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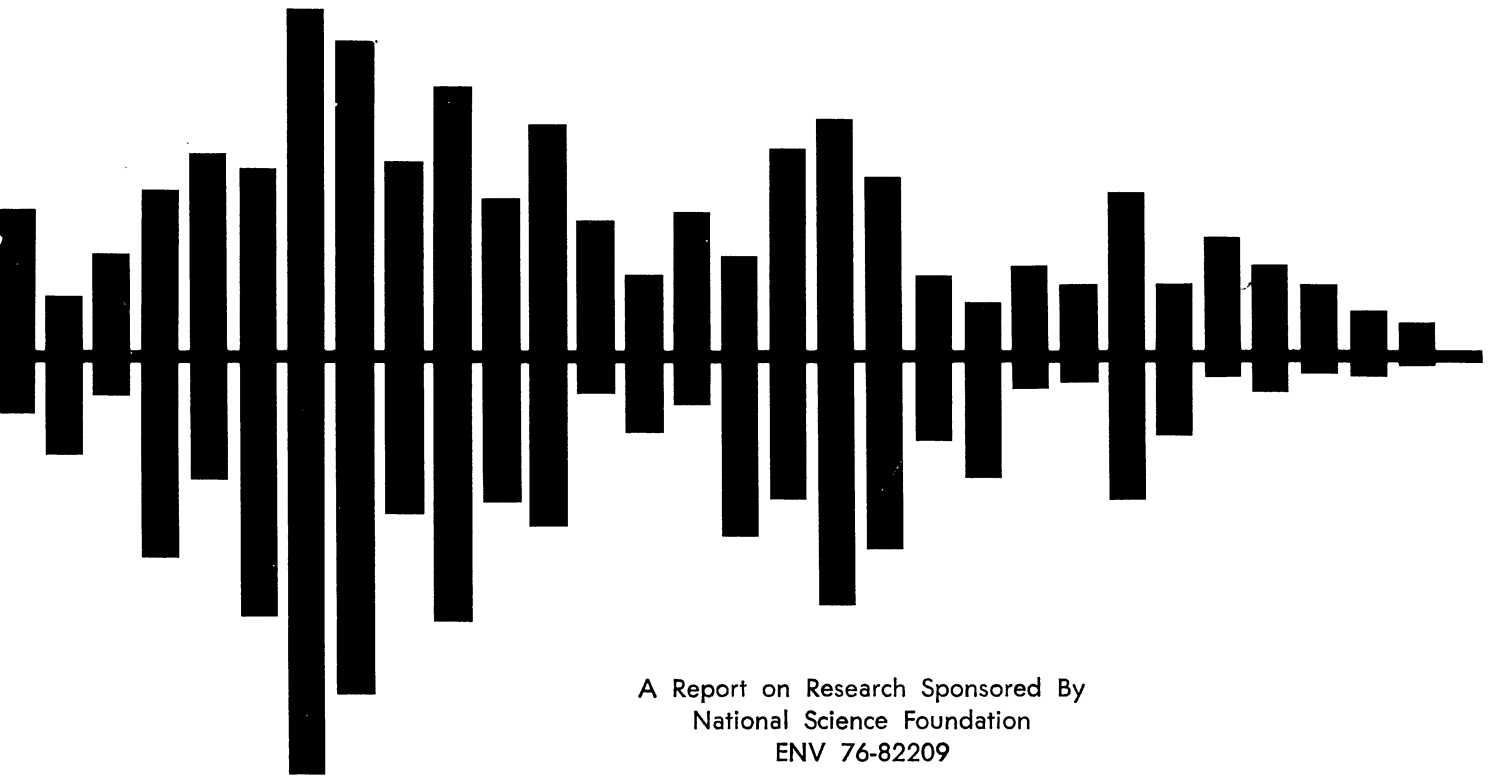
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HYSTERESIS MODELS FOR STEEL MEMBERS SUBJECTED TO CYCLIC BUCKLING OR CYCLIC END MOMENTS AND BUCKLING (USER'S GUIDE FOR DRAIN-2D:EL9 AND EL10)

**Ashok K. Jain
Subhash C. Goel**

The University of Michigan Department of Civil Engineering



A Report on Research Sponsored By
National Science Foundation
ENV 76-82209

American Iron and Steel Institute
Project 301

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(USER'S GUIDE FOR DRAIN-2D:EL9 AND EL10)

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ABSTRACT

Elements EL9 and EL10 are general purpose programs for steel members subjected to cyclic buckling, or cyclic end moment-buckling, respectively. These elements are developed for use with DRAIN-2D computer program. This manual describes the essential features of these two new elements along with their FORTRAN listing. The development of axial load-axial displacement hysteresis model as used for these elements has been described in a previous report.

ACKNOWLEDGEMENTS

The computer programs for the Buckling Element (EL9) and End Moment-Buckling Element (EL10) have been written on the basis of analytical and experimental research conducted at the University of Michigan over the period of past ten years. The financial assistance for the current effort was provided by the National Science Foundation, Grant ENV 76-82209 and the American Iron and Steel Institute, Project 301. Constructive suggestions from American Iron and Steel Institute Task Force on Project 301, Earthquake Resistant Design of Braced Steel Frame Structures, consisting of A. L. Collin, L. W. Lu, M. H. Mark, W. A. Milek, Jr., D. R. Sherman, E. P. Popov and L. A. Wyllie, Jr., were very helpful. Special thanks are due to Professor R. D. Hanson for introducing the idea of writing subroutine VRTX9. The authors are also thankful to Dr. J. B. Scalzi of NSF and Mr. A. C. Kuentz of AISI for the encouragement they provided in carrying out this research.

For compatibility and consistency, the format of presentation of this manual has been styled after the other DRAIN-2D elements developed by A. E. Kanaan, G. H. Powell and Pritam Singh.

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CHAPTER 1

INTRODUCTION

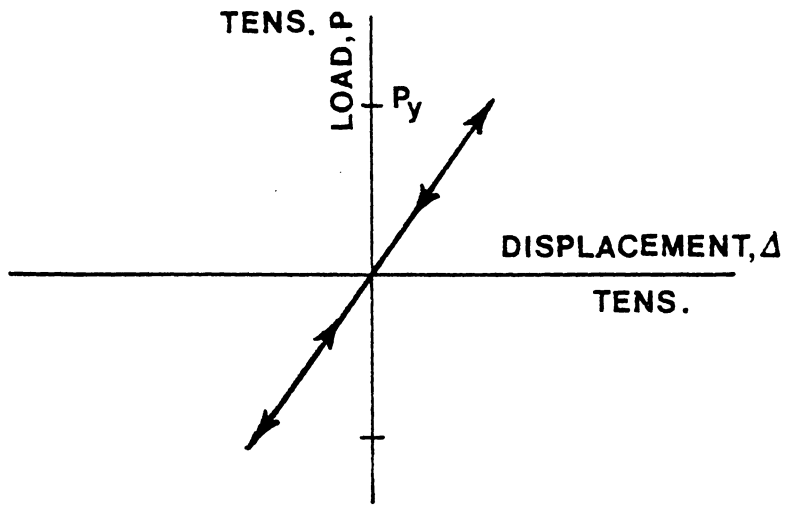
Tall braced frame structures are constructed in seismically active regions throughout the world. Such frames are generally more efficient in terms of lateral stiffness per unit volume of material than open moment resisting steel frames. American Petroleum Institute Code (API-RP2A, Ref. 1) now contains strength and ductility requirements for offshore braced steel platforms. The strength requirements insure that the structure is adequately sized for strength and stiffness to maintain all nominal stresses within yield or buckling for the level of earthquake activity which is normally expected during the life of the structure. The ductility requirements are intended to insure that the structure has sufficient energy absorption capacity to prevent its collapse during rare intense earthquake motions. Bracing members are considered effective earthquake resistant elements as they help satisfy the above two requirements when used in a frame.

Different member arrangements and proportions are used in braced frames (3,4,5). Bracing arrangements may be either concentric or eccentric type. The connections of bracing members may be designed as simple or moment resistant. The members in the former situation are generally treated as primarily axially loaded, whereas, the

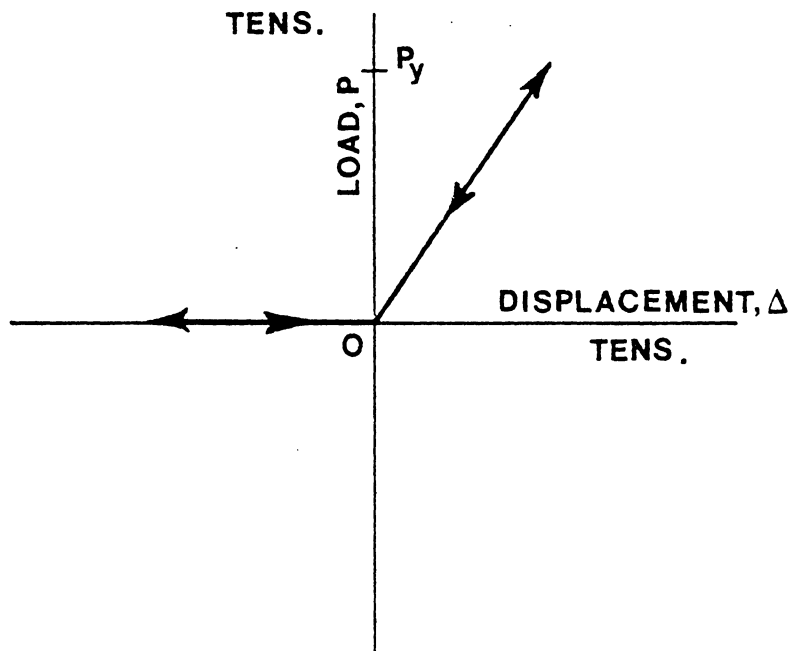
latter type may develop significant end moments.

In past studies of braced frames, the hysteresis behavior of primarily axially loaded bracing members has been modeled in one of several ways, such as: elastic in tension and compression (Figure 1a), tension-only elastic model (Figure 1b), tension-yield and compression-yield (Figure 1c), or tension-yield and compression-buckling (Figure 1d). These models neglected the energy dissipation characteristics of bracing members in the post-buckling range. Later, Higginbotham and Hanson (Ref. 2, Figure 1e), Nilforoushan (Ref. 12, Figure 1f), Prathuangsit (Ref. 14), Singh (Ref. 15, Figure 1g), Wakabayashi (Ref. 17), and Marshall (Ref. 10, Figure 1h) presented hysteresis models which represented the post-buckling behavior of bracing members in a more realistic manner. Experimental studies (3,8) on small specimens have pointed out two significant characteristics in the hysteresis behavior which were not included in these analytical models. These characteristics are: increase in member length and reduction in compressive strength with number of cycles. Jain (Ref. 3, Figure 2) presented a hysteresis model which accounts for these two parameters. Minor changes have been made in this model and the latest version is described in Chapter 2 of this manual. This model is called as Buckling Element (EL9) for use in DRAIN-2D Computer program (9).

There is one model available for primarily end

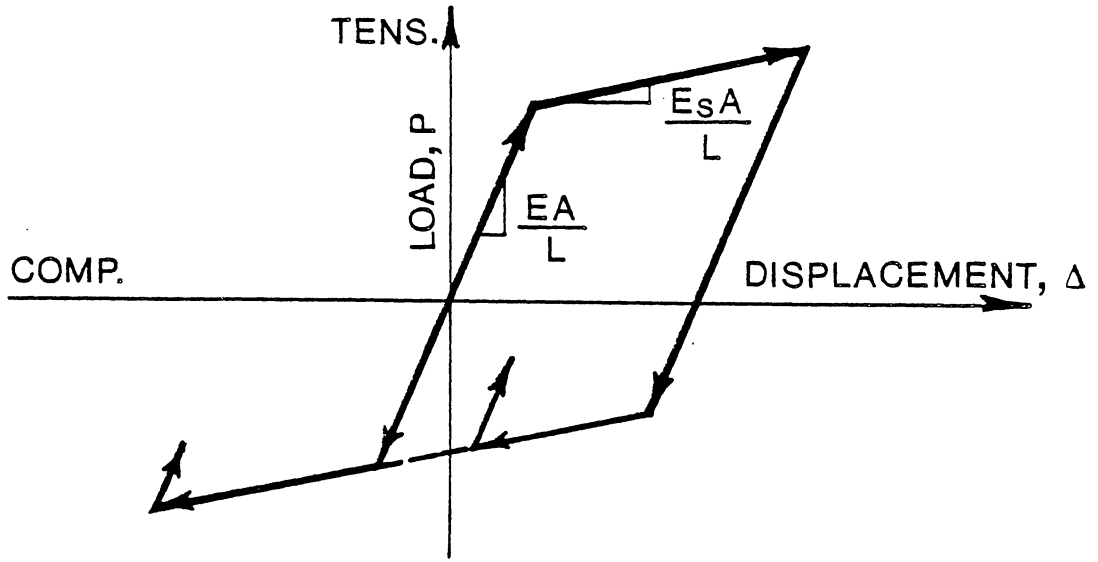


(a) Elastic Tension — Compression Model

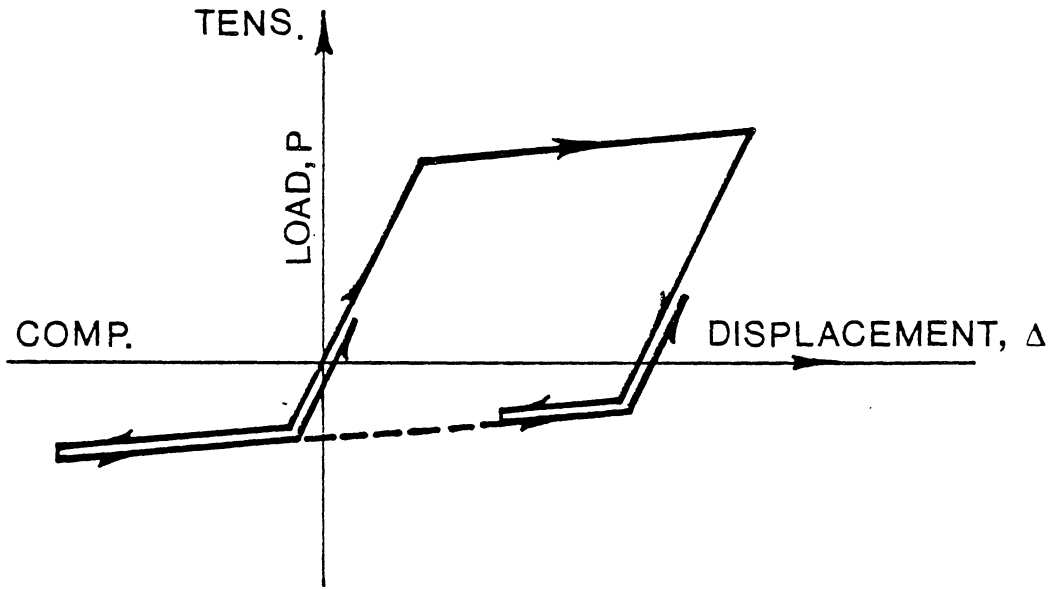


(b) Elastic Tension — Only Model

Figure 1 - Axial Load - Displacement Behavior

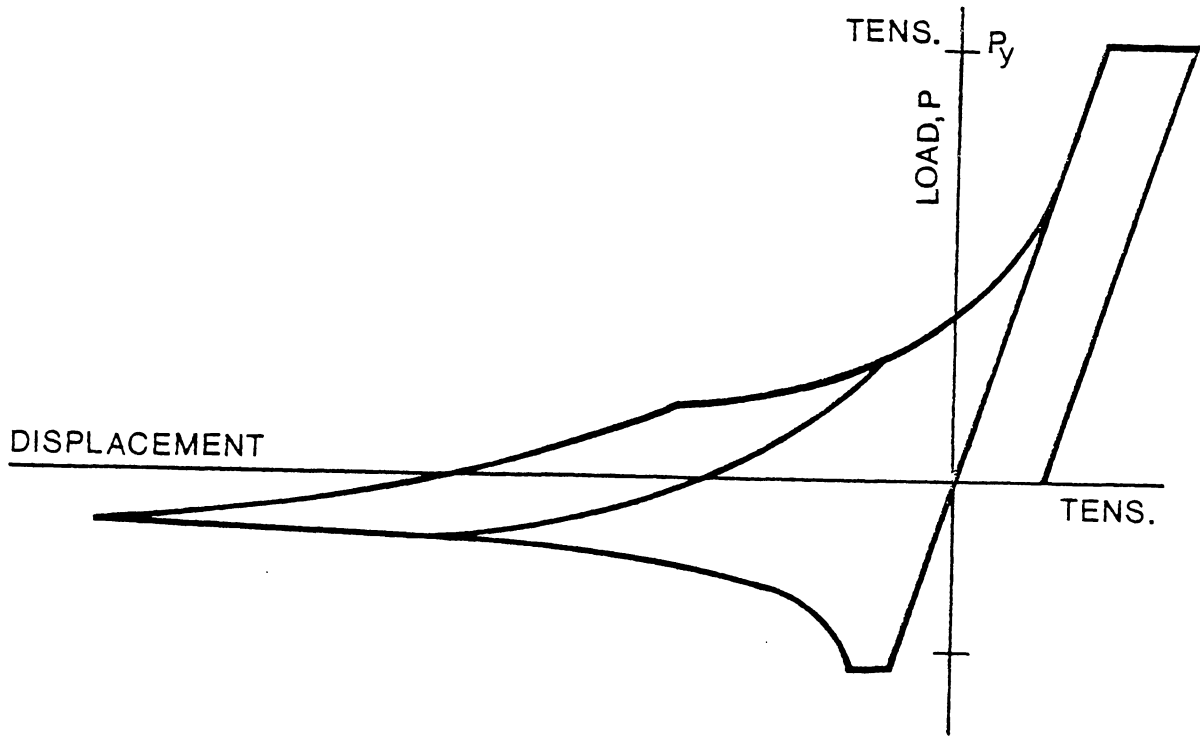


(c) Yield in Tension and Compression

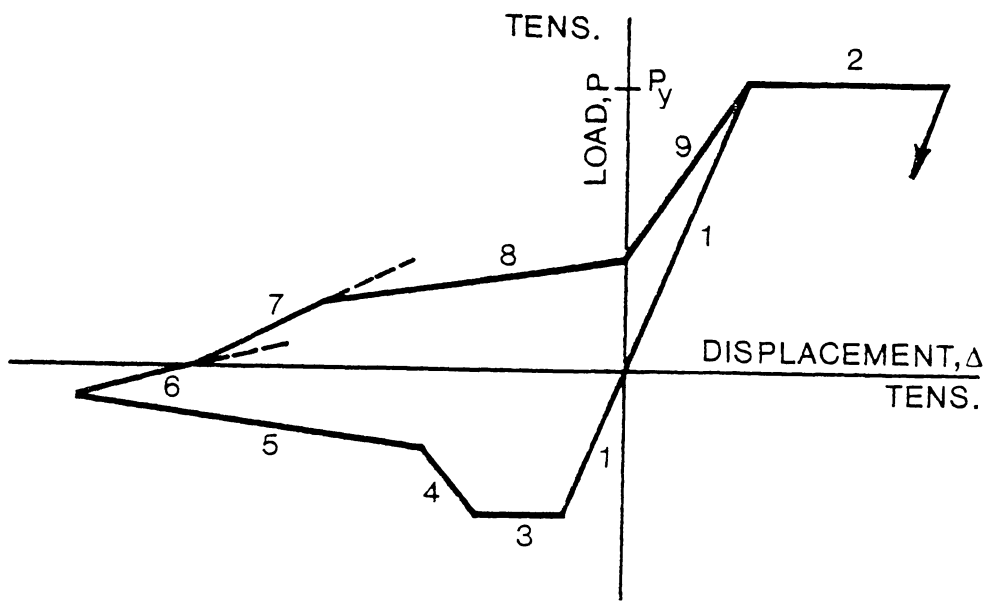


(d) Yield in Tension, Buckling in Compression

Figure 1 - Axial Load - Displacement Behavior (Cont.)

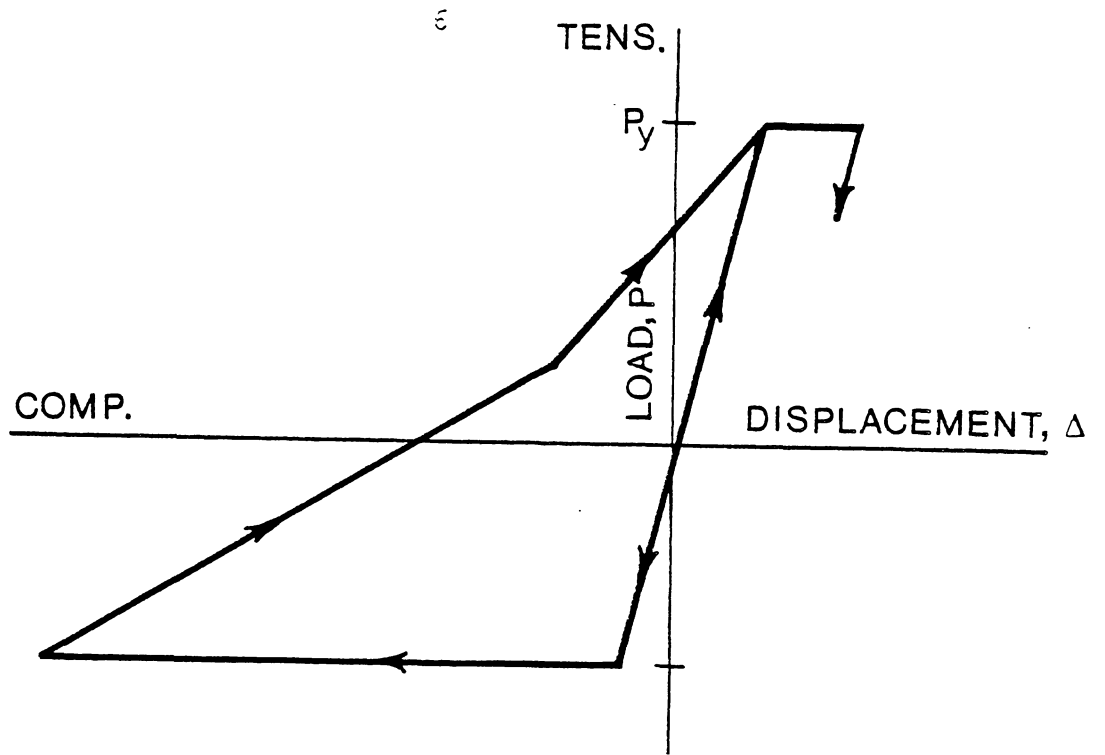


(e) HIGGINBOTHAM AND HANSON, Ref. 2

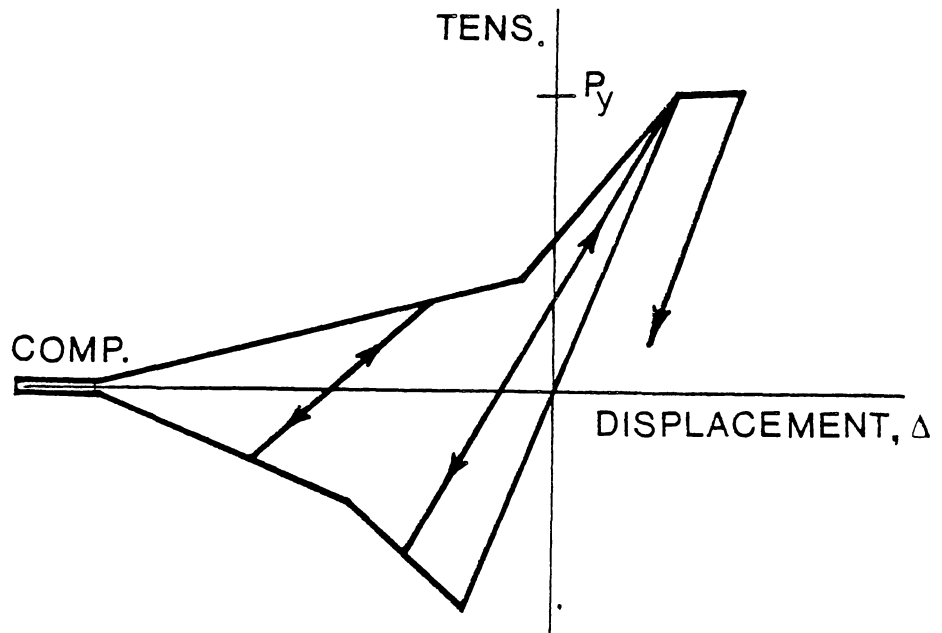


(f) NILFOROUSHAN, Ref. 12

Figure 1 - Axial Load-Displacement Behavior (cont.)



(g) Singh, Ref. 15



(h) Marshall, Ref. 10

Figure 1 - Axial Load-Displacement Behavior (cont.)

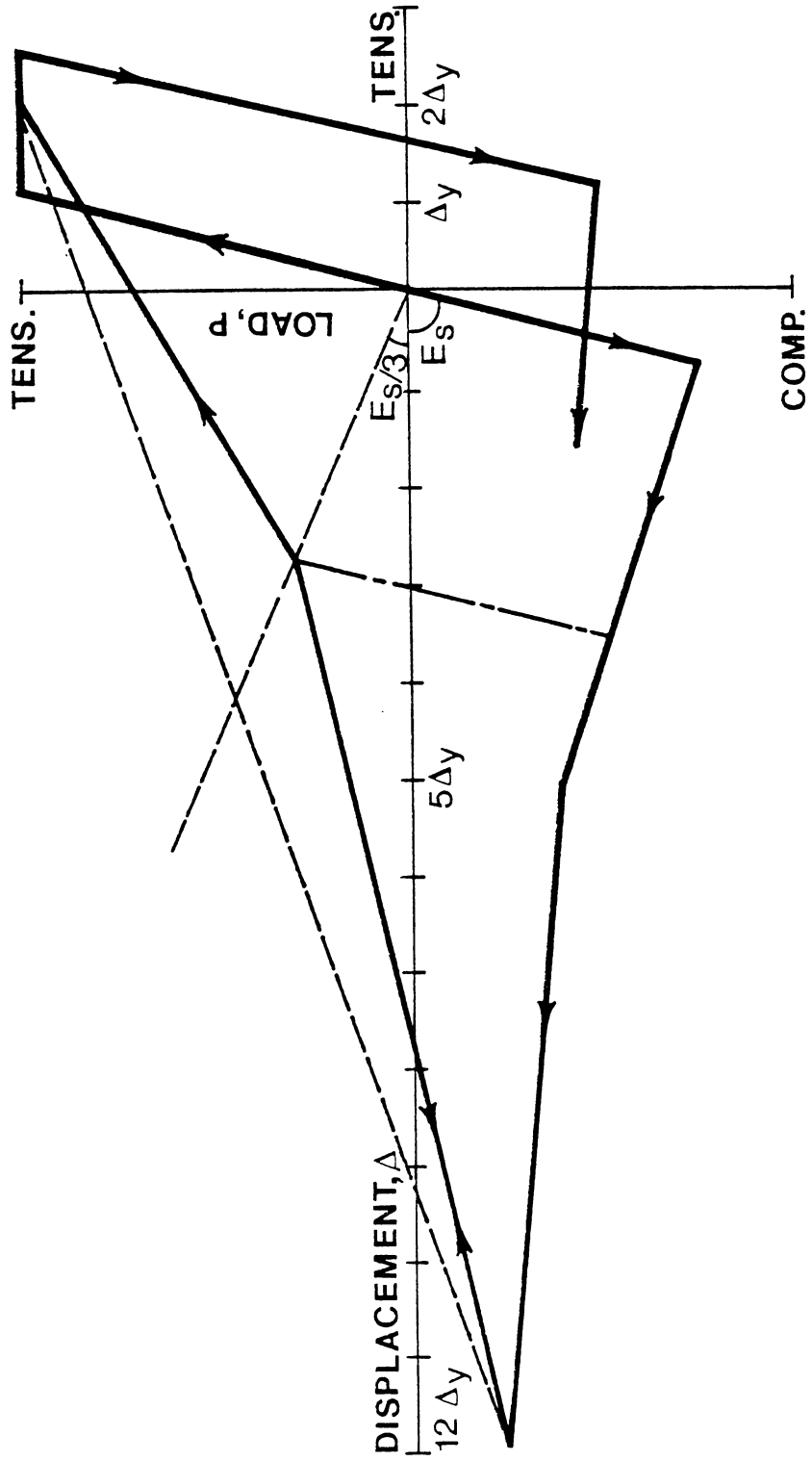


Figure 2 - Axial Load-Displacement Behavior, Ref. 3

moment resisting members and is known as beam-column element (Ref. 9, Figure 3). This model does not consider buckling and, therefore, should be used for full moment connected (or rigid-connected) non-buckling type bracing or column members.

Jain (3) analyzed 18 concentrically braced (X and K) and eccentrically braced (open or split K) frames under monotonic elastic, monotonic inelastic and dynamic loading conditions. The purpose of this analysis was (i) to determine the situations in which end moments dominate over axial forces in bracing members and vice versa, and (ii) to develop an understanding of the inelastic dynamic response of these frames with different member proportions. It was concluded that there is a need to develop a hysteresis model for rigid-connected buckling type steel members. Such a model has been developed by combining buckling element (EL9) and beam-column element (EL2) and is described in Chapter 3 of this manual. This model is called as End Moment-Buckling Element (EL10) for use in DRAIN-2D computer program (9).

DRAIN-2D COMPUTER PROGRAM

DRAIN-2D is a general purpose computer program for the inelastic response of plane structures subjected to earthquake forces, and was developed by Kannan and Powell (9). The program concepts and features are described in Reference 9. User's Guide (13) describes the extensions made to the program and presents input data procedures.

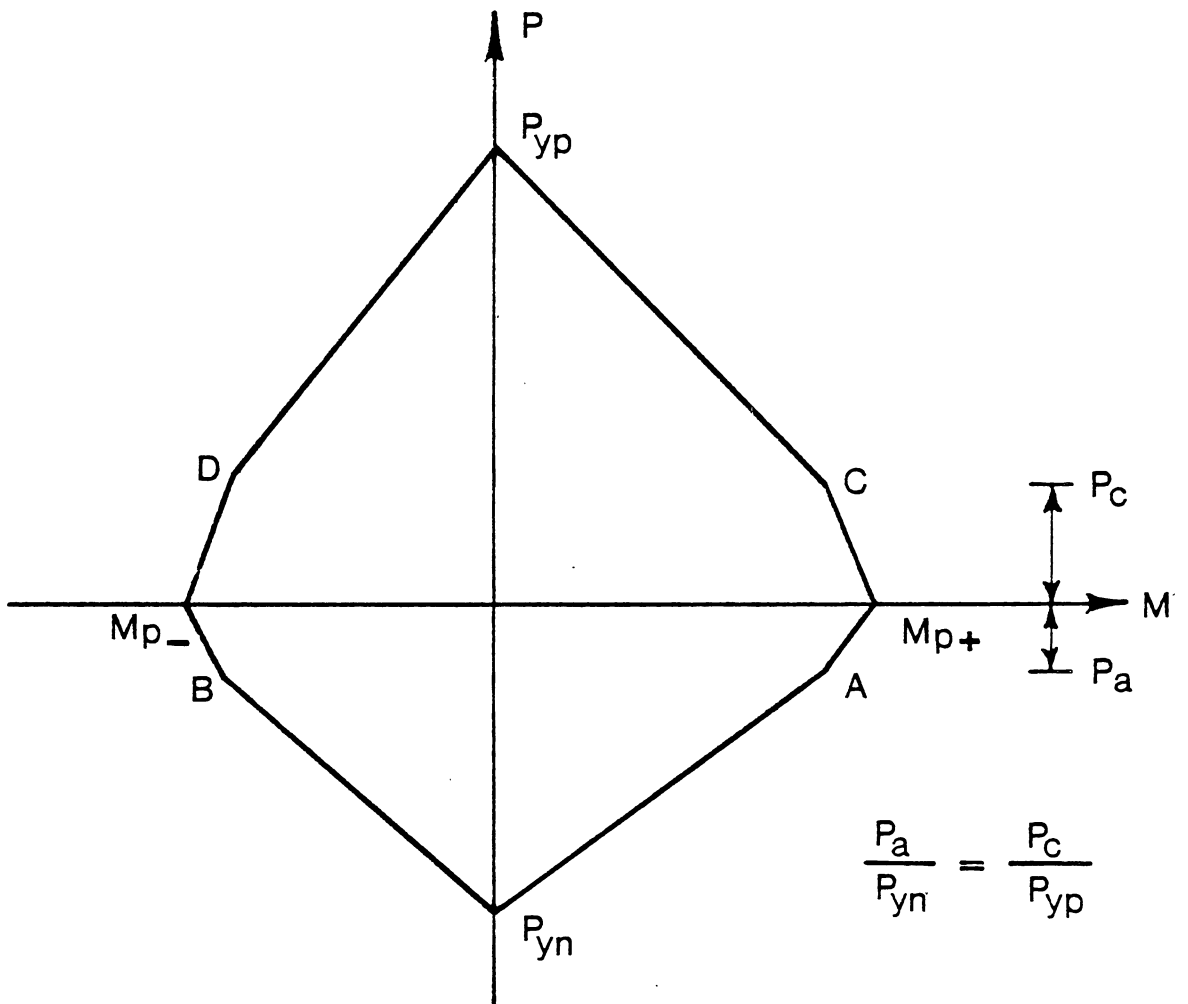


Figure 3 - Beam-Column Model, Ref. 9

This manual supplements References 3, 9 and 13, and should be used in conjunction with them. For compatibility the format of presentation in this manual has also been styled after these references. The procedure followed in adding the new elements EL9 and EL10 to the DRAIN-2D conforms to Chapter 4 of Reference 9. The four main subroutines developed for each element are as follows. The number at the end of the subroutine name corresponds to the element type.

1. INEL9, INEL10: Input and initialization of element data.
2. STIF9, STIF10: Calculation of element tangent stiffness at different time steps.
3. RESP9, RESP10: Determination of increments of element deformations (strains) and forces (stress-
es), determination of yield status, and output of time history results. This may be called as "state determination phase".
4. OUT9, OUT10: Output of final envelope values for element deformations and forces.

This arrangement is used in DRAIN-2D program and is taken directly from it. The variable names have been kept the same as for other elements (9,15). FORTRAN listing for elements EL9 and EL10 are given in Appendices A-1 and A-2, respectively. Several COMMENT statements are given for understanding the underlying logic. These programs have been used on AMDAHL 470V/6 computer at the University

of Michigan using MTS. It is believed that they can be easily used on other systems.

If the user has other element subroutines which are also called either EL9 or EL10, then the suffix 9 or 10 from all the subroutines of these elements including CALL statements should be changed. The new suffix should be less than 10, otherwise, significant additions and changes would have to be made in the main DRAIN-2D program (Cards B to AB) in order to accomodate more than ten elements.

CHAPTER 2

BUCKLING ELEMENT (EL9)

Singh (15) presented a multilinear hysteresis model (EL7) for axially loaded pin-ended bracing members and used in the seismic analysis of multistory braced frames. Jain, Goel and Hanson (7) compared their experimental hysteresis curves with analytical curves obtained by using Singh's model and suggested that this model could be improved if modifications were made in compression envelope and tension envelope regions to include the change in compression loads and residual elongation. The new buckling model accounts for these two parameters, yet, retains the simplicity of Singh's model.

The buckling model is described in the following section. Tension load and displacement are taken as positive, and compression load and displacement are taken as negative.

GENERAL CHARACTERISTICS

Assuming that an initially straight member is loaded first in tension, the member follows segment AE elastically as shown in Figure 4a (computer print out code for this segment = 0). The member yields at E and follows segment EE' (Code = 9). If the direction of displacement is reversed at E', the member unloads elastically, parallel to the initial elastic slope AE (code = 0). Continued compression will result in the first buckling of the member

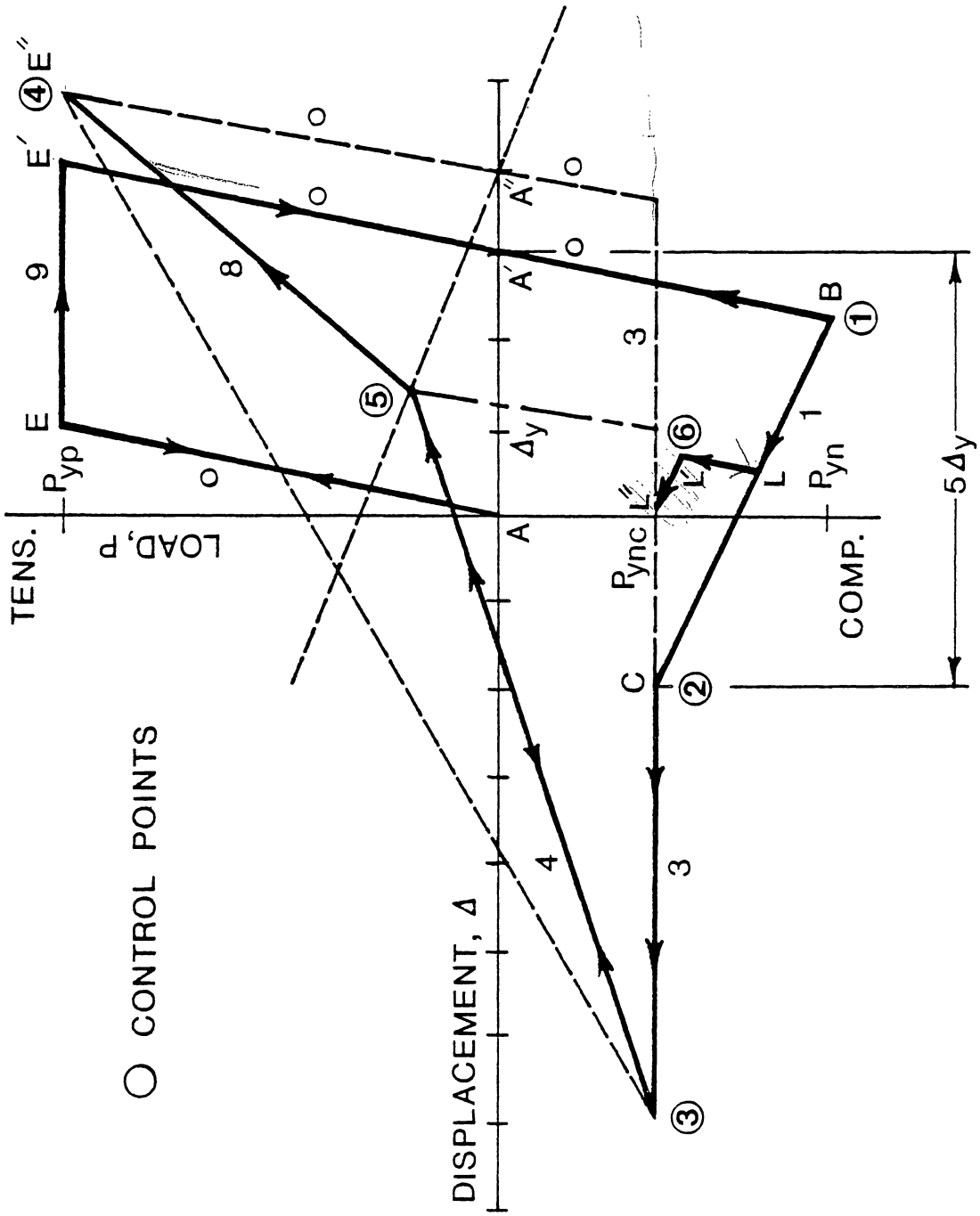


Figure 4a - Axial Hysteresis Behavior Used in Buckling Model

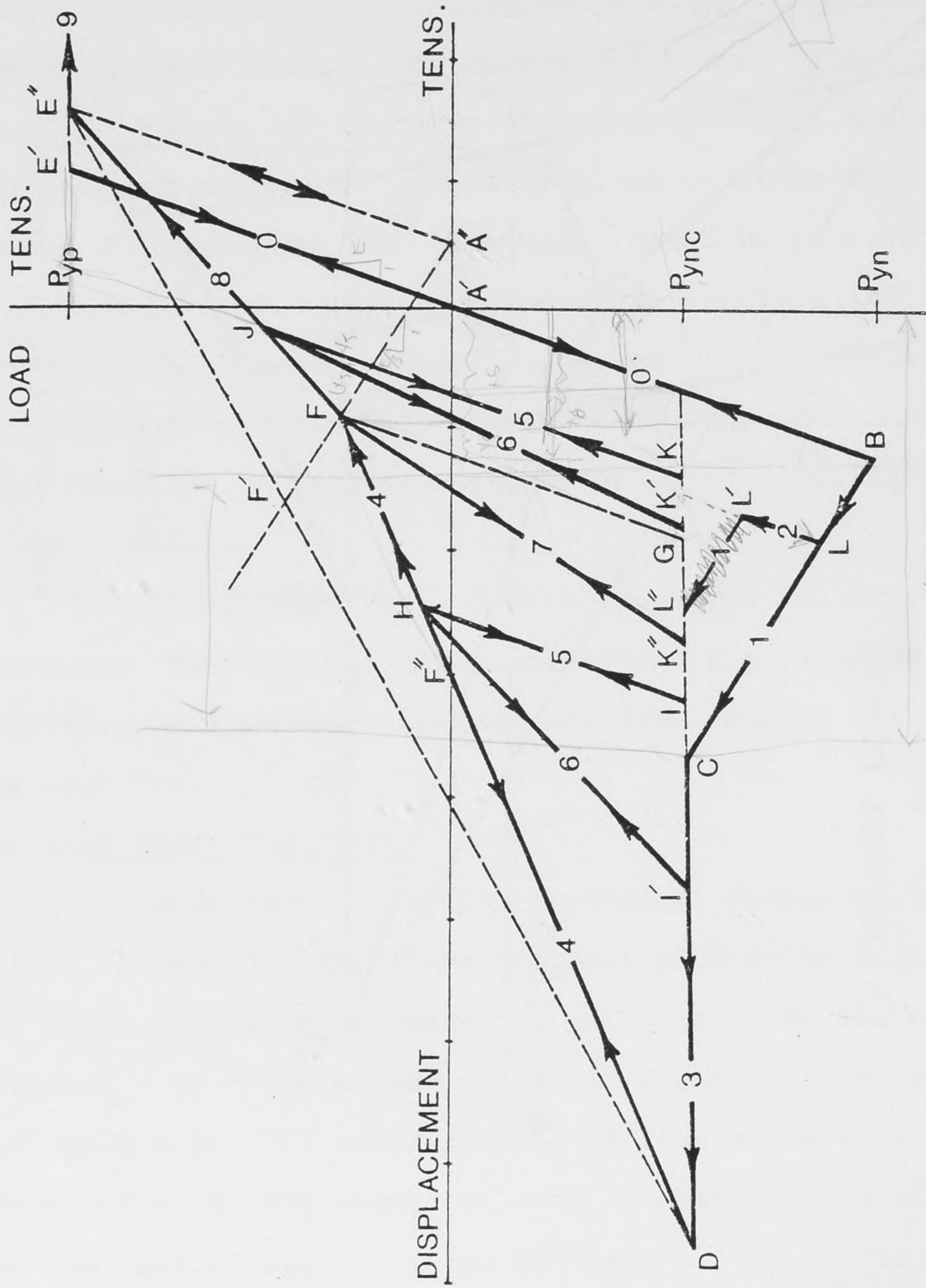


Figure 4b - Axial Hysteresis Behavior Used in Buckling Model (cont.)

at point B. The load at point B corresponds to the first cycle buckling load P_{yn} for the member which is significantly higher than the buckling load P_{ync} used for subsequent cycles. After buckling at point B the member follows segment BC (code = 1). The point C corresponds to a compression displacement equal to five times the tension yield displacement Δy of the member (Ref. 3). If the direction of axial displacement is reversed at L, the member follows segment LL' (code = 2), parallel to the initial elastic slope AB until it hits the post-buckling load level, P_{ync} . However, if the direction of axial displacement is reversed at L', the member follows segment L'L" (code = 1) which is parallel to segment BC. L" lies at the P_{ync} load level.

Once the member hits the post-buckling load level P_{ync} , it comes out of Subroutine VRTX9 and reenters into the main Subroutine RESP9 for further state determination (Figure 4b).

From point C or L", continued compression results in segment CD or L"D, respectively (code = 3). If the direction of axial displacement is reversed at D, it results in compression load decreasing to zero and followed by an increasing tensile load along the path DFE" (code = 4 for segment DF and code = 8 for segment FE").

To locate the point F, a line A"F' is drawn from the new origin A" ($\overline{AA''} = \overline{EE''}$) at a slope of 1/3 times the

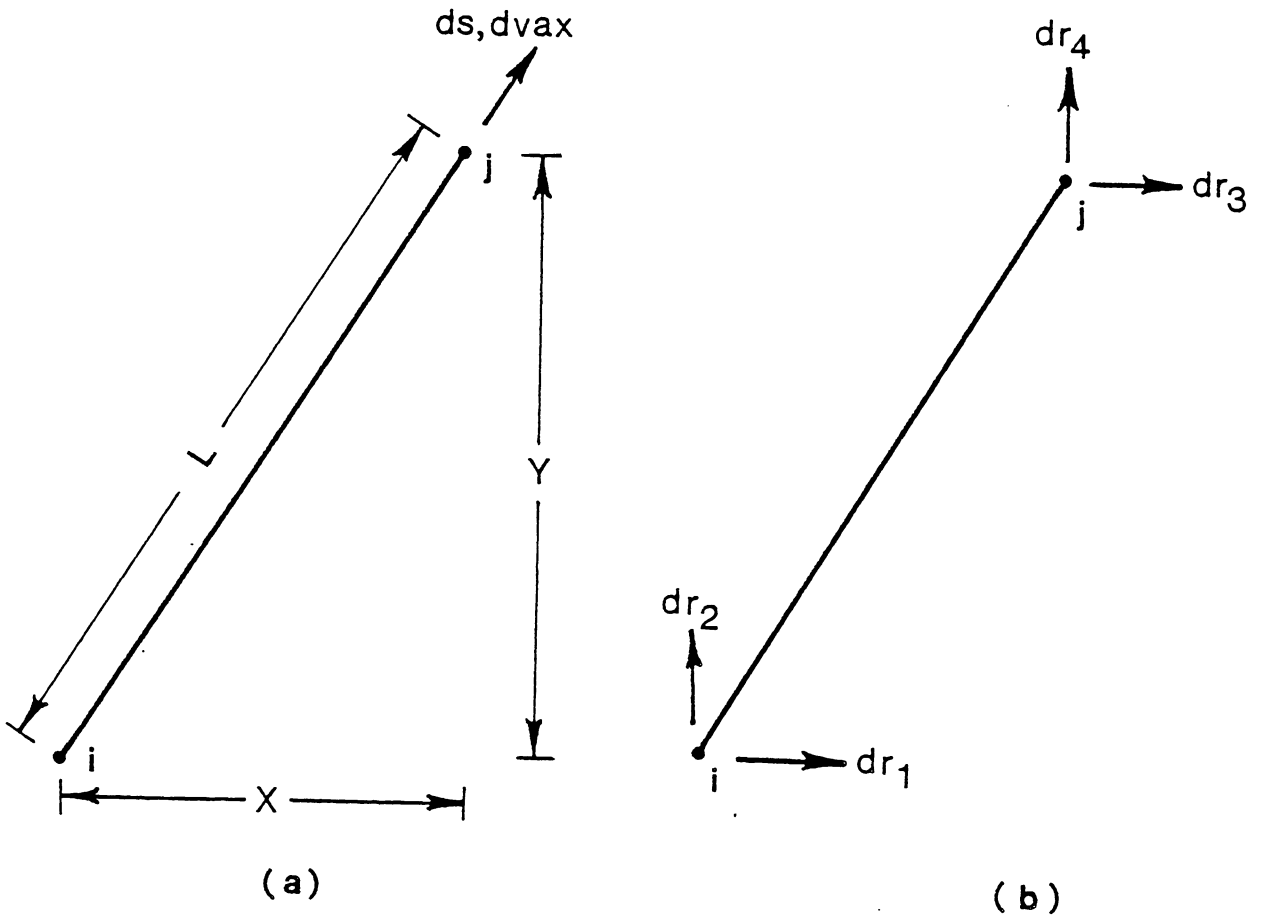


Figure 5 - Deformations and Displacements

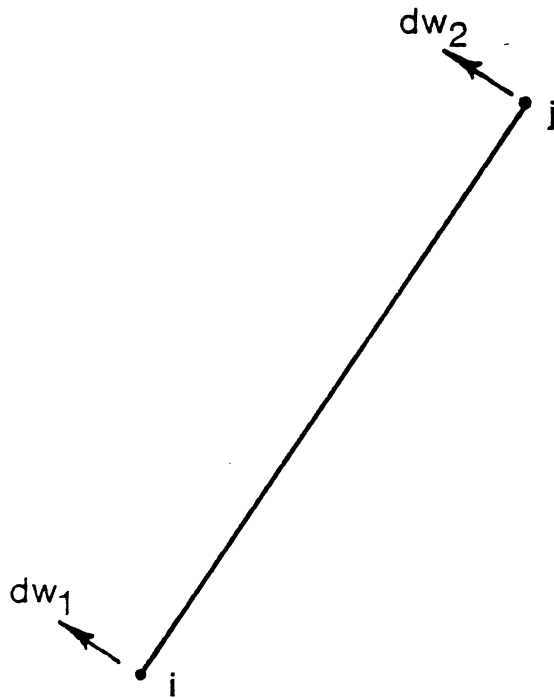


Figure 6 - Displacement for Geometric Stiffness

ELEMENT STIFFNESS

The tangent stiffness in term of deformations is

$$dS = \frac{E_T A}{L} dv \quad 2.4$$

$$\text{or, } \{dS\} = [k_T] \{dv\} \quad 2.5$$

where, E_T = tangent modulus in current state, and A = element cross sectional area.

The tangent stiffness in terms of nodal displacements is

$$[K_T] = [a]^T [k_T] [a] \quad 2.6$$

where, $[a]$ is given by equations 2.2 and 2.3.

The geometric stiffness in the element coordinates dw_1 and dw_2 is (Figure 6):

$$[k_G] = \frac{S}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad 2.7$$

or, in terms of nodal displacements

$$[K_G] = [a_1]^T [k_G] [a_1] \quad 2.8$$

where, $[a_1]$ is given by

$$\begin{Bmatrix} dw_1 \\ dw_2 \end{Bmatrix} = \begin{bmatrix} -\frac{Y}{L} & \frac{X}{L} & 0 & 0 \\ 0 & 0 & -\frac{Y}{L} & \frac{X}{L} \end{bmatrix} \begin{Bmatrix} dr_1 \\ dr_2 \\ dr_3 \\ dr_4 \end{Bmatrix} \\ = [a_1] \{dr\} \quad 2.9$$

FIXED END AND INITIAL FORCES

The effects of static loads applied along the element length rather than at the nodes can be taken into account by specifying fixed end force patterns. Static thermal effects can also be considered in the same way. The forces to be specified are the forces on the element ends required to prevent them from displacing, with the sign convention shown in Figure 7. If axial forces having different magnitudes at ends i and j are specified, the average value is assumed for determining the yield status of the element and for computing the geometric stiffness.

Elements may be stressed under static load but it may be incorrect or inconvenient to determine the element forces by applying static loads to the structure. To allow for such cases, provision is made for initial forces to be specified in the elements. These forces will typically be the forces in the elements under static loading as calculated by a separate analysis. For consistency, they should be in equilibrium with the static load producing them, but this is not essential. The computer program does not make corrections for any equilibrium unbalance resulting from the specification of initial forces.

To satisfy the requirement that the structure remain elastic under static loading, the initial element forces should be less than the yield strengths of the element. If desired, static loads as well as initial forces may be specified. The element forces will then be the sum of the

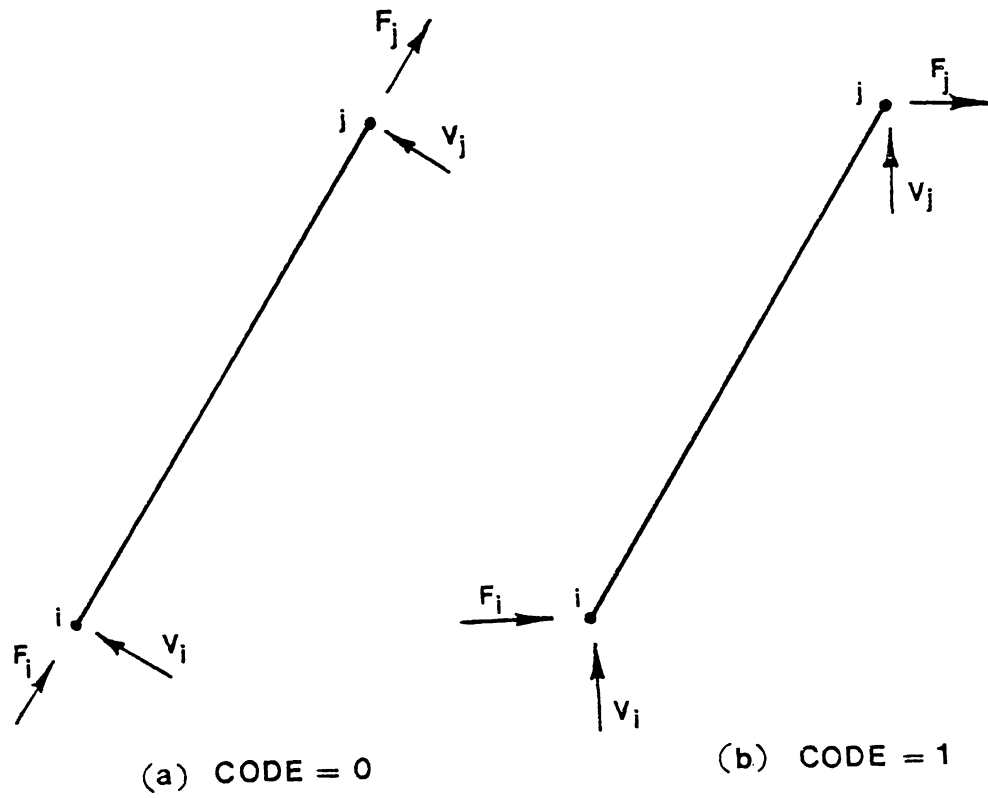


Figure 7 - End Clamping and Initial Forces

initial forces and those due to the static loads. The geometric stiffness effect is not included in the static analysis.

OUTPUT RESULTS

The following results are printed for the static loading condition (time = 0) and at each output time if a time history is requested. The static results are output for all elements, and the time history results for only those elements for which time histories are requested.

1. Yield code: 0 to 9 as explained earlier in Figures 4a and 4b.
2. Axial force, tension positive.
3. Net axial extension.
4. Accumulated positive and negative plastic extensions up to the current time.

These accumulated deformations are computed by accumulating the plastic extensions during all positive and negative plastic excursions. These accumulated deformations, together with the maximum positive and negative total extensions, provide information on the amount of plastic deformation imposed on the element. The maximum positive and negative values of axial force, maximum positive and negative extension and accumulated plastic extension are printed at the time intervals requested for results envelopes. The times at which the maximum forces and extensions were produced are also printed.

INPUT DATA PREPARATIONE9. BUCKLING ELEMENTS - EL9

Number of words of information per element = 53.

E9(a). CONTROL INFORMATION FOR GROUP (415) - ONE CARD

Columns 5: Punch 9 (to indicate that group consists of buckling elements).

 6 - 10: Number of elements in group.

 11 - 15: Number of different element stiffness types (max. 40). See Section E9(b).

 16 - 20: Number of different fixed end force patterns (max. 40). See Section E9(c).

E9(b). STIFFNESS TYPES (I5, 7F10.0) - ONE CARD FOR EACH STIFFNESS TYPE

Columns 1 - 5: Stiffness type number, in sequence beginning with 1.

 6 - 15: Young's modulus of elasticity.

 16 - 25: Average cross sectional area.

 26 - 35: Tension yield force, P_{yp}

 36 - 45: Compression yield force, P_{yn} (first cycle)

 46 - 55: Radius of gyration

 56 - 65: Effective length factor

 66 - 75: Strength reduction factor, PHI

E9(c). FIXED END FORCE PATTERNS (2I5, 4F10.0) - ONE CARD FOR EACH
FIXED END FORCE PATTERN

Omit if there are no fixed end forces. See Figure 7.

Columns 1 - 5: Pattern number, in sequence beginning with 1.

 10: Axis code, as follows:

Code = 0: Forces are in the element coordinate system, as in Figure 7a.

Code = 1: Forces are in the global coordinate system, as in Figure 7b.

Columns 11 - 20: Clamping force F_i .
 21 - 30: Clamping force V_i .
 31 - 40: Clamping force F_j .
 41 - 50: Clamping force V_j .

E9(d). ELEMENT GENERATION COMMANDS (9I5, 2F5.0, F10.0) - ONE CARD FOR EACH GENERATION COMMAND

Elements must be specified in increasing numerical order. Cards for the first and last elements must be included. See Note 7 of User's Guide (13) for explanation of generation procedure.

Columns 1 - 5: Element number, or number of first element in a sequentially numbered series of elements to be generated by this command.

6 - 10: Node number of element end i .

11 - 15: Node number of element end j .

16 - 20: Node number increment for element generation
 If zero or blank, assumed to be equal to 1.

21 - 25: Stiffness type number.

30: Code for including geometric stiffness.
 Punch 1 if geometric stiffness is to be included. Leave blank or punch zero if geometric stiffness is to be ignored.

35: Time history output code. If a time history

of element results is not required for the elements covered by this command, punch zero or leave blank. If a time history printout, at the intervals specified on card D1, is required, punch 1.

Columns	36 - 40:	Fixed end force pattern number for static dead loads on element. Leave blank if there are no dead loads. See note below.
	41 - 45:	Fixed end force pattern number for static live load on element. Leave blank if there are no live loads.
	46 - 50:	Scale factor to be applied to fixed end forces due to static dead loads. Leave blank if there are no dead loads.
	51 - 55:	Scale factor to be applied to fixed end forces due to static live loads. Leave blank if there are no live loads.
	56 - 65:	Initial axial force on element, tension positive.

NOTE: If static load code, Card C1, is zero but fixed end forces are still specified for some elements, an inconsistency results. In effect any such fixed end forces will be treated as initial element forces.

CHAPTER 3

END MOMENT-BUCKLING ELEMENT (EL10)

The end moment-buckling element is a combination of beam-column element (EL2) and buckling element (EL9). This element considers the interaction between the end moments and axial force in the beam-column component EL2, the axial force being determined by the buckling element EL9. In this formulation, the flexural stiffness is assumed to be independent of the axial force. Workman (18) studied the influence of axial force-flexural stiffness interaction in the elastic state on the seismic response of braced steel frames. He concluded that the effect of this interaction was not significant for the structural response. Nigam (11) proposed a more consistent procedure for considering the interaction between forces existing at sections where yielding occurs, but it is very complex and, therefore, not considered for this interactive element EL10. It is believed that the axial force-end moment interaction as modeled herein should be adequate for practical applications.

GENERAL CHARACTERISTICS

End moment-buckling element has six degrees of freedom and may be arbitrarily oriented in the X-Y plane. The element possess axial and flexural stiffnesses. Variable cross-sections can be considered by specifying average area and appropriate flexural stiffness

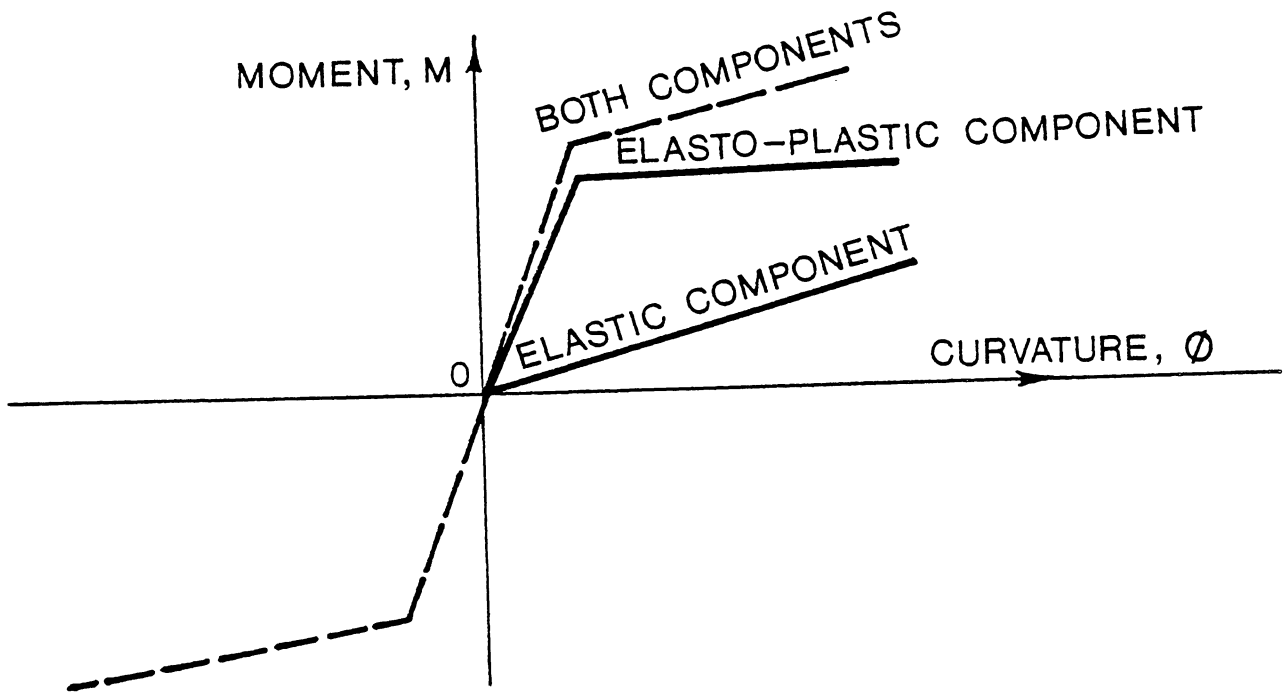


Figure 8 - Moment-Curvature Relation

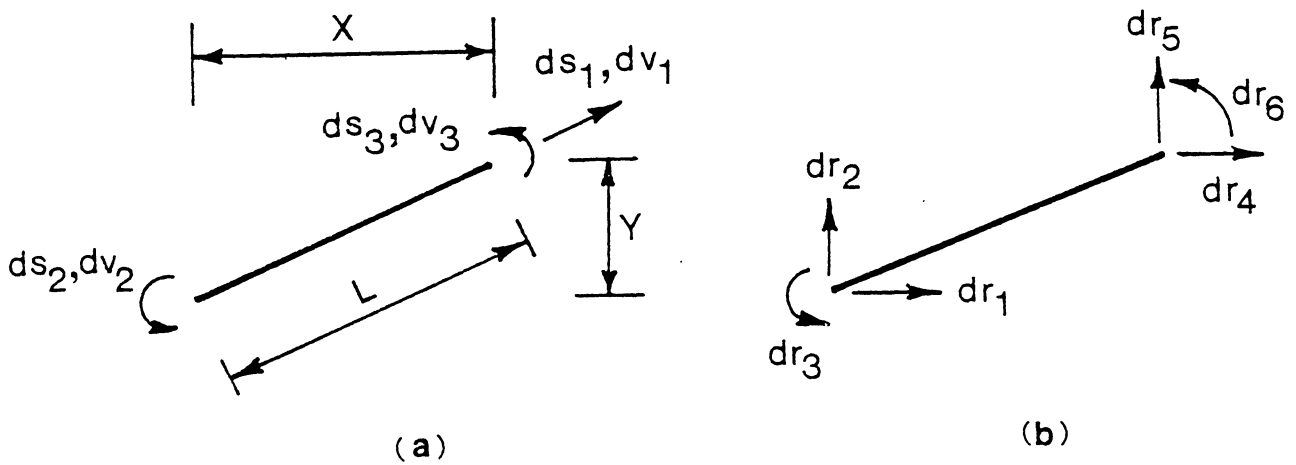


Figure 9 - Deformations and Displacements

coefficients. Flexural shear deformations can also be taken into account.

Strain hardening is considered in the moment-rotation relationship but not in the axial force-axial displacement relationship. Strain hardening is approximated by assuming that the element consists of elastic and elasto-plastic components in parallel as shown in Figure 8. Flexural yielding may take place only in concentrated plastic hinges at the ends of the element. The plastic hinges in the elasto-plastic component rotate under constant moment, but the moment in the elastic component may continue to increase.

The plastic moment capacities may be specified to be different at the two ends of an element and also for positive and negative bending at each end. If tension yield and compression strengths are different at the two ends of an element, minimum values of tension yield and compression strengths are used.

Static loads applied along any element length may be taken into account by specifying fixed end force values. The results of separate static load analyses can be imposed by specifying initial force values.

Large displacement effects may be approximated in the dynamic analysis by including simple geometric stiffnesses based on the element axial forces under static load.

ELEMENT DEFORMATIONS

An end moment-buckling element has three modes of

deformation, namely axial extension, flexural rotation at end i, and flexural rotation at end j. The displacement transformation relating increments of deformation and displacement (Figure 9) is:

$$\begin{Bmatrix} dv_1 \\ dv_2 \\ dv_3 \end{Bmatrix} = \begin{bmatrix} -X/L & -Y/L & 0 & X/L & Y/L & 0 \\ -Y/L^2 & X/L^2 & 1 & Y/L^2 & -X/L^2 & 0 \\ -Y/L^2 & X/L^2 & 0 & Y/L^2 & -X/L^2 & 1 \end{bmatrix} \begin{Bmatrix} dr_1 \\ dr_2 \\ dr_3 \\ dr_4 \\ dr_5 \\ dr_6 \end{Bmatrix} \quad 3.1$$

$$\text{or, } \{dv\} = [a] \{dr\} \quad 3.2$$

As for the buckling element, X, Y and L are assumed to remain constant.

A plastic hinge forms when the moment in the elasto-plastic component of the element reaches its plastic moment. A hinge is then introduced into this component, the elastic component remaining unchanged. The measure of flexural plastic deformation is the plastic hinge rotation.

For any increments of total flexural rotation, dv_2 and dv_3 , the corresponding increments of plastic hinge rotation, dv_{p2} and dv_{p3} , are given by

$$\begin{Bmatrix} dv_{p2} \\ dv_{p3} \end{Bmatrix} = \begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{Bmatrix} dv_2 \\ dv_3 \end{Bmatrix} \quad 3.3$$

where, A, B, C and D are given in Table 1.

Unloading occurs at a hinge when the increment in hinge rotation is opposite in sign to that of the bending moment.

Inelastic axial deformations obey the same hysteresis law as the Buckling Element EL9 does (Figures 4a and 4b).

INTERACTION SURFACES

The End Moment-Buckling Element uses two types of interaction surfaces. For axial force-axial displacement interaction, it uses the same as used by Buckling Element EL9. For axial force-end moment interaction it uses the envelope as shown in Figure 10.

Knowing the axial deformations, the program first determines the axial state of the element as for EL9. It calculates the axial force and the unbalanced axial force, if any. Then, it calculates the yield moment by using the axial force-moment interaction curve as for beam-column element EL2. If the moment lies on or outside the surface, a plastic hinge is introduced at that end. Combinations outside the yield surface are permitted only temporarily, being compensated for by applying corrective loads in the succeeding time step (Figures 11a and 11b).

Once the axial load in the post-buckling range becomes equal to P_{ync} , the program redefines the four branches of M-P interaction curve in the compression region as shown in Figure 10. Maximum compressive strength of the member for all subsequent cycles remains

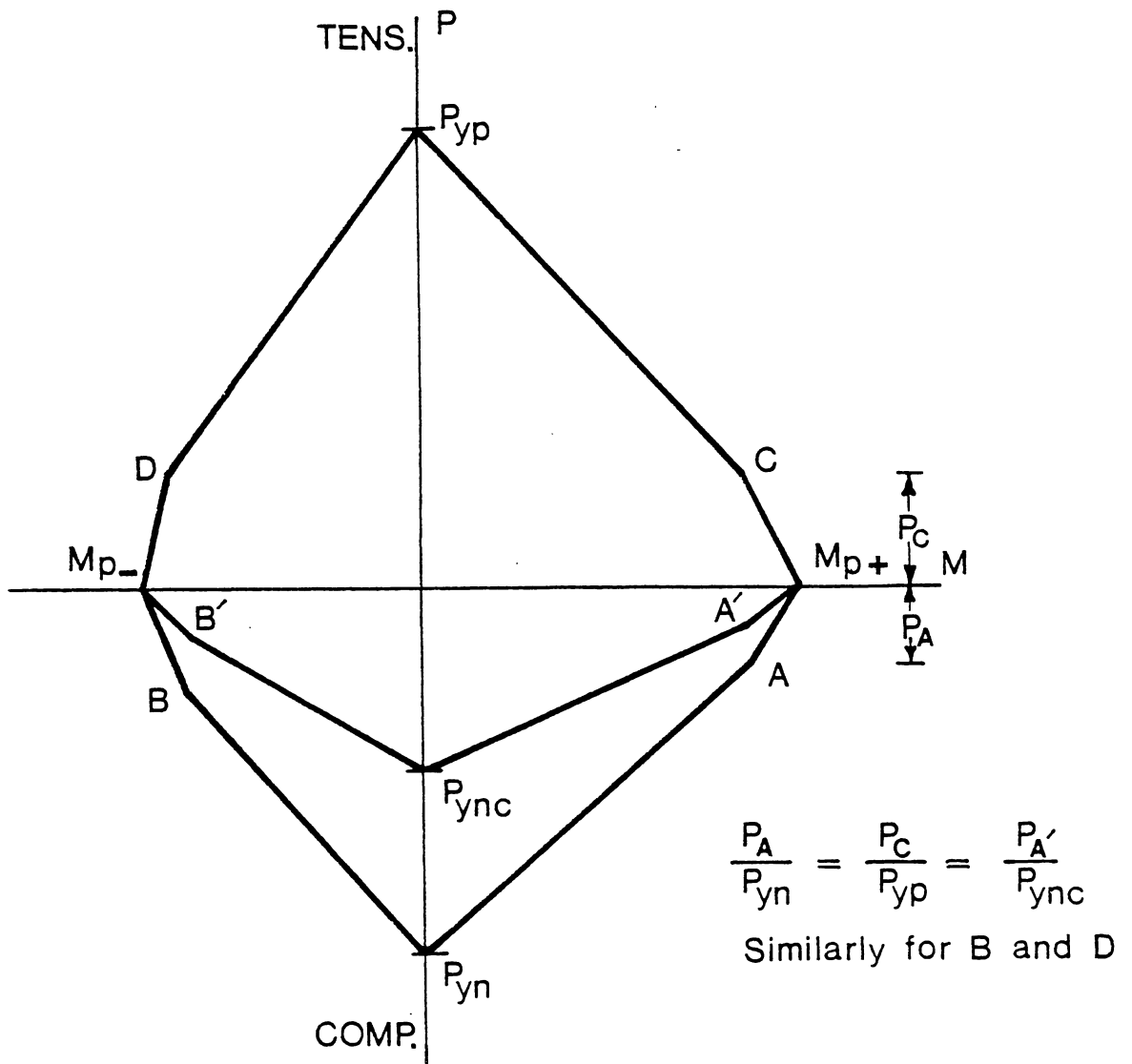


Figure 10 - M-p Interaction Curve Used in EL10

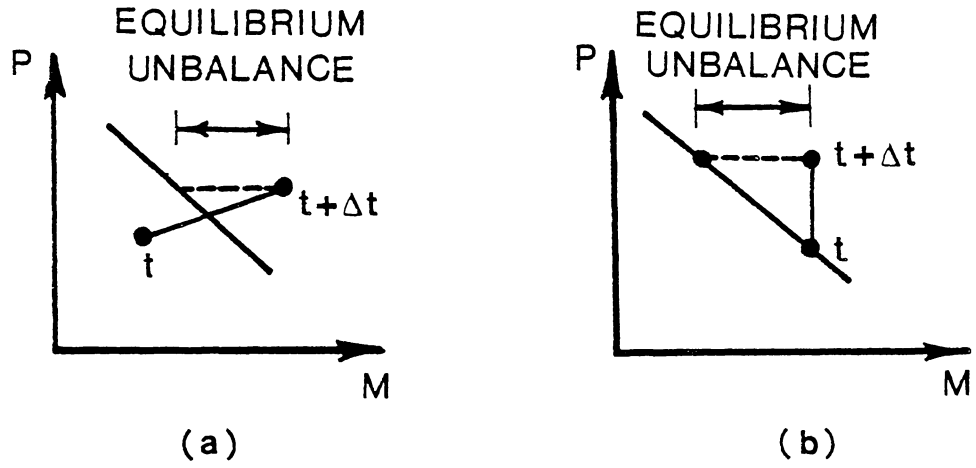


Figure 11 - Equilibrium Correction for Yield Surface Overshoot

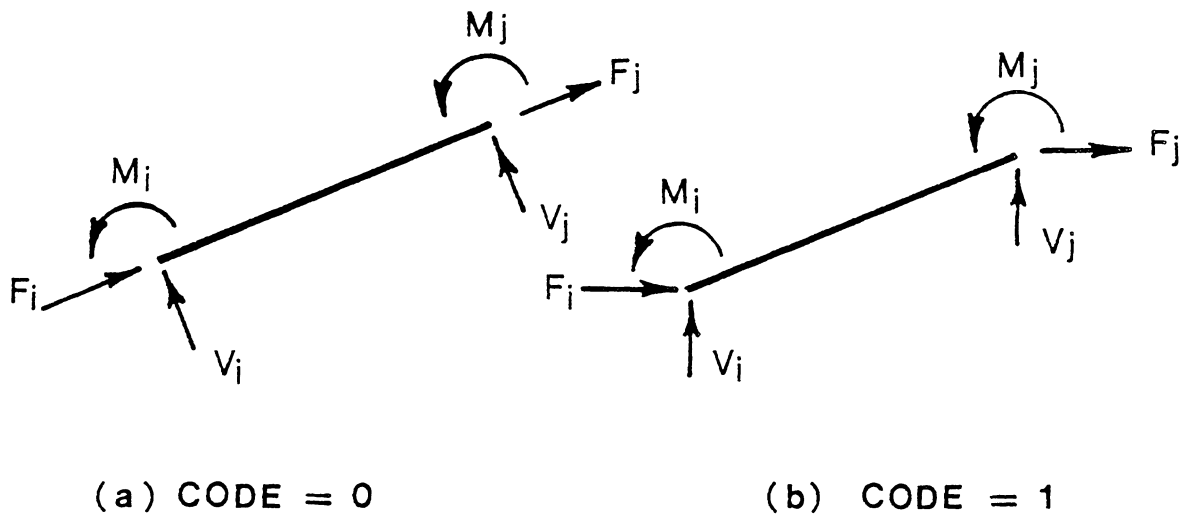


Figure 12 - End Clamping and Initial Forces

at P_{ync} . When the axial load is either P_{yp} (axial yield code = 9), P_{yn} or P_{ync} (axial yield code = 3), the member behaves as a pin-ended member in bending.

ELEMENT STIFFNESS

The element deformations and displacements are shown in Figures 9a and 9b. The axial stiffness is given by

$$dS_1 = \frac{E_T A}{L} dv_1 \quad 3.4$$

where, E_T = tangent modulus in current state, and A = average cross sectional area.

The elastic flexural stiffness is given by

$$\begin{Bmatrix} dS_2 \\ dS_3 \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} k_{ii} & k_{ij} \\ k_{ij} & k_{jj} \end{bmatrix} \begin{Bmatrix} dv_2 \\ dv_3 \end{Bmatrix} \quad 3.5$$

where, I = reference moment of inertia; and k_{ii} , k_{ij} , k_{jj} are coefficients which depend on the cross section variation. For a uniform element, I = actual moment of inertia, $k_{ii} = k_{jj} = 4$, and $k_{ij} = 2$. The coefficients must be specified by the program user, and may, if desired, account for such effects as shear deformations and non-rigid end connections as well as cross section variations.

After one or more hinges form, the coefficients for the elasto-plastic component change to k'_{ii} , k'_{ij} and k'_{jj} , as follows:

$$k'_{ii} = k_{ii}(1-A) - k_{ij}C \quad 3.6$$

$$k'_{ij} = k_{ij}(1-D) - k_{ii}B \quad 3.7$$

$$k'_{jj} = k_{jj}(1-D) - k_{ij}B \quad 3.8$$

TABLE 1
COEFFICIENTS FOR PLASTIC ROTATIONS

Yield Condition	COEFFICIENT			
	A	B	C	D
Both Ends Elastic	0	0	0	0
Plastic hinge at end i only	1	k_{ij}/k_{ii}	0	0
Plastic hinge at end j only	0	0	k_{ij}/k_{jj}	1
Plastic hinges at both ends i and j	1	0	0	1

Note:
Stiffness Coefficients k_{ii} , k_{ij} , and k_{jj} are defined by
Equation 3.5

where, A, B, C and D are defined in Table 1.

Stiffness in term of nodal displacements is obtained as

$$[K_T] = [a]^T [k_T] [a] \quad 3.9$$

where, [a] is given by equations 3.1 and 3.2.

The geometric stiffness used is exactly the same as for the buckling element. This is not the exact geometric stiffness for an end moment-buckling element, but is sufficiently accurate for taking into account the P- Δ effect in building frames.

FIXED END AND INITIAL FORCES

Static loads applied along the lengths of end moment-buckling elements may be taken into account by specifying end clamping forces as shown in Figure 12. These forces are those which must act on the element ends to prevent end displacement.

Initial member forces may be specified for structures in which static analyses are carried out separately. The sign convention for these forces is as shown in Figure 12a. These forces are not converted to loads on the nodes of the structure but simply used to initialize the element end actions. Any end forces due to other loadings are then added to the initial forces.

Initial element forces may be specified in addition to static nodal loads and element end clamping forces in which case the element forces due to the static loading are added to the initial forces. The geometric stiffness, if used, is based on the initial axial force plus any axial force due to static loading, and is included only for the dynamic loading, not for the static loading.

Fixed end and initial forces are defined as standard patterns, and each element can be identified with a standard pattern for dead load fixed end force, live load fixed end force and initial force. In addition multiplication factors for scaling the standard patterns can be specified.

LIVE LOAD REDUCTION

Live load reductions based on area supported may have important effects in buildings and, therefore, should be taken into account. The fixed end forces specified for any element, after scaling by the factors specified for the element, should account for any live load reductions permitted for that element.

The fixed end forces for any element will, when changed in sign, constitute static loadings on the nodes to which the element connects, and these loadings are taken into account by the program. Frequently, however, the live load reduction factor permitted for a column in a building will exceed that for the beams it supports, because columns support tributary loads from several floors. Therefore, if the full live load fixed end shears for each beam are applied at the structure nodes the accumulated loads on the columns may be unnecessarily large. This could be compensated for by reducing the fixed end shears to provide the correct column loads but the shear forces computed for the beams would then be too low. A preferable approach is to take advantage of the live load reduction factors which may be specified with the fixed end force patterns and are used as follows.

For initialization of the element shear and axial forces the full specified fixed end forces are used. However, for computation of the static loads on the nodes connected to the element, the fixed end shear and axial

forces due to live load (but not the moments) are first multiplied by the specified reduction factor. The forces producing axial loads in the columns may, therefore, be reduced to account for difference in permissible live load reductions between the beams and columns, yet the shear forces computed for the beams will still be correct. The reduction factor is ignored for dead loads.

SHEAR DEFORMATIONS

If desired, effective flexural shear areas may be specified. The program then modifies the flexural stiffness to account for the additional shear deformations. However, the fixed end forces are not changed, so that if shear deformations may be important the specified fixed end force patterns should take these deformations into account.

OUTPUT RESULTS

The following results are printed for the static loading condition (all elements, time = 0) and at each output time if a time history is requested. The time-history results are output only for those elements for which time histories are requested.

1. Yield Code:

- (a) Flexural yield code (at each end of an element). Zero indicates the element end is elastic, and 1 indicates that a plastic hinge has formed.

(b) Axial yield code (for the whole element).

0 to 9 as shown in Figures 4a and 4b.

2. Bending moment, shear force and axial force acting at each end of an element, with the sign convention as shown in Figure 12a.
3. Current plastic hinge rotations at each end.
4. Accumulated positive and negative plastic hinge rotations up to the current time.
5. Net axial extension, positive means extension, negative means shortening.

The maximum positive and negative values of bending moment, shear force, axial force, plastic hinge rotations and axial extension, with their time of occurrence, are printed at the time intervals requested for envelopes.

The envelope values of accumulated positive and negative plastic hinge rotations (PRACP(2), PRACN(2)) as well as of accumulated positive and negative axial elongations (VPACP, VPACN) are not printed, although they are computed within the program. Program users interested in these values can easily insert appropriate print statements in Subroutine OUT10.

INPUT DATA PREPARATIONE10. END MOMENT-BUCKLING ELEMENTS - EL10

Number of words of information per element = 170.

E10(a) CONTROL INFORMATION FOR GROUP (6I5) - ONE CARD.

Columns 1 - 5: Punch 10 (to indicate that group consists of end moment-buckling elements).

 6 - 10: Number of elements in group.

 11 - 15: Number of different element stiffness types (max. 40). See Sections E10(b) and E10(c).

 16 - 20: Number of different yield interaction surfaces for cross sections (max. 40). See Section E10(d).

 21 - 25: Number of different fixed end force patterns (max. 35). See Section E10(e).

 26 - 30: Number of different initial element force patterns (max. 30). See Section E10(f).

E10(b). STIFFNESS TYPES (I5, 4F10.0, 3F5.0, 2F10.0) - ONE CARD FOR EACH STIFFNESS TYPE.

Columns 1 - 5: Stiffness type number, in sequence beginning with 1.

 6 - 15: Young's modulus of elasticity.

 16 - 25: Strain hardening modulus, as a proportion of Young's modulus.

 26 - 35: Average cross sectional area.

 36 - 45: Reference moment of inertia.

 46 - 50: Flexural stiffness factor k_{ii} .

Columns 51 - 55: Flexural stiffness factor k_{jj} .
 56 - 60: Flexural stiffness factor k_{ij} .
 61 - 70: Effective shear area. Leave blank or punch zero if shear deformations are to be ignored, or if shear deformations have already been taken into account in computing the flexural stiffness factors.
 71 - 80: Poisson's ratio (used for computing shear modulus, and required only if shear deformations are to be considered).

E10(c). INPUT RADIUS OF GYRATION, K AND PHI FACTORS (I5, 3F10.0) - ONE CARD FOR EACH STIFFNESS TYPE

Columns 1 - 5: Stiffness type number, in sequence beginning with 1.
 6 - 15: Radius of gyration.
 16 - 25: Effective length factor.
 26 - 35: Strength reduction factor, PHI.

E10(d). CROSS SECTION M-P YIELD INTERACTION SURFACES (I5, 4F10.0, 4F5.0) - ONE CARD FOR EACH YIELD SURFACE.

See Figure 10 for explanation.

Columns 1 - 5: Yield surface number, in sequence beginning with 1.
 6 - 15: Positive plastic moment, M_{p+}
 16 - 25: Negative plastic moment, M_{p-}
 26 - 35: Compression yield force in first cycle, P_{yn} .
 36 - 45: Tension yield force, P_{yp} .
 46 - 50: M - coordinate of balance point A, as a

proportion of M_{p+} .

Columns 51 - 55: P - coordinate of balance point A, as a proportion of P_{yn} .

56 - 60: M - coordinate of balance point B, as a proportion of M_{p-} .

61 - 65: P - coordinate of balance point B, as a proportion of P_{yn} .

E10(e). FIXED END FORCE PATTERNS (2I5, 7F10.0) - ONE CARD FOR EACH FIXED END FORCE PATTERN.

Omit if there are not fixed end forces. See Figure 12.

Columns 1 - 5: Pattern number, in sequence beginning with 1.

10: Axis code, as follows:

Code = 0: Forces are in the element coordinate system, as in Figure 12a.

Code = 1: Forces are in the global coordinate system, as in Figure 12b.

11 - 20: Clamping force, F_i .

21 - 30: Clamping force, V_i .

31 - 40: Clamping moment, M_i .

41 - 50: Clamping force, F_j .

51 - 60: Clamping force, V_j .

61 - 70: Clamping moment, M_j .

71 - 80: Live load reduction factor, for computation of live load forces to be applied to nodes.

E10(f). INITIAL ELEMENT FORCE PATTERNS (I5, 6F10.0) - ONE CARD FOR EACH INITIAL FORCE PATTERN.

Omit if there are no initial forces. See Figure 12a.

Columns	1 - 5:	Pattern number, in sequence beginning with 1.
	6 - 15:	Initial axial force, F_i .
	16 - 25:	Initial shear force, V_i .
	26 - 35:	Initial moment, M_i .
	36 - 45:	Initial axial force, F_j .
	46 - 55:	Initial shear force, V_j .
	56 - 65:	Initial moment, M_j .

E10(g). ELEMENT GENERATION COMMANDS (11I5, 2F5.0, I5, F5.0) - ONE CARD FOR EACH GENERATION COMMAND.

Elements must be specified in increasing numerical order. Cards for the first and last elements must be included. See Note 7 of User's Guide (13) for explanation of generation procedure.

Columns	1 - 5:	Element number, or number of first element in a sequentially numbered series of elements to be generated by this command.
	6 - 10:	Node number at element end i.
	11 - 15:	Node number at element end j.
	16 - 20:	Node number increment for element generation. If zero or blank, assumed to be equal to 1.
	21 - 25:	Stiffness type number.
	26 - 30:	Yield surface number for element end i.
	31 - 35:	Yield surface number for element end j.
	40:	Code for including geometric stiffness. Punch 1 if geometric stiffness is to be included. Leave blank or punch zero if geometric stiffness is to be ignored.
	45:	Time history output code. If a time history

of element results is not required for the element covered by this command, punch zero or leave blank. If a time history printout, at the intervals specified on card D1, is required, punch 1.

- Columns 46 - 50: Fixed end force pattern number for static dead loads on element. Leave blank or punch zero if there are no dead loads. See Note below.
- 51 - 55: Fixed end forces pattern number for static live loads on element. Leave blank or punch zero if there are no live loads.
- 56 - 60: Scale factor to be applied to fixed end forces due to static dead loads.
- 61 - 65: Scale factor to be applied to fixed end forces due to static live loads.
- 66 - 70: Initial force pattern number. Leave blank or punch zero if there are not initial forces.
- 71 - 75: Scale factor to be applied to initial element forces.

Note: If the static load code, Card C1, is zero but fixed end forces are still specified for some elements, an inconsistency results. In effect, any such fixed end forces will be treated as initial element forces.

CHAPTER 4

EXAMPLE

The braced frame example shown in Figure 13 may be used to check the execution of the DRAIN-2D program with elements EL9 and EL10 when it is implemented on a computer installation. Program decks received through the Department of Civil Engineering, University of Michigan or the National Information Service for Earthquake Engineering will include a data deck and computer output for this example.

The input cards for the structure are listed in Table 2 and identified by the corresponding sections in the User's Guide (13) and in this report. The user should be able to obtain guidance in data preparation procedures by studying this sample data.

The columns in the example structure are represented by element EL10 in group 1, the beams are represented by element EL5 in group 2 and the bracing members (assumed as pin-connected at the ends) are represented by element EL9 in group 3. Node numbers are shown at the ends of the members and element numbers are shown near the middle.

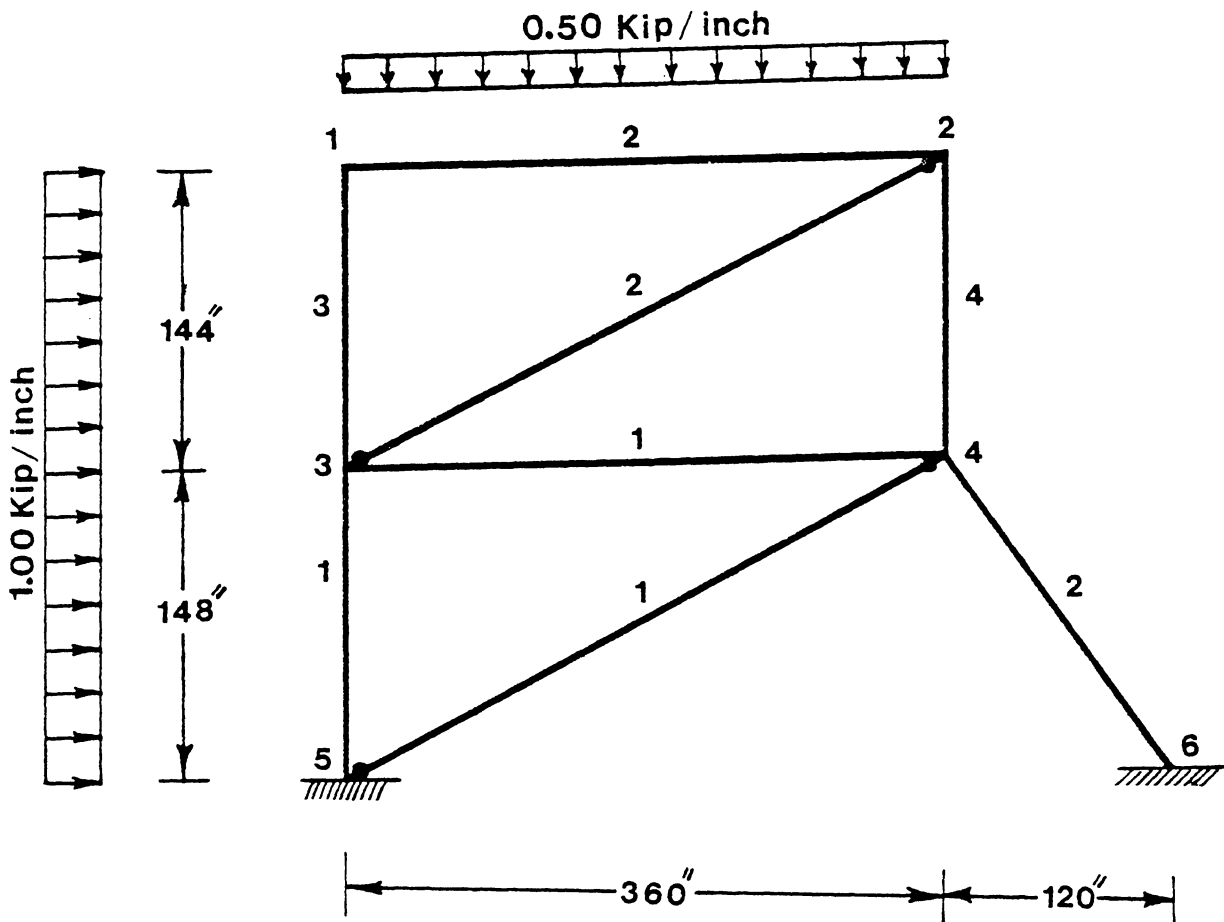


Figure 13 - Test Example

TABLE 2 - SAMPLE INPUT DATA

START	TEST	EXAMPLE	FOR MICHIGAN EL9 AND EL10 ELEMENTS						A
6	6	1	2	1	3			B1	
1		292.							
2	360.	292.							
3		148.						B2	
4	360.	148.							
5									
6	480.								
5	1	1	1	6				B4	
1	2	1	2					B5	
1	2	3	4						
1	1.5				3	2	1.	B6	
1		100	0.01	575.6	1.	1.	1.	20.	C1
4		1	IMPULSE LOADING						C3
		0.03	1.0	0.10	2.0			C4	
10	10		2	4	4				
1	3							D	
1	2	3	4						
1	2	3	4						
10	4	2	2						
1	29000.	0.01	49.1	2020.	4.	4.	2.		
2	29000.	0.01	35.0	1370.	4.	4.	2.		
1	4.01	0.7	0.5						
2	3.75	0.7	0.4					E10	
1	10900.	-10900.	-1700.	1770.	1.	0.15	1.	0.15	
2	7600.	-7600.	-1200.	1275.	1.	0.15	1.	0.15	
1		74.	1825.		74.			-1825.	
2	1	-72.	1728.		-72.			-1728.	
1	5	3	1	1	1	1	1	1.5	
2	4	6	1	1	1	1	1		
3	3	1	2	2	2	1	1		
4	2	4	2	2	2	1	1	2	
								1.25	
5	2	1	2	1					
1	29000.	0.05	41.8	3470.	4.	4.	2.		
1	12850.		-12850.						
2	40000.		-40000.					E5	
1		50.	5400.		50.			-5400.	
1	3	4	1	1	2	1	1		
2	1	2	1	1	1	1	1	1.5	
9	2	2							
1	29000.	17.5	350.	-250.	2.45	0.90	0.55		
2	30000.	15.6	300.	-200.	1.90	1.00	0.35	E9	
1	5	4	1	1	1		50.		
2	3	4	2	1	1		40.		

STOP

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APPENDIX A-1

FORTRAN LISTING OF BUCKLING ELEMENT EL9

SUBROUTINE INEL9 (/KCONT/,/FCONT/,/NDOF/,/NINFC/,/ID/,/X/,/Y/,/NN/
1)

C

COMMON /INPEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVPTX,
3 IELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(147)

COMMON /WDRK/ FTYP(40,7),FEF(40,4),KDFEF(40),DD(4),GA(4,4),
1 PFEF(4),SFF(4),SSFF(4),NMEM,NMBT,NFEF,SLOP1,INEL,
2 INODI,INODJ,INC,IINC,IMBT,IIMBT,IKGM,IKDT,KPDL,
3 IKFDL,KFLL,IKFLL,FDL,FDL,FLL,PFLL,FINIT,PFINIT,
4 XL,YL,AREA,RAD,SLEND,H(*460)

C

DIMENSION KCONT(1),ID(NM,1),X(1),Y(1),COM(1)
DIMENSION AST(2),YESNC(2)
DATA AST/2H ,2H */
DATA YESNO/4H YFS,4H NO /
EQUIVALENCE (IMEM,COM(1))

C

C

C

DATA INPUT, BUCKLING ELEMENTS

NDOF=4
NINFC=53
NMEM=KCONT(2)
NMBT=KCONT(3)
NFEF=KCONT(4)
PRINT 10, (KCONT(I),I=2,4)

20 FORMAT (27H BUCKLING ELEMENTS (TYPE 9)////

1 25H NO. OF ELEMENTS =I4/
2 25H NO. OF STIFFNESS TYPES =I4/
3 25H NO. OF F.E.F. PATTERNS =I4)

C

C

C

INPUT STIFFNESS PROPERTIES

PRINT 20

20 FORMAT (////16H STIFFNESS TYPES//

1 5H TYPE,6X,7H YOUNGS,6X,8H SECTION,3X,
2 15H YIELD FORCES,10X,10H RADIUS OF,5X,
3 12H EQU. LENGTH,5X,19H STRENGTH REDUCTION/
4 5H NO.,5X,8H MODULUS,6X,7H AREA,5X,
5 3H TENSION,5X,8H COMPN,6X,9H GYRATION,4X,
6 12H COEFFICIENT,9X,7H FACTOR/)

C

DO 50 IT=1,NMBT
READ 30, I, (FTYP(IT,J),J=1,7)
50 PRINT 40, IT, (FTYP(IT,J),J=1,7)
30 FORMAT (I5,7F10.0)
40 FORMAT (I4,E14.4,6F13.2)

C

C

C

FIXED END FORCE PATTERNS

IF (NFEF.EQ.0) GO TO 100
PRINT 60


```

60 FORMAT (///25H FIXED END FORCE PATTERNS//
1      3H PATTERN,3X,4HAXIS,2 (7X,5HAXIAL,7X,5HSHEAR)/
2      3H NO. ,3X,4HCODE,2 (7X,5HAT I),2 (7X,5HAT J)/)

```

C

```

DD 90 NF=1,NPEF
READ 70, I,KDFEF(NF), (FEF(NF,J),J=1,4)
70 FORMAT (2I5,4F10.0)
80 FORMAT (I6,I8,1X,4F12.2)
90 PRINT 80, NF,KDFEF(NF), (FEF(NF,J),J=1,4)

```

C

C

C

ELEMENT DATA

100 PRINT 110

110 FORMAT (///22H ELEMENT SPECIFICATION//

```

1      3X,4HELEM,3X,4HNODF,2X,4HNODE,2X,4HNODE,2X,4HSTIF,2X,
2      4HGEOM,2X,4HTIME,3X,12HFEF PATTERNS,3X,17HFEF SCALE FACTORS,
3      5X,7HINITIAL/
4      3X,4H NO.,3X,4H I ,2X,4H J ,2X,4HDIFF,2X,4HTYPE,2X,
5      4HSTIF,2X,4HHIST,3X,12H DL LL ,3X,17H DL LL ,
6      5X,7H FORCE /)

```

C

KDDYX=0

KDDY=0

KST=0

KPP=0.

DD 120 J=19,47

120 COM(J)=0.

C

IMEM=1

130 READ 140, INEL,INODI,INODJ,IINC,IIMBT,IKGM,IKDT,IKFDL,IKFLL,FFDL,F

1FLL,FFINIT

140 FORMAT (9I5,2F5.0,F10.0)

IF (INEL.GT.IMEM) GO TO 170

150 NODI=INODI

NODJ=INODJ

INC=IINC

IF (INC.EQ.0) INC=1

IMBT=IIMBT

KGEOM=IKGM

KOUTDT=IKDT

YNG=YESNO(2)

IF (KGEOM.NE.0) YNG=YESNC(1)

YNT=YESNO(2)

IF (KOUTDT.NE.0) YNT=YESNC(1)

KFDL=IKFDL

KFLL=IKFLL

FDL=FFDL

FLL=FFLL

FINIT=FFINIT

ASTT=AST(1)

IF (INEL.NMEM) 130,170,130

C

160 NODI=NODI+INC

NODJ=NODJ+INC

ASTT=AST(2)

```

C
170 PRINT 180, ASTT, IMEM, NODI, NODJ, INC, IMBT, YNG, YNT, KF DL, KF LL, FDL, FLL,
1FINIT
180 FORMAT (A2, I4, I7, 3I6, 3X, A4, 2X, A4, I7, I6, F11.2, F10.2, F11.2)
C
C   LOCATION MATRIX
C
DO 190 L=1,2
LM(L)=ID(NODI,L)
190 LM(L+2)=ID(NODJ,L)
CALL BAND
C
C   ELEMENT PROPERTIES
C
XL=X(NODJ)-X(NODI)
YL=Y(NODJ)-Y(NODI)
FL=SQRT(XL**2+YL**2)
CO SA=XL/FL
SI NA=YL/FL
AREA=FTYP(IMBT,2)
EAL=FTYP(IMBT,1)*AREA/FL
PYP=FTYP(IMBT,3)
PYN=-ABS(FTYP(IMBT,4))
PHI=FTYP(IMBT,7)
PYNC=PHI*PYN
RAD=FTYP(IMBT,5)
AK=FTYP(IMBT,6)
SLEND=AK*FL/RAD
PFAC=ABS(PYNC/PYP)
SLOP1=(PFAC*(1.-PHI))/(PFAC-5.*PHI)
EAL1=SLOP1*EAL
X3RE=-PFAC
RATIO=60.0/SLEND
DELT Y=PYP/EAL
C
C   LOADS DUE TO FIXED END FORCES
C
SFEF=0.
IF (KF DL+KF LL.EQ.0) GO TO 310
DO 200 I=1, NDOF
DO 200 J=1, NDOF
200 GA(I,J)=0.
GA(1,1)=CO SA
GA(1,2)=SI NA
GA(2,1)=-SI NA
GA(2,2)=CO SA
GA(3,3)=CO SA
GA(3,4)=SI NA
GA(4,3)=-SI NA
GA(4,4)=CO SA
DO 210 I=1,4
SFF(I)=0.
210 SSFF(I)=0.
IF (KF DL.EQ.0) GO TO 250
DO 220 I=1,4

```

```

220 FFEF(I)=PEF(KFDL,I)*FDL
    IF (KDPEF(KFDL).EQ.0) GO TO 230
    CALL MULT (GA,FFE F,SFF,4,4,1)
    GO TO 250
230 DO 240 I=1,4
240 SFF(I)=FFE F(I)
C
250 IF (KFLLEQ.0) GO TO 290
    DO 260 I=1,4
260 FFEF(I)=FEF(KFLL,I)*FLL
    IF (KDPEF(KFLL).EQ.0) GO TO 270
    CALL MULT(GA,FFE F,SSFF,4,4,1)
    GO TO 290
270 DO 280 I=1,4
280 SSFF(I)=FFE F(I)
C
290 DO 300 I=1,4
300 SSFF(I)=SSFF(I)+SFF(I)
C
    CALL MULT (GA,SSFF,DD,4,4,1)
    CALL SFORCE (DD)
C
    INITIALIZE ELEMENT FORCE
C
    SFEF=(SSFF(3)-SSFF(1))*0.5
310 FF=FINIT+SFEF
    SEP=FF
    IF (FINIT.LT.0.) GO TO 320
    SENP=FINIT
    SENN=0.
    GO TO 330
320 SENN=FINIT
    SENP=0.
C
330 CALL FINISH
C
    GENERATE MISSING ELEMENTS
C
    IF (IMEM.EQ.NMEM) RETURN
    IMEM=IMEM+1
    IF (IMEM.EQ.INEL) GO TO 150
    GO TO 160
C
    END

```

```

SUBROUTINE STIF9 (/MSTEP/,/NDOP/,/NINFC/,/COMS/,/FK/,/DFAC/)
C
COMMON /INFEL/ IMEM,KST,LM(4),KGFOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3 IELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(147)
COMMON /WORK/ STIF,STIFF,SST(2,2),AA(2,4),AATK(4,2),FFK(4,4),
1 W(1962)
C
DIMENSION COM(1),COMS(1),FK(4,4)
EQUIVALENCE (IMEM,COM(1))
C
C STIFFNESS FORMULATION, BUCKLING ELEMENTS
C
DO 10 J=3,23
10 COM(J)=COMS(J)
C
CURRENT STIFFNESS
C
CALL FST9 (STIF,KODY)
C
PREVIOUS STIFFNESS
C
IF (MSTEP.LT.2) GO TO 20
CALL FST9 (STIFF,KODYX)
C
STIFFNESS DIFFERENCE
C
STIF=STIF-STIFF
20 FK(1,1)=STIF*COSA**2
FK(1,2)=STIF*SINA*COSA
FK(1,3)=-FK(1,1)
FK(1,4)=-FK(1,2)
FK(2,2)=STIF*SINA**2
FK(2,3)=FK(1,4)
FK(2,4)=-FK(2,2)
FK(3,3)=FK(1,1)
FK(3,4)=FK(1,2)
FK(4,4)=FK(2,2)
DO 30 I=2,4
JJ=I-1
DO 30 J=1,JJ
30 FK(I,J)=FK(J,I)
IF (MSTEP.GT.1) GO TO 80
C
INITIAL STIFFNESS FOR STEP 0, BETA=0 ALLOWANCE FOR STEP 1
C
CC=1.
IF (MSTEP.EQ.1) CC=DFAC
DO 40 I=1,16
40 FK(I,1)=FK(I,1)*CC
C
ADD GEOMETRIC STIFFNESS

```

```
C      IF (MSTEP.EQ.0.OR.KGECM.EQ.0) GO TO 80
      PFL=COMS(34)/PL
      DO 50 I=1,4
50     SST(I,1)=PFL
      DO 60 I=1,8
60     AA(I,1)=J.
      AA(1,1)=-SINA
      AA(1,2)=COSA
      AA(2,3)=SINA
      AA(2,4)=-COSA
      CALL MULTST(AA,SST,AATK,FEK,4,2)
      DO 70 I=1,16
70     FK(I,1)=FK(I,1)+FEK(I,1)
C
80     RETURN
      END
```

```

SUBROUTINE RESP9 (/NDOF//,/NINFC//,/KBAL//,/KPR//,/COMS//,/DDISM//,/DD/
1,/TIME//,/VELM//,/DFAC//,/DELTA/)
C
COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3 IEL OG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(*47)
COMMON /WORK/ EALE,DSL,DSEP,SLIN,FAC,FACTOR,FACAC,DSUB,FVRTX,
1 SLOP6,SLOP7,DVP,DS4,DS5,DS6,DS7,DS8,POUT(1983)
C
DIMENSION COM(1),COMS(1),DDISM(1),DD(1),VELM(1)
EQUIVALENCE (IMEM,COM(1))
C
STATE DETERMINATION, BUCKLING ELEMENTS
C
DO 10 I=1,NINFC
10 COM(I)=COMS(I)
KODYX=KODY
IF (IMEM.EQ.1) THED=0
C
EXTENSION INCREMENT
C
DVAX=COSA*(DDISM(3)-DDISM(1))+SINA*(DDISM(4)-DDISM(2))
VFOT=VTOI+DVAX
C
LINEAR FORCE INCREMENT
C
CALL FST9 (EAL,KODY)
SLIN=SEP+EAL*DVAX
C
INITIALIZE
C
FVRTX=0.
C
CHECK VERTEX STATE
C
IF (IVRTX.EQ.0) CALL VRTX9 (FVRTX,DVAX,TIME)
C
IF (IVRTX.EQ.0) GO TO 120
C
FACAC=FVRTX
C
20 FACTOR=1.-FACAC
KODYI=KODY+1
GO TO (30,120,120,40,50,60,70,80,90,100),KODYI
C
ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
30 DSEP=EAL*DVAX
IF (DSEP) 31,110,32
31 FAC=(PYNC-SEP)/DSFP
IF (FAC.GE.FACTOR) GO TO 32
FACTOR=FAC

```

```

SEP=PYN C
KODY=3
GO TO 110
C
32 FAC=(PYP-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 33
FACTOR=FAC
SEP=PYP
KODY=9
GO TO 110
C
33 SEP=SEP+FACTOR*DSEP
GO TO 110
C
ON SLOPE 3, BUCKLING AND CONTINUING
C
40 IF (DVAX.GT.0.) GO TO 41
C
UPDATE PLASTIC DEFORMATIONS
C
DVP=FACTOR*DVAX
VPACN=VPACN+DVP
GO TO 120
C
BUCKLING AND UNLOADING
C
41 XTOT=(VTOT-DVAX)/DELTY
IF (XTOT.GE.X3RE) GO TO 42
C
ESTABLISH NEW STIFFNESS FOR REVERSE
C
X3RE=XTOT
IELOG=1
CALL LAW9
P5=Y5*PYP
KODY=4
GO TO 50
C
USE OLD STIFFNESS FOR REVERSE
C
42 IF (INDRE.EQ.2.AND.XTOT.LE.X6) GO TO 43
IF ((X5D-XTOT).EQ.0.) GO TO 44
SLOP6=(Y5D+PFAC)/(X5D-XTOT)
EAL6=EAL*SLOP6
KODY=6
GO TO 70
C
43 SLOP7=(Y5+PFAC)/(X5-XTOT)
EAL7=EAL*SLOP7
KODY=7
GO TO 80
44 KODY=5
GO TO 60
C
ON SLOPE 4, GET FACTOR FOR STATUS CHANGE

```

C

```

50 DS4=EAL4*DVAX
   IF (DS4) 51,110,54
51 IF (SEP) 52,52,53
52 FAC=(PYNC-SEP)/DS4
   IF (FAC.GE.FACTOR) GO TO 55
   FACTOR=FAC
   SEP=PYNC
   KODY=3
   GO TO 110
53 X5D=VTOT/DELTY
   Y5D=SEP/PYP
   P5D=SEP
   INDR=1
   KODY=5
   GO TO 60
54 FAC=(P5-SEP)/DS4
   IF (FAC.GE.FACTOR) GO TO 55
   FACTOR=FAC
   SEP=P5
   KODY=3
   GO TO 110
55 SEP=SEP+FACTOR*DS4
   GO TO 110

```

C

C

C

ON SLOPE 5, GET FACTOR FOR STATUS CHANGE

```

60 DS5=EAL*DVAX
   IF (DS5) 61,110,62
61 FAC=(PYNC-SEP)/DS5
   IF (FAC.GE.FACTOR) GO TO 65
   FACTOR=FAC
   SEP=PYNC
   KODY=3
   GO TO 110
62 FAC=(P5D-SEP)/DS5
   IF (FAC.GE.FACTOR) GO TO 65
   FACTOR=FAC
   SEP=P5D
   GO TO (63,64),INDR
63 KODY=4
   GO TO 110
64 KODY=3
   GO TO 110
65 SEP=SEP+FACTOR*DS5
   GO TO 110

```

C

C

C

ON SLOPE 6, GET FACTOR FOR STATUS CHANGE

```

70 DS6=EAL6*DVAX
   IF (DS6) 71,110,72
71 FAC=(PYNC-SEP)/DS6
   IF (FAC.GE.FACTOR) GO TO 75
   FACTOR=FAC
   SEP=PYNC

```



```

      KODY=3
      GO TO 110
72  FAC=(P5D-SEP)/DS6
      IF (FAC.GE.FACTOR) GO TO 75
      FACTOR=FAC
      SEP=P5D
      GO TO (73,74),INDRE
73  KODY=4
      GO TO 110
74  KODY=3
      GO TO 110
75  SEP=SEP+FACTOR*DS6
      GO TO 110
C
C      ON SLOPE 7, GET FACTOR FOR STATUS CHANGE
C
80  DS7=EAL7*DVAX
      IF (DS7) 31,110,82
81  FAC=(PYNC-SEP)/DS7
      IF (FAC.GE.FACTOR) GO TO 83
      FACTOR=FAC
      SEP=PYNC
      KODY=3
      GO TO 110
82  FAC=(P5-SEP)/DS7
      IF (FAC.GE.FACTOR) GO TO 83
      FACTOR=FAC
      SEP=P5
      KODY=8
      GO TO 110
83  SEP=SEP+FACTOR*DS7
      GO TO 110
C
C      ON SLOPE 3, GET FACTOR FOR STATUS CHANGE
C
90  DS8=EAL8*DVAX
      IF (DS8) 91,110,92
91  X5D=VTOT/DELTY
      Y5D=SEP/PYP
      P5D=SEP
      INDRE=2
      KODY=5
      GO TO 60
C
92  FAC=(PYP-SEP)/DS8
      IF (FAC.GE.FACTOR) GO TO 93
      FACTOR=FAC
      SEP=PYP
      KODY=9
      GO TO 110
93  SEP=SEP+FACTOR*DS8
      GO TO 110
C
C      ON SLOPE 9, YIELDED BUT CONTINUING
C

```

```

100 IF (DVAX.LT.0.) GO TO 107
C
C   UPDATE PLASTIC DEFORMATIONS
C
      DVP=FACTOR*DVAX
      XPP=XPP+DVP/DELTY
      VPACP=VPACP+DVP
      GO TO 120
C
C   YIELDED BUT UNLOADING
C
101 KJDY=0
C
C   RESIDUAL ELONGATION, RE
C
      IF (IELOG.NE.1) GO TO 105
      SLEND=60./RATIO
      RE=0.0175*(0.55*X3RE/SLEND+0.0002*X3RE**2)
      RE=RE*FL/DELTY
      XPP=XPP+RE
      IELOG=0
C
105 X3RE=XPP-PFAC
      GO TO 30
C
C   CHECK FOR COMPLETION OF CYCLE
C
110 FACAC=FACAC+FACTOR
      IF (FACAC.LT.0.9999999) GO TO 20
C
C   NEW FORCE, UNBALANCED FORCE DUE TO YIELD
C
120 ST=SEP
      DSUB=SLIN-SEP
      IF (ABS(DSUB).GT.1.E-8) KBAL=1
C
C   DEFORMATION RATE FOR DAMPING
C
      IF (DFAC.EQ.0.0.AND.DELTA.FQ.0.0) GO TO 140
      IF (TIME.EQ.0.) GO TO 150
      KBAL=1
      DVAX=COSA*(VELM(3)-VELM(1))+SINA*(VELM(4)-VELM(2))
C
C   BETA=0 DAMPING FORCE
C
      IF (DFAC.EQ.0.) GO TO 130
      DSUB=DSUB+DFAC*FAL*DVAX
C
C   STRUCTURAL DAMPING FORCE
C
130 IF (DELTA.EQ.0.) GO TO 140
      DSL=DELTA*SIGN(ABS(ST),DVAX)
      DSUB=DSUB-DSL+SDFO
      SDFO=DSL
C

```

```

C      UNBALANCED LOAD VECTOR
C
140 IF (KBAL.EQ.0) GO TO 150
      DD (3) = DSJB * COSA
      DD (4) = DSUB * SINA
      DD (1) = -DD (3)
      DD (2) = -DD (4)
C
C      EXTRACT ENVELOPES
C
150 IF (SENP.GE.ST) GO TO 160
      SENP=ST
      ISENP=TIME
      GO TO 170
160 IF (SENN.LE.ST) GO TO 170
      SENN=ST
      ISENN=TIME
170 IF (VENP.GE.VTOT) GO TO 180
      VENP=VTOT
      IVENP=TIME
      GO TO 190
180 IF (VENN.LE.VTOT) GO TO 190
      VENN=VTOT
      IVENN=TIME
190 CONTINUE
C
C      PRINT TIME HISTORY
C
      IF (KPR.LT.0) GO TO 200
      IF (KPR.EQ.0.OR.KOJTDI.EQ.0) GO TO 240
200 IF (IHED.NE.0) GO TO 220
      KKPR=IABS(KPR)
      PRINT 210, KKPR, TIME
210 FORMAT (///18H RESULTS FOR GROUP,I3,
1         27H, BUCKLING ELEMENTS, TIME =,F8.3
2         //5X,5H FLEM,3X,4HNODE,3X,4HNODE,3X,5HYIELD,8X,5HAXIAL,4X,
3         9H NET ,3X,25HACCU. PLASTIC EXTENSIONS/5X,
4         5H NO.,3X,4H I ,3X,4H J ,3X,5H CODE,8X,5HFORCE,4X,
5         9HEXTENSION,5X,8HPOSITIVE,5X,8HNEGATIVE/)
      IHED=1
220 PRINT 230, IMEM,NODI,NODJ,KODY,ST,VTOT,VPACP,VPACN
230 FORMAT (I9,2I7,I8,F14.2,3F13.5)
C
C      SET INDICATOR FOR STIFFNESS CHANGE
C
240 KST=0
      IF (KODYX.NE.KODY) KST=1
C
C      UPDATE INFORMATION IN COMS
C
      DO 250 J=15,47
250 COMS(J)=COM(J)
      COMS(2)=COM(2)
C
      RETURN
      END

```

```

SUBROUTINE OUT9 (/COMS/,/NINFC/)
C
COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1 DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2 X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,
3 TELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4 TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5 PYP,PYN,PYNC,REST(147)
C
DIMENSION COM(1),COMS(1)
EQUIVALENCE (IMEM,COM(1))
C
ENVELOPE OUTPUT, BUCKLING ELEMENTS
C
DO 10 J=1,NINFC
10 COM(J)=COMS(J)
C
IF (IMEM.EQ.1) PRINT 20
20 FORMAT (27H BUCKLING ELEMENTS (TYPE 9)////
1 5H ELEM,3X,4HNODE,3X,4HNODE,21X,20HMAXIMUM AXIAL FORCES,
2 19X,18HMAXIMUM EXTENSIONS,12X,25HACCUM. PLASTIC EXTENSIONS/
3 5H NO.,3X,4H I,3X,4H J,5X,7HTENSION,3X,4HTIME,
4 6X,5HCOMPEN,3X,4HTIME,5X,8HPOSITIVE,3X,4HTIME,
5 3X,8HNEGATIVE,3X,4HTIME,7X,8HPOSITIVE,5X,8HNEGATIVE/)
C
PRINT 30,IMEM,NODI,NODJ,SENP,TSENP,SENN,TSENN,VENP,TVENP,VENN,TVEN
1N,VPACP,VPACN
30 FORMAT (I4,I7,I7,2X,2(F11.2,F7.2),2X,2(F11.5,F7.2),2X,2F13.5)
C
RETURN
END

```

```

C
C
C
      SJBROUTINE FST9 (/STIF/,/KOD/)
      FORM AXIAL STIFFNESS
      COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1      DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8
2      REST(177)
      KYY=KOD+1
      GO TO (10,20,30,40,50,60,70,80,90,100),KYY
10  STIF=EAL
      GO TO 110
20  STIF=EAL1
      GO TO 110
30  STIF=EAL
      GO TO 110
40  STIF=0.001*EAL
      GO TO 110
50  STIF=EAL4
      GO TO 110
60  STIF=EAL
      GO TO 110
70  STIF=EAL6
      GO TO 110
80  STIF=EAL7
      GO TO 110
90  STIF=EAL8
      GO TO 110
100 STIF=0.001*EAL
C
110 RETURN
      END

```

```

SUBROUTINE LA79
C
C   GENERATE P-DELTA HYSTERESIS CURVE
C
COMMON /IMFEL/ IMEM,KST,LM(4),KGEOM,EAL,FL,COSA,SINA,PFAC,RATIO,
1             DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2             X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,IVRTX,REST(168)
C
C   RESIDUAL ELONGATION, RE
C
C   SLEND=60./RATIO
RE=0.0175*(0.55*X3RE/SLEND+0.0002*X3RE**2)
RE=RE*FL/DELTY
C
C   XPP=XPP+RE
XPP1=1.+XPP
BETA=1./3.
DENOM=(XPP1-X3RE)+(1.+PFAC)/BETA
XNUMP=XPP-X3RE-PFAC
Y5=RATIO*XNUMP/DENOM
X5=XPP-Y5/BETA
SLOP4=(Y5+PFAC)/(X5-X3RE)
EAL4=EAL*SLOP4
SLOP8=(1.-Y5)/(XPP1-X5)
EAL8=EAL*SLOP8
X6=X5-Y5-PFAC
XPP=XPP-RE
C
RETURN
END

```

```

SUBROUTINE VRTX9 (/FACAC/,/DVAX/,/TIME/)
C
C   AXIAL STATE DETERMINATION IN TENSION AND VERTEX REGIONS
C
COMMON /INFEL/ IMEM,KST,LM(4),KGEOM,EAL,PL,COSA,SINA,PFAC,RATIO,
1  DELTY,KODYX,KODY,XPP,X3RE,EAL1,EAL4,EAL6,EAL7,EAL8,
2  X5,Y5,P5,X6,X5D,Y5D,P5D,INDBE,IVRTX,
3  IELOG,SEP,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
4  TVENN,SENP,TSENP,SENN,TSENN,SDFO,NODI,NODJ,KOUTDT,
5  PYP,PYN,PYNC,REST(147)
C
C   INITIALIZE
C
FACAC=0.
10 FACTOR=1,-FACAC
KODYI=KODY+1
IF (KODYI.EQ.10) KODYJ=4
GO TO (20,30,40,50),KODYI
C
C   ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
20 DSEP=EAL*DVAX
IF (DSEP) 21,50,22
21 FAC=(PYN-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 23
FACTOR=FAC
SEP=PYN
KODY=1
C
C   SENN=PYN
C   TSENN=TIME
C   GO TO 60
C
22 FAC=(PYP-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 23
FACTOR=FAC
SEP=PYP
KODY=9
GO TO 60
C
23 SEP=SEP+FACTOR*DSEP
GO TO 60
C
C   ON SLOPE 1, GET FACTOR FOR STATUS CHANGE
C
30 JS1=EAL1*DVAX
IF (EAL1.NE.0) GO TO 34
KODY=3
IVRTX=1
GO TO 80
C
34 IF (DS1) 31,60,32
C
C   BUCKLED BUT LOADING
C

```

```

31 KODY=2
   GO TO 40
C
C   BUCKLING AND CONTINUING
C
32 FAC=(PYNC-SEP)/DS1
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYNC
   KODY=3
C
C   UPDATE PLASTIC DEFORMATION
C
   VPACN=VPACN+FACTOR*DVAX
   GO TO 70
C
33 SEP=SEP+FACTOR*DS1
C
C   UPDATE PLASTIC DEFORMATION
C
   VPACN=VPACN+FACTOR*DVAX
   GO TO 60
C
C   ON SLOPE 2, GET FACTOR FOR STATUS CHANGE
C
40 DS2=EAL*DVAX
   IF (DS2) 41,60,42
C
C   BUCKLING AND CONTINUING
C
41 KODY=1
   GO TO 30
C
C   BUCKLED BUT LOADING
C
42 FAC=(PYNC-SEP)/DS2
   IF (FAC.GE.FACTOR) GO TO 43
   FACTOR=FAC
   SEP=PYNC
   KODY=3
   GO TO 70
C
43 SEP=SEP+FACTOR*DS2
   GO TO 60
C
C   ON SLOPE 9, TENSION YIELDING
C
50 IF (DVAX.LT.0) GO TO 51
   DVP=FACTOR*DVAX
   XPP=XPP+DVP/DELTY
   VPACP=VPACP+DVP
   GO TO 80
C
C   YIELDED BUT UNLOADING
C

```



```
51 KODY=0
   X3RE=XPP-PFAC
   GO TO 20
C
60 FACAC=FACAC+FACTOR
   IF (FACAC.LT.0.9999999) GC TO 10
   RETURN
C
70 FACAC=FACAC+FACTOR
   IVRTX=1
C
80 RETURN
   END
```

APPENDIX A-2
FORTRAN LISTING OF END MOMENT-BUCKLING
ELEMENT EL10

SUBROUTINE INEL10 (/KCONT/,/FCONT/,/NDOF/,/NINFC/,/ID/,/X/,/Y/,/NN/
1)

C

```
COMMON /INFEL/ IMEM,KST,IM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1 EK22,EK12,FSH,EAL,EK11H,EK22H, KODYX(2),
2 KODY(2),BMTOT(2),SPTOT(2),PTOT(2),PRTOT(2),SENP(8),
3 SENN(8),TENF(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4 SDACT(3),EMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5 PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6 PFAC,RATIC,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7 EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8 IVRTX,IELCG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9 TVENN,PYP,PYN,PYNC,REST(30)
COMMON /WORK/ SFF(8),SSFF(8),DD(6),GA(6,6),FFEF(6),FF(6),
1 PMAX(5,2,40),AA1(4,2,40),AA2(4,2,40),XM(5,2),
2 FIYP(40,9),FEF(35,7),KDFEF(35),FINIT(30,6),
3 NMEM,NMBT,NSURF,NFEF,NINT,INODI,INODJ,INC,IINC,
4 IMBT,IIMBT,IKSFI,IKSPJ,IKGM,IKDT,KFDL,IKFDL,KFLL,
5 IKFLL,FDL,FFDL,FLLF,AK,SLEND,PYP1,PYP2,PYN1,PYN2,
6 KS1,KS2,XL,YL,DET,PLL,SS,W(25)
```

C

```
DIMENSION KCONT(1),ID(NN,1),X(1),Y(1),COM(1),AST(2),YESNO(2),
1 KSF(2),FTYP1(40,3)
EQUIVALENCE (IMEM,COM(1))
DATA AST/2H ,2H */
DATA YESNO/4H YES,4H NC /
```

C

C

DATA INPUT, END MOMENT-BUCKLING ELEMENTS

C

```
NDOF=6
NINFC=170
KCOM=KCONT(1)
NMEM=KCONT(2)
NMBT=KCONT(3)
NSURF=KCONT(4)
NFEF=KCONT(5)
NINT=KCONT(6)
PRINT 10, (KCONT(I),I=2,6)
```

10 FORMAT (39H END MOMENT-BUCKLING ELEMENTS (TYPE *0)////

```
1 34H NO. OF ELEMENTS =I4/
2 34H NO. OF STIFFNESS TYPES =I4/
3 34H NO. OF YIELD SURFACES =I4/
4 34H NO. OF FIXED END FORCE PATTERNS =I4/
5 34H NO. OF INITIAL FORCE PATTERNS =I4)
```

C

C

INPUT STIFFNESS PROPERTIES

C

```
PRINT 20
20 FORMAT (///16H STIFFNESS TYPES//
1 5H TYPE,6X,7H YOUNGS,4X,9HHARDENING,6X,7HSECTION,
2 3X,9HREFERENCE,6X,26HFLEXURAL STIFFNESS FACTORS,
3 8X,5HSHEAR,5X,7HFCISSON/
4 5H NO.,6X,7HMODULUS,4X,9H RATIC ,6X,7H AREA ,
5 3X,9H INERTIA ,6X,26H II JJ TJ ,
6 8X,5H AREA,5X,7H RATIC /)
```

```

DO 30 N=1, NMBT
  READ 40, I, (FTYP(N,J), J=1,9)
30 PRINT 50, N, (FTYP(N,J), J=1,9)
40 FORMAT (I5, 4F10.0, 3F5.0, 2F10.0)
50 FORMAT (I4, E14.4, E13.4, 2F12.2, 3X, 3F10.3, F13.2, F11.3)
C
C   INPUT RADIUS OF GYRATION , K AND PHI FACTORS
C
  PRINT 60
60 FORMAT (////16H STIFFNESS TYPES//
1      5H TYPE, 6X, 10H RADIUS OF, 6X, 18H EQUIVALENT LENGTH,
2      6X, 19H STRENGTH REDUCTION/
3      5H NO., 6X, 10H GYRATION , 12X, 12H COEFFICIENT,
4      12X, 7H FACTOR)
DO 70 N=1, NMBT
  READ 80, I, (FTYP1(N,J), J=1,3)
70 PRINT 90, N, (FTYP1(N,J), J=1,3)
80 FORMAT (I5, 3F10.0)
90 FORMAT (I4, 4X, F10.3, 2(10X, F10.3))
C
C   INPUT M-P YIELD SURFACE PROPERTIES
C
  PRINT 110
110 FORMAT (////25H YIELD SURFACE PROPERTIES//
1      8H SURFACE,          9X, 13HYIELD MOMENTS, 15X,
2      12HYIELD FORCES, 9X, 16HCOORDINATES OF A, 6X,
3      16HCOORDINATES OF E/
4      8H NO. ,          5X, 8HPOSITIVE, 5X, 8HNEGATIVE, 8X,
5      5HCOMPX, 6X, 7HTENSION, 6X, 16HMOMENT FORCE, 6X,
6      16HMOMENT FORCE/)
DO 200 IYT=1, NSURF
  READ 120, I, (SFF(J), J=1,8)
  PRINT 130, IYT, (SFF(J), J=1,8)
120 FORMAT (I5, 4F10.0, 4F5.0)
130 FORMAT (I5, F15.2, 3F13.2, 2(2X, 2F10.3))
  SFF(2) = -ABS(SFF(2))
  SFF(3) = -ABS(SFF(3))
  SFF(4) = ABS(SFF(4))
  IF (SFF(6) .EQ. 0.) SFF(6) = 1.E-6
  IF (SFF(8) .EQ. 0.) SFF(8) = 1.E-6
C
C   STEEL TYPE
C
  PMAX(1,1,IYT) = SFF(3)
  PMAX(1,2,IYT) = SFF(3)
  PMAX(2,1,IYT) = SFF(3) * SFF(6)
  PMAX(2,2,IYT) = SFF(3) * SFF(8)
  PMAX(3,1,IYT) = 0.
  PMAX(3,2,IYT) = 0.
  PMAX(4,1,IYT) = SFF(4) * SFF(6)
  PMAX(4,2,IYT) = SFF(4) * SFF(8)
  PMAX(5,1,IYT) = SFF(4)
  PMAX(5,2,IYT) = SFF(4)
  XM(1,1) = 0.
  XM(1,2) = 0.

```

```

XM (2,1)=SFF(1)*SFF(5)
XM (2,2)=SFF(2)*SFF(7)
XM (3,1)=SFF(1)
XM (3,2)=SFF(2)
XM (4,1)=XM (2,1)
XM (4,2)=XM (2,2)
XM (5,1)=0.
XM (5,2)=0.
DO 190 J=1,2
  PP2=PMAX(1,J,IYT)
  XM2=XM (1,J)
DO 190 I=1,4
  PP1=PP2
  XM1=XM2
  PP2=PMAX(I+1,J,IYT)
  XM2=XM (I+1,J)
  DENOM=XM1*PP2-XM2*PP1
  AA2(I,J,IYT)=(PP2-PP1)/DENOM
190 AA1(I,J,IYT)=(XM1-XM2)/DENOM
200 CONTINUE

```

C
C
C

FIXED END FORCE PATTERNS

```

IF (NFEF.EQ.0) GO TO 250
PRINT 210
210 FORMAT (///25H FIXED END FORCE PATTERNS//
1      8H PATTERN,3X,4HAXIS,7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,
2      7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,5X,8HLL. RED./
3      8H NO. ,3X,4HCODE,7X,5HAT I,7X,5HAT I,6X,6H AT I ,
4      7X,5HAT J,7X,5HAT J,6X,6H AT J ,5X,8H FACTOR /)
DO 220 N=1,NFEF
  READ 230, I,KDFEF(N), (FEF(N,J),J=1,7)
220 PRINT 240, N,KDFEF(N), (FEF(N,J),J=1,7)
230 FORMAT (2I5,7F10.0)
240 FORMAT (I5,I9,F13.2,5F12.2,F12.3)

```

C
C
C

INITIAL FORCE PATTERNS

```

250 IF (NINT.EQ.0) GO TO 300
PRINT 260
260 FORMAT (///28H INITIAL END FORCE PATTERNS //
1      8H PATTERN,7X,5HAXIAL,7X,5HSHEAR,6X,6HMOMENT,7X,5HAXIAL,
2      7X,5HSHEAR,6X,6HMOMENT/
3      8H NO. ,7X,5HAT I,7X,5HAT I,6X,6H AT I ,7X,5HAT J,
4      7X,5HAT J,6X,6H AT J /)
DO 270 N=1,NINT
  READ 280, I, (FINIT(N,J),J=1,6)
270 PRINT 290, N, (FINIT(N,J),J=1,6)
280 FORMAT (I5,6F10.0)
290 FORMAT (I5,3X,6F12.2)

```

C
C
C

ELEMENT SPECIFICATION

```

300 PRINT 310
310 FORMAT (///22H ELEMENT SPECIFICATION//

```

```

1      3X,4HELEM,3X,4HNCDE,2X,4HNODE,2X,4HNODE,2X,4HSTIF,
2      2X,14HYIELD SURFACES,2X,4HGEOM,2X,4HTIME,3X,
3      12HFEF PATTERNS,3X,17HFEF SCALE FACTORS,3X,
4      16H INITIAL FORCES /
5      3X,4H NO.,3X,4H I ,2X,4H J ,2X,4HDIFF,2X,4HTYPE,
6      2X,14H END I  END J ,2X,4HSTIF,2X,4HHIST,3X,
7      12H DL   LL  ,3X,17H DL   LL  ,3X,
8      17H NO.  SCALE FAC./)

```

C

```

      DO 320 J=36,84
320  COM(J)=0.
      KODYX(1)=0
      KODYX(2)=0
      KODY(1)=0
      KODY(2)=0
      KST=0
      DO 325 J=145,167
325  COM(J)=0.
      KODYX1=0
      KODY1=0
      KPP=0.

```

C

```

      IMEM=1
330  READ 340, INEL,INODI,INODJ,IINC,IIMBT,          IKSPI,IKSFJ,IKGM,IKDT,
      1IKPDL,IKFLL,FFDL,FLL,IIINIT,FFINIT
340  FORMAT (11I5,2F5.0,I5,F5.0)

```

C

```

      IF (INEL.GT.IMEM) GO TO 380
350  NODI=INODI
      NODJ=INODJ
      INC=IINC
      IF (INC.EQ.0) INC=1
      IMBT=IIMBT
      KSF(1)=IKSPI
      KSF(2)=IKSFJ
      KGEOM=IKGM
      KOUTDI=IKDT
      YNG=YESNO(2)
      IF (KGEOM.NE.0) YNG=YESNC(1)
      YNT=YESNO(2)
      IF (KOUTDI.NE.0) YNT=YESNC(1)
      KFDL=IKFDL
      KFLL=IKFLL
      FDL=FFDL
      FLLM=FLL
      FLLF=1.
      IF (KFLL.EQ.0) GO TO 360
      FLLF=FEF(1KFLL,7)
      IF (FLLF.EQ.0.) FLLF=1.E-6
360  INIT=IIINIT
      FINI=FFINIT
      ASTT=AST(1)
      IF (INEL.NMEM) 330,380,330

```

C

```

370  NODI=NODI+INC

```

```

      NODJ=NODJ+INC
      ASTT=AST(2)
C
380 PRINT 390, ASTT,IMEM,NCDI,NODJ,INC,IMBT,      KSF(1),KSF(2),YNG,YNT
      1,KFDL,KFL,FDL,FLM,INIT,FINT
390 FORMAT(A2,I4,I7,3I6,2I7,5X,A4,2X,A4,I7,I6,F11.2,F10.2,I7,F11.2)
C
      LOCATION MATRIX
C
      DO 400 I=1,3
      LM(I)=ID(NODI,I)
400 LM(I+3)=ID(NODJ,I)
      CALL BAND
C
      ELEMENT PROPERTIES
C
      XL=X(NODJ)-X(NODI)
      YL=Y(NODJ)-Y(NODI)
      FL=SQRT(XL**2+YL**2)
      COSA=XL/FL
      SIN A=Y L/FL
      YMOD=FTYP(IMBT,1)
      PSH=FTYP(IMBT,2)
      PPSH=1.-PSH
      PSH=PSH/PPSH
      AREA=FTYP(IMBT,3)
      EAL=YMOD*AREA/FL
      RAD=FTYP1(IMBT,1)
      AK=FTYP1(IMBT,2)
      SLEND=AK*FL/RAD
      RATIO=60./SLEND
      KS1=KSF(1)
      KS2=KSF(2)
      PYP1=PMAX(5,1,KS1)
      PYP2=PMAX(5,1,KS2)
      IF(PYP1.LE.PYP2) GO TO 420
      PYP=PYP2
      GO TO 421
420 PYP=PYP1
421 CONTINUE
      PYN1=PMAX(1,1,KS1)
      PYN2=PMAX(1,1,KS2)
      IF(PYN1.LE.PYN2) GO TO 425
      PYN=PYN2
      GO TO 426
425 PYN=PYN1
426 CONTINUE
      PHI=FTYP1(IMBT,3)
      PYNC=PHI*PYN
      PFAC=ABS(PYNC/PYP)
      SLOP1=(PFAC*(1.-PHI))/(PFAC-5.*PHI)
      EAL1=SLOP1*EAL
      X3RE=-PFAC
      DELTY=PYP/EAL
      EIL=YMOD*FTYP(IMBT,4)*PPSH/FL

```

```

FACL=FTYP(IMBT,5)
FACR=FTYP(IMBT,6)
FACLR=FTYP(IMBT,7)
IF (FACL.EQ.0.) FACL=1.E-6
IF (FACR.EQ.0.) FACR=1.E-6
IF (FTYP(IMBT,8).EQ.0.) GC TO 430
SHFAC=EIL/(FTYP(IMBT,1)/(2.*(1.+FTYP(IMBT,9))))*FTYP(IMBT,8)*FL*PPS
1H)
DET=FACL*FACR-FACLR**2
FII=FACR/DET+SHFAC
FJJ=FACL/DET+SHFAC
FIJ=-FACLR/DET+SHFAC
DET=FII*FJJ-FIJ**2
FACR=FII/DET
FACL=FJJ/DET
FACLR=-FIJ/DET
430 EK11=EIL*FACL
EK22=EIL*FACR
EK12=EIL*FACLR
EK11H=EK11-EK12**2/EK22
EK22H=EK22-EK12**2/EK11
ER12=EK12/EK22
ER21=EK12/EK11
C
C 1-P YIELD SURFACE EQUATION DATA FOR EACH END OF AN ELEMENT
C
DO 450 K=1,2
KK=KSF(K)
DO 450 J=1,2
DO 440 I=1,3
440 PMX(I,J,K)=PMAK(I+1,J,KK)
DO 450 I=1,4
A2(I,J,K)=PPSH/AA2(I,J,KK)
450 A1(I,J,K)=AA1(I,J,KK)*A2(I,J,K)
C
C DISPLACEMENT TRANSFORMATION
C
A(1,1)=-SINA/FL
A(1,2)=COSA/FL
A(1,3)=1.
A(1,4)=-A(1,1)
A(1,5)=-A(1,2)
A(1,6)=0.
A(2,1)=A(1,1)
A(2,2)=A(1,2)
A(2,3)=0.
A(2,4)=A(1,4)
A(2,5)=A(1,5)
A(2,6)=1.
C
C LOADS DUE TO FIXED END FORCES
C
DO 480 I=1,6
SFF(I)=0.
480 SSFF(I)=0.

```



```

      IF (KFDL+KFLL.EQ.0) GC TC 610
      DO 490 I=1,6
      DO 490 J=1,6
490  GA(I,J)=0.
      GA(1,1)=CO SA
      GA(1,2)=SINA
      GA(2,1)=-SINA
      GA(2,2)=CO SA
      GA(3,3)=1.
      GA(4,4)=CO SA
      GA(4,5)=SINA
      GA(5,4)=-SINA
      GA(5,5)=CO SA
      GA(6,6)=1.
C
      IF (KFDL.EQ.0) GO TO 530
      DO 500 I=1,6
500  PFEF(I)=FEF(KFDL,I)*FDL
      IF (KDFEF(KFDL).EQ.0) GC TO 510
      CALL MULT (GA,PFEF,SFF,6,6,1)
      GO TO 530
510  DO 520 I=1,6
520  SFF(I)=PFEF(I)
C
530  IF (KFLL.EQ.0) GO TO 570
      DO 540 I=1,6
      FLL=FLLF*FLLM
      IF (I.EQ.3.OR.I.EQ.6) FII=FLLM
540  PFEF(I)=FEF(KFLL,I)*FLL
      IF (KDFEF(KFLL).EQ.0) GC TC 550
      CALL MULT (GA,PFEF,SSFF,6,6,1)
      GO TO 570
550  DO 560 I=1,6
560  SSFF(I)=PFEF(I)
C
570  DO 580 I=1,6
580  FF(I)=SFF(I)+SSFF(I)
C
      CALL MULT (GA,FF,DD,6,6,1)
      CALL SFORCE (DD)
C
C      MODIFY TO GET INITIAL ELEMENT FORCES
C
      DO 600 I=1,6
      FLL=1./FLLF
      IF (I.EQ.3.OR.I.EQ.6) FII=1.
600  SFF(I)=SFF(I)+SSFF(I)*FII
C
C      INITIAL FORCES
C
610  IF (INIT.EQ.0) GO TO 630
      DO 620 I=1,6
620  SFF(I)=SFF(I)+FINIT(INIT,I)*PINT
C
C      INITIALIZE ARRAYS

```

```
C
630 BMEP(1)=SFF(3)*PPSH
    BMEP(2)=SFF(6)*PPSH
    FIOT(1)=SFF(1)
    FIOT(2)=SFF(4)
    SFTOT(1)=SFF(2)
    SFTOT(2)=SFF(5)
    BMTOT(1)=SFF(3)
    BMTOT(2)=SFF(6)
    DO 650 I=1,6
    SS=BMTOT(I)
    IF (SS.LT.0.) GO TO 640
    SENP(I)=SS
    GO TO 650
640 SENN(I)=SS
650 CONTINUE

C
C   YIELD MOMENTS FOR INITIAL FORCE STATE
C
C   CALL YMOM10
C
C   CALL FINISH
C
C   GENERATE MISSING ELEMENTS
C
    IF (IMEM.EQ.NMEM) RETURN
    IMEM=IMEM+1
    IF (IMEM.EQ.INEL) GO TO 350
    GO TO 370

C
    END
```

```

SJBROUTINE STIF10 (/MSTEP/,/NDOF/,/NINFC/,/COMS/,/FK/,/DFAC/)
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),FK11,
1 EK22,EK12,PSH,EAL,EK11H,EK22H, KODYX(2),
2 KODY(2),EMTCT(2),SPTOT(2),FTOT(2),PRTOT(2),SENP(8),
3 SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4 SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5 PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6 PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7 EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8 IVRTX,IFLOG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9 TVENN,EYF,EYN,PYNC,REST(30)
COMMON /WORK/ ST(2,2),STIT(2,2),ATK(6,2),AA(2,6),PFL,AXK,FAC,
1 FFK(6,6),FSK(6,6),W(1893)
C
DIMENSION COM(1),COMS(1),FK(6,6)
EQUIVALENCE (IMEM,COM(1))
C
C STIFFNESS FORMULATION, END MOMENT-BUCKLING ELEMENTS
C
DO 10 J=3,35
10 COM(J)=COMS(J)
DO 15 J=141,149
15 COM(J)=COMS(J)
C
C CURRENT AXIAL STIFFNESS
C
CALL FST10A (STIF,KODY1)
C
C PREVIOUS STIFFNESS
C
IF(MSTEP.LT.2) GO TO 20
CALL FST10A (STIFF,KODYX1)
C
C STIFFNESS DIFFERENCE
C
STIF=STIF-STIFF
20 CONTINUE
DO 30 I=1,36
30 FSK(I,1)=0.
AXK=STIF*CCOSA**2
FSK(1,1)=AXK
FSK(1,4)=-AXK
FSK(4,4)=AXK
AXK=STIF*SINA**2
FSK(2,2)=AXK
FSK(2,5)=-AXK
FSK(5,5)=AXK
AXK=STIF*SINA*COSA
FSK(1,2)=AXK
FSK(1,5)=-AXK
FSK(2,4)=-AXK
FSK(4,5)=AXK
DO 40 I=1,6
DO 40 J=I,6

```

```

40 FSK(J,I)=FSK(I,J)
C
C   CURRENT FLEXURAL STIFFNESS, ELASTO-PLASTIC PART
C
C   CALL FST10B (ST,KODY)
C
C   PREVIOUS STIFFNESS
C
C   IF(MSTEP.LT.2) GO TO 50
C   CALL FST10B (ST,KODYX)
C
C   STIFFNESS DIFFERENCE
C
C   DO 60 I=1,4
60 ST(I,1)=ST(I,1)-STT(I,1)
   CALL MULIST (A,ST,ATK,FF,6,2)
C
C   GET TOTAL STIFFNESS
C
C   DO 70 I=1,6
C   DO 70 J=1,6
70 FK(I,J)=FK(I,J)+FSK(I,J)
   RETURN
C
C   ORIGINAL STIFFNESS AT STEP 0, EETA=0, CORRKN AT STEP 1
C
C   50 FAC=1.
C   IF (MSTEP.NE.1) GO TO 85
C   FAC=DFAC
C   DO 80 I=1,36
80 FSK(I,1)=FSK(I,1)*FAC
85 CC=(1.+PSH)*FAC
C   DO 90 I=1,4
90 ST(I,1)=ST(I,1)*CC
   CALL MULIST (A,ST,ATK,FK,6,2)
C
C   GET TOTAL INITIAL STIFFNESS
C
C   DO 100 I=1,6
C   DO 100 J=1,6
100 FK(I,J)=FK(I,J)+FSK(I,J)
C
C   ADD GEOMETRIC STIFFNESS
C
C   IF (MSTEP.EQ.0.OR.KGFCM.EQ.0) GO TO 120
C   PFL=(COMS(41)-CGMS(40))/(2.*FL)
C   DO 110 I=1,4
110 ST(I,1)=PFL
C   DO 130 I=1,12
130 AA(I,1)=0.
   AA(1,1)=-SINA
   AA(1,2)=COSA
   AA(2,4)=SINA
   AA(2,5)=-COSA
   CALL MULIST (AA,ST,ATK,FFK,6,2)

```

```
      DO 140 I=1,36
140 FK (I,1)=FK (I,1)+PFK (I,1)
C
120 RETURN
C
      END
```

```

SUBROUTINE RESP10 (/NDOF/, /NINFC/, /KBAL/, /KPR/, /COMS/, /DDISM/, /DD/,
1/TIME/, /VELM/, /DFAC/, /DELTA/)

```

C
C
C

```

STATE DETERMINATION, END MOMENT-BUCKLING ELEMENTS

```

```

COMMON /INFEL/ IMEM, KST, IM (6), KGEOM, FL, COSA, SINA, A (2,5), EK11,
1      EK22, EK12, FSH, EAL, EK11H, EK22H,      KODYX (2),
2      KODY (2), BMTOT (2), SFTOT (2), FTOT (2), PRTOT (2), SENP (8),
3      SENN (8), TFNE (8), TENN (8), PRACP (2), PFACN (2), BMEP (2),
4      SDACT (3), BMY (2,2), NODI, NODJ, KOUTDT, PR12, PR21,
5      PMX (3,2,2), A1 (4,2,2), A2 (4,2,2),
6      PFAC, RATIC, DELTY, KODYX1, KODY1, XPP, X3RE, EAL1, EAL4,
7      FAL6, EAL7, EAL8, X5, Y5, P5, X6, X5D, Y5D, P5D, INDRE,
8      IVRTX, IETOG, VTOT, XTOT, VPACP, VPACN, VENP, TVENP, VENN,
9      TVFNN, PYP, PYN, PYNC, REST (30)
COMMON /WORK/ DVR (2), DPP (2), DBM (2), BMTOT (2), BML (2), BMFL (2),
1      DVAX, SLIN, FACAC, FACTOR, FAC, DSP, BMIUB, BMJUB, SFUB,
2      DSEP, DVP, SLCF6, SLOP7, DS4, DS5, DS6, DS7, DS8, EALE, FOU8,
3      KBAL1, KST1, W (1966)

```

C

```

DIMENSION COM (1), COMS (1), DDISM (1), DD (1), VELM (1), NOD (2)
EQUIVALENCE (IMEM, COM (1)), (NODI, NOD (1))

```

C

```

DO 10 J=1, NINFC
10 COM (J) = COMS (J)
KODYX (1) = KODY (1)
KODYX (2) = KODY (2)
KODYX1 = KODY1
IF (IMEM.EQ.1) IPED=0

```

C

C

C

```

DEFORMATION INCREMENTS

```

```

DVAX=COSA*(DDISM(4)-DDISM(1))+SINA*(DDISM(5)-DDISM(2))
ROT=(SINA*(DDISM(4)-DDISM(1))+COSA*(DDISM(2)-DDISM(5)))/FL
DVR (1) =DDISM(3) +ROT
DVR (2) =DDISM(6) +ROT
VTOT=VTOT+DVAX
SEP=(FTOT(2)-FTOT(1))*0.5

```

C

C

C

```

AXIAL FORCE INCREMENT

```

```

CALL FST10A (EALF, KODY1)
SLIN=SEP+EALE*DVAX

```

C

C

C

```

AXIAL STATE DETERMINATION

```

```

FVRTX=0.

```

C

C

C

```

CHECK VERTEX STATE

```

C

```

IF (IVRTX.EQ.0) CALL VRTX10 (FVRTX, DVAX, SEP, TIME)

```

C

```

IF (IVRTX.EQ.0) GO TO 120

```

C

```

FACAC= FVRTX

```

```

C
20 FACTOR=1.-FACAC
   KODYI=KODY1+1
   GO TO (30,120,120,40,50,60,70,80,90,100),KODYI
C
C   ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
30 DSEP=EAL*DVAX
   IF (DSEP) 31,110,32
31 FAC=(PYNC-SEP)/DSEP
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYNC
   KODY1=3
   GO TO 110
C
32 FAC=(PYP-SEP)/DSEP
   IF (FAC.GE.FACTOR) GO TO 33
   FACTOR=FAC
   SEP=PYP
   KODY1=9
   GO TO 110
C
33 SEP=SEP+FACTOR*DSEP
   GO TO 110
C
C   ON SLOPE 3, BUCKLING AND CONTINUING
C
40 IF (DVAX.GT.0.) GO TO 41
C
C   UPDATE PLASTIC DEFORMATIONS
C
   DVP=FACTOR*DVAX
   VPACN=VPACN+DVP
   GO TO 120
C
C   BUCKLING AND UNLOADING
C
41 XTOT=(VTOT-DVAX)/DELTY
   IF (XTOT.GE.X3RE) GO TO 42
C
C   ESTABLISH NEW STIFFNESS FOR REVERSE
C
   X3RE=XTOT
   IELOG=1
   CALL LAW10
   P5=Y5*PYP
   KODY1=4
   GO TO 50
C
C   USE OLD STIFFNESS FOR REVERSE
C
42 IF (INDRE.EQ.2.AND.YTOT.LE.X6) GO TO 43
   IF ((X5D-XTOT).EQ.0.) GO TO 44
   SLOP5=(Y5D+PFAC)/(X5D-XTOT)

```

```

EAL6=EAL*SLOP6
KODY1=6
GO TO 70
C
43 SLOP7=(Y5+PFAC)/(X5-XTOT)
EAL7=EAL*SLOP7
KODY1=7
GO TO 80
44 KODY1=5
GO TO 60
C
C ON SLOPE 4, GET FACTOR FOR STATUS CHANGE
C
50 DS4=EAL4*DVAX
IF (DS4) 51,110,54
51 IF (SEP) 52,52,53
52 FAC=(PYNC-SEP)/DS4
IF (FAC.GE.FACTOR) GO TO 55
FACTOR=FAC
SEP=PYNC
KODY1=3
GO TO 110
53 X5D=VTOT/DELTY
Y5D=SEP/PYP
P5D=SEP
INDRE=1
KODY1=5
GO TO 60
54 FAC=(P5-SEP)/DS4
IF (FAC.GE.FACTOR) GO TO 55
FACTOR=FAC
SEP=P5
KODY1=8
GO TO 110
55 SEP=SEP+FACTOR*DS4
GO TO 110
C
C ON SLOPE 5, GET FACTOR FOR STATUS CHANGE
C
60 DS5=EAL*DVAX
IF (DS5) 61,110,62
61 FAC=(PYNC-SEP)/DS5
IF (FAC.GE.FACTOR) GO TO 65
FACTOR=FAC
SEP=PYNC
KODY1=3
GO TO 110
62 FAC=(P5D-SEP)/DS5
IF (FAC.GE.FACTOR) GO TO 65
FACTOR=FAC
SEP=P5D
GO TO (63,64),INDEE
63 KODY1=4
GO TO 110
64 KODY1=8

```


GO TO 110
 65 SEP=SEP+FACTOR*DS5
 GO TO 110

C
 C
 C

ON SLOPE 6, GET FACTOR FOR STATUS CHANGE

70 DS6=EAL6*DVAX
 IF (DS6) 71,110,72
 71 FAC=(PYNC-SEP)/DS6
 IF (FAC.GE.FACTOR) GO TO 75
 FACTOR=FAC
 SEP=PYNC
 KODY1=3
 GO TO 110
 72 FAC=(P5D-SEP)/DS6
 IF (FAC.GE.FACTOR) GO TO 75
 FACTOR=FAC
 SEP=P5D
 GO TO (73,74),INDRE
 73 KODY1=4
 GO TO 110
 74 KODY1=8
 GO TO 110
 75 SEP=SEP+FACTOR*DS6
 GO TO 110

C
 C
 C

ON SLOPE 7, GET FACTOR FOR STATUS CHANGE

80 DS7=EAL7*DVAX
 IF (DS7) 81,110,82
 81 FAC=(PYNC-SEP)/DS7
 IF (FAC.GE.FACTOR) GO TO 83
 FACTOR=FAC
 SEP=PYNC
 KODY1=3
 GO TO 110
 82 FAC=(P5-SEP)/DS7
 IF (FAC.GE.FACTOR) GO TO 83
 FACTOR=FAC
 SEP=P5
 KODY1=8
 GO TO 110
 83 SEP=SEP+FACTOR*DS7
 GO TO 110

C
 C
 C

ON SLOPE 8, GET FACTOR FOR STATUS CHANGE

90 DS8=EAL8*DVAX
 IF (DS8) 91,110,92
 91 X5D=VIOT/DELTY
 Y5D=SEP/PYP
 P5D=SEP
 INDRE=2
 KODY1=5
 GO TO 60

```

C
  92 FAC=(PYP-SEP)/DS8
    IF (FAC.GE.FACTOR) GO TO 93
    FACTOR=FAC
    SEP=PYP
    KODY1=9
    GO TO 110
  93 SEP=SEP+FACTOR*DS8
    GO TO 110

C
  ON SLOPE 9, YIELDED BUT CONTINUING
C
  100 IF (DVAX.LI.0.) GO TO 101
C
  UPDATE PLASTIC DEFORMATIONS
C
  DVP=FACTOR*DVAX
  XPP=XPP+DVP/DELTY
  VPACP=VPACP+DVP
  GO TO 120

C
  YIELDED BUT UNLOADING
C
  101 KODY1=0
C
  RESIDUAL ELONGATION, RE
C
  IF (IELOG.NE.1) GO TO 105
  SLEND=60./RATIO
  RE=0.0175*(0.55*X3RE/SLEND+0.0002*X3RE**2)
  RE=RE*FL/DELTY
  XPP=XPP+RE
  IELOG=0

C
  105 X3RE=XPP-PFAC
    GO TO 30

C
  CHECK FOR COMPLETION OF CYCLE
C
  110 FACAC=FACAC+FACTOR
    IF (FACAC.LT.0.9999999) GO TO 20
C
  NEW FORCE, UNBALANCED FORCE DUE TO YIELD
C
  120 FTOT(2)=SEP
    FTOT(1)=-SEP
    F0UB=SLIN-SEP
    IF (ABS(F0UB).GT.1.E-8) KBAL1=1

C
  ACCUMULATE EXTENSION ENVELOPES
C
  IF (VENP.GE.VTOT) GO TO 180
  VENP=VTOT
  IVENP=TIME
  GO TO 190

```

```

180 IF (VENN.LE.VTOT) GO TO 190
    VENN=VTOT
    TVENN=TIME
190 CONTINUE

C
C   SET INDICATOR FOR AXIAL STIFFNESS CHANGE
C
    KST1=0
    IF (KODYX1.NE.KODY1) KST1=1

C
C   FLEXURAL STATE DETERMINATION
C
C   LINEAR MOMENT INCREMENTS
C
    CALL BM10
    BML(1)=BMEP(1)+DBM(1)
    BML(2)=BMEP(2)+DBM(2)
    BMEL(1)=BMTOT(1)-BMEP(1)
    BMEL(2)=BMTOT(2)-BMEP(2)

C
C   TRACE OUT NONLINEAR PATH ON M-P SURFACE
C
    FACAC=0.
    KBAL=0
250 FACTOR=1.-FACAC

C
C   PLASTIC HINGE ROTATIONS
C
    IF (KODY(1)+KODY(2)-1) 280,260,270
260 DPR(1)=DVR(1)+PR21*DVR(2)
    DPR(2)=DVR(2)+PR12*DVR(1)
    GO TO 280
270 DPR(1)=DVR(1)
    DPR(2)=DVR(2)

C
280 KPAC=0
    DO 320 IEND=1,2

C
C   ELASTIC, GET FACTOR FOR STATUS CHANGE
C
    IF (KODY(IEND).NE.0) GO TO 310
    IF (DBM(IEND)) 290,320,300
290 FAC=(BMY(IEND,2)-BMEP(IEND))/DBM(IEND)
    IF (FAC.GE.FACTOR) GO TO 320
    FACTOR=FAC
    BMY=BMY(IEND,2)
    KPAC=IEND
    GO TO 320
300 FAC=(BMY(IEND,1)-BMEP(IEND))/DBM(IEND)
    IF (FAC.GE.FACTOR) GO TO 320
    FACTOR=FAC
    BMY=BMY(IEND,1)
    KPAC=IEND
    GO TO 320

C

```

```

C     CONTINUING TO YIELD - POSITIVE ELASTIC WORK
C
310  IF (BM EP(IEND)*DPP(IEND).GE.0.) GO TO 320
C
C     UNLOADING - PLASTIC WORK NEGATIVE
C
      FACTOR=0.
      KFAC=0
      KODY(IEND)=0
C
320  CONTINUE
C
C     UPDATE MOMENTS AND HINGE ROTATIONS
C
      DO 360 IEND=1,2
      IF (IEND.EQ.KFAC) GO TO 350
      IF (KODY(IEND).NE.0) GO TO 330
      BM EP(IEND)=BM EP(IEND)+FACTOR*DBM(IEND)
      GO TO 360
330  DPPR=FACTOR*DPR(IEND)
      PRTOT(IEND)=PRTOT(IEND)+DPPR
      IF (DPPR.LT.0.) GO TO 340
      PRACP(IEND)=PRACP(IEND)+DPPR
      GO TO 360
340  PRACN(IEND)=PRACN(IEND)+DPPR
      GO TO 360
350  BM EP(IEND)=BMY
      KODY(IEND)=1
360  CONTINUE
C
C     CHECK COMPLETION OF CYCLE
C
      FACAC=FACAC+FACTOR
      IF (FACAC.GT.0.99999) GO TO 370
      CALL BM13
      KBAL=1
      GO TO 250
C
C     YIELD MOMENTS FOR NEXT STEP
C
370  CALL YMOM10
C
C     CHECK FOR OVERSHOOT OF M-F SURFACE
C
      DO 420 IEND=1,2
      IF (KODY(IEND).EQ.0) GO TO 400
      IF (BM EP(IEND).LE.BMY(IEND,1)) GO TO 380
      BM EP(IEND)=BMY(IEND,1)
      KBAL=1
      GO TO 420
380  IF (BM EP(IEND).GE.BMY(IEND,2)) GO TO 390
      BM EP(IEND)=BMY(IEND,2)
      KBAL=1
      GO TO 420
390  IF (BM EP(IEND).LT.BMY(IEND,1)*0.98.AND.BM EP(IEND).GT.BMY(IEND,2)*0

```

```

1.98) KODY(IEND)=0
GO TO 420
400 IF (BMEP(IEND).LE.BMY(IEND,1)) GO TO 410
BMEP(IEND)=BMY(IEND,1)
KBAL=1
KODY(IEND)=1
GO TO 420
410 IF (BMEP(IEND).GE.BMY(IEND,2)) GO TO 420
BMEP(IEND)=BMY(IEND,2)
KBAL=1
KODY(IEND)=1
420 CONTINUE
C
C ELASTIC AND TOTAL FORCES
C
BBMTOT(1)=BMTOT(1)
BBMTOT(2)=BMTOT(2)
BMTOT(1)=BMEP(1)+BMEL(1)+(EK11*DVR(1)+EK12*DVR(2))*PSH
BMTOT(2)=BMEP(2)+BMEL(2)+(EK12*DVR(1)+EK22*DVR(2))*PSH
DSF=(BMTOT(1)-BBMTOT(1)+BMTOT(2)-BBMTOT(2))/FL
SFTOT(1)=SFTOT(1)+DSF
SFTOT(2)=SFTOT(2)-DSF
C
C UNBALANCED LOADS DUE TO YIELD
C
BMIUB=0.
BMJUB=0.
IF (KBAL.EQ.0) GO TO 430
BMIUB=BML(1)-BMEP(1)
BMJUB=BML(2)-BMEP(2)
C
C DEFORMATION RATES FOR DAMPING
C
430 IF (DFAC.EQ.0.0.AND.DELTA.EQ.0.0) GO TO 460
IF (TIME.EQ.0.) GO TO 470
KBAL=1
DVAX=COSA*(VELM(4)-VELM(1))+SINA*(VELM(5)-VELM(2))
ROT=(SINA*(VELM(4)-VELM(1))+COSA*(VELM(2)-VELM(5)))/FL
DVR(1)=VELM(3)+ROT
DVR(2)=VELM(6)+ROT
C
C BETA-D DAMPING
C
IF (DFAC.EQ.0.) GO TO 450
FAC=DFAC*(1.+PSH)
BMIUB=BMIUB+(EK11*DVR(1)+EK12*DVR(2))*FAC
BMJUB=BMJUB+(EK12*DVR(1)+EK22*DVR(2))*FAC
FOUB=EAL*DVAX*DFAC+FOUB
C
C STRUCTURAL DAMPING LOAD
C
450 IF (DELTA.EQ.0.) GO TO 460
SDMI=DELTA*ABS(BMTOT(1))*SIGN(1.,DVR(1))
SDMJ=DELTA*ABS(BMTOT(2))*SIGN(1.,DVR(2))
SDFO=DELTA*ABS((FTOT(2)-FTCT(1))/2.)*SIGN(1.,DVAX)

```

```

      BMIUB=BMIJB-SDMI+SDACT(1)
      BMJUB=BMJUB-SDMJ+SDACT(2)
      FOUB=FOUB-SDFO+SDACT(3)
      SDACT(1)=SDMI
      SDACT(2)=SDMJ
      SDACT(3)=SDFO
C
C      SET UP UNBALANCED LOAD VECTOR
C
460 IF (KBAL+KBAL1.EQ.0) GO TO 470
      KBAL=1
      SFUB=(BMIUB+BMJUB)/FL
      DD(1)=-SFUB*SINA-FOUB*COSA
      DD(2)=SFUB*COSA-FOUB*SINA
      DD(3)=BMIUB
      DD(4)=-DD(1)
      DD(5)=-DD(2)
      DD(6)=BMJUB
C
C      EXTRACT ENVELOPES
C
470 DO 490 I=1,8
      S=BMTOT(I)
      IF (S.LF.SENP(I)) GO TO 480
      SENP(I)=S
      TENP(I)=TIME
480 IF (S.GE.SENN(I)) GO TO 490
      SENN(I)=S
      TENN(I)=TIME
490 CONTINUE
C
C      PRINT TIME HISTORY
C
      IF (KPR.LT.0) GO TO 500
      IF (KPR.EQ.0.OR.KOUTDT.EQ.0) GO TO 540
500 IF (IHED.NE.0) GO TO 520
      KKPR=IABS(KPR)
      PRINT 510, KKPR, TIME
510 FORMAT (///18H RESULTS FOR GROUP,I3,
1          38H, END MOMENT-BUCKLING ELEMENTS, TIME =,F8.3///5X,
2          5H ELEM,4X,4HNODF,2X,10HPLX. YIELD,2X,7HBENDING,7X,5HSHEAR,
3          7X,5HAXIAL,12X,23HELASTIC HINGE ROTATIONS,
4          15X,5HAXIAL/5X,
5          5H NO.,4X,4H NO.,3X,5H CODE,6X,7H MOMENT,7X,5H FORCE,
6          7X,5HFORCF,8X,7HCURRENT,4X,9HACC. POS.,3X,9HACC. NEG.,
7          3X,10HYIELD CODE,3X,9HEXTENSION/)
      IHED=1
520 PRINT 530, IMEM, (NOD(I), KODY(I), BMTOT(I), SFTOT(I), FTOT(I), PRTOT(I)
1, PRACN(I), PRACN(I), I=1,2), KODY1, VTOT
530 FORMAT (I9, I8, I7, 3X, 3F12.2, 3X, 3F12.5/
1          9X, I8, I7, 3X, 3F12.2, 3X, 3F12.5, I10, F11.5)
C
C      SET INDICATOR FOR STIFFNESS CHANGE
C
540 KST=0

```

```
IF (KODYX(1).NE.KODY(1).OR.KODYX(2).NE.KODY(2)) KST=1  
IF (KST.EQ.0.AND.KST1.NE.0) KST=1
```

C

C

```
UPDATE INFORMATION IN CCMS
```

C

```
DO 550 J=32,88
```

```
550 COMS(J)=COM(J)
```

```
COMS(2)=COM(2)
```

```
DO 560 J=141,167
```

```
560 COMS(J)=COM(J)
```

C

```
RETURN
```

```
END
```

```

SUBROUTINE OUT10 (/COMS/,/NINFC/)
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,
1 EK22,EK12,ESH,EAL,EK11H,EK22H,KODYX(2),
2 KODY(2),BMTCT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3 SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4 SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5 PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6 PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3PE,EAL1,EAL4,
7 EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8 IVRTX,IEICG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9 TVENN,PYP,PYN,PYNC,REST(30)

C
DIMENSION COM(1),COMS(1)
EQUIVALENCE (IMEM,COM(1))

C
ENVELOPE OUTPUT, END MOMENT-BUCKLING ELEMENTS

C
DO 10 J=1,NINFC
10 COM(J)=COMS(J)

C
IF (IMEM.EQ.1) PRINT 20
20 FORMAT (39H END MOMENT-BUCKLING ELEMENTS (TYPE 10)///
1 5H ELEM,3X,4HNODE,17X,7HBENDING,14X,5HSHEAR,14X,5HAXIAL,
2 13X,8HPL HINGE,7Y,14H MAX. AXIAL/
3 5H NO.,3X,4H NO.,17X,7H MOMENT,3X,4HTIME,7X,5HFORCE,3X,
4 4HTIME,7X,5HFORCE,3X,4HTIME,6X,8HROTATION,3X,4HTIME,
5 5X,9HEXTENSION,3X,4HTIME/)

C
PRINT 30, IMEM,NODI,(SENP(I),TENP(I),I=1,7,2),VENP,TVENP,(SENN(I),
1IENN(I),I=1,7,2),VENN,TVENN,NODJ,(SENP(I),TENP(I),I=2,8,2),(SENN(I)
2),TENN(I),I=2,8,2)
30 FORMAT (I4,I7,5X,8HPOSITIVE,3(F12.2,F7.3),2(F14.5,F7.3)/
1 16X,8HNEGATIVE,3(F12.2,F7.3),2(F14.5,F7.3)/
2 7X,I4,5X,8HPOSITIVE,3(F12.2,F7.3),F14.5,F7.3/
3 15X,8HNEGATIVE,3(F12.2,F7.3),F14.5,F7.3/)

C
RETURN
END

```



```

SUBROUTINE FST10A (/STIF/,/KOD/)
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1      EK22,EK12,PSH,EAL,EK11H,EK22H,      KODYX(2),
2      KODY(2),BMTOT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3      SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4      SDACT(3),BMY(2,2),NODI,MODJ,KOUTDT,PR12,PR21,
5      PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6      PFAC,RATIO,DELT, KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7      EAL6,EAL7,EAL8,POUT(51)
C
FORM AXIAL STIFFNESS
C
      KYY=KOD+1
      GO TO (10,20,30,40,50,60,70,80,90,100),KYY
10 STIF=EAL
   GO TO 110
20 STIF=EAL1
   GO TO 110
30 STIF=EAL
   GO TO 110
40 STIF=.001*EAL
   GO TO 110
50 STIF=EAL4
   GO TO 110
60 STIF=EAL
   GO TO 110
70 STIF=EAL6
   GO TO 110
80 STIF=EAL7
   GO TO 110
90 STIF=EAL8
   GO TO 110
100 STIF=.001*EAL
C
110 RETURN
    END

```

SJBROUTINE LAW10

```

C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1      EK22,EK12,PSH,EAL,EK11H,EK22H,      KODYX(2),
2      KODY(2),BMTOT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3      SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4      SDACT(3),EMY(2,2),NODI,NODJ,KOBTDT,PR12,PR21,
5      PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6      PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7      EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8      REST(44)

C
C      GENERATE P-DELTA HYSTERESIS CURVE
C
C      RESIDUAL ELONGATION, RE
C
      SLEND=60./RATIO
      RE=0.0175*(0.55*X3RE/SLEND+0.0002*X3RE**2)
      RE=RE*FL/DELTY

C
      XPP=XPP+RE
      XPP1=1.+XPP
      BETA=1./3.
      DENOM=(XPP1-X3RE)+(1.+PFAC)/BETA
      XNUMR=XPP-X3RE-PFAC
      Y5=RATIO*XNUMR/DENOM
      X5=XPP-Y5/BETA
      SLOP4=(Y5+PFAC)/(X5-X3RE)
      EAL4=EAL*SLOP4
      SLOP8=(1.-Y5)/(XPP1-X5)
      EAL8=EAL*SLOP8
      X6=X5-Y5-PFAC
      XPP=XPP-RE

C
      RETURN
      END

```

```

SUBROUTINE VRTX10 (/FACAC/,/DVAX/,/SEP/,/TIME/)
C
C   AXIAL STATE DETERMINATION IN TENSION AND VERTEX REGIONS
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,
1   EK22,EK12,FSH,EAL,EK11H,EK22H,      KODYX(2),
2   KODY(2),BMTOT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3   SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4   SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5   PMX(3,2,2),A1(4,2,2),A2(4,2,2),
6   PFAC,RATIO,DELTY,KODYX1,KODY1,XPP,X3RE,EAL1,EAL4,
7   EAL6,EAL7,EAL8,X5,Y5,P5,X6,X5D,Y5D,P5D,INDRE,
8   IVP TX,IELOG,VTOT,XTOT,VPACP,VPACN,VENP,TVENP,VENN,
9   TVENN,PYP,PYN,PYNC,REST(30)
C
C   INITIALIZE
C
FACAC=0.
10 FACTOR=1.-FACAC
KODYI=KODY1+1
IF (KODYI.EQ.10) KODYI=4
GO TO (20,30,40,50),KODYI
C
C   ON SLOPE 0, ELASTIC, GET FACTOR FOR STATUS CHANGE
C
20 DSEP=EAL*DVAX
IF (DSEP) 21,60,22
21 FAC=(PYN-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 23
FACTOR=FAC
SEP=PYN
KODY1=1
C
SENP(5)=-PYN
SENN(6)=PYN
TENP(5)=TIME
TENN(6)=TIME
GO TO 60
C
22 FAC=(PYP-SEP)/DSEP
IF (FAC.GE.FACTOR) GO TO 23
FACTOR=FAC
SEP=PYP
KODY1=9
GO TO 60
C
23 SEP=SEP+FACTOR*DSFP
GO TO 60
C
C   ON SLOPE 1, GET FACTOR FOR STATUS CHANGE
C
30 DS1=EAL1*DVAX
IF (EAL1.NE.0) GO TO 34
KODY1=3
IVRTX=1

```

```

      GO TO 80
C
34 IF (DS1) 31,60,32
C
      BUCKLED BUT LOADING
C
31 KODY1=2
      GO TO 40
C
      BUCKLING AND CONTINUING
C
32 FAC=(PYN-C-SEP)/DS1
      IF (FAC.GE.FACTOR) GO TO 33
      FACTOR=FAC
      SEP=PYN-C
      KODY1=3
C
      UPDATE PLASTIC DEFORMATION
C
      VPACN=VPACN+FACTOR*DVAX
      GO TO 70
C
33 SEP=SEP+FACTOR*DS1
C
      UPDATE PLASTIC DEFORMATION
C
      VPACN=VPACN+FACTOR*DVAX
      GO TO 60
C
      ON SLOPE 2, GET FACTOR FOR STATUS CHANGE
C
40 DS2=EAL*DVAX
      IF (DS2) 41,60,42
C
      BUCKLING AND CONTINUING
C
41 KODY1=1
      GO TO 30
C
      BUCKLED BUT LOADING
C
42 FAC=(PYN-C-SEP)/DS2
      IF (FAC.GE.FACTOR) GO TO 43
      FACTOR=FAC
      SEP=PYN-C
      KODY1=3
      GO TO 70
C
43 SEP=SEP+FACTOR*DS2
      GO TO 60
C
      ON SLOPE 9, TENSION YIELDING
C
50 IF (DVAX.LT.0) GO TO 51
      DVP=FACTOR*DVAX

```

```

XPP=XPP+DVP/DELTY
VPACP=VPACP+DVP
GO TO 110
C
C   YIELDED BUT UNLOADING
C
51 KDDY1=0
   X3RE=XPP-PFAC
   GO TO 20
C
60 FACAC=FACAC+FACTOR
   IF (FACAC.LT.0.9999999) GO TO 10
   RETURN
C
70 FACAC=FACAC+FACTOR
   IVRTX=1
C
80 PHI=PYNC/PYN
C
C   MODIFY THE SLOPES OF 4 SEGMENTS ON M-P CURVE IN COMPRESSION
C
DO 90 K=1,2
DO 90 J=1,2
DO 90 I=1,2
90 A1(I,J,K)=A1(I,J,K)/PHI
C
DO 100 K=1,2
DO 100 J=1,2
100 PMX(1,J,K)=PHI*PMX(1,J,K)
C
110 RETURN
    END

```

```

      SJBROUTINE FST1QB (/ST//KCD/)
C
C     FORM 2*2 FLEXURAL STIFFNESS
C
      COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,5),EK11,
1      EK22,EK12,FSH,EAL,EK11H,EK22H,REST(169)
      DIMENSION ST(2,2),KOD(2)
C
      KYY=KOD(1)+2*KOD(2)+1
      GO TO (10,20,30,40),KYY
10  ST(1,1)=EK11
      ST(2,2)=EK22
      ST(1,2)=EK12
      GO TO 60
20  ST(1,1)=0.
      ST(2,2)=EK22H
      GO TO 50
30  ST(1,1)=EK11H
      ST(2,2)=0.
      GO TO 50
40  ST(1,1)=0.
      ST(2,2)=0.
50  ST(1,2)=0.
60  ST(2,1)=ST(1,2)
C
C     RETURN
      END

```

```

SUBROUTINE BM10
C
C   END MOMENT INCREMENT
C
COMMON /INFEL/ IMEM, KST, LM(6), KGEOM, FL, COSA, SINA, A(2,5), EK11,
1          EK22, EK12, PSH, EAL, EK11H, EK22H,      KODYX(2),
2          KODY(2), PEST(165)
COMMON /WORK/ DVR(2), DPR(2), DBM(2), W(1994)
C
      KYX=KODY(1)+2*KODY(2)+1
      GO TO (10,20,30,40), KYX
10  DBM(1)=EK11*DVR(1)+EK12*DVR(2)
     DBM(2)=EK12*DVR(1)+EK22*DVR(2)
     GO TO 50
20  DBM(1)=0.
     DBM(2)=EK22H*DVR(2)
     GO TO 50
30  DBM(1)=EK11H*DVR(1)
     DBM(2)=0.
     GO TO 50
40  DBM(1)=0.
     DBM(2)=0.
C
C
50  RETURN
     END

```

```

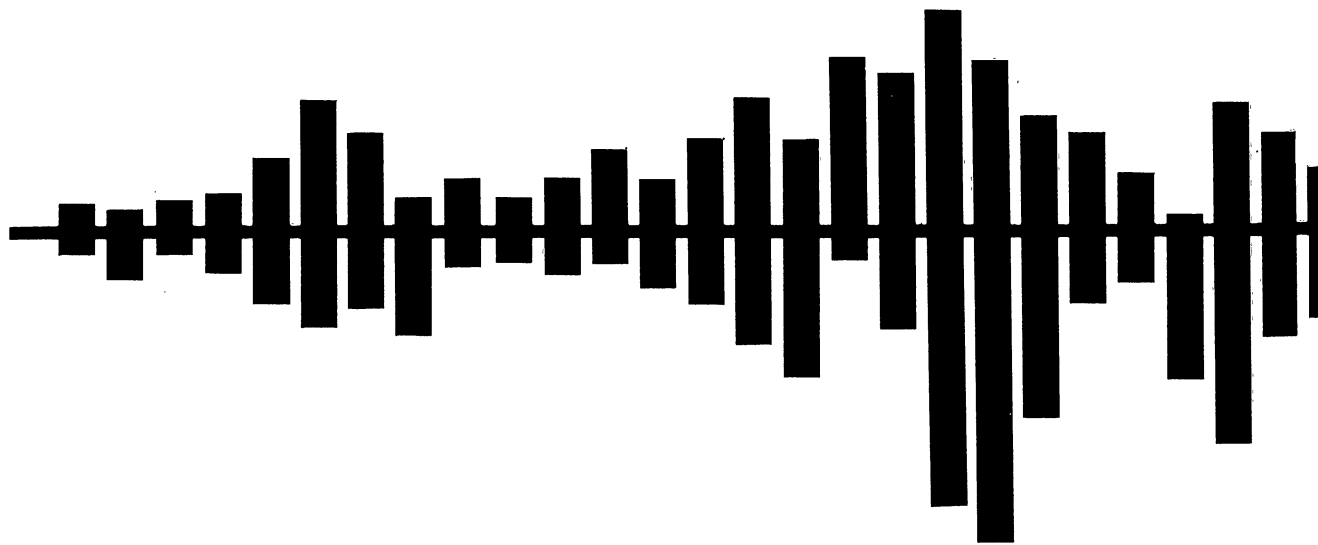
C      SJBROUTINE YMOM10
C
C      YIELD MOMENTS FOR CURRENT AXIAL FORCE
C
COMMON /INFEL/ IMEM,KST,LM(6),KGEOM,FL,COSA,SINA,A(2,6),EK11,
1          EK22,EK12,PSH,EAL,EK11H,EK22H,          KODYX(2),
2          KODY(2),EMTCT(2),SFTOT(2),FTOT(2),PRTOT(2),SENP(8),
3          SENN(8),TENP(8),TENN(8),PRACP(2),PRACN(2),BMEP(2),
4          SDACT(3),BMY(2,2),NODI,NODJ,KOUTDT,PR12,PR21,
5          PMX(3,2,2),A1(4,2,2),A2(4,2,2),POUT(63)
C
      FACC=-1.
      DO 30 IEND=1,2
      FFT=FTOT(IEND)*FACC
      FACC=1.
      FAC=1.
      DO 30 J=1,2
      FAC=-FAC
      DO 10 I=1,3
      IF (FFT.LT.PMX(I,J,IEND)) GO TO 20
10  CONTINUE
      I=4
20  BMY=A2(I,J,IEND)-A1(I,J,IEND)*FFT
      IF (FAC*BMY.GT.0.) BMY=0.
30  BMY(IEND,J)=BMY
      FAC=-BMY(1,1)
      BMY(1,1)=-BMY(1,2)
      BMY(1,2)=FAC
C
      RETURN
      END

```


UNIVERSITY OF MICHIGAN



3 9015 09400 8078



AIIM SCANNER TEST CHART # 2

Spectra

4 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 6 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 8 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 10 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789

Times Roman

4 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 6 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 8 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 10 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789

Century Schoolbook Bold

4 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 6 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 8 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 10 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789

News Gothic Bold Reversed

4 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 6 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 8 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 10 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789

Bodoni Italic

4 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 6 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 8 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789
 10 PT ABCDEFGHIJKLMNOPQRSTUVWXYZabcdefghijklmnopqrstuvwxyz;"/?0123456789

Greek and Math Symbols

4 PT ΑΒΓΔΕΕΘΗΙΚΑΜΝΟΠΦΡΣΤΥΩΧΨΖαβγδεξθηικλμνοπφρστνωχψζ≥≠",./≤±=≠' > < > < > < ≡
 6 PT ΑΒΓΔΕΕΘΗΙΚΑΜΝΟΠΦΡΣΤΥΩΧΨΖαβγδεξθηικλμνοπφρστνωχψζ≥≠",./≤±=≠' > < > < > < ≡
 8 PT ΑΒΓΔΕΕΘΗΙΚΑΜΝΟΠΦΡΣΤΥΩΧΨΖαβγδεξθηικλμνοπφρστνωχψζ≥≠",./≤±=≠' > < > < > < ≡
 10 PT ΑΒΓΔΕΕΘΗΙΚΑΜΝΟΠΦΡΣΤΥΩΧΨΖαβγδεξθηικλμνοπφρστνωχψζ≥≠",./≤±=≠' > < > < > < ≡

White



Black



Isolated Characters

e	m	1	2	3	a
4	5	6	7	o	-
8	9	0	h	l	B

MESH HALFTONE WEDGES

65

85

100

110

133

150

