

ANN ARBOR

MODEL STUDY OF AN OFFSHORE DRILLING STRUCTURE

by '

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Technical Report No. 6 Lake Hydraulics Laboratory Department of Civil Engineering

Project 2092 Bethlehem Steel Company, Shipbuilding Division

Beaumont, Texas

ENGINEERING RESEARCH INSTITUTE UNIVERSITY OF MICHIGAN ANN ARBOR

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MODEL STUDY OF AN OFFSHORE DRILLING STRUCTURE

INTRODUCTION

These model tests were undertaken to determine the horizontal forces, piling stresses, and drilling platform displacements produced in a proposed offshore drilling structure by the action of waves. This information was needed to provide a basis for design and also to provide an indication of operating conditions. Measurements were made on a 1:50 scale model for various depths of water with waves of various periods and heights and with the barge in two different positions.

The study was made in accordance with a contract, dated September 30, 1952, between the University of Michigan, Ann Arbor, Michigan, and the Bethlehem Steel Company, Shipbuilding Division, Beaumont, Texas. The tests were conducted in the University of Michigan Lake Hydraulics Laboratory*.

The men most directly concerned with the model tests were Mr. E. C. Rechtin, General Manager, and Mr. J. E. Steele, Chief of Design, Bethlehem Steel Company, Shipbuilding Division, Beaumont, Texas. The model was designed on the basis of preliminary plans supplied by Mr. Steele. The test results were transmitted to Mr. Steele as they were obtained and the model operations and test sequences were arranged to meet the needs of the design group.

Consultations were also held with Mr. H. deLuce and Mr. D. F. MacNaught of the Central Technical Division, Bethlehem Steel Company, Quincy, Massachusetts. Their suggestions were very helpful in planning and conducting the tests.

* The Lake Hydraulics Laboratory is a facility of the Engineering Research Institute and the Department of Civil Engineering of the College of Engineering.

Mr. J. R. Aikins and Mr. H. J. EnDean of the Gulf Research and Development Company, Pittsburgh, Pennsylvania, made helpful suggestions regarding the tests and also supplied some of the recording instruments used during the tests. Other personnel of the Gulf Oil Company who visited the laboratory during the tests were Mr. D. W. Gower of the Construction Department, Mr. W. D. Stine and Mr. C. W. Laas of the Producing Department, Houston, Texas, and Mr. O. W. Crisman of the Producing Department, New Orleans, Lousiana.

Mr. L. D. Stair, Research Associate in the Engineering Research Institute, worked with the authors on all phases of the testing program and in the preparation of the report. Dr. T. A. Hunter, Assistant Professor of Engineering Mechanics, assisted with the instrumentation and other phases of the work.

THE PROTOTYPE

The proposed structure is being designed for use in the Gulf of Mexico at water depths up to 100 feet. It is expected that storm waves as high as 35 feet will be directed against the structure. The wave period selected for design purposes was 12 seconds; the corresponding wave length in deep water would be 736 feet.

The structure consists of a movable barge 200 feet long, 88 feet wide and, 17 feet high and a drilling platform 165 feet long and 88 feet wide. During transit, the drilling platform is supported 40 feet above the deck of the barge by sixteen 48-inch tubes which are part of the barge superstructure. During operation, the platform is to be supported on sixteen 36-inch piles driven through the 48-inch tubes; at the same time the barge is lowered to a position below the water surface, where it is held in place by clamps located near both ends of the 48-inch tubes.

THE MODEL

The model was constructed to a scale of 1:50 in accordance with plans supplied by the Bethlehem Steel Company. Drawing 1 shows the general arrangement and dimensions of the model. Photographs of the model during assembly are shown in Fig. 1, p. 4.



Drewing Number 1





Figure I. Model during Final Stages of Assembly.

Barge and Drilling Platform

In order to reproduce the stiffness of the prototype as accurately as possible, the internal bulkheads were designed and constructed to simulate those shown in detailed drawings supplied by the Bethlehem Steel Company. A drawing showing the internal construction of the model barge is given as Drawing 2, Appendix A, p. 23, and a photograph is shown in Fig. 2.

The weights of the barge and platform were reproduced to scale (1:50³) by adding weights where necessary, placing them so as to duplicate the locations of the centers of gravity.



Figure 2. Interior of Model Barge.

Piling

It was decided to use aluminum for the model piling rather than steel in order to magnify the strains induced by the model forces. The prototype piling consisted of 36-inch-outside-diameter pipe having a wall thickness of 3/4 inch. The corresponding model dimensions were 0.72-inch diameter and 0.015inch wall thickness. The 48-inch-outside-diameter tubes (Fig. 2) which will support the drilling platform while the structure is in transit and through which the piling will be driven when the structure is in place have the model dimensions of 0.96-inch diameter and 0.015-inch wall thickness. The construction of the model was delayed for some time until material having approximately these dimensions could be found. The problem was solved by reducing the wall thickness of standard aluminum tubing by a centerless grinding process*. The tubing used for the model piling had an outside diameter of 0.711 inch. The wall thickness of the tubes was measured and the deviations from 0.015 inch were found to be less than 0.001 inch for thirteen piles and just slightly greater than 0.001 inch for three piles. Figure 3a is a photograph showing the piling before assembly.

It was decided that the degree of fixity of the piling in the mud bottom might be simulated by restraining the model piling against rotation at a location corresponding to a point 40 feet below the mud line. This was accomplished by carefully driving the model piling into holes drilled in pieces of cypress 1-5/8 inches thick. The pieces of cypress were bolted to a 1/4-inch aluminum plate as shown in Figs. 1 and 3b. The fit between the aluminum tubes and the wood was as tight as could be obtained without injuring the piling. A similar tight fit was produced in the platform by drilling holes in hard wood 3 inches thick. Four strain gages were placed on each pile (Fig. 3a, p. 7) at a point 1.6 inches (80 inches in prototype) above the point where they emerged from their supports. When the piles were placed in the supporting plate the positions of the strain gages were oriented with the principal axes of the model.

The clamping arrangement for fastening the barge to the piling was simulated by placing four set screws in the corresponding locations at the top and bottom of the 48-inch supporting tubes. It was found that the eight set screws on each of the sixteen piles produced sufficient resistance to support the barge in any desired position without serious injury to the model piling.

* These tubes were prepared by the Central Specialty Division of the King Seely Corporation, Ypsilanti, Michigan, under the direction of Mr. Louis Chadwick.

beams placed at fastened to the Fig. 1. The bea base of the mode was simulated by abova in Fig. 5. without restrain veight of 115 po

Dynamome terre

Figure 3a. Piling before Assembly.



Figure 3 b. Base Plate and Aluminum Portion of Supporting Box.

Model Suspension

The model was suspended from four piano wires 11-1/2 inches long located as shown in Drawing 3. The suspending wires were supported by two small beams placed at the elevation of the mud line, the other ends of the wires being fastened to the base plate. One of the supporting beams is shown in place in Fig. 1. The beams were, in turn, supported by the walls of a box in which the base of the model was enclosed. The lower portion of this box was made of aluminum (Figs. 1 and 3a) and the upper portion was made of wood. The sea floor was simulated by nailing 1-inch boards to the top of the supporting box as shown in Fig. 5, p. 14. The sixteen piles passed through holes in these boards without restraint. In order to resist the overturning force of the waves, a weight of 115 pounds was placed on the base plate of the model.

Dynamometers

The horizontal displacement of the base plate of the model was restricted by three dynamometers located as shown in Drawing 3, p. 9. The dynamometers consisted of aluminum cantilevers having the dimensions shown in Drawing 4. They were fastened to the bottom of the rigid aluminum box shown in Fig. 3b. The upper ends of the dynamometers were connected to the base plate of the model by the aluminum bars shown in Drawing 4. Holes were drilled in the ends of these connecting bars to produce a tight fit on 1/8-inch-diameter pins, one of which was located on the top of the dynamometer and the others on the base plate of the model. The dynamometers were placed with their longer cross-sectional dimensions parallel to the direction of the wave forces in order to provide the maximum amount of stiffness.

Strain gages were placed on the sides of each dynamometer 1.72 inches from the top. Static calibrations were made with the model in place before and after each series of tests. This was done by applying known horizontal loads and comparing their magnitudes with the forces calculated from the strain gage measurements. Points of application of the loads were varied in both the horizontal and vertical directions in order to insure that the static tests covered the range of conditions produced by the waves. It was found that the calibration factor varied very little with the location of the point of application of the loads.

The calibration coefficient varied gradually from about 1.12 to 1.3 during the several months of testing. A calibration factor of 1.25 would indicate that the horizontal force computed from dynamometer readings was 80 per cent of the applied load.







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DYNAMOMETER





SIDE VIEW



CONNECTING BAR





END VIEW

UNIVERSITY OF MICHIGAN ENGINEERING RESEARCH INSTITUTE MODEL STUDY OF AN OFFSHORE DRILLING STRUCTURE

DYNAMOMETER DETAILS

Scole : 1:2

Drawing Number 4

INSTRUMENTATION

The strain gages used on the dynamometers were standard A-7 resistance gages and those used on the piling were standard A-12 gages. The gages were obtained from the Baldwin-Southwark Company. They were attached to the dynamometers and piling by means of regular Duco cement. Some difficulty was encountered in waterproofing the gages adequately; however, it was found from a series of preliminary tests that the following procedure gave excellent results. The gages and connections were given three coats of standard Glyptal paint, each coat being dried several hours under heat. This paint was then covered with Krylon acrylic plastic spray, which was in turn covered with standard Permatex No. 1.

The strain gages on the two principal dynamometers (R and T), which measured the resultant wave forces, were connected directly to separate Brush strain amplifiers and the results were recorded by means of Brush oscillographs. The two gages on opposite sides of a dynamometer were attached to opposite arms of the Wheatstone bridge, thus obtaining twice the sensitivity that would have been obtained from one gage. The gages on the third dynamometer (Y) and those on eight of the piling were connected to one strain amplifier and oscillograph by means of an Anderson balancing and switching unit. This arrangement operated in a satisfactory manner. It was found, however, that the stress in the remaining piling could not be measured even with the balancing unit because the balancing of the amplifier could not be maintained when these additional piling gages were attached. The general arrangement of the measuring equipment is shown in Fig. 4a.

The horizontal displacement of the platform was obtained by mounting three Federal dial gages in a wood frame as shown in Fig. 4b. The exact locations of these gages are shown in Drawing 3, p. 9.

TESTING FACILITIES

The wave tank is 90 feet long and 55 feet wide. For these tests, a temporary wall, a-b in Drawing 5, was constructed to form a channel 14 feet wide. The wave machine was located at one end of the channel and, the model near the other end as shown in Drawing 5. The aluminum box in which the model was mounted was set in a pit at such an elevation that the surfaces of the boards which simulated the sea floor were at the same elevation as the floor of the tank.



Figure 40 Recording Instruments and Amplifiers.



Figure 4b Displacement Gages



During deep-water tests the depth of water was uniform from the wave machine to the model. For tests at 62 feet of depth and less, gravel was placed in the tank and graded from zero thickness at the wave machine to a thickness simulating 40 feet at the model, and the walls of the model pit were built up to this new elevation as shown in Fig. 5.

The portion of the wave tank outside the 14-foot channel served as a wave-damping area. The wave energy was reflected by the curved wall c-d (Drawing 5) in such a manner as to prevent reflected waves from returning through the testing channel. The reflected waves were destroyed on beaches built in suitable locations. Some small reflected waves were present in the testing channel, but these were caused by reflections from the model itself.



Figure 5. Model in Position for Testing.

Generation of Waves

Waves were generated by the plunger-type wave machine, shown in Fig. 6a. The speed of the plunger could be controlled to produce any desired wave period. The wave height was varied by changing the amplitude of the plunger.

It was found that the original plunger was unable to produce waves of sufficient height at the large wave periods required for these tests. Therefore, a larger plunger was constructed of plywood. With the new plunger, waves up to 0.35 foot were generated at a wave period of 1.65 seconds and wave heights up to 0.54 foot at a wave period of 1.23 seconds.

Measurement of Wave Heights

A continuous record of wave heights was obtained at a point 6.67 feet from the model at the center of the testing channel by means of an electrically operated wave rod. Readings were also taken at a location 1.67 feet from the model by means of a point gage. These devices are shown in Fig. 6b. A wave rod was also located at one side of the model for the purpose of comparing the wave profile at the model with the variations in the horizontal force on the model.

The wave rod acted as one resistance in a Wheatstone bridge. Variations in the water depth produced variations in the resistance, which were recorded by means of an oscillograph. The gage was calibrated by lowering and raising the wave rod known amounts in still water and noting the corresponding oscillograph readings. Calibrations were made before and after each series of tests.

Measurement of Wave Lengths

Wave lengths were determined by noting the positions of two consecutive wave crests. One point of observation was near the wave rod (6.67 feet in front of the model) and the other, one wave length nearer the wave machine. Observations were made by two observers, one of whom varied his position until the locations of simultaneous crests were found.



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Figure 6b Wave Rod.

SCOPE OF THE TESTS

The variable factors in the tests were water depth, vertical positions of the barge and platform, wave period and wave height. During each series of tests the water depth and barge location were held constant. Tests were conducted with water depths of 100 feet, 62 feet, 40 feet, 20 feet, and 15 feet. With a water depth of 100 feet, tests were conducted with the bottom of the barge 41.5 feet above the mud line (center of barge at mid-depth) and with the bottom of the barge 3 feet above the mud line. For all other water depths the barge was placed only at the latter location. Figure 5 is a photograph showing the barge in this position.

During each series of tests, at least two wave periods were studied. In each case a wave period of approximately 1.65 seconds was used. This period corresponds approximately to a prototype period of 12 seconds. One or more shorter-period waves were also studied because it was possible to generate higher waves at the smaller periods.

For each wave period, waves of three or four different heights were projected against the model. The wave heights were varied from the maximum that could be generated to one having a prototype height of less than 10 feet.

TEST PROCEDURE

At the beginning of each series of tests the calibration of the dynamometers was checked as previously described (p. 8). The tank was then filled to the proper water-surface elevation. The zero readings of the displacement gages and the point gage were taken. The wave machine was set at the desired period and plunger amplitudes. The strain amplifiers were then balanced and the wave rod calibrated. After the instruments were set in operation, the wave machine was started and the test observations made.

During each test run, continuous records were obtained from the wave rod and from the two principal dynamometers (R and T, Fig. 3, p. 7). At the same time, another oscillograph channel, connected with a switch box, was used to obtain shorter-period records from dynamometer S, from eight piles (see Table I, p. 29), and from the auxiliary wave rod located at the side of the model. The piles for which stresses were measured during each test may be determined from

Table I, p. 29 and the locations of these piles may be determined from Drawing 2, p. 25. The time of beginning and ending of each of the above records was noted on the wave-height oscillograph chart. The three dial gages giving platform displacements were read and the value of wave height at the point gage was observed. Finally, the wave length was determined.

The wave machine was then shut down and the amplitude of the plunger changed to produce a wave of a different height. When all reflections had disappeared from the tank the procedure described above was repeated.

For each model setup the natural frequency of the model was determined with water in the tank. This was done by giving the model an initial impulse and recording the oscillations of the force in one of the dynamometers.

TEST RESULTS

The results of measurements made during the wave tests are given in Table 1, Appendix B, p. 27 to 31. The tabulated results are given in the order in which the tests were conducted. Values shown in Column 9 $[(R + T) \times \text{factor}]$ represent the total corrected horizontal force on the model in the direction of wave travel. Displacements of the platform are represented by the symbol, Δ , in columns 10-13. For water depths of 40 feet or greater, the values in Column 13, $(\Delta_1 + \Delta_2)/2$, are the total average amplitudes of the displacements in the direction parallel to the direction of wave travel. The displacements in one direction from the still-water position may be obtained by dividing tabulated values by 2. For water depths of 20 feet and 15 feet the breaking waves caused the displacements in the direction of wave travel to be much larger than those in the reverse direction. Therefore, the values given in Columns 10-13 for water depths of 20 and 15 feet are displacements in the direction of wave travel to be much larger than those the neutral position.

The locations of the piles referred to in Columns 14-22 may be found from Drawing 2, p. 25. It should be noted that these stresses were observed at strain gages placed 1.6 inches above the top of the base support.

The test results have also been presented in graphical form in Appendix B, Figs. 7-30, pages 33-56 Figures 7-12 are summaries of some of the test results. Complete sets of data for all tests are given in Figs. 13-30. The general arrangement used in presenting the graphs was to place those giving the total horizontal forces, the piling stresses, and the platform displacements in that order. Thus, in the summaries, Figs. 7 and 8 give forces, Figs. 9 and 10 deal with piling stresses, and Figs. 11 and 12 show displacements. In a

similar manner the more detailed results for forces are given in Figs. 13-18, for stresses in Figs. 19-24 and for displacements in Figs. 25-30.

It should be noted that the summary graphs do not show the actual test results, but only the lines drawn through the resulting points. The measured values are all plotted in the more detailed graphs (Figs. 13-30). The reader must refer to these latter figures in order to determine the nature and extent of the extrapolation used to apply the results to a prototype wave height of 35 feet.

The straight lines drawn through the sets of experimental points give dependable comparative values of the test results and may be used to indicate the effect of the variables. However, actual numerical values read from these lines outside the range of the test values are subject to the uncertainty resulting from the straight-line extrapolation.

The summary curves indicate that the forces, piling stresses, and displacements increased as the water depth was decreased for the case where the bottom of the barge was located 3 feet above the mud line. A comparison of Figs. 13 and 14 indicates that moving the barge from mid-depth to a point near the bottom had little or no effect on the magnitude of the horizontal forces. The effect of wave period on the total horizontal forces is well illustrated by the results shown on Fig. 13, p. 39. It should be noted that the displacements and corresponding forces measured during static calibration tests were in very close agreement with the displacements and corresponding forces measured during the wave tests, thus indicating that consistent and dependable results were being obtained.

The results of the determinations of the natural frequencies of the model both in and out of water are presented in Table II, p. 32.

The results of the static tests, the calibration data for the wave rod, the oscillograph charts for all the tests, and many other data have not been included in this report but are on file in the University of Michigan Lake Hydraulics Laboratory.

CONVERSION OF RESULTS FROM MODEL TO PROTOTYPE

Horizontal Forces

Based on the Froude law, the conversion factors for force is the product of the cube of the length ratio and the ratio of the specific weights of the fluid. For these studies, this value would be

$F_r = 50^3 \times 64/62.4 = 125,000 \times 1.025 = 128,000.$

However, this conversion factor should be modified by the ratio of the model and prototype drag coefficients and the ratio of the model and prototype virtualmass coefficients in proportion to the relative values of the drag forces and acceleration forces. It could be expected that the drag coefficient at the higher Reynolds numbers found in the prototype would be smaller than that of the model. Little is known about the variations in the virtual-mass coefficient. However, because the indications were that the drag force was less important than the acceleration forces, it was believed that for the present it would be necessary, for safety, to convert on the basis of the Froude law. The test results indicated the predominant force was that on the barge. Consequently, the use of larger piles than the 36-inch ones simulated in the model would produce only a small increase in forces over those measured in the model.

Piling Streeses

The conversion factor to be applied to model results to obtain prototype values is 50 if $36 \times 3/4$ -inch piles are used and 21.2 if 48×1 -inch piles are used. The development of these conversion factors is shown in Appendix C, p. 59. A procedure for comparing the stresses at the base of the piles with experimental values (1.6 inches above the base) is also given in Appendix C.

Platform Displacements

Model values of platform displacements may be converted to prototype values by multiplying by 833 if $36 \ge 3/4$ -inch piles are used and by 266 if $48 \ge 1$ -inch piles are used. Computations for determining these conversion factors are given in Appendix C, p. 59.

Natural Period

The conversion of national frequencies from model to prototype is discussed in Appendix C, p. 60

Magnification Factor

No correction of the experimental results has been made for the dynamic magnification effect. The magnification factor depends on the natural period of the structure, the wave period, and the amount of damping. The amount of

damping in both model and prototype is uncertain. Therefore, there is no apparent advantage in attempting to correct for this effect. It is possible however, that the magnification factor may be higher in the prototype than in the model, thus resulting in prototype displacements and stresses which would be larger than those computed by means of the conversion factors shown above.

APPENDIX A

DRAWING 2

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TEST RESULTS

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APPENDIX B

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2 9	1.63	17.5	3.00	7.10		10.10	1.1	01.11															
2 3 0	1.63	13.0	2.50	5.67	1.18	8.17	1.1	9.00	811.	.083	.018	960.	700	700		88	630	710	Ş	2- Off	94	675	
В.	1.63	12.3	1.93	4°87		6.80	1.1	7.48															
30 20	1.63	9.8	1.89	4.10	6.	5.99	1.1	6.60	.079	• 055	.013	.067	06†	ħ90		0 <i>L</i> †	420	۰ <u>۲</u>	° ₽	140 24	8	99 1	
Jla	1.63	6.8	1.29	2.92		4.21	1.1	4.63											,				
31 b	1.63	5.5	1.09	2.12	04.	3.21	1.1	3.54	•074	150.	100.	.0 3 7	300	300		290	270	300	230	\$00 3	60	264	

SUMMART OF TABLE I.

2.0

54		Remarks				Depth of water	5 feet above	mud line.						Denth of water	62 feet. Barge 3 feet above	mud line.						Depth of water 40 feet. Baroe	3 feet above			
23	Avg.	Pile Stress		100		162		234	295	189	154	242	387	655	4 <u>7</u> 4	264	540		Ŕ		525		356	341	315	478
52		ф,		125		200		250	310	200	190	30	9 81	750	575	350	400		20		550		350	200	90 1	ŝ
ស		м		100		125		200	270	190	150	550	350	680	<u>8</u>	82	370		450		450		350	1 25	8	1 52
20	в. ²	Ъ		8		150		250	õ	200	150	200	350	620	4 50	230	8		550		20		<u>Š</u>	125	275	ş
19	Lbs/I	ტ		100		200		80	350	ß	200	230	6	650	h50	250	310		0 9		550		8	1 27	800	450
18	ess in	₿4		100		סדו		2 00	2 80	175	130	80 80	350	550	90 4	250	õ		550		20		Ř	125	800	84
17	ng Str	E		100		200		52	350	175	130	230	8	200	20	230	350							·		
1 6	P111	A																	700		80		0 04	150	350	575
15		υ		100		160		200	250	175	130	260	350	650	420	2 30	£		650		<u>8</u>		350	150	800	450
14		A		8		150		250	250	175	150	580	h20	200	200	590	350		650		550		90 1	175	Ř	550
13	A1 + A2	2		•U14		8		120.	.0 3 3	.025	.020	.023	₩£0.	.053	6 £0•	.023	030.	.036	1 70.	.052	- O42		920.	•07 4	.025	140.
12	bes	۵3		. 03		1 00.		.006	800.	<u>.</u> 00	.005	80.	.03	8	.003	10.	8									
ц	in Incl	ଷ		10.		.018		.œ7	0£0.	80.	910.	210.	.027	. of 5	.0 3 3	.016	8	620.	.038	043	¥0.		.026	8.	.018	а <u>к</u> о-
10	۲ ۲	4		.017		.025		<u>رد</u> ه.	-035	030.	.025	.028	Ę.	88.	• of 2	88.	۰0 ،	. OH3	.050	<u>8</u>	640.		760.	610.	3 £0•	<u>8</u>
6	R + T	x Factor	84.4	4.24	7.82	6.83	10.80	9.30	12.19	7.10	5.17	7.95	00.SI	19.45	14.00	7.85	10.25	12.30	19.4	21.1	16.3	7.8	4°6	4.9	10.0	13.7
8	Cal.	Factor	1.23	1.23	1.23	1.23	1.23	1.23	1.23	1.23	1.23	1.30	1.30	1,30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30
7	Lbs	1- + 2-	3.65	3.44	6.36	5.56	8.77	7.57	9.9	5.78	4.21	6.11	9.23	14.98	10.75	6.03	7.89	9.45	9.41	16.2	12.5	6.0	7.2	3.8	7.7	10.5
9	nt gu	т		66.		66.		64.	1.18	.67	74.	62.	.39	.47	66.	66.	62.		8.		.79		74.	66.	۲¥.	ŝ
ŝ	. Readi	E	2.35	2.23	4.05	3.65	5.75	4.95	6.20	3.77	2.64	4.42	6.36	10.55	1.7	4.42	5.68	6.76	10.1	† .1	8.7	4.6	5.7	2.8	5.7	7.3
4	Dyn	24	1.30	1.21	2. 31	1.91	3.02	2.62	3.70	2.01	1.57	1.69	2.86	4.43	3.04	1.61	2.21	2.69	8.4	8° †	3.8	4.1	1.5	1.0	2.0	3.2
r	Wave	leight	7.6	6.5	12.3	10.5	15.0	13.6	23.0	13.5	8.7	8.0	12.5	21.8	24.5	8.4	10.3	12.8	17.8	8.5	16.0	7.5	9.8	3.5	8.8	22.3
2	ave.	eriod i	1.63	1.63	1.60	1.60	1.60	1.60	1.20	1.21	1.2	1.24	1.24	1.24	1.65	1.65	1.65	1.65	1.20	1.20	1.20	1.20	1.20	1.60	1.60	1.60
г	Test	Ио.	R	\$	55	3 7.	ħ	a fr	R	*	37	\$	6 2	0 1	14	¥	4 3	4	ŝ.	94	74	84	40	64	8	2

TABLE I. SUMMARY OF TEST RESULTS (continued)

24	•	Remerks						20 feet. Barge	> ree t move > mud line.	in downstream							Depth of water 15 feet. Rarow	5 feet above	*Displacement in downstream	direction only.	
23	Avg.	Pile Stress		627			<u>i - </u>		<i>9</i> 75				653		00 9				814		610
ଷ		е,		750	•				1125				%	į	8				B 25		80
ដ		м		800					8000				%		525				675		525
20	ی 1	5		450					88				800		450				750		525
19	Lbs/I1	t		%					1050				80		ŝ				8		80
18	ess in	₿ 4 j		<u>8</u>					1050				675		600				750		%
17	ng Str	ы																			
16	Pili	A		675					1200		•		600		750				800		750
15		υ		0 09																	
41		۷		750			_		1125				000		675				8		675
13	41 + 42	5		***20.					*O†2*				•035*		•030				• 0+5*		•026*
ମ	Je B	۵3																			
я	In Incl	ଷ		.025					010.				.032		.025				.038		ħЗО.
ទ	Ā	ង		78.					.050				.038		.035				.053		-œ.
6	E1 + : 64	x Factor	1.71	25.2	28.3	п.5	15.2	35.2	38.9	8.3	13.6	35.8	29.4	8.5	19.8	1.11	15.2	27.8	35.0	۲.4	19.8
ø	Cal.	Factor	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30	1.30
2	cbs	R + T	13.3	19.4	21.8	8.9	7.11	27.1	29.9	6.4	10.5	27.4	22.6	6.5	15.3	8.5	7.11	21.4	27.0	5.7	15.3
ور	ng in l	Y	•	1, 38					1.96				1.96		.78				1.58		1.18
5	Read1	E+	9.7	13.8	16.6	6.5	8.9	21.1	24.3	4.8	7.3	20.2	15.4	4.5	10.9	6.1	8.5	15.0	19 . 4	4.1	ц.3
4	Dyn.	ezi	3.6	5.6	5.2	2.4	2.8	6.0	5.6	1.6	3.2	7.2	7.2	2.0	† °†	2.4	3.2	6.4	7.6	1.6	h. 0
۴	Wave	Height	8.0	13.0	14.0	5.0	8.0	16.0	0.71	3.0	6.3	15.0	12.0	4.5	9.5	4. 8	8.0	13.3	14.5	0.4	8.5
5	Wave	eriod	1.66	1.66	1.66	1.25	1.25	1.25	1.25	1.25	1.25	1.25	1.25	1.25	1.25	1.45	1.45	1.45	1.45	1.45	1.45
ч	Test	No. F	53a	530	53e	af.	aff.	546	PH2	55a	<u>5</u> 5b	550	554	56 a	56b	57a	57b	57 c	57a	8 2	æ

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TABLE I. SUMMARY OF TEST RESULTS (concluded)

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TABLE II

SUMMARY OF THE NATURAL FREQUENCIES OF THE MODEL (Primary Mode)

Depth of Water (Prototype), Feet	Distance of Bottom of Barge above Mud Line (Prototype), Feet	Frequency of Model in Water, cycles/sec
100	41.5	3
100	3.0	4-1/2
62	3.0	4-3/4
40	3.0	5-1/4



























APPENDIX C

CONVERSION FACTORS

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CONVERSION FACTORS

In this appendix, the subscript p denotes prototype quality and the subscript m denotes model quality.

Displacements

For $36 \times 3/4$ -inch piles

 $\Delta = c \quad \frac{FL^3}{EI}$ $\Delta \propto c \quad \frac{F}{EL} \quad .$

or

 $\frac{\Delta_{\mathbf{p}}}{\Delta_{\mathbf{m}}} = \frac{\mathbf{F}_{\mathbf{p}}}{\mathbf{E}_{\mathbf{p}}\mathbf{L}_{\mathbf{p}}} \cdot \frac{\mathbf{E}_{\mathbf{m}}\mathbf{L}_{\mathbf{m}}}{\mathbf{F}_{\mathbf{m}}}$

If

then

Then

$$\frac{L_p}{L_m}$$
 = 50, $\frac{F_p}{F_m}$ = 50³, and $\frac{E_m}{E_p}$ = $\frac{1}{3}$

$$\Delta_{\rm p} = \frac{50^{\circ}}{50 \times 3} \quad \Delta_{\rm m} = 833 \quad \Delta_{\rm m}$$

_ . 7

For 48 x 1-inch piles

$$\Delta_{\rm p} = 833 \text{ x} \frac{13,000}{40,800} \Delta_{\rm m} = 266 \Delta_{\rm m}$$

Piling Stresses

For 36 x 3/4-inch piles

$$f = c \frac{FLd/2}{T}$$

or

then

$$\mathbf{f}_{\mathbf{p}} = \left(\frac{\mathbf{F}_{\mathbf{p}}}{\mathbf{F}_{\mathbf{m}}}\right) \left(\frac{\mathbf{L}_{\mathbf{m}}}{\mathbf{L}_{\mathbf{p}}}\right)^2 \mathbf{f}_{\mathbf{m}}$$

If

or

$$\frac{L_{p}}{L_{m}} = 50 \text{ and } \frac{F_{p}}{F_{m}} = 50^{3},$$

$$f_p = 50 f_m$$
.

For 48 x 1-inch piles

$$f_p = 50 \times \frac{13,000}{40,800} \times \frac{23.5}{17.63} f_m = 21.2 f_m$$
.

Maximum Stress in Piles

The point of measurement of the stress in the model piling was 1.6 inches above the bottom. If the piles are assumed to have equal rigidity at the base and at the lower clamp of the barge, and the distance between these points is d, then there is a point of contra-flexure midway between the base and the clamp, or d/2 from the base.

Therefore $\frac{S_{max}}{S_{meas}} = \frac{d/2}{d/2-1.6}$

$$S_{max} = \frac{d}{d-3.2} S_{meas}$$

Example:

For 100 feet of water, d in model is 12.3 inches.

Therefore $S_{max} = \frac{12.3}{9.1}$ $S_{meas} = 1.36$ S_{meas} .

Thus the maximum stress in the model piling is 1.36 times the measured stress.

Natural Period

For $36 \times 3/4$ -inch piles

$$P_{p} = \frac{L_{p}}{L_{m}} \sqrt{\frac{E_{m}}{E_{p}}} P_{m}$$
$$P_{p} = 28.8 P_{m}.$$

or

Natural frequency
$$T_p = \frac{1}{28.8} T_m$$

For 48 x 1-inch piles

$$P_{p} = 28.8 \times \sqrt{\frac{13,000}{40,800}} P_{m} = 16.2 P_{m}.$$

(Natural frequency $T_p = \frac{1}{16.2} T_m$
