Improving the wind-induced human comfort of the Beijing Olympic Tower by a double-stage pendulum tuned mass damper

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Abstract: With five sub towers and a maximum height of 246.8 m, the Beijing Olympic Tower (BOT) is a landmark of Beijing. The complex structural properties and slenderness of the BOT render it prone to wind loading. As far as the wind-induced performance of this structure is concerned, this paper thus aims at a tuned mass damper-based mitigation system for controlling the wind-induced acceleration response of the BOT. To this end, the three-dimensional wind loading of various wind directions are simulated based on the fluctuating wind force obtained by the wind tunnel test, by which the wind-induced vibration is evaluated in the time domain by using the finite element model (FEM). A double-stage pendulum tuned mass pamper (DPTMD), which is capable of controlling the long period dynamic response and requires only a limited space of installation, is optimally designed at the upper part of the tower. Finally, the wind-induced response of the structure with and without DPTMD is compared with respect to various wind directions and in both the time

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and frequency domain. The comparative results show that the wind-induced accelerations atop the tower with the wind directions of 45°, 135°, 225°, and 315° are larger than those with the other directions. The DPTMD significantly reduces the wind-induced response by the maximum acceleration reduction ratio of 30.05%. Moreover, it is revealed that the control effect varies noticeably for the five sub towers, depending on the connection rigidity between Tower1 and each sub tower.

Keyword: Multi-tower structure; Wind-induced responses; Human comfort; Double-stage pendulum tuned mass damper; Vibration mitigation; High-rise building

1. Introduction

With the increasing use of high-strength materials, light-weight floors, and curtain wall systems, the mass and stiffness of modern high-rise buildings and towers are decreasing (Ghorbani-Tanha et al. 2009; Kim et al. 2008). Thus, they suffer from excessive wind-induced oscillations (Chen et al. 2018; Li et al. 2011). The undesirably excessive vibration may cause not only the structural damage or failure but also the discomfort of occupants. Hence, very often those structures may need specific measures to mitigate the wind-induced responses (Kareem et al. 1999). Generally, increasing the structural stiffness is effective in reducing dynamic response, but it is often economically inefficient. Structural control technology, which aims to enhance structural safety and serviceability against dynamic excitations, is now a widely-used alternative (Basu et al. 2014; Spencer&Nagarajaiah 2003; Housner et al. 1997; Soong&Spencer 2002; Ikeda 2009).

Tuned mass damper (TMD) is one of the simplest and practical control devices and especially
 useful in mitigating dynamic response within the narrow frequency range. Generally, a typical TMD

The frequency of the TMD is tuned to a particular vibration frequency so that when the structural vibration at that frequency is excited, the damper will resonate out of phase and mitigate the induced vibration. A portion of vibration energy input to the structure may be imparted to the TMD and dissipated by the damping element. Due to the convenient implementation procedure, low cost and no requirement of external energy, TMD has attracted many researches, most of which are focused on the configuration, design, and application of the TMD. A proper configuration is essential to realize the TMD in engineering practice. Options of the effective stiffness may include the springs (Ohtake et al. 1992; Chen et al. 2012), the suspended pendulum (Roffel et al. 2013; Nagase&Hisatoku 1992), and the laminated rubber bearings (Saito et al. 2001). Although the oil dampers are typical, there are still others such as the friction (Chung et al. 2013), the eddy current (Lu et al. 2018), the sub tuned absorbers (Sarkar&Gudmestad 2013), and the pounding (Song et al. 2016; Zhang et al. 2013). Moreover, the analysis and design methods of TMD are under active investigation, including the design formulas of the TMD's parameters. These researches may date back to (Den Hartog 1956), such as simplified expressions for optimum TMD parameters of undamped/damped systems considering various combinations of responses (e.g., displacement, velocity, acceleration and force) and excitations (e.g., harmonic and white noise random excitations) (Chang&Qu 1998; Sadek et al. 1997; Warburton 1982; Chung et al. 2013; Hoang et al. 2008; Fujino&Abe 1993). On the other hand, the metaheuristic optimization methods have been utilized in the design of TMD (Farshidianfar&Soheili 2013; Leung et al. 2008; Bekdaş&Nigdeli 2011; Bekdaş et al. 2018; Carlo Marano et al. 2010; Jin et al. 2018).

can be modeled as a device consisting of a mass element, a stiffness element, and a damping element.

With the development in the past four decades, the TMD has been successfully installed on

practical buildings to mitigate dynamic responses including pedestrian-induced vibration (An et al. 2015; Chen et al. 2012), wind-induced vibration (Ghorbani-Tanha et al. 2009; Nagase&Hisatoku 1992) and earthquake-induced vibration (Chen et al. 2018; Hoang et al. 2008). Researches and applications show that the TMD in the high-rise tower can suppress the wind-induced vibration significantly, such as John Hancock Tower in Boston (1975), Taipei 101 Tower (Kao et al. 2011), Shanghai Tower (Zhou et al. 2018), and many others (Ghorbani-Tanha et al. 2009; Ohtake et al. 1992; Kang et al. 2012; Li et al. 2011; McNamara 1977; Nagase&Hisatoku 1992). Most of these cases are a TMD on a single-tower building, but most recently some complicated buildings with two or more towers have been constructed out of their architecture benefits. However, the researches of TMD on these irregular buildings are rare. Hence, further investigations on the wind-induced vibration and mitigation are still needed for multi-tower buildings.

With a height of 246.8m, Beijing Olympic Tower (BOT) in the northern of Beijing, China, is a new landmark. The BOT consists of five towers, each of which has an atop large mass. Preliminary studies demonstrate that the upper part of the tower suffers from excessive wind-induced acceleration beyond the human comfortability limit. This paper presents a study on the TMD installed on the BOT. The remainder of this paper is organized as follows. Section 2 introduces the structural system of the BOT. Section 3 presents the finite element model (FEM) of the structure and discusses its dynamic characteristics. The following section introduces the results of the wind tunnel test for this structure and the simulation of the wind loading in the time domain. Section 5 designs a double-stage pendulum tuned mass damper (DPTMD) for this tower and assesses the mitigation effect of the tower by comparing the acceleration response with and without the DPTMD. Finally, the main conclusions of the numerical study are summarized in Section 6.

81 **2. Description of the Beijing Olympic Tower**

82 Beijing Olympic Tower (BOT), shown in Fig.1, is adjacent to the landscape avenue on the axis 83 line of the Beijing Olympic Forest Park. It boasts one of the two buildings around the world officially with Olympic rings permanently on the top (BeijingInternational 2016; OlympicNews -84 85 2016). This tower consists of five sub towers with different heights: one central tower (Tower1) 86 locates in the middle and four lower towers (Tower2, Tower3, Tower4, and Tower5) stands around 87 the Tower1. The geometry of these sub towers is shown in Table.1. Towers 2-5 are connected to 88 Tower1 by the corridors which are spatial steel girders with a height of 3.0m and a width of 2.7m. 89 The sections of these sub towers are regular hexagon or circular, the dimension of which increases 90 with the height. The structure of Tower1 is a tube-in-tube system; its outer tube is composed of 16 concrete-filled steel tube (CFST) circular columns stiffened by I-shape steel beams and H-shape steel 91 braces. Towers 2-5 has the structure of tube systems composed of 6 CFST circular columns 92 93 connected by steel beams and braces.

94 **3. Dynamic characteristics of the structure**

95 *3.1 Finite element model*

As shown in Fig.2, a three-dimensional (3D) finite element model (FEM) of the BOT was established by using CSI SAP2000 based on the structural design drawings. Two kinds of elements were employed in the FEM: the beam elements for the beams, columns, and braces, and the shell elements for the floors and walls. The nonstructural components were modeled as the dead loads, while the mass was transformed from the dead loads and live loads. The connections between the braces and other components were hinged constraints, and the connections between the beams and columns and between the structure and its foundation were fixed constraints.

103 *3.2 Natural frequencies and modal shapes*

104 The structural dynamic characteristics of the BOT were analyzed and shown in Table.2 and Fig.3. Here, 45° and 135° denotes the angles between the vibration directions and the x-axis in the 105 106 x-y plane. It can be observed that the first natural period can be as long as 5.687s and the 107 torsional-translation period ratio is 0.775. The first two modes of the tower are overall bending in the 108 direction of 45° and 135°, respectively, where the third mode is torsional. The first six modes are 109 overall vibration modes, indicating that the corridors in this tower are stiff enough to combine the five sub towers. The modes from the sixth to ninth are all 2nd order torsional modes with different 110 111 vibration shapes of the sub towers with dense-distributed frequencies, which may influence the 112 vibration of the tower.

113 **4. Wind loads simulation**

114

4.1 Introduction of the wind tunnel test

115 Wind tunnel testing is an effective tool to evaluate the wind effects on buildings, especially 116 those with complex silhouettes. In this study, wind tunnel tests of wind pressure on the BOT were 117 carried out in the atmospheric boundary layer wind tunnel TJ-2 at the Tongji University using a 118 synchronous multi-pressure sensing system (Ming et al. 2010). The dimension of the working section 119 of the wind tunnel is 3.0m wide ×2.5m high. In order to reproduce the building shape more detailed, 120 the rigid sectional models are used. The whole building is divided into four sections vertically. The 121 sectional models are made of plexiglass and ABS. These four scaled segments are shown in Fig.4(a), 122 and the tower bodies and the tower crowns are 1:70 and 1:100 scaled, respectively. The wind 123 direction is defined as an angle from the tower south along a clockwise direction (see Fig.4(b)), and 124 the measurements were made with the wind direction varying from 0° to 360° with the increment of

125 15°. 1944 pressure taps were installed on these four sectional models. Due to the lack of instruments, 126 the scaled are divided into eight synchronous pressure measuring segments, and not all of them use 127 the same sampling rate. Hence, the pressure data were acquired at five sampling rates with a 128 sampling length of 6000 steps. The sectional model testing was conducted in a turbulent wind field 129 with the turbulence intensity in Table.3, but the mean wind speed was 12.0 m/s and constant along 130 the height. In this experiment, significant mutual aerodynamic interference effects among the sub 131 towers existed, and complicated wind pressure and force distributions were found. Therefore, the 132 wind loads for the BOT is undoubtedly different from the conventional high-rise towers.

133 *4.2 Simulation of the three-dimensional wind loads*

134 In this paper, because of the nonlinearity of the damping devices, the time-domain method is introduced to compute the wind-induced vibration of the BOT. To this end, one should simulate a 135 136 three-dimensional wind loading field that is applicable to the FEM of the BOT firstly. In this study, 137 the fluctuating wind pressure coefficients given by the wind tunnel tests are used to model the 138 three-dimensional wind field. However, there are some difficulties to utilize these data directly: (1) 139 As mentioned above, the data were acquired at five different sampling rates, including 6.0391Hz, 140 8.1145 Hz, 7.3379 Hz, 6.8065 Hz, and 6.2723 Hz. (2) The directions of the tested wind pressure 141 coefficients are perpendicular to the model surfaces. (3) There are 1944 pressure taps and 6000 steps 142 per testing wind direction, and locations of the pressure taps do not strictly one-to-one coincide with 143 the structural nodes in the FEM. These difficulties may lead to unsynchronized time steps, difficulty 144 in the loads' assignment and extensive computing time in the FEM time history analysis. Therefore, a 145 method for the three-dimensional wind loading field simulation is proposed as depicted by Fig.5 146 including four steps:

147 (1) In order to unify the sampling rates of all data into 6.0391Hz, the nearest interpolation
148 method is used. Fig.6 shows the comparison of the data from a pressure tap with and without
149 interpolation and illustrates a good agreement not only in the time domain but also in the frequency
150 domain.

151 (2) The wind pressure coefficients are decomposed into three orthogonal directions based on the 152 global coordinate system of the FEM. Hence, the wind pressure coefficients in the x, y, and 153 *z*-direction of global coordinate can be obtained as follows:

$$\mu_{si,x}(t) = \mu_{si}(t) \cdot \cos \varphi \cdot \sin \theta \tag{1}$$

$$\mu_{si,y}(t) = \mu_{si}(t) \cdot \cos \varphi \cdot \cos \theta \tag{2}$$

$$\iota_{si,z}(t) = \mu_{si}(t) \cdot \sin \varphi \tag{3}$$

where μ_{si} is the wind pressure coefficient of the *i*th pressure tap; $\mu_{si,x}$, $\mu_{si,y}$ and $\mu_{si,z}$ are the decomposed wind pressure coefficients in the *x*, *y*, and *z*-direction of the *i*th pressure tap; φ and θ , shown in Fig.5, are the angles between the normal direction of the surface and the horizontal and vertical axis direction respectively.

(3) The surface of the tower is divided into 1367 sub-areas in which the values of the wind
 pressure coefficients of the embodied pressure taps are regarded as the same. In each subarea, the
 nominal wind pressure coefficients can be calculated as:

$$\mu_{sk} = \frac{\sum_{i=1}^{n} A_{i} \mu_{si}}{\sum_{i=1}^{n} A_{i}}$$
(4)

where μ_{sk} is the wind pressure coefficient of the *k*th subarea, A_i is the influence area of *i*th pressure tap, *n* is the total number of the pressure taps included in the *k*th area. Then the wind pressure of the *k*th subarea can be written as:

$$w_k(t) = \mu_{sk}(t)\mu_{zk}w_0 \tag{5}$$

169 where w_k is the wind pressure of the *k*th subarea, μ_{zk} is the height coefficients of the wind pressure, 170 w_0 is the basic wind pressure, in this study $w_0=0.30$ kPa according to 10-year return period (2012).

171 Both the Eqs. (4) and (5) applies to *x*, *y*, and *z*-direction.

(4) Lastly, the loading nodes in the FEM are selected for the subareas in Step (3), and the wind
loading time histories can be calculated as:

$$F_m(t) = w_k(t)A_m \tag{6}$$

where F_m is the *m*th loading nodes, A_m is the influence area of the *m*th loading nodes. Finally, 4101 time histories in FEM global coordinate system are obtained.

177 **5. Vibration control using DPTMD**

178 5.1 Design of the Double-stage Pendulum Tuned Mass Damper

179 A TMD system is installed in the BOT in order to mitigate the excessive wind-induced vibration. 180 Constrained by the architectural requirements, the space available for the mitigation service in this 181 building is quite limited: only a room of 4.80m×4.20m×5.80m can be utilized. In order to take full 182 advantage of the space without a significant increase in the total weight, the fire water tank is utilized 183 as the mass element of the absorber, because the volume of the tank excludes the possibility as a 184 sloshing-based mitigation. In consideration of all these factors, a Double-stage Pendulum Tuned 185 Mass Damper (DPTMD) was designed and installed on the floor of the Tower1 at the height of 186 232.5m(Fig.7).

There are some requirements, including long period, limit space and all motion directions, for the TMD used in this tower. If a conventional pendulum TMD scheme was used in this project, at least a space with more than 7.90m in height should be needed. However, the height of the available space for the TMD is only 5.80m. Therefore, this DPTMD system utilizes the fire water tank as the

191 mass element and hangs the tank by a double-stage suspension system. As shown in Fig.8, this 192 system is composed of the fire water tank, outer cables, inner cables, suspension frame, tank support, 193 viscous dampers, limiters, structure connections, and support connections. The tank support supports the fire water tank, which has a total mass of 50t. They together are hung by the double-stage 194 195 suspension system consisting of the outer cables, inner cables, and suspension frame. Four viscous 196 dampers are installed between the floor and the tank support to suppress the motion between the tank 197 and its support. Moreover, four limiters are used to restrict the maximum displacement of the tank. 198 The parameters of the DPTMD are shown in Table.4, determined by the limitations of the total mass, 199 maximum stroke and installation space of the DPTMD. The mass ratio of the DPTMD to the 200 structure is only 0.058%. The frequency ratio of the DPTMD in the directions of the first two modes are 1.055 and 1.025, while the corresponding damping ratios are 0.120 and 0.116 respectively. 201 202 Considering the uncertainty in the construction, one can adjust the length of the cables after the field 203 testing.

204 *5.2 Numerical simulation of the wind-induced vibration*

205 5.2.1 Spatial distribution

The wind-induced vibrations of the BOT with and without TMD are analyzed by inputting the three-dimensional wind field simulated in Section 4.2 to the FEM model. Fig.9 shows the spatial distribution of the maximum node acceleration over time under the wind directions of 45° and 135°. The combination of the horizontal accelerations and of all the three acceleration components are $\sqrt{a_x^2 + a_y^2}$ and $\sqrt{a_x^2 + a_y^2 + a_z^2}$, in which a_x , a_y , and a_z are the maximum node accelerations over time in the directions of X, Y and Z. It can be observed that: (1) The acceleration distributions of the BOT with and without TMD are similar. After the installation of the TMD, the total horizontal

213 accelerations are all reduced. However, the TMD does not affect the vertical acceleration responses.
214 (2) In general, the acceleration increases from the bottom to the top and from the inner part to the
215 outer part. The accelerations of the tower crown are larger than those of the body, especially in the
216 vertical direction. The maximum accelerations the main bodies of the Tower 2 and 4 are larger than
217 those of the other sub towers, while those of the crowns of the Tower 3 and 5 are larger than the other
218 towers.

219 5.2.2 Response variation with wind directions

220 There are two main public scenic sites in each sub tower: one is the viewing platform at the top 221 of each tower, and the other is the viewing hall, which is the floor immediately below the platform. 222 As there will be many people on these sites to watch the scenery of Beijing city, the issue of human comfort is of concern. Therefore, by considering eight typical wind directions, the mean and 223 maximum horizontal accelerations of the viewing platform and hall are assessed and shown in Fig.10 224 225 and Fig.11. It can be concluded from the Fig.10 that: (1) In general, the maximum acceleration 226 occurs at the edge nodes of the viewing platform. The structural constraint on these nodes is much 227 weaker than those at the middle. Thus the reduction effect of the TMD on the mean accelerations is 228 better than on the maximum accelerations. (2) The wind-induced accelerations of the tower subjected 229 to the wind loads with the directions of 45°, 135°, 225°, and 315° are larger than those subjected to 230 the wind loads with the other directions. This phenomenon is mainly due to the first two modal 231 shapes of the BOT. (3) The best reduction effect is with the wind direction of 270°, when the reduction ratio of Tower1, Tower2, and Tower3 are 16.44%, 30.05%, and 22.63%, respectively. (4) 232 Because the DPTMD was installed only on Tower1, it has a slight effect on some of the sub towers, 233 among which reduction effect on Tower5 is limited. The reduction effect in the viewing hall as 234

shown in Fig.11 is similar to the effect in the viewing platform. Moreover, the maximum reduction ratio of the viewing hall is 24.56%, which also occurs in the wind direction of 270°. As the connections between Tower1 and Tower5 are weaker than those between Tower1 and the other sub towers, the acceleration reduction effect, both in the viewing platform and in the viewing hall of Tower5, is the weakest among these sub towers. Moreover, the reduction effect in the viewing hall is much better than in the viewing platform.

241 5.2.3 Response time history and PSD

242 The time history responses and the corresponding power spectral densities of two nodes on the 243 viewing platform of the Tower1 and Tower5 in the x-direction are shown in Fig.12 and Fig.13 with 244 the wind direction of 45°. Meanwhile, these responses in the y-direction are shown in Fig.14 and Fig.15. In the time domain, the reduction of the displacements is better than the accelerations at both 245 246 x and y directions. In the frequency domain, the DPTMD has a considerable reduction effect on both 247 the accelerations and displacements at the first bending mode but does not affect the higher modes. 248 However, the higher modes do have some influences on the wind-induced responses, and the 249 accelerations are affected much more than the displacements. Comparing with the node in Tower 5, 250 the reduction effect of the node in Tower1 is much better, and this confirms the conclusions from the 251 analysis in the section above.

252 6. Concluding remarks

The wind-induced response of the BOT and the performance of its vibration control are presented in this paper. The three-dimensional wind loading for the FEM analysis of the BOT is established based on the wind tunnel test results. Moreover, based on the dynamic characteristics and responses of the BOT, a DPTMD was designed and installed on the tower. Then, the wind-induced

257	responses of the tower with and without DPTMD are compared with respect to various wind
258	directions and both in the time and frequency domain. Based on the results and discussions presented,
259	the following remarks can be obtained:
260	(1) Mostly, the wind-induced accelerations of the tower subjected to the wind loads with the
261	directions of 45°, 135°, 225°, and 315° are larger than those with the other directions.
262	(2) The DPTMD design utilizes a shorter pendulum length to achieve a long period, thus a lot of
263	spatial space can be saved given the excellent control performance on the BOT can be ensured.
264	The design serves as a reference for other slender high-rise towers.
265	(3) Only with 0.058% structural mass does the DPTMD mitigate the wind-induced vibration of the
266	tower effectively. The maximum acceleration reduction ratio for the mean accelerations of each
267	floor, which occurs in Tower2 with the wind direction of 270°, reaches 30.05%. Because of the
268	relatively weak connections between sub towers, the reduction effect of the Tower5 is lower than
269	the other sub towers.
270	(4) Because of the limitation of the physical space, only one TMD with a small mass can be installed
271	in this tower. Only the response induced by the first two modes can be suppressed, although
272	higher modes can affect the towers' wind-induced accelerations as well.
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The support from the Natural Science Foundation of China under Grant No. 51408389, the Natural Science Foundation of Jiangsu Province under Grant No. BK20161581 and BK20181078, the Six Talent Climax Foundation of Jiangsu under Grant No. JZ-004, the Natural Science Foundation of Jiangsu Education Department under Grant No. 19KJA430019, Jiangsu Government Scholarship for

- 279 Overseas Studies and A Project Funded by the Priority Academic Program Development of Jiangsu
- 280 Higher Education Institutions is gratefully acknowledged.
- 281

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Table.1 Structural geometry of the BOT

Tower	Tower1	Tower 2	Tower 3	Tower 4	Tower 5
Architectural diameter/ m	16.20	9.60	9.60	8.30	8.30
Structural diameter/ m	14.00	7.30	7.30	6.00	6.00
Structural height/ m	244.35	228.00	210.00	198.00	186.00
Slenderness ratio	17.45	31.23	28.77	33.00	31.00
Diameter of the top/ m	51.20	33.60	32.40	30.00	26.40
Distance to Tower1/m	-	19.17	15.75	15.10	16.90

374

Table.2 Natural frequencies of the tower

_	Modes	Frequency/ Hz	Description
	1	0.176	1st overall bending (45°)
	2	0.181	1st overall bending (135°)
	3	0 227	1st overall torsional (anti-symmetric bending of the Tower2 and 4 (45°) +
	5	0.227	anti-symmetric bending of the Tower3 and 5 (135°))
	4	0.512	2nd overall bending (45°)
	5	0.559	2nd overall bending (135°)
5	6	0.598	2nd overall torsional (symmetric bending of the Tower2 and 4 (45°) +
			anti-symmetric bending of the Tower3 and 5 (135°))
	7 0.703	0.702	2nd overall torsional (symmetric bending of the Tower2 and 4 (45°) + symmetric
\frown		0.703	bending of the Tower3 and 5 (135°))
1	0 0 701	2nd overall torsional (anti-symmetric bending of the Tower2 and 4 (45°) +	
_	8	0.721	symmetric bending of the Tower3 and 5 (135°))
	0	9 0.790	2nd overall torsional (local vibration of the Tower4 crown + anti-symmetric
	9		bending of the Tower3 and 5 (135°))
_	10	1.019	3rd overall bending (45°)
-	11	1.088	3rd overall bending (135°)

376
377

Table.3 Simulated turbulence intensity in the model tests

_	Segment model	Elevation/ m	Reference height/ m	Reference turbulence intensity/ %	Simulated turbulence intensity/ %
	1	0-49.2	30-40	18-20	18
	2	49.2-109.2	70-90	15-16	15
	3	109.2-169.2	130-150	13-14	14
	4	145.8-244.35	180-200	12	11.5

3	8	0
3	8	1

Table 4. Parameters of the DPTMD system

 Direction	x	У
 Effective length of cable/ m	7.90	7.90
Mass/ t	50	50
 Damping exponent of the damper	1	1
Damping coefficient of the damper/ N·s/m	6997	6788



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TAL_1704_Fig.1b.tif





TAL_1704_Fig.3a.tif



TAL_1704_Fig.3b.tif

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Note: T is short for Tower.

TAL_1704_Fig.4b.tif



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TAL_1704_Fig.8a.tif


















































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TAL_1704_Fig.10f.tif









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