

**ROBERT D. HANSON**

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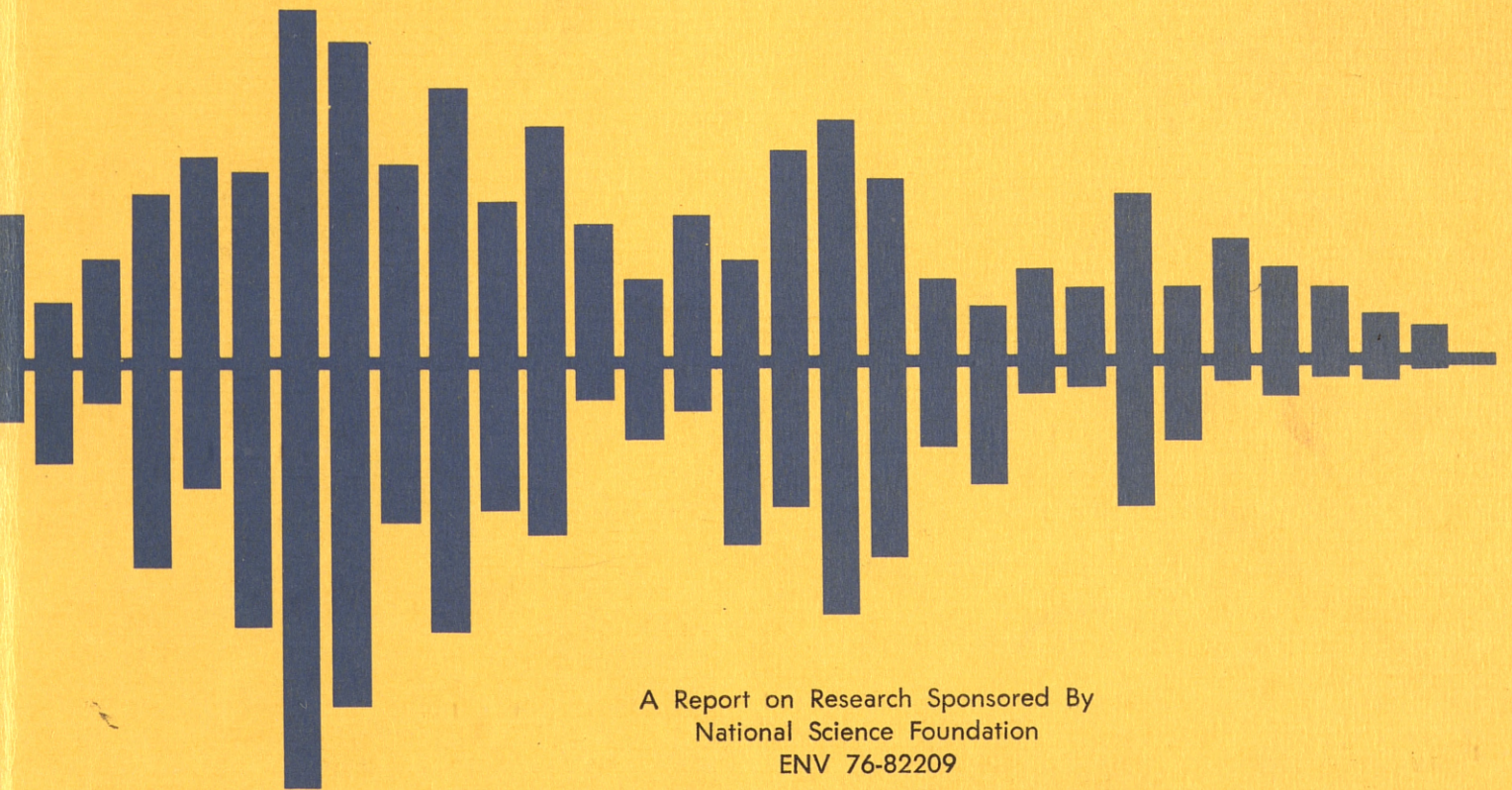
# **HYSTERESIS MODELS FOR STEEL MEMBERS SUBJECTED TO CYCLIC BUCKLING OR CYCLIC END MOMENTS AND BUCKLING (USER'S GUIDE FOR DRAIN-2D:EL9 AND EL10)**

**Ashok K. Jain**

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**The University of Michigan**

**Department of Civil Engineering**



A Report on Research Sponsored By  
National Science Foundation  
ENV 76-82209

American Iron and Steel Institute  
Project 301



HYSTERESIS MODELS FOR STEEL MEMBERS SUBJECTED TO  
CYCLIC BUCKLING

OR

CYCLIC END MOMENTS AND BUCKLING  
(USER'S GUIDE FOR DRAIN-2D:EL9 AND EL10)

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## ABSTRACT

Elements EL9 and EL10 are general purpose programs for steel members subjected to cyclic buckling, or cyclic end moment-buckling, respectively. These elements are developed for use with DRAIN-2D computer program. This manual describes the essential features of these two new elements along with their FORTRAN listing. The development of axial load-axial displacement hysteresis model as used for these elements has been described in a previous report.

## ACKNOWLEDGEMENTS

The computer programs for the Buckling Element (EL9) and End Moment-Buckling Element (EL10) have been written on the basis of analytical and experimental research conducted at the University of Michigan over the period of past ten years. The financial assistance for the current effort was provided by the National Science Foundation, Grant ENV 76-82209 and the American Iron and Steel Institute, Project 301. Constructive suggestions from American Iron and Steel Institute Task Force on Project 301, Earthquake Resistant Design of Braced Steel Frame Structures, consisting of A. L. Collin, L. W. Lu, M. H. Mark, W. A. Milek, Jr., D. R. Sherman, E. P. Popov and L. A. Wyllie, Jr., were very helpful. Special thanks are due to Professor R. D. Hanson for introducing the idea of writing subroutine VRTX9. The authors are also thankful to Dr. J. B. Scalzi of NSF and Mr. A. C. Kuentz of AISI for the encouragement they provided in carrying out this research.

For compatibility and consistency, the format of presentation of this manual has been styled after the other DRAIN-2D elements developed by A. E. Kanaan, G. H. Powell and Pritam Singh.

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## CHAPTER 1

### INTRODUCTION

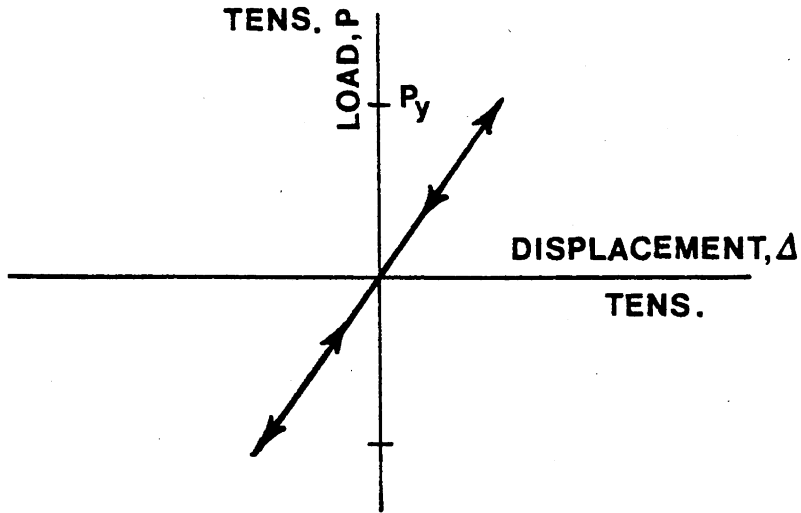
Tall braced frame structures are constructed in seismically active regions throughout the world. Such frames are generally more efficient in terms of lateral stiffness per unit volume of material than open moment resisting steel frames. American Petroleum Institute Code (API-RP2A, Ref. 1) now contains strength and ductility requirements for offshore braced steel platforms. The strength requirements insure that the structure is adequately sized for strength and stiffness to maintain all nominal stresses within yield or buckling for the level of earthquake activity which is normally expected during the life of the structure. The ductility requirements are intended to insure that the structure has sufficient energy absorption capacity to prevent its collapse during rare intense earthquake motions. Bracing members are considered effective earthquake resistant elements as they help satisfy the above two requirements when used in a frame.

Different member arrangements and proportions are used in braced frames (3,4,5). Bracing arrangements may be either concentric or eccentric type. The connections of bracing members may be designed as simple or moment resistant. The members in the former situation are generally treated as primarily axially loaded, whereas, the

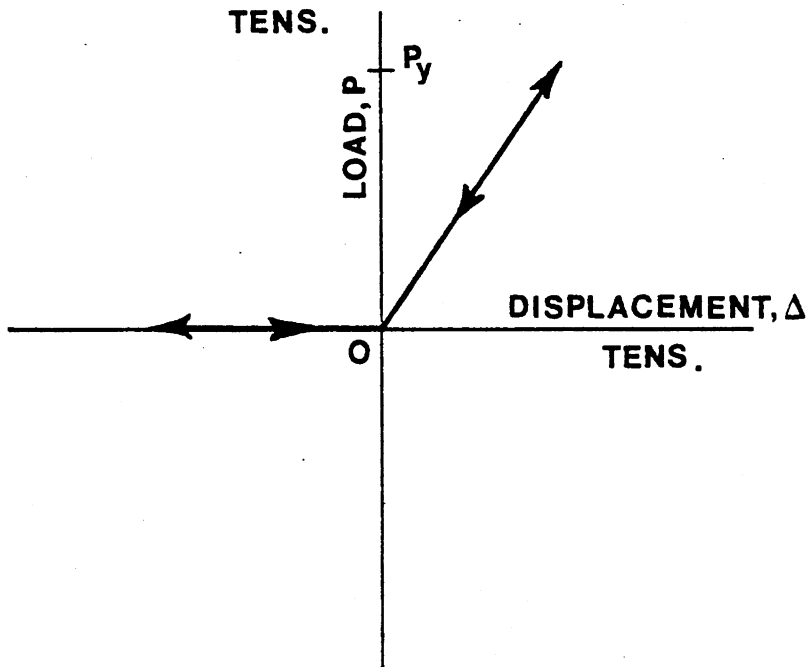
latter type may develop significant end moments.

In past studies of braced frames, the hysteresis behavior of primarily axially loaded bracing members has been modeled in one of several ways, such as: elastic in tension and compression (Figure 1a), tension-only elastic model (Figure 1b), tension-yield and compression-yield (Figure 1c), or tension-yield and compression-buckling (Figure 1d). These models neglected the energy dissipation characteristics of bracing members in the post-buckling range. Later, Higginbotham and Hanson (Ref. 2, Figure 1e), Nilforoushan (Ref. 12, Figure 1f), Prathuangsit (Ref. 14), Singh (Ref. 15, Figure 1g), Wakabayashi (Ref. 17), and Marshall (Ref. 10, Figure 1h) presented hysteresis models which represented the post-buckling behavior of bracing members in a more realistic manner. Experimental studies (3,8) on small specimens have pointed out two significant characteristics in the hysteresis behavior which were not included in these analytical models. These characteristics are: increase in member length and reduction in compressive strength with number of cycles. Jain (Ref. 3, Figure 2) presented a hysteresis model which accounts for these two parameters. Minor changes have been made in this model and the latest version is described in Chapter 2 of this manual. This model is called as Buckling Element (EL9) for use in DRAIN-2D Computer program (9).

There is one model available for primarily end

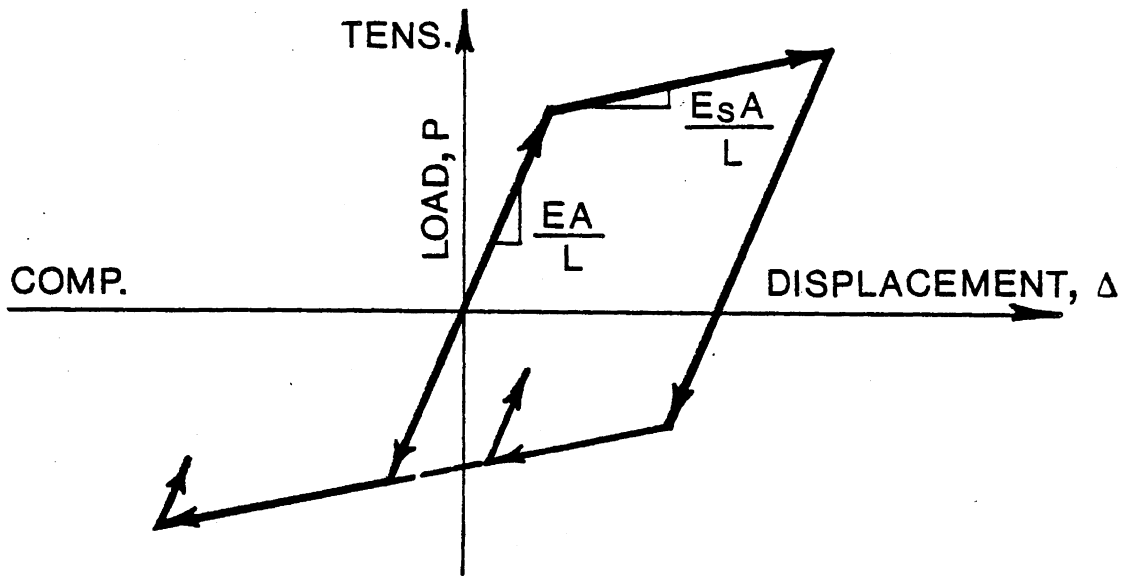


(a) Elastic Tension — Compression Model

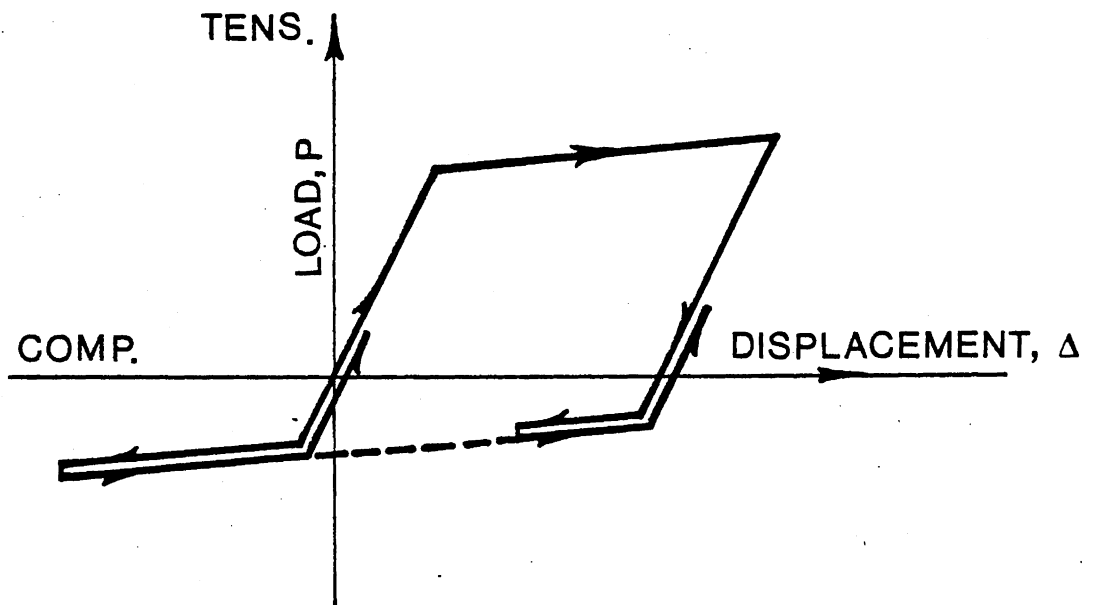


(b) Elastic Tension — Only Model

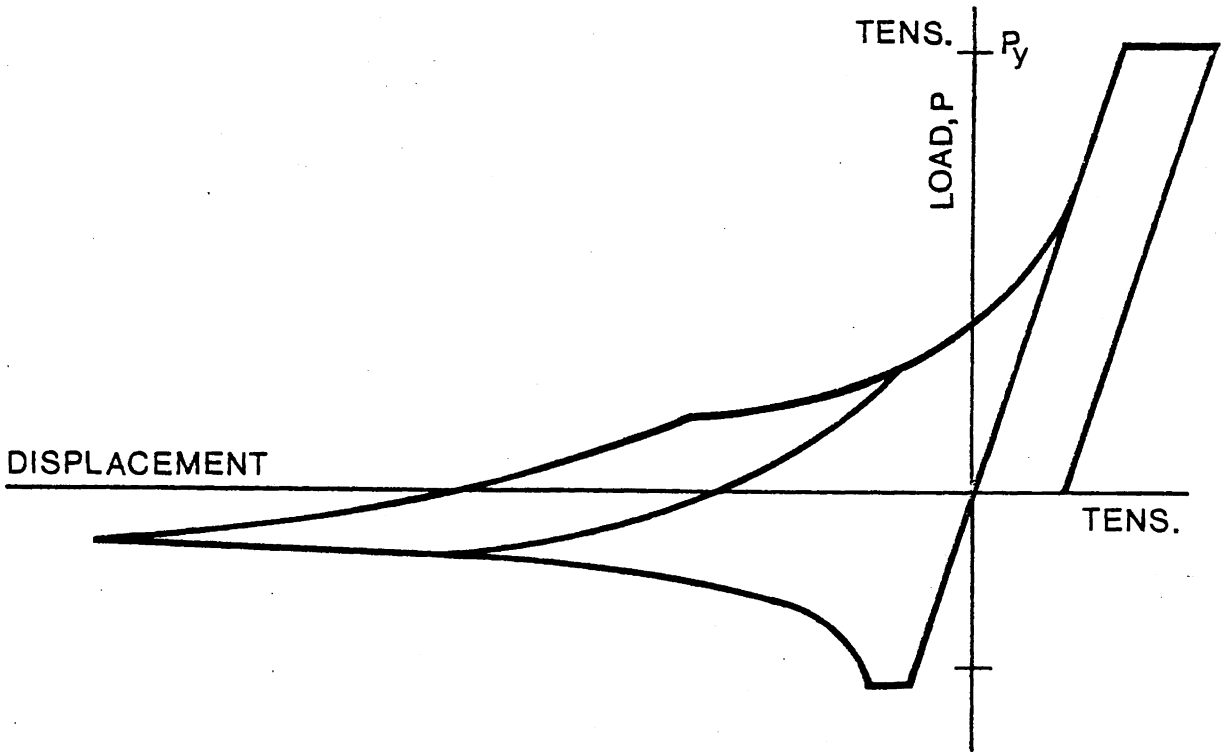
Figure 1 - Axial Load - Displacement Behavior



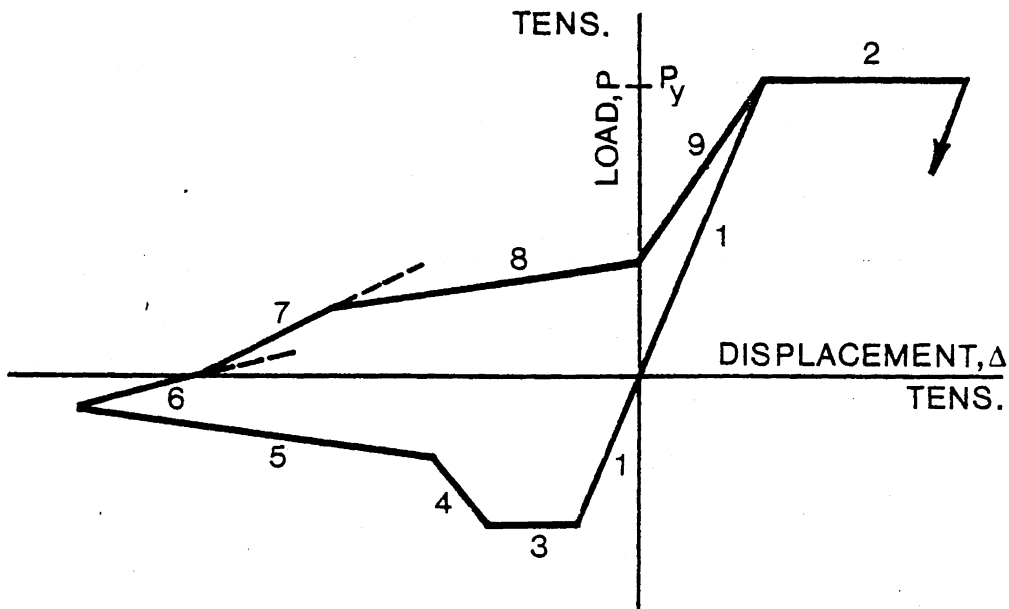
(c) Yield in Tension and Compression



(d) Yield in Tension, Buckling in Compression

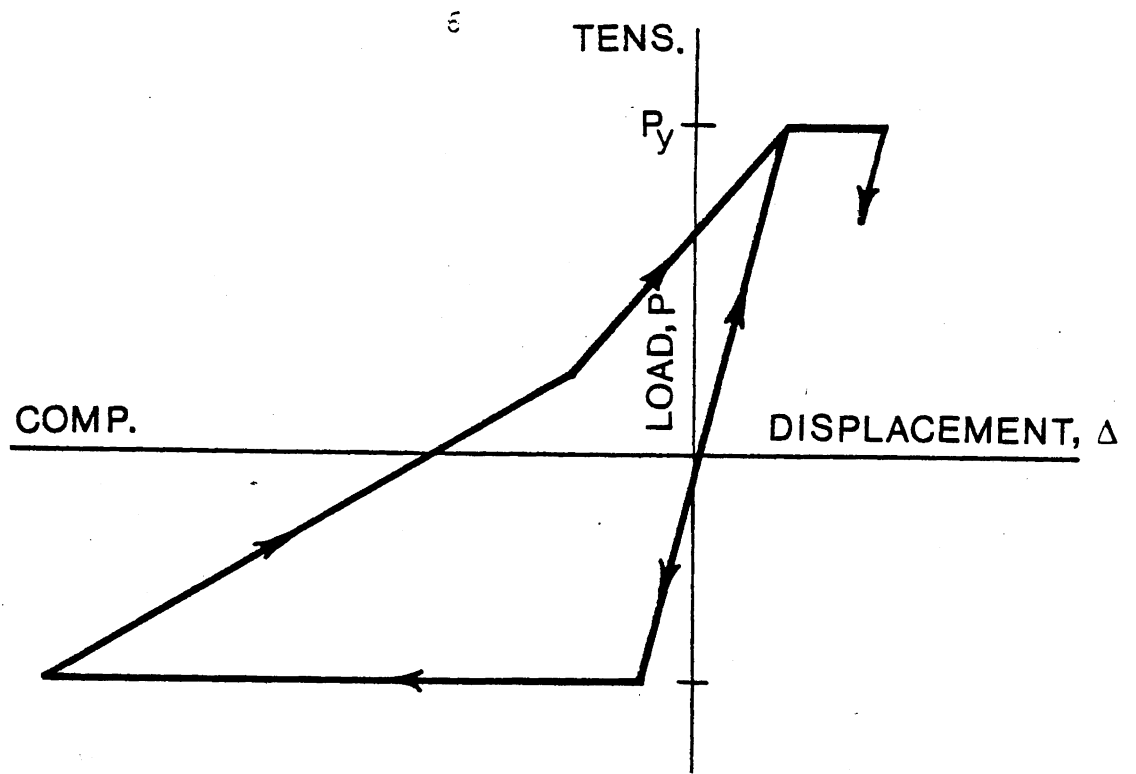


(e) HIGGINBOTHAM AND HANSON, Ref. 2

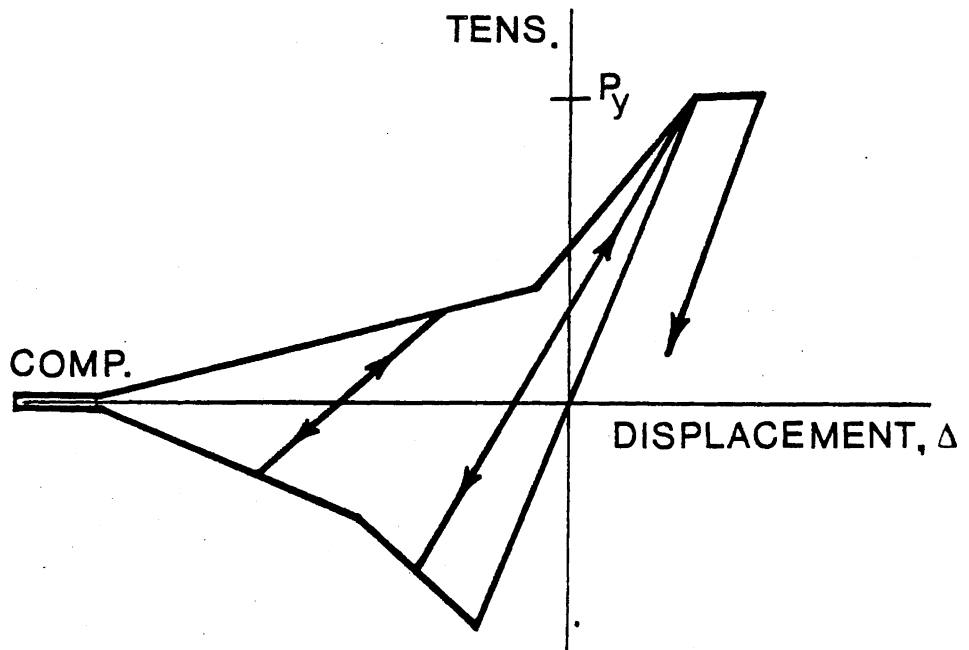


(f) NILFOROUSHAN, Ref. 12

Figure 1 - Axial Load-Displacement Behavior (cont.)



(g) Singh, Ref. 15



(h) Marshall, Ref. 10

Figure 1 - Axial Load-Displacement Behavior (cont.)

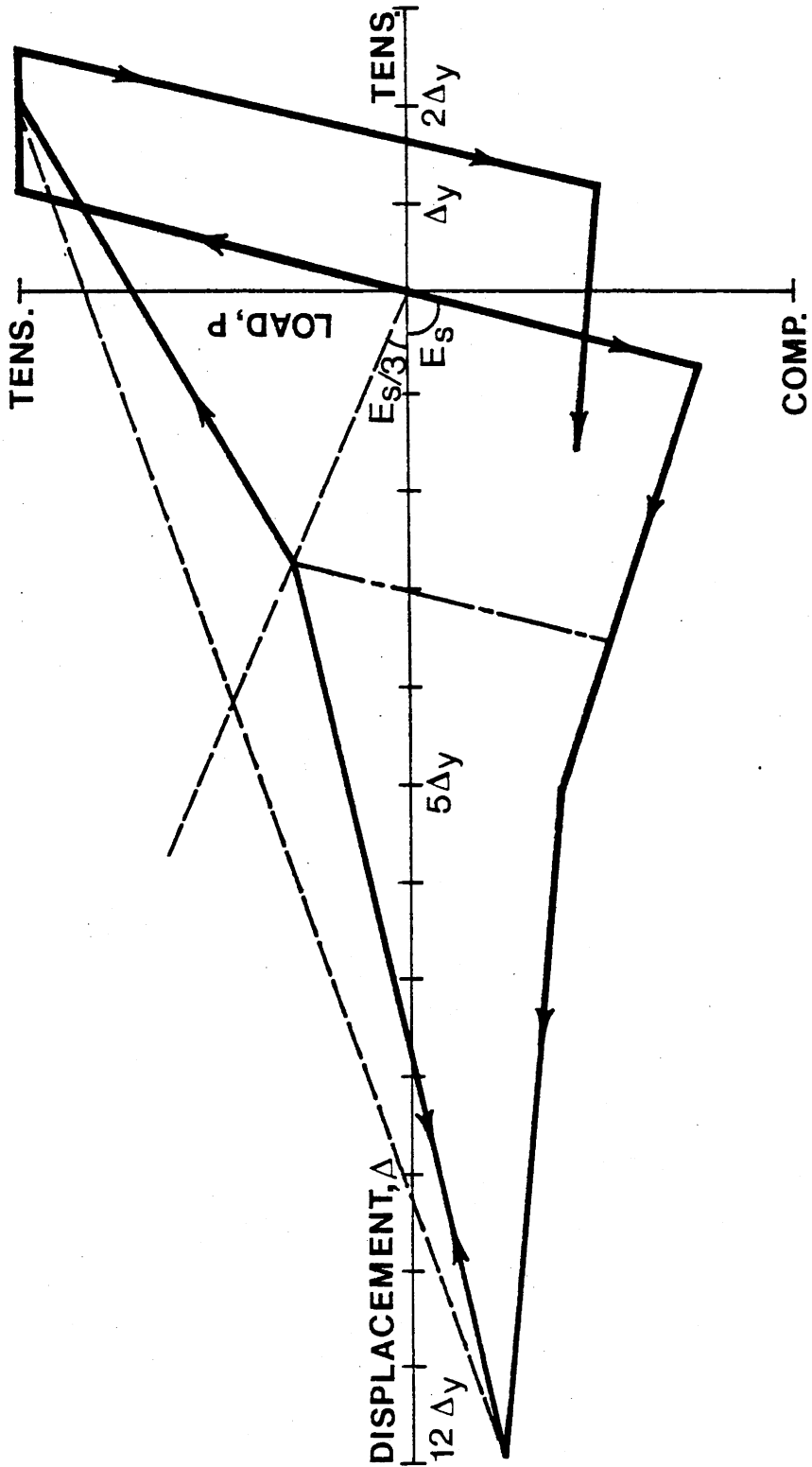


Figure 2 - Axial Load-Displacement Behavior, Ref. 3

moment resisting members and is known as beam-column element (Ref. 9, Figure 3). This model does not consider buckling and, therefore, should be used for full moment connected (or rigid-connected) non-buckling type bracing or column members.

Jain (3) analyzed 18 concentrically braced (X and K) and eccentrically braced (open or split K) frames under monotonic elastic, monotonic inelastic and dynamic loading conditions. The purpose of this analysis was (i) to determine the situations in which end moments dominate over axial forces in bracing members and vice versa, and (ii) to develop an understanding of the inelastic dynamic response of these frames with different member proportions. It was concluded that there is a need to develop a hysteresis model for rigid-connected buckling type steel members. Such a model has been developed by combining buckling element (EL9) and beam-column element (EL2) and is described in Chapter 3 of this manual. This model is called as End Moment-Buckling Element (EL10) for use in DRAIN-2D computer program (9).

#### DRAIN-2D COMPUTER PROGRAM

DRAIN-2D is a general purpose computer program for the inelastic response of plane structures subjected to earthquake forces, and was developed by Kannan and Powell (9). The program concepts and features are described in Reference 9. User's Guide (13) describes the extensions made to the program and presents input data procedures.



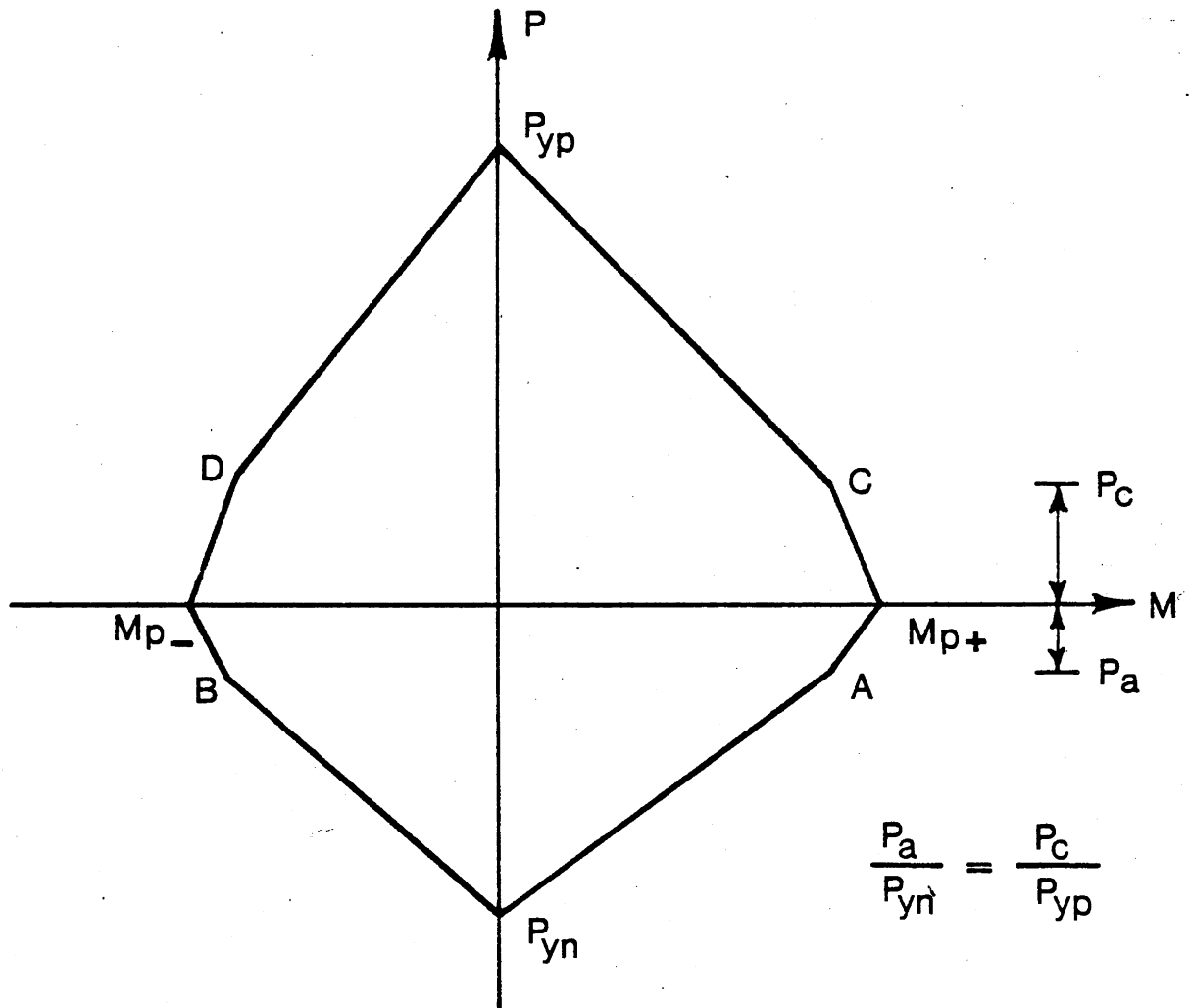


Figure 3 - Beam-Column Model, Ref. 9

This manual supplements References 3, 9 and 13, and should be used in conjunction with them. For compatibility the format of presentation in this manual has also been styled after these references. The procedure followed in adding the new elements EL9 and EL10 to the DRAIN-2D conforms to Chapter 4 of Reference 9. The four main subroutines developed for each element are as follows. The number at the end of the subroutine name corresponds to the element type.

1. INEL9, INEL10: Input and initialization of element data.
2. STIF9, STIF10: Calculation of element tangent stiffness at different time steps.
3. RESP9, RESP10: Determination of increments of element deformations (strains) and forces (stresses), determination of yield status, and output of time history results. This may be called as "state determination phase".
4. OUT9, OUT10: Output of final envelope values for element deformations and forces.

This arrangement is used in DRAIN-2D program and is taken directly from it. The variable names have been kept the same as for other elements (9,15). FORTRAN listing for elements EL9 and EL10 are given in Appendices A-1 and A-2, respectively. Several COMMENT statements are given for understanding the underlying logic. These programs have been used on AMDAHL 470V/6 computer at the University

of Michigan using MTS. It is believed that they can be easily used on other systems.

If the user has other element subroutines which are also called either EL9 or EL10, then the suffix 9 or 10 from all the subroutines of these elements including CALL statements should be changed. The new suffix should be less than 10, otherwise, significant additions and changes would have to be made in the main DRAIN-2D program (Cards B to AB) in order to accomodate more than ten elements.

## CHAPTER 2

### BUCKLING ELEMENT (EL9)

Singh (15) presented a multilinear hysteresis model (EL7) for axially loaded pin-ended bracing members and used in the seismic analysis of multistory braced frames. Jain, Goel and Hanson (7) compared their experimental hysteresis curves with analytical curves obtained by using Singh's model and suggested that this model could be improved if modifications were made in compression envelope and tension envelope regions to include the change in compression loads and residual elongation. The new buckling model accounts for these two parameters, yet, retains the simplicity of Singh's model.

The buckling model is described in the following section. Tension load and displacement are taken as positive, and compression load and displacement are taken as negative.

#### GENERAL CHARACTERISTICS

Assuming that an initially straight member is loaded first in tension, the member follows segment AE elastically as shown in Figure 4a (computer print out code for this segment = 0). The member yields at E and follows segment EE' (Code = 9). If the direction of displacement is reversed at E', the member unloads elastically, parallel to the initial elastic slope AE (code = 0). Continued compression will result in the first buckling of the member

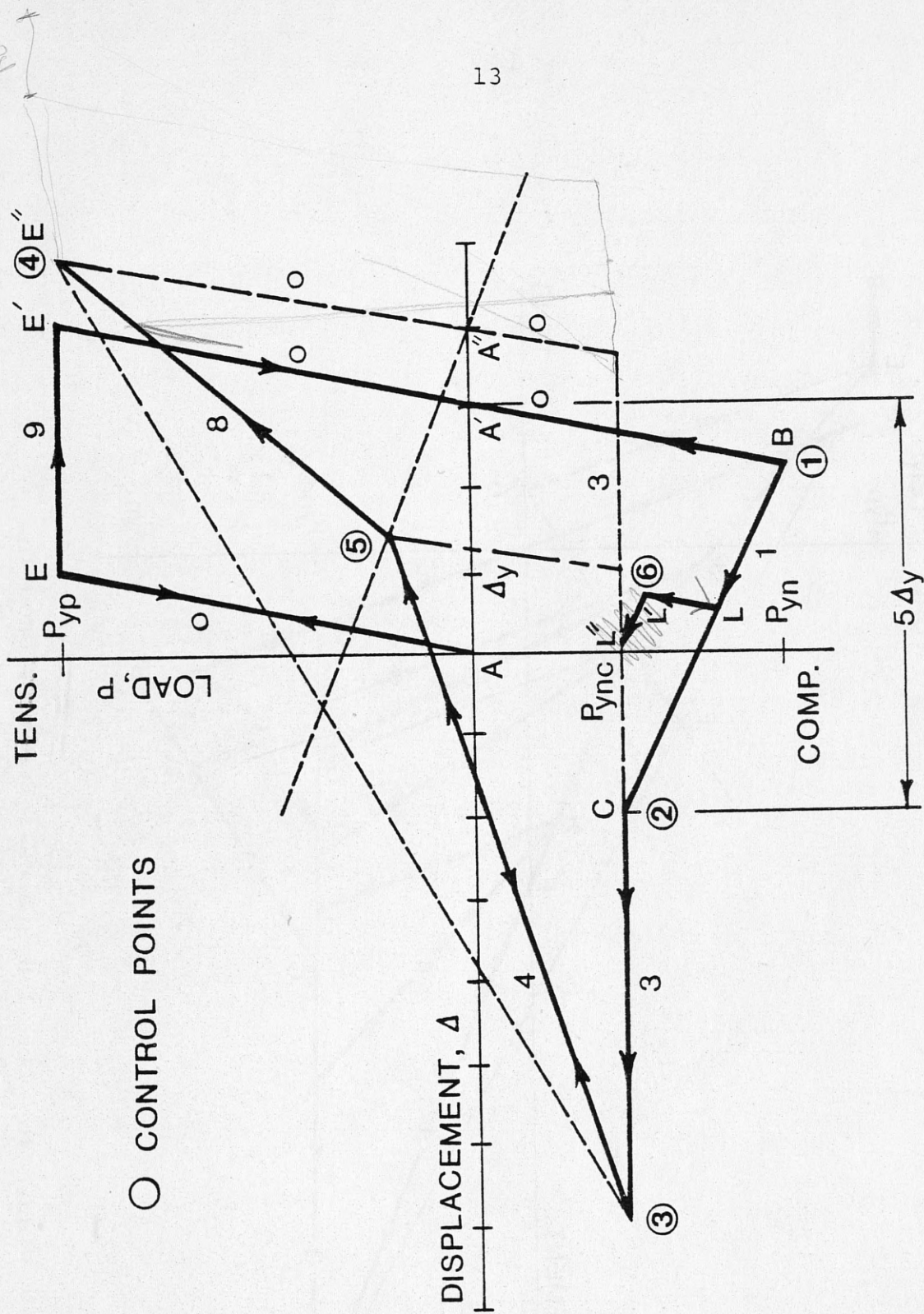


Figure 4a - Axial Hysteresis Behavior Used in Buckling Model

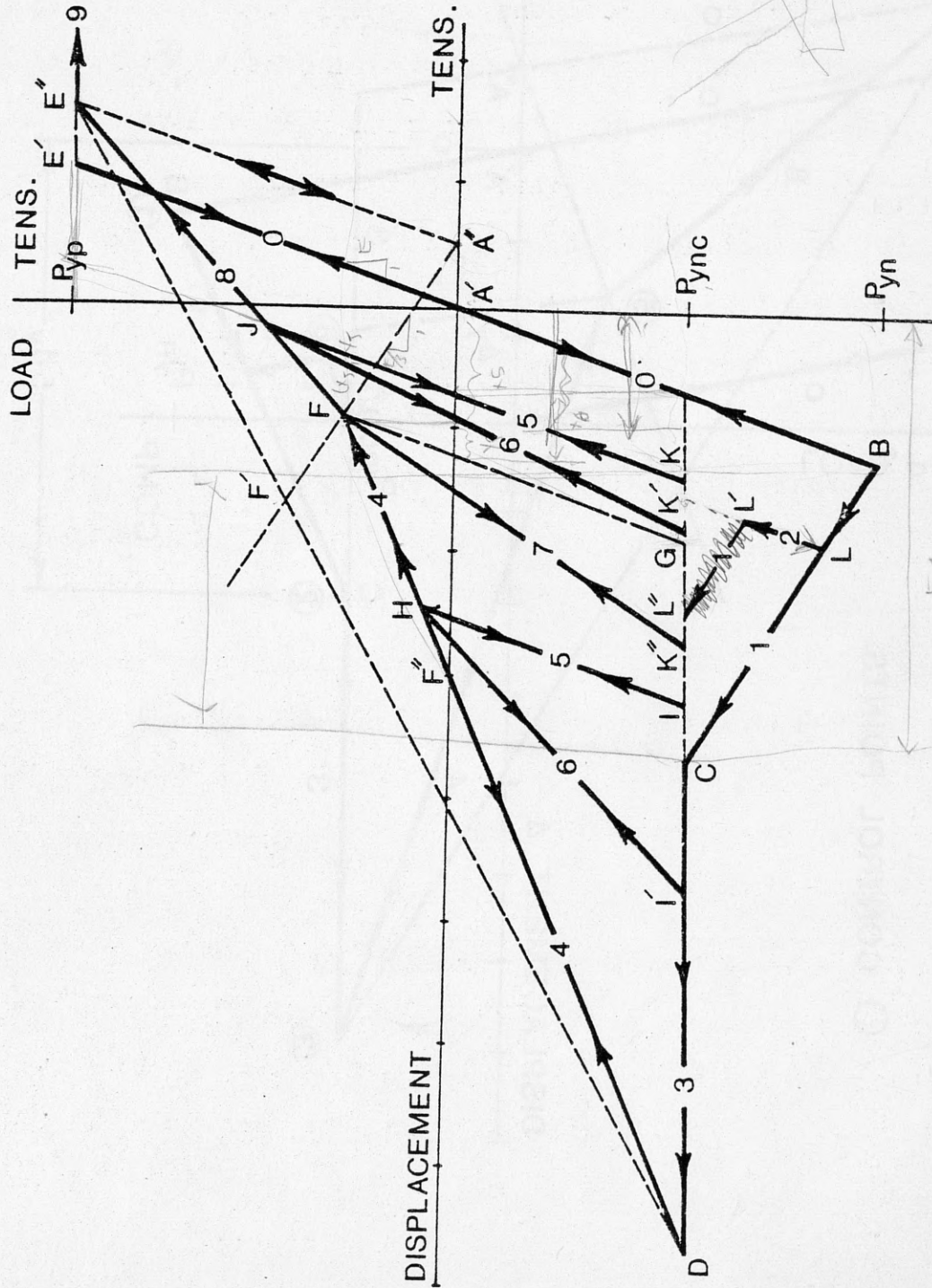


Figure 4b - Axial Hysteresis Behavior Used in Buckling Model (cont.)

at point B. The load at point B corresponds to the first cycle buckling load  $P_{yn}$  for the member which is significantly higher than the buckling load  $P_{ync}$  used for subsequent cycles. After buckling at point B the member follows segment BC (code = 1). The point C corresponds to a compression displacement equal to five times the tension yield displacement  $\Delta y$  of the member (Ref. 3). If the direction of axial displacement is reversed at L, the member follows segment LL' (code = 2), parallel to the initial elastic slope AB until it hits the post-buckling load level,  $P_{ync}$ . However, if the direction of axial displacement is reversed at L', the member follows segment L'L" (code = 1) which is parallel to segment BC. L" lies at the  $P_{ync}$  load level.

Once the member hits the post-buckling load level  $P_{ync}$ , it comes out of Subroutine VRTX9 and reenters into the main Subroutine RESP9 for further state determination (Figure 4b).

From point C or L", continued compression results in segment CD or L"D, respectively (code = 3). If the direction of axial displacement is reversed at D, it results in compression load decreasing to zero and followed by an increasing tensile load along the path DFE" (code = 4 for segment DF and code = 8 for segment FE").

To locate the point F, a line A"F' is drawn from the new origin A" ( $AA'' = EE''$ ) at a slope of  $1/3$  times the

initial elastic slope  $\overline{AE'}$  or  $\overline{AB}$ , which intersects the line  $DE''$  at  $F'$ . Intercept  $A''F$  is taken as  $60/(KL/r)$  times the distance  $A''F'$ .  $EE''$  is the residual elongation term and is calculated from equation 2.1. This equation was developed from the data obtained on small square steel tubes in Reference 3. The coefficients used in this equation can be changed very easily, if program user so desires. One card for RE (residual elongation expressed in terms of residual strain) should be changed in each of the Subroutines RESP9 and LAW9.

$$RE = 0.0175 * (0.55 * X3RE / SLEND + 0.0002 * X3RE ** 2) \quad 2.1$$

The line  $FG$  drawn parallel to the initial elastic slope is used to distinguish the loading history in the region  $DFG$  from that in the region  $A''E''FG$ . If the direction of axial displacement is changed along  $DF''$  the member follows the same path (code = 4). If the member reverses at point  $H$ , it follows segment  $HI$  (code = 5) which is parallel to segment  $\overline{AB}$ . Continued compression results in segment  $II'$  (code = 3). If the direction of displacement is again reversed at  $I'$ , the member follows segment  $I'H$  (code = 6). If the direction of axial displacement is reversed at  $J$ , the member follows segment  $JK$  (code = 5). Continued compression results in segment  $KK'$  (code = 3) or in segment  $KK''$  (code = 3). If reversal occurs at  $K'$ , the member follows segment  $K'J$  (code = 6). If member surpasses point  $G$  and returns from  $K''$ , it follows  $K''F$



$$KE = \left[ \frac{0.55 \Delta}{KL/r} + 0.0002 \Delta^2 \right]$$

(code = 7) and then FE" (code = 8). New control points are determined if the member surpasses points D or E" during a cycle. In subsequent cycles the maximum compression load is  $P_{ync}$ .

The load  $P_{ync}$  is input as a fraction of maximum compressive load  $P_{yn}$  in the first cycle. The load  $P_{yn}$  can be calculated by using AISC equations (3,16). Data obtained in Reference 3 suggests that the strength reduction factor PHI ( $\phi = P_{ync}/P_{yn}$ ) can be approximated as 0.3-0.6 depending upon effective slenderness ratio of the member.

#### ELEMENT DEFORMATIONS

The buckling element has four degrees of freedom. The only deformation to be considered is its axial extension. The displacement transformation relating increments of deformation and displacement (Figure 5) is:

$$dv = \begin{bmatrix} -\frac{X}{L} & -\frac{Y}{L} & \frac{X}{L} & \frac{Y}{L} \end{bmatrix} \begin{Bmatrix} dr_1 \\ dr_2 \\ dr_3 \\ dr_4 \end{Bmatrix} \quad 2.2$$

$$\text{or, } \{dv\} = [a] \{dr\} \quad 2.3$$

Large displacement effects are not taken into account. X, Y and L are assumed to remain constant and the displacement transformation matrix also remains constant.

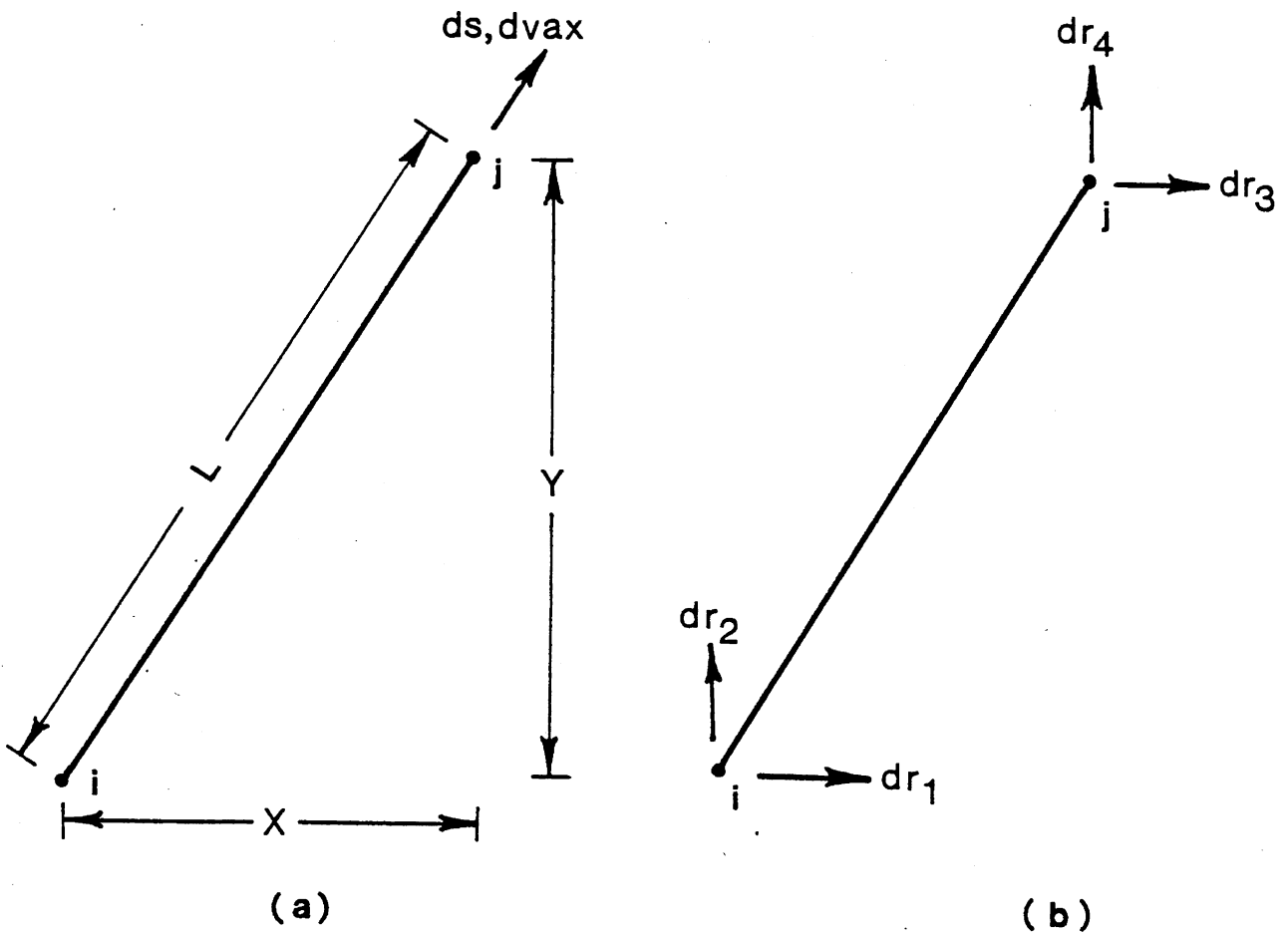


Figure 5 - Deformations and Displacements

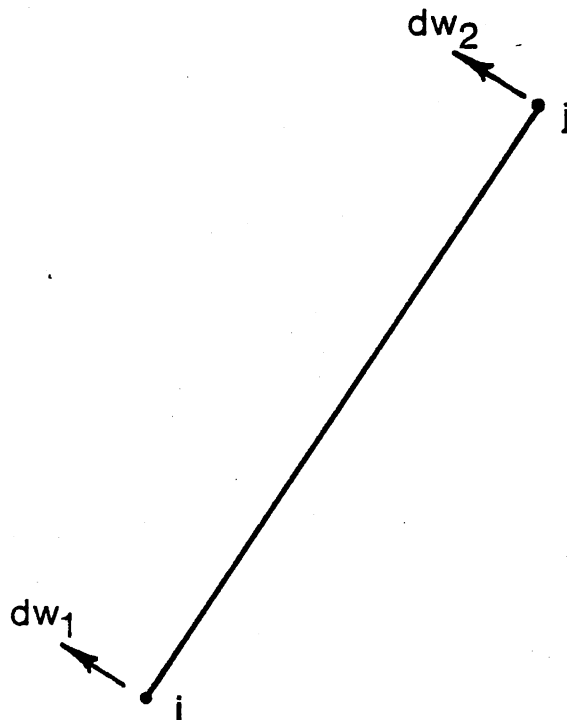


Figure 6 - Displacement for Geometric Stiffness

ELEMENT STIFFNESS

The tangent stiffness in term of deformations is

$$dS = \frac{E_T A}{L} dv \quad 2.4$$

$$\text{or, } \{dS\} = [k_T] \{dv\} \quad 2.5$$

where,  $E_T$  = tangent modulus in current state, and  $A$  = element cross sectional area.

The tangent stiffness in terms of nodal displacements is

$$[K_T] = [a]^T [k_T] [a] \quad 2.6$$

where,  $[a]$  is given by equations 2.2 and 2.3.

The geometric stiffness in the element coordinates  $dw_1$  and  $dw_2$  is (Figure 6):

$$[k_G] = \frac{S}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad 2.7$$

or, in terms of nodal displacements

$$[K_G] = [a_1]^T [k_G] [a_1] \quad 2.8$$

where,  $[a_1]$  is given by

$$\begin{Bmatrix} dw_1 \\ dw_2 \end{Bmatrix} = \begin{bmatrix} -\frac{Y}{L} & \frac{X}{L} & 0 & 0 \\ 0 & 0 & -\frac{Y}{L} & \frac{X}{L} \end{bmatrix} \begin{Bmatrix} dr_1 \\ dr_2 \\ dr_3 \\ dr_4 \end{Bmatrix} \\ = [a_1] \{dr\} \quad 2.9$$

### FIXED END AND INITIAL FORCES

The effects of static loads applied along the element length rather than at the nodes can be taken into account by specifying fixed end force patterns. Static thermal effects can also be considered in the same way. The forces to be specified are the forces on the element ends required to prevent them from displacing, with the sign convention shown in Figure 7. If axial forces having different magnitudes at ends  $i$  and  $j$  are specified, the average value is assumed for determining the yield status of the element and for computing the geometric stiffness.

Elements may be stressed under static load but it may be incorrect or inconvenient to determine the element forces by applying static loads to the structure. To allow for such cases, provision is made for initial forces to be specified in the elements. These forces will typically be the forces in the elements under static loading as calculated by a separate analysis. For consistency, they should be in equilibrium with the static load producing them, but this is not essential. The computer program does not make corrections for any equilibrium unbalance resulting from the specification of initial forces.

To satisfy the requirement that the structure remain elastic under static loading, the initial element forces should be less than the yield strengths of the element. If desired, static loads as well as initial forces may be specified. The element forces will then be the sum of the

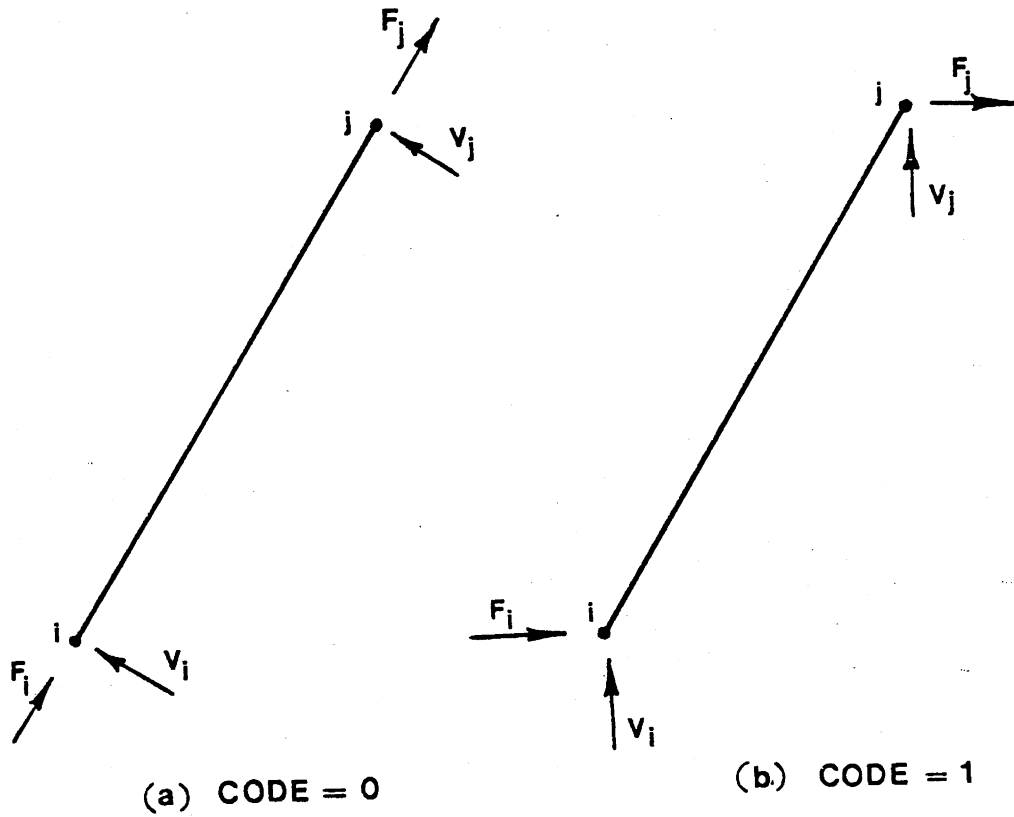


Figure 7 - End Clamping and Initial Forces

initial forces and those due to the static loads. The geometric stiffness effect is not included in the static analysis.

#### OUTPUT RESULTS

The following results are printed for the static loading condition (time = 0) and at each output time if a time history is requested. The static results are output for all elements, and the time history results for only those elements for which time histories are requested.

1. Yield code: 0 to 9 as explained earlier in Figures 4a and 4b.
2. Axial force, tension positive.
3. Net axial extension.
4. Accumulated positive and negative plastic extensions up to the current time.

These accumulated deformations are computed by accumulating the plastic extensions during all positive and negative plastic excursions. These accumulated deformations, together with the maximum positive and negative total extensions, provide information on the amount of plastic deformation imposed on the element. The maximum positive and negative values of axial force, maximum positive and negative extension and accumulated plastic extension are printed at the time intervals requested for results envelopes. The times at which the maximum forces and extensions were produced are also printed.

INPUT DATA PREPARATIONE9. BUCKLING ELEMENTS - EL9

Number of words of information per element = 53.

## E9(a). CONTROL INFORMATION FOR GROUP (4I5) - ONE CARD

Columns            5: Punch 9 (to indicate that group consists of buckling elements).

                  6 - 10: Number of elements in group.

                  11 - 15: Number of different element stiffness types (max. 40). See Section E9(b).

                  16 - 20: Number of different fixed end force patterns (max. 40). See Section E9(c).

## E9(b). STIFFNESS TYPES (I5, 7F10.0) - ONE CARD FOR EACH STIFFNESS TYPE

Columns            1 - 5: Stiffness type number, in sequence beginning with 1.

                  6 - 15: Young's modulus of elasticity.

                  16 - 25: Average cross sectional area.

                  26 - 35: Tension yield force,  $P_{yp}$

                  36 - 45: Compression yield force,  $P_{yn}$  (first cycle)

                  46 - 55: Radius of gyration

                  56 - 65: Effective length factor

                  66 - 75: Strength reduction factor, PHI

E9(c). FIXED END FORCE PATTERNS (2I5, 4F10.0) - ONE CARD FOR EACH  
FIXED END FORCE PATTERN

Omit if there are no fixed end forces. See Figure 7.

Columns            1 - 5: Pattern number, in sequence beginning with 1.

                  10: Axis code, as follows:

Code = 0: Forces are in the element coordinate system, as in Figure 7a.

Code = 1: Forces are in the global coordinate system, as in Figure 7b.

Columns 11 - 20: Clamping force  $F_i$ .  
 21 - 30: Clamping force  $V_i$ .  
 31 - 40: Clamping force  $F_j$ .  
 41 - 50: Clamping force  $V_j$ .

E9(d). ELEMENT GENERATION COMMANDS (9I5, 2F5.0, F10.0) - ONE CARD FOR EACH GENERATION COMMAND

Elements must be specified in increasing numerical order.

Cards for the first and last elements must be included. See Note 7 of User's Guide (13) for explanation of generation procedure.

Columns 1 - 5: Element number, or number of first element in a sequentially numbered series of elements to be generated by this command.

6 - 10: Node number of element end  $i$ .

11 - 15: Node number of element end  $j$ .

16 - 20: Node number increment for element generation. If zero or blank, assumed to be equal to 1.

21 - 25: Stiffness type number.

30: Code for including geometric stiffness. Punch 1 if geometric stiffness is to be included. Leave blank or punch zero if geometric stiffness is to be ignored.

35: Time history output code. If a time history



of element results is not required for the elements covered by this command, punch zero or leave blank. If a time history printout, at the intervals specified on card D1, is required, punch 1.

## Columns

- 36 - 40: Fixed end force pattern number for static dead loads on element. Leave blank if there are no dead loads. See note below.
- 41 - 45: Fixed end force pattern number for static live load on element. Leave blank if there are no live loads.
- 46 - 50: Scale factor to be applied to fixed end forces due to static dead loads. Leave blank if there are no dead loads.
- 51 - 55: Scale factor to be applied to fixed end forces due to static live loads. Leave blank if there are no live loads.
- 56 - 65: Initial axial force on element, tension positive.

NOTE: If static load code, Card C1, is zero but fixed end forces are still specified for some elements, an inconsistency results. In effect any such fixed end forces will be treated as initial element forces.

## CHAPTER 3

### END MOMENT-BUCKLING ELEMENT (EL10)

The end moment-buckling element is a combination of beam-column element (EL2) and buckling element (EL9). This element considers the interaction between the end moments and axial force in the beam-column component EL2, the axial force being determined by the buckling element EL9. In this formulation, the flexural stiffness is assumed to be independent of the axial force. Workman (18) studied the influence of axial force-flexural stiffness interaction in the elastic state on the seismic response of braced steel frames. He concluded that the effect of this interaction was not significant for the structural response. Nigam (11) proposed a more consistent procedure for considering the interaction between forces existing at sections where yielding occurs, but it is very complex and, therefore, not considered for this interactive element EL10. It is believed that the axial force-end moment interaction as modeled herein should be adequate for practical applications.

#### GENERAL CHARACTERISTICS

End moment-buckling element has six degrees of freedom and may be arbitrarily oriented in the X-Y plane. The element possess axial and flexural stiffnesses. Variable cross-sections can be considered by specifying average area and appropriate flexural stiffness

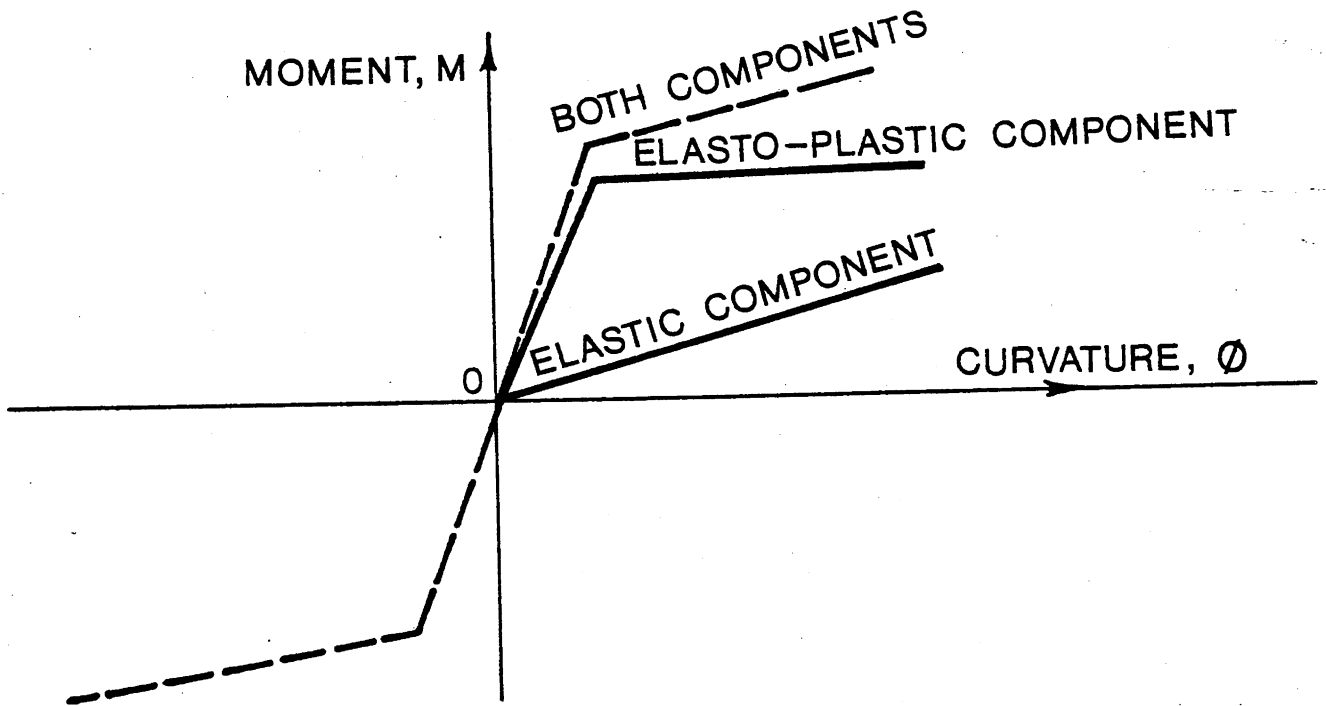


Figure 8 - Moment-Curvature Relation

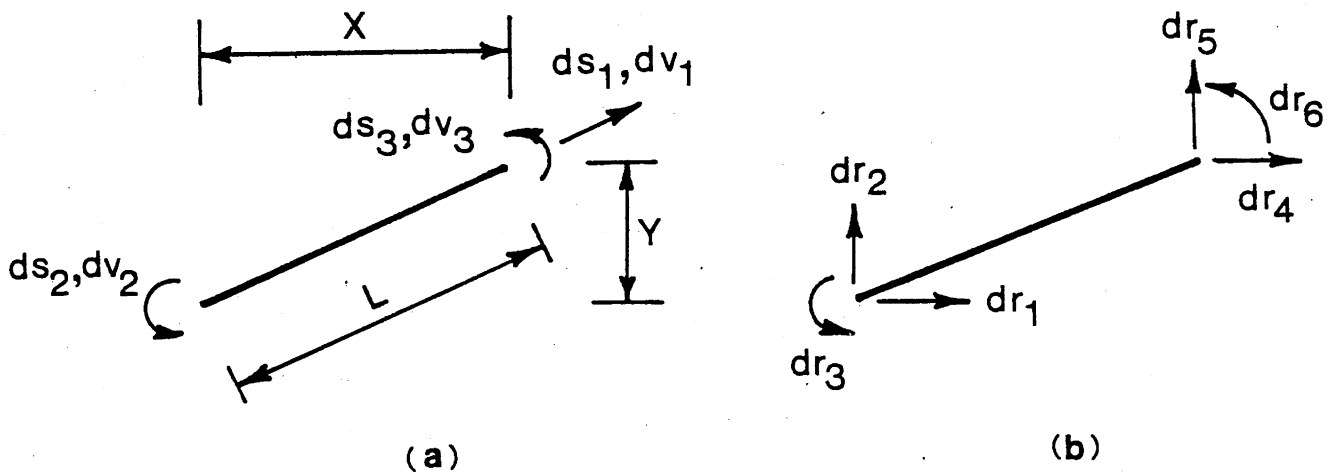


Figure 9 - Deformations and Displacements

coefficients. Flexural shear deformations can also be taken into account.

Strain hardening is considered in the moment-rotation relationship but not in the axial force-axial displacement relationship. Strain hardening is approximated by assuming that the element consists of elastic and elasto-plastic components in parallel as shown in Figure 8. Flexural yielding may take place only in concentrated plastic hinges at the ends of the element. The plastic hinges in the elasto-plastic component rotate under constant moment, but the moment in the elastic component may continue to increase.

The plastic moment capacities may be specified to be different at the two ends of an element and also for positive and negative bending at each end. If tension yield and compression strengths are different at the two ends of an element, minimum values of tension yield and compression strengths are used.

Static loads applied along any element length may be taken into account by specifying fixed end force values. The results of separate static load analyses can be imposed by specifying initial force values.

Large displacement effects may be approximated in the dynamic analysis by including simple geometric stiffnesses based on the element axial forces under static load.

#### ELEMENT DEFORMATIONS

An end moment-buckling element has three modes of

deformation, namely axial extension, flexural rotation at end i, and flexural rotation at end j. The displacement transformation relating increments of deformation and displacement (Figure 9) is:

$$\begin{Bmatrix} dv_1 \\ dv_2 \\ dv_3 \end{Bmatrix} = \begin{bmatrix} -X/L & -Y/L & 0 & X/L & Y/L & 0 \\ -Y/L^2 & X/L^2 & 1 & Y/L^2 & -X/L^2 & 0 \\ -Y/L^2 & X/L^2 & 0 & Y/L^2 & -X/L^2 & 1 \end{bmatrix} \begin{Bmatrix} dr_1 \\ dr_2 \\ dr_3 \\ dr_4 \\ dr_5 \\ dr_6 \end{Bmatrix} \quad 3.1$$

$$\text{or, } \{dv\} = [a] \{dr\} \quad 3.2$$

As for the buckling element, X, Y and L are assumed to remain constant.

A plastic hinge forms when the moment in the elasto-plastic component of the element reaches its plastic moment. A hinge is then introduced into this component, the elastic component remaining unchanged. The measure of flexural plastic deformation is the plastic hinge rotation.

For any increments of total flexural rotation,  $dv_2$  and  $dv_3$ , the corresponding increments of plastic hinge rotation,  $dv_{p2}$  and  $dv_{p3}$ , are given by

$$\begin{Bmatrix} dv_{p2} \\ dv_{p3} \end{Bmatrix} = \begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{Bmatrix} dv_2 \\ dv_3 \end{Bmatrix} \quad 3.3$$

where, A, B, C and D are given in Table 1.

Unloading occurs at a hinge when the increment in hinge rotation is opposite in sign to that of the bending moment.

Inelastic axial deformations obey the same hysteresis law as the Buckling Element EL9 does (Figures 4a and 4b).

#### INTERACTION SURFACES

The End Moment-Buckling Element uses two types of interaction surfaces. For axial force-axial displacement interaction, it uses the same as used by Buckling Element EL9. For axial force-end moment interaction it uses the envelope as shown in Figure 10.

Knowing the axial deformations, the program first determines the axial state of the element as for EL9. It calculates the axial force and the unbalanced axial force, if any. Then, it calculates the yield moment by using the axial force-moment interaction curve as for beam-column element EL2. If the moment lies on or outside the surface, a plastic hinge is introduced at that end. Combinations outside the yield surface are permitted only temporarily, being compensated for by applying corrective loads in the succeeding time step (Figures 11a and 11b).

Once the axial load in the post-buckling range becomes equal to  $P_{ync}$ , the program redefines the four branches of M-P interaction curve in the compression region as shown in Figure 10. Maximum compressive strength of the member for all subsequent cycles remains

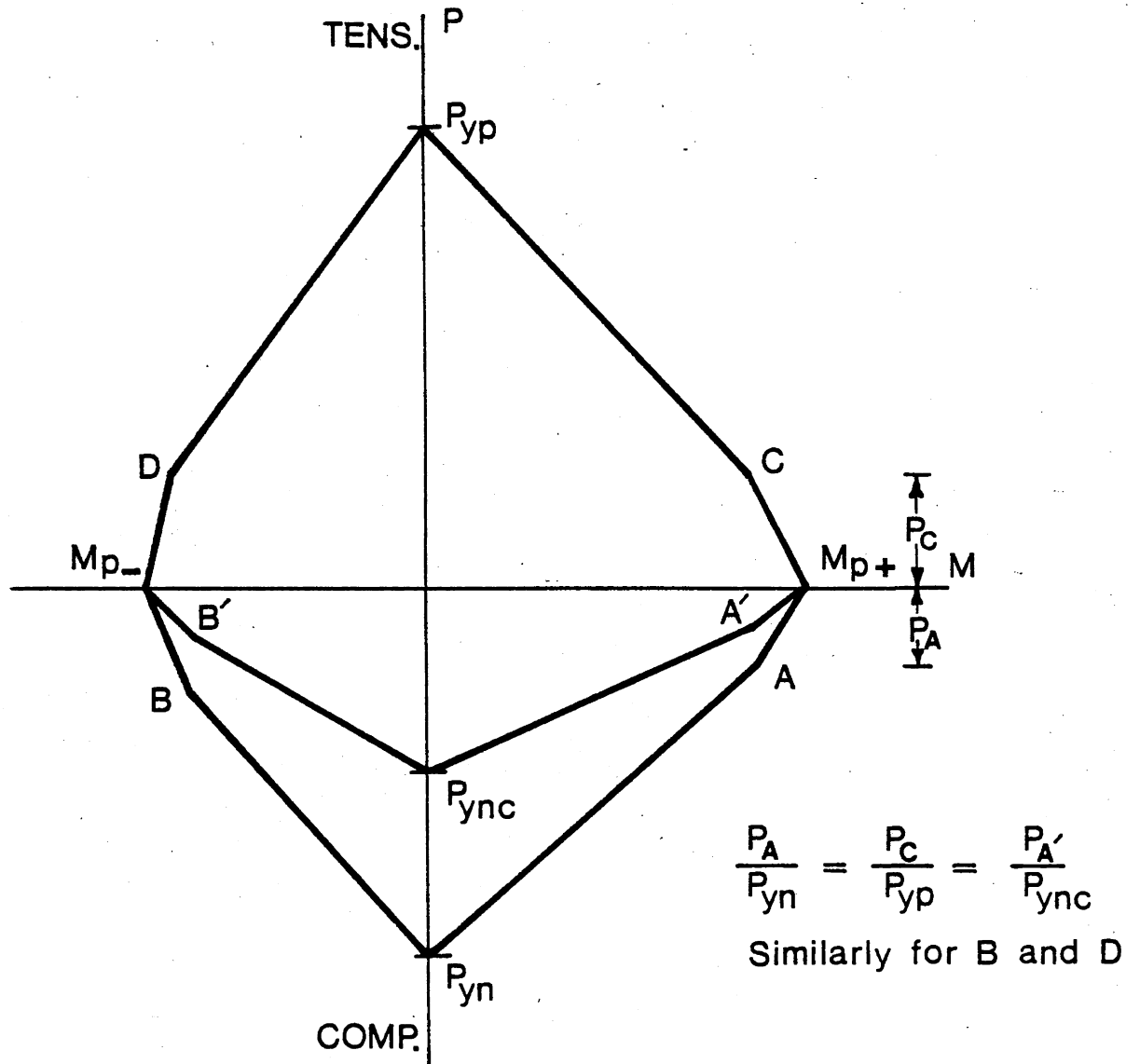


Figure 10 - M-p Interaction Curve Used in EL10

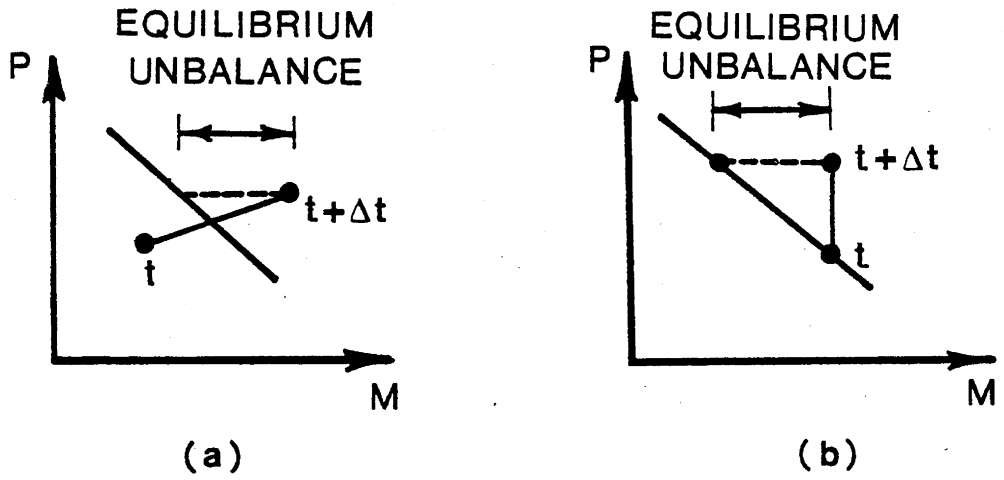


Figure 11 - Equilibrium Correction for Yield Surface Overshoot

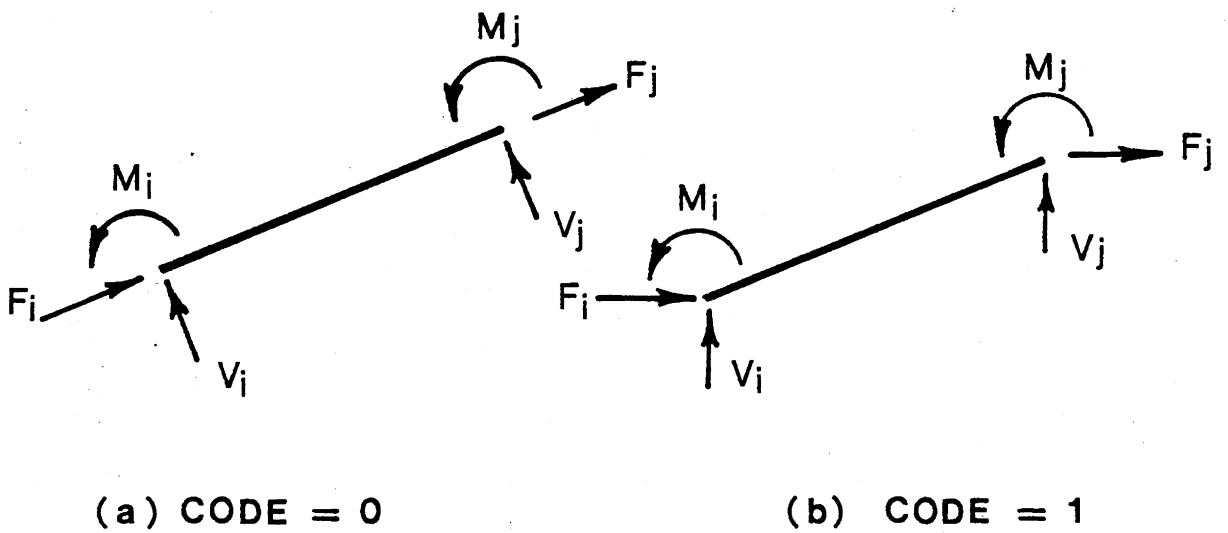


Figure 12 - End Clamping and Initial Forces



at  $P_{ync}$ . When the axial load is either  $P_{yp}$  (axial yield code = 9),  $P_{yn}$  or  $P_{ync}$  (axial yield code = 3), the member behaves as a pin-ended member in bending.

### ELEMENT STIFFNESS

The element deformations and displacements are shown in Figures 9a and 9b. The axial stiffness is given by

$$dS_1 = \frac{E_T A}{L} dv_1 \quad 3.4$$

where,  $E_T$  = tangent modulus in current state, and  $A$  = average cross sectional area.

The elastic flexural stiffness is given by

$$\begin{Bmatrix} dS_2 \\ dS_3 \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} k_{ii} & k_{ij} \\ k_{ij} & k_{jj} \end{bmatrix} \begin{Bmatrix} dv_2 \\ dv_3 \end{Bmatrix} \quad 3.5$$

where,  $I$  = reference moment of inertia; and  $k_{ii}$ ,  $k_{ij}$ ,  $k_{jj}$  are coefficients which depend on the cross section variation. For a uniform element,  $I$  = actual moment of inertia,  $k_{ii} = k_{jj} = 4$ , and  $k_{ij} = 2$ . The coefficients must be specified by the program user, and may, if desired, account for such effects as shear deformations and non-rigid end connections as well as cross section variations.

After one or more hinges form, the coefficients for the elasto-plastic component change to  $k'_{ii}$ ,  $k'_{ij}$  and  $k'_{jj}$ , as follows:

$$k'_{ii} = k_{ii}(1-A) - k_{ij}C \quad 3.6$$

$$k'_{ij} = k_{ij}(1-D) - k_{ii}B \quad 3.7$$

$$k'_{jj} = k_{jj}(1-D) - k_{ij}B \quad 3.8$$

TABLE 1  
COEFFICIENTS FOR PLASTIC ROTATIONS

Yield Condition	COEFFICIENT			
	A	B	C	D
Both Ends Elastic	0	0	0	0
Plastic hinge at end i only	1	$k_{ij}/k_{ii}$	0	0
Plastic hinge at end j only	0	0	$k_{ij}/k_{jj}$	1
Plastic hinges at both ends i and j	1	0	0	1

Note:  
Stiffness Coefficients  $k_{ii}$ ,  $k_{ij}$ , and  $k_{jj}$  are defined by Equation 3.5

---

where, A, B, C and D are defined in Table 1.

Stiffness in term of nodal displacements is obtained as

$$[K_T] = [a]^T [k_T] [a] \quad 3.9$$

where, [a] is given by equations 3.1 and 3.2.

The geometric stiffness used is exactly the same as for the buckling element. This is not the exact geometric stiffness for an end moment-buckling element, but is sufficiently accurate for taking into account the P- $\Delta$  effect in building frames.

### FIXED END AND INITIAL FORCES

Static loads applied along the lengths of end moment-buckling elements may be taken into account by specifying end clamping forces as shown in Figure 12. These forces are those which must act on the element ends to prevent end displacement.

Initial member forces may be specified for structures in which static analyses are carried out separately. The sign convention for these forces is as shown in Figure 12a. These forces are not converted to loads on the nodes of the structure but simply used to initialize the element end actions. Any end forces due to other loadings are then added to the initial forces.

Initial element forces may be specified in addition to static nodal loads and element end clamping forces in which case the element forces due to the static loading are added to the initial forces. The geometric stiffness, if used, is based on the initial axial force plus any axial force due to static loading, and is included only for the dynamic loading, not for the static loading.

Fixed end and initial forces are defined as standard patterns, and each element can be identified with a standard pattern for dead load fixed end force, live load fixed end force and initial force. In addition multiplication factors for scaling the standard patterns can be specified.

### LIVE LOAD REDUCTION

Live load reductions based on area supported may have important effects in buildings and, therefore, should be taken into account. The fixed end forces specified for any element, after scaling by the factors specified for the element, should account for any live load reductions permitted for that element.

The fixed end forces for any element will, when changed in sign, constitute static loadings on the nodes to which the element connects, and these loadings are taken into account by the program. Frequently, however, the live load reduction factor permitted for a column in a building will exceed that for the beams it supports, because columns support tributary loads from several floors. Therefore, if the full live load fixed end shears for each beam are applied at the structure nodes the accumulated loads on the columns may be unnecessarily large. This could be compensated for by reducing the fixed end shears to provide the correct column loads but the shear forces computed for the beams would then be too low. A preferable approach is to take advantage of the live load reduction factors which may be specified with the fixed end force patterns and are used as follows.

For initialization of the element shear and axial forces the full specified fixed end forces are used. However, for computation of the static loads on the nodes connected to the element, the fixed end shear and axial

forces due to live load (but not the moments) are first multiplied by the specified reduction factor. The forces producing axial loads in the columns may, therefore, be reduced to account for difference in permissible live load reductions between the beams and columns, yet the shear forces computed for the beams will still be correct. The reduction factor is ignored for dead loads.

#### SHEAR DEFORMATIONS

If desired, effective flexural shear areas may be specified. The program then modifies the flexural stiffness to account for the additional shear deformations. However, the fixed end forces are not changed, so that if shear deformations may be important the specified fixed end force patterns should take these deformations into account.

#### OUTPUT RESULTS

The following results are printed for the static loading condition (all elements, time = 0) and at each output time if a time history is requested. The time-history results are output only for those elements for which time histories are requested.

1. Yield Code:

- (a) Flexural yield code (at each end of an element). Zero indicates the element end is elastic, and 1 indicates that a plastic hinge has formed.

(b) Axial yield code (for the whole element).

0 to 9 as shown in Figures 4a and 4b.

2. Bending moment, shear force and axial force acting at each end of an element, with the sign convention as shown in Figure 12a.
3. Current plastic hinge rotations at each end.
4. Accumulated positive and negative plastic hinge rotations up to the current time.
5. Net axial extension, positive means extension, negative means shortening.

The maximum positive and negative values of bending moment, shear force, axial force, plastic hinge rotations and axial extension, with their time of occurrence, are printed at the time intervals requested for envelopes.

The envelope values of accumulated positive and negative plastic hinge rotations (PRACP(2), PRACN(2)) as well as of accumulated positive and negative axial elongations (VPACP, VPACN) are not printed, although they are computed within the program. Program users interested in these values can easily insert appropriate print statements in Subroutine OUT10.

INPUT DATA PREPARATIONE10. END MOMENT-BUCKLING ELEMENTS - E10

Number of words of information per element = 170.

## E10(a) CONTROL INFORMATION FOR GROUP (6I5) - ONE CARD.

Columns      1 - 5: Punch 10 (to indicate that group consists of end moment-buckling elements).

              6 - 10: Number of elements in group.

            11 - 15: Number of different element stiffness types (max. 40). See Sections E10(b) and E10(c).

            16 - 20: Number of different yield interaction surfaces for cross sections (max. 40). See Section E10(d).

            21 - 25: Number of different fixed end force patterns (max. 35). See Section E10(e).

            26 - 30: Number of different initial element force patterns (max. 30). See Section E10(f).

## E10(b). STIFFNESS TYPES (I5, 4F10.0, 3F5.0, 2F10.0) - ONE CARD FOR EACH STIFFNESS TYPE.

Columns      1 - 5: Stiffness type number, in sequence beginning with 1.

              6 - 15: Young's modulus of elasticity.

            16 - 25: Strain hardening modulus, as a proportion of Young's modulus.

            26 - 35: Average cross sectional area.

            36 - 45: Reference moment of inertia.

            46 - 50: Flexural stiffness factor  $k_{ii}$ .

Columns 51 - 55: Flexural stiffness factor  $k_{jj}$ .  
 56 - 60: Flexural stiffness factor  $k_{ij}$ .  
 61 - 70: Effective shear area. Leave blank or punch zero if shear deformations are to be ignored, or if shear deformations have already been taken into account in computing the flexural stiffness factors.  
 71 - 80: Poisson's ratio (used for computing shear modulus, and required only if shear deformations are to be considered).

E10(c). INPUT RADIUS OF GYRATION, K AND PHI FACTORS (I5, 3F10.0) - ONE CARD FOR EACH STIFFNESS TYPE

Columns 1 - 5: Stiffness type number, in sequence beginning with 1.  
 6 - 15: Radius of gyration.  
 16 - 25: Effective length factor.  
 26 - 35: Strength reduction factor, PHI.

E10(d). CROSS SECTION M-P YIELD INTERACTION SURFACES (I5, 4F10.0, 4F5.0) - ONE CARD FOR EACH YIELD SURFACE.

See Figure 10 for explanation.

Columns 1 - 5: Yield surface number, in sequence beginning with 1.  
 6 - 15: Positive plastic moment,  $M_{p+}$   
 16 - 25: Negative plastic moment,  $M_{p-}$   
 26 - 35: Compression yield force in first cycle,  $P_{yn}$ .  
 36 - 45: Tension yield force,  $P_{yp}$ .  
 46 - 50: M - coordinate of balance point A, as a



proportion of  $M_{p+}$ .

Columns 51 - 55: P - coordinate of balance point A, as a proportion of  $P_{yn}$ .

56 - 60: M - coordinate of balance point B, as a proportion of  $M_{p-}$ .

61 - 65: P - coordinate of balance point B, as a proportion of  $P_{yn}$ .

E10(e). FIXED END FORCE PATTERNS (2I5, 7F10.0) - ONE CARD FOR EACH FIXED END FORCE PATTERN.

Omit if there are not fixed end forces. See Figure 12.

Columns 1 - 5: Pattern number, in sequence beginning with 1.

10: Axis code, as follows:

Code = 0: Forces are in the element coordinate system, as in Figure 12a.

Code = 1: Forces are in the global coordinate system, as in Figure 12b.

11 - 20: Clamping force,  $F_i$ .

21 - 30: Clamping force,  $V_i$ .

31 - 40: Clamping moment,  $M_i$ .

41 - 50: Clamping force,  $F_j$ .

51 - 60: Clamping force,  $V_j$ .

61 - 70: Clamping moment,  $M_j$ .

71 - 80: Live load reduction factor, for computation of live load forces to be applied to nodes.

E10(f). INITIAL ELEMENT FORCE PATTERNS (I5, 6F10.0) - ONE CARD FOR EACH INITIAL FORCE PATTERN.

Omit if there are no initial forces. See Figure 12a.

Columns	1 - 5:	Pattern number, in sequence beginning with 1.
	6 - 15:	Initial axial force, $F_i$ .
	16 - 25:	Initial shear force, $V_i$ .
	26 - 35:	Initial moment, $M_i$ .
	36 - 45:	Initial axial force, $F_j$ .
	46 - 55:	Initial shear force, $V_j$ .
	56 - 65:	Initial moment, $M_j$ .

E10(g). ELEMENT GENERATION COMMANDS (11I5, 2F5.0, I5, F5.0) - ONE CARD FOR EACH GENERATION COMMAND.

Elements must be specified in increasing numerical order.

Cards for the first and last elements must be included. See Note 7 of User's Guide (13) for explanation of generation procedure.

Columns	1 - 5:	Element number, or number of first element in a sequentially numbered series of elements to be generated by this command.
	6 - 10:	Node number at element end i.
	11 - 15:	Node number at element end j.
	16 - 20:	Node number increment for element generation. If zero or blank, assumed to be equal to 1.
	21 - 25:	Stiffness type number.
	26 - 30:	Yield surface number for element end i.
	31 - 35:	Yield surface number for element end j.
	40:	Code for including geometric stiffness. Punch 1 if geometric stiffness is to be included. Leave blank or punch zero if geometric stiffness is to be ignored.
	45:	Time history output code. If a time history

of element results is not required for the element covered by this command, punch zero or leave blank. If a time history printout, at the intervals specified on card D1, is required, punch 1.

- Columns 46 - 50: Fixed end force pattern number for static dead loads on element. Leave blank or punch zero if there are no dead loads. See Note below.
- 51 - 55: Fixed end forces pattern number for static live loads on element. Leave blank or punch zero if there are no live loads.
- 56 - 60: Scale factor to be applied to fixed end forces due to static dead loads.
- 61 - 65: Scale factor to be applied to fixed end forces due to static live loads.
- 66 - 70: Initial force pattern number. Leave blank or punch zero if there are not initial forces.
- 71 - 75: Scale factor to be applied to initial element forces.

Note: If the static load code, Card C1, is zero but fixed end forces are still specified for some elements, an inconsistency results. In effect, any such fixed end forces will be treated as initial element forces.

## CHAPTER 4

### EXAMPLE

The braced frame example shown in Figure 13 may be used to check the execution of the DRAIN-2D program with elements EL9 and EL10 when it is implemented on a computer installation. Program decks received through the Department of Civil Engineering, University of Michigan or the National Information Service for Earthquake Engineering will include a data deck and computer output for this example.

The input cards for the structure are listed in Table 2 and identified by the corresponding sections in the User's Guide (13) and in this report. The user should be able to obtain guidance in data preparation procedures by studying this sample data.

The columns in the example structure are represented by element EL10 in group 1, the beams are represented by element EL5 in group 2 and the bracing members (assumed as pin-connected at the ends) are represented by element EL9 in group 3. Node numbers are shown at the ends of the members and element numbers are shown near the middle.

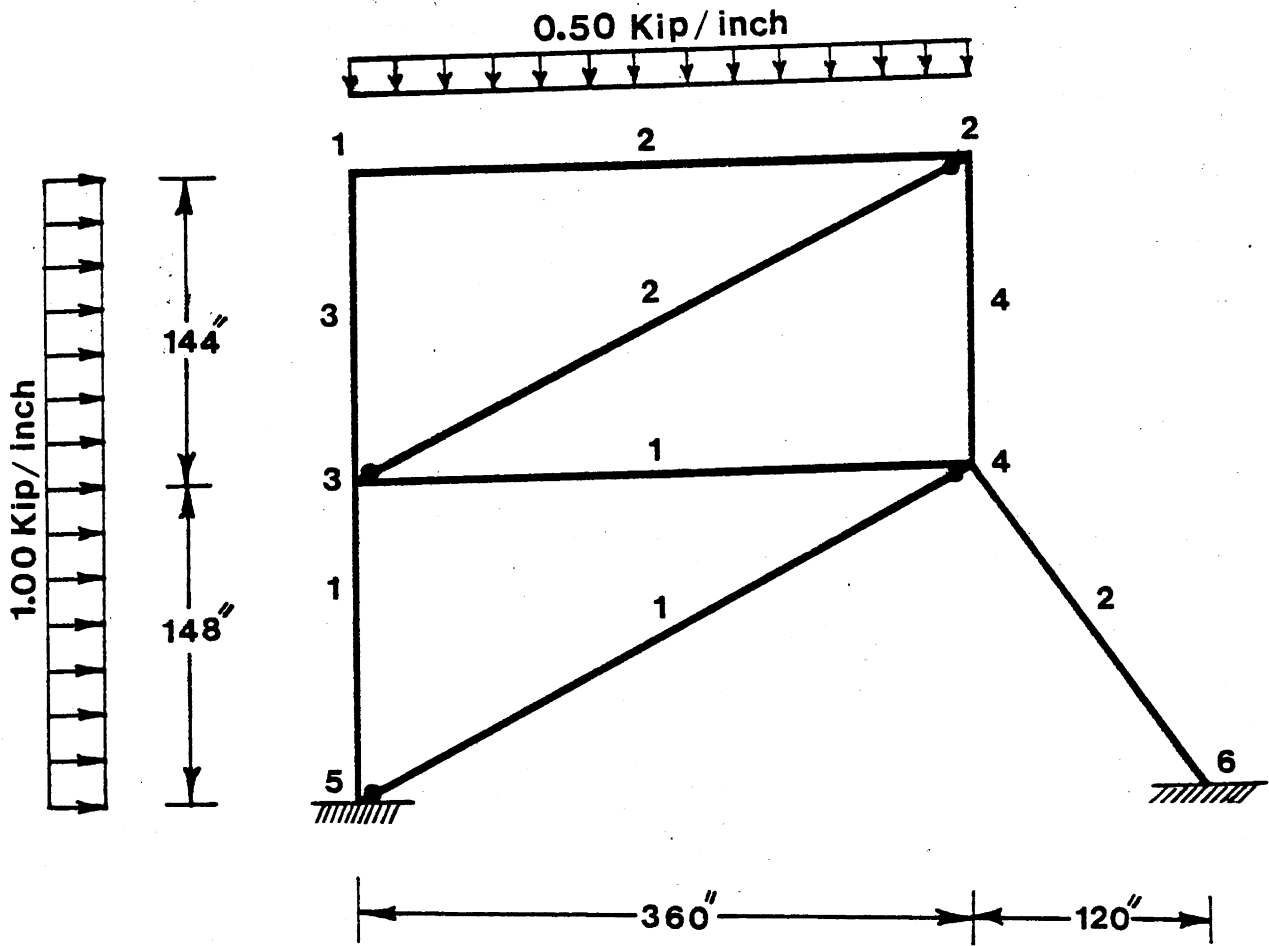


Figure 13 - Test Example

TABLE 2 - SAMPLE INPUT DATA

START	TEST EXAMPLE FOR MICHIGAN EL9 AND EL10 ELEMENTS										A		
6	6	1	2	1	3							B1	
1						292.							
2	360.					292.							
3						148.						B2	
4	360.					148.							
5													
6	480.												
5	1	1	1	6								B4	
1	2	1	2										
1	2	3	4									B5	
1	1.5						3	2	1.			B6	
1		100	0.01		575.6		1.	1.	1.	20.		C1	
4		1	IMPULSE LOADING										
		0.03	1.0	0.10		2.0						C3	
												C4	
10	10		2	4	4								
1	3											D	
1	2	3	4										
1	2	3	4										
10	4	2	2	2									
1	29000.	0.01		49.1		2020.	4.	4.	2.				
2	29000.	0.01		35.0		1370.	4.	4.	2.				
1	4.01	0.7		0.5									
2	3.75	0.7		0.4								E10	
1	10900.	-10900.		-1700.		1770.	1.	0.15	1.	0.15			
2	7600.	-7600.		-1200.		1275.	1.	0.15	1.	0.15			
1			74.		1825.			74.		-1825.			
2	1	-72.			1728.		-72.			-1728.			
1	5	3		1	1	1	1	1		1.5			
2	4	6		1	1	1	1	1					
3	3	1		2	2	2	1	1		2	1.25		
4	2	4		2	2	2	1	1					
5	2	1		2	1								
1	29000.	0.05		41.8		3410.	4.	4.	2.			E5	
1		12850.		-12850.									
2		40000.		-40000.									
1			50.		5400.			90.		-5400.			
1	3	4		1	1	2		1					
2	1	2		1	1	1		1	1	1.5			
9	2	2											
1	29000.	17.5		350.		-250.	2.45	0.90	0.55				
2	30000.	15.6		300.		-200.	1.90	1.00	0.35			E9	
1	5	4		1	1	1			50.				
2	3	2		2	1	1			40.				
STOP													

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APPENDIX A-1

FORTRAN LISTING OF BUCKLING ELEMENT EL9

```

SUBROUTINE INEL9 (/KCONT/,/FCONT/,/NDOF/,/NINFC/,/ID/,/X/,/Y/,/NN/
1)
C
COMMON /INPEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3 IELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVFNN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(147)
COMMON /NCRK/ FTYP(40,7),FEF(40,4),KDFEF(40),DD(4),GA(4,4),
1 FFEF(4),SFF(4),SSFF(4),NMEM,NMBT,NFEF,SLOP1,INEL,
2 INODI,INODJ,INC,IINC,IMBT,IIMBT,IKGM,IKDT,KFDL,
3 IKFDL,KFLL,IKFLL,FDL,FFDL,FLL,PFLL,FINIT,PFINIT,
4 XL,YL,APEA,RAD,SLEND,W(1460)
C
DIMENSION KCONT(1),ID(NN,1),X(1),Y(1),COM(1)
DIMENSION AST(2),YESNC(2)
DATA AST/2H ,2H */
DATA YESNO/4H YES,4H NO /
EQUIVALENCE (IMEM,COM(1))
C
C DATA INPUT, BUCKLING ELEMENTS
C
NDOF=4
NINFC=53
NMEM=KCONT(2)
NMBT=KCONT(3)
NFEF=KCONT(4)
PRINT 10, (KCONT(I),I=2,4)
10 FORMAT (27H BUCKLING ELEMENTS (TYPE 9)////
1 25H NO. OF ELEMENTS =I4/
2 25H NO. OF STIFFNESS TYPES =I4/
3 25H NO. OF F.E.F. PATTERNS =I4)
C
C INPUT STIFFNESS PROPERTIES
C
PRINT 20
20 FORMAT (////16H STIFFNESS TYPES//
1 5H TYPE,6X,7H YOUNGS,6X,8H SECTION,3X,
2 15H YIELD FORCES,10X,10H RADIUS OF,5X,
3 12H EQU. LENGTH,5X,19H STRENGTH REDUCTION/
4 5H NO.,5X,8H MODULUS,6X,7H APEA ,5X,
5 3H TENSION,5X,8H COMPN ,6X,9H GYRATION,4X,
6 12H COEFFICIENT,9X,7H FACTOR/)
C
DO 50 IT=1,NMBT
READ 30, I, (FTYP(IT,J),J=1,7)
50 PRINT 40, IT, (FTYP(IT,J),J=1,7)
30 FORMAT (I5,7F10.0)
40 FORMAT (I4,E14.4,6F13.2)
C
C FIXED END FORCE PATTERNS
C
IF (NFEF.EQ.0) GO TO 100
PRINT 60

```

```

60 FORMAT (///25H FIXED END FORCE PATTERNS//
1      3H PATTERN,3X,4HAXIS,2 (7X,5HAXIAL,7X,5HSHEAR)/
2      3H NO. ,3X,4HCODE,2 (7X,5HAT I),2 (7X,5HAT J)/)

```

```

C
DO 90 NF=1,NPEF
READ 70, I,KDFEF(NF), (FEF(NF,J),J=1,4)
70 FORMAT (2I5,4F10.0)
80 FORMAT (I6,I8,1X,4F12.2)
90 PRINT 80, NF,KDFEF(NF), (FEF(NF,J),J=1,4)

```

```

C
C
C
ELEMENT DATA

```

```

100 PRINT 110
110 FORMAT (///22H ELEMENT SPECIFICATION//
1      3X,4HELEM,3X,4HNODF,2X,4HNODE,2X,4HNODE,2X,4HSTIF,2X,
2      4HGgeom,2X,4HTIME,3X,12HFEEF PATTERNS,3X,17HFEEF SCALE FACTORS,
3      5X,7HINITIAL/
4      3X,4H NO.,3X,4H I ,2X,4H J ,2X,4HDIFF,2X,4HTYPE,2X,
5      4HSTIF,2X,4HHIST,3X,12H DL LL ,3X,17H DL LL ,
6      5X,7H FORCE /)

```

```

C
KDDYX=0
KDDY=0
KST=0
KPP=0.
DO 120 J=19,47
120 COM(J)=0.

```

```

C
IMEM=1
130 READ 140, INEL,INODI,INODJ,IINC,IIMBT,IKGM,IKDT,IKFDL,IKFLL,FFDL,F
1FLL,FFINIT
140 FORMAT (9I5,2F5.0,F10.0)
IF (INEL.GT.IMEM) GO TO 170
150 NODI=INODI
NODJ=INODJ
INC=IINC
IF (INC.EQ.0) INC=1
IMBT=IIMBT
KGEOM=IKGM
KOUTDT=IKDT
YNG=YESNO(2)
IF (KGEOM.NE.0) YNG=YESNC(1)
YNT=YESNO(2)
IF (KOUTDT.NE.0) YNT=YESNC(1)
KFDL=IKFDL
KFLL=IKFLL
FDL=FFDL
FLL=FFLL
FINIT=FFINIT
ASTT=AST(1)
IF (INEL-NMEM) 130,170,130

```

```

C
160 NODI=NODI+INC
NODJ=NODJ+INC
ASTT=AST(2)

```

```

C
170 PRINT 180, ASTT, IMEM, NODI, NODJ, INC, IMBT, YNG, YNT, KFDL, KFLL, FDL, FLL,
1FINIT
180 FORMAT (A2, I4, I7, 3I6, 3X, A4, 2X, A4, I7, I6, F11.2, F10.2, F11.2)
C
C      LOCATION MATRIX
C
      DO 190 L=1, 2
      LM(L)=ID(NODI, L)
190  LM(L+2)=ID(NODJ, L)
      CALL BAND
C
C      ELEMENT PROPERTIES
C
      XL=X(NODJ)-X(NODI)
      YL=Y(NODJ)-Y(NODI)
      FL=SQRT(XL**2+YL**2)
      COSA=XL/FL
      SINA=YL/FL
      AREA=FTYP(IMBT, 2)
      EAL=FTYP(IMBT, 1)*AREA/FL
      PYP=FTYP(IMBT, 3)
      PYN=-ABS(FTYP(IMBT, 4))
      PHI=FTYP(IMBT, 7)
      PYNC=PHI*PYN
      RAD=FTYP(IMBT, 5)
      AK=FTYP(IMBT, 6)
      SLEND=AK*FL/RAD
      PFAC=ABS(PYNC/PYP)
      SLOP1=(PFAC*(1.-PHI))/(PFAC-5.*PHI)
      EAL1=SLOP1*EAL
      K3RE=-PFAC
      RATIO=60.0/SLEND
      DELTY=PYP/EAL
C
C      LOADS DUE TO FIXED END FORCES
C
      SFEF=0.
      IF (KFDL+KFLL.EQ.0) GO TO 310
      DO 200 I=1, NDOF
      DO 200 J=1, NDOF
200  GA(I, J)=0.
      GA(1, 1)=COSA
      GA(1, 2)=SINA
      GA(2, 1)=-SINA
      GA(2, 2)=COSA
      GA(3, 3)=COSA
      GA(3, 4)=SINA
      GA(4, 3)=-SINA
      GA(4, 4)=COSA
      DO 210 I=1, 4
      SFF(I)=0.
210  SSFF(I)=0.
      IF (KFDL.EQ.0) GO TO 250
      DO 220 I=1, 4

```

```

220 FFEF(I)=FEF(KFDL,I)*FDL
    IF (KDFEF(KFDL).EQ.0) GO TO 230
    CALL MULT (GA,FFE F,SFF,4,4,1)
    GO TO 250
230 DO 240 I=1,4
240 SFF(I)=FFE F(I)
C
250 IF (KFLL.EQ.0) GO TO 290
    DO 260 I=1,4
260 FFEF(I)=FEF(KFLL,I)*FLL
    IF (KDFEF(KFLL).EQ.0) GO TO 270
    CALL MULT (GA,FFE F,SSFF,4,4,1)
    GO TO 290
270 DO 280 I=1,4
280 SSFF(I)=FFE F(I)
C
290 DO 300 I=1,4
300 SSFF(I)=SSFF(I)+SFF(I)
C
    CALL MULT (GA,SSFF,DD,4,4,1)
    CALL SFORCE (DD)
C
C    INITIALIZE ELEMENT FORCE
C
    SFEF=(SSFF(3)-SSFF(1))*0.5
310 FF=FINIT+SEF
    SEP=FF
    IF (FINIT.LT.0.) GO TO 320
    SENP=FINIT
    SENN=0.
    GO TO 330
320 SENN=FINIT
    SENP=0.
C
330 CALL FINISH
C
C    GENERATE MISSING ELEMENTS
C
    IF (IMEM.EQ.NMEM) RETURN
    IMEM=IMEM+1
    IF (IMEM.EQ.INEL) GO TO 150
    GO TO 160
C
END

```

SUBROUTINE STIF9 (/MSTEP//,/NDOF//,/NINFC//,/COMS//,/FK//,/DFAC/)

C  
 COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,  
 1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,  
 2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,  
 3 IELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,  
 4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,  
 5 PYP,PYN,PYNC,REST(147)  
 COMMON /WORK/ STIF,STIFF,SST(2,2),AA(2,4),AATK(4,2),FFK(4,4),  
 1 W(1962)

C  
 DIMENSION COM(1),COMS(1),FK(4,4)  
 EQUIVALENCE (IMEM,COM(1))

C  
 C C STIFFNESS FORMULATION, BUCKLING ELEMENTS  
 C

DO 10 J=3,23  
 10 COM(J)=COMS(J)

C  
 C C CURRENT STIFFNESS  
 C

CALL FST9 (STIF,KODY)

C  
 C C PREVIOUS STIFFNESS  
 C

IF (MSTEP.LT.2) GO TO 20  
 CALL FST9 (STIFF,KODYX)

C  
 C C STIFFNESS DIFFERENCE  
 C

STIF=STIF-STIFF  
 20 FK(1,1)=STIF\*COSA\*\*2  
 FK(1,2)=STIF\*SINA\*COSA  
 FK(1,3)=-FK(1,1)  
 FK(1,4)=-FK(1,2)  
 FK(2,2)=STIF\*SINA\*\*2  
 FK(2,3)=FK(1,4)  
 FK(2,4)=-FK(2,2)  
 FK(3,3)=FK(1,1)  
 FK(3,4)=FK(1,2)  
 FK(4,4)=FK(2,2)  
 DO 30 I=2,4  
 JJ=I-1  
 DO 30 J=1, JJ  
 30 FK(I,J)=FK(J,I)  
 IF (MSTEP.GT.1) GO TO 80

C  
 C C INITIAL STIFFNESS FOR STEP 0, BETA=0 ALLOWANCE FOR STEP 1  
 C

CC=1.  
 IF (MSTEP.EQ.1) CC=DFAC  
 DO 40 I=1,16  
 40 FK(I,1)=FK(I,1)\*CC

C  
 C C ADD GEOMETRIC STIFFNESS

```
C
  IF (MSTEP.EQ.0.OR.KGECM.EQ.0) GO TO 80
  PFL=COMS(34)/PL
  DO 50 I=1,4
50  SST(I,1)=PFL
  DO 60 I=1,8
60  AA(I,1)=J.
  AA(1,1)=-SINA
  AA(1,2)=COSA
  AA(2,3)=SINA
  AA(2,4)=-COSA
  CALL MULTST (AA,SST,AATK,FFK,4,2)
  DO 70 I=1,16
70  FK(I,1)=FK(I,1)+FFK(I,1)
C
80  RETURN
  END
```

```

SUBROUTINE RESP9 (/NDCF//,/NINFC//,KBAL//,KPR//,COMS//,DDISM//,DD/
1,/TIME//,VELM//,DFAC//,DELTA/)
C
COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,PL,COSA,SINA,PFAC,RATIO,
1      DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2      X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3      IEL OG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4      TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5      PYP,PYN,PYNC,REST(*47)
COMMON /WORK/ EALE,DSL,DSEP,SLIN,FAC,FACTOR,FACAC,DSUB,FVRTX,
1      SLOP6,SLOP7,DVP,DS4,DS5,DS6,DS7,DS8,POUT(1983)
C
DIMENSION COM(1),COMS(1),DDISM(1),DD(1),VELM(1)
EQUIVALENCE (IMEM,COM(1))
C
STATE DETERMINATION, BUCKLING ELEMENTS
C
DO 10 I=1,NINFC
10 COM(I)=COMS(I)
KODYX=KODY
IF (IMEM.EQ.1) THFD=0
C
EXTENSION INCREMENT
C
DVAX=COSA*(DDISM(3)-DDISM(1))+SINA*(DDISM(4)-DDISM(2))
VTOT=VTOI+DVAX
C
LINEAR FORCE INCREMENT
C
CALL FSTS (EALE,KODY)
SLIN=SEP+EALE*DVAX
C
INITIALIZE
C
FVRTX=0.
C
CHECK VERTEX STATE
C
IF (IVRTX.EQ.0) CALL VRTX9 (FVRTX,DVAX,TIME)
C
IF (IVRTX.EQ.0) GO TO 120
C
FACAC=FVRTX
C
20 FACTOR=1.-FACAC
KODYI=KODY+1
GO TO (30,120,120,40,50,60,70,80,90,100),KODYI
C
ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
30 DSEP=EAL*DVAX
IF (DSEP) 31,110,32
31 FAC=(PYNC-SEP)/DSFP
IF (FAC.GE.FACTOR) GO TO 33
FACTOR=FAC

```



```

SEP=PYNC
KODY=3
GO TO 110
C
32 FAC=(PYP-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 33
FACTOR=FAC
SEP=PYP
KODY=9
GO TO 110
C
33 SEP=SEP+FACTOR*DSEP
GO TO 110
C
ON SLOPE 3, BUCKLING AND CONTINUING
C
40 IF (DVAX.GT.0.) GO TO 41
C
UPDATE PLASTIC DEFORMATIONS
C
DVP=FACTOR*DVAX
VPACN=VPACN+DVP
GO TO 120
C
BUCKLING AND UNLOADING
C
41 XTOT=(VTOT-DVAX)/DELTY
IF (XTOT.GE.X3RE) GO TO 42
C
ESTABLISH NEW STIFFNESS FOR REVERSE
C
X3RE=XTOT
IELOG=1
CALL LAW9
P5=Y5*PYP
KODY=4
GO TO 50
C
USE OLD STIFFNESS FOR REVERSE
C
42 IF (INDRE.EQ.2.AND.XTOT.LE.X6) GO TO 43
IF ((X5D-XTOT).EQ.0.) GO TO 44
SLOP5=(Y5D+PFAC)/(X5D-XTOT)
EAL6=EAL*SLOP6
KODY=6
GO TO 70
C
43 SLOP7=(Y5+PFAC)/(X5-XTOT)
EAL7=EAL*SLOP7
KODY=7
GO TO 80
44 KODY=5
GO TO 60
C
ON SLOPE 4, GET FACTOR FOR STATUS CHANGE

```

C

```

50 DS4=EAL4*DVAX
   IF (DS4) 51,110,54
51 IF (SEP) 52,52,53
52 FAC=(PINC-SEP)/DS4
   IF (FAC.GE.FACTOR) GO TO 55
   FACTOR=FAC
   SEP=PINC
   KODY=3
   GO TO 110
53 XSD=VIOT/DELTY
   YSD=SEP/PYP
   PSD=SEP
   INDRE=1
   KODY=5
   GO TO 60
54 FAC=(P5-SEP)/DS4
   IF (FAC.GE.FACTOR) GO TO 55
   FACTOR=FAC
   SEP=P5
   KODY=3
   GO TO 110
55 SEP=SEP+FACTOR*DS4
   GO TO 110

```

C

C

C

ON SLOPE 5, GET FACTOR FOR STATUS CHANGE

```

60 DS5=EAL*DVAX
   IF (DS5) 61,110,62
61 FAC=(PINC-SEP)/DS5
   IF (FAC.GE.FACTOR) GO TO 65
   FACTOR=FAC
   SEP=PINC
   KODY=3
   GO TO 110
62 FAC=(P5D-SEP)/DS5
   IF (FAC.GE.FACTOR) GO TO 65
   FACTOR=FAC
   SEP=P5D
   GO TO (63,64),INDRE
63 KODY=4
   GO TO 110
64 KODY=3
   GO TO 110
65 SEP=SEP+FACTOR*DS5
   GO TO 110

```

C

C

C

ON SLOPE 6, GET FACTOR FOR STATUS CHANGE

```

70 DS6=EAL6*DVAX
   IF (DS6) 71,110,72
71 FAC=(PINC-SEP)/DS6
   IF (FAC.GE.FACTOR) GO TO 75
   FACTOR=FAC
   SEP=PINC

```

```

KODY=3
GO TO 110
72 FAC=(P5D-SEP)/DS6
   IF (FAC.GE.FACTOR) GO TO 75
   FACTOR=FAC
   SEP=P5D
   GO TO (73,74),INDRE
73 KODY=4
   GO TO 110
74 KODY=8
   GO TO 110
75 SEP=SEP+FACTOR*DS6
   GO TO 110

```

C  
C  
C

```

ON SLOPE 7, GET FACTOR FOR STATUS CHANGE

80 DS7=EAL7*DVAX
   IF (DS7) 31,110,82
81 FAC=(PYNC-SEP)/DS7
   IF (FAC.GE.FACTOR) GO TO 83
   FACTOR=FAC
   SEP=PYNC
   KODY=3
   GO TO 110
82 FAC=(P5-SEP)/DS7
   IF (FAC.GE.FACTOR) GO TO 83
   FACTOR=FAC
   SEP=P5
   KODY=8
   GO TO 110
83 SEP=SEP+FACTOR*DS7
   GO TO 110

```

C  
C  
C

```

ON SLOPE 3, GET FACTOR FOR STATUS CHANGE

90 DS8=EAL8*DVAX
   IF (DS8) 91,110,92
91 K5D=VTOT/DELTY
   Y5D=SEP/PYP
   P5D=SEP
   INDRE=2
   KODY=5
   GO TO 60

```

C

```

92 FAC=(PYP-SEP)/DS8
   IF (FAC.GE.FACTOR) GO TO 93
   FACTOR=FAC
   SEP=PYP
   KODY=9
   GO TO 110
93 SEP=SEP+FACTOR*DS8
   GO TO 110

```

C  
C  
C

```

ON SLOPE 9, YIELDED BUT CONTINUING

```

```

100 IF (DVAX.LT.0.) GO TO 101
C
C   UPDATE PLASTIC DEFORMATIONS
C
   DVP=FACTOR*DVAX
   XPP=XPP+DVP/DELTY
   VPACP=VPACP+DVP
   GO TO 120
C
C   YIELDED BUT UNLOADING
C
101 KDDY=0
C
C   RESIDUAL ELONGATION, RE
C
   IF (IELOG.NE.1) GO TO 105
   SLEND=60./RATIC
   RE=0.0175*(0.55*X3RE/SLEND+0.0002*X3RE**2)
   RE=RE*FL/DELTY
   XPP=XPP+RE
   IELOG=0
C
105 X3RE=XPP-PFAC
   GO TO 30
C
C   CHECK FOR COMPLETION OF CYCLE
C
110 FACAC=FACAC+FACTOR
   IF (FACAC.LT.0.9999999) GO TO 20
C
C   NEW FORCE, UNBALANCED FORCE DUE TO YIELD
C
120 ST=SEP
   DSUB=SLIN-SEP
   IF (ABS(DSUB).GT.1.E-8) KBAL=1
C
C   DEFORMATION RATE FOR DAMPING
C
   IF (DFAC.EQ.0.0.AND.DELTA.EQ.0.0) GO TO 140
   IF (TIME.EQ.0.) GO TO 150
   KBAL=1
   DVAX=COSA*(VELM(3)-VELM(1))+SINA*(VELM(4)-VELM(2))
C
C   BETA=0 DAMPING FORCE
C
   IF (DFAC.EQ.0.) GO TO 130
   DSUB=DSUB+DFAC*FAL*DVAX
C
C   STRUCTURAL DAMPING FORCE
C
130 IF (DELTA.EQ.0.) GO TO 140
   DSL=DELTA*SIGN(ABS(ST),DVAX)
   DSUB=DSUB-DSL+SDFO
   SDFO=DSL
C

```

```

C      UNBALANCED LOAD VECTOR
C
140 IF (KBAL.EQ.0) GO TO 150
      DD (3) = DSJB * COSA
      DD (4) = DSUB * SINA
      DD (1) = -DD (3)
      DD (2) = -DD (4)
C
C      EXTRACT ENVELOPES
C
150 IF (SENP.GE.ST) GO TO 160
      SENP=ST
      ISENP=TIME
      GO TO 170
160 IF (SENN.LE.ST) GO TO 170
      SENN=ST
      ISENN=TIME
170 IF (VENP.GE.VTOT) GO TO 180
      VENP=VTOT
      IVENP=TIME
      GO TO 190
180 IF (VENN.LE.VTOT) GO TO 190
      VENN=VTOT
      IVENN=TIME
190 CONTINUE
C
C      PRINT TIME HISTORY
C
      IF (KPR.LT.0) GO TO 200
      IF (KPR.EQ.0.OR.KOJTDI.EQ.0) GO TO 240
200 IF (IHED.NE.0) GO TO 220
      KKPR=IABS(KPR)
      PRINT 210, KKPR, TIME
210 FORMAT (///18H RESULTS FOR GROUP, I3,
1         27H, BUCKLING ELEMENTS, TIME =, F8.3
2         //5X, 5H FLEM, 3X, 4HNCDE, 3X, 4HNODE, 3X, 5HYIELD, 8X, 5HAXIAL, 4X,
3         9H NET , 3X, 25HACCU. PLASTIC EXTENSIONS/5X,
4         5H NO., 3X, 4H I , 3X, 4H J , 3X, 5H CODE, 8X, 5HFORCE, 4X,
5         9HEXTENSION, 5X, 8HPOSITIVE, 5X, 8HNEGATIVE/)
      IHED=1
220 PRINT 230, IMEM, NODI, NODJ, KODY, ST, VTOT, VPACP, VPACN
230 FORMAT (I9, 2I7, I8, F14.2, 3F13.5)
C
C      SET INDICATOR FOR STIFFNESS CHANGE
C
240 KST=0
      IF (KODYX.NE.KODY) KST=1
C
C      UPDATE INFORMATION IN COMS
C
      DO 250 J=15, 47
250 COMS(J) = COM (J)
      COMS(2) = COM (2)
C
      RETURN
      END

```

```

SUBROUTINE OUT9 (/COMS/,/NINFC/)
C
COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3 TELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(147)
C
DIMENSION COM(1),COMS(1)
EQUIVALENCE (IMEM,COM(1))
C
ENVELOPE OUTPUT, BUCKLING ELEMENTS
C
DO 10 J=1,NINFC
10 COM(J)=COMS(J)
C
IF (IMEM.EQ.1) PPINT 20
C
20 FORMAT (27H BUCKLING ELEMENTS (TYPE 9)////
1 5H ELEM,3X,4HNODE,3X,4HNODE,11X,20HMAXIMUM AXIAL FORCES,
2 19X,18HMAXIMUM EXTENSIONS,12X,25HACCUM. PLASTIC EXTENSIONS/
3 5H NO.,3X,4H I,3X,4H J,5X,7HTENSION,3X,4HTIME,
4 6X,5HCOMP,3X,4HTIME,5X,8HPOSITIVE,3X,4HTIME,
5 3X,8HNEGATIVE,3X,4HTIME,7X,8HPOSITIVE,5X,8HNEGATIVE/)
C
PRINT 30,IMEM,NODI,NODJ,SENP,TSENP,SENN,TSENN,VENP,TVENP,VENN,TVEN
1N,VPACP,VPACN
30 FORMAT (I4,17,I7,2X,2(F11.2,F7.2),2X,2(F11.5,F7.2),2X,2F13.5)
C
RETURN
END

```

SUBROUTINE FST9 (/STIF/,/KOD/)

C  
C  
C  
C

FORM AXIAL STIFFNESS

COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,  
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8  
2 REST(177)

KYY=KOD+1

GO TO (10,20,30,40,50,60,70,80,90,100),KYY

10 STIF=EAL

GO TO 110

20 STIF=EAL1

GO TO 110

30 STIF=EAL

GO TO 110

40 STIF=0.001\*EAL

GO TO 110

50 STIF=EAL4

GO TO 110

60 STIF=EAL

GO TO 110

70 STIF=EAL6

GO TO 110

80 STIF=EAL7

GO TO 110

90 STIF=EAL8

GO TO 110

100 STIF=0.001\*EAL

C

110 RETURN

END

SUBROUTINE LAW9

C  
C  
C

GENERATE P-DELTA HYSTERESIS CURVE

COMMON /INPFL/ IMEM, KST, LM(4), KGEOM, EAL, FL, COSA, SINA, PFAC, RATIO,  
1 DELTY, KODYX, KODY, XPP, X3RE, EAL1, EAL4, EAL6, EAL7, EAL8,  
2 X5, Y5, P5, X6, X5D, Y5D, P5D, INDRE, IVRTX, REST(168)

C  
C  
C

RESIDUAL ELONGATION, RE

SLEND=60./RATIO  
RE=0.0175\*(0.55\*X3RE/SLEND+0.0002\*X3RE\*\*2)  
RE=RE\*FL/DELTY

C

XPP=XPP+RE  
XPP1=1.+XPP  
BETA=1./3.  
DENOM=(XPP1-X3RE)+(1.+PFAC)/BETA  
XNUMR=XPP-X3RE-PFAC  
Y5=RATIO\*XNUMR/DENOM  
X5=XPP-Y5/BETA  
SLOP4=(Y5+PFAC)/(X5-X3RE)  
EAL4=EAL\*SLOP4  
SLOP8=(1.-Y5)/(XPP1-X5)  
EAL8=EAL\*SLOP8  
X6=X5-Y5-PFAC  
XPP=XPP-RE

C

RETURN  
END



SUBROUTINE VRTX9 (/FACAC/,/DVAX/,/TIME/)

AXIAL STATE DETERMINATION IN TENSION AND VERTEX REGIONS

COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,PL,COSA,SINA,PFAC,RATIO,  
 1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,  
 2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,  
 3 TELOG,SEP,VIOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,  
 4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,  
 5 PYP,PYN,PYNC,REST(147)

INITIALIZE

FACAC=0.

10 FACTOR=1.-FACAC  
 KCDYI=KODY+1  
 IF (KCDYI.EQ.10) KODYI=4  
 GO TO (20,30,40,50),KCDYI

ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE

20 DSEP=EAL\*DVAX  
 IF (DSEP) 21,50,22  
 21 FAC=(PYN-SEP)/DSEP  
 IF (FAC.GE.FACTOR) GO TO 23  
 FACTOR=FAC  
 SEP=PYN  
 KODY=1

SENN=PYN  
 TSENN=TIME  
 GO TO 60

22 FAC=(PYP-SEP)/DSEP  
 IF (FAC.GE.FACTOR) GO TO 23  
 FACTOR=FAC  
 SEP=PYP  
 KODY=9  
 GO TO 60

23 SEP=SEP+FACTOR\*DSEP  
 GO TO 60

ON SLOPE 1, GET FACTOR FOR STATUS CHANGE

30 JS1=EAL1\*DVAX  
 IF (EAL1.NE.0) GO TO 34  
 KODY=3  
 IVRTX=1  
 GO TO 80

34 IF (DS1) 31,60,32

BUCKLED BUT LOADING

```

31 KODY=2
   GO TO 40
C
C   BUCKLING AND CONTINUING
C
32 FAC=(PYNC-SEP)/DS1
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYNC
   KODY=3
C
C   UPDATE PLASTIC DEFORMATION
C
   VPACN=VPACN+FACTOR*DVAX
   GO TO 70
C
33 SEP=SEP+FACTOR*DS1
C
C   UPDATE PLASTIC DEFORMATION
C
   VPACN=VPACN+FACTOR*DVAX
   GO TO 60
C
C   ON SLOPE 2, GET FACTOR FOR STATUS CHANGE
C
40 DS2=EAL*DVAX
   IF (DS2) 41,60,42
C
C   BUCKLING AND CONTINUING
C
41 KODY=1
   GO TO 30
C
C   BUCKLED BUT LOADING
C
42 FAC=(PYNC-SEP)/DS2
   IF (FAC.GE.FACTOR) GO TO 43
   FACTOR=FAC
   SEP=PYNC
   KODY=3
   GO TO 70
C
43 SEP=SEP+FACTOR*DS2
   GO TO 60
C
C   ON SLOPE 9, TENSION YIELDING
C
50 IF (DVAX.LT.0) GO TO 51
   DVP=FACTOR*DVAX
   XPP=XPP+DVP/DELTY
   VPACP=VPACP+DVP
   GO TO 80
C
C   YIELDED BUT UNLOADING
C

```

```
51 KDDY=0
   X3RE=XPP-PFAC
   GO TO 20
```

C

```
60 FACAC=FACAC+FACTOR
   IF (FACAC.LT.0.9999999) GC TO 10
   RETURN
```

C

```
70 FACAC=FACAC+FACTOR
   IVRTX=1
```

C

```
80 RETURN
   END
```

## APPENDIX A-2

FORTRAN LISTING OF END MOMENT-BUCKLING  
ELEMENT EL10

SUBROUTINE INEL10 (/KCCNT//,/FCONT//,/NDOF//,/NINFC//,/ID//,/X//,/Y//,/NN/  
1)

C

```
COMMON /INFEL/ IMEM, KST, IM(6), RGEOM, FL, COSA, SINA, A(2,6), EK11,
1 EK22, EK12, FSH, EAL, EK11H, EK22H, KODYX(2),
2 KODY(2), BMTOT(2), SPTOT(2), FTOT(2), PRTOT(2), SENP(8),
3 SENN(8), TENE(8), TENN(8), PRACP(2), PRACN(2), BMEP(2),
4 SDACT(3), EMY(2,2), NODI, NODJ, KOUTDT, PR12, PR21,
5 PMX(3,2,2), A1(4,2,2), A2(4,2,2),
6 PFAC, RATIC, DELTY, KODYX1, KODY1, XPP, X3RE, EAL1, EAL4,
7 EAL6, EAL7, EAL8, X5, Y5, P5, X6, X5D, Y5D, P5D, INDRE,
8 IVRTX, IELCG, VTOT, XTOT, VPACP, VPACN, VENP, TVENP, VENN,
9 TVENN, PYP, PYN, PYNC, REST(30)
COMMON /WORK/ SPF(8), SSFF(8), DD(6), GA(6,6), FFEF(6), FF(6),
1 PMAX(5,2,40), AA1(4,2,40), AA2(4,2,40), XM(5,2),
2 FIYP(40,9), FEF(35,7), KDFEF(35), FINIT(30,6),
3 NMEM, NMBT, NSURF, NFEF, NINT, INODI, INODJ, INC, IINC,
4 IMBT, IIMBT, IKSEI, IKSPJ, IKGM, IKDT, KFDL, IKPDL, KFLI,
5 IKFLL, FDI, FFEL, FLLF, AK, SLEND, PYP1, PYP2, PYN1, PYN2,
6 KS1, KS2, XL, YL, DET, PLL, SS, W(25)
```

C

```
DIMENSION KCONT(1), ID(MN,1), X(1), Y(1), COM(1), AST(2), YESNO(2),
1 KSF(2), FTYP1(40,3)
EQUIVALENCE (IMEM, COM(1))
DATA AST/2H ,2H */
DATA YESNO/4H YES,4H NC /
```

C

C  
C

DATA INPUT, END MOMENT-BUCKLING ELEMENTS

```
NDOF=6
NINFC=170
KCOM=KCONT(1)
NMEM=KCCNT(2)
NMBT=KCONT(3)
NSURF=KCONT(4)
NFEF=KCONT(5)
NINT=KCONT(6)
PRINT 10, (KCONT(I), I=2,6)
10 FORMAT(39H END MOMENT-BUCKLING ELEMENTS (TYPE '0')////
1 34H NO. OF ELEMENTS =I4/
2 34H NO. OF STIFFNESS TYPES =I4/
3 34H NO. OF YIELD SURFACES =I4/
4 34H NO. OF FIXED END FORCE PATTERNS =I4/
5 34H NO. OF INITIAL FORCE PATTERNS =I4)
```

C

C  
C

INPUT STIFFNESS PROPERTIES

```
PRINT 20
20 FORMAT(///16H STIFFNESS TYPES//
1 5H TYPE,6X,7H YOUNGS,4X,9HHARDENING,6X,7HSECTION,
2 3X,9HREFERENCE,6X,26HFLEXURAL STIFFNESS FACTORS,
3 8X,5HSHEAR,5X,7HFCISSON/
4 5H NO.,6X,7HMODULUS,4X,9H RATIO ,6X,7H AREA ,
5 3X,9H INERTIA ,6X,26H II JJ IJ ,
6 8X,5H AREA,5X,7H RATIO /)
```

```

DO 30 N=1, NMBT
  READ 40, I, (FTYP(N,J), J=1,9)
30 PRINT 50, N, (FTYP(N,J), J=1,9)
40 FORMAT (15, 4F10.0, 3F5.0, 2F10.0)
50 FORMAT (I4, E14.4, E13.4, 2F12.2, 3X, 3F10.3, F13.2, F11.3)

C
C
C   INPUT RADIUS OF GYRATION , K AND PHI FACTORS

  PRINT 60
60 FORMAT (////16H STIFFNESS TYPES//
1      5H TYPE, 6X, 10H RADIUS OF, 6X, 18H EQUIVALENT LENGTH,
2      6X, 19H STRENGTH REDUCTION/
3      5H NO., 6X, 10H GYRATION , 12X, 12H COEFFICIENT,
4      12X, 7H FACTOR)
  DO 70 N=1, NMBT
  READ 80, I, (FTYP1(N,J), J=1,3)
70 PRINT 90, N, (FTYP1(N,J), J=1,3)
80 FORMAT (15, 3F10.0)
90 FORMAT (I4, 4X, F10.3, 2(10X, F10.3))

C
C
C   INPUT M-P YIELD SURFACE PROPERTIES

  PRINT 110
110 FORMAT (////25H YIELD SURFACE PROPERTIES//
1      8H SURFACE,          9X, 13HYIELD MOMENTS, 15X,
2      12HYIELD FORCES, 9X, 16HCOORDINATES OF A, 6X,
3      16HCOORDINATES OF E/
4      8H NO. ,          5X, 8HPOSITIVE, 5X, 8HNEGATIVE, 8X,
5      5HCOMPX, 6X, 7HTENSION, 6X, 16HMOMENT FORCE, 6X,
6      16HMOMENT FORCE/)
  DO 200 IYT=1, NSURF
  READ 120, I, (SFF(J), J=1,8)
  PRINT 130, IYT, (SFF(J), J=1,8)
120 FORMAT (15, 4F10.0, 4F5.0)
130 FORMAT (15, F15.2, 3F13.2, 2(2X, 2F10.3))
  SFF(2) = -ABS(SFF(2))
  SFF(3) = -ABS(SFF(3))
  SFF(4) = ABS(SFF(4))
  IF (SFF(6) .EQ. 0.) SFF(6) = 1.E-6
  IF (SFF(8) .EQ. 0.) SFF(8) = 1.E-6

C
C
C   STEEL TYPE

  PMAX(1,1,IYT) = SFF(3)
  PMAX(1,2,IYT) = SFF(3)
  PMAX(2,1,IYT) = SFF(3) * SFF(6)
  PMAX(2,2,IYT) = SFF(3) * SFF(8)
  PMAX(3,1,IYT) = 0.
  PMAX(3,2,IYT) = 0.
  PMAX(4,1,IYT) = SFF(4) * SFF(6)
  PMAX(4,2,IYT) = SFF(4) * SFF(8)
  PMAX(5,1,IYT) = SFF(4)
  PMAX(5,2,IYT) = SFF(4)
  XM(1,1) = 0.
  XM(1,2) = 0.

```

```

XM (2,1)=SFF (1)*SFF (5)
XM (2,2)=SFF (2)*SFF (7)
XM (3,1)=SFF (1)
XM (3,2)=SFF (2)
XM (4,1)=XM (2,1)
XM (4,2)=XM (2,2)
XM (5,1)=0.
XM (5,2)=0.
DO 190 J=1,2
PP2=PMAX (1,J,IYT)
XM2=XM (1,J)
DO 190 I=1,4
PP1=PP2
XM1=XM2
PP2=PMAX (I+1,J,IYT)
XM2=XM (I+1,J)
DENOM=XM1*PP2-XM2*PP1
AA2 (I,J,IYT)=(PP2-PP1)/DENOM
190 AA1 (I,J,IYT)=(XM1-XM2)/DENOM
200 CONTINUE

```

```

C
C   FIXED END FORCE PATTERNS
C

```

```

IF (NFEF.EQ.0) GO TO 250
PRINT 210
210 FORMAT (////25H FIXED END FORCE PATTERNS//
1      8H PATTERN,3X,4HAXIS,7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,
2      7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,5X,8HLL. RED./
3      8H NO. ,3X,4HCODE,7X,5HAT I,7X,5HAT I,6X,6H AT I ,
4      7X,5HAT J,7X,5HAT J,6X,6H AT J ,5X,8H FACTOR /)
DO 220 N=1,NFEF
READ 230, I,KDFEF (N) ,(FEF (N,J) ,J=1,7)
220 PRINT 240, N,KDFEF (N) ,(FEF (N,J) ,J=1,7)
230 FORMAT (215,7F10.0)
240 FORMAT (I5,I9,F13.2,5F12.2,F12.3)

```

```

C
C   INITIAL FORCE PATTERNS
C

```

```

250 IF (NINT.EQ.0) GO TO 300
PRINT 260
260 FORMAT (////28H INITIAL END FORCE PATTERNS //
1      8H PATTERN,7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,7X,5HAXIAL,
2      7X,5HSHEAR,6X,6HMOMENT/
3      8H NO. ,7X,5HAT I,7X,5HAT I,6X,6H AT I ,7X,5HAT J,
4      7X,5HAT J,6X,6H AT J /)
DO 270 N=1,NINT
READ 280, I,(FINIT (N,J) ,J=1,6)
270 PRINT 290, N,(FINIT (N,J) ,J=1,6)
280 FORMAT (I5,6F10.0)
290 FORMAT (I5,3X,6F12.2)

```

```

C
C   ELEMENT SPECIFICATION
C

```

```

300 PRINT 310
310 FORMAT (////22H ELEMENT SPECIFICATION//

```

```

1      3X,4HELEM,3X,4HNCDE,2X,4HNODE,2X,4HNODE,2X,4HSTIF,
2      2X,14HYIELD SURFACES,2X,4HGEOM,2X,4HTIME,3X,
3      12HFEF PATTERNS,3X,17HFEF SCALE FACTORS,3X,
4      16H INITIAL FORCES /
5      3X,4H NO.,3X,4H I ,2X,4H J ,2X,4HDIFF,2X,4HTYPE,
6      2X,14H END I END J ,2X,4HSTIF,2X,4HHIST,3X,
7      12H DL LL ,3X,17H DL LL ,3X,
8      17H NO. SCALE FAC./)

```

C

```

      DO 320 J=36,84
320 COM(J)=0.
      KJDYX(1)=0
      KJDYX(2)=0
      KODY(1)=0
      KODY(2)=0
      KST=0
      DO 325 J=145,167
325 COM(J)=0.
      KODY1=0
      KODY1=0
      XPP=0.

```

C

```

      IMEM=1
330 READ 340, INEL, INODI, INODJ, IINC, IMBT,          IKSPI, IKSPJ, IKGM, IKDT,
      1IKFDL, IKFLL, FFDL, FLL, IINIT, FFINIT
340 FORMAT (11I5,2F5.0, I5, F5.0)

```

C

```

      IF (INEL.GT.IMEM) GO TO 380
350 NODI=INODI
      NODJ=INODJ
      INC=IINC
      IF (INC.EQ.0) INC=1
      IMBT=IMBT
      KSF(1)=IKSPI
      KSF(2)=IKSPJ
      KGEOM=IKGM
      KOUTDI=IKDT
      YNG=YESNO(2)
      IF (KGEOM.NE.0) YNG=YESNC(1)
      YNT=YESNO(2)
      IF (KOUTDI.NE.0) YNT=YESNC(1)
      KFDL=IKFDL
      KFLL=IKFLL
      FDL=FFDL
      FLLM=FFLL
      FLLF=1.
      IF (KFLL.EQ.0) GO TO 360
      FLLF=FEF(IKFLL,7)
      IF (FLLF.EQ.0.) FLLF=1.E-6
360 IINIT=IINIT
      FINI=FFINIT
      ASTT=AST(1)
      IF (INEL-NMEM) 330,380,330

```

C

```

370 NODI=NODI+INC

```



```

NODJ=NODJ+INC
ASTT=AST(2)
C
380 PRINT 390, ASTT, IMEM, NCDI, NODJ, INC, IMBT, KSF(1), KSF(2), YNG, YNT
1, KFDL, KFL, FDL, FLLM, INIT, FINT
390 FORMAT (A2, I4, I7, 3I6, 2I7, 5X, A4, 2X, A4, I7, I6, F11.2, F10.2, I7, F11.2)
C
C LOCATION MATRIX
C
DO 400 I=1, 3
LM(I)=ID(NODI, I)
400 LM(I+3)=ID(NODJ, I)
CALL BAND
C
C ELEMENT PROPERTIES
C
XL=X(NODJ)-X(NODI)
YL=Y(NODJ)-Y(NODI)
FL=SQRT(XL**2+YL**2)
COSA=XL/FL
SINA=YL/FL
YMOD=FTYP(IMBT, 1)
PSH=FTYP(IMBT, 2)
PPSH=1.-PSH
PSH=PSH/PPSH
AREA=FTYP(IMBT, 3)
EAL=YMOD*AREA/FL
RAD=FTYP1(IMBT, 1)
AK=FTYP1(IMBT, 2)
SLEND=AK*FL/RAD
RATIO=60./SLEND
KS1=KSF(1)
KS2=KSF(2)
PYP1=PMAX(5, 1, KS1)
PYP2=PMAX(5, 1, KS2)
IF(PYP1.LE.PYP2) GO TO 420
PYP=PYP2
GO TO 421
420 PYP=PYP1
421 CONTINUE
PYN1=PMAX(1, 1, KS1)
PYN2=PMAX(1, 1, KS2)
IF(PYN1.LE.PYN2) GO TO 425
PYN=PYN2
GO TO 426
425 PYN=PYN1
426 CONTINUE
PHI=FTYP1(IMBT, 3)
PYNC=PHI*PYN
PFAC=ABS(PYNC/PYP)
SLOP1=(PFAC*(1.-PHI))/(PFAC-5.*PHI)
EAL1=SLOP1*EAL
K3RE=-PFAC
DELTY=PYP/EAL
EIL=YMOD*FTYP(IMBT, 4)*PPSH/FL

```

```

FACL=FTYP(IMBT,5)
FACR=FTYP(IMBT,6)
FACLR=FTYP(IMBT,7)
IF (FACL.EQ.0.) FACL=1.E-6
IF (FACR.EQ.0.) FACR=1.E-6
IF (FTYP(IMBT,8).EQ.0.) GC TO 430
SHFAC=EIL/(FTYP(IMBT,1)/(2.*(1.+FTYP(IMBT,9))))*FTYP(IMBT,8)*FL*PPS
1H)
DET=FACL*FACR-FACLR**2
FII=FACR/DET+SHFAC
FJJ=FACL/DET+SHFAC
FIJ=-FACLR/DET+SHFAC
DET=FII*FJJ-FIJ**2
FACR=FII/DET
FACL=FJJ/DET
FACLR=-FIJ/DET
430 EK11=EIL*FACL
EK22=EIL*FACR
EK12=EIL*FACLR
EK11H=EK11-EK12**2/EK22
EK22H=EK22-EK12**2/EK11
PR12=EK12/EK22
PR21=EK12/EK11

```

```

C
C 1-P YIELD SURFACE EQUATION DATA FOR EACH END OF AN ELEMENT
C

```

```

DO 450 K=1,2
KK=KSF(K)
DO 450 J=1,2
DO 440 I=1,3
440 PMX(I,J,K)=PMAX(I+1,J,KK)
DO 450 I=1,4
A2(I,J,K)=PPSH/AA2(I,J,KK)
450 A1(I,J,K)=AA1(I,J,KK)*A2(I,J,K)

```

```

C
C DISPLACEMENT TRANSFORMATION
C

```

```

A(1,1)=-SINA/FL
A(1,2)=COSA/FL
A(1,3)=1.
A(1,4)=-A(1,1)
A(1,5)=-A(1,2)
A(1,6)=0.
A(2,1)=A(1,1)
A(2,2)=A(1,2)
A(2,3)=0.
A(2,4)=A(1,4)
A(2,5)=A(1,5)
A(2,6)=1.

```

```

C
C LOADS DUE TO FIXED END FORCES
C

```

```

DO 480 I=1,6
SFF(I)=0.
480 SSFF(I)=0.

```

```

      IF (KF DL+KF LL.EQ.0) GC TC 610
      DO 490 I=1,6
      DO 490 J=1,6
490 GA (I,J)=0.
      GA (1,1)=COS A
      GA (1,2)=SIN A
      GA (2,1)=-SIN A
      GA (2,2)=COS A
      GA (3,3)=1.
      GA (4,4)=COS A
      GA (4,5)=SIN A
      GA (5,4)=-SIN A
      GA (5,5)=COS A
      GA (6,6)=1.
C
      IF (KF DL.EQ.0) GO TO 530
      DO 500 I=1,6
500 PFEF (I)=FEF (KF DL,I)*PDL
      IF (KDFEF (KF DL).EQ.0) GC TO 510
      CALL MULT (GA,PFEF,SFF,6,6,1)
      GO TO 530
510 DO 520 I=1,6
520 SFF (I)=PFEF (I)
C
530 IF (KF LL.EQ.0) GO TO 570
      DO 540 I=1,6
      FLL=FLLF*FLLM
      IF (I.EQ.3.OR.I.EQ.6) FII=FLLM
540 PFEF (I)=FEF (KF LL,I)*FLL
      IF (KDFEF (KF LL).EQ.0) GC TC 550
      CALL MULT (GA,PFEF,SSFF,6,6,1)
      GO TO 570
550 DO 560 I=1,6
560 SSFF (I)=PFEF (I)
C
570 DO 580 I=1,6
580 PF (I)=SFF (I)+SSFF (I)
C
      CALL MULT (GA,PF,DD,6,6,1)
      CALL SFORCE (DD)
C
C      MODIFY TO GET INITIAL ELEMENT FORCES
C
      DO 600 I=1,6
      FLL=1./FLLF
      IF (I.EQ.3.OR.I.EQ.6) FII=1.
600 SFF (I)=SFF (I)+SSFF (I)*FII
C
C      INITIAL FORCES
C
610 IF (INIT.EQ.0) GO TO 630
      DO 620 I=1,6
620 SFF (I)=SFF (I)+FINIT (INIT,I)*FINT
C
C      INITIALIZE ARRAYS

```

```
C
630 BMEP(1)=SFF(3)*PPSH
    BMEP(2)=SFF(6)*PPSH
    FTOT(1)=SFF(1)
    FTOT(2)=SFF(4)
    SFTOT(1)=SFF(2)
    SFTOT(2)=SFF(5)
    BMTOT(1)=SFF(3)
    BMTOT(2)=SFF(6)
    DO 650 I=1,6
    SS=BMTOT(I)
    IF (SS.LL.0.) GO TO 640
    SENP(I)=SS
    GO TO 650
640 SENN(I)=SS
650 CONTINUE

C
C   YIELD MOMENTS FOR INITIAL FORCE STATE
C
C   CALL YMOM10
C
C   CALL FINISH
C
C   GENERATE MISSING ELEMENTS
C
    IF (IMEM.EQ.NMEM) RETURN
    IMEM=IMEM+1
    IF (IMEM.EQ.INEL) GO TO 350
    GO TO 370

C
    END
```

SUBROUTINE STIF10 (/MSTEP/,/NDOF/,/NINFC/,/COMS/,/FK/,/DFAC/)

C

```

COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1 EK22,EK12,PSH,EAL,EK11H,EK22H, KODYX(2),
2 KODY(2),EMICT(2),SPTOT(2),FTOT(2),PRTOT(2),SENP(8),
3 SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4 SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5 PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6 PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7 EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8 IVRTX,IFLOG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9 TVENN,EYF,EYN,PYNC,REST(30)
COMMON /WORK/ ST(2,2),SIT(2,2),ATK(6,2),AA(2,6),PFL,AXK,FAC,
1 FFK(6,6),FSK(6,6),N(1893)

```

C

```

DIMENSION COM(1),COMS(1),FK(6,6)
EQUIVALENCE (IMEM,COM(1))

```

C

C

C

STIFFNESS FORMULATION, END MOMENT-BUCKLING ELEMENTS

```

DO 10 J=3,35
10 COM(J)=COMS(J)
DO 15 J=141,149
15 COM(J)=COMS(J)

```

C

C

C

CURRENT AXIAL STIFFNESS

C

CALL FST10A (STIF,KODY1)

C

C

C

PREVIOUS STIFFNESS

C

```

IF(MSTEP.LT.2) GO TO 20
CALL FST10A (STIFF,KODYX1)

```

C

C

C

STIFFNESS DIFFERENCE

```

STIF=STIF-STIFF
20 CONTINUE
DO 30 I=1,36
30 FSK(I,1)=J.
AXK=STIF*CGSA**2
FSK(1,1)=AXK
FSK(1,4)=-AXK
FSK(4,4)=AXK
AXK=STIF*SINA**2
FSK(2,2)=AXK
FSK(2,5)=-AXK
FSK(5,5)=AXK
AXK=STIF*SINA*COXA
FSK(1,2)=AXK
FSK(1,5)=-AXK
FSK(2,4)=-AXK
FSK(4,5)=AXK
DO 40 I=1,6
DO 40 J=1,6

```

```

40 FSK(J,I)=FSK(I,J)
C
C   CURRENT FLEXURAL STIFFNESS, ELASTO-PLASTIC PART
C
C   CALL FST10B (ST,KODY)
C
C   PREVIOUS STIFFNESS
C
C   IF (MSTEP.LT.2) GO TO 50
C   CALL FST10B (STT,KODYX)
C
C   STIFFNESS DIFFERENCE
C
C   DO 60 I=1,4
60 ST(I,1)=ST(I,1)-STT(I,1)
   CALL MULIST (A,ST,ATK,FF,6,2)
C
C   GET TOTAL STIFFNESS
C
C   DO 70 I=1,6
C   DO 70 J=1,6
70 FK(I,J)=FK(I,J)+FSK(I,J)
   RETURN
C
C   ORIGINAL STIFFNESS AT STEP 0, BETA-0, CORRKN AT STEP 1
C
C   50 FAC=1.
C   IF (MSTEP.NE.1) GO TO 85
C   FAC=DFAC
C   DO 80 I=1,36
80 FSK(I,1)=FSK(I,1)*FAC
85 CC=(1.+PSH)*FAC
C   DO 90 I=1,4
90 ST(I,1)=ST(I,1)*CC
   CALL MULIST (A,ST,ATK,FK,6,2)
C
C   GET TOTAL INITIAL STIFFNESS
C
C   DO 100 I=1,6
C   DO 100 J=1,6
100 FK(I,J)=FK(I,J)+FSK(I,J)
C
C   ADD GEOMETRIC STIFFNESS
C
C   IF (MSTEP.EQ.0.OR.KGFCM.EQ.0) GO TO 120
C   PFL=(COHS(41)-CGMS(40))/(2.*FL)
C   DO 110 I=1,4
110 ST(I,1)=PFL
C   DO 130 I=1,12
130 AA(I,1)=0.
   AA(1,1)=-SINA
   AA(1,2)=COISA
   AA(2,4)=SINA
   AA(2,5)=-COISA
   CALL MULIST (AA,ST,ATK,FFK,6,2)

```

```
      DO 140 I=1,36  
140 FK(I,1)=FK(I,1)+PPK(I,1)  
C  
120 RETURN  
C  
      END
```

SUBROUTINE RESP10 (/NDOF/, /NINFC/, /KBAL/, /KPR/, /COMS/, /DDISM/, /DD/,  
1/TIME/, /VELM/, /DFAC/, /DELTA/)

C  
C  
C

STATE DETERMINATION, END MOMENT-BUCKLING ELEMENTS

COMMON /INFEL/ IMEM, KST, IM(6), KGEOM, FL, COSA, SINA, A(2,5), EK11,  
1 EK22, EK12, FSH, EAL, EK11H, EK22H, KODYX(2),  
2 KODY(2), BMTOT(2), SFTOT(2), FTOT(2), PRTOT(2), SENP(8),  
3 SENN(8), TFNP(8), TENN(8), PRACP(2), PFACN(2), BMEP(2),  
4 SDACT(3), BMY(2,2), NODI, NODJ, KOUTDT, PR12, PR21,  
5 PMX(3,2,2), A1(4,2,2), A2(4,2,2),  
6 PFAC, RATIC, DELTY, KODYX1, KODY1, XPP, X3RE, EAL1, EAL4,  
7 EAL6, EAL7, EAL8, X5, Y5, P5, X6, X5D, Y5D, P5D, INDRE,  
8 IVRTX, IETOG, VTOT, XTOT, VPACP, VPACN, VENP, TVENP, VENN,  
9 TVFNN, PYP, PYN, PYNC, REST(30)  
COMMON /WORK/ DVR(2), DPR(2), DBM(2), BBMTOT(2), BML(2), BMEL(2),  
1 DVAX, SLIN, FACAC, FACTOR, FAC, DSP, BMIUB, BMJUB, SFUB,  
2 DSEP, DVP, SLCP6, SLOP7, DS4, DS5, DS6, DS7, DS8, EALE, FOU8,  
3 KBAL1, KST1, W(1966)

C  
C  
C

DIMENSION COM(1), COMS(1), DDISM(1), DD(1), VELM(1), NOD(2)  
EQUIVALENCE (IMEM, COM(1)), (NODI, NOD(1))

DO 10 J=1, NINFC  
10 COM(J) = COMS(J)  
KODYX(1) = KODY(1)  
KODYX(2) = KODY(2)  
KODYX1 = KODY1  
IF (IMEM.EQ.1) IPED=0

C  
C  
C

DEFORMATION INCREMENTS

DVAX = COSA \* (DDISM(4) - DDISM(1)) + SINA \* (DDISM(5) - DDISM(2))  
ROT = (SINA \* (DDISM(4) - DDISM(1)) + COSA \* (DDISM(2) - DDISM(5))) / FL  
DVR(1) = DDISM(3) + ROT  
DVR(2) = DDISM(6) + POT  
VTOT = VTOT + DVAX  
SEP = (FTOT(2) - FTOT(1)) \* 0.5

C  
C  
C

AXIAL FORCE INCREMENT

CALL FST10A (EALF, KODY1)  
SLIN = SEP + EALE \* DVAX

C  
C  
C

AXIAL STATE DETERMINATION

FVRTX = 0.

C  
C  
C

CHECK VERTEX STATE

IF (IVRTX.EQ.0) CALL VRTX10 (FVRTX, DVAX, SEP, TIME)

C  
C  
C

IF (IVRTX.EQ.0) GO TO 120

FACAC = FVRTX



```

C
20 FACTOR=1.-FACAC
   KJDYI=KODY1+1
   GO TO (30,120,120,40,50,60,70,80,90,100),KODYI
C
C   ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
30 DSEP=EAL*DVAX
   IF (DSEP) 31,110,32
31 FAC=(PYNC-SEP)/DSEP
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYNC
   KJDY1=3
   GO TO 110
C
32 FAC=(PYP-SEP)/DSEP
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYP
   KJDY1=9
   GO TO 110
C
33 SEP=SEP+FACTOR*DSEP
   GO TO 110
C
C   ON SLOPE 3, BUCKLING AND CONTINUING
C
40 IF (DVAX.GT.0.) GO TO 41
C
C   UPDATE PLASTIC DEFORMATIONS
C
   DVP=FACTOR*DVAX
   VPACN=VPACN+DVP
   GO TO 120
C
C   BUCKLING AND UNLOADING
C
41 XTOT=(VTOT-DVAX)/DELTY
   IF (XTOT.GE.X3RE) GO TO 42
C
C   ESTABLISH NEW STIFFNESS FOR REVERSE
C
   X3RE=XTOT
   IELOG=1
   CALL LAW10
   P5=Y5*PYP
   KJDY1=4
   GO TO 50
C
C   USE OLD STIFFNESS FOR REVERSE
C
42 IF (INDRE.EQ.2.AND.YTOT.LE.X6) GO TO 43
   IF ((X5D-XTOT).EQ.0.) GO TO 44
   SLOP5=(Y5D+PFAC)/(X5D-XTOT)

```

```

EAL6=EAL*SLOP6
KODY1=6
GO TO 70

C
43 SLOP7=(Y5+PFAC)/(X5-XTOT)
EAL7=EAL*SLOP7
KODY1=7
GO TO 80
44 KODY1=5
GO TO 60

C
C
C
ON SLOPE 4, GET FACTOR FOR STATUS CHANGE

50 DS4=EAL4*DVAX
IF (DS4) 51,110,54
51 IF (SEP) 52,52,53
52 FAC=(PYNC-SEP)/DS4
IF (FAC.GE.FACTOR) GO TO 55
FACTOR=FAC
SEP=PYNC
KODY1=3
GO TO 110
53 X5D=VTOT/DELTY
Y5D=SEP/PYP
P5D=SEP
INDEE=1
KODY1=5
GO TO 60
54 FAC=(P5-SEP)/DS4
IF (FAC.GE.FACTOR) GO TO 55
FACTOR=FAC
SEP=P5
KODY1=8
GO TO 110
55 SEP=SEP+FACTOR*DS4
GO TO 110

C
C
C
ON SLOPE 5, GET FACTOR FOR STATUS CHANGE

60 DS5=EAL*DVAX
IF (DS5) 61,110,62
61 FAC=(PYNC-SEP)/DS5
IF (FAC.GE.FACTOR) GO TO 65
FACTOR=FAC
SEP=PYNC
KODY1=3
GO TO 110
62 FAC=(P5D-SEP)/DS5
IF (FAC.GE.FACTOR) GO TO 65
FACTOR=FAC
SEP=P5D
GO TO (63,64),INDEE
63 KODY1=4
GO TO 110
64 KODY1=8

```

GO TO 110  
 65 SEP=SEP+FACTOR\*DS5  
 GO TO 110

C  
 C  
 C

ON SLOPE 6, GET FACTOR FOR STATUS CHANGE

70 DS6=EAL6\*DVAX  
 IF (DS6) 71,110,72  
 71 FAC=(PYNC-SEP)/DS6  
 IF (FAC.GE.FACTOR) GO TO 75  
 FACTOR=FAC  
 SEP=PYNC  
 KODY1=3  
 GO TO 110  
 72 FAC=(P5D-SEP)/DS6  
 IF (FAC.GE.FACTOR) GO TO 75  
 FACTOR=FAC  
 SEP=P5D  
 GO TO (73,74),INDRE  
 73 KODY1=4  
 GO TO 110  
 74 KODY1=8  
 GO TO 110  
 75 SEP=SEP+FACTOR\*DS6  
 GO TO 110

C  
 C  
 C

ON SLOPE 7, GET FACTOR FOR STATUS CHANGE

80 DS7=EAL7\*DVAX  
 IF (DS7) 81,110,82  
 81 FAC=(PYNC-SEP)/DS7  
 IF (FAC.GE.FACTOR) GO TO 83  
 FACTOR=FAC  
 SEP=PYNC  
 KODY1=3  
 GO TO 110  
 82 FAC=(P5-SEP)/DS7  
 IF (FAC.GE.FACTOR) GO TO 83  
 FACTOR=FAC  
 SEP=P5  
 KODY1=8  
 GO TO 110  
 83 SEP=SEP+FACTOR\*DS7  
 GO TO 110

C  
 C  
 C

ON SLOPE 8, GET FACTOR FOR STATUS CHANGE

90 DS8=EAL8\*DVAX  
 IF (DS8) 91,110,92  
 91 K5D=VFOT/DELTY  
 Y5D=SEP/PYP  
 P5D=SEP  
 INDRE=2  
 KODY1=5  
 GO TO 60

```

C
  92 FAC=(PYP-SEP)/DS8
    IF (FAC.GE.FACTOR) GO TO 93
    FACTOR=FAC
    SEP=PYP
    KODY1=9
    GO TO 110
  93 SEP=SEP+FACTOR*DS8
    GO TO 110

C
  ON SLOPE 9, YIELDED BUT CCNTINUING

C
  100 IF (DVAX.LT.0.) GO TO 101

C
  UPDATE PLASTIC DEFORMATIONS

C
  DVP=FACTOR*DVAX
  XPP=XPP+DVP/DELTY
  VPACP=VPACP+DVP
  GO TO 120

C
  YIELDED BUT UNLOADING

C
  101 KODY1=0

C
  RESIDUAL ELONGATION, RE

C
  IF (IELOG.NE.1) GO TO 105
  SLEND=60./RATIO
  RE=0.0175*(0.55*X3RE/SLEND+C.0002*X3RE**2)
  RE=RE*FL/DELTY
  XPP=XPP+RE
  IELOG=0

C
  105 K3RE=XPP-PFAC
    GO TO 30

C
  CHECK FOR COMPLETION OF CYCLE

C
  110 FACAC=FACAC+FACTOR
    IF (FACAC.LT.0.9999999) GO TO 20

C
  NEW FORCE, UNBALANCED FORCE DUE TO YIELD

C
  120 FTOT(2)=SEP
    FTOT(1)=-SEP
    FOUB=SLIN-SEP
    IF (ABS(FOUB).GT.1.E-8) KBAL1=1

C
  ACCUMULATE EXTENSTION ENVELOPES

C
  IF (VENP.GE.VTOT) GO TO 180
  VENP=VTOT
  IVENP=TIME
  GO TO 190

```

```

180 IF (VENN.LE.VTOT) GO TO 190
    VENN=VTOT
    TVENN=TIME
190 CONTINUE
C
C     SET INDICATOR FOR AXIAL STIFFNESS CHANGE
C
    KST1=0
    IF (KODY1.NE.KODY1) KST1=1
C
C     FLEXURAL STATE DETERMINATION
C
C     LINEAR MOMENT INCREMENTS
C
    CALL BM10
    BML(1)=BMEP(1)+DBM(1)
    BML(2)=BMEP(2)+DBM(2)
    BMEEL(1)=BMTOT(1)-BMEP(1)
    BMEEL(2)=BMTOT(2)-BMEP(2)
C
C     TRACE OUT NONLINEAR PATH ON M-P SURFACE
C
    FACAC=0.
    KBAL=0
250 FACTOR=1.-FACAC
C
C     PLASTIC HINGE ROTATIONS
C
    IF (KODY(1)+KODY(2)-1) 280,260,270
260 DPR(1)=DVR(1)+PR21*DVR(2)
    DPR(2)=DVR(2)+PR12*DVR(1)
    GO TO 280
270 DPR(1)=DVR(1)
    DPR(2)=DVR(2)
C
280 KFAC=0
    DO 320 IEND=1,2
C
C     ELASTIC, GET FACTOR FOR STATUS CHANGE
C
    IF (KODY(IEND).NE.0) GO TO 310
    IF (DBM(IEND)) 290,320,300
290 FAC=(BMY(IEND,2)-BMEP(IEND))/DBM(IEND)
    IF (FAC.GE.FACTOR) GO TO 320
    FACTOR=FAC
    BMY=BMY(IEND,2)
    KFAC=IEND
    GO TO 320
300 FAC=(BMY(IEND,1)-BMEP(IEND))/DBM(IEND)
    IF (FAC.GE.FACTOR) GO TO 320
    FACTOR=FAC
    BMY=BMY(IEND,1)
    KFAC=IEND
    GO TO 320
C

```

```

C     CONTINUING TO YIELD - POSITIVE PLASTIC WORK
C
C 310 IF (BMEP(IEND)*DPP(IEND).GE.0.) GO TO 320
C
C     UNLOADING - PLASTIC WORK NEGATIVE
C
C     FACTOR=0.
C     KFAC=0
C     KODY(IEND)=0
C
C 320 CONTINUE
C
C     UPDATE MOMENTS AND HINGE ROTATIONS
C
C     DO 360 IEND=1,2
C     IF (IEND.EQ.KFAC) GO TO 350
C     IF (KODY(IEND).NE.0) GO TO 330
C     BMEP(IEND)=BMEP(IEND)+FACTOR*DBM(IEND)
C     GO TO 360
C 330 DPPR=FACTOR*DPR(IEND)
C     PRTOT(IEND)=PRTOT(IEND)+DPPR
C     IF (DPPR.LT.0.) GO TO 340
C     PRACP(IEND)=PRACP(IEND)+DPPR
C     GO TO 360
C 340 PRACN(IEND)=PRACN(IEND)+DPPR
C     GO TO 360
C 350 BMEP(IEND)=BMY
C     KODY(IEND)=1
C 360 CONTINUE
C
C     CHECK COMPLETION OF CYCLE
C
C     FACAC=FACAC+FACTOR
C     IF (FACAC.GT.0.99999) GO TO 370
C     CALL BM10
C     KBAL=1
C     GO TO 250
C
C     YIELD MOMENTS FOR NEXT STEP
C
C 370 CALL YMOM10
C
C     CHECK FOR OVERTHOOT OF M-F SURFACE
C
C     DO 420 IEND=1,2
C     IF (KODY(IEND).EQ.0) GO TO 400
C     IF (BMEP(IEND).LE.BMY(IEND,1)) GO TO 380
C     BMEP(IEND)=BMY(IEND,1)
C     KBAL=1
C     GO TO 420
C 380 IF (BMEP(IEND).GE.BMY(IEND,2)) GO TO 390
C     BMEP(IEND)=BMY(IEND,2)
C     KBAL=1
C     GO TO 420
C 390 IF (BMEP(IEND).LT.BMY(IEND,1)*0.98.AND.BMFP(IEND).GT.BMY(IEND,2)*0

```

```

1.98) KODY(IEND)=0
GO TO 420
400 IF (BMEP(IEND).LE.BMY(IEND,1)) GO TO 410
    BMEP(IEND)=BMY(IEND,1)
    KBAL=1
    KODY(IEND)=1
    GO TO 420
410 IF (BMEP(IEND).GE.BMY(IEND,2)) GO TO 420
    BMEP(IEND)=BMY(IEND,2)
    KBAL=1
    KODY(IEND)=1
420 CONTINUE

```

C  
C  
C

ELASTIC AND TOTAL FORCES

```

BBMTOT(1)=BMTOT(1)
BBMTOT(2)=BMTOT(2)
BMTOT(1)=BMEP(1)+BMEL(1)+(FK11*DVR(1)+EK12*DVR(2))*PSH
BMTOT(2)=BMEP(2)+BMEL(2)+(EK12*DVR(1)+EK22*DVR(2))*PSH
DSF=(BMTOT(1)-BBMTOT(1)+BMTOT(2)-BBMTOT(2))/FL
SFTOT(1)=SFTOT(1)+DSF
SFTOT(2)=SFTOT(2)-DSF

```

C  
C  
C

UNBALANCED LOADS DUE TO YIELD

```

BMIUB=0.
BMJUB=0.
IF (KBAL.EQ.0) GO TO 430
BMIUB=BML(1)-BMEP(1)
BMJUB=BML(2)-BMEP(2)

```

C  
C  
C

DEFORMATION RATES FOR DAMPING

```

430 IF (DFAC.EQ.0.0.AND.DELTA.EQ.0.0) GO TO 460
IF (TIME.EQ.0.) GO TO 470
KBAL=1
DVAX=COSA*(VELM(4)-VELM(1))+SINA*(VELM(5)-VELM(2))
ROT=(SINA*(VELM(4)-VELM(1))+COSA*(VELM(2)-VELM(5)))/FL
DVR(1)=VELM(3)+ROT
DVR(2)=VELM(6)+ROT

```

C  
C  
C

BETA-D DAMPING

```

IF (DFAC.EQ.0.) GO TO 450
FAC=DFAC*(1.+PSH)
BMIUB=BMIUB+(EK11*DVR(1)+EK12*DVR(2))*FAC
BMJUB=BMJUB+(EK12*DVR(1)+EK22*DVR(2))*FAC
FOUB=EAL*DVAX*DFAC+FOUB

```

C  
C  
C

STRUCTURAL DAMPING LOAD

```

450 IF (DELTA.EQ.0.) GO TO 460
SDMI=DELTA*ABS(BMTOT(1))*SIGN(1.,DVR(1))
SDMJ=DELTA*ABS(BMTOT(2))*SIGN(1.,DVR(2))
SDFO=DELTA*ABS(FTOT(2)-FTCT(1))/2.*SIGN(1.,DVAX)

```

```

BMIUB=BMIUB-SDMI+SDACT(1)
BMJUB=BMJUB-SDMJ+SDACT(2)
FOJUB=FOUB-SDFO+SDACT(3)
SDACT(1)=SDMI
SDACT(2)=SDMJ
SDACT(3)=SDFO

```

```

C
C   SET UP UNBALANCED LOAD VECTOR
C

```

```

460 IF (KBAL+KBAL1.EQ.0) GO TO 470
    KBAL=1
    SFUB=(BMIUB+BMJUB)/FL
    DD(1)=-SFUB*SINA-FOUB*COSA
    DD(2)=SFUB*COSA-FOUB*SINA
    DD(3)=BMIUB
    DD(4)=-DD(1)
    DD(5)=-DD(2)
    DD(6)=BMJUB

```

```

C
C   EXTRACT ENVELOPES
C

```

```

470 DO 490 I=1,8
    S=BMTOT(I)
    IF (S.LF.SENP(I)) GO TO 480
    SENP(I)=S
    IENP(I)=TIME
480 IF (S.GE.SENN(I)) GO TO 490
    SENN(I)=S
    IENN(I)=TIME
490 CONTINUE

```

```

C
C   PRINT TIME HISTORY
C

```

```

IF (KPR.LT.0) GO TO 500
IF (KPR.EQ.0.OR.KOUTDT.EQ.0) GO TO 540
500 IF (IHED.NE.0) GO TO 520
    KKPR=IABS(KPR)
    PRINT 510, KKPR, TIME
510 FORMAT (///18H RESULTS FOR GROUP,I3,
1      38H, END MOMENT-BUCKLING ELEMENTS, TIME =,F8.3///5X,
2      5H ELEM,4X,4HNODF,2X,10HFLX. YIFLD,2X,7HBRNDING,7X,5HSHEAR,
3      7X,5HAXIAL,12X,23HELASTIC HINGE ROTATIONS,
4      15X,5HAXIAL/5X,
5      5H NO.,4X,4H NO.,3X,5H CODE,6X,7H MOMENT,7X,5H FORCE,
6      7X,5HFORCE,8X,7HCURRENT,4X,9HACC. POS.,3X,9HACC. NEG.,
7      3X,10HYIELD CODE,3X,9HEXTENSION/)
    IHED=1
520 PRINT 530, IMEM, (NOD(I), KODY(I), BMTOT(I), SFTOT(I), FTOT(I), PRTOT(I)
1, PRACP(I), PRACN(I), I=1,2), KODY1, VTOT
530 FORMAT (I9, I8, I7, 3X, 3F12.2, 3X, 3F12.5/
1      9X, I8, I7, 3X, 3F12.2, 3X, 3F12.5, I10, F11.5)

```

```

C
C   SET INDICATOR FOR STIFFNESS CHANGE
C

```

```

540 KST=0

```



```
IF (KODYX(1).NE.KODY(1).OR.KODYX(2).NE.KODY(2)) KST=1  
IF (KST.EQ.0.AND.KST1.NE.0) KST=1
```

C

C

```
JUPDATE INFORMATION IN CCMS
```

C

```
DO 550 J=32,88
```

```
550 COMS(J)=COM(J)
```

```
COMS(2)=COM(2)
```

```
DO 560 J=141,167
```

```
560 COMS(J)=COM(J)
```

C

```
RETURN
```

```
END
```

SUBROUTINE OUT10 (/COMS/,/NINFC/)

C

```

COMMON /INFEL/ IMEM,KST,LE(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,
1      EK22,EK12,FSH,EAL,EK11H,EK22H,          KODYX(2),
2      KODY(2),BMTCT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3      SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4      SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5      PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6      PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3PE,EAL1,EAL4,
7      EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8      IVRTX,IEICG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9      TVENN,PYP,PYN,PYNC,REST(30)

```

C

```

DIMENSION COM(1),COMS(1)
EQUIVALENCE (IMEM,COM(1))

```

C

C

ENVELOPE OUTPUT, END MOMENT-BUCKLING ELEMENTS

C

```

DO 10 J=1,NINFC
10 COM(J)=COMS(J)

```

C

```

IF (IMEM.EQ.1) PRINT 20
20 FORMAT (39H END MOMENT-BUCKLING ELEMENTS (TYPE 10)///
1      5H ELEM,3X,4HNODE,17X,7HBENDING,14X,5HSHEAR,14X,5HAXIAL,
2      13X,8HPL HINGE,7X,14H      MAX. AXIAL/
3      5H NO.,3X,4H NO.,17X,7H MOMENT,3X,4HTIME,7X,5HFORCE,3X,
4      4HTIME,7X,5HFORCE,3X,4HTIME,6X,8HROTATION,3X,4HTIME,
5      5X,9HEXTENSION,3X,4HTIME/)

```

C

```

PRINT 30, IMEM,NODI,(SENP(I),TENP(I),I=1,7,2),VFNp,TVENP,(SENN(I),
1      1IENN(I),I=1,7,2),VENN,TVENN,NODJ,(SENP(I),TENP(I),I=2,8,2),(SENN(I
2      2),TENN(I),I=2,8,2)
30 FORMAT (I4,I7,5X,8HPOSITIVE,3(F12.2,F7.3),2(F14.5,F7.3)/
1      16X,8HNEGATIVE,3(F12.2,F7.3),2(F14.5,F7.3)/
2      7X,I4,5X,8HPOSITIVE,3(F12.2,F7.3),F14.5,F7.3/
3      15X,8HNEGATIVE,3(F12.2,F7.3),F14.5,F7.3/)

```

C

```

RETURN
END

```

SUBROUTINE FST10A (/STIF/,/KOD/)

C

```

COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1      EK22,EK12,PSH,EAL,EK11H,EK22H,      KODYX(2),
2      KODY(2),BMTOT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3      SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4      SDACT(3),BNY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5      PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6      PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7      EAL6,EAL7,EAL8,POUT(51)

```

C

C

C

FORM AXIAL STIFFNESS

```

      KYY=KOD+1
      GO TO (10,20,30,40,50,60,70,80,90,100),KYY
10  STIF=EAL
      GO TO 110
20  STIF=EAL1
      GO TO 110
30  STIF=EAL
      GO TO 110
40  STIF=0.001*EAL
      GO TO 110
50  STIF=EAL4
      GO TO 110
60  STIF=EAL
      GO TO 110
70  STIF=EAL6
      GO TO 110
80  STIF=EAL7
      GO TO 110
90  STIF=EAL8
      GO TO 110
100 STIF=0.001*EAL
C
110 RETURN
      END

```

SJBROUTINE LAW10

C

```

COMMON /INFEL/ IMEM, KST, LM(6), KGEOM, FL, COSA, SINA, A(2,6), EK11,
1 EK22, EK12, PSH, EAL, EK11H, EK22H, KODYX(2),
2 KODY(2), BMTOT(2), SFTOT(2), FTOT(2), PRTOT(2), SENP(8),
3 SENN(8), TENP(8), TENN(8), PRACP(2), PRACN(2), BMEP(2),
4 SDACT(3), EMY(2,2), NODI, NODJ, KONTDT, PR12, PR21,
5 PMX(3,2,2), A1(4,2,2), A2(4,2,2),
6 PFAC, RATIO, DELTY, KODYX1, KODY1, XPP, X3RE, EAL1, EAL4,
7 EAL6, EAL7, EAL8, X5, Y5, P5, X6, X5D, Y5D, P5D, INDR2,
8 REST(44)

```

C

C

GENERATE P-DELTA HYSTERESIS CURVE

C

C

RESIDUAL ELONGATION, RE

C

```

SIEND=60./RATIO
RE=0.0175*(0.55*X3RE/SIEND+0.0002*X3RE**2)
RE=RE*FL/DELTY

```

C

```

XPP=XPP+RE
XPP1=1.+XPP
BETA=1./3.
DENOM=(XPP1-X3RE)+(1.+PFAC)/BETA
XNUMR=XPP-X3RE-PFAC
Y5=RATIO*XNUMR/DENOM
X5=XPP-Y5/BETA
SLOP4=(Y5+PFAC)/(X5-X3RE)
EAL4=EAL*SLOP4
SLOP8=(1.-Y5)/(XPP1-X5)
EAL8=EAL*SLOP8
X6=X5-Y5-PFAC
XPP=XPP-RE

```

C

```

RETURN
END

```

SUBROUTINE VRTX10 (/FACAC/,/DVAX/,/SEP/,/TIME/)

AXIAL STATE DETERMINATION IN TENSION AND VERTEX REGIONS

COMMON /INFEL/ INFEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,  
 1 EK22,EK12,FSH,EAL,EK11H,EK22H, KODYX(2),  
 2 KODY(2),BMTOT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),  
 3 SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),  
 4 SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,  
 5 PMX(3,2,2),A1(4,2,2),A2(4,2,2),  
 6 PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,  
 7 EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,  
 8 IVPTX,IELOG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,  
 9 TVENN,PYP,PYN,PYNC,REST(30)

INITIALIZE

FACAC=0.

10 FACTOR=1.-FACAC

KODYI=KODY1+1

IF (KODYI.EQ.10) KODYI=4

GO TO (20,30,40,50),KODYI

ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE

20 DSEP=EAL\*DVAX

IF (DSEP) 21,60,22

21 FAC=(PYN-SEP)/DSEP

IF (FAC.GE.FACTOR) GO TO 23

FACTOR=FAC

SEP=PYN

KODY1=1

SENP(5)=-PYN

SENN(6)=PYN

TENP(5)=TIME

TENN(6)=TIME

GO TO 60

22 FAC=(PYP-SEP)/DSEP

IF (FAC.GE.FACTOR) GO TO 23

FACTOR=FAC

SEP=PYP

KODY1=9

GO TO 60

23 SEP=SEP+FACTOR\*DSEP

GO TO 60

ON SLOPE 1, GET FACTOR FOR STATUS CHANGE

30 DS1=EAL1\*DVAX

IF (EAL1.NE.0) GO TO 34

KODY1=3

IVRTX=1

```
      GO TO 80
C
34 IF (DS1) 31,60,32
C
      BUCKLED BUT LOADING
C
31 KODY1=2
      GO TO 40
C
      BUCKLING AND CONTINUING
C
32 FAC=(PYN-C-SEP)/DS1
      IF (FAC.GE.FACTOR) GO TO 33
      FACTOR=FAC
      SEP=PYN-C
      KODY1=3
C
      UPDATE PLASTIC DEFORMATION
C
      VPACN=VPACN+FACTOR*DVAX
      GO TO 70
C
33 SEP=SEP+FACTOR*DS1
C
      UPDATE PLASTIC DEFORMATION
C
      VPACN=VPACN+FACTOR*DVAX
      GO TO 60
C
      ON SLOPE 2, GET FACTOR FOR STATUS CHANGE
C
40 DS2=EAL*DVAX
      IF (DS2) 41,60,42
C
      BUCKLING AND CONTINUING
C
41 KODY1=1
      GO TO 30
C
      BUCKLED BUT LOADING
C
42 FAC=(PYN-C-SEP)/DS2
      IF (FAC.GE.FACTOR) GO TO 43
      FACTOR=FAC
      SEP=PYN-C
      KODY1=3
      GO TO 70
C
43 SEP=SEP+FACTOR*DS2
      GO TO 60
C
      ON SLOPE 9, TENSION YIELDING
C
50 IF (DVAX.LT.0) GO TO 51
      DVP=FACTOR*DVAX
```

```
XPP=XPP+DVP/DELTY
VPACP=VPACP+DVP
GO TO 110
C
C   YIELDED BUT UNLOADING
C
51 KDDY1=0
   X3RE=XPP-PFAC
   GO TO 20
C
60 FACAC=FACAC+FACTOR
   IF (FACAC.LT.0.9999999) GO TO 10
   RETURN
C
70 FACAC=FACAC+FACTOR
   IVRTX=1
C
80 PHI=PYNC/PYN
C
C   MODIFY THE SLOPES OF 4 SEGMENTS ON M-P CURVE IN COMPRESSION
C
   DO 90 K=1,2
   DO 90 J=1,2
   DO 90 I=1,2
90 A1(I,J,K)=A1(I,J,K)/PHI
C
   DO 100 K=1,2
   DO 100 J=1,2
100 PMX(1,J,K)=PHI*PMX(1,J,K)
C
110 RETURN
    END
```

```

C      SUBROUTINE FST10B (/ST/,/KCD/)
C
C      FORM 2*2 FLEXURAL STIFFNESS
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,
1      EK22,EK12,PSH,EAL,EK11H,EK22H,REST(169)
DIMENSION ST(2,2),KOD(2)
C
      KYY=KOD(1)+2*KOD(2)+1
      GO TO (10,20,30,40),KYY
10  ST(1,1)=EK11
    ST(2,2)=EK22
    ST(1,2)=EK12
    GO TO 60
20  ST(1,1)=0.
    ST(2,2)=EK22H
    GO TO 50
30  ST(1,1)=EK11H
    ST(2,2)=0.
    GO TO 50
40  ST(1,1)=0.
    ST(2,2)=0.
50  ST(1,2)=0.
60  ST(2,1)=ST(1,2)
C
      RETURN
      END

```



```

SUBROUTINE BM10
C
C   END MOMENT INCREMENT
C
COMMON /INFEL/ IMEM, KST, LM(6), KGEOM, FL, COSA, SINA, A(2,5), EK11,
1      EK22, EK12, FSH, EAL, EK11H, EK22H,      KODYX(2),
2      KODY(2), PEST(165)
COMMON /WORK/ DVR(2), DPR(2), DBM(2), W(1994)
C
KYY=KODY(1)+2*KODY(2)+1
GO TO (10, 20, 30, 40), KYY
10 DBM(1) = EK11*DVR(1) + EK12*DVR(2)
   DBM(2) = EK12*DVR(1) + EK22*DVR(2)
   GO TO 50
20 DBM(1) = 0.
   DBM(2) = EK22H*DVR(2)
   GO TO 50
30 DBM(1) = EK11H*DVR(1)
   DBM(2) = 0.
   GO TO 50
40 DBM(1) = 0.
   DBM(2) = 0.
C
C
50 RETURN
END

```

SUBROUTINE YMOM10

C  
C  
C

YIELD MOMENTS FOR CURRENT AXIAL FORCE

COMMON /INFEL/ IMEM, KST, LM(6), KGEOM, FL, COSA, SINA, A(2,6), EK11,  
1 EK22, EK12, PSH, FAL, EK11H, EK22H, KODYX(2),  
2 KODY(2), BMTCT(2), SFTOT(2), FTOT(2), PRTOT(2), SENP(8),  
3 SENN(8), TENP(8), TENN(8), PRACP(2), PRACN(2), BMEP(2),  
4 SDACT(3), BMY(2,2), NODI, NODJ, KOUTDT, PR12, PR21,  
5 PMX(3,2,2), A1(4,2,2), A2(4,2,2), POUT(63)

C

FACC=-1.  
DO 30 IEND=1,2  
FFT=FTOT(IEND)\*FACC  
FACC=1.  
FAC=1.  
DO 30 J=1,2  
FAC=-FAC  
DO 10 I=1,3  
IF (FFT.LI.PMX(I,J,IEND)) GO TO 20  
10 CONTINUE  
I=4  
20 BMY=A2(I,J,IEND)-A1(I,J,IEND)\*FFT  
IF (FAC\*BMY.GT.0.) BMY=0.  
30 BMY(IEND,J)=BMY  
FAC=-BMY(1,1)  
BMY(1,1)=-BMY(1,2)  
BMY(1,2)=FAC

C

RETURN  
END



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