

Calving multiplier effect controlled by melt undercut geometry

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Key Points:

- An elastic beam model is used to analyse calving driven by melt undercutting at tidewater glaciers
- The presence of a calving multiplier effect is found to be sensitive to the shape of melt undercutting
- The approach offers a promising route to parameterising calving driven by melt undercutting in large-scale ice sheet models

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Abstract

Quantifying the impact of submarine melting on calving is central to understanding the response of marine-terminating glaciers to ocean forcing. Modeling and observational studies suggest the potential for submarine melting to amplify calving (the calving multiplier effect), but there is little consensus as to under what conditions this occurs. Here, by viewing a marine-terminating glacier as an elastic beam, we propose an analytical basis for understanding the presence or absence of the calving multiplier effect. We show that as a terminus becomes undercut it becomes more susceptible to both serac failure (calving only of ice that is undercut, driven by vertical imbalance) and rotational failure (full thickness calving of ice behind the grounding line, driven by rotational imbalance). By deriving analytical stress thresholds for these two forms of calving, we suggest that the dominant of the two calving styles is determined principally by the shape of melt undercutting. Uniform undercutting extending from the bed to the waterline promotes serac failure and no multiplier effect, while glaciers experiencing linear undercutting that is greatest at the bed and zero at the waterline are more likely to experience rotational failure and a multiplier effect. Our study offers a quantitative framework for understanding where and when the calving multiplier effect occurs, and, therefore, a route to parameterising the effect in ice sheet-scale models.

1 Introduction

Calving and submarine melting are key forms of mass loss at glaciers that terminate in water. For the Greenland Ice Sheet, the combination of calving and submarine melting is the largest source of mass loss in a typical year (van den Broeke et al., 2016) and an increase in calving and submarine melting is responsible for half of Greenland’s 1992–2018 sea level contribution of 10.8 mm (The IMBIE Team, 2020). Given the likelihood of continuing global ice mass loss over the coming decades and centuries (Hock et al., 2019; Goelzer et al., 2020; Seroussi et al., 2020), understanding calving and submarine melting and their interplay, and ensuring their faithful representation in models are high priorities.

Calving occurs in many styles depending on the geometry of the ice body and can take the form of frequent small events or infrequent large events (Benn et al., 2007; Åström et al., 2014). Calving may be driven purely by geometric effects or may be sensitive to external influences such as surface melt, ice mélange or submarine melting (Benn, Cowton, et al., 2017; Catania et al., 2020). The focus of this study is on glaciers with grounded termini and on the interaction of calving with submarine melting. Submarine melting removes ice from the submerged portion of the terminus, and, depending on the distribution of melting, can give rise to termini that are preferentially undercut at depth, at the water surface or in confined chimneys above subglacial channels (Rignot et al., 2015; Slater et al., 2018; Wagner et al., 2019; Fried et al., 2019; Sutherland et al., 2019). In turn, undercutting may influence calving through (i) changing buoyant forces at the terminus giving rise to rotational or vertical imbalances (Hughes, 1992; Hanson & Hooke, 2000), (ii) the removal of lateral support provided to the glacier by side walls (Cowton et al., 2019) or (iii) the removal of basal friction resulting in ice flow acceleration and increased stresses within the ice (Morlighem et al., 2016). The potential for submarine melting, through each and any of these processes, to drive calving at many times the rate of melting has become known as the ‘calving multiplier effect’ (O’Leary & Christoffersen, 2013; Benn, Astrom, et al., 2017; Ma & Bassis, 2019).

Due to coarse model resolution and a lack of process understanding and suitable parameterisations, the representation of the impact of submarine melting on calving in ice sheet-scale models is currently limited. In a recent leading sea level projection exercise, most ice sheet models did not have sufficient resolution to resolve calving and relied on a pragmatic but heavy-handed parameterisation for glacier retreat (Slater et al.,

2019; Goelzer et al., 2020). Only the more advanced models resolved calving and submarine melting (Morlighem et al., 2019; Aschwanden et al., 2019), but even these state-of-the-art approaches do not, explicitly at least, account for the first two of the three influences of submarine melting on calving described above. Because resolving all of the relevant processes is beyond current computational capabilities, work towards calving parameterisations is of great importance.

Observational constraints on the impact of submarine melting on calving are hard to come by due to the difficulty and danger of measuring submarine melting, the shape of the terminus beneath the water and the magnitude of calving events. The link between submarine melting and calving has, therefore, often been inferred from large-scale remote observations connecting glacier retreat to ocean warming (e.g. Straneo & Heimbach, 2013). At the glacier scale, Luckman et al. (2015) found a significant relationship between near-glacier water temperature and frontal ablation rate at three tidewater glaciers in Svalbard, strongly suggestive of a tight relationship between submarine melting and calving. Bartholomaeus et al. (2013) and How et al. (2019) have similarly argued on observational grounds for calving being paced by melt undercutting. At each of the glaciers considered in these studies, however, most of the calving was small scale, high frequency events above the water line suggesting the calving rate would be equal to the submarine melt rate. A significant calving multiplier effect has not, to our knowledge, been conclusively observed.

Much of our knowledge on the calving multiplier effect, therefore, comes from models, yet such studies are equivocal on the impact of submarine melting on calving. Using flowband simulations in Elmer/Ice, Cook et al. (2014), Todd and Christoffersen (2014) and Krug et al. (2015) all argued for a limited impact of submarine melting on calving, while Mercenier et al. (2020) suggest that increased melting can, in fact, decrease total mass loss. Vallot et al. (2018), Todd et al. (2018), Todd et al. (2019) and Cowton et al. (2019) have argued for a nuanced effect of submarine melting on calving, in which melt is important to the seasonal details of the calving, or is important depending on the magnitude and distribution of melting. Three recent studies very explicitly simulate a calving multiplier effect. Ma and Bassis (2019) used flowband simulations, accounting for damage evolution and both tensile and shear failure, to find a calving multiplier effect for certain melt rates and undercut shapes. Benn, Astrom, et al. (2017) identified a calving multiplier effect for certain water depth, ice thickness and undercut length combinations. In simulations of Bowdoin Glacier, van Dongen et al. (2020) recently argued for a multiplier effect for certain distributions of submarine melting. The latter two studies used a discrete particle model from which calving styles appear naturally without the need to impose modes of failure.

The emerging picture then, is that the calving multiplier effect does prevail in some situations, depending on the topographic regime of the glacier, the relative magnitude of the submarine melt rate and ice velocity, the distribution of submarine melting, or the water depth and ice thickness. As such, there is little consensus on the impact of submarine melting on calving, precluding the development of a unifying parameterisation for incorporating this essential link into ice sheet-scale models. Here, we suggest framing the presence or absence of the calving multiplier effect based on a simple thresholding of the tensile and shear stresses obtained when considering the glacier as an elastic beam subject to a torque and shear at one end. The basic idea is that a multiplier effect is active if an undercut can stably grow sufficiently large that the torque exerted on the terminus results in a bending moment in the ice that exceeds a threshold value. If, instead, the undercut ice calves through shear/serac failure before the bending moment threshold is reached, no multiplier effect is observed. The elastic beam approach we take offers a promising analytical route towards parameterising the presence or absence of the calving multiplier effect in ice sheet-scale models.

2 Methods

2.1 Elastic beam theory

The fundamental tool used in this study is elastic thin beam theory and is not new to glaciology, having been extensively used to study buoyant flexure of ice shelves (e.g. Vaughan, 1995; Sergienko, 2010; Sayag & Worster, 2011; Mosbeux et al., 2020) and tide-water glaciers (Wagner et al., 2016). It has not, however, been used to model the stress response of a tidewater glacier to melt undercutting at its terminus. Beam theory describes the shape and stress response of an elastic beam to loading along its length or at one end. Let the glacier (i.e. beam) flow from left to right with coordinate x measuring the along flow distance and let the ice divide and grounding line be at $x = -L$ and $x = 0$, respectively (Fig. 1a). The glacier has a constant ice thickness H and a constant bending stiffness $D = EH^3/12(1 - \nu^2)$, where E is the elastic modulus and ν is Poisson's ratio. The ice has density $\rho_i = 910 \text{ kg m}^{-3}$, is subject to gravity $g = 9.81 \text{ m s}^{-2}$ and rests on a Winkler foundation: that is, a flat elastic bed of stiffness k . The glacier surface elevation from the ice divide to the grounding line, $w(x)$, is then given by (e.g. Sayag & Worster, 2011)

$$Dw'''' + kw = -\rho_i g H \quad (1)$$

where the primes denote derivatives with respect to x . All solutions to Eq. 1 used in this study are obtained analytically, and, except where stated otherwise, we take $E = 1 \text{ GPa}$ and $\nu = 0.3$. The bed stiffness is not well constrained by observations, though Sayag and Worster (2013) have argued that values of approximately $k = 1 \text{ MPa/m}$ to 1 GPa/m are appropriate for deformable glacial till at the bed. Unless stated otherwise, we assume a value $k = 1 \text{ MPa/m}$, appropriate for soft till, as has been inferred to persist beneath Greenland's largest tidewater glaciers (e.g. Shapero et al., 2016).

The glacier terminates in water of depth d and density $\rho_w = 1030 \text{ kg m}^{-3}$ and the terminus of the glacier is taken to be vertical or undercut. Although none of the mathematical set-up is specific to an undercut shape, we will in the results section consider two undercut shapes: a linear undercut when undercutting increases linearly from zero at the waterline to a length of u at the bed (Fig. 1b) and a uniform undercut when all of the submerged ice is undercut to a length u (Fig. 1c). This undercutting and the imbalance between ice and water pressure at the terminus impose a torque and shear on the glacier that are represented in beam theory by boundary conditions.

The use of a thin beam model (Eq. 1) is an approximation to the full two-dimensional flowline problem. The thin beam approach is, in general, appropriate when the ice thickness is not large compared to the flexural wavelength of the glacier. In the present application, this is certainly the case for the thinner glaciers considered ($H \sim 100 \text{ m}$), but for thick glaciers ($H \sim 900 \text{ m}$) additional effects beyond the thin beam model may become more important. We expect, however, that these additional effects will make quantitative rather than qualitative differences to our results; that is, the use of a more complex model would change some of the numbers, but would not change the findings or conclusions. The great benefit of using a thin beam model is the opportunity to make analytical progress on a topic that is dominated by models of high complexity and we therefore consider its use to be justified for the present application.

From the outset it is also important to note that our analysis considers a glacier flowline (Fig. 1) and, therefore, neglects across-glacier variability in stress and terminus shape. As such, our results will be most appropriate where the glacier has a width that is large compared to the ice thickness and where undercutting extends across a broad swath of the terminus. Because observations of the submerged shape of calving fronts do, however, show significant across-glacier variability (Fried et al., 2019), including localised undercut chimneys, the flowline nature of our analysis is a limitation that should be borne in mind throughout. We nevertheless consider it an important step towards un-

167 understanding the impact of submarine melting on calving, and particularly towards pa-
 168 rameterising this process in ice sheet models.

169 2.2 Boundary conditions

170 Solution of Eq. 1 requires four boundary conditions that, for a glacier that is un-
 171 dercut at its terminus, are taken as $w(-L) = -\rho_i g H/k$, $w'(-L) = 0$, $w''(0) = M/D$
 172 and $w'''(0) = Q/D$. The first two conditions ensure that far inland from the ground-
 173 ing line, the weight of the glacier is fully supported by the bed and the surface elevation
 174 is flat. Note that the solutions presented later are given in the limit $L \rightarrow -\infty$. The third
 175 condition imposes a torque at the grounding line that arises from the imbalance between
 176 ice and water pressure (Fig. 1b) and the fourth condition imposes a shear at the ground-
 177 ing line that is required to support the weight of undercut ice beyond the grounding line
 178 (Figs. 1b and c). Note that the undercut region is not modeled explicitly; rather, the ef-
 179 fect of the undercut region is imposed through the torque and shear boundary conditions
 180 applied at the grounding line.

181 The depth-integrated torque at the grounding line, M , is evaluated on the neutral
 182 plane of the beam (i.e. halfway between the bed and ice surface) and is defined as $M =$
 183 $\int F \times r \, dr$. Here, F is the net force resulting from the ice and water pressure, r is the
 184 vector from the neutral plane at the grounding line (point ‘o’ on Figs. 1b and c) to the
 185 terminus and the integral runs over the subaerial and submerged interfaces of the ter-
 186 minus (Fig. 1b). Note that the depth-integrated torque acts to rotate the terminus top-
 187 forwards into the ocean when $M < 0$ and bottom-forwards into the ocean when $M >$
 188 0 . The depth-integrated shear at the grounding line, Q , is calculated as the difference
 189 between the weight of ice beyond the grounding line and the weight of water that it dis-
 190 places.

191 2.3 Solution of elastic beam theory

192 The elastic beam model (Eq. 1) has an analytical solution given by

$$w = \frac{\lambda^2}{2D} \left[(M - Q\lambda) \cos\left(\frac{x}{\lambda}\right) + M \sin\left(\frac{x}{\lambda}\right) \right] \exp\left(\frac{x}{\lambda}\right) - \frac{\rho_i g H}{k} \quad (2)$$

193 in which $\lambda = \left(\frac{4D}{k}\right)^{1/4} = \left[\frac{EH^3}{3k(1-\nu^2)}\right]^{1/4}$ is the characteristic length scale that controls
 194 how far up-glacier the terminus boundary conditions are felt. For the adopted values of
 195 the ice and bed modulus, $\lambda = 450$ m when the ice thickness is $H = 500$ m. Note that
 196 the term in the fully general solution that is proportional to $\exp\left(-\frac{x}{\lambda}\right)$ vanishes in the
 197 limit $L \rightarrow -\infty$ assumed here. The longitudinal stress on the top surface of the glacier,
 198 σ_r , is given by (e.g. Wagner et al., 2016)

$$\sigma_r = -\frac{6D}{H^2} w'' = \frac{6}{H^2} \left[(M - Q\lambda) \sin\left(\frac{x}{\lambda}\right) - M \cos\left(\frac{x}{\lambda}\right) \right] \exp\left(\frac{x}{\lambda}\right) \quad (3)$$

199 Lastly, the depth-mean shear at the grounding line, σ_s , can be obtained by dividing the
 200 depth-integrated shear by the grounding line ice thickness H_{gl} :

$$\sigma_s = Q/H_{gl} \quad (4)$$

201 2.4 Calving criteria

202 With the described framework, we are able consider two calving styles: serac fail-
 203 ure and rotational failure (Fig. 2). Rather than adopting a scheme to calculate crevasse
 204 depth (e.g. Benn et al., 2007), we assume that full-depth crevassing and calving occurs
 205 when the shear or tensile stresses exceed a threshold value.

206 Serac failure involves calving only of ice that is undercut and results from high shear
 207 stress (Fig. 2a). It can be shown that the highest shear stress at any point along the un-
 208 dercut ice is almost always at the grounding line (Supporting Information) and so serac
 209 failure is most likely to occur at the grounding line itself. We impose serac failure (Fig. 2a)
 210 when the depth-mean shear stress at the grounding line, σ_s , exceeds a threshold value
 211 that is taken as $\sigma_s^{max} = 0.5$ MPa (e.g. Ma & Bassis, 2019). An alternative serac fail-
 212 ure mechanism involving downward bending of the cantilever beam formed by uniform
 213 undercutting is considered in supporting information but does not give significantly dif-
 214 ferent results.

215 Rotational failure is calving resulting from a high bending moment in the ice (Fig. 2b).
 216 This bending moment is a response to the torque and shear exerted at the terminus and
 217 results in tensile and compressive stresses on the top and bottom of the glacier, respec-
 218 tively. We impose rotational failure (Fig. 2b) when the longitudinal stress, σ_r , at any point
 219 on the top surface of the glacier exceeds a threshold value that is taken as $\sigma_r^{max} = 1$
 220 MPa (e.g. Ultee et al., 2020). Failure is assumed to occur at the surface stress maximum,
 221 which from Eq. 3 is located at the point x_0 given by

$$\tan\left(\frac{x_0}{\lambda}\right) = \left(\frac{2M}{Q\lambda} - 1\right)^{-1} \quad (5)$$

222 and the maximum value can be obtained by substituting x_0 into Eq. 3. Note that due
 223 to the boundary conditions (section 2.2) and because for vertical or undercut ice the depth-
 224 integrated shear Q is always zero or positive, the maximum of the longitudinal stress on
 225 the top surface of the glacier is always located at or upstream of the grounding line. Ro-
 226 tational failure, therefore, involves full-thickness calving of ice upstream of the ground-
 227 ing line.

228 For melt undercutting to be the process driving calving, we require the glacier to
 229 be stable to calving when the front is vertical. Large glaciers terminating in shallow wa-
 230 ter are unstable to ice cliff failure before undercutting and are, therefore, not observed
 231 in reality (Bassis & Walker, 2012). As such, in this study we only consider glaciers that
 232 are stable to ice cliff failure according to the metric of Bassis and Walker (2012), which
 233 restricts the possible ice thickness and water depth combinations to those that satisfy

$$\frac{1}{4}\rho_i g H \left[1 - \frac{\rho_w}{\rho_i} \left(\frac{d}{H} \right)^2 \right] < \sigma_s^{max} \quad (6)$$

234 For a given ice thickness, this condition imposes a minimum water depth on the glaciers
 235 we consider. For example, if $H = 500$ m, the water depth d must exceed 350 m. Al-
 236 though ice thicknesses up to 1900 m are allowable under Eq. 6, with Greenlandic tide-
 237 water glaciers in mind we will below consider a maximum ice thickness $H = 900$ m.

238 For a glacier that is stable when the front is vertical, we define the dominant calving
 239 style by which failure threshold (serac or rotational) is reached first as the terminus
 240 becomes progressively more undercut. If the calving length, l , is the distance from the
 241 most advanced part of the glacier to the calving position and u is the undercut length
 242 at the grounding line (Fig. 2), then we define the calving multiplier as $\beta = l/u$. Serac
 243 failure by definition occurs at the grounding line, hence $l = u$ and $\beta = 1$, and there
 244 is no multiplier effect under serac failure. Rotational failure occurs upstream of the ground-
 245 ing line, hence $l > u$ and $\beta > 1$, and there is a multiplier effect. The presence of a mul-
 246 tiplier effect, therefore, depends on the dominant calving style, which is now analysed
 247 using elastic beam theory and the described framework.

3 Results

3.1 Serac failure

Because the depth-mean shear stress at the grounding line is entirely determined by the geometry of the undercut, serac failure can be analysed independently of beam theory. The depth-mean shear stress (i.e. the depth-mean of σ_{xz}) at the grounding line is given by

$$\sigma_s = \frac{Q}{H} = \rho_i g u \left[1 - \frac{d}{2H} \left(1 + \frac{\rho_w}{\rho_i} \right) \right] \quad (7)$$

for a linear undercut and $\sigma_s = Q/(H-d) = \rho_i g u$ for a uniform undercut, with details of the calculation given in Supporting Information. For linear undercutting, the shear stress at the grounding line increases with greater undercutting because there is more ice beyond the grounding line that must be supported (Fig. 3a). The shear stress decreases with deeper water because more of the weight of ice beyond the grounding line is hydrostatically compensated (Fig. 3a). For uniform undercutting, the shear stress similarly increases with greater undercutting, but is independent of the water depth because none of the undercut ice is supported by the ocean at all (Fig. 3b).

The imposition of the shear stress threshold for failure gives a critical undercut length, u_s , at which serac failure occurs, given from Eq. 7 as

$$u_s = \frac{\sigma_s^{max}}{\rho_i g} \left[1 - \frac{d}{2H} \left(1 + \frac{\rho_w}{\rho_i} \right) \right]^{-1} \quad (8)$$

for linear undercutting, or $u_s = \sigma_s^{max}/\rho_i g$ for uniform undercutting. As would be expected from Figs. 3a and b, the critical undercut length increases sensitively with the water depth for linear undercutting but is independent of the water depth for uniform undercutting (Fig. 3c). With $\sigma_s^{max} = 0.5$ MPa, serac failure occurs at an undercut of $u_s = 56$ m for uniform undercutting regardless of water depth, but not until an undercut of $u_s = 220$ m for linear undercutting when the water depth fraction, d/H , is 0.7.

Note that these undercutting thresholds assume fully intact ice at the grounding line of thickness H in the linearly undercut case and $H-d$ in the uniformly undercut case. If, instead, the glacier has surface crevasses over the grounding line such that the remaining intact ice thickness is a fraction f of the fully intact ice thickness, then the undercut length for serac failure (Fig. 3c, Eq. 8) is reduced by a factor f . For the examples just given, if $f = 0.5$, so that the crevasse depth is $(H-d)/2$ in the uniformly undercut case and $H/2$ in the linearly undercut case, then serac failure would occur at an undercut of $u_s = 28$ m for uniform undercutting and $u_s = 110$ m for linear undercutting when the water depth fraction is 0.7.

3.2 Depth-integrated torque on the calving front

Analysing rotational failure is more complex, but before looking at beam theory it is instructive to first examine the depth-integrated torque, M , exerted on the calving front by ice and water pressure. We here analyse only the linear undercut case, but emphasize that the uniformly undercut case is similar (Supporting Information). The depth-integrated torque for the linearly undercut case is

$$M = \frac{1}{12} \rho_i g H^3 \left[1 - \frac{\rho_w}{\rho_i} \left(3 \frac{d^2}{H^2} - 2 \frac{d^3}{H^3} \right) - 6 \left(1 - \frac{2d}{3H} - \frac{1}{3} \frac{\rho_w}{\rho_i} \frac{d}{H} \right) \frac{u^2}{H^2} \right] \quad (9)$$

with details of the calculation given in supporting information.

Firstly, for a given water depth fraction d/H , the depth-integrated torque always decreases as the length of the undercut increases (Fig. 4a). Thus, undercutting always increases the propensity of the terminus to tip top-forwards into the ocean (Fig. 4a). Sec-

289 only, for a vertical ($u/H = 0$) or slightly undercut ($u/H = 0.25$) terminus, the depth-
 290 integrated torque decreases as the water depth increases, because the greater water pres-
 291 sure acts to tip the terminus top-forwards (Fig. 4b). In particular, for a vertical termi-
 292 nus, the depth-integrated torque on the calving front acts to tip the terminus top-forwards
 293 only when the fractional water depth d/H exceeds 0.78. For deep undercutting ($u/H =$
 294 0.5), however, the depth-integrated torque increases with deeper water because the wa-
 295 ter increasingly supports the weight of the undercut ice (Fig. 4b). Viewing the depth-
 296 integrated torque as a function of both the water depth and undercut length shows that
 297 for shallow water and shallow undercutting, the torque is positive and, therefore, acts
 298 to tip the terminus bottom-forwards (Fig. 4c). As an aside, when sufficiently large this
 299 positive torque has been implicated in ice cliff failure (Hanson & Hooke, 2000), though
 300 in this study we use a longitudinal stress-based ice cliff failure criteria (Eq. 6). For deep
 301 water or deep undercutting, the torque is negative and acts to tip the terminus top-forwards
 302 (Fig. 4c).

303 3.3 Elastic beam solutions

304 Some example solutions of the elastic beam model (Eqs. 2 & 3) are shown in Fig. 5.
 305 For a vertical terminus, there is no shear acting on the front, and if in addition the wa-
 306 ter depth is sufficiently small, the torque on the terminus acts to tip it bottom-forwards
 307 and so the glacier flexes upwards at the front (Fig. 5a) and the resulting longitudinal stress
 308 on the glacier surface is compressive close to the terminus (Fig. 5b). For deeper water
 309 (but still a vertical terminus), the torque acts to tip the front top-forwards and so the
 310 glacier flexes downwards at the front and the surface stress is tensile, with a maximum
 311 at the grounding line. Once undercutting is imposed, there is both a torque and a shear
 312 acting on the terminus. The glacier increasingly flexes downwards at the terminus (and
 313 upwards some distance upstream), the surface stress increases in magnitude and the sur-
 314 face stress maximum is upstream of the grounding line (Fig. 5). The distance upstream
 315 to the stress maximum is largely controlled by the characteristic length λ .

316 An important distinction is between scenarios where the glacier flexes upwards or
 317 downwards at the terminus, as this determines whether the surface stress is compressive
 318 or tensile close to the terminus (e.g. Fig. 5). Based on the analytical solution (Eq. 2) it
 319 can be shown that the glacier flexes downwards when $2M < Q\lambda$. If the terminus is ver-
 320 tical (so that there is no shear and $Q = 0$), then the glacier flexes downwards when the
 321 torque acts to tip the glacier top-forwards ($M < 0$). If the torque acts to tip the glacier
 322 bottom-forwards ($M > 0$), it is still possible for the glacier to flex downwards provided
 323 undercutting results in sufficient shear on the terminus to pull it downwards. Thus, the
 324 flexure of the glacier and the nature of the surface stress is determined by both the torque
 325 and the shear acting on the terminus, which are in turn functions of the ice thickness,
 326 water depth, undercut length and undercut shape.

327 3.4 Rotational failure

328 For an ice thickness $H = 500$ m but variable water depth and linear undercut length,
 329 the magnitude and position of the stress maximum are given through Eq. 5 and shown
 330 in Fig. 6. The magnitude of the stress maximum increases with the degree of undercut-
 331 ting (Fig. 6a, see also Fig. 5b), and except for very large undercut lengths, also increases
 332 with the water depth. The position of the stress maximum (x_0 , which is negative because
 333 it is upstream of the grounding line) is furthest behind the grounding line at small wa-
 334 ter depths and undercut lengths (Fig. 6b), but because the magnitude of the stress max-
 335 imum in this region of the parameter space is small (Fig. 6a), calving is unlikely to oc-
 336 cur. The position of the stress maximum moves closer to the grounding line as the wa-
 337 ter depth is increased. When the front is vertical and the glacier is flexing downwards
 338 at the terminus, the maximum is located at the grounding line.

339 We have suggested imposing rotational failure when the surface longitudinal stress
 340 maximum exceeds the threshold value $\sigma_r^{max} = 1$ MPa, shown on Fig. 6a. The critical
 341 undercut length at which this occurs, u_r , can be obtained by inverting the equation

$$\sigma_r^{max} = \frac{6}{H^2} \left[(M - Q\lambda) \sin\left(\frac{x_0}{\lambda}\right) - M \cos\left(\frac{x_0}{\lambda}\right) \right] \exp\left(\frac{x_0}{\lambda}\right) \quad (10)$$

342 for the undercut length u (see also Eqs. 3 & 5). Since all of M , Q and x_0 are functions
 343 of u , this cannot in general be done analytically, but it is a very quick numerical calcu-
 344 lation. The critical linear undercut length at which rotational failure occurs for an ice
 345 thickness of 500 m and varying water depth is shown in Fig. 6c, and increases non-linearly
 346 with water depth to reach 400 m when the glacier is at flotation. Meanwhile, the loca-
 347 tion of the stress maximum moves closer to the grounding line for larger water depths.
 348 The total calving length measured at the glacier surface is the sum of the undercut length
 349 and the distance to the stress maximum and has a complex form, but is roughly 600 m
 350 regardless of water depth (Fig. 6c).

351 Because many tidewater glaciers terminate in water depths that make them close
 352 to flotation, it is illuminating to consider these results as a function of ice thickness as-
 353 suming the glacier is at flotation (Fig. 7). In these cases, the glacier always flexes down-
 354 wards at the terminus. For a given ice thickness, the stress maximum increases with the
 355 undercut length (Fig. 7a). The position of the stress maximum shows a complex depen-
 356 dence on undercut length, first moving upstream from the grounding line then moving
 357 back towards the grounding line as undercutting increases (Fig. 7b). The position of the
 358 stress maximum is much simpler when we consider only its value at the critical under-
 359 cut length, scaling approximately as $H^{3/4}$ (Fig. 7c), as might be expected from the char-
 360 acteristic length λ . The critical undercut length for rotational failure scales approximately
 361 as $H^{1/2}$ (Fig. 7c), which can be motivated by considering that the right hand side of Eq. 10
 362 scales roughly as M/H^2 and M itself scales as Hu^2 based on Eq. 9.

363 Although all of Figs. 4-7 show results assuming linear undercutting, we emphasize
 364 that the uniformly undercut results are relatively similar, because the depth-integrated
 365 torque M and shear Q are not overly sensitive to the choice of undercut shape.

366 3.5 Dominant calving style

367 We have argued that serac failure driven by shear stress occurs when the under-
 368 cut length reaches a critical value given by Eq. 8, while rotational failure driven by a high
 369 bending moment occurs when the undercut length reaches a critical value given through
 370 Eq. 10 (Fig. 2). The dominant of the two calving styles and the calving multiplier is then
 371 determined by which of these critical undercut lengths is reached first, which is in turn
 372 determined by the fractional water depth, ice thickness and undercut shape.

373 An example for a glacier having an ice thickness $H = 500$ m is shown in Fig. 8a.
 374 With uniform undercutting (Fig. 1c), serac failure occurs once the undercut length reaches
 375 $u_s = 56$ m (Fig. 8a; see also Fig. 3). This is smaller than the rotational failure under-
 376 cut threshold u_r for any water depth, so that in this case, uniform undercutting results
 377 only in serac failure. With linear undercutting (Fig. 1b), the critical undercut length for
 378 serac failure is a function of water depth (Fig. 8a). For water depths less than 325 m,
 379 the undercut length threshold for serac failure is reached before that for rotational fail-
 380 ure. Conversely, for linear undercutting and water depths greater than 325 m, the calv-
 381 ing front is relatively stable to serac failure and so rotational failure is the dominant calv-
 382 ing style (Fig. 8a). In reality, large glaciers having an ice thickness of 500 m are not found
 383 terminating in shallow water because they would be unstable to ice cliff failure (Eq. 6).
 384 Water depths less than $d = 350$ m are, therefore, unlikely to be observed in reality and
 385 only the furthest right portion of Fig. 8a is realistic.

386 This analysis can instead be viewed as a function of ice thickness if a choice of wa-
 387 ter depth is made; in Fig. 8b we choose this to put the terminus at flotation. In this case,
 388 there is a clear separation between the critical undercut lengths for serac failure and ro-
 389 tational failure. For all ice thicknesses between 100 and 900 m, the critical undercut length
 390 for serac failure is smaller than for rotational failure under uniform undercutting, and
 391 larger than for rotational failure under linear undercutting (Fig. 8b). Note that the crit-
 392 ical undercut length for serac failure differs so markedly between uniform and linear un-
 393 dercutting in large part due to the differing thickness of ice at the grounding line avail-
 394 able to support the shear stress (Fig. 1 and Eq. 7).

395 Similar results are found when extending the analysis to all ice thickness and frac-
 396 tional water depth combinations that are stable to ice cliff failure (Fig. 9). Considering
 397 first linear undercutting, for fractional water depths less than 0.5, serac failure dominates
 398 for certain ice thicknesses, but otherwise rotational failure is the dominant calving style
 399 (Fig. 9a). For fractional water depths greater than 0.5, rotational failure is the dominant
 400 calving style regardless of ice thickness. Because most tidewater glaciers are found to be
 401 close to flotation (e.g. Bassis & Walker, 2012), rotational failure will dominate under lin-
 402 ear undercutting. In contrast, for uniform undercutting almost all ice thickness and wa-
 403 ter depth combinations have serac failure as a dominant calving style, including those
 404 stable to ice cliff failure (Fig. 9b). Only rather thin glaciers sitting in shallow water un-
 405 dergo rotational failure.

406 These results can certainly be modified by changing the thresholds for serac and
 407 rotational failure (set here at $\sigma_s^{max} = 0.5$ MPa and $\sigma_r^{max} = 1$ MPa), but the sensible
 408 choice of these thresholds, together with the clear separation of undercut lengths in Fig. 8b
 409 and emphatic results in Fig. 9, suggests that uniform undercutting intrinsically promotes
 410 serac failure while linear undercutting intrinsically promotes rotational failure.

411 3.6 Calving multiplier

412 By considering the dominant calving style together with the calving position we
 413 can analyse the potential for a calving multiplier effect (Fig. 2). Under serac failure, the
 414 calving position is at the grounding line so that the calving length is equal to the un-
 415 dercut length and the calving multiplier is $\beta = 1$ (i.e. no calving multiplier effect). Un-
 416 der rotational failure, the calving position is upstream of the grounding line, the total
 417 calving length exceeds the undercut length and the calving multiplier β exceeds 1 (e.g.
 418 Figs. 6c and 7c).

419 Because uniform undercutting is almost exclusively associated with serac failure,
 420 there is little potential for a calving multiplier effect under uniform undercutting. Lin-
 421 ear undercutting does, however, give a calving multiplier effect whenever rotational fail-
 422 ure is the dominant calving style (Fig. 10). The effect is weakest for thin ice close to flota-
 423 tion, when the surface stress maximum is close to the grounding line, and is strongest
 424 for thin ice in shallow water (Figs. 6c, 7c & 10a). For large glaciers that are close to flota-
 425 tion the calving multiplier takes a value in the range $\beta = 1.4$ to 1.6 (Figs. 7c & 10a).

426 It is important, however, to stress that the magnitude of the calving multiplier is
 427 influenced strongly by the material parameters. We have chosen reasonable values, but
 428 in a practical application of our results these parameters could be chosen by compari-
 429 son to observations. An analysis of the sensitivity to the surface stress threshold is given
 430 in Supporting Information; here, we consider the ice and bed strength E and k while hold-
 431 ing the stress threshold constant. These parameters influence the position (Eq. 5) and,
 432 to a lesser extent, the value (Eq. 3) of the surface stress maximum through the charac-
 433 teristic length scale λ , which is proportional to $E^{1/4}$ and $k^{-1/4}$. Strengthening the ice
 434 (or weakening the bed) by an order of magnitude increases the characteristic length scale
 435 by 80%, and for a glacier of thickness 500 m at flotation, increases the calving multiplier
 436 by 26% (Fig. 10b). Similarly, weakening the ice (or strengthening the bed) by an order

437 of magnitude decreases the characteristic length by 44%, and for the same glacier, de-
438 creases the calving multiplier by 9%. As such, the magnitude of the calving multiplier
439 effect is sensitive to the ice and bed strength through their effect on the distance up-glacier
440 over which the terminus effects decay. But, for glaciers close to flotation, the clear sep-
441 aration between critical undercut lengths for serac and rotational failure (Fig. 8b) means
442 that the ice and bed strength do not influence the dominant calving style.

443 4 Discussion

444 4.1 Calving, undercut shape and submarine melting

445 We have derived a metric for serac failure based on depth-mean shear stress at the
446 grounding line and a metric for rotational failure based on the tensile stress at the glacier
447 surface induced by shear and torque at the terminus (Fig. 2). Once a choice of stress thresh-
448 old is made, these metrics give a critical undercut length at which the particular style
449 of calving occurs. By comparing undercut lengths between failure mechanisms a dom-
450 inant calving style emerges, which in turn determines the presence or absence of a calv-
451 ing multiplier effect. We find that the dominant calving style is critically dependent on
452 the shape of undercutting, with uniform undercutting promoting serac failure and lin-
453 ear undercutting promoting rotational failure.

454 We have considered only two undercut shapes - linear and uniform (Fig. 1) - but
455 clearly many other shapes are possible, including convex or concave undercutting (e.g.
456 Ma & Bassis, 2019), linear or uniform undercutting that does not extend all the way to
457 the fjord surface (e.g. Slater et al., 2017), waterline notching (Petlicki et al., 2015) or
458 overcutting and projecting ice feet (Wagner et al., 2016; Fried et al., 2019). Our calv-
459 ing metrics can be easily generalised to other terminus shapes by appropriate calcula-
460 tion of the depth-integrated torque M and the depth-integrated shear Q (Supporting In-
461 formation). In particular, the depth-integrated shear has to compensate for the hydro-
462 static imbalance of ice beyond the grounding line. For a given undercut length at the
463 bed, therefore, the more ice volume remaining below the water the less likely serac fail-
464 ure is to occur. Uniform undercutting reaching the fjord surface (Fig. 1c) is the shape
465 most likely to drive serac failure as it has no ice below the water beyond the grounding
466 line. Linear undercutting (Fig. 1b) or uniform undercutting that does not reach the sur-
467 face both have ice below the water, reducing the hydrostatic imbalance and making larger
468 undercuts more stable to serac failure (e.g. Fig. 3).

469 The sensitivity of calving style to undercut shape (Figs. 8 & 9) then transfers fo-
470 cus to the vertical profile of submarine melting that generates the undercut; submarine
471 melt rate that is uniform with depth will generate uniform undercutting while subma-
472 rine melt rate that increases with depth will generate linear undercutting. Much work
473 has been undertaken to understand submarine melting and its impact on undercut shape,
474 showing that fjord stratification, subglacial discharge and grounding line depth are all
475 key controls (e.g. Jenkins, 2011; Xu et al., 2013; Slater et al., 2016, 2017; Sutherland et
476 al., 2019; Jackson et al., 2020). We will only highlight one interesting link here in rela-
477 tion to grounding line depth. The highest rates of submarine melting are found in up-
478 welling plumes, generated by the emergence of fresh subglacial discharge from the ground-
479 ing line, that flush warm water along the terminus (e.g. Mankoff et al., 2016). The height
480 reached by the plume then determines the vertical extent of the region of high subma-
481 rine melting (De Andrés et al., 2020). At smaller glaciers with shallow grounding lines,
482 the plume is likely to reach the surface, promoting uniform undercutting reaching the
483 fjord surface (Fig. 1b; Carroll et al., 2016) and serac failure. At large glaciers with deep
484 grounding lines, the plume is more likely to be trapped in the subsurface, promoting lin-
485 ear undercutting that does not reach the surface (Carroll et al., 2016) and, therefore, ro-
486 tational failure. As such, the dynamics of submarine melting together with the sensitiv-

ity of calving style to undercut shape may result in contrasts in calving style between small and large glaciers.

4.2 Parameter choices and comparison to previous work

Our results are clearly sensitive to the choice of the ice and bed strength and to the two stress thresholds. Bassis and Walker (2012) and Ultee et al. (2020) contain extensive discussions of shear and tensile failure thresholds, respectively. While the values we have adopted are within the ranges used by previous studies, these parameters are also rather idealised notions that assume the ice and bed are perfect and uniform. In reality, the ice will have crevasses and smaller imperfections and inhomogeneities that will reduce the ability of the ice to resist rotational and vertical imbalances. As such, in a real-world application of our results, such as to form a calving parameterisation, a pragmatic choice would be to choose the values of these parameters to best match observations. A lack of contemporaneous observations of calving and undercut morphology at tidewater glaciers makes this difficult, but some constraints can be obtained.

Approximately linear undercutting has been observed to reach 350 m at Store Glacier (Rignot et al., 2015) in the presence of a water depth of 500 m and an ice thickness of 570 m (Ryan et al., 2015). Such a degree of undercutting would not be stable within our framework unless $\sigma_s^{max} > 0.2$ MPa and $\sigma_r^{max} > 0.8$ MPa (Figs. 3 & 7). The presence of undercutting reaching 220 m at Kangerlussuup Sermia, where the water depth is approximately 250 m and the ice thickness 300 m (Fried et al., 2019) places similar bounds on the stress thresholds. The additional assertion of Fried et al. (2019) that the dominant calving style at Kangerlussuup Sermia is serac failure suggests that the shear stress threshold is smaller than the tensile failure threshold. One very important point to note in comparing our framework to real glaciers is that we have not accounted for across-glacier bridging stresses that could easily play a role in stabilising undercut regions (Cowton et al., 2019).

Our results compare well with previous particle modeling studies in which calving styles emerge naturally rather than through a choice of a stress metric for calving. Benn, Astrom, et al. (2017) simulated a calving multiplier effect of magnitude $\beta = 1$ to 4.6 for an ice thickness $H = 100$ m, fractional water depths of 0.58 to 0.88 and linear undercut lengths $u = 5$ to 15 m. The magnitude of the calving multiplier effect was found to decrease with deeper water. The results of Benn, Astrom, et al. (2017) are, therefore, consistent with our conclusions on the calving multiplier effect in terms of both magnitude and variability, given appropriate parameter choices (e.g. Fig. 10b). Ma and Bassis (2019) suggested that linear undercutting induces a significant calving multiplier effect but uniform undercutting generally reduces the mass of ice that is calved. Lastly, when imposing linear undercutting across the full width of Bowdoin Glacier, van Dongen et al. (2020) predict rotational rather than serac failure. The consistency of our results with all of these studies suggests that the elastic beam approach presented here is a promising avenue for capturing calving due to melt-undercutting in a computationally efficient manner.

4.3 Limitations and missing processes

We have not made much explicit mention of crevasses, yet they are very obviously central to any study on calving. In section 3.1 we described how the existence of a crevasse would modify the threshold for serac failure and indeed we believe that serac failure would be most likely to occur when a crevasse that is formed upstream is advected across the grounding line. Similarly, the existence of a bending moment in the ice that resists top-forwards rotation would deepen surface crevasses, leaving less ice thickness to support the moment and potentially leading to unstable crevasse growth. In some ways, therefore, rather than considering the inception of crevassing, this study focuses on quanti-

537 fying whether the overall geometry of the terminus is conducive to the unstable growth
538 of existing crevasses that leads to calving.

539 A factor not accounted for in this study is ice mélange. It has been suggested that
540 the backstress induced on a glacier terminus by icebergs and sea-ice could be sufficient
541 to suppress calving (Amundson et al., 2010; Krug et al., 2015; Burton et al., 2018; Todd
542 et al., 2018). This suppression could occur by lowering the depth-mean force imbalance
543 at the terminus, thereby reducing longitudinal stretching in the ice. Since mélange is lo-
544 cated at or close to the fjord surface, however, the mélange will also induce a torque on
545 the terminus that opposes the torque resulting from ice and water pressure. As such, mélange
546 could subdue rotational failure and allow a terminus to support greater undercutting.
547 Given a mélange backstress and a range of depths over which it acts on the terminus,
548 it would be simple to incorporate mélange into the depth-integrated torque (Eq. 9) and
549 the critical undercut length for rotational failure (Eq. 10).

550 The limitations and caveats of our study are numerous and lie largely in its ide-
551 alised nature. We have assumed the water depth is fixed and that the ice geometry other
552 than the undercut does not change. In reality, these will evolve in time due to tides, ice
553 dynamics and grounding line advance or retreat; the analysis presented in this study es-
554 sentially assumes that all processes are paused except for undercutting.

555 By employing an elastic beam model, we have neglected any viscous response to
556 the loads applied at the terminus. The viscous bending timescale for glaciers has been
557 estimated to lie between 2 months and 21 years (Sayag & Worster, 2013). Given that
558 the time interval between calving events at tidewater glaciers is typically smaller than
559 2 months (e.g. Schild & Hamilton, 2013; How et al., 2019), treating the glacier response
560 to undercutting as elastic is a reasonable approximation, but it is likely that viscous stresses
561 will play a role. A related point is that we have neglected longitudinal stresses other than
562 those associated with bending. These additional stresses could influence rotational calv-
563 ing or themselves drive calving independently of glacier bending (e.g. Benn et al., 2007).
564 A more complete treatment of the calving problem could, therefore, seek to characterise
565 calving associated with horizontal imbalance at the same time as the calving driven by
566 rotational and vertical imbalance analysed in this study.

567 Lastly, and perhaps most limiting of all, we have neglected across-glacier variabil-
568 ity, which is a significant simplification in view of the evidence that lateral stress bridg-
569 ing is likely to provide support to localised undercutting (Cowton et al., 2019; van Don-
570 gen et al., 2020).

571 4.4 Prospects for a frontal ablation parameterisation

572 The quantitative metrics for serac and rotational failure proposed in this study of-
573 fer a promising step towards parameterisation of these calving styles. In the represen-
574 tation of a grounded tidewater glacier in a depth-integrated ice sheet model (e.g. Morlighem
575 et al., 2016), the ice-ocean boundary is at the grounding line ($x = 0$ in Fig. 1). It is prac-
576 tical, then, to formulate a frontal ablation parameterisation, incorporating the sum of
577 calving and submarine melting, that represents along-flow values at the grounding line.

578 If the glacier and undercut geometry are such that a glacier undergoes serac fail-
579 ure before rotational failure (Eqs. 8 & 10, Fig. 9), then frontal ablation can be represented
580 as a rate that is equal to the submarine melt rate. If instead the glacier and undercut
581 geometry promotes rotational failure, then by combining with a predicted grounding line
582 submarine melt rate, Eqs. 5 & 10 give the timing and distance upstream at which calv-
583 ing occurs. Averaged over many calving events, frontal ablation could in this case be rep-
584 resented as a rate equal to the product of the calving multiplier (Fig. 10) and the ground-
585 ing line submarine melt rate.

586 These calving (or frontal ablation) formulations require only basic inputs such as
587 the ice thickness, water depth, undercut shape and submarine melt rate, together with
588 ice and bed strength parameters, making them suitable for including in large-scale, depth-
589 integrated ice sheet models used for sea level projection. Further work could strengthen
590 these formulations by considering the impact of bed slope, bed friction and ice mélange.

591 5 Conclusions

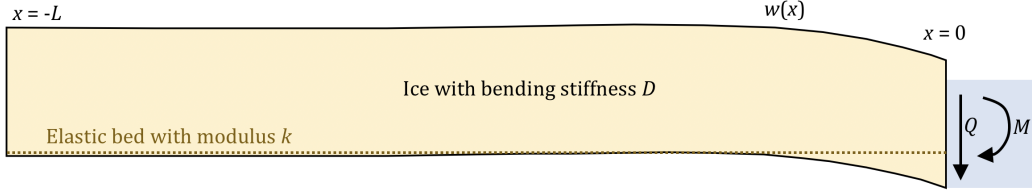
592 We have presented an analytical framework based on elastic beam theory for un-
593 derstanding the impact of submarine melt-undercutting on calving. We have shown that,
594 as undercutting by submarine melting progresses, both the depth-integrated torque and
595 the depth-integrated shear experienced by the terminus increase. Our premise is that
596 serac failure, i.e. calving only of ice that is undercut, occurs when the shear stress ex-
597 ceeds a threshold value, or equivalently when the undercut length reaches a critical value
598 (Eq. 8). Similarly, we suggest that rotational failure, i.e. full-thickness calving of ice up-
599 stream from the grounding line, occurs when the tensile stress at the glacier surface re-
600 sulting from downwards flexure of the glacier exceeds a threshold value, or when the un-
601 dercut length reaches an equivalent critical value (Eq. 10). Within this framework, the
602 dominant calving style and the presence or absence of the calving multiplier effect is de-
603 termined by which critical undercut length is shorter and will, therefore, be reached first.

604 The two critical undercut lengths depend on the ice thickness, water depth and shape
605 of melt undercutting. As a result, glaciers experiencing uniform undercutting are par-
606 ticularly vulnerable to serac failure and experience no calving multiplier effect. Glaciers
607 experiencing linear undercutting are much more stable to serac failure, but are more vul-
608 nerable to rotational failure and can experience a calving multiplier effect whereby sub-
609 marine melting significantly amplifies calving. As such, the geometry of melt undercut-
610 ting determines the presence or absence of the calving multiplier effect. This study pro-
611 vides an intuitive analytical framework to be challenged by further observations and mod-
612 eling, and a promising step towards the parameterisation of melt undercut-driven calv-
613 ing in large-scale, depth-integrated ice sheet models.

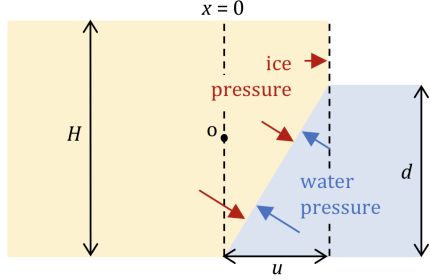
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a elastic beam



b linear undercutting



c uniform undercutting

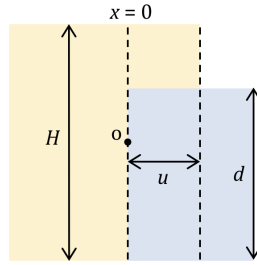
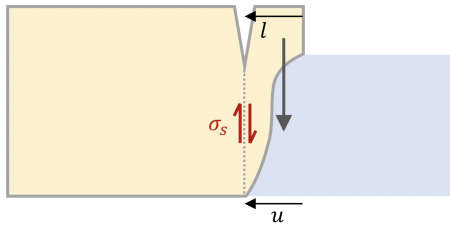


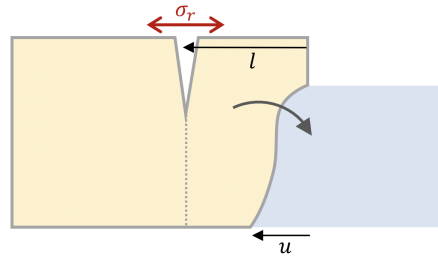
Figure 1. Schematic of the glacier and terminus geometry. (a) A tidewater glacier considered as an elastic beam, with ice/water pressure and undercutting exerting shear and torque on the terminus such that the glacier flexes. The flat profile of the elastic bed shows its position before loading by the glacier. Note that this illustration is vertically exaggerated. The detail at the terminus is shown for the (b) linearly undercut geometry and (c) uniformly undercut geometry. All variables are defined in the text.

a serac failure



when σ_s exceeds $\sigma_s^{max} = 0.5$ MPa

b rotational failure



when σ_r exceeds $\sigma_r^{max} = 1$ MPa

Figure 2. Schematic of calving styles considered. (a) Serac failure is imposed when the depth-mean shear stress at the grounding line, σ_s , exceeds a threshold value taken as $\sigma_s^{max} = 0.5$ MPa. (b) Rotational failure is imposed when and where the longitudinal stress at the glacier surface, σ_r , has a maximum value that exceeds $\sigma_r^{max} = 1$ MPa. As a glacier becomes progressively more undercut at the terminus, the dominant calving style is determined by which of these two thresholds is reached first. The undercut shape shown in these schematics is non-specific; in this study we consider only linear and uniform undercutting (Fig. 1).

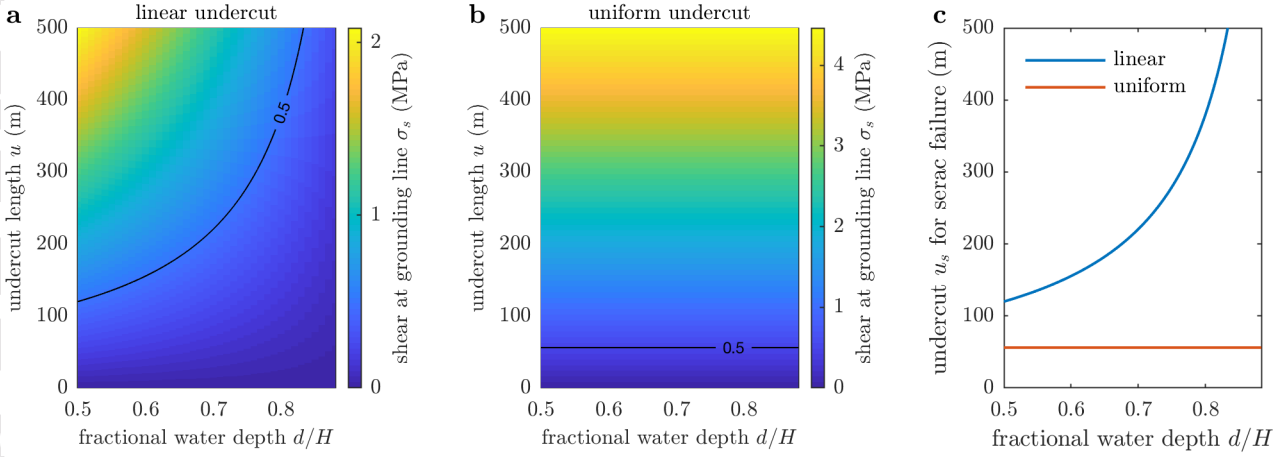


Figure 3. (a) Depth-mean shear stress at the grounding line for linearly undercut ice as a function of fractional water depth and undercut length. The fractional water depth ranges from $d/H = 0.5$ to $d/H = 0.88$ (flotation). The black contour shows the failure threshold $\sigma_s^{max} = 0.5$ MPa. (b) The equivalent depth-mean shear stress at the grounding line for uniformly undercut ice. Note the different colour scale compared to (a). (c) The critical undercut length, u_s , at which the depth-mean grounding line shear reaches the threshold value ($\sigma_s = \sigma_s^{max}$). This is the undercut length at which serac failure occurs.

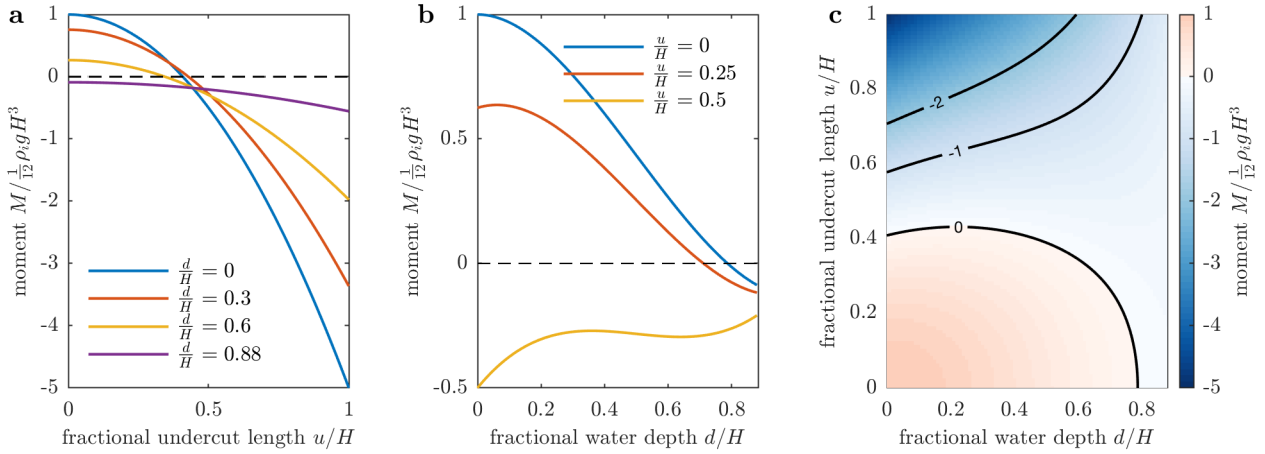


Figure 4. The rotational moment acting on the calving front as a result of hydrostatic ice and water pressure and linear undercutting. Negative values of the rotational moment indicate a calving front that wants to tip top-forwards into the water. (a) The moment as a function of undercutting for various water depths. The moment is normalised by $\frac{1}{12}\rho_i g H^3$, which is its value when the calving front is vertical ($u/H = 0$) and there is no water ($d/H = 0$). (b) The normalised moment as a function of water depth for three undercutting scenarios. (c) The normalised moment as a function of both water depth and undercutting.

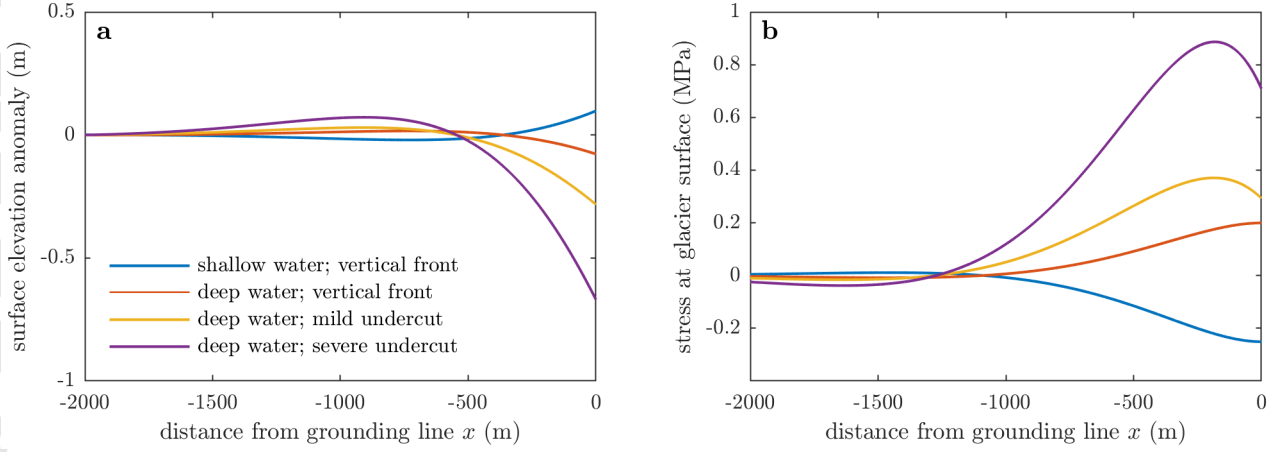


Figure 5. Example solutions of elastic beam model for various water depth and linear undercut scenarios, showing (a) glacier surface elevation and (b) longitudinal stress at the glacier surface. All solutions have ice thickness $H = 500$ m. Examples shown are blue: shallow water $d = 350$ m and a vertical front $u = 0$ m, red: deep water $d = 442$ m (flotation) and a vertical calving front $u = 0$ m, yellow: deep water $d = 442$ m and mild undercut $u = 150$ m, purple: deep water $d = 442$ m and severe undercut $u = 350$ m.

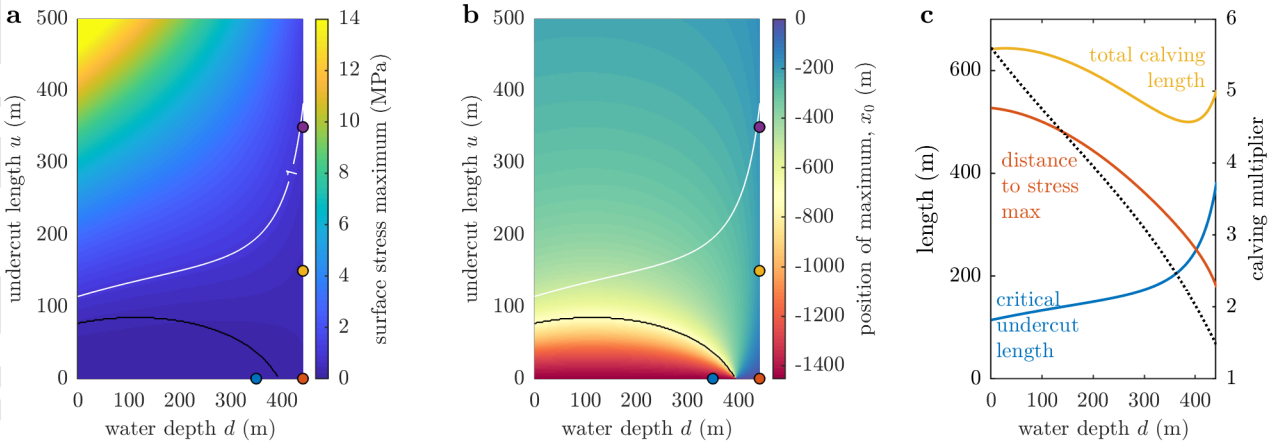


Figure 6. (a) Surface longitudinal stress maximum and (b) location of maximum using the elastic beam model for an ice thickness $H = 500$ m and variable water depth and linear undercut length. The white contours on (a) and (b) show the failure threshold $\sigma_r^{max} = 1$ MPa that also defines the critical undercut length. The coloured markers on (a) correspond to the examples shown in Fig. 5. The black line separates glaciers that flex upwards (below the line) and downwards (above the line) at the terminus. (c) Left axis: the critical undercut length u_r (blue) at which the surface stress maximum hits the threshold value and rotational failure occurs, together with the calving position $|x_0|$ (red) and the total calving length $l = u_r + |x_0|$ (yellow). Right axis and black dotted line: the calving multiplier $\beta = l/u_r$.

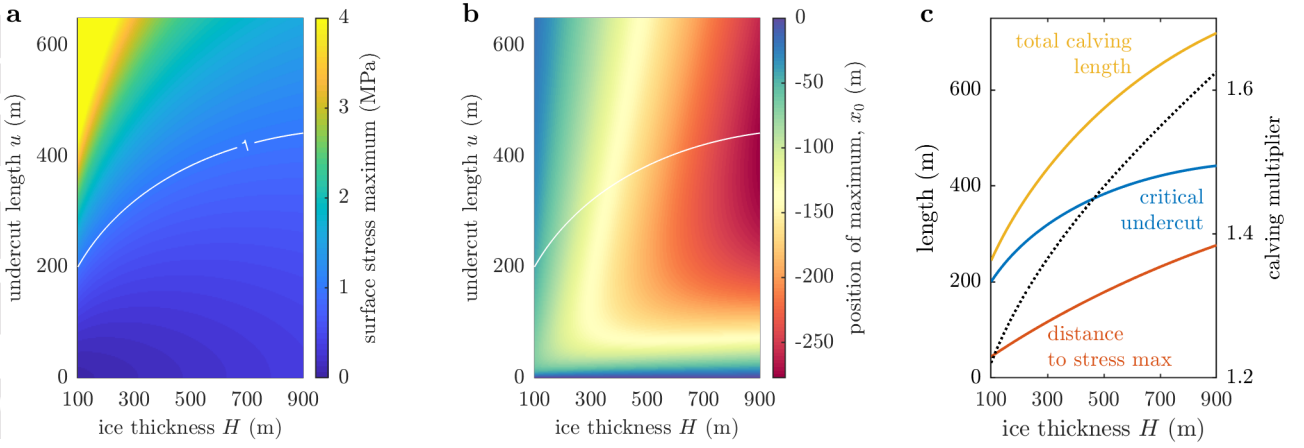


Figure 7. The equivalent of Fig. 6, but considered as a function of ice thickness assuming the water depth is such that the glacier is at flotation. (a) Surface longitudinal stress maximum, and (b) position of surface stress maximum. (c) left axis: the three lengths that determine when and where rotational failure calving occurs, and right axis and black dotted line: the calving multiplier.

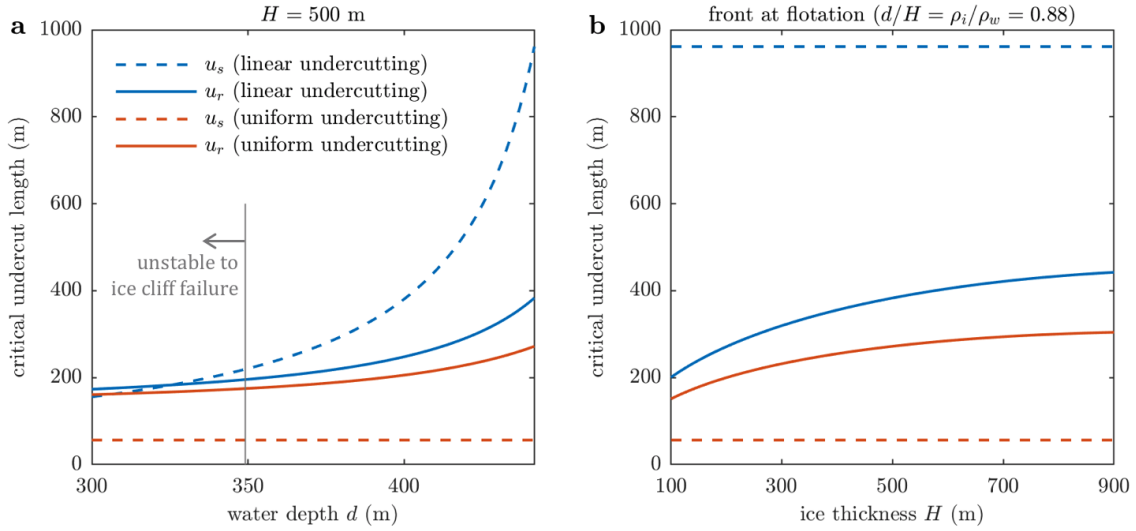


Figure 8. The dependence of calving style on undercut shape. (a) A fixed ice thickness of $H = 500$ m but variable water depth. The grey dashed line shows the minimum water depth required for the front to be stable to ice cliff failure according to Eq. 6. (b) Variable ice thickness, assuming the water depth is such that the front is at flotation.

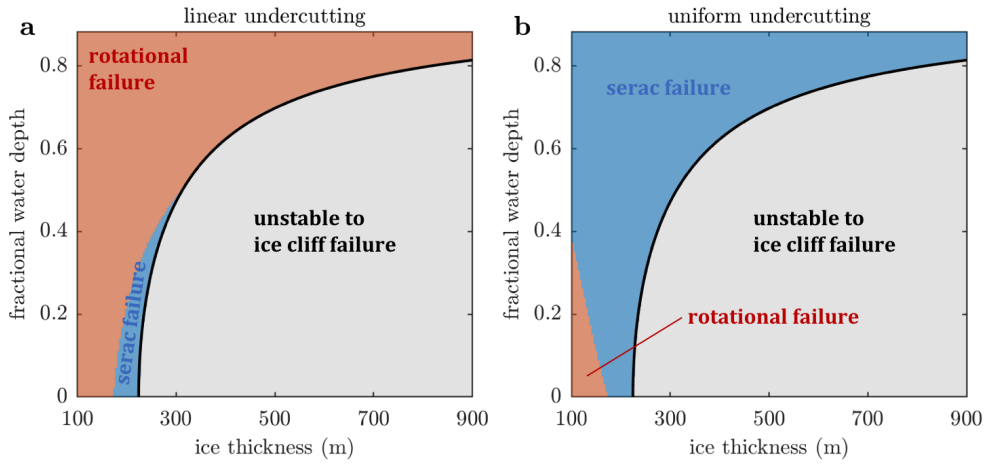


Figure 9. Dominant calving style as a function of fractional water depth and ice thickness for (a) linear undercutting and (b) uniform undercutting.

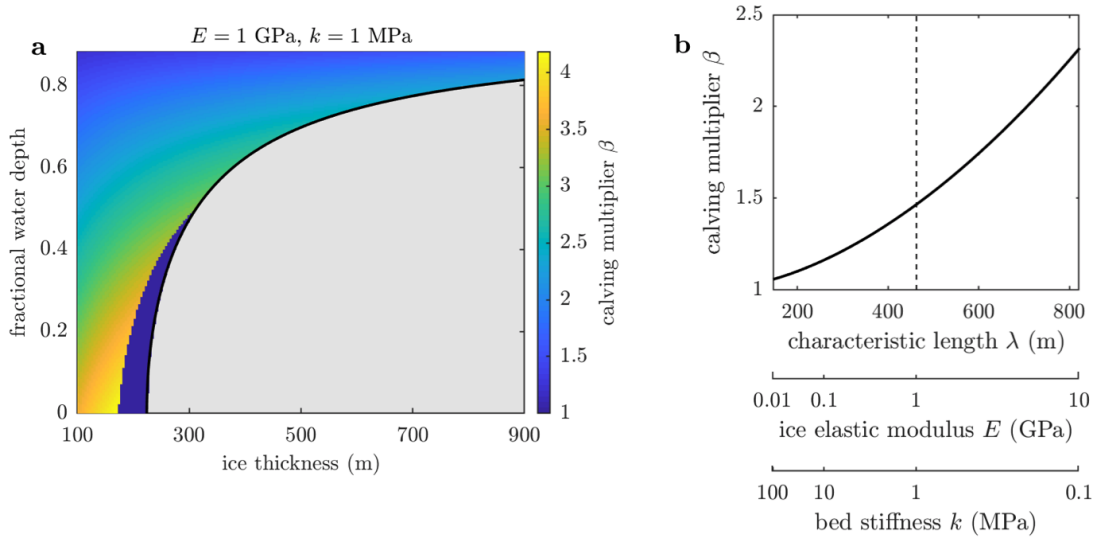


Figure 10. The calving multiplier effect resulting from linear undercutting. The calving multiplier is defined as the ratio of the total calving length (measured at the glacier surface) to the undercut length at the time of calving. (a) Calving multiplier as a function of ice thickness and fractional water depth using the default material parameters for ice and bed strength as throughout the paper. (b) The sensitivity of the calving multiplier to the ice and bed strength, which can also be expressed in terms of the characteristic length. An ice thickness $H = 500$ m at flotation is assumed and the dashed line shows the default parameter values.

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