

UMTRI-83-16

STEERING AND SUSPENSION SYSTEM

Descriptive Parameters Used in Analyzing  
the Braking and Handling of Heavy Trucks

Volume 4  
2nd Edition

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## 1.0 INTRODUCTION

The purpose of this volume is to provide a source of data, describing the mechanical properties of heavy vehicle suspensions. Properties of interest herein are the compliance, kinematic, and Coulomb friction properties which are of interest in the study of vehicle ride, handling and/or braking.

This document contains data derived from the measurement of some 23 heavy vehicle suspensions. All measurements were made using the UMTRI Heavy Vehicle Suspension Measurement Facility. The facility is described in [1]. Data describing the properties of truck and tractor front and rear suspensions, trailer suspensions, and dolly suspensions are included. Tandem and single axle suspensions are included as are suspensions using a variety of springing elements and linkage arrangements.

All of the data provided herein represent measurements of suspension characteristics alone and do not include tire characteristics. That is, referenced linear (e.g., vertical displacement) and angular (e.g., roll angle) displacements are motions of the suspension, wheel, or axle (all measured suspensions are of solid axle configuration) motion relative to the vehicle frame. They are not measurements of body or frame motions relative to the ground plane.

All data presented herein were gathered with the vehicle frame parallel to the ground plane (in pitch and roll) unless otherwise noted. Steering axle data was gathered at a nominally zero steer angle unless otherwise noted.

Section 2.0 of this document contains a discussion of measurement and data reduction techniques. The reader is encouraged to read and understand this material prior to making use of the data presented.

In Section 3.0 of this document, the available suspension data is presented in a suspension-by-suspension format. For each suspension, a "summary sheet" is provided wherein simplified numerics describing the suspension performance are given. This presentation is made in order to provide the reader with data in a convenient, readily useable form. The reader must be aware, however, that a great deal of information regarding the behavior of the suspension is lost, when raw measurement data is reduced

to the simple (usually linear approximation) numerics provided in Section 3.0. Accordingly, for each suspension, the summary sheet is followed by data in unreduced, graphical form as obtained directly from the measurement facility. This presentation retains all of the complex, nonlinear behavioral characteristics of the measured suspensions.

## 2.0 MEASUREMENT AND DATA REDUCTION PROCEDURES

All of the data provided herein represent measurement of suspension characteristics alone and do not include tire characteristics. That is, referenced linear (e.g. vertical displacement) and angular (e.g. roll angle) displacements are motions of the suspension, wheel, or axle (all measured suspensions are of solid axle configuration) motion relative to the vehicle frame. They are not measurements of body or frame motions relative to the plane of the road.

All data presented herein were gathered with the vehicle frame parallel to the ground plane (in pitch and roll) unless otherwise noted. Steering axle suspension data were gathered at a nominal zero steer condition unless otherwise noted.

### 2.1 Loads

Where applicable, referenced suspension loads are defined in accordance with the definitions of "TIRE FORCES AND MOMENTS" given by the SAE in [2]. That is, the terms vertical load, lateral force, brake force, and aligning moment all refer to tire forces and moments. In particular, with respect to compliance steer properties, the measurements presented derive from forces and moments applied relative to SAE tire axis system (and not necessarily relative to the kingpin axis for steering suspensions). Finally, where dual tires are involved, each dual tire pair is considered as one tire, for purposes of applying the SAE definitions. The exception to this general rule is "Roll Moment." Roll moment applied to a given axle is the sum of (1) the couple derived from the difference in vertical loads, side-to-side, plus, (2) the overtuning moments present at each tire of the axle.

### 2.2 Vertical Rate and Coulomb Friction

Figure 2.1 presents a qualitative facsimile of the data derived from a vertical rate test. The ordinate is the average (i.e., per wheel) vertical load applied to the suspension. The abscissa is the average vertical deflection of the wheel spindles of the suspension. The test is conducted under conditions of equal side-to-side loading and zero tire shear forces and moment. As implied by the figure, the suspension is exercised cyclically

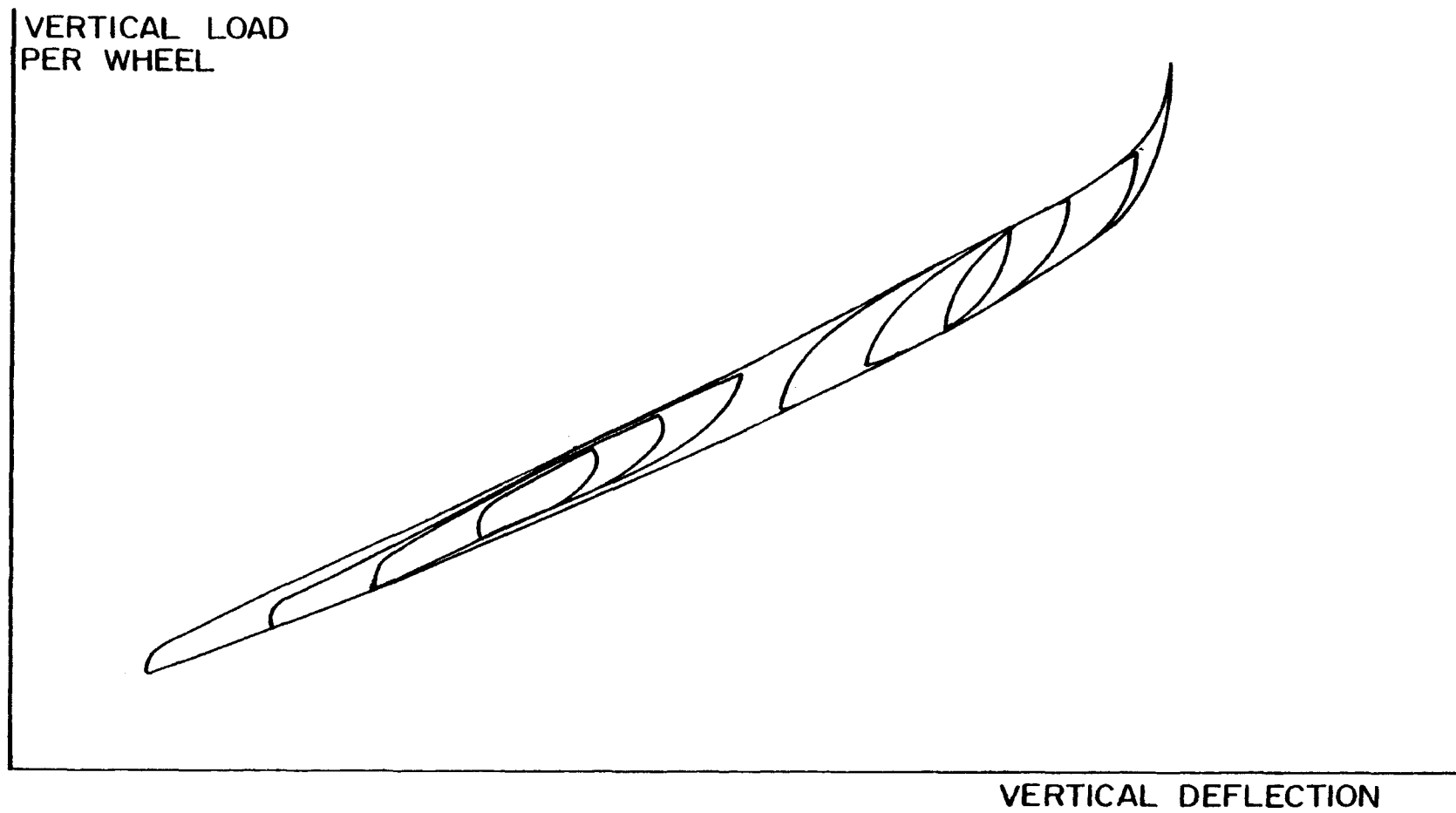


Figure 2.1 Typical leaf spring vertical rate performance.

over various levels of vertical stroke at various nominal vertical loads. Exercising the suspension cyclically over its full range of stroke in a single cycle generates the "outer envelope" curve. (The "tail" at the right end of the curve indicates contact with the axle bump stops.) Exercising the suspension over shorter strokes produces the smaller hysteretic loops.

The numerics reported as "vertical rate" and "Coulomb friction" on the summary sheets are derived from the full stroke, "outer envelope" data. The vertical rate numeric is the average slope of the upper and lower curves (in pounds per inch). Coulomb friction is one half of the ordinate distance between the upper and lower curves (in pounds). These summary numerics are, then, appropriate only for describing the large stroke performance of the suspension. They are not appropriate for short stroke behavior (e.g., as may be involved in ride vibrations). Reference [3] describes two useful methods for deriving representative numerics from the graphical data.

### 2.3 Roll Rate

Figure 2.2 is an example of data describing suspension roll rate. The ordinate represents axle roll moment (in inch-pounds). The abscissa gives axle roll displacement (in degrees). Roll moment data is presented on a per axle basis whether the suspension is single or tandem axle.

Test conditions under which such data are gathered include (1) a constant value of applied vertical load on the suspension, and (2) zero values of tire shear forces and moment.

The example graph shows that suspension roll performance contains much of the hysteretic quality shown in suspension vertical performance. Although not apparent from this example, it should be noted that truck suspensions often exhibit significantly greater roll stiffness than would be implied by a simple model using the vertical spring stiffness and the lateral spring spacing [1]. The example data does show the influence of spring lash. That is, the portions of the curve showing very low roll rate are the result of the lightly loaded suspension spring passing through lash as it goes from compression to tension [4].

The roll rate numeric represents the average slope of the upper and lower, large deflection curves, taken at the zero roll condition.

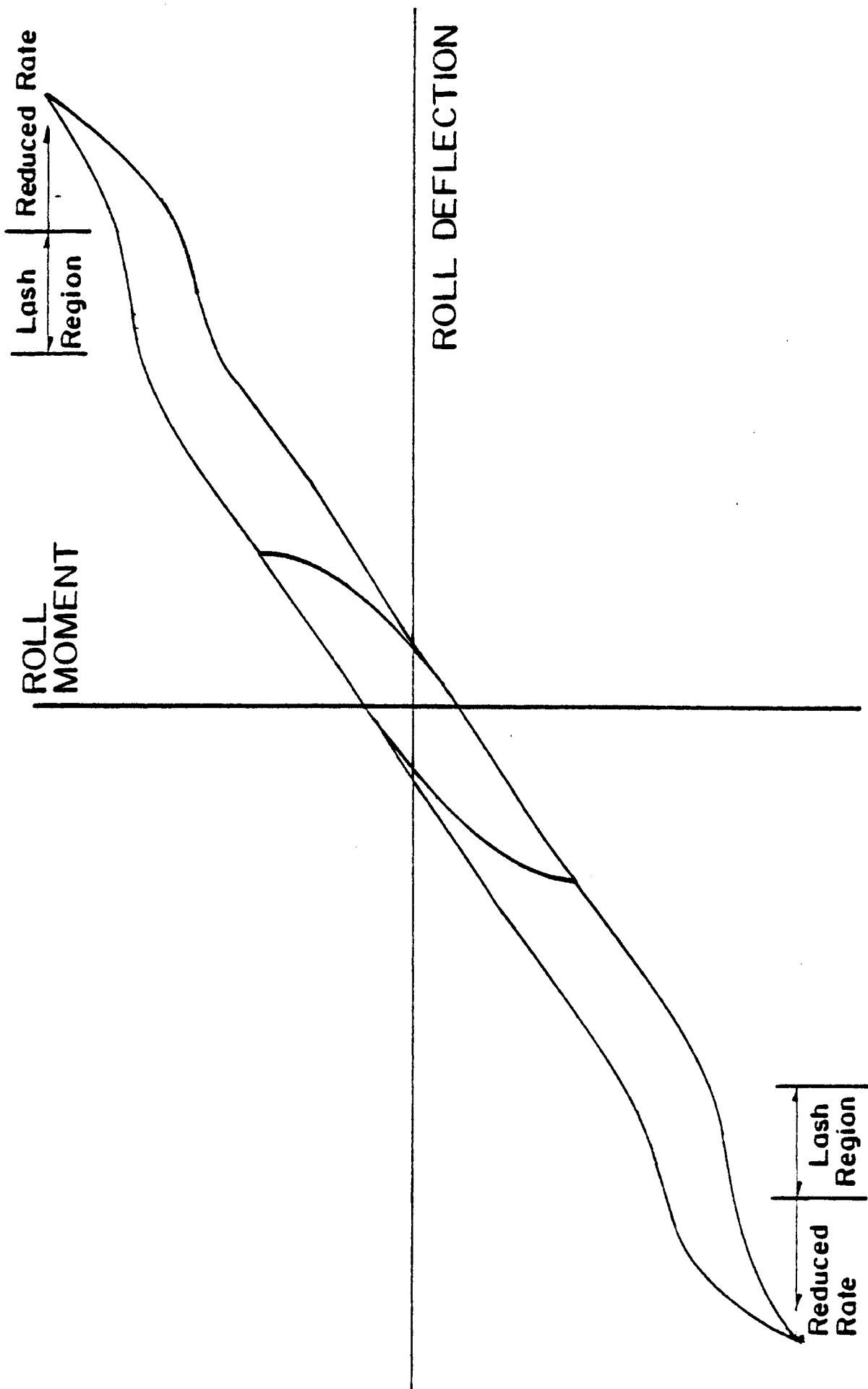


Figure 2.2 Typical leaf spring roll rate performance.

## 2.4 Roll Center Height

Roll center height is measured under the same test conditions described for roll rate. It is presented only as a numeric on the summary sheets. The numeric presented locates the instant center of the axle and frame at the zero roll angle condition. In most cases, the roll center height is given relative to the top surface of the vehicle frame rather than relative to the ground in order to avoid the need to track roll center height as a function of tire rolling radius. In the absence of specific data, the reader might assume frame height to be approximately 42 inches above the ground.

## 2.5 Roll Steer

Roll steer data is collected under the same test conditions as noted for roll rate. The graphical presentations plot road wheel steer angle vs. axle roll angle. For steering axles, separate presentations are made for left and right wheels. For non-steering axles, only one presentation is made, assuming that both wheels of an axle steer equally. For tandem suspensions, separate presentations are made for each axle. In the summary data, the numeric given represents the average slope of the graphical data at the zero roll angle condition.

## 2.6 Compliance Steers

Data describing steer response to three tire shear loads, viz (1) aligning moment, (2) lateral force, and (3) brake force are given in Section 3.0. The following conditions prevail for tests of each of these steer responses. The specified total suspension vertical load is established with equal vertical load side-to-side. Henceforth, the vertical and roll positions of the suspension are held fixed (average vertical and roll positions of the two axles, for tandem suspensions) to avoid contamination of the data by kinematic steer effects. The shear plane load of interest is applied equally at all wheels of the suspension while all other shear plane loads are maintained at zero.

2.6.1 Aligning Moment Steer. Aligning moment steer data is presented graphically in plots of road wheel steer angle (in degrees) vs. applied aligning moment (in inch-pounds per wheel). For steering axles, data is presented



for left and right sides. For nonsteering axles, it is assumed that right and left wheels steer equally; however, data is presented separately by axle for tandem suspensions.

The data is generally well-behaved for nonsteering axles, in that it is quite linear and shows little influence of vertical load. Steering axle response to aligning moment tends to be nonlinear due to hysteresis and the influence of steering system lash around zero applied moment. The summary sheet numeric for aligning moment steer is the approximate slope of the graphical data, ignoring the influence of lash.

2.6.2 Lateral and Brake Force Compliance Steers. It must be candidly noted that the measurement of shear force compliance steers is extremely difficult to accomplish with a high degree of accuracy. *Shear force compliance steer data presented herein (and elsewhere) should be received and employed with caution.*

The experimental difficulty derives from the following: (1) aligning moment compliance steer mechanisms (particularly for steering axles) is generally quite "soft" relative to shear force steer mechanisms, and (2) the accurate maintenance of the orientation of the applied shear force vector along the reference axis of interest (in this case, the SAE tire x- or y-axis) throughout the experiment is extremely difficult. That is, since aligning moment effects tend to generate larger steer responses than do shear force effects, the experiment must take extra care to applying shear forces in the complete absence of aligning moment (i.e., the applied shear force must be oriented on the reference axis of interest), to avoid contamination of the response data. However, since the application of the shear force itself generates steer and translational motions of the road wheel, the necessary shear force orientation is difficult to maintain, let alone establish initially.

Shear force compliance steer data is presented graphically in plots of steer angle (in degrees) vs. applied shear force (in pounds per wheel). Left and right side data are presented for steering axles and the data is presented on a per axle basis for tandem suspensions. Brake force steer is not measured for nonsteering axles. The summary sheet numeric is the linear approximation of the slope of the graphical data.

## 2.7 Tandem Suspension Properties

The following two suspension properties apply to tandem axle suspensions only.

2.7.1 Interaxle Load Leveling. The majority of tandem axle suspensions include a mechanism intended to maintain equal vertical loading on the two axles of the suspension. Data describing this interaxle load leveling function is gathered under the same conditions described for vertical rate, and is presented in the form of plots of leading axle vertical load (in pounds) vs. trailing axle vertical load (in pounds) for both right and left sides. "Perfect" load leveling would then be represented by a slope of unity. No summary numeric is presented.

2.7.2 Interaxle Load Transfer. The same tandem axle mechanisms which provide load leveling under free running generally induce the transfer of load from one axle of the tandem set to the other during the application of longitudinal tire shear force (braking or driving). Data describing interaxle load transfer due to braking is gathered in the following manner. The specified total suspension load is applied with equal (average) load distribution side-to-side and these conditions are maintained throughout the test. Brake forces are then applied, equally at all four wheels of the suspension. Graphical data is gathered in which trailing axle vertical wheel load (in pounds) is plotted on the ordinate, and applied brake force per wheel (in pounds) is plotted on the abscissa. Since total and side-to-side vertical loading are held constant, vertical load changes at the trailing axle are known to be mirrored at the leading axle. Data is gathered for right and left sides separately. The summary sheet numeric is the representative slope of the graphical data in vertical load transferred per side (pounds), per brake force applied per wheel (pounds).

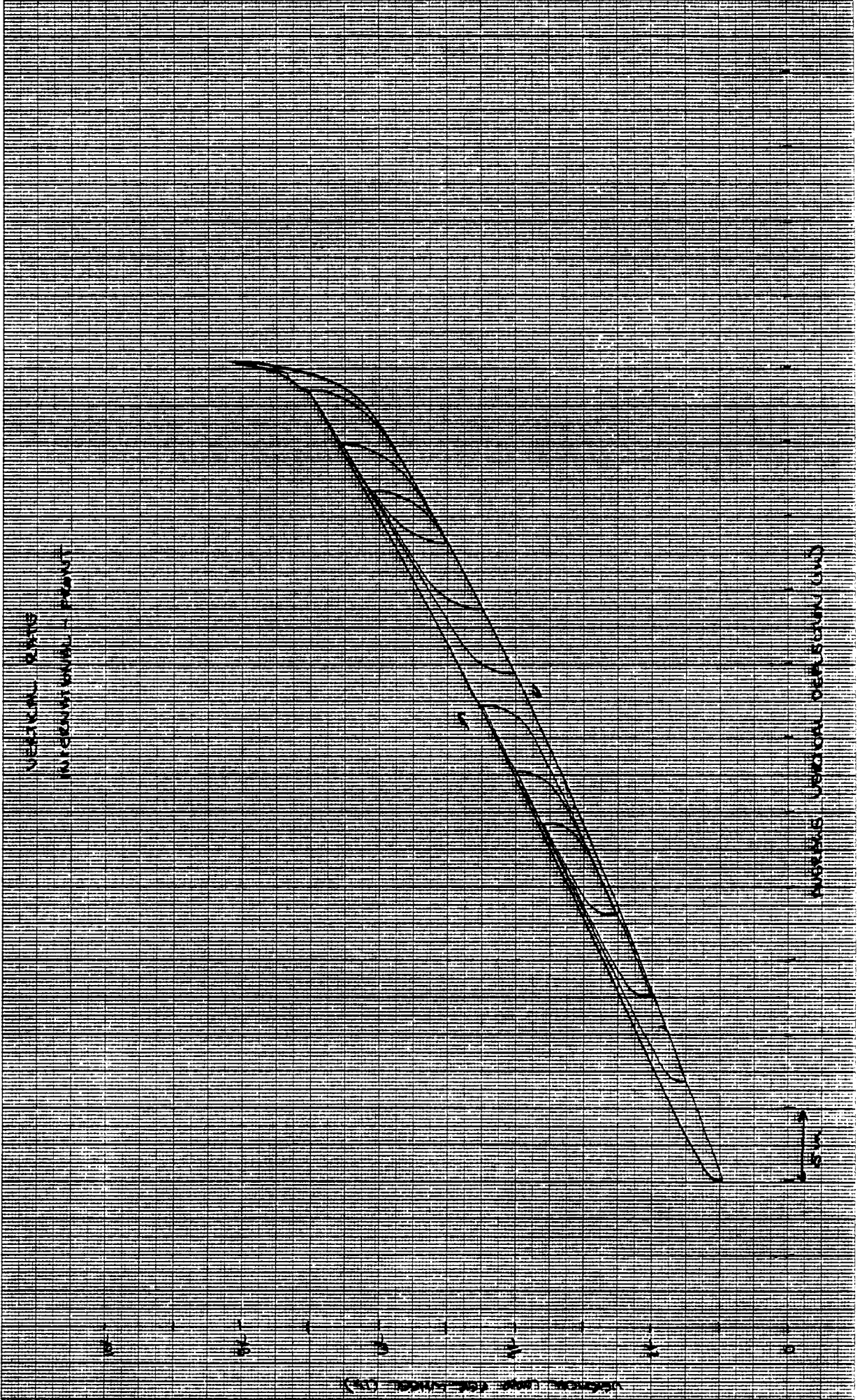
### 3.0 SUSPENSION DATA

This section presents measured suspension properties in reduced and graphical forms. The numerics presented are, in most cases, linear approximations of system performance, at specific operating points as defined by total suspension load. More complete descriptions of measurement and data reduction techniques are given in Section 2.0. The reader is encouraged to read and understand these discussions prior to making use of the data presented in this section.



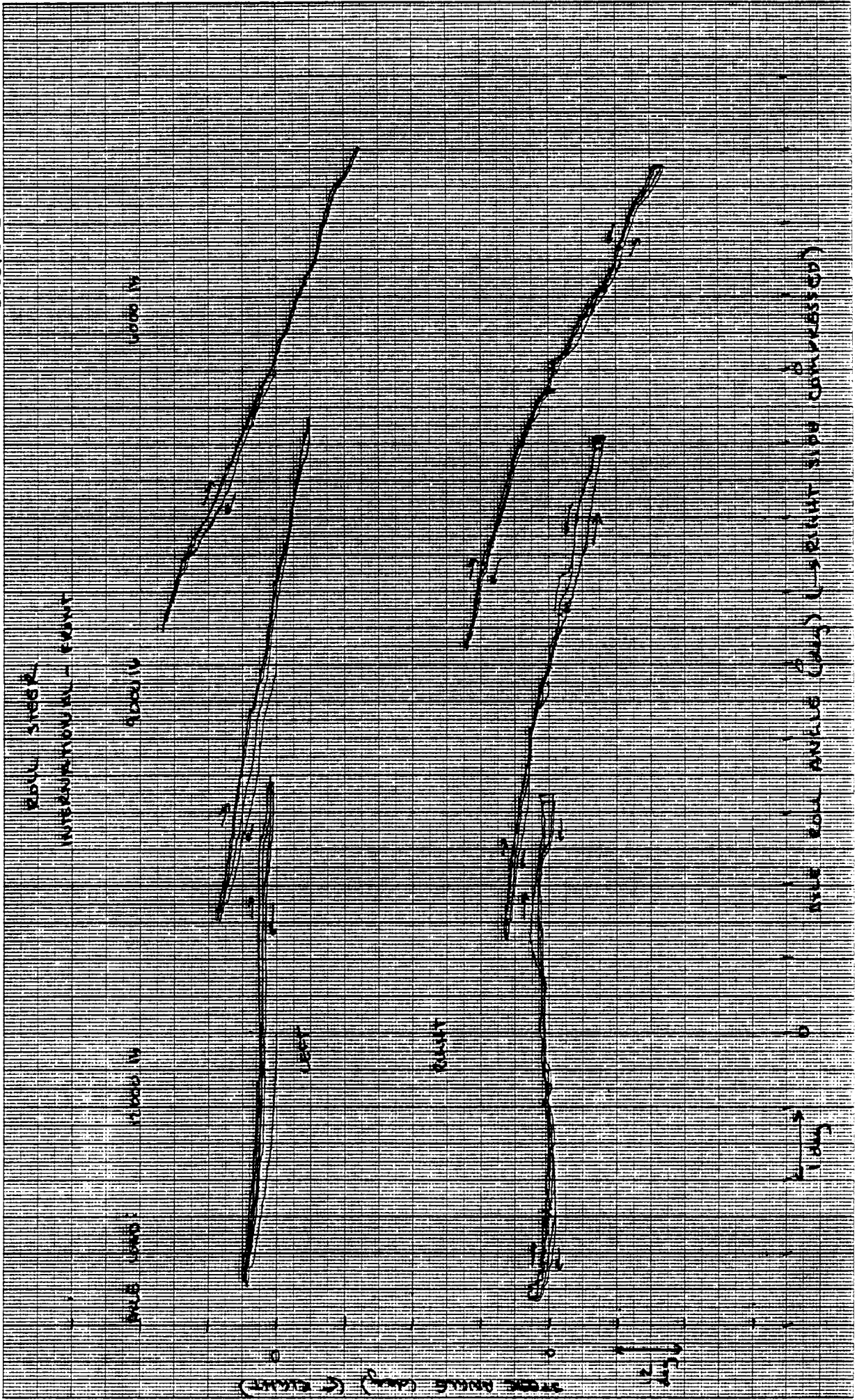
Suspension: International - Front

		Susp Load		
		<u>12000</u>	<u>9000</u>	<u>6000</u>
Vertical Rate (lb/in)		1200	1100	1100
Coulomb Friction (lb)		400	375	325
Roll RATE (in/lb/deg)	F	25K	20K	18K
	R			
Roll CENTER HT (in below top of frame)	F	NA		
	R			
Roll STEER (deg/deg) @ 0° sw	L	nil	-0.04	-0.08
	R	nil	-0.06	-0.10
ALIGNING MOMENT COMPLIANCE STEER (deg/in lb)	L	$5 \times 10^{-5}$	$5.5 \times 10^{-5}$	$5.5 \times 10^{-5}$
	R	$5 \times 10^{-5}$	$6.5 \times 10^{-5}$	$7 \times 10^{-5}$
LATERAL FORCE COMPLIANCE (deg/lb)	R	_____	nil	_____
	L	_____	nil	_____
Brake Force Compliance (deg/lb)	Steer L	$2.2 \times 10^{-4}$ out	$2 \times 10^{-4}$ out	$2 \times 10^{-4}$ out
	R	nil	nil	$1.2 \times 10^{-4}$ out

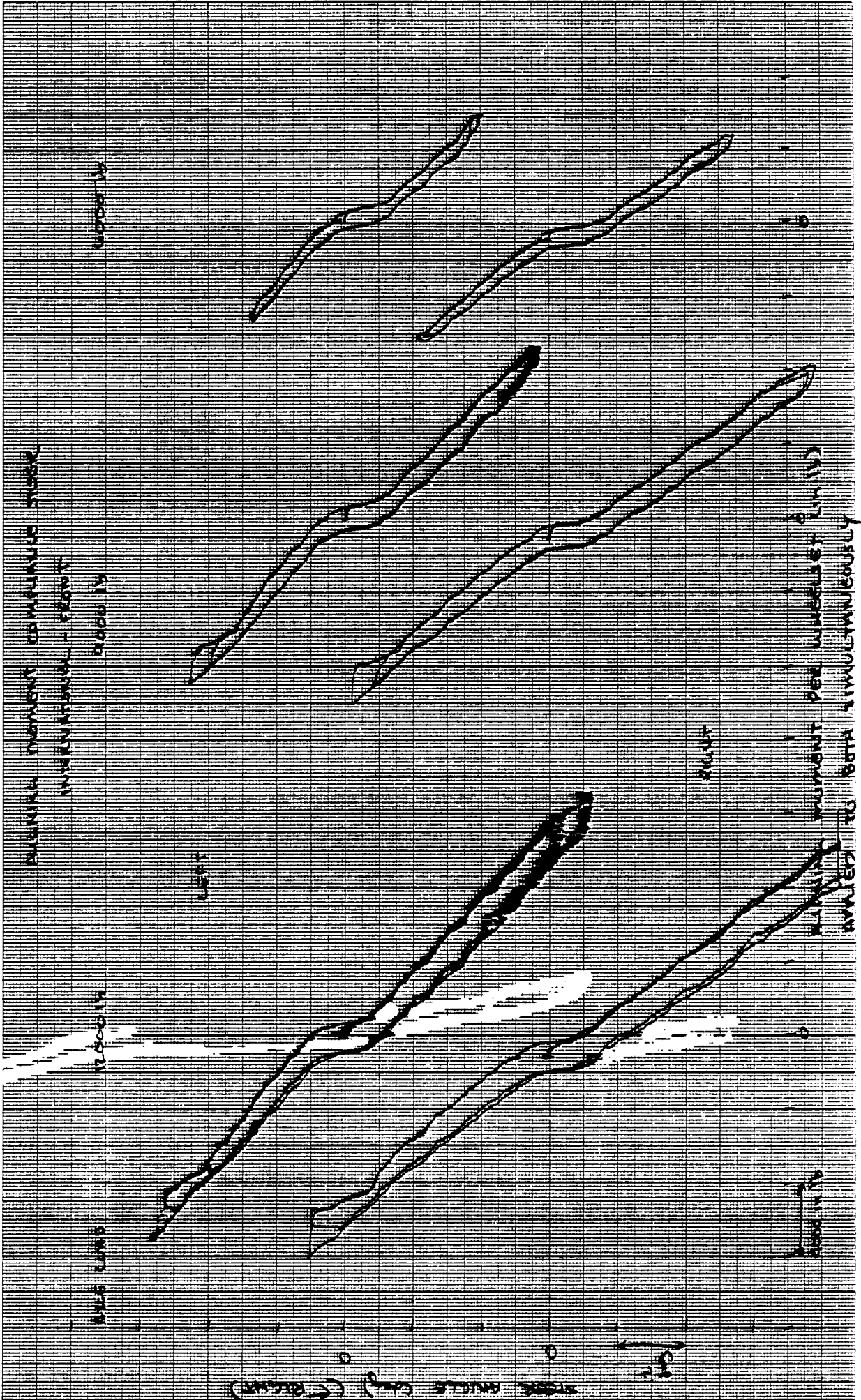


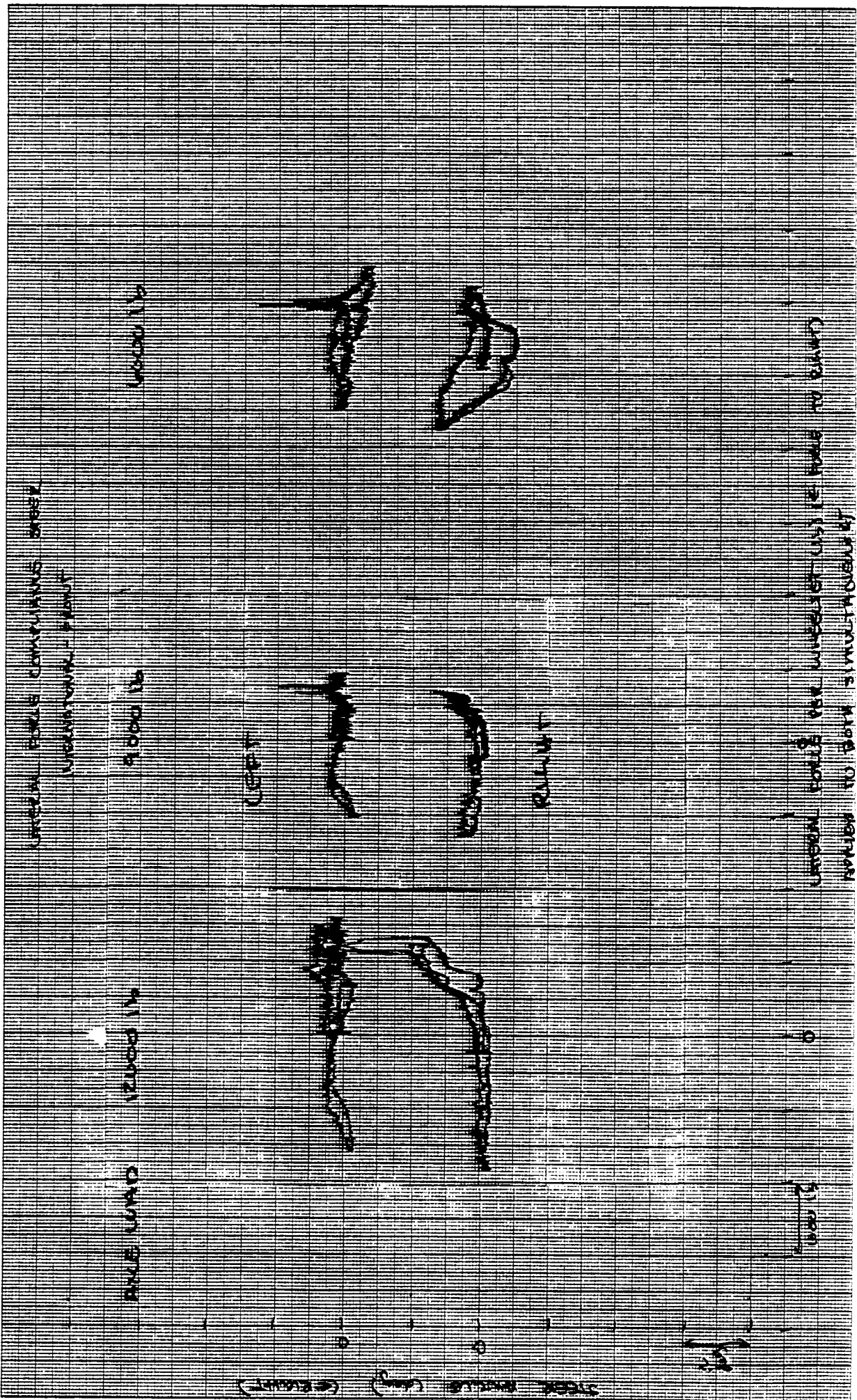


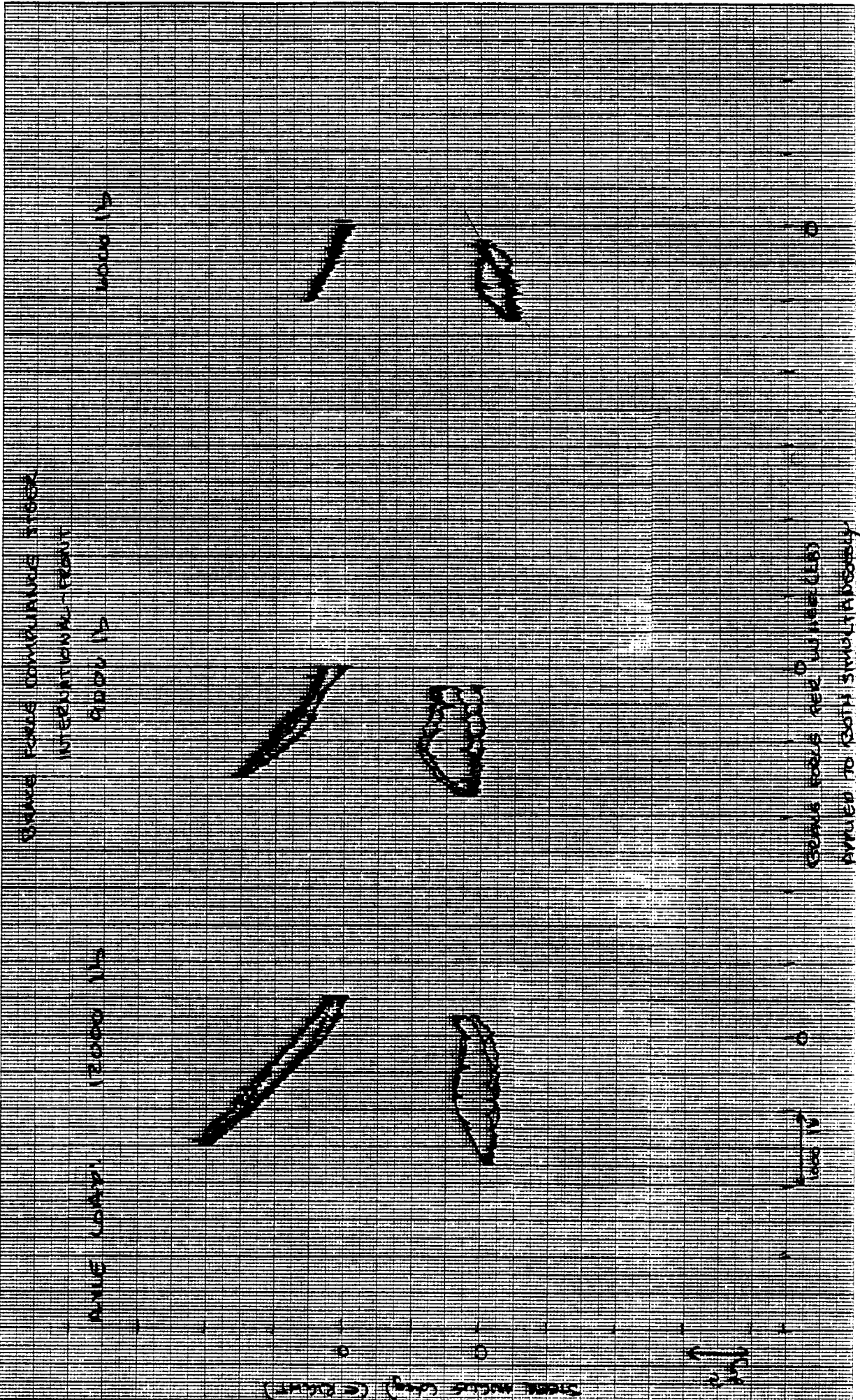
STEER 0°











Suspension: Ford - Front

	Susp Load		
	<u>12000</u>	<u>9000</u>	<u>6000</u>
Vertical Rate (lb/in)	1250	1250	1250
Coulumb Friction (lb)	650	550	400
Roll RATE F (in./lb/deg)	30K	25K	25K
R			

Roll CENTER HT F (in below top of frame)	_____		NA	_____
R				

Roll STEER L (deg / deg) @ 0° SW	.11	.12	.17
R	.10	.12	.17

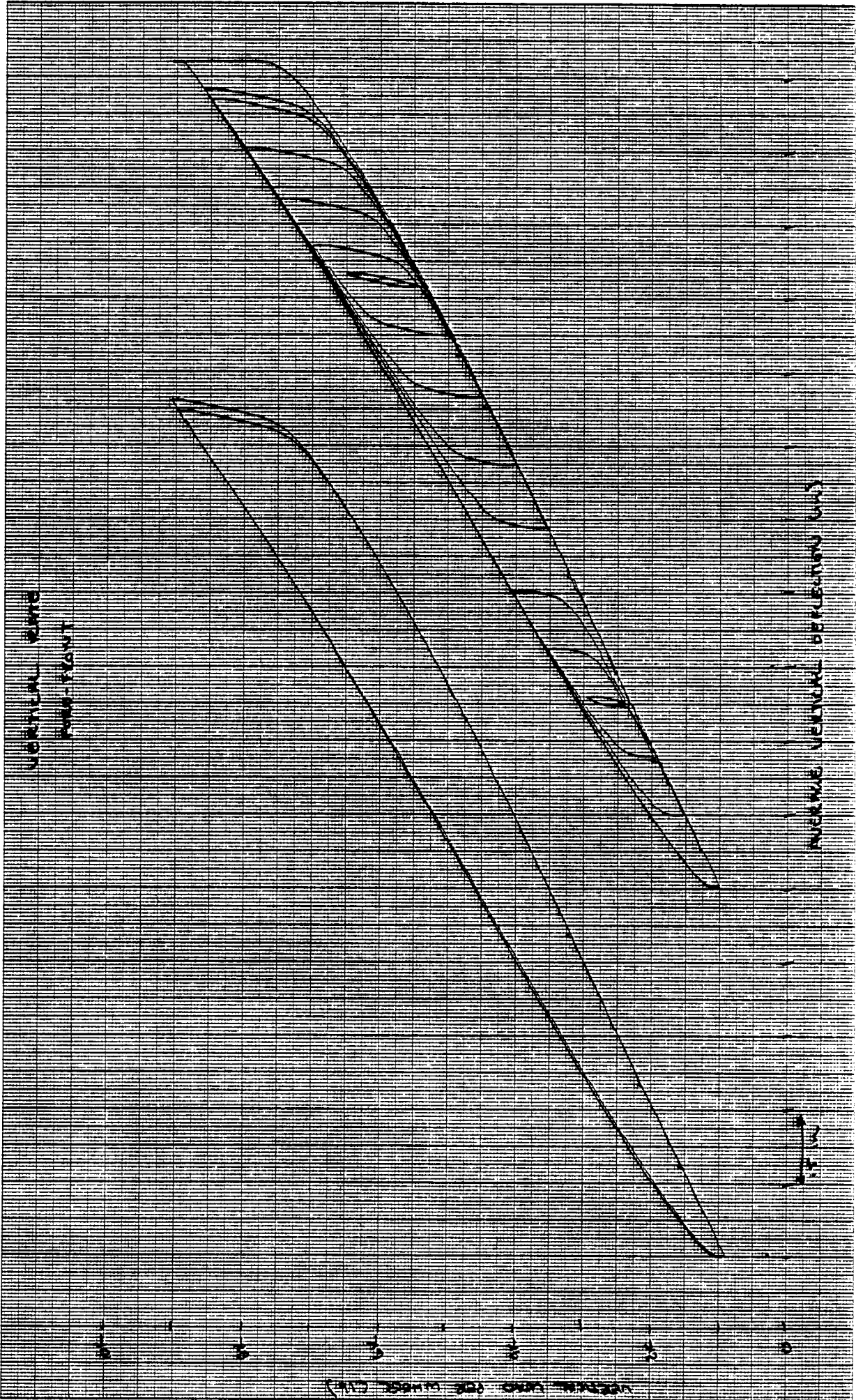
ALIGNING MOMENT COMPLIANCE STEER L (deg in lb)	$1.5 \times 10^{-9}$	$3 \times 10^{-9}$	$3.5 \times 10^{-9}$
R	$4.5 \times 10^{-9}$	$3.8 \times 10^{-9}$	$4 \times 10^{-9}$

LATERAL FORCE COMPLIANCE L (deg / lb)	R
---------------------------------------	---

INTERAXLE LOAD TRANSFER (STEER)  
(lb / lb)

Brake Force Compliance Steer L	_____	nil	_____
R		$2 \times 10^{-5}$	$8 \times 10^{-5}$







NOMINAL STRESS = 0

ROLL STRESS  
FIXED - FRONT

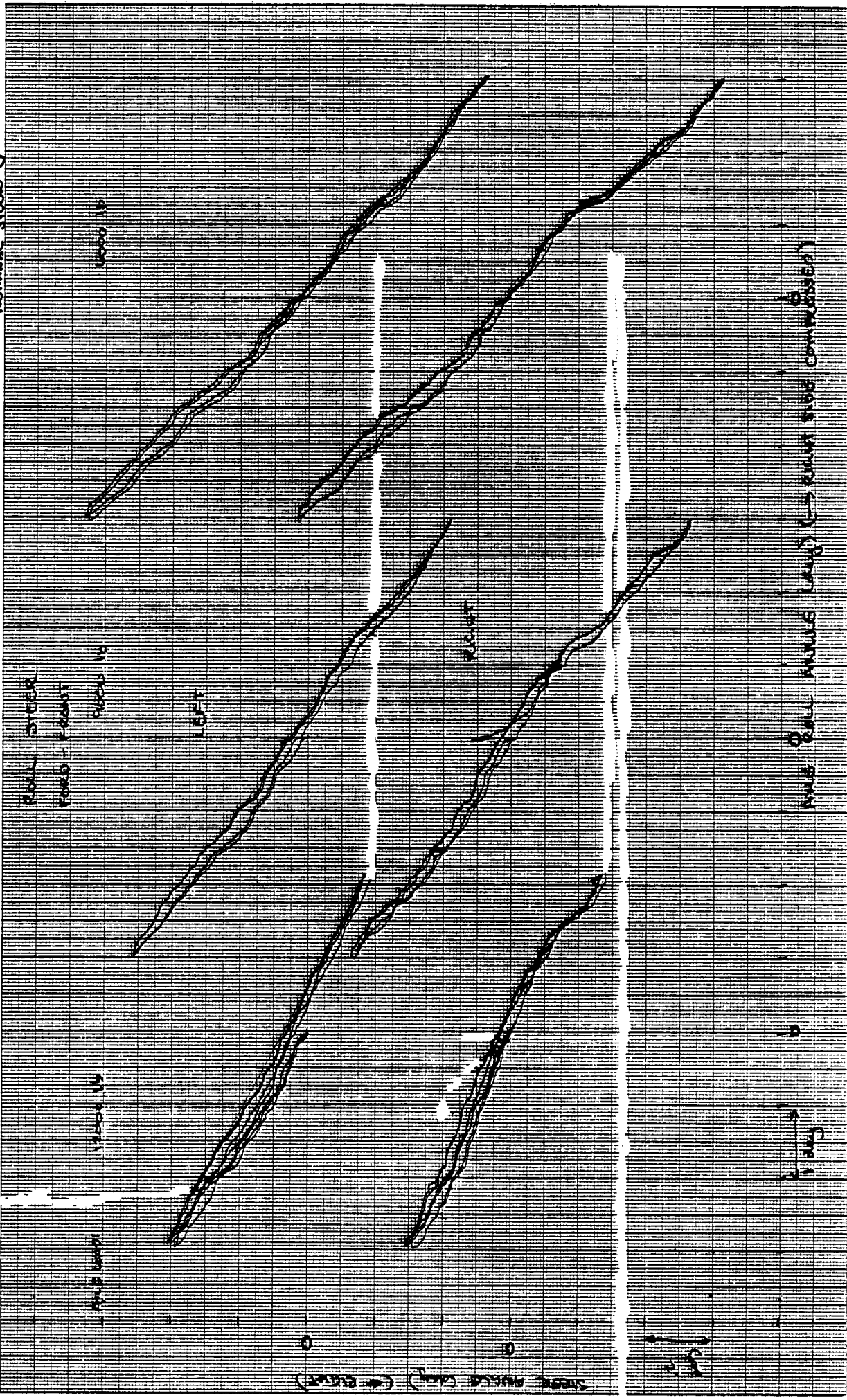
10000 LB

10000 LB

10000 LB

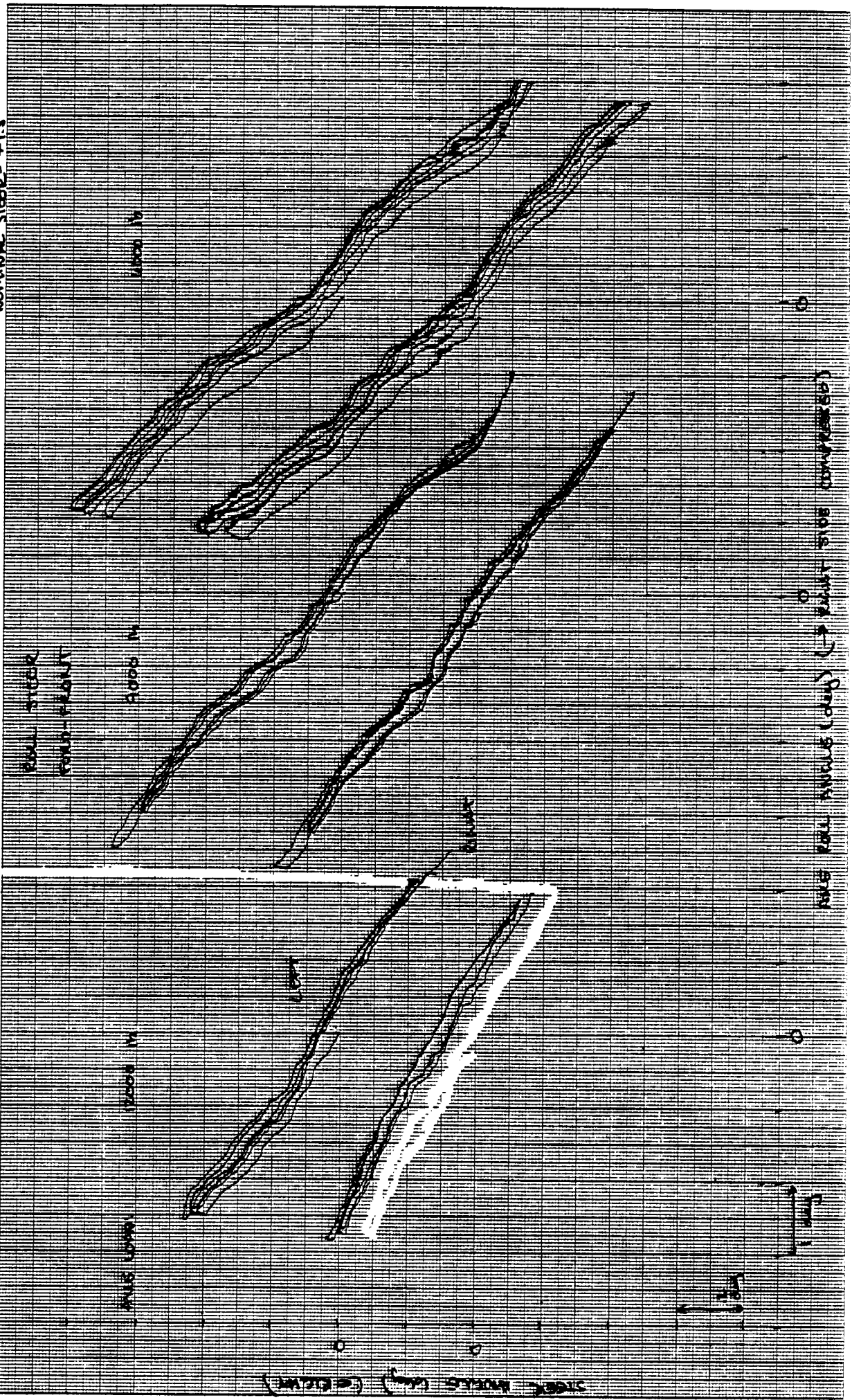
10000 LB

ROLL STRESS (ROLL STRESS COMPRESSED)



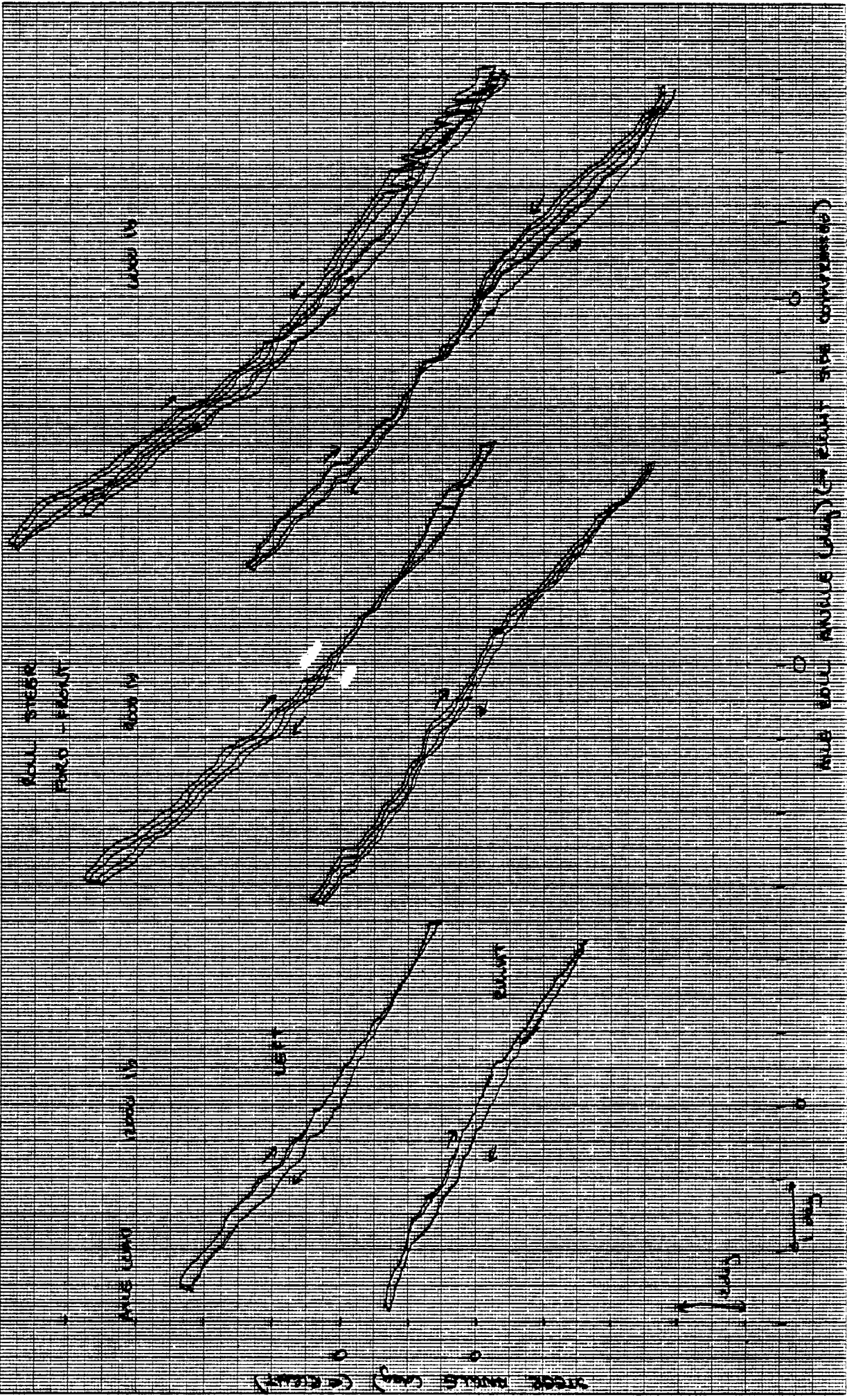


51 + 2825 MINIMUM





NORMAL STEEL = -1.5°





6x

ANTERIOR FORCE COMPLIANCE SKEW  
FORCE FLUCT

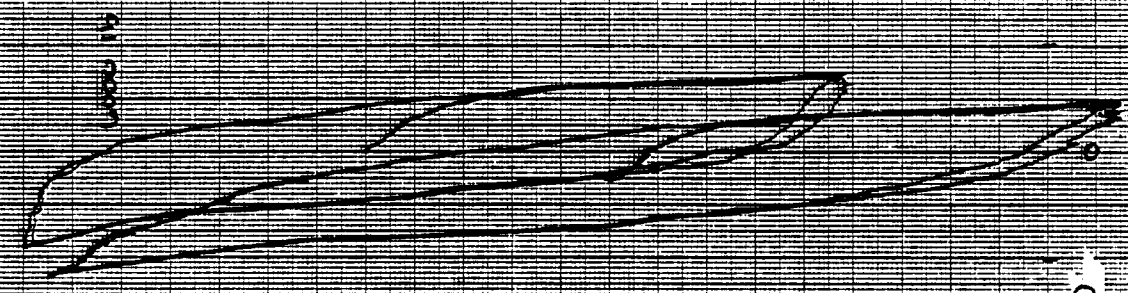
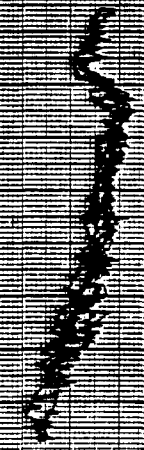
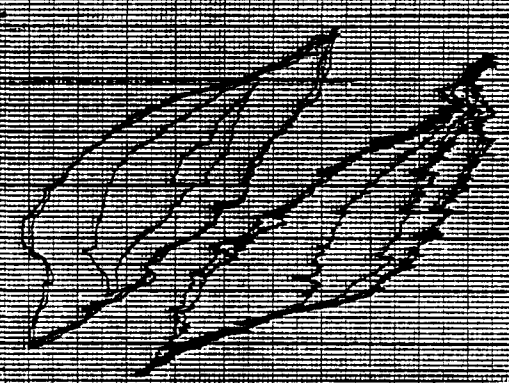
1000 IN

1000 IN

1000 IN

LEFT

RIGHT

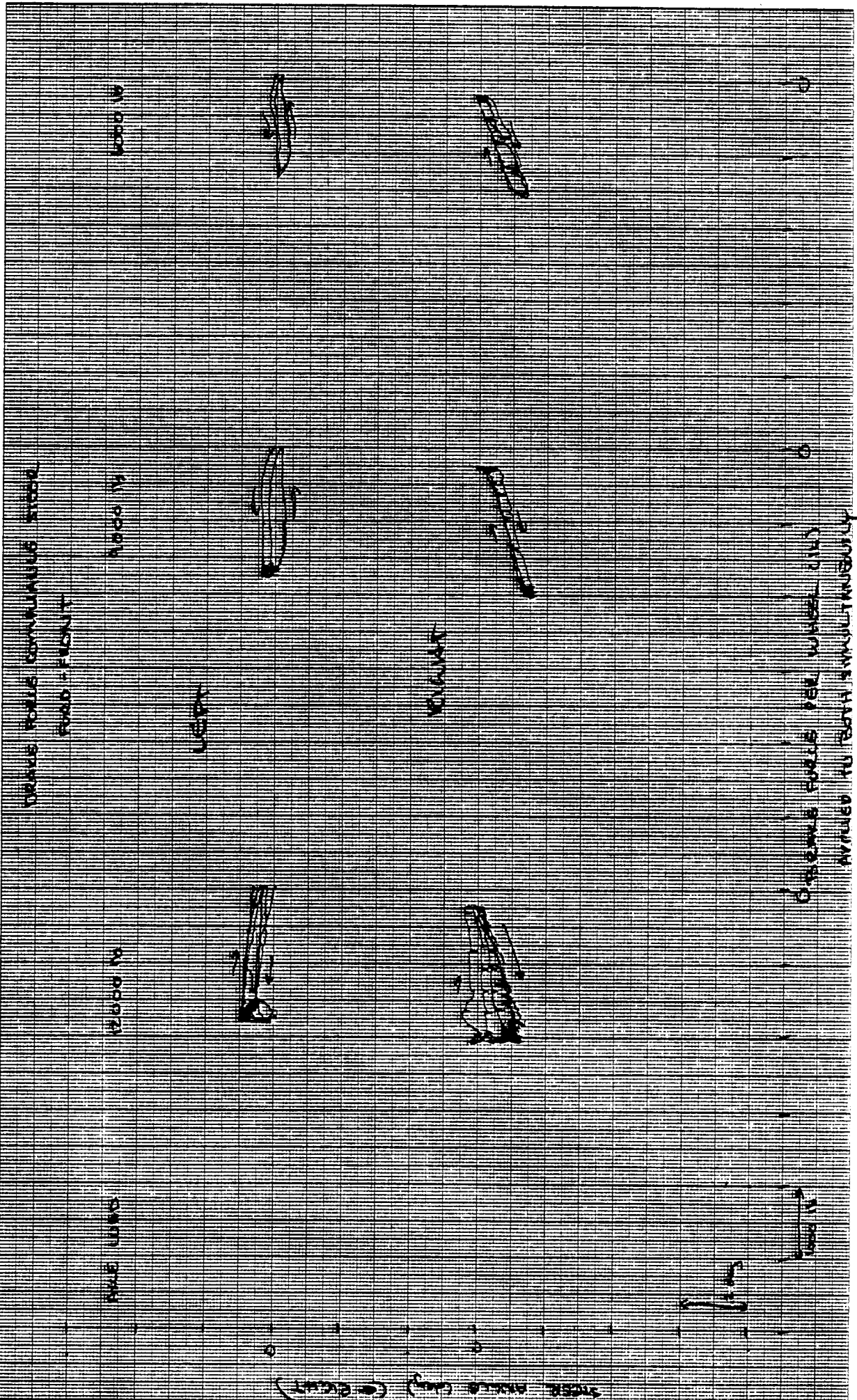


ANTERIOR FORCE (SEE WHEEL AND) (= RIGHT)  
APPLIED TO BOTH SIMULTANEOUSLY

1000 INCHES (LEFT) (RIGHT)

1000 IN





Suspension: Rayco Taper Leaf Front  
 Ford CL9000

Susp Load

12000

10000

8000

tr

Vertical Rate (lb/in)		1000	1100	1100
Coulomb Friction (lb)		300	300	250
Roll RATE (in/lb/deg)	F	36K	28K	28K
	R			

Roll CENTER HT (in below top of frame)	F	_____	NA	_____
	R			

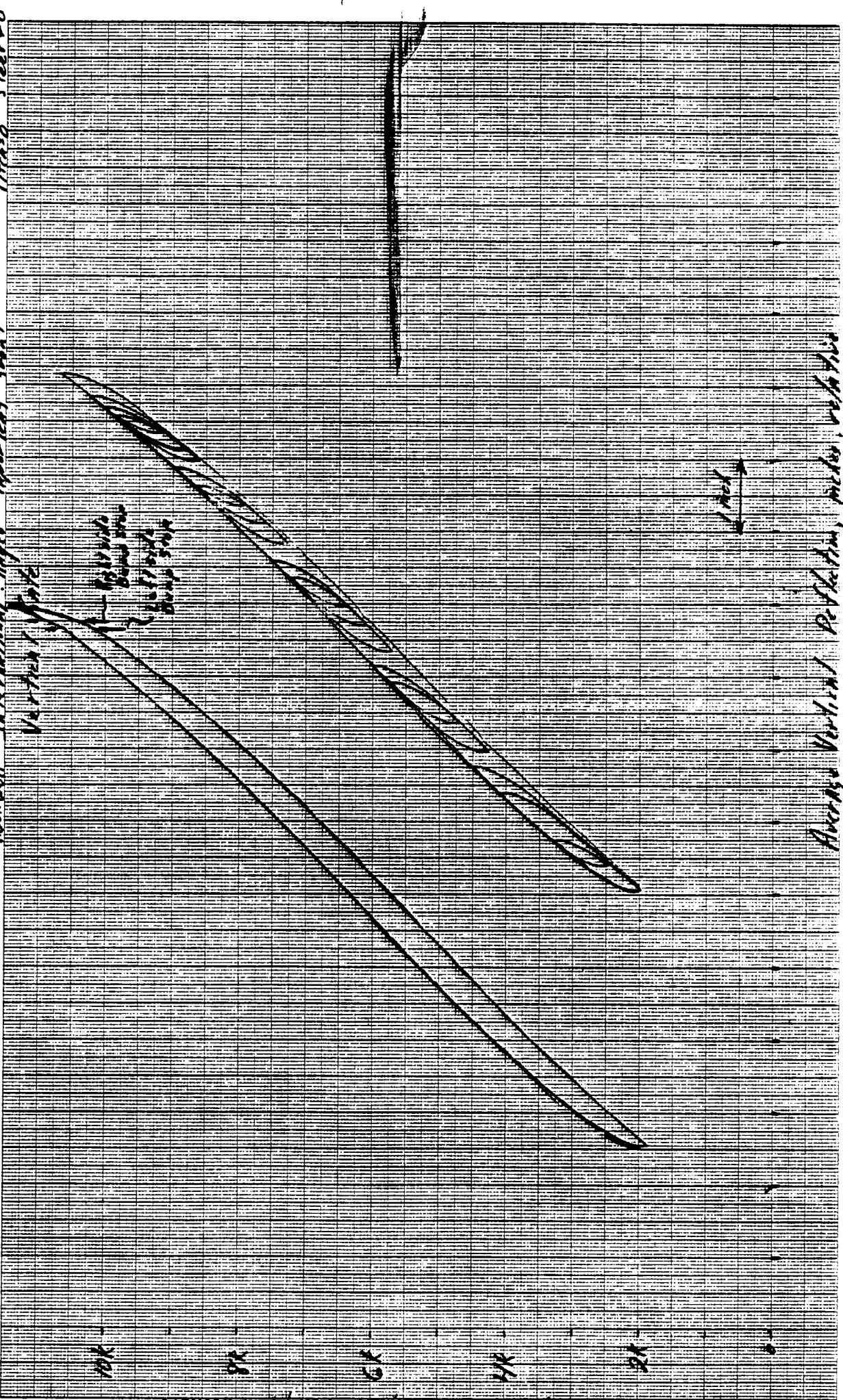
Roll STEER (DEG / DEG)	F	_____	NA	_____
	R			

ALIGNING MOMENT COMPLIANCE STEER (deg/in/lb)	F	NA	$7.0 \times 10^{-4}$	NA
	R	NA	$7.0 \times 10^{-4}$	NA

LATERAL FORCE COMPLIANCE (deg/lb)	F	_____	NA	_____
	R			

INTERAXLE LOAD TRANSFER (STEER)  
 (lb/lb)

Rockwell International: Super Two-leaf front Pitkin Steero



Vertical  
 Pitkin Steero  
 Average Vertical Performance, Pitkin, with 1/2 inch

1 inch

Average Vertical Performance, Pitkin, with 1/2 inch

Vertical load per wheel, lbs. Absolute

Rockwell International: Joyce Turn Leaf Fault

Alpha Station

Case #14  
Date 12/10/86

1000 ft

1000 ft

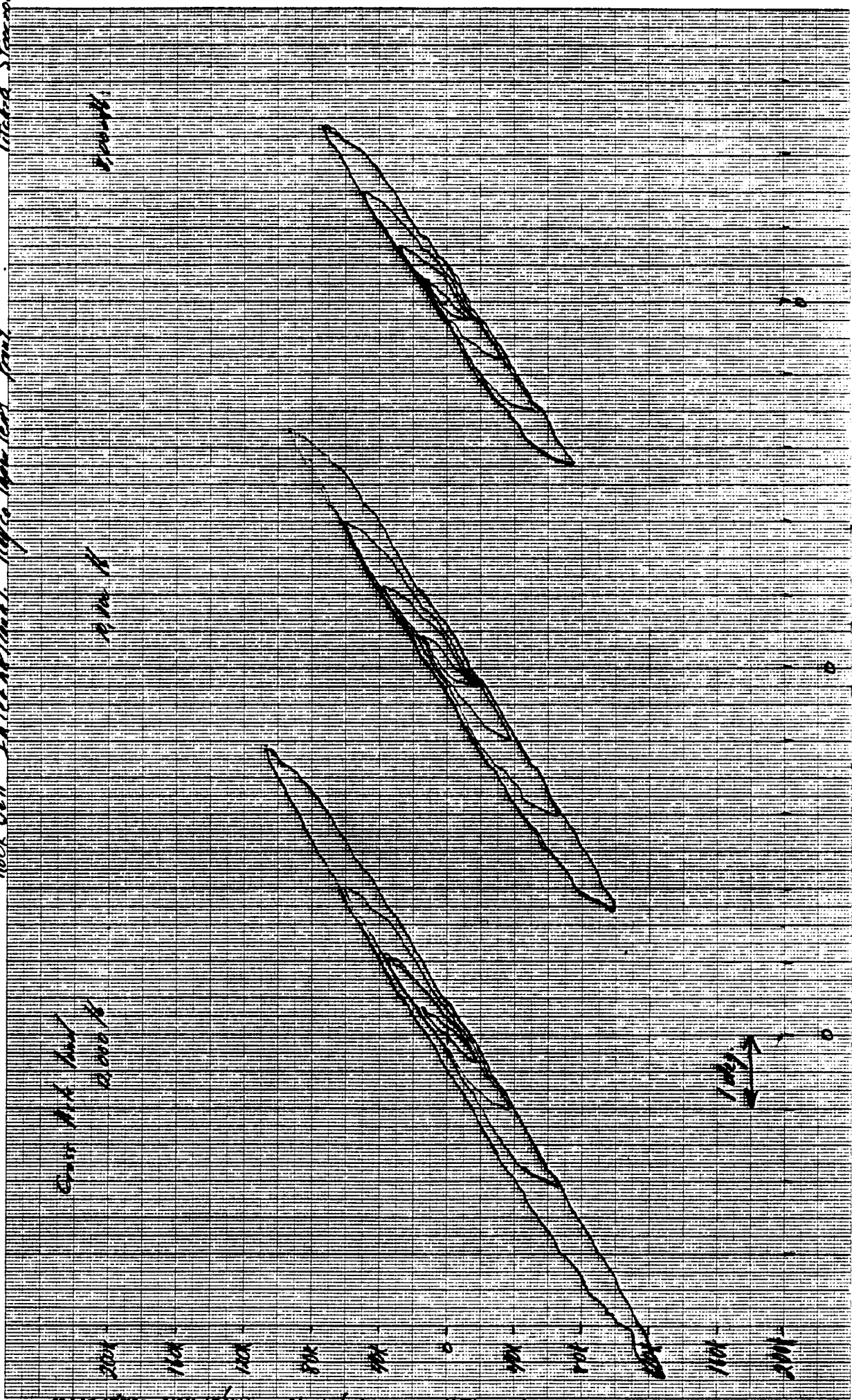
Axle Roll Moment, in-lb (Right side compressed)

2000  
1000  
0  
1000  
2000

1000

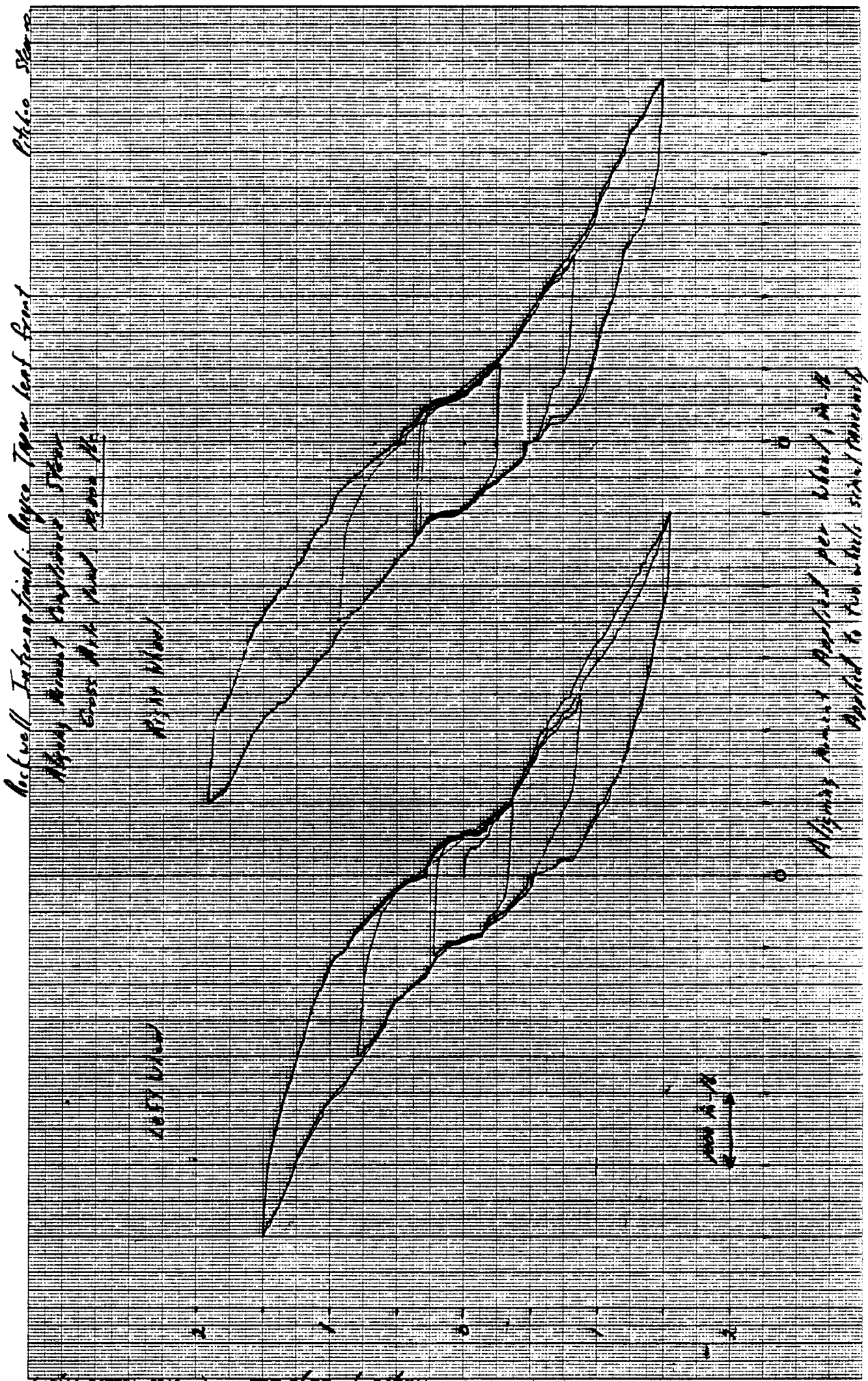
1000  
2000

Axle Roll Angle, deg.





Wheel Steer Angle, degrees (← Steer toward right)



Rockwell Steer

Rockwell Intercontinental - Gyro Topor Leaf Front

Aligns moment wheel, in ft  
Even this view shows it

Right View

Left View

Aligns moment Applied to two wheels, same moment

1000 ft



Suspension: Rayco Multileaf Front  
 Ford CL9000

	Susp Load		
	<u>12000</u>	<u>10000</u>	<u>8000</u>
Vertical Rate (lb/in)	1100	1000	1000
Coulomb Friction (lb)	625	600	550
Roll Rate (in/lb/deg)	F 35K	24K	24K
	R		

Roll CENTER HT F \_\_\_\_\_ N/A \_\_\_\_\_  
 (in below top of frame) R

Roll Steer (deg/deg)	L	.1	-.11	-.13
	R	.17	.10	.16

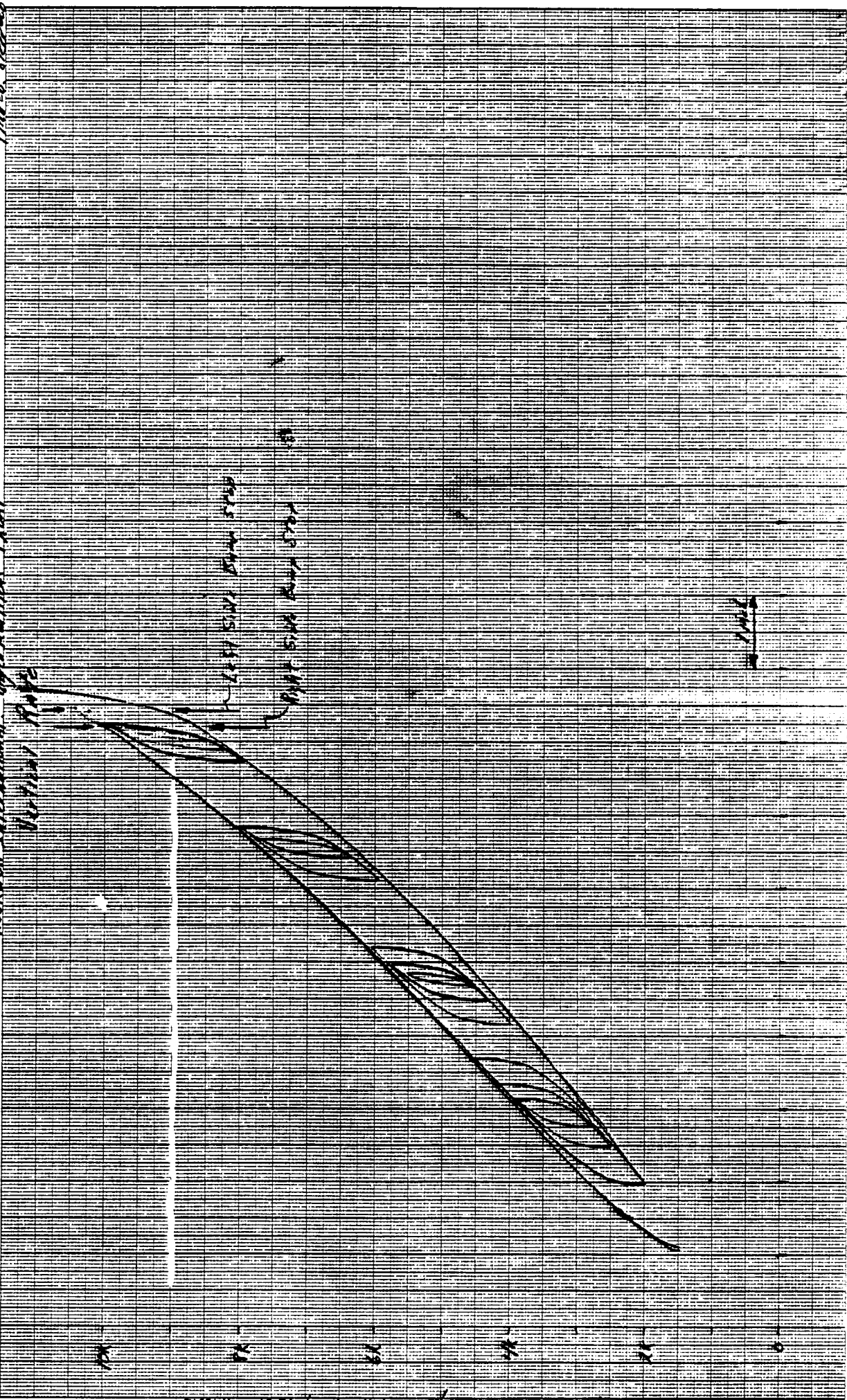
ALIGNING MOMENT COMPLIANCE STEER F \_\_\_\_\_ N/A \_\_\_\_\_  
 (deg/in/lb) R

LATERAL FORCE COMPLIANCE F \_\_\_\_\_ N/A \_\_\_\_\_  
 (deg/lb) R

INTERAXLE LOAD TRANSFER (STEER) (lb/lb)

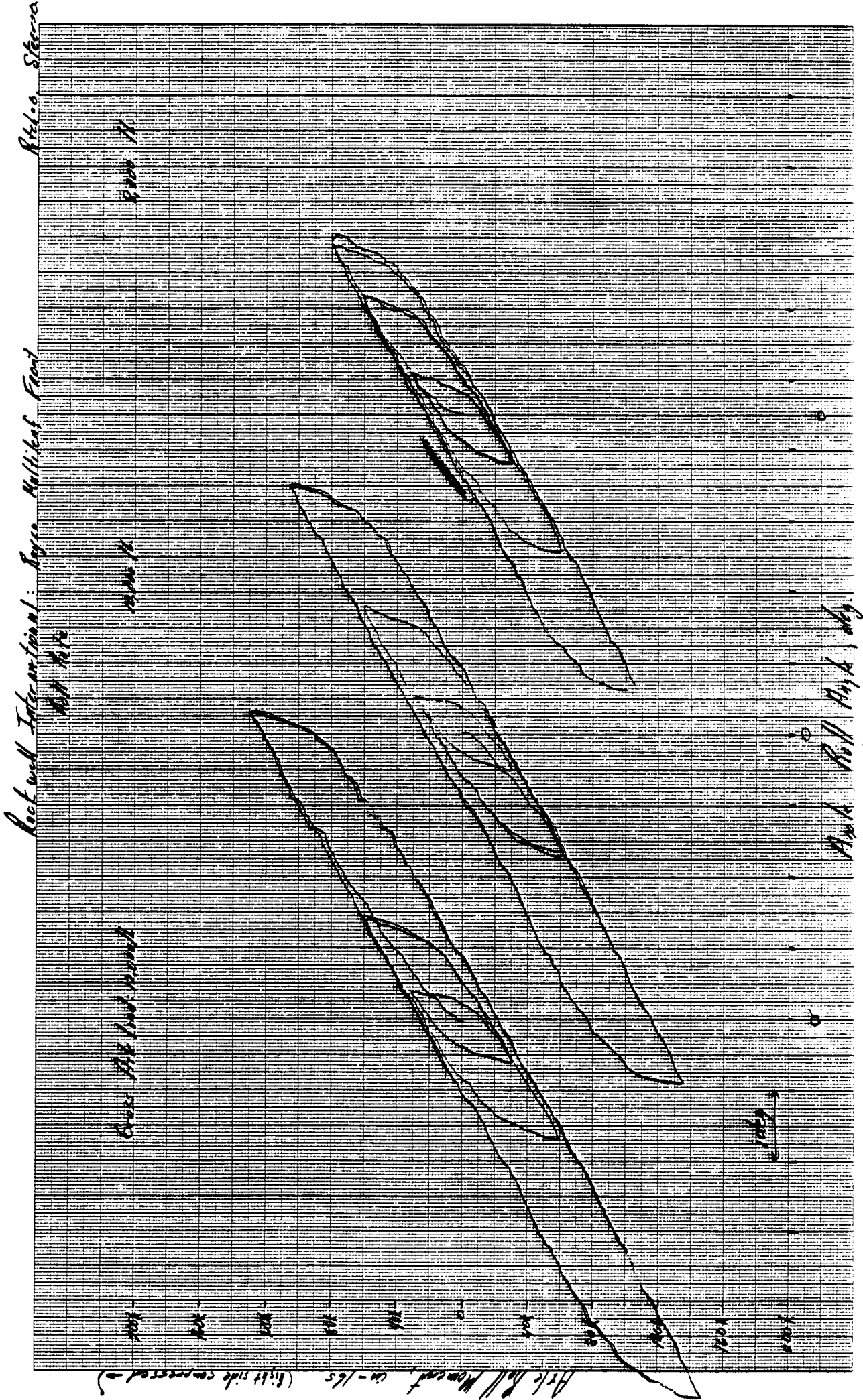
Rockwell International, Open Market, Front

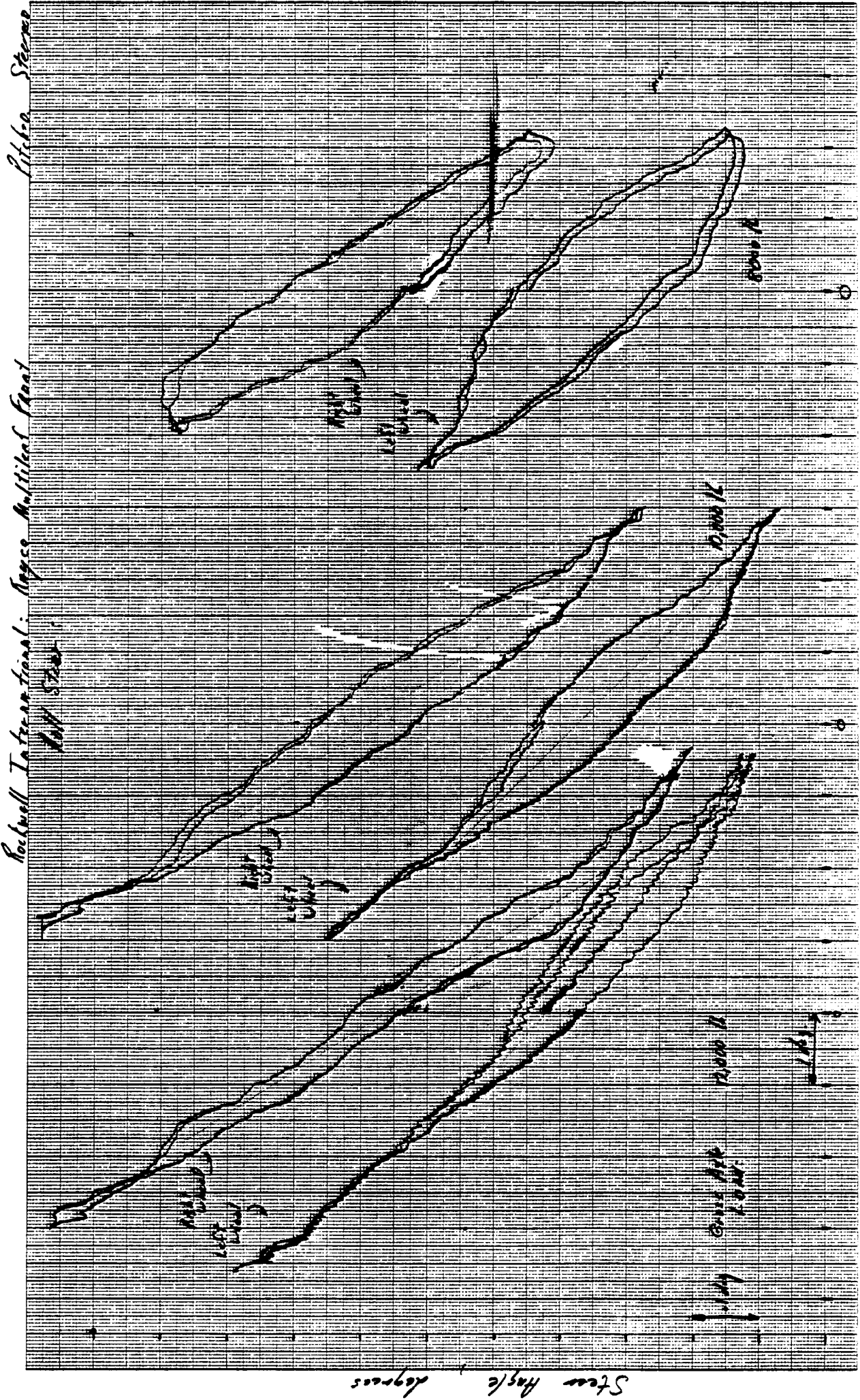
Vertical Probe



Average Vertical deflection, inches, relative

Vertical Load per Wheel, lbs. Absolute





Angle Roll Angle, degrees



Suspension: IH Single

		Susp Load (lb)		
		<u>19000</u>	<u>14250</u>	<u>9500</u>
Vertical Rate (lb/in)		4400	3400	2000
Coulomb Friction (lb)		1250	1175	900
Roll RATE	F	92K	80K	50K
(in-lb/deg)	R	<hr/>		

Roll CENTER HT	F	<hr/>	NA	<hr/>
(in below top of frame)	R	<hr/>		

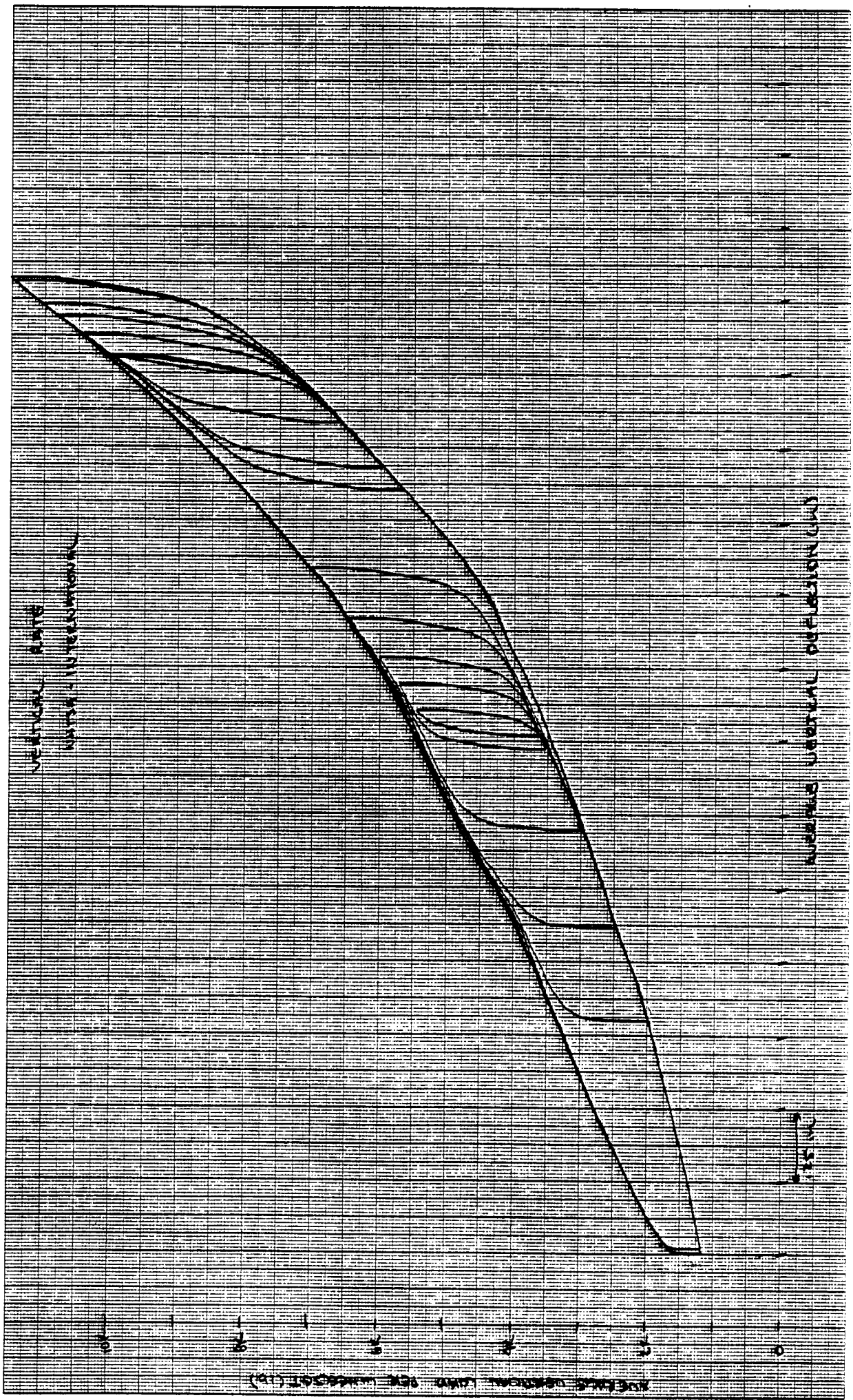
Roll STEER	F	nil	nil	.05
(deg / deg)	R	<hr/>		

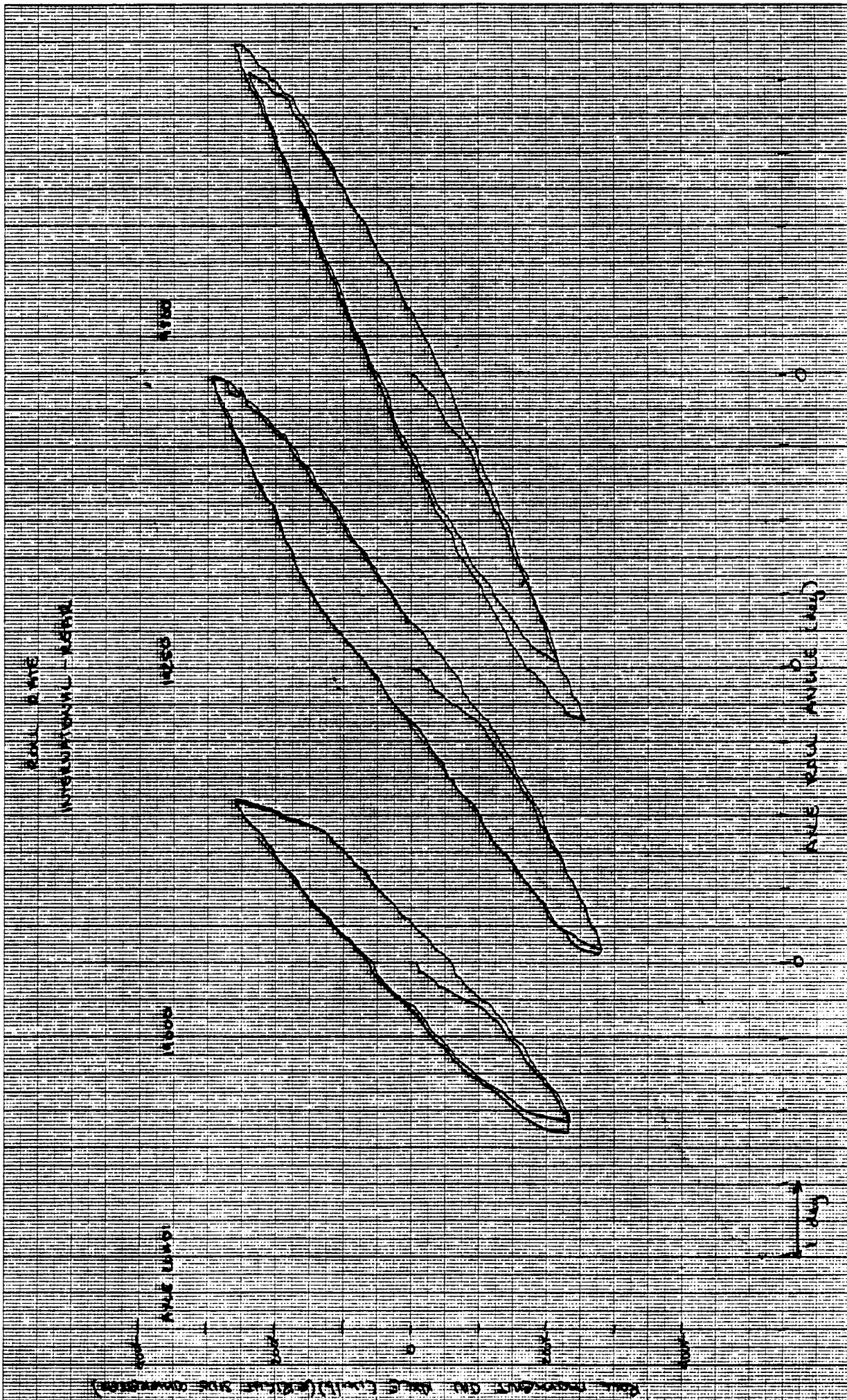
ALIGNING MOMENT COMPLIANCE STEER	F	$5.9 \times 10^{-5}$	$6.3 \times 10^{-5}$	$7.1 \times 10^{-5}$
(deg / in lb)	R	<hr/>		

LATERAL FORCE COMPLIANCE	F	$3.1 \times 10^{-4}$	$4.3 \times 10^{-4}$	nil
(deg / lb)	R	<hr/>		

INTERAXLE LOAD TRANSFER (BANK)		<hr/>	NA	<hr/>
(lb / lb)		<hr/>		

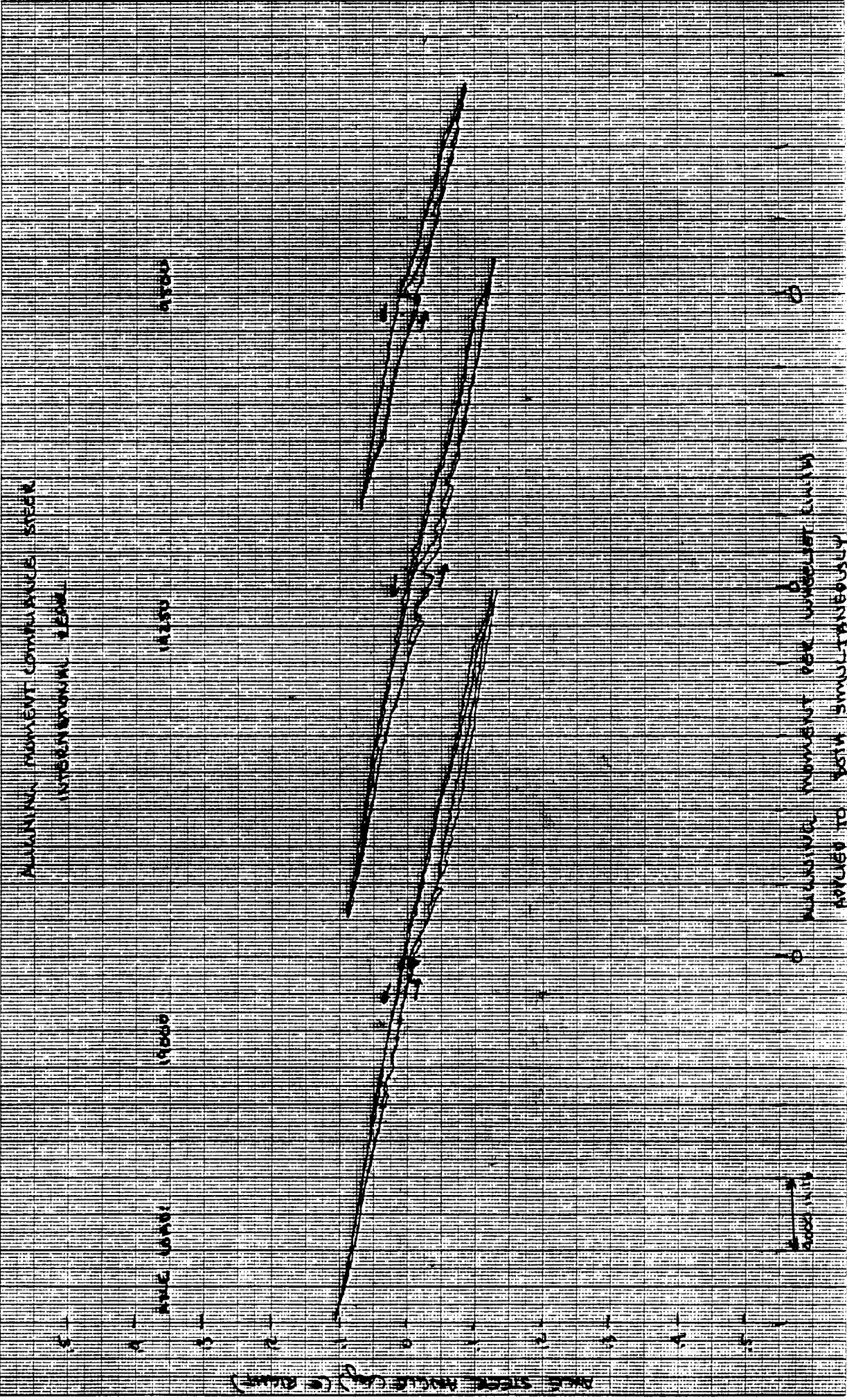


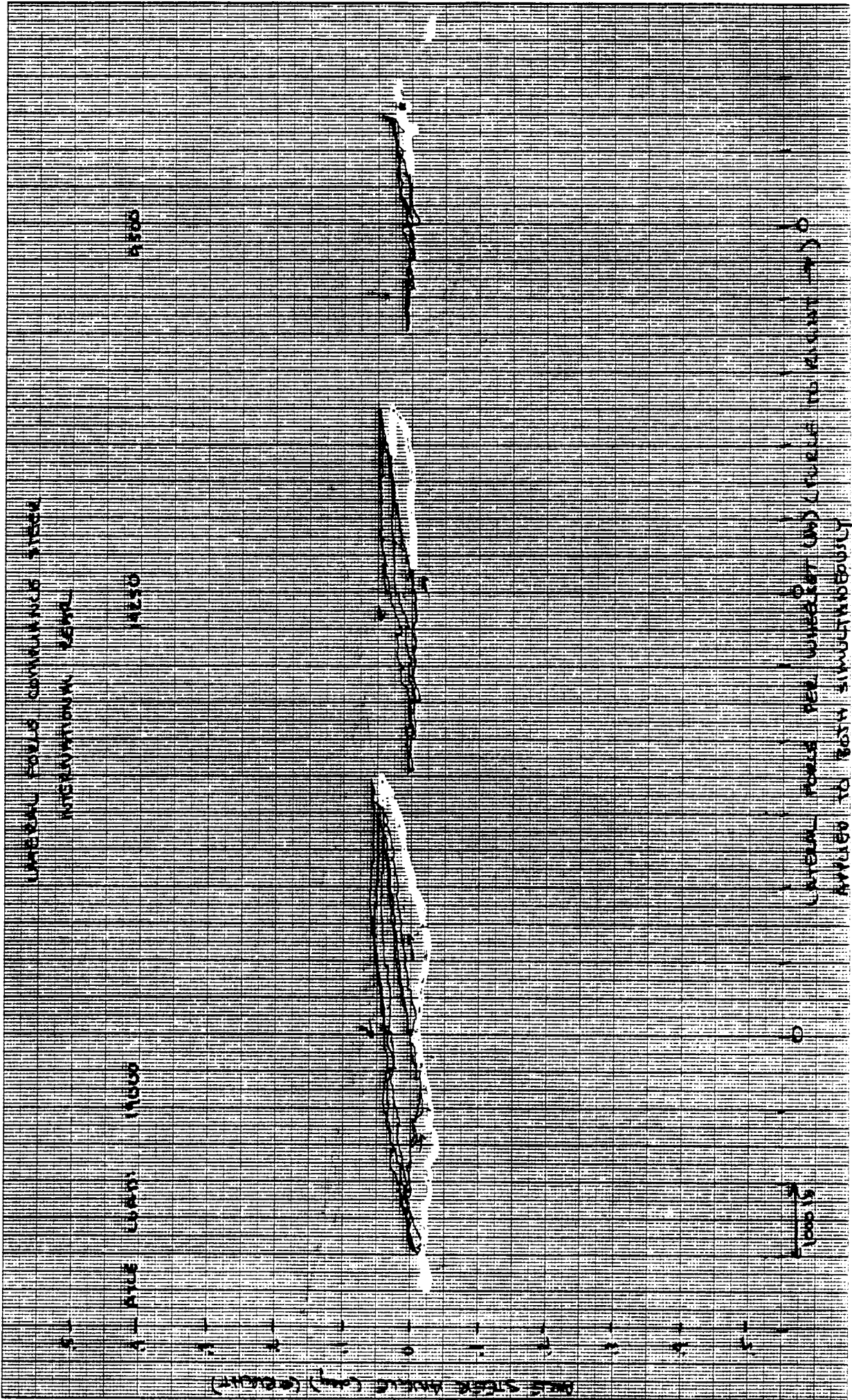












LATERAL FORCE COMPARING STEEL  
INTERMEDIATE LEVEL

1000

14250

5500

WIRE STEEL ANGLE (CM) (RIGHT)

5000

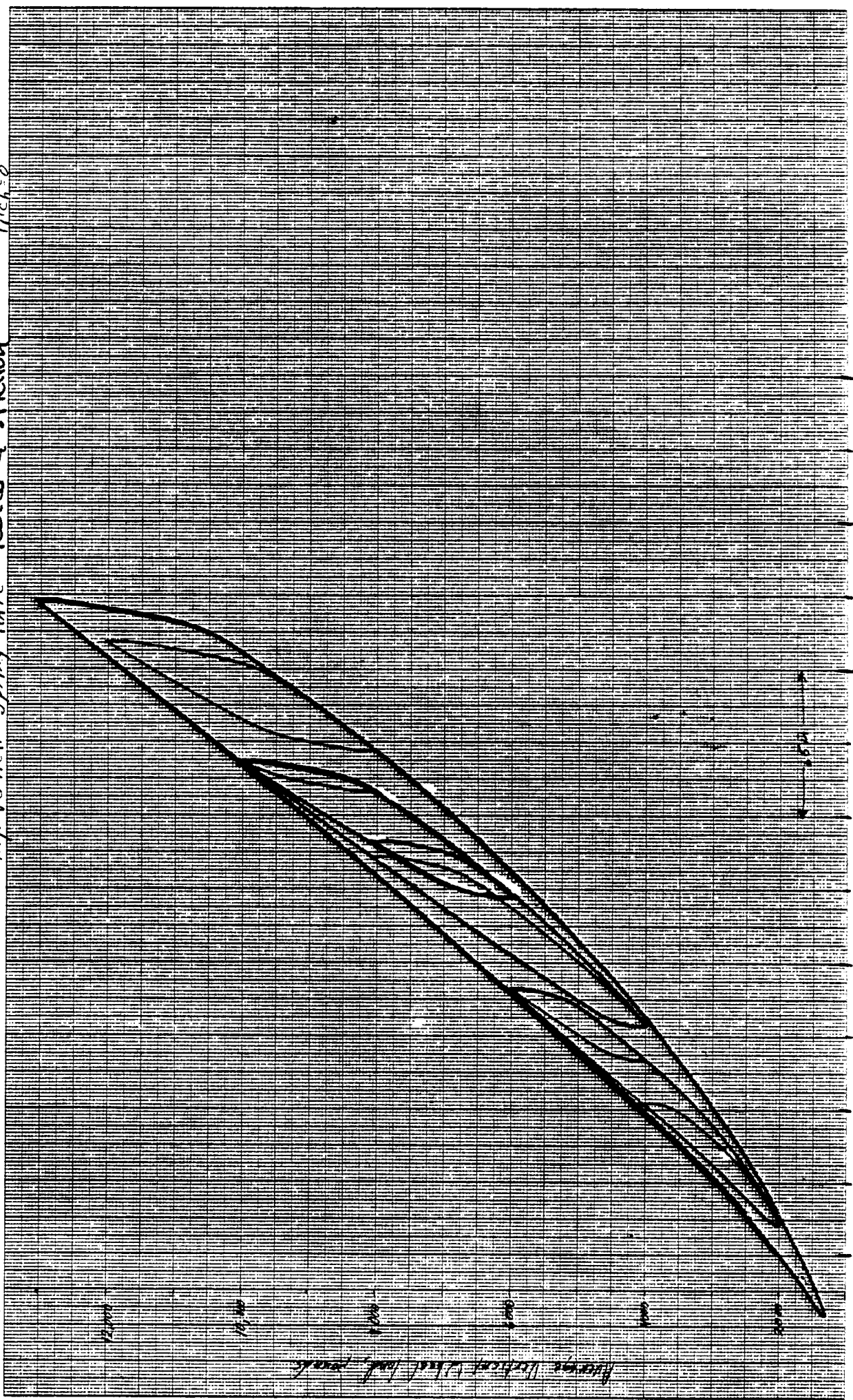
LATERAL FORCE PER WIRE (FT AND) LARGER TO RESIST 1000  
APPLIED TO BOTH SIMULTANEOUSLY

Suspension: Rayco 4-Spring #1  
(From Ford)

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>re</del>				
Vertical Rate (lb/in)		5200	4900	4500
Coulomb Friction (lb)		1050	875	550
Roll RATE	F	78K	67.5K	57K
(in/lb/deg)	R	74.5K	64K	50K
Roll CENTER HT	F	11.6	12.2	12.5
(in below top of frame)	R	11.1	10.9	10.7
Roll STEER	F	-.12	-.15	-.18
(deg/deg)	R	.09	0	-.10
ALIGNING MOMENT COMPLIANCE	F	$10^{-5}$	$1.1 \times 10^{-5}$	$1.3 \times 10^{-5}$
(deg/in/lb)	R	$8 \times 10^{-6}$	$8 \times 10^{-6}$	$8 \times 10^{-6}$
LATERAL FORCE COMPLIANCE	F	$4.5 \times 10^{-5}$	$4.6 \times 10^{-5}$	$5 \times 10^{-5}$
(deg/lb)	R	_____	nil	_____
INTERAXLE LOAD TRANSFER (BY SPRING)	L	.25	.33	.29
(lb/lb) + to rear	R	.3	.31	.28
Pitch = 0°				

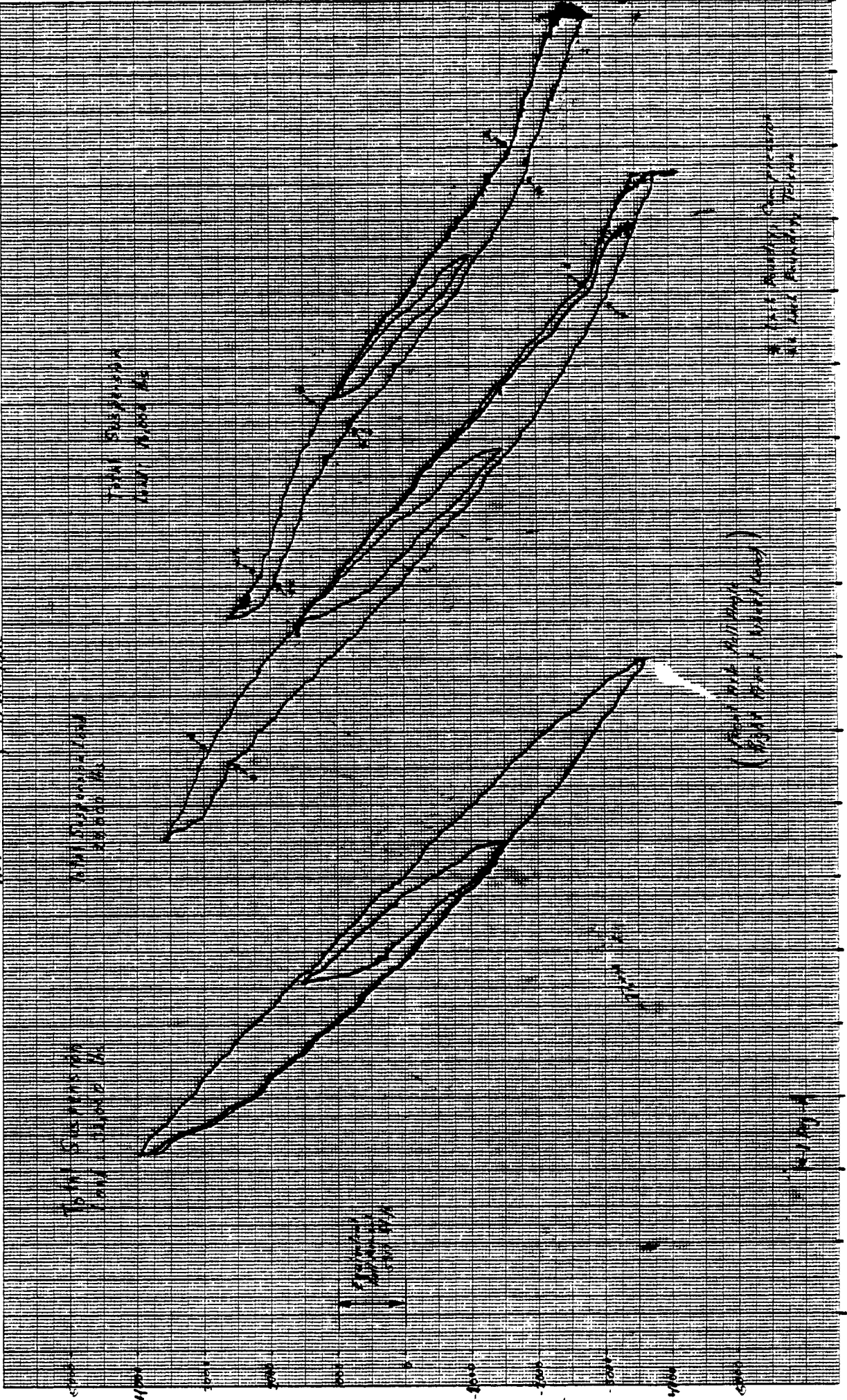


Avg. Vertical Spring Rate RETCO 4-SPRNGH Pitch 6'



Average Vertical Displacement

Roll Rate Front Axle Pitch=0°

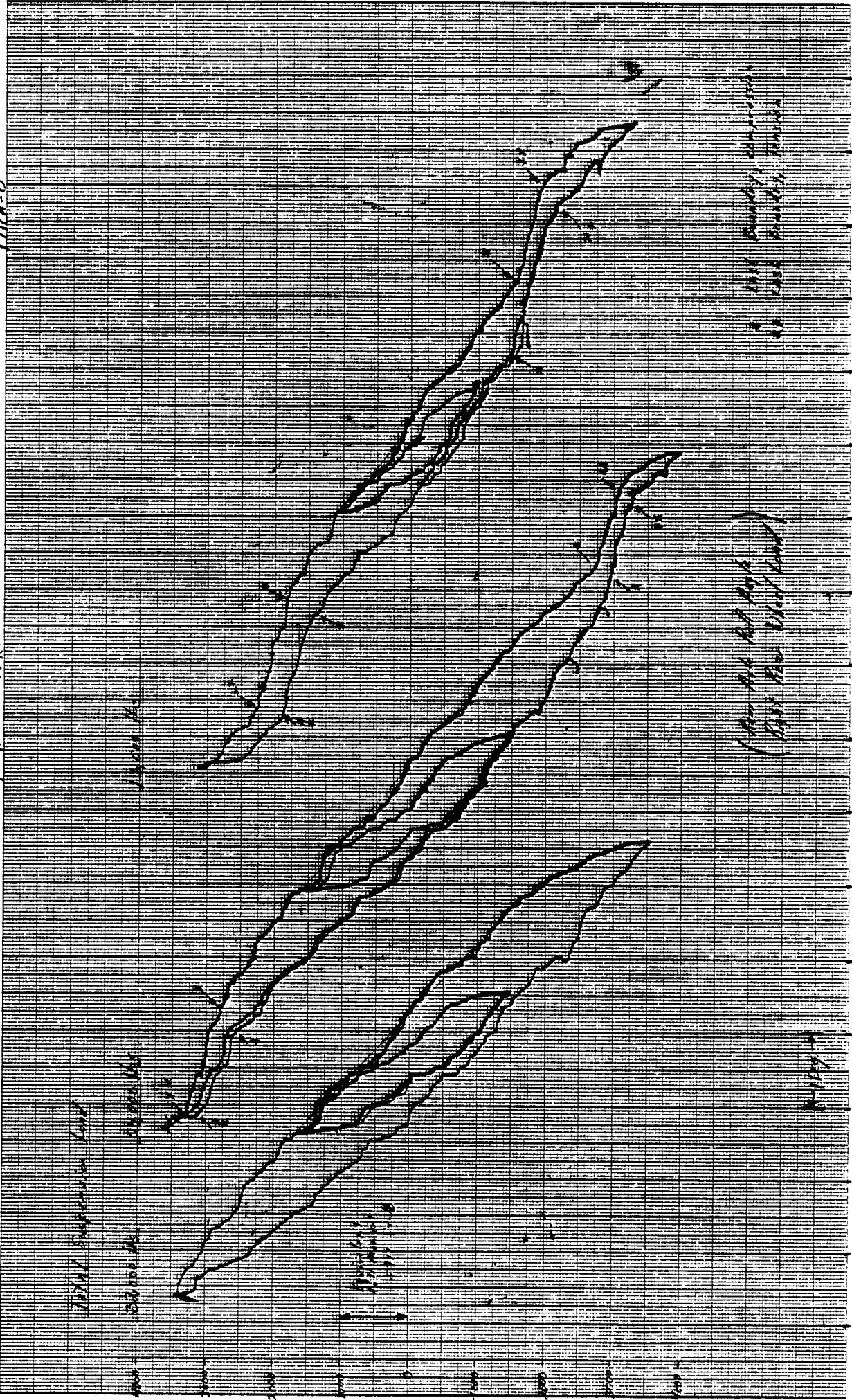


Right Side Differential Wheel Load, pounds

Roll Angle

Roll Rate, Rear Axle

Pitch



Roll Angle

Right Side Differential Wheel Load, pounds

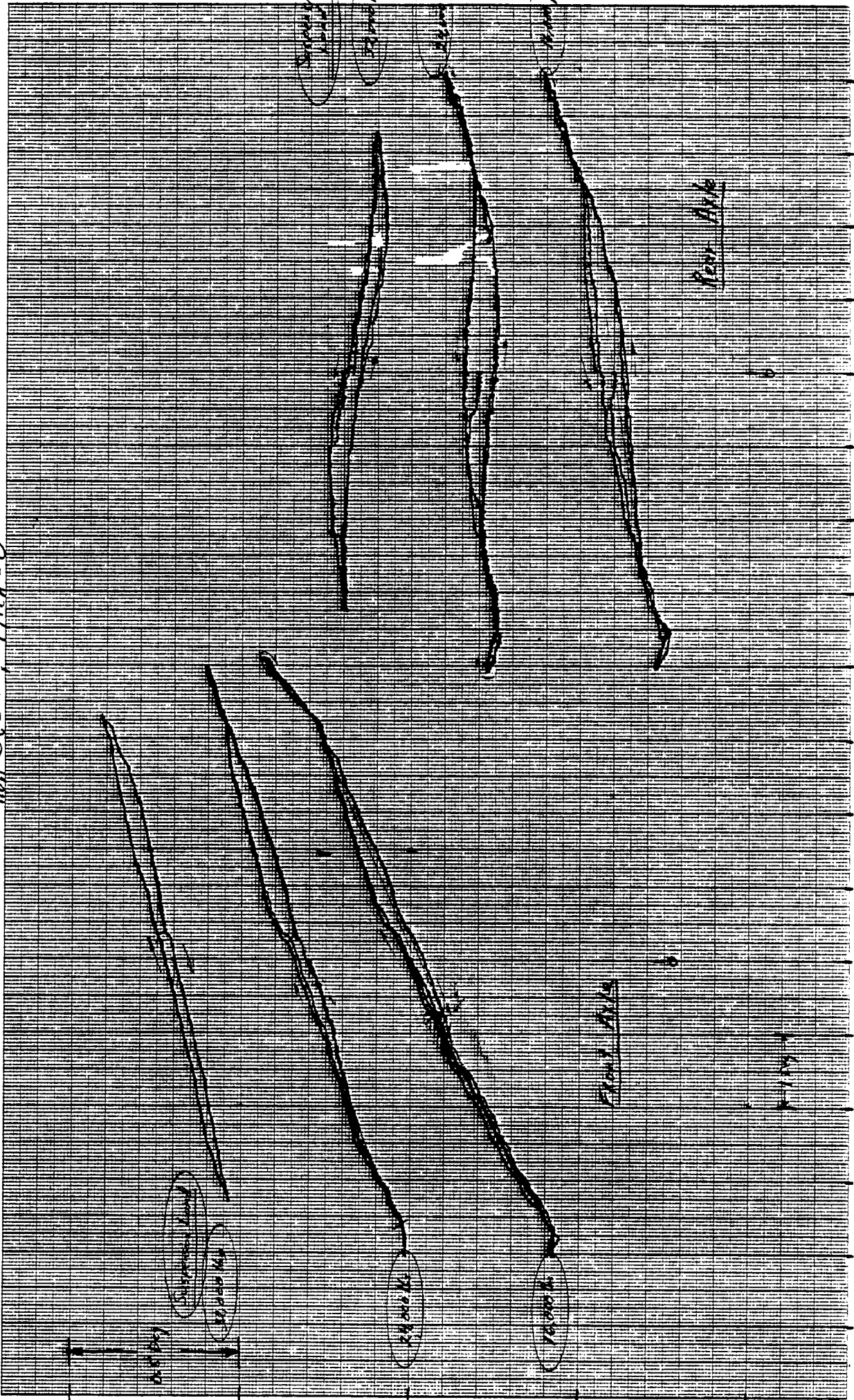
47 1512

10 X 12 TO 14 CENTIMETERS X 30 CM

11/2/52



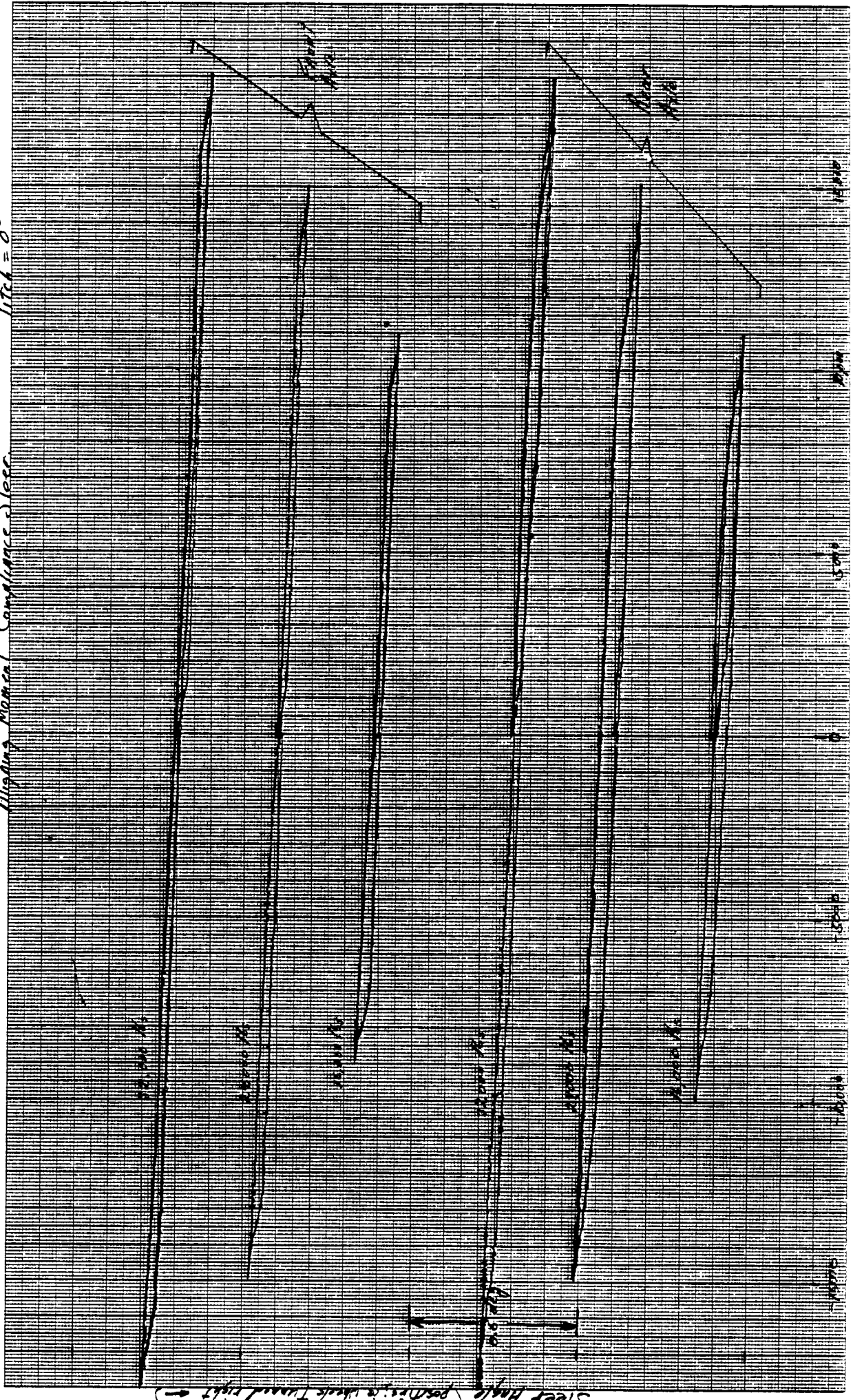
Roll Steer, Pitch = 0°



Roll Angle, (Right Spring Compressed: ←)



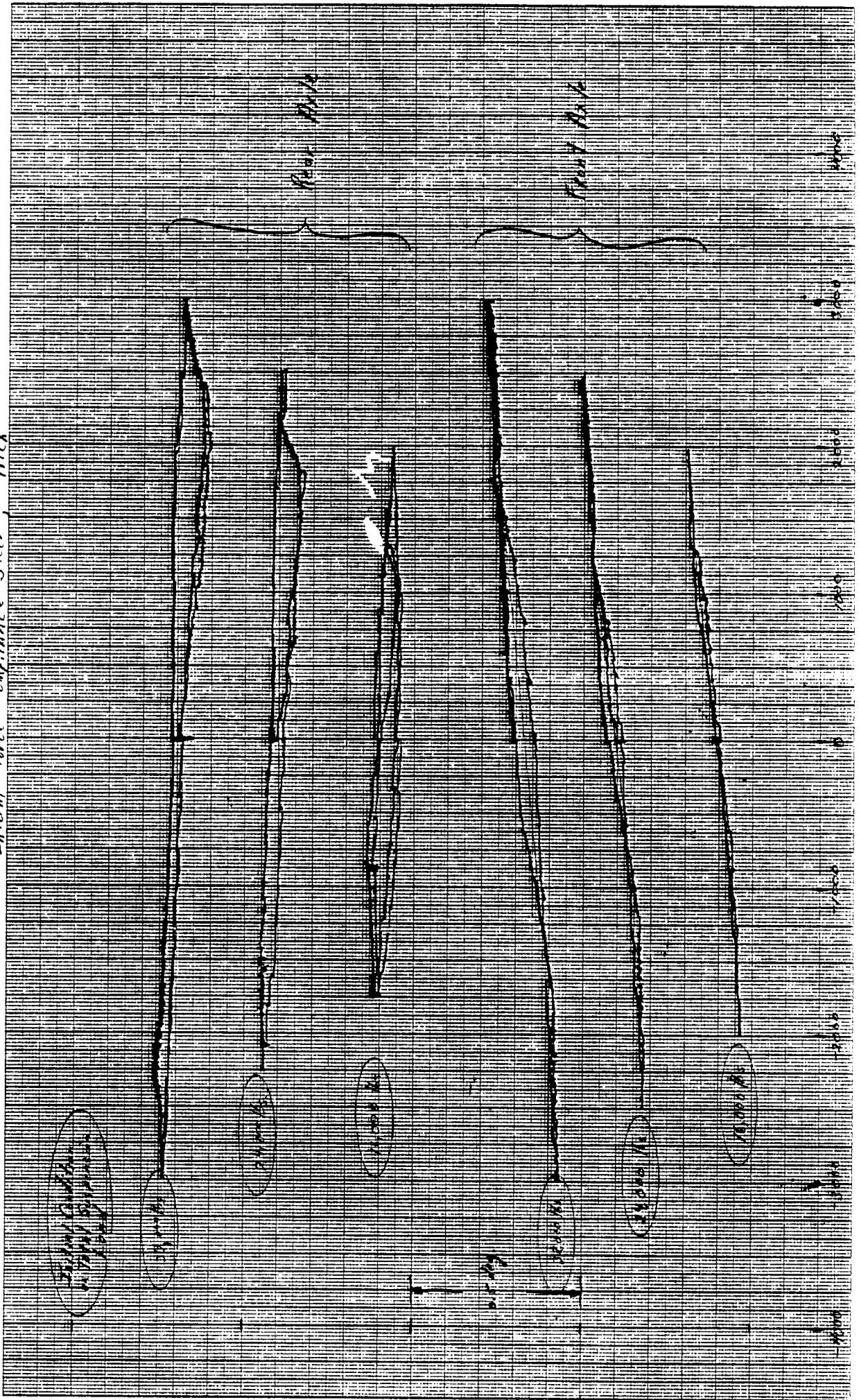
Aligning Moment Compliance Steer Pitch = 0°



Steer Angle (positive is wheel turned right →)

Aligning Moment Per Wheel (positive is turn wheel toward right →) in  $lb_0$

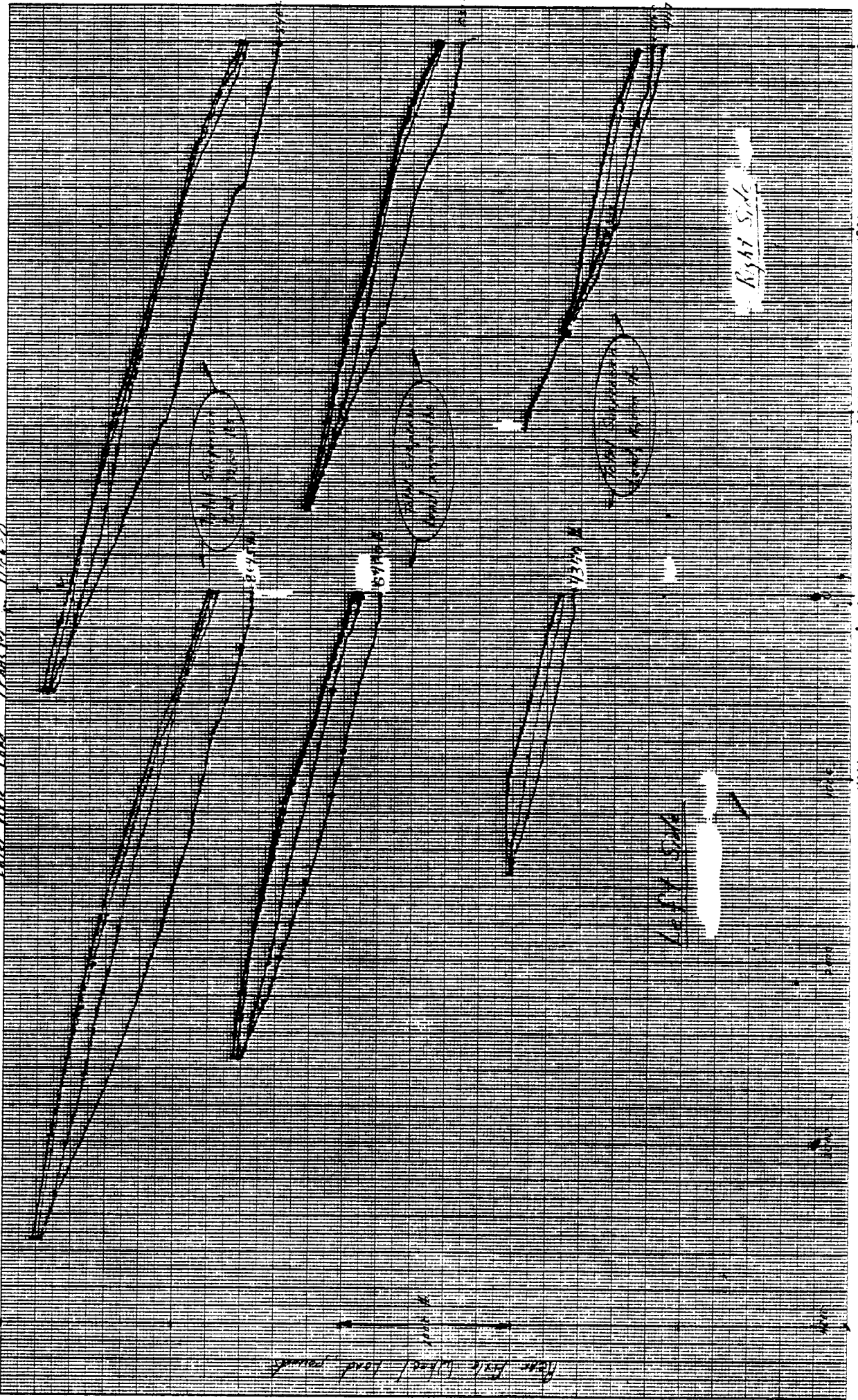
Lateral Force Compliance Steer, Pitch



Steer Angle (positive is wheel toward right →)

Side Force Per Wheel, (Force toward right side →) pounds

Inter-Axle load Trans. & Pk. 120



From axle load/ load points

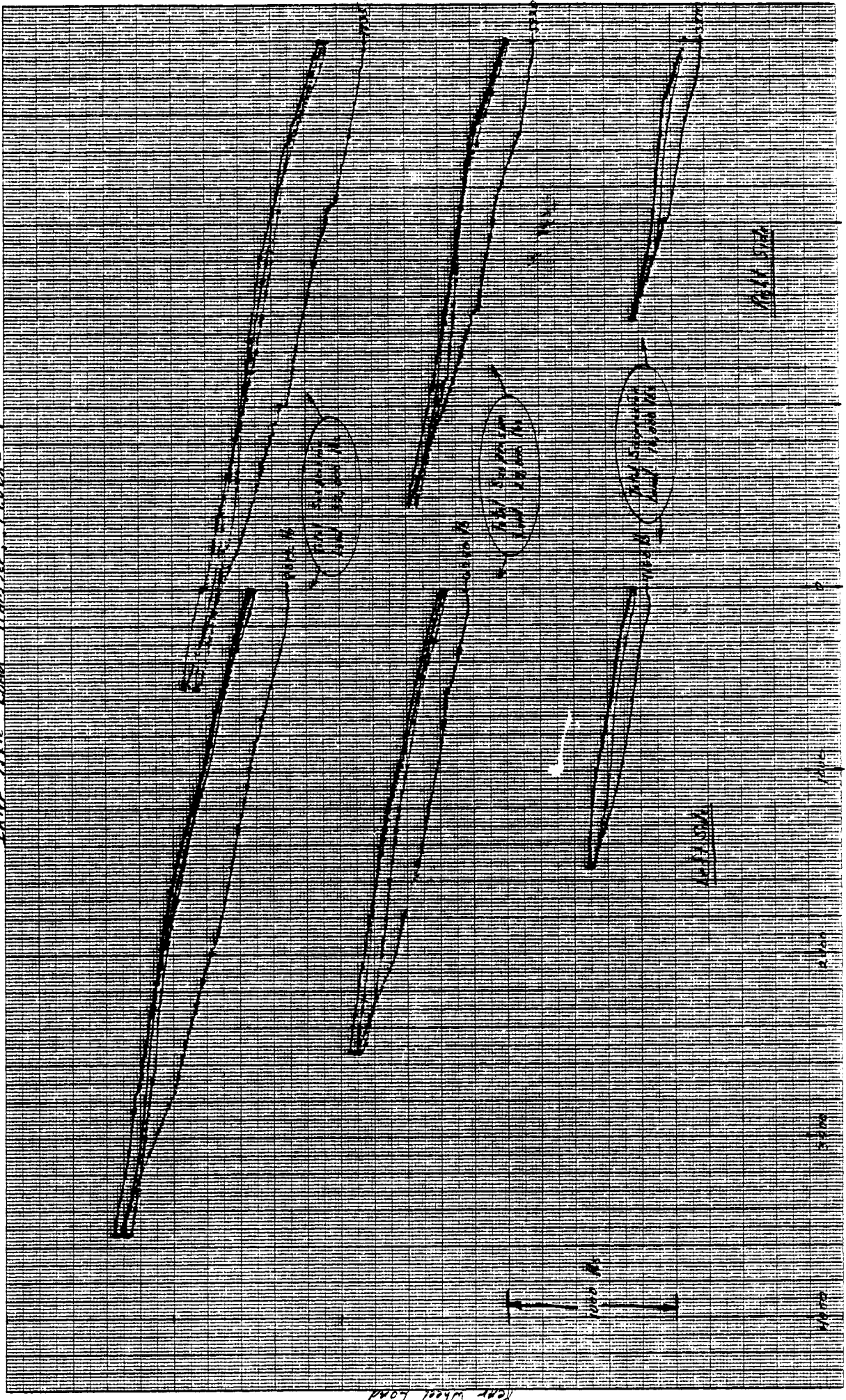
Left Side

Right Side

Applied Axle Force for Wheel mounts (all loads)



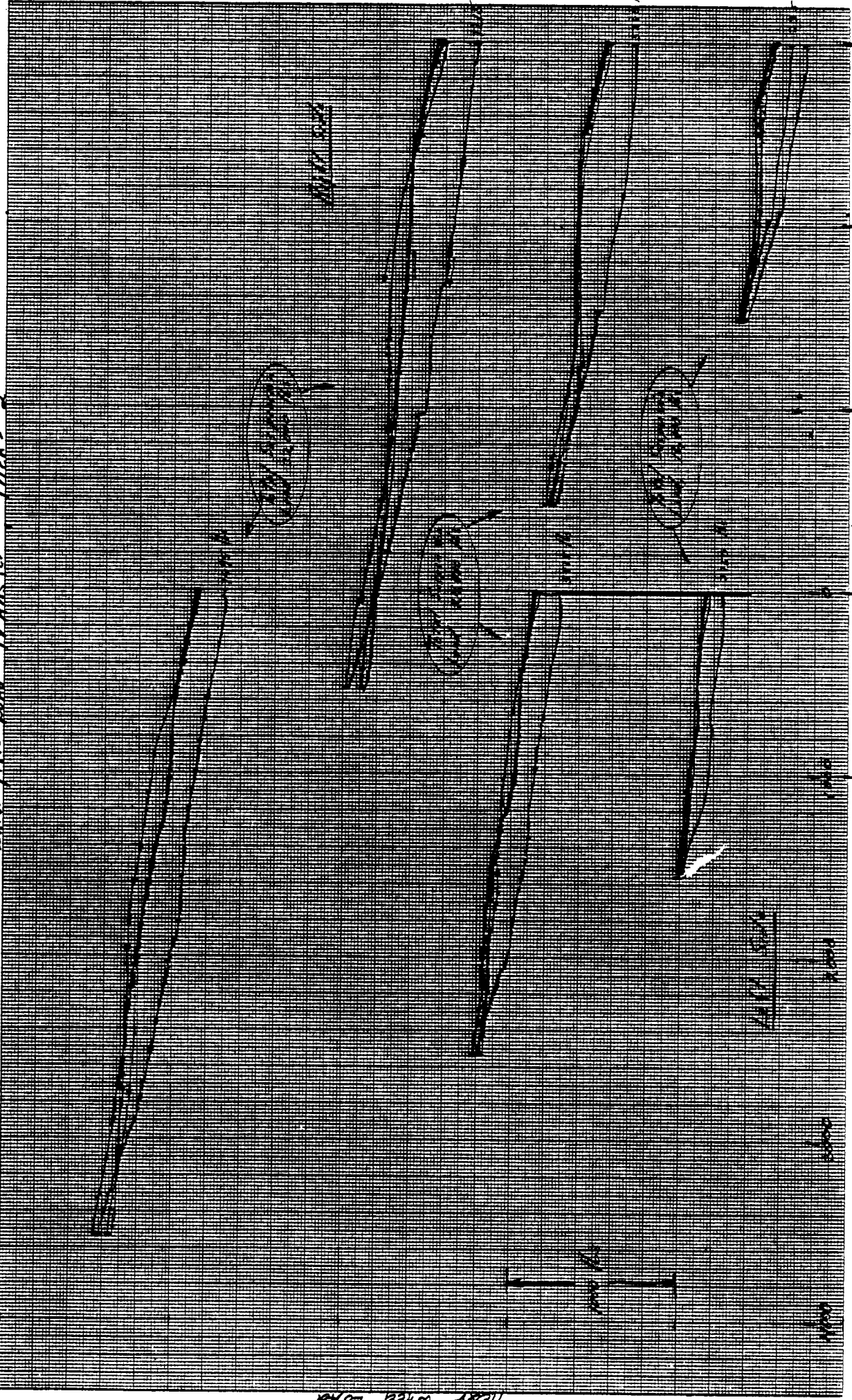
Intr. Axle Load Transfer Pitch = -1°



Rear Wheel Load

Applied Brake Force Per Wheel, pounds (all wheels)

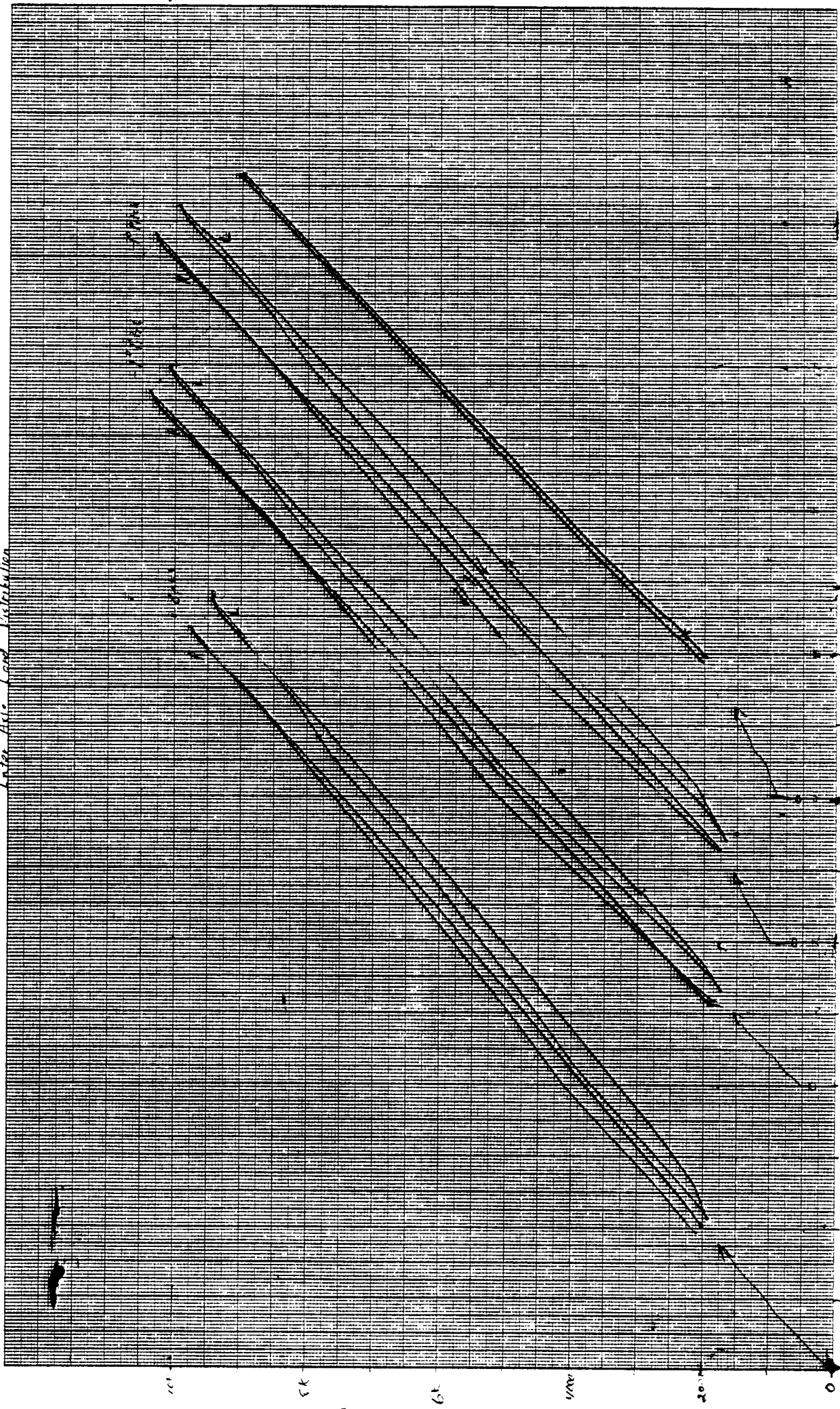
Inter Axle Load Transfer Plot,  $\gamma = 2^\circ$



Rear Wheel Load

Applied Brake Force Per Wheel, pounds (All wheels)

Inter Axis Load Distribution

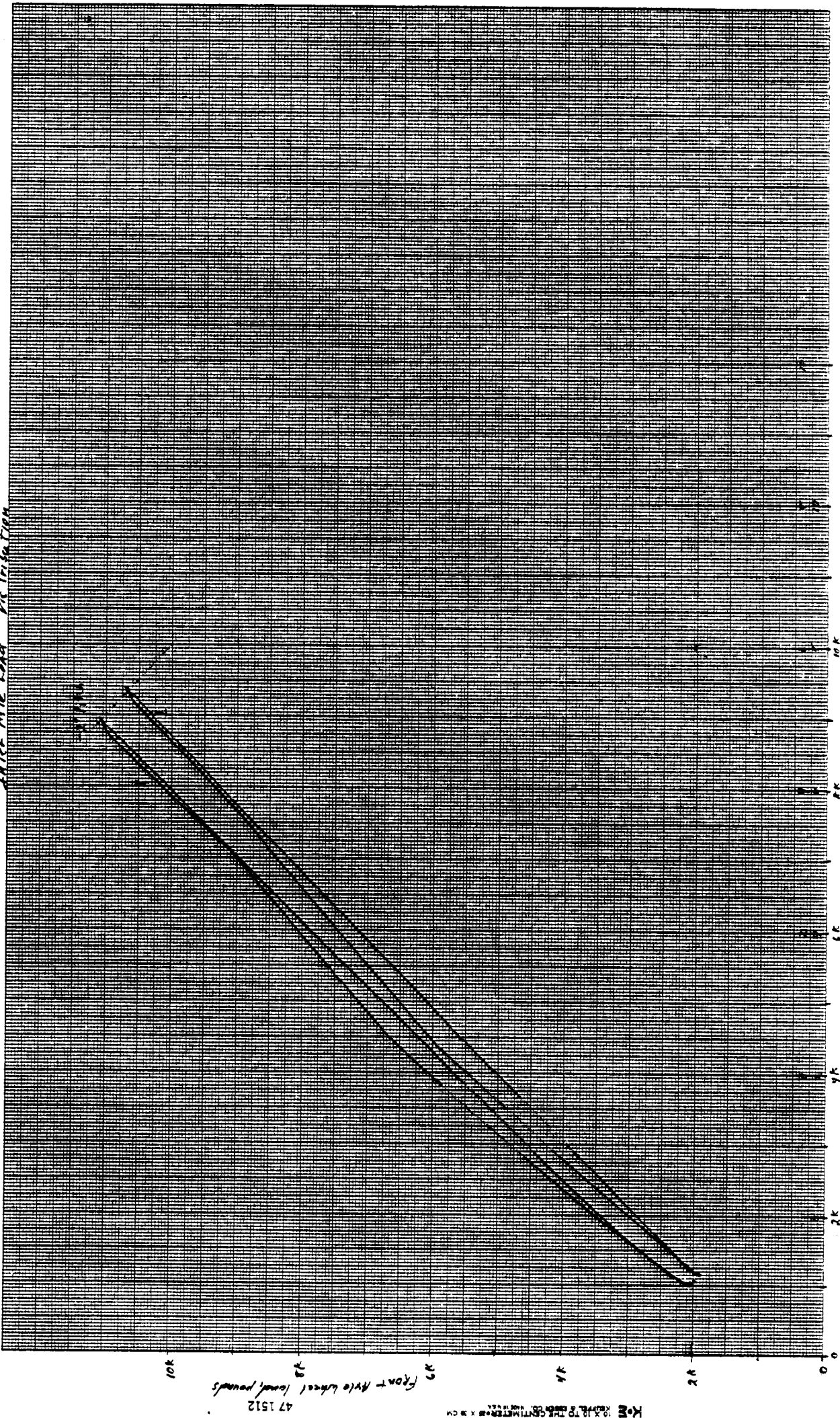


47 1512

10 X 10 TO THE CENTIMETER X 30 CM



*Inter Axle Load Distribution*



*Rear Axle Wheel Load, pounds*

47 1512  
K.E. 10 X 10 IN. CENTIMETER X 3 CM  
FRONT AXLE LOAD, POUNDS

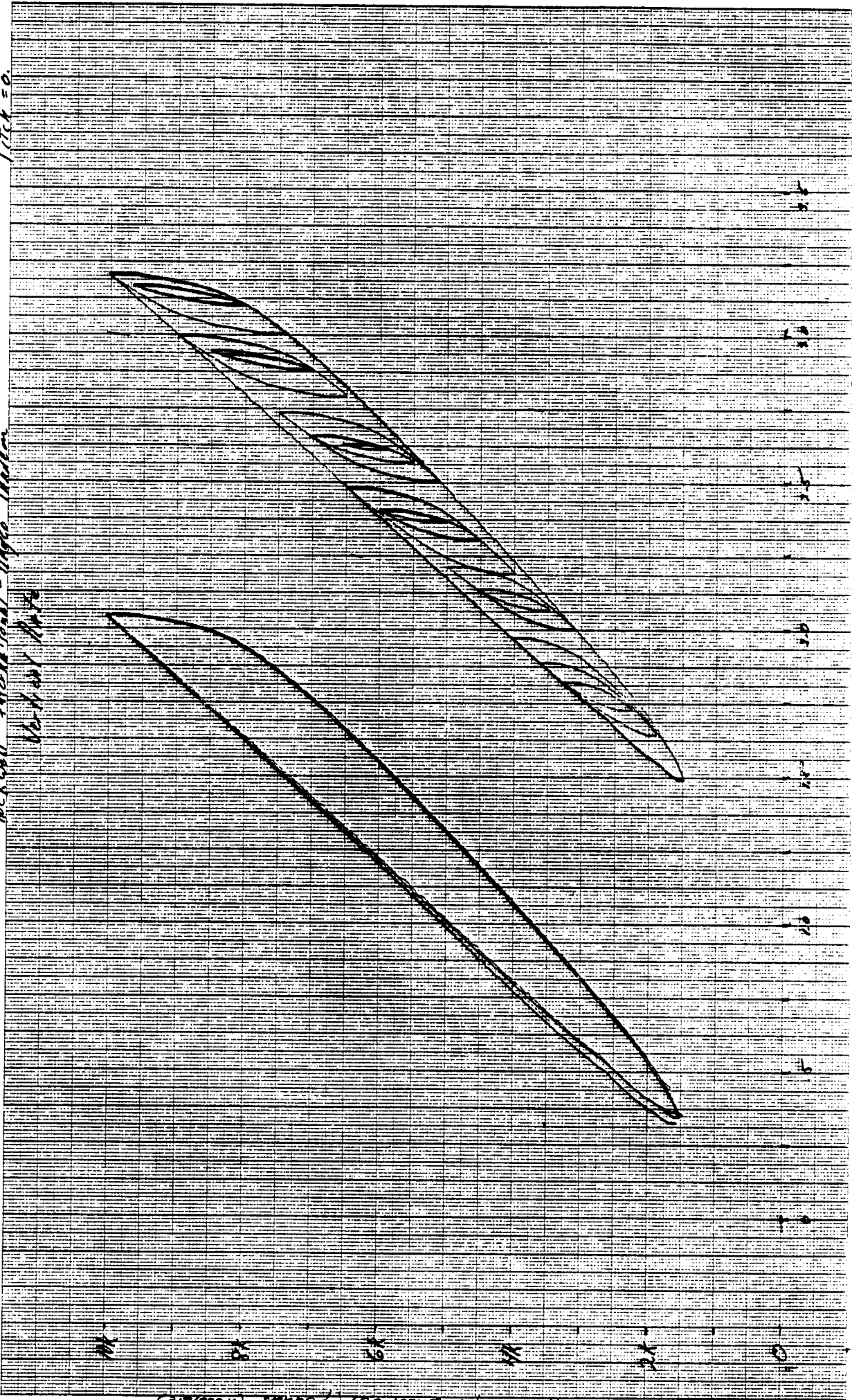
Suspension: Reyco 4 Spring #2  
(from Rockwell)

		<u>32000</u>	<u>24000</u>	<u>16000</u>
			Susp Load	
Vertical Rate (lb/in)		4600	4600	4600
Coulomb Friction (lb)		825	700	550
Roll RATE	F	80K	75K	65K
(in. lb/deg)	R	75K	72K	67K
Roll CENTER HT	F	NA		
(in below top of beam)	R			
Roll STEER	F	.02	-.01	-.04
(DEG / DEG)	R	nil	-.005	-.01
ALIGNING MOMENT COMPLIANCE	F	$4.5 \times 10^{-6}$	$5 \times 10^{-6}$	$5 \times 10^{-6}$
(deg (in lb))	R	$1 \times 10^{-6}$	$1 \times 10^{-6}$	$3.4 \times 10^{-6}$
LATERAL FORCE COMPLIANCE	F	$2.2 \times 10^{-5}$	$4.5 \times 10^{-5}$	$7 \times 10^{-5}$
(deg / lb)	R		nil	
INTERAXLE LOAD TRANSFER (BRANK)	L	.52	.56	.42
(lb / lb) + to rear	R	.36	.38	.38
PITCH = 0				

Pitch = 0.

Beckwith International - Super Tracker  
Vertical Motion

Average Vertical Wheel Motion, inches (relative)



Vertical Load per wheel set, pounds (relative)

Rockwell International - Cygo Trade  
Roll Rate, Country Risk

Roll = 0

Roll Moment Limit: 52,000 ft-lb

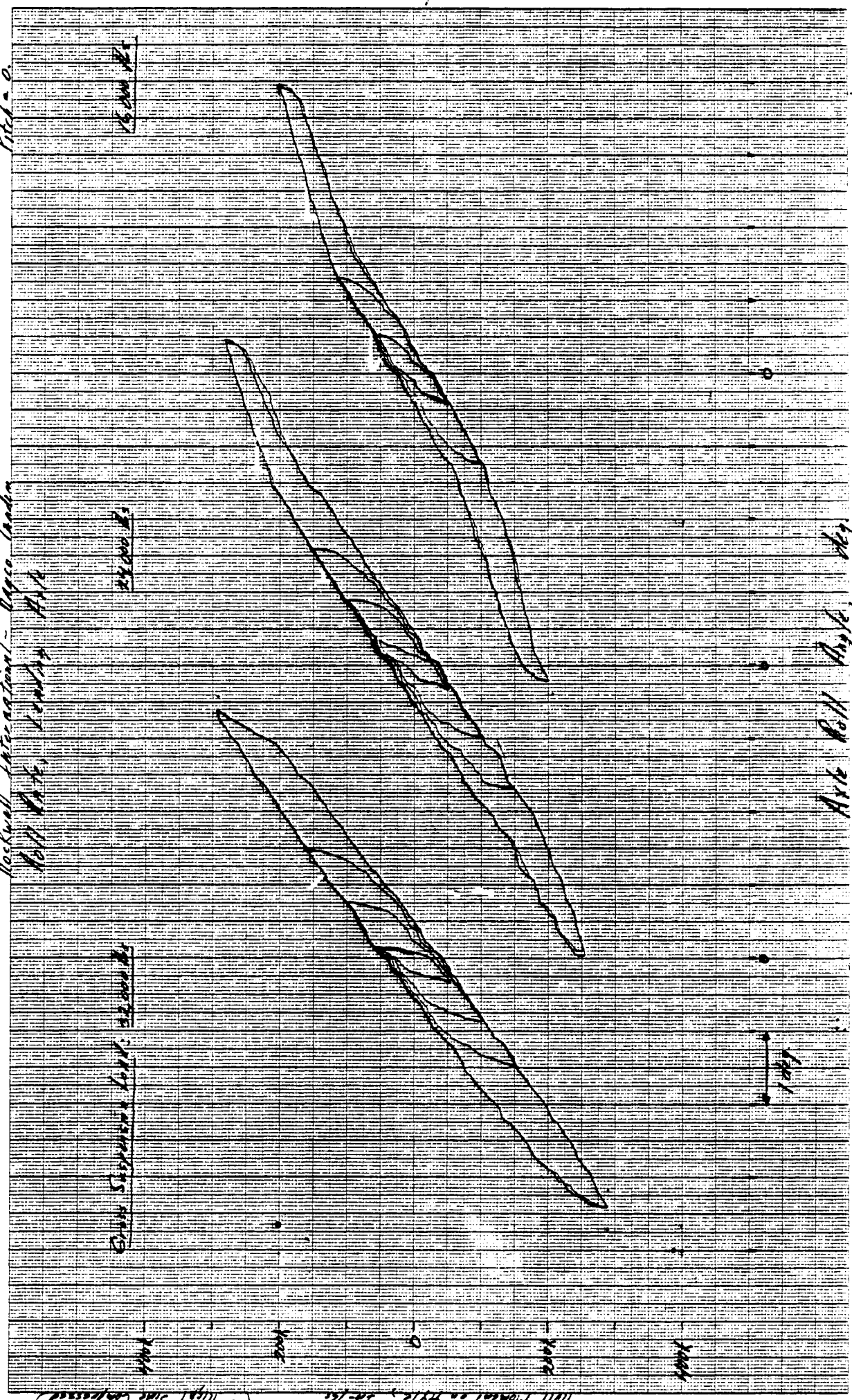
Roll Rate Limit: 24,000 ft/s

Roll Rate Limit: 16,000 ft/s

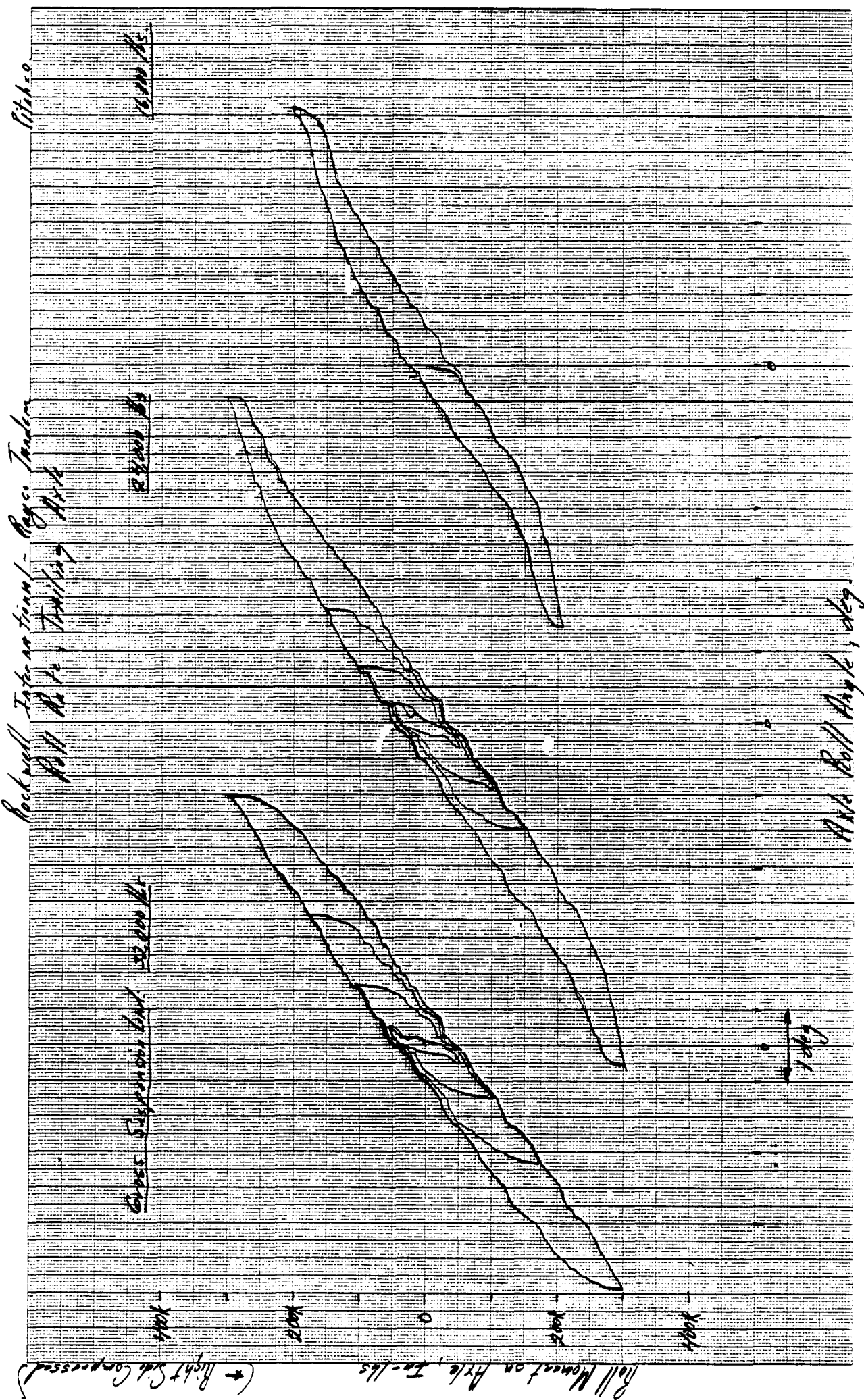
(→ Right Side Compressed)

Roll Moment on Axle, In-lb

Axle Roll Angle, deg







Rockwell International - Pogo Tendon  
Roll Angle, Training Axis

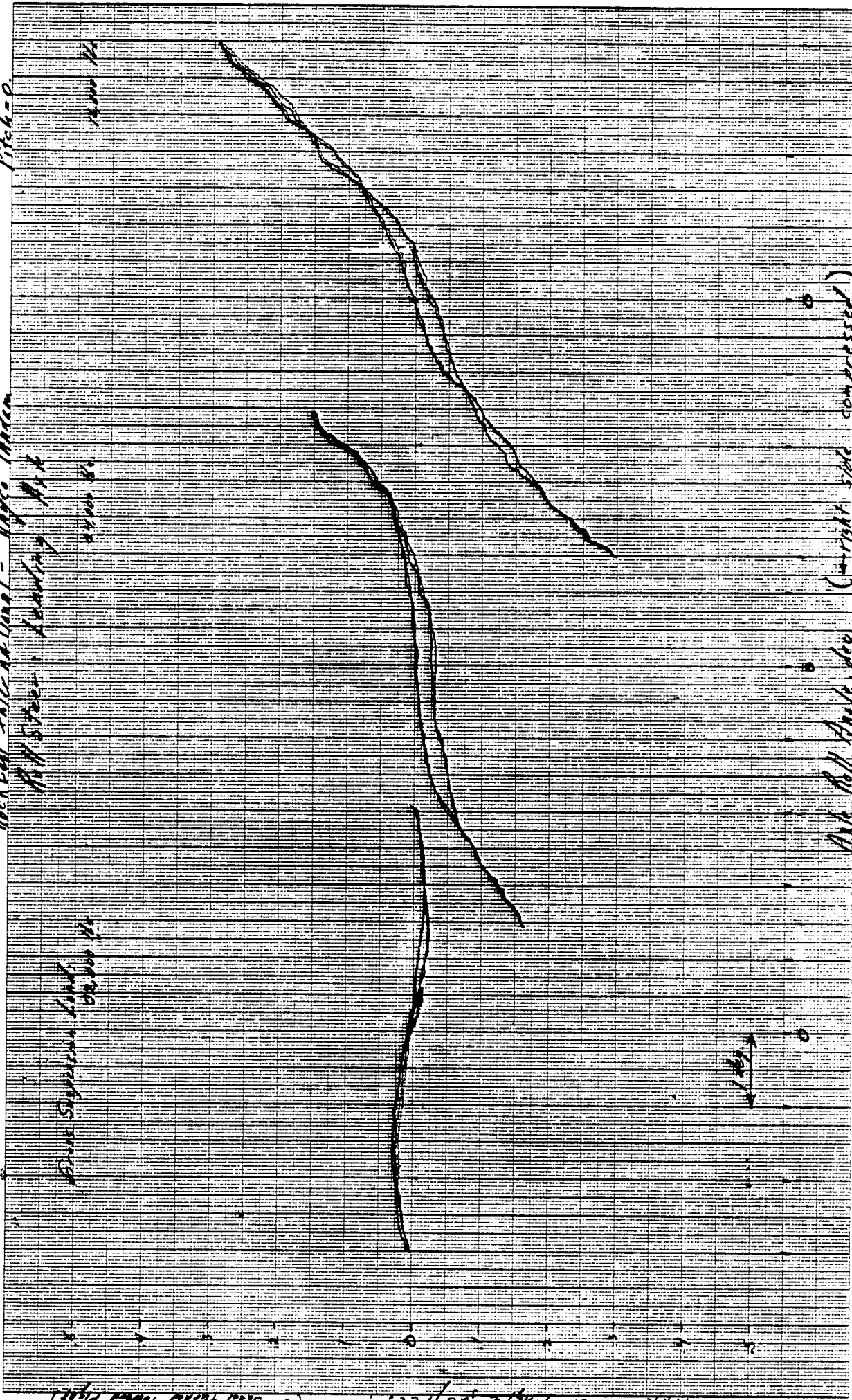
Upper Suspension Unit

10000/15

16000/15

Roll Angle, deg

(→ Right Side Compressed)



Rockwell International - Space Tendon

Roll Steer - Landing Axle

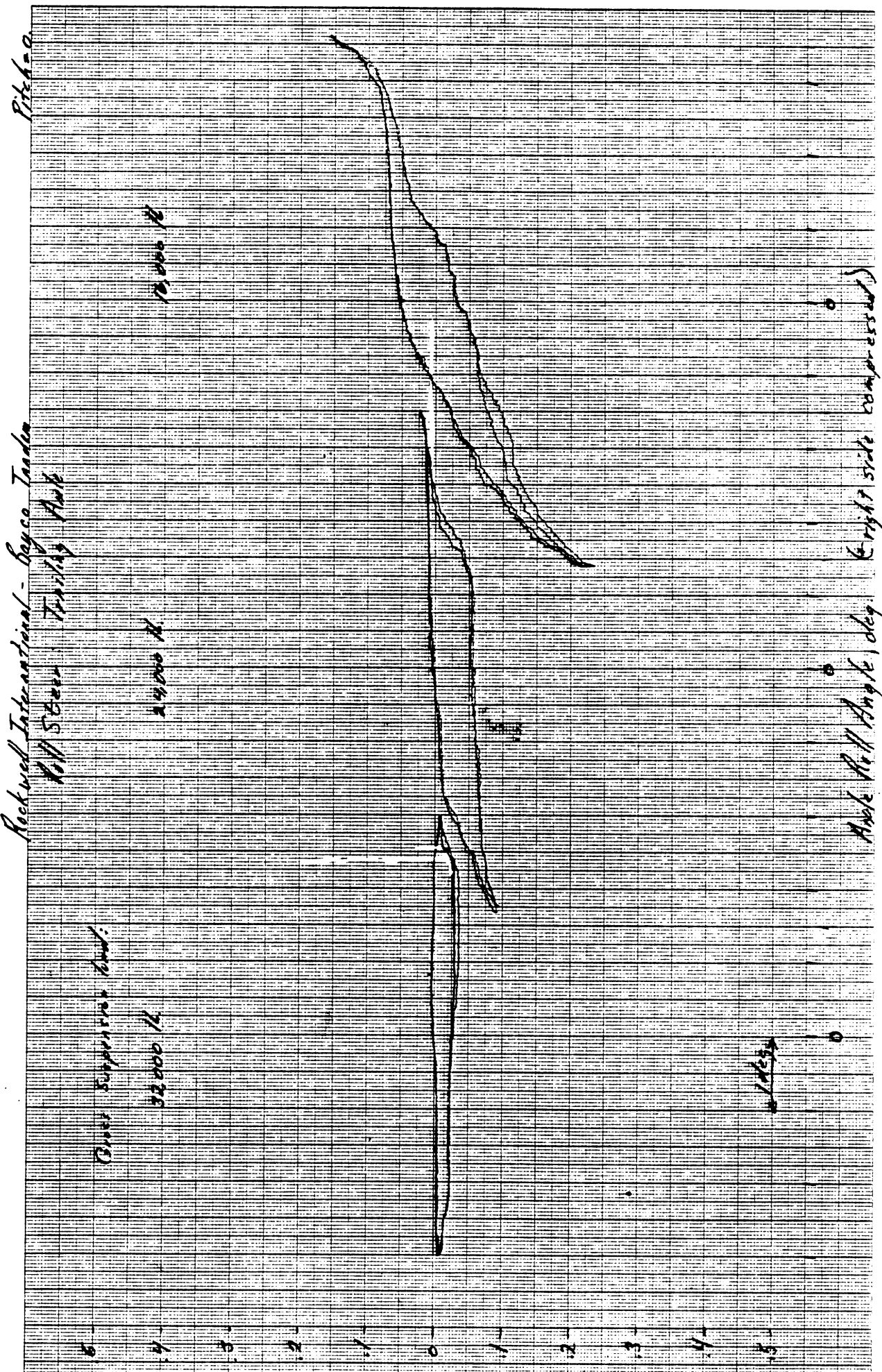
Direct Suspension Load

Axle Roll Angle, deg. (← right side compressed)

Axle Steer Angle, degrees (← Wheel turned toward right)



Angle Steer Angle, degrees. (→ Wheel turned toward right)



Over suspension

Roll Steer

Pitch

(right side comparison)

Angle Roll Angle, deg

Pitch = 0

Rockwell International - Logco Sweden

Always Mount Engineered Steel  
Axle Assembly

Class Suspension  
2000 lb

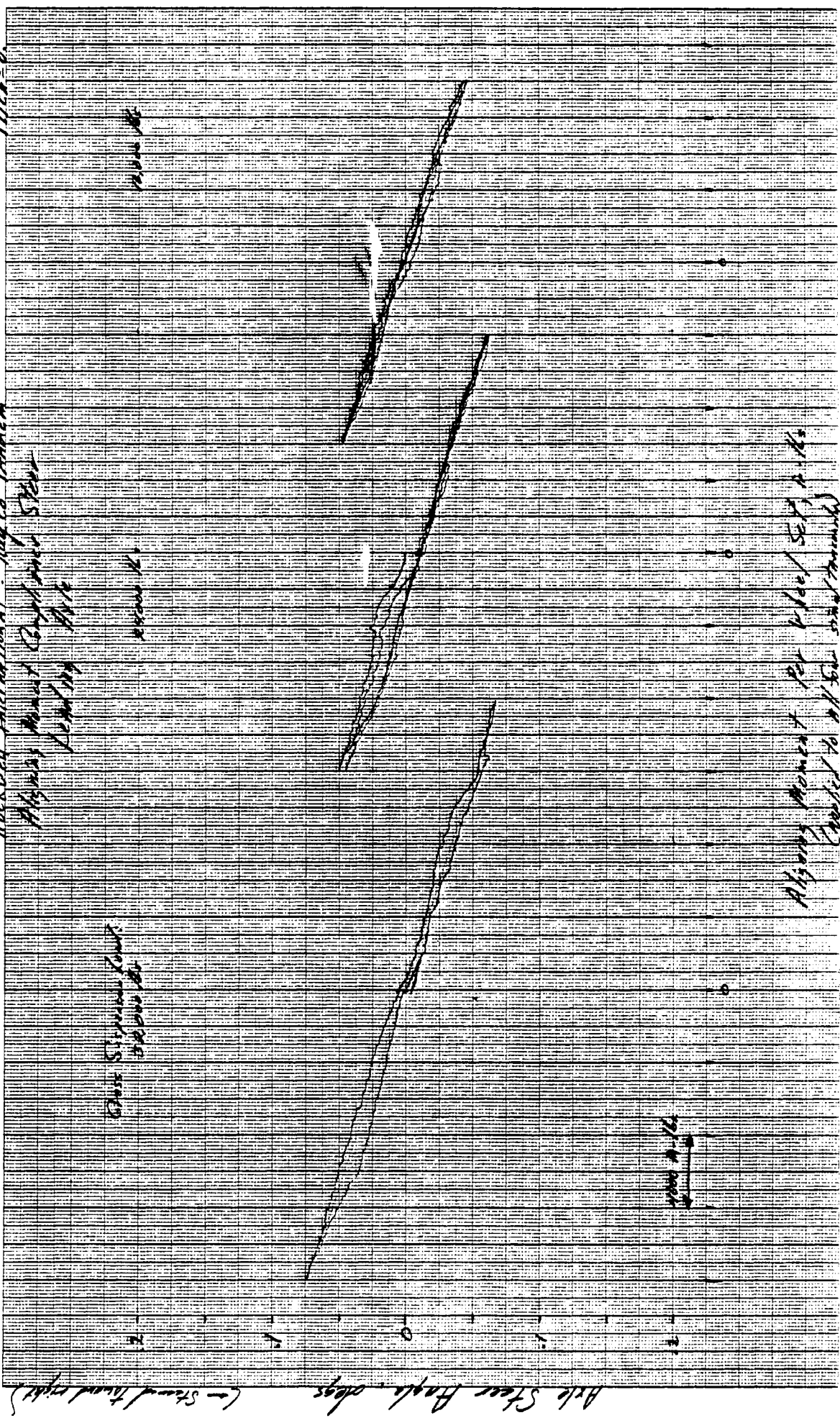
2000 lb

2000 lb

Arle Steer Angle - degs  
(- Steer toward right)

47 1512

K&E 10 X 11 1/2 SHEET METAL X 3 CM  
K&E 10 X 11 1/2 SHEET METAL X 3 CM



Always Moment per wheel set  
Applied to all four wheels

Pitch = 0.

Rockwell International - Rocco Tomlin

Aligning Moment for Wheel 500 ft. m.H.  
(Applied to all four wheels)

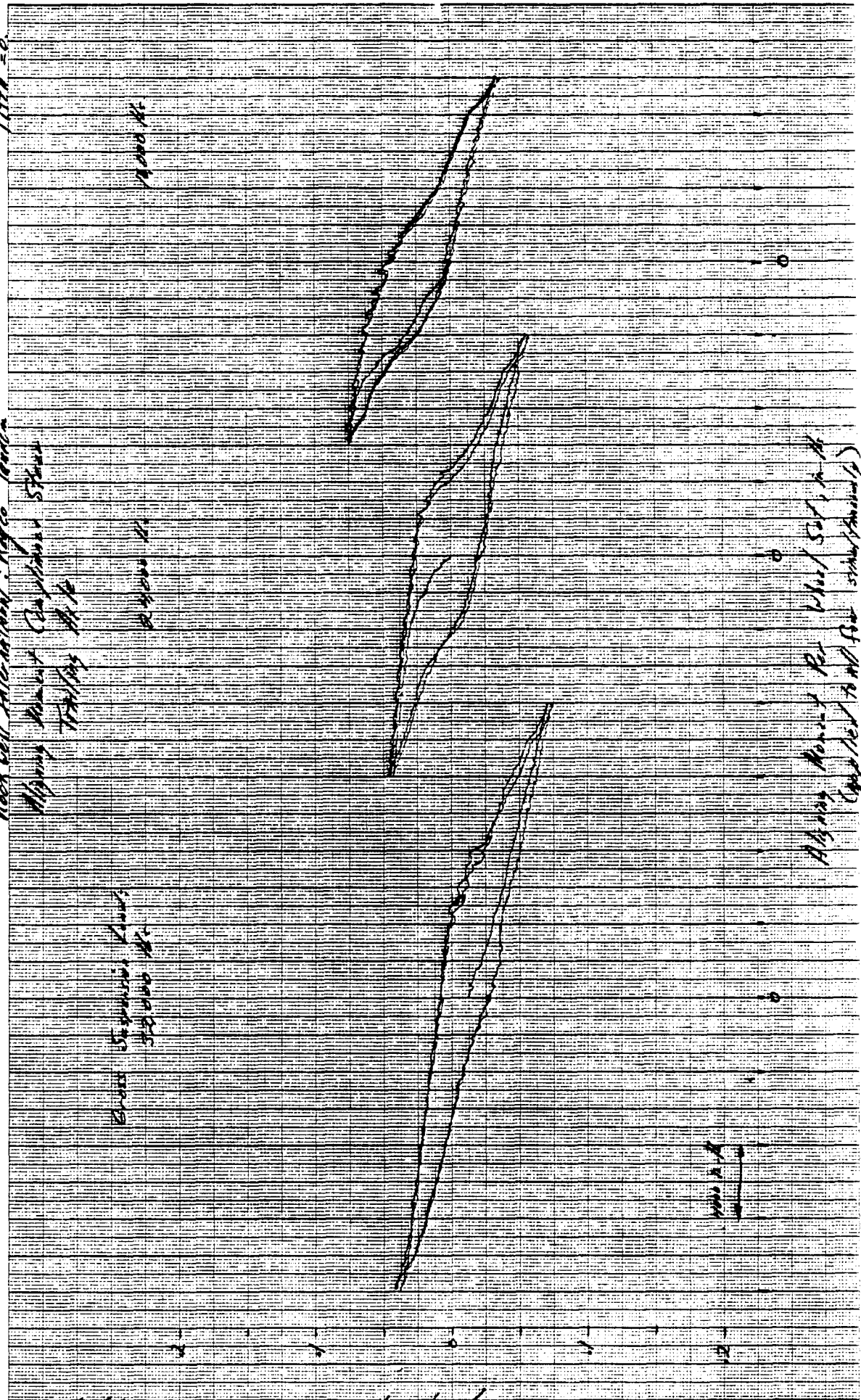
Aligning Moment for Wheel 500 ft. m.H.  
(Applied to all four wheels)

Aligning Moment for Wheel 500 ft. m.H.  
(Applied to all four wheels)

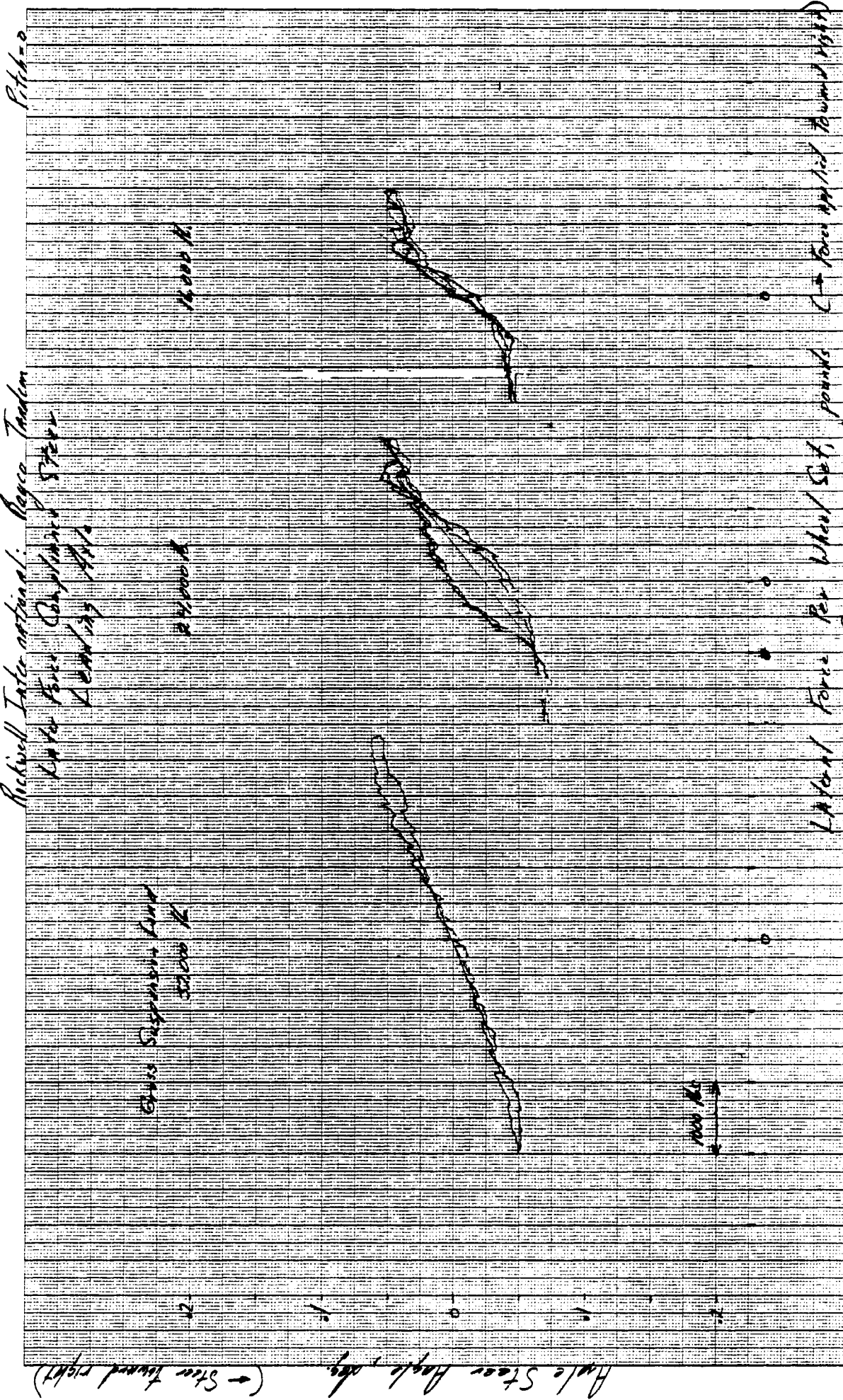
Aligning Moment for Wheel 500 ft. m.H.  
(Applied to all four wheels)

Aligning Moment for Wheel 500 ft. m.H.  
(Applied to all four wheels)

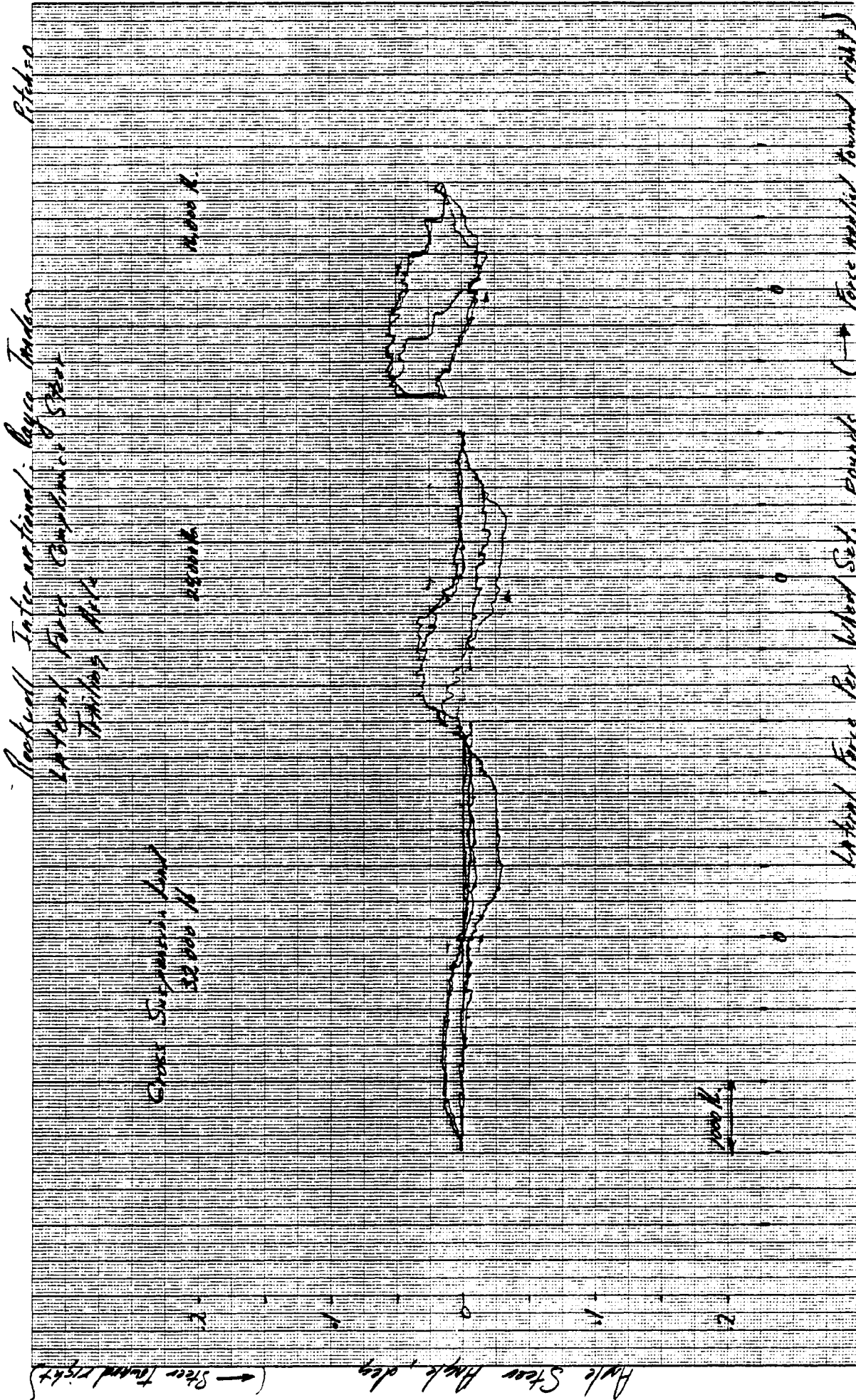
Axle Steer Angle, degrees ( = Steered Toward Right )







(→ Steer toward right)



Applied Force Per Wheel Set, Pounds  
 (Applied to all four simultaneously)

Steer Toward right

Backwell International - Soyco Tender

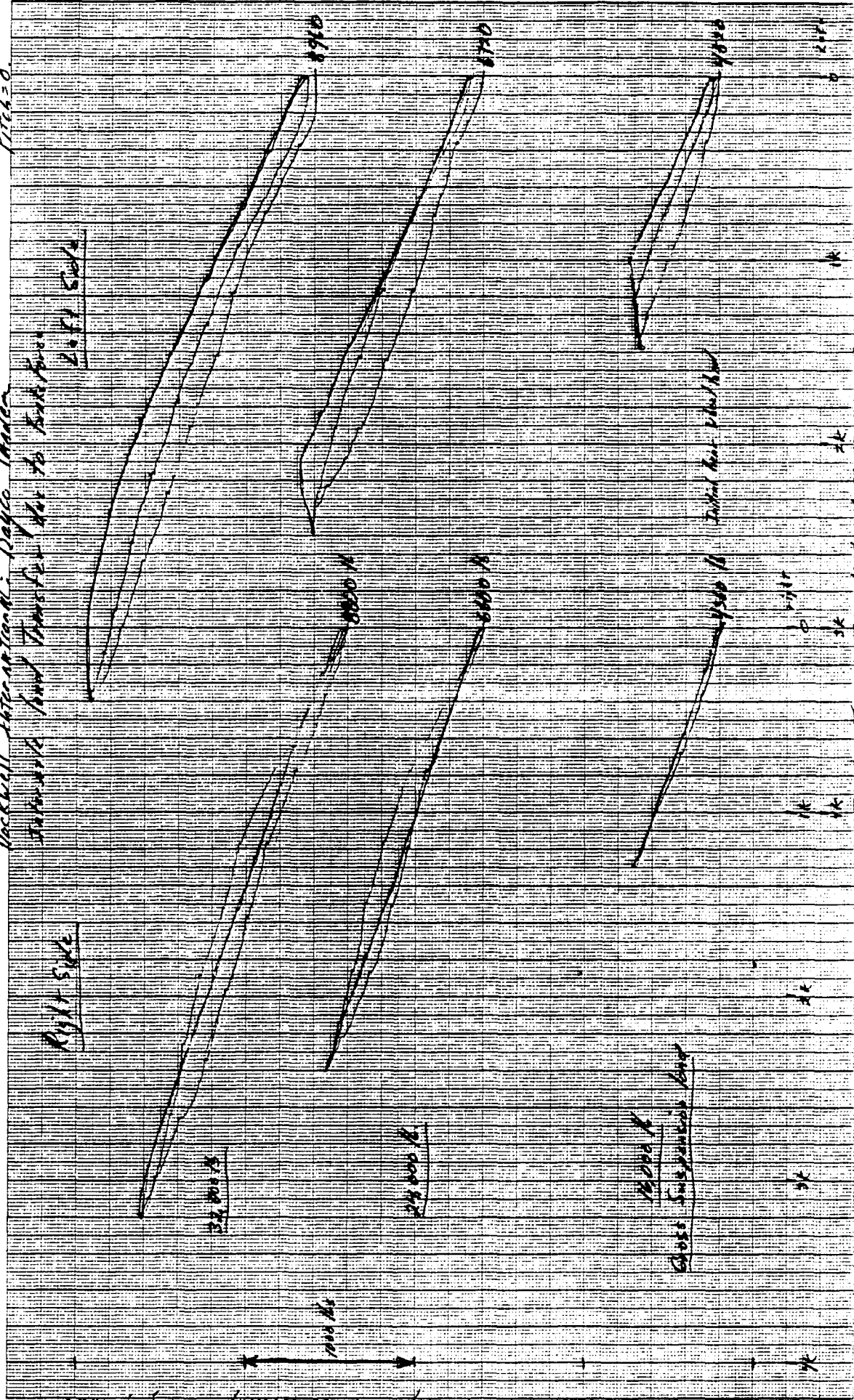
Pitch = 0

Forward Load Transfer due to Braking

RIGHT SIDE

Left Side

Load Transfer from leading wheel to Trailing wheel, pounds



Brake Force per Wheel Set, pounds  
Applied to All Four Simultaneously



Pitch = -1°

Richard International: Super Tractor

Maximum Power Output Due to Brake Force

Right Side

Excess Suspension Load

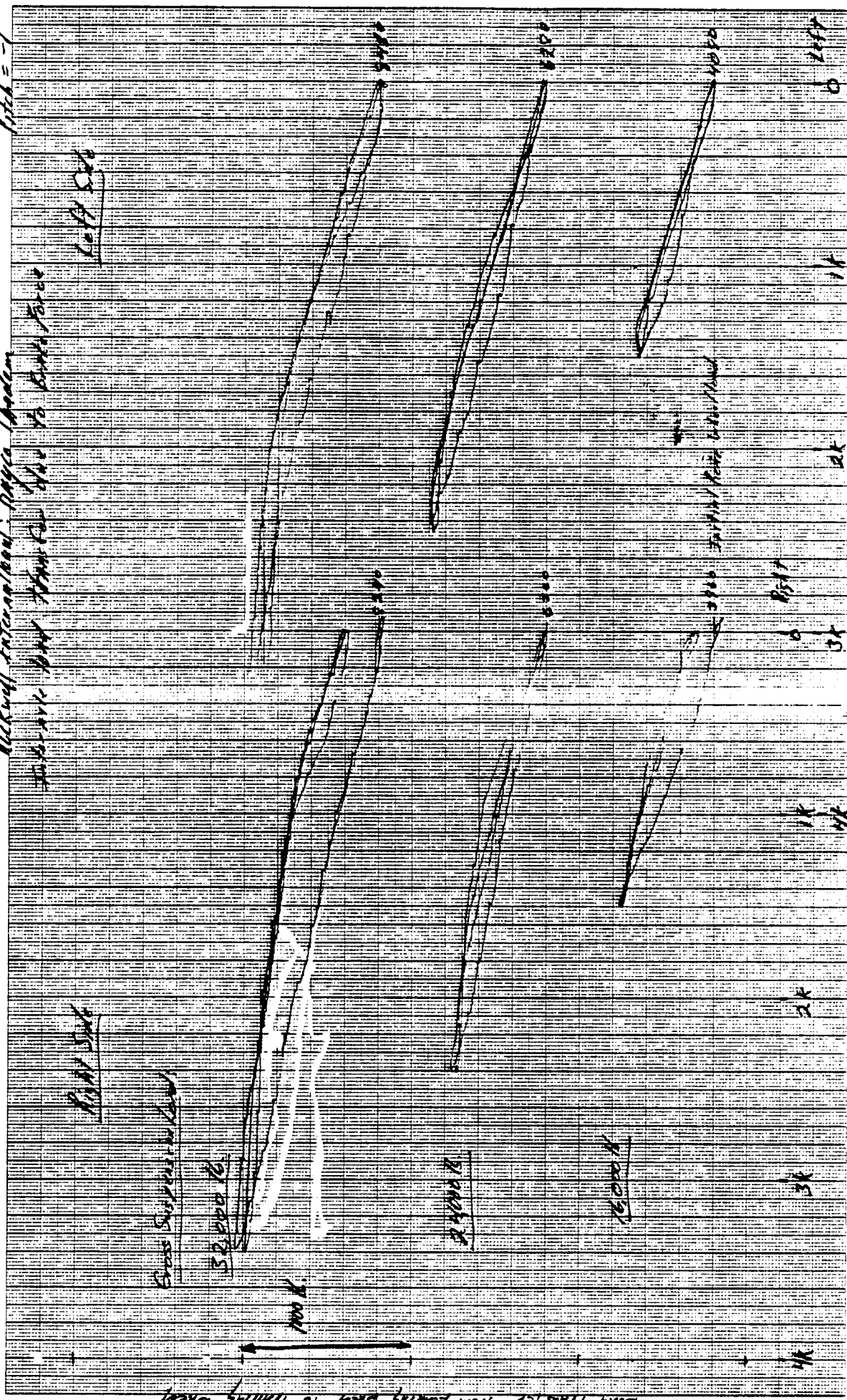
52000 lb

1000 lb

21000 lb

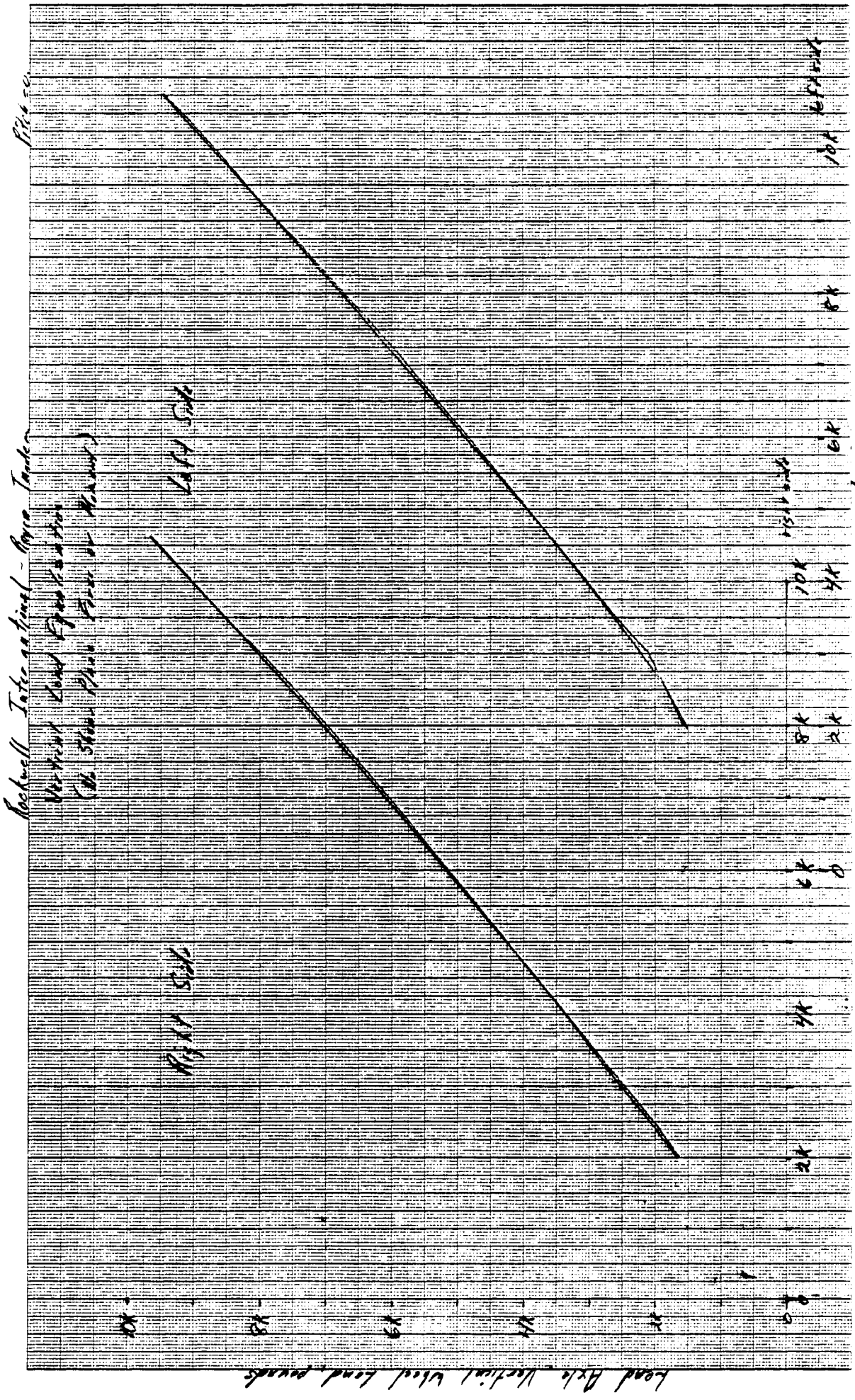
16000 lb

Left Side



Load Transfer from Leading Wheel to Trailing Wheel

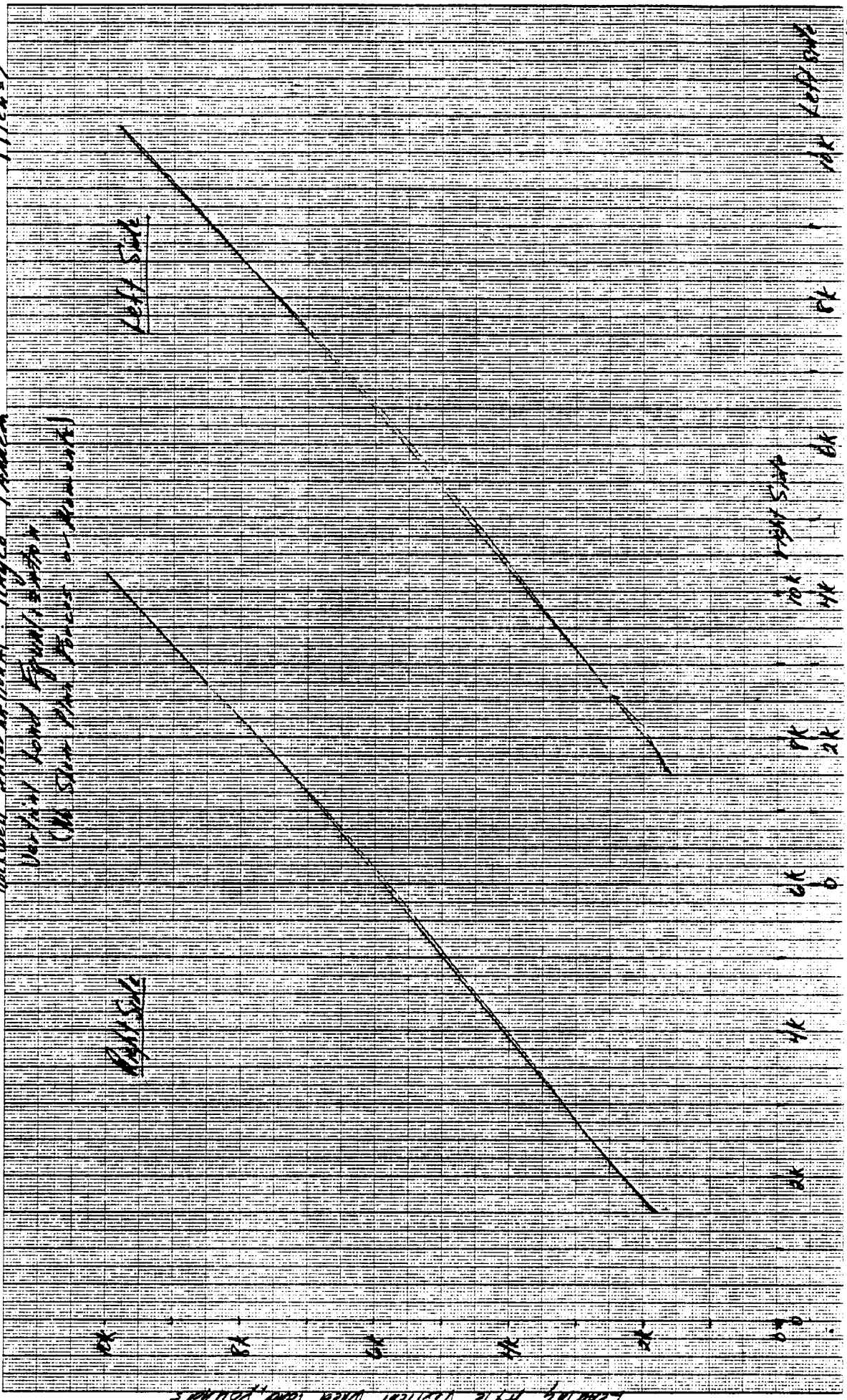
Brake force per wheel set, pounds  
Applied to all four simultaneously



Trailing Axle, Vertical Wheel Load, Pounds

Pitch = 7°

Rockwell International - Rayco Tandem  
Vertical Wheel Load  
(All Star Plus Trucks - 4-wheel drive)



Trailing Axle Vertical Wheel load, pounds.

Suspension: Rayco 4-Spring (MacK)

		<u>32000</u>	<u>24000</u>	<u>16000</u>
			Susp Load	
Vertical Rate (lb/in)		8250	7000	6000
Coulomb Friction (lb)		1200	1000	650
Roll RATE (in/lb/deg)	F	120K	105K	90K
	R	145K	120K	85K

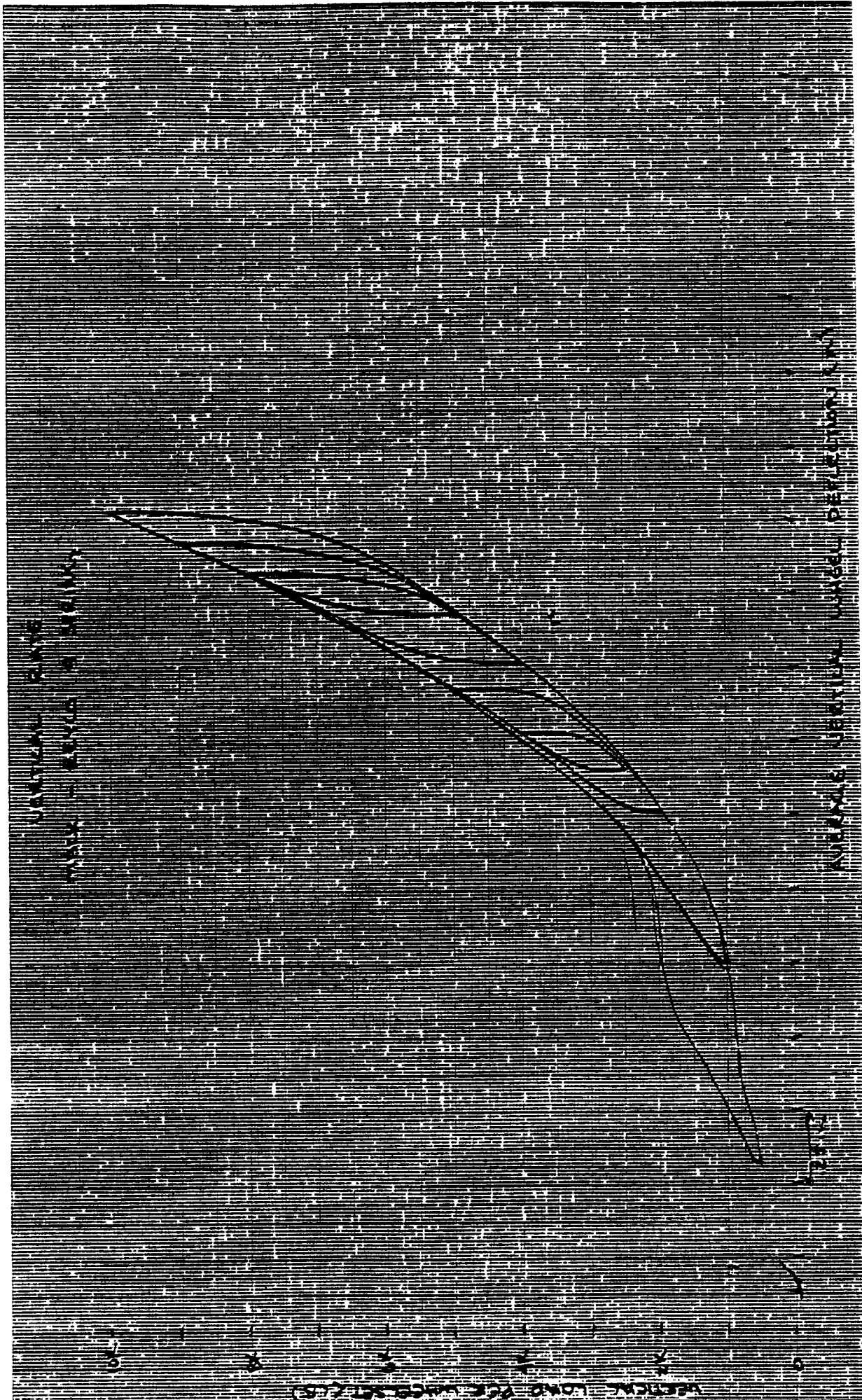
Roll CENTER HT (in below top of frame)	F	9.1	9.0	9.3
	R	9.7	10.0	10.6

Roll STEER (deg / deg)	F			
	R			

ALIGNING MOMENT COMPLIANCE STEER F  
 (deg / in lb) R

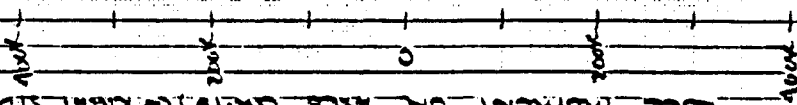
LATERAL FORCE COMPLIANCE F  
 (deg / lb) R

INTERAXLE LOAD TRANSFER (STEERING)  
 (lb / lb)



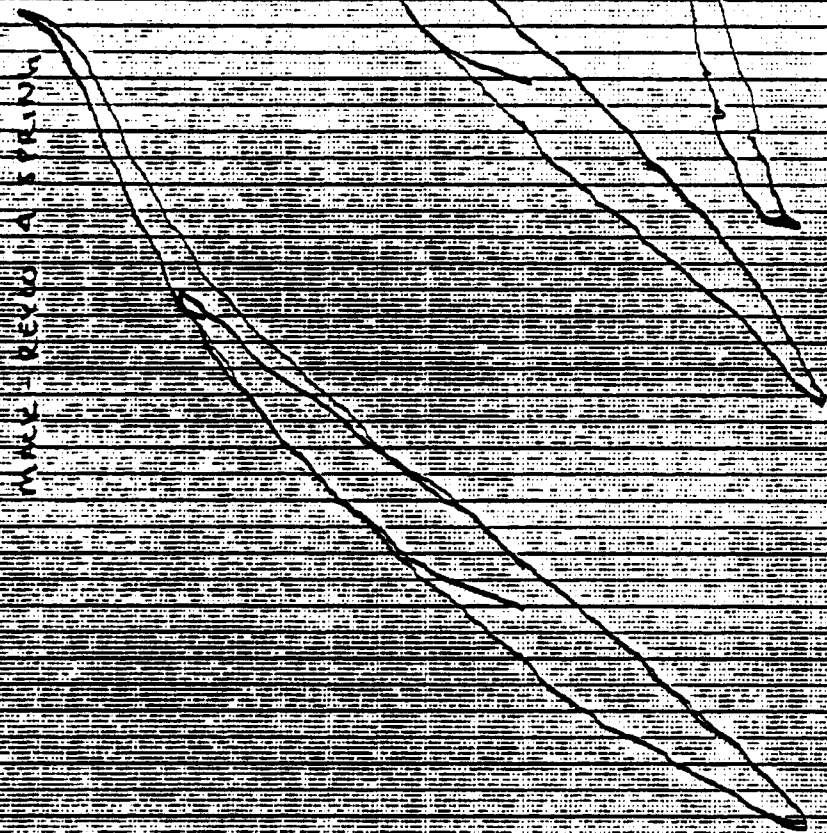


ROLL MOMENT ON AXES (in lb) (RIGHT SIDE COMPRESSED)



1000

ROLL AXIS - LEADING AXLE  
 YAW AXIS - REAR SPRING

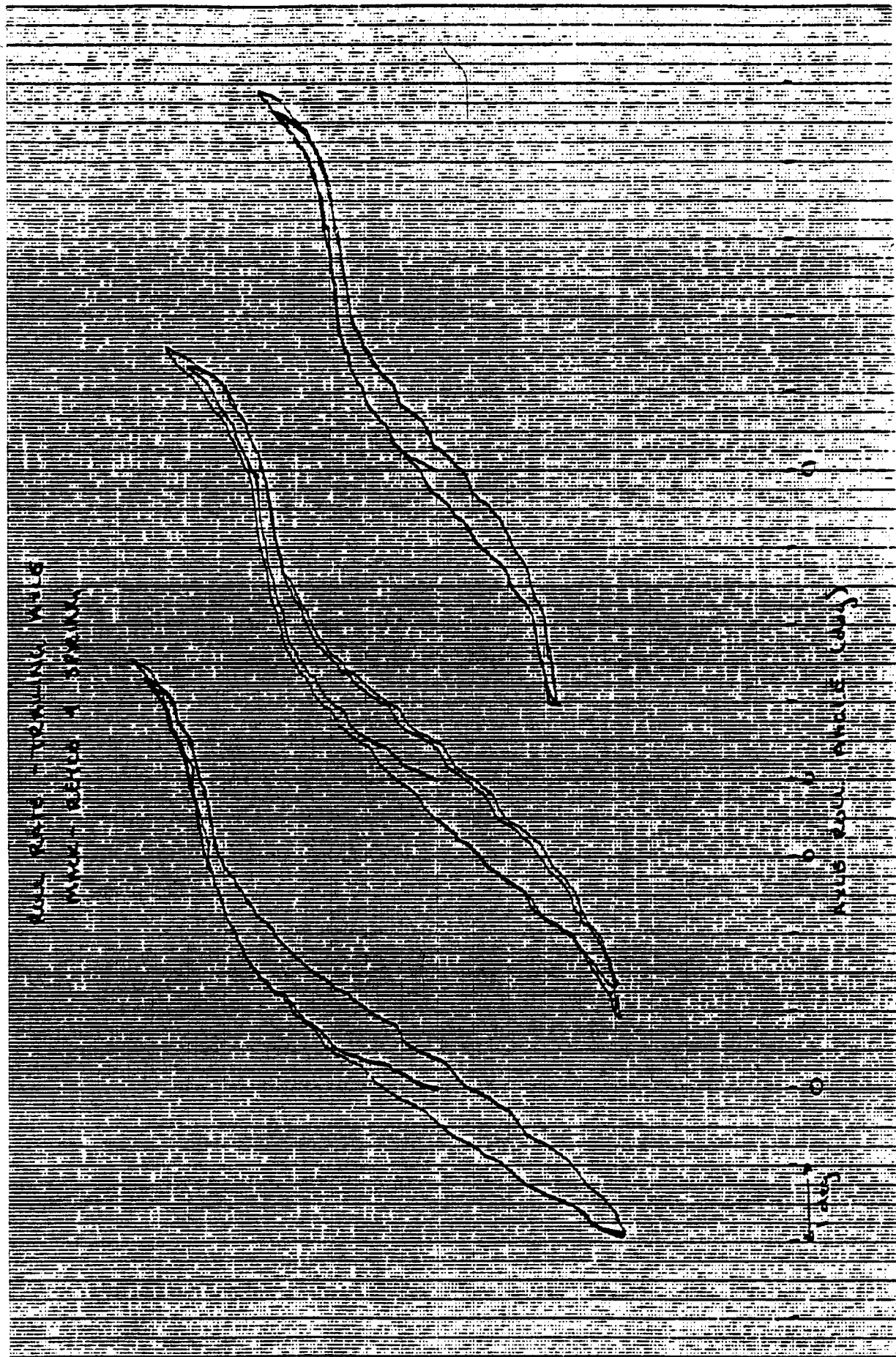


AXIS ROLL ANGLES (deg)

ALL DATA - YOUNG WALS  
MAY 1950 - 4 SPAN

ALL DATA - YOUNG WALS  
MAY 1950 - 4 SPAN

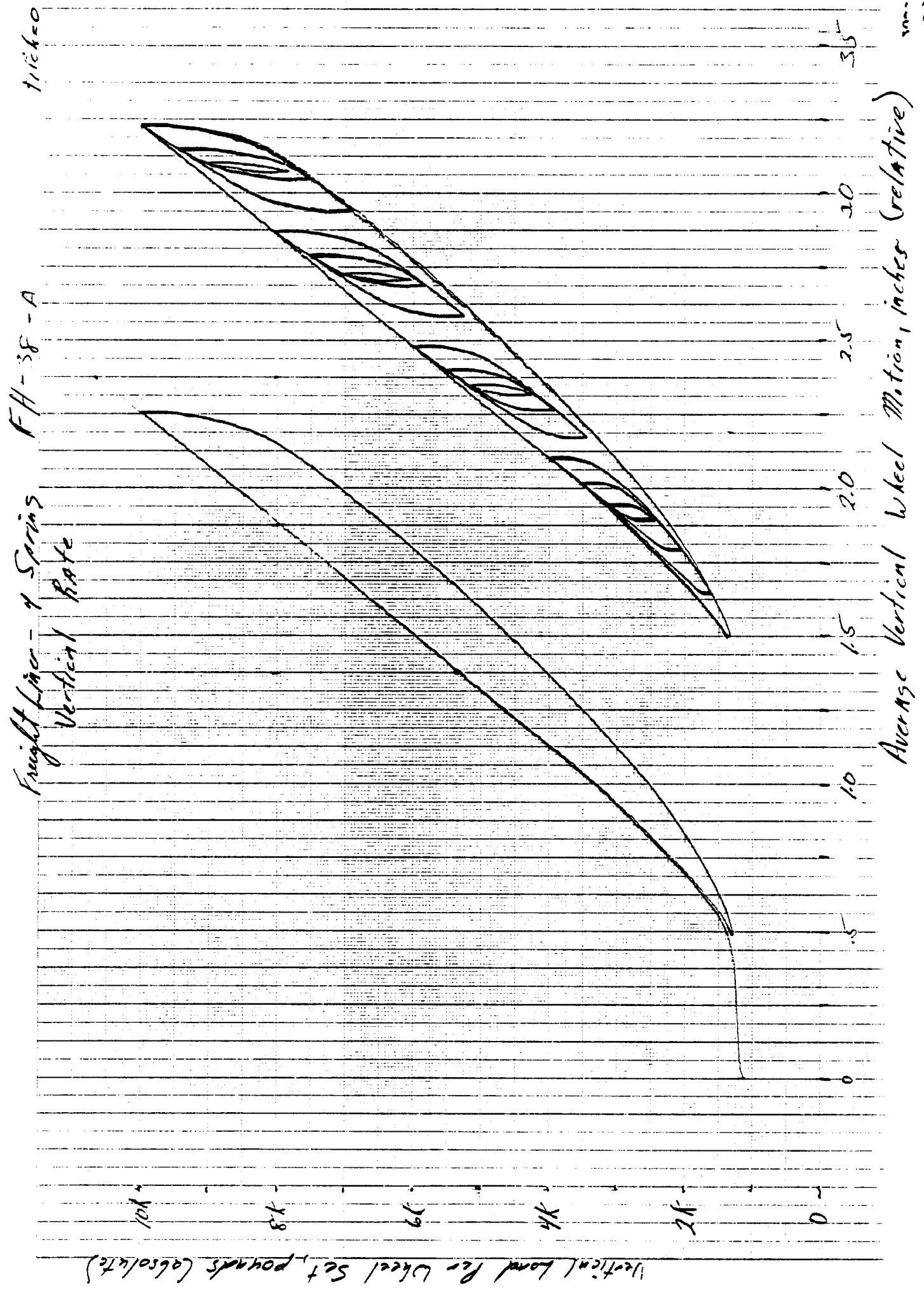
ALL MOMENT ON WALS (IN-LB) (← RIGHT SIDE COMPRESSION)



Suspension: FREIGHTLINER 4 SPRING (FH-38-A)

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<hr/>				
Vertical Rate (lb/in)		5300	5300	5300
Coulomb Friction (lb)		800	700	500
Roll Rate (in/lb/deg)	F	95K	90K	82K
	R	105K	95K	90K
<hr/>				
Roll Center HT (in below top of frame)	F	_____	NA	_____
	R			
<hr/>				
Roll Steer (deg/deg)	F	-0.04	-0.06	-0.07
	R	-0.04	-0.05	-0.06
<hr/>				
Aligning Moment Compliance Steer (deg/in lb)	F	$5 \times 10^{-6}$	$5.5 \times 10^{-6}$	$5.5 \times 10^{-6}$
	R	$2.5 \times 10^{-6}$	$4.5 \times 10^{-6}$	$5.5 \times 10^{-6}$
Lateral Force Compliance (deg/lb)	F	$2.2 \times 10^{-5}$	$3 \times 10^{-5}$	$4 \times 10^{-5}$
	R	$7 \times 10^{-6}$	$1 \times 10^{-5}$	$1.3 \times 10^{-5}$
Interaxle Load Transfer (lb/lb)	L	.34	.29	.30
	R	.26	.32	.20

4 SPRINGS



Vertical Load Per Wheel Set, pounds (absolute)

Average Vertical Wheel Motion, inches (relative)

Freight Liner - 4 Springs  
Vertical Rate

FH-38-A

11 inch = 0

Pitch no

Freightliner 4-Spring  
Vertical Load Equilibrium

Right side

Left side

Trailing Axle, Vertical Load Pounds.

10K 8K 6K 4K 2K 0 0

10K

8K

6K

10K

4K

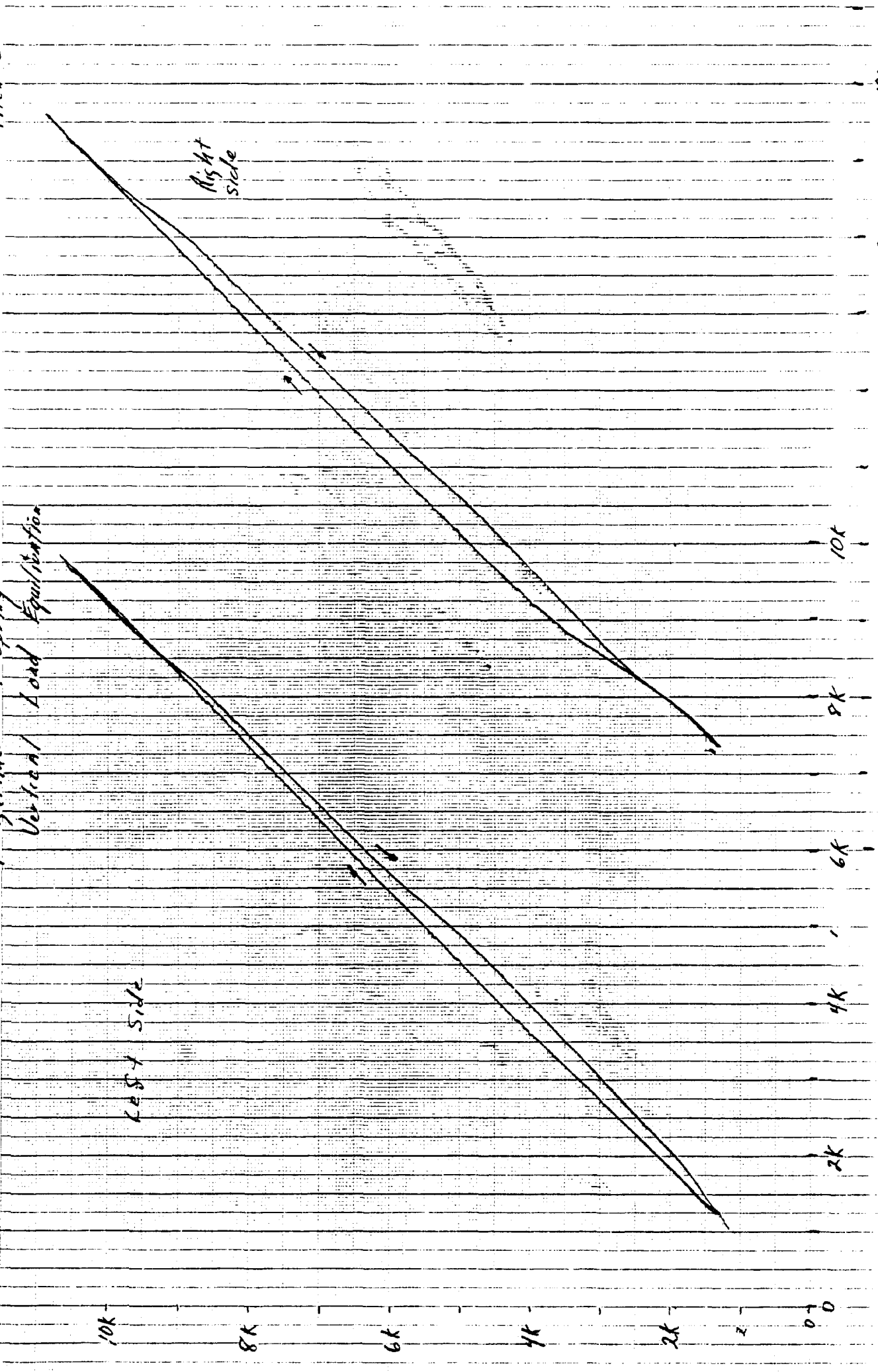
2K

6K

4K

2K

Lead Axle, Vertical Load, Pounds





Freightliner - 4 Springs

Roll Rate - Trailing Axle

Gross Suspensions brch.

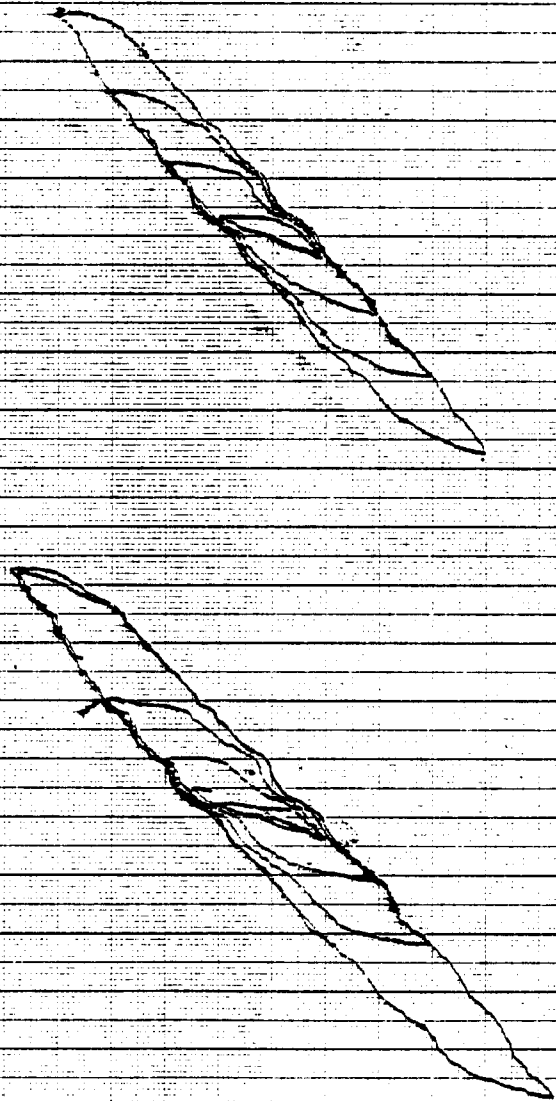
32,000 lb

29,000 lb

16,000 lb

Roll Moment on Axle, In-lb's (Right side Comp →)

4000  
2000  
0  
2000  
4000



Axle Roll Angle, deg.

Pos.

Pitch = 0

Freightliner 4-5 Spring  
Roll Rate, Landing Axle

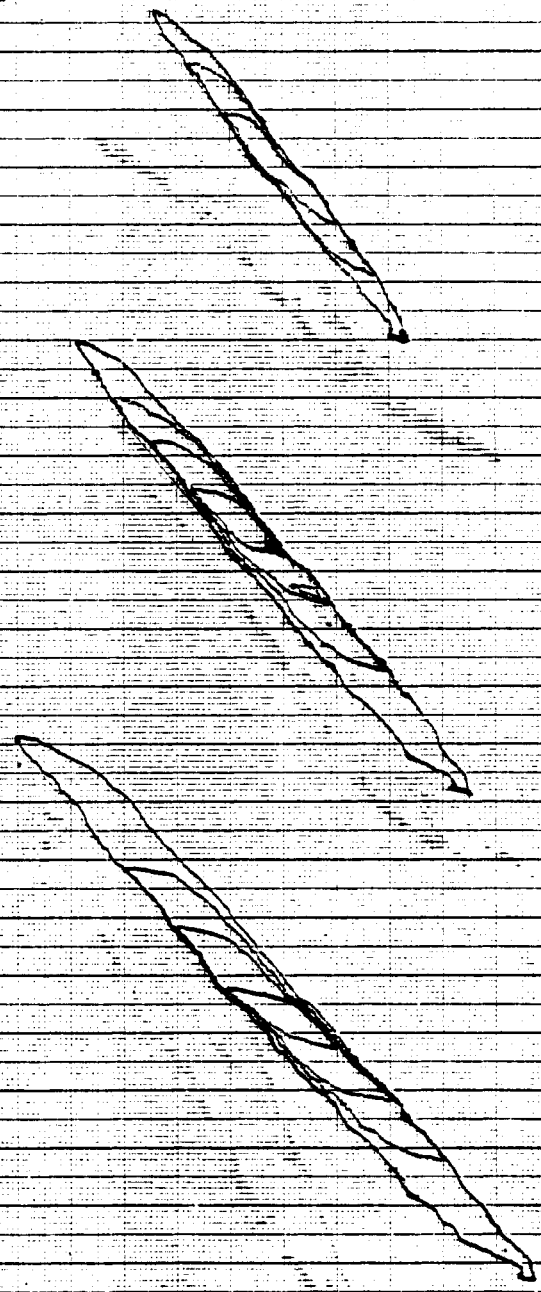
Grise Suspension Load  
32,000 lb

24,000 lb

16,000 lb

Roll Moment on Axle, In-lbs (Right side Comp. →)

4000  
2000  
0  
2000  
4000



Axle Roll Angle, deg

PITCH = 0

FREIGHTLINER - 4 SPRING

ROLL STEER - LEADING AXLE

AXLE STEER ANGLE (← STEERED TO RIGHT)

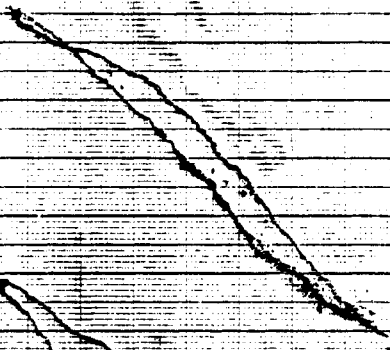
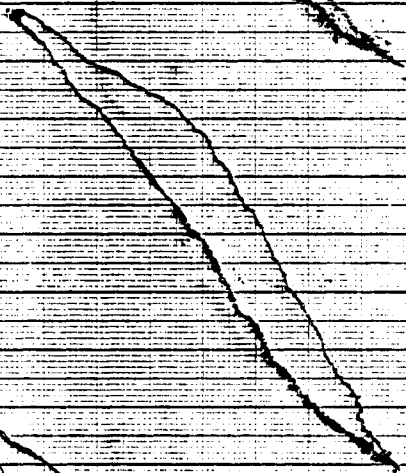
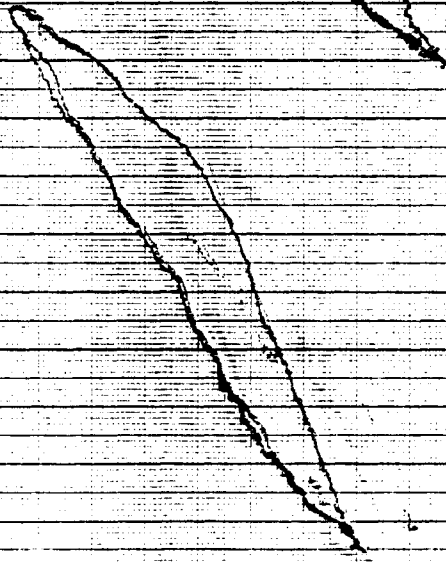
5  
4  
3  
2  
1  
0  
1  
2  
3  
4  
5

Gross Suspension Load

32,000 lb

24,000 lb

16,000 lb



1 DEG

AXLE ROLL ANGLE (DEG) (← RIGHT SIDE COMPRESSED)

PITCH = 0

FREIGHTLINER - 4 SPRING  
ROLL STEER - TRAILING AXLE

AXLE STEER ANGLE (DEG) (← STEER TO RIGHT)

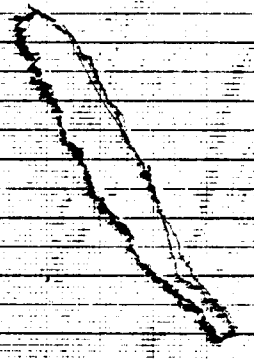
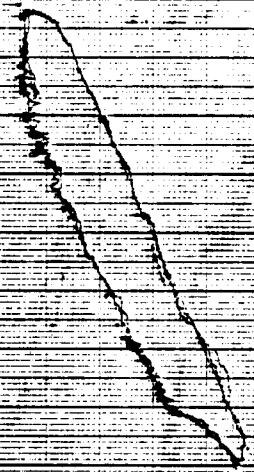
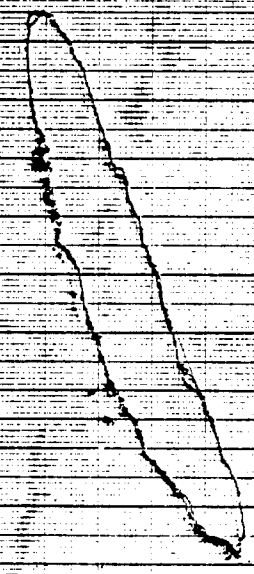
5  
4  
3  
2  
1  
0  
1  
2  
3  
4  
5

Cross Suspension Limit

32,000 lb

24,000 lb

16,000 lb



1 DEGREE

AXLE ROLL ANGLE (DEG)

(← RIGHT SIDE COMPRESSED)

PITCH = 0

Freightliner - 4 Spring  
Aligning Moment Compliance Steer Landing Axle

Aligning Moment Compliance Steer Landing Axle

Gross Suspension Load

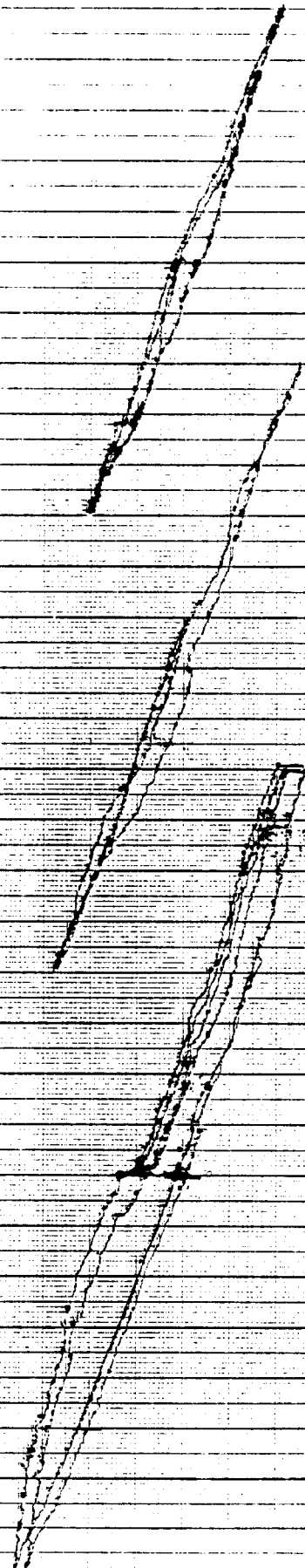
32000 LBS

24000 LBS

14000 LBS

Axle Steer Angle, Degs (w/ wheel toward front right)

2  
1  
0  
-1  
-2



ALIGNING MOMENT PER WHEEL SET (IN-LB)  
(APPLIED TO ALL FOUR SIMULTANEOUSLY)

4000 in-lb



PITCH = 0

FREIGHTLINER - 4 SPRING

ALIGNING MOMENT COMPLIANCE STEER

TRAILING ~~ANGLE~~ AXLE

Gross Suspension Load

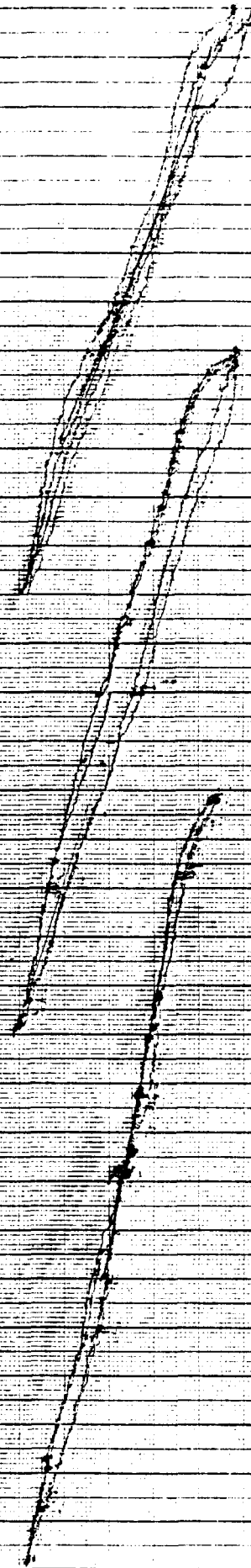
32,000 lbs

24,000 lbs

16,000 lbs

AXLE STEER ANGLE (DEG) (← STEER TO RIGHT)

2  
1  
0  
1  
2

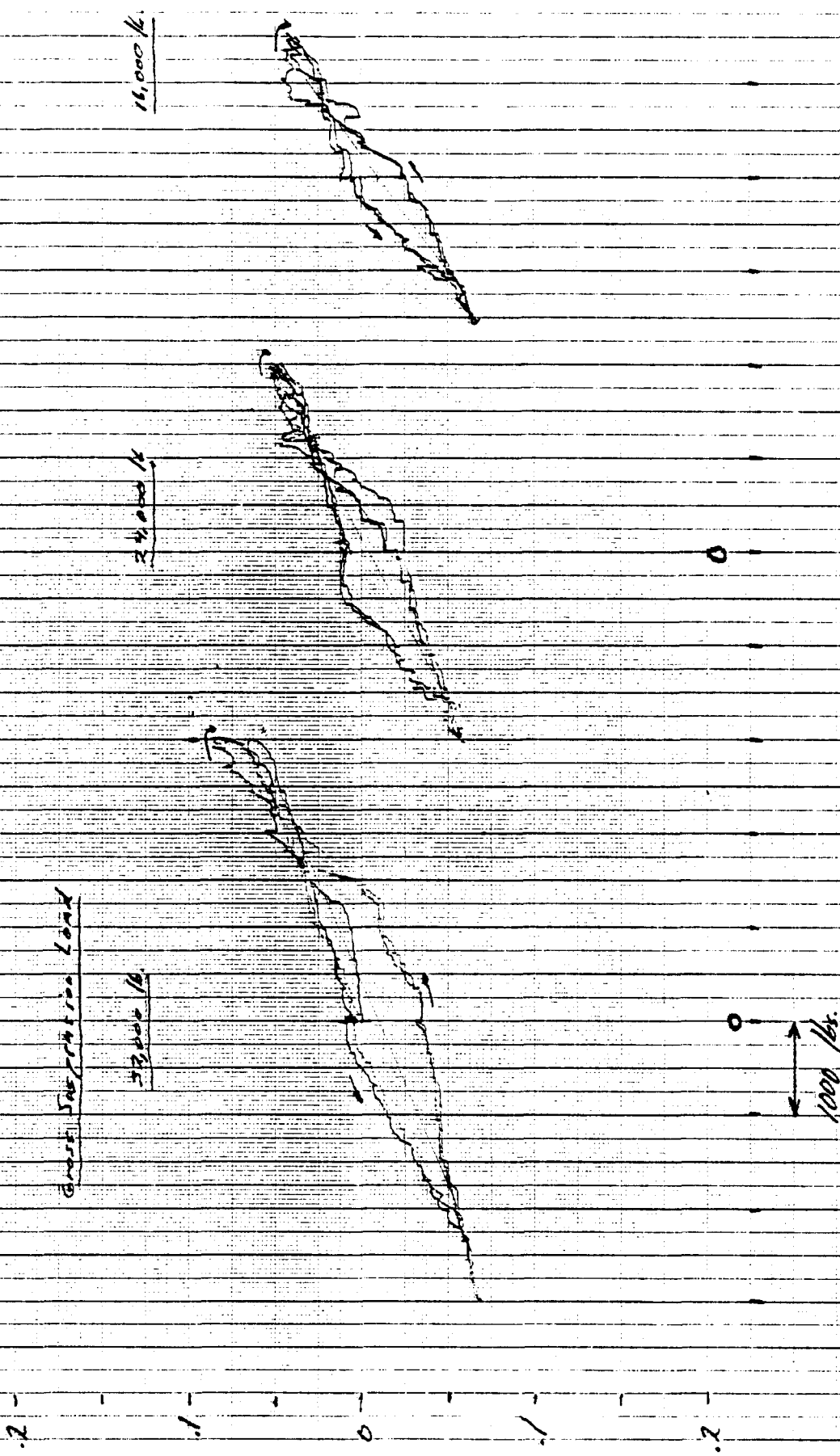


ALIGNING MOMENT PER WHEEL SBT  
(APPLIED TO ALL FOUR SIMULTANEOUSLY)

4000 units

Pitch=0

Freight liner H-Spring  
Lateral Force Compliance Steer-  
Leading Axle



Lateral Force Per Wheel Set, pounds (Force Applied toward right)  
(Applied to all four wheel sets)

Pitch 6-2

Freight Car - 4 Spring  
Lateral Force Compliance Steer  
Trailing Axle

Gross Suspension Load

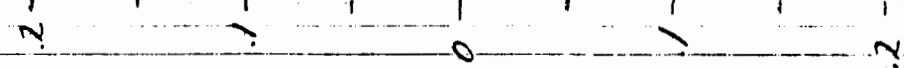
33,000 lb

22,000 lb

1000 lb

Axle Steer Angle, deg (Steer toward right ←)

Lateral Force Per Wheel Set, pounds (Force applied toward right →)  
(Applied to all four simultaneously)



11/16/40

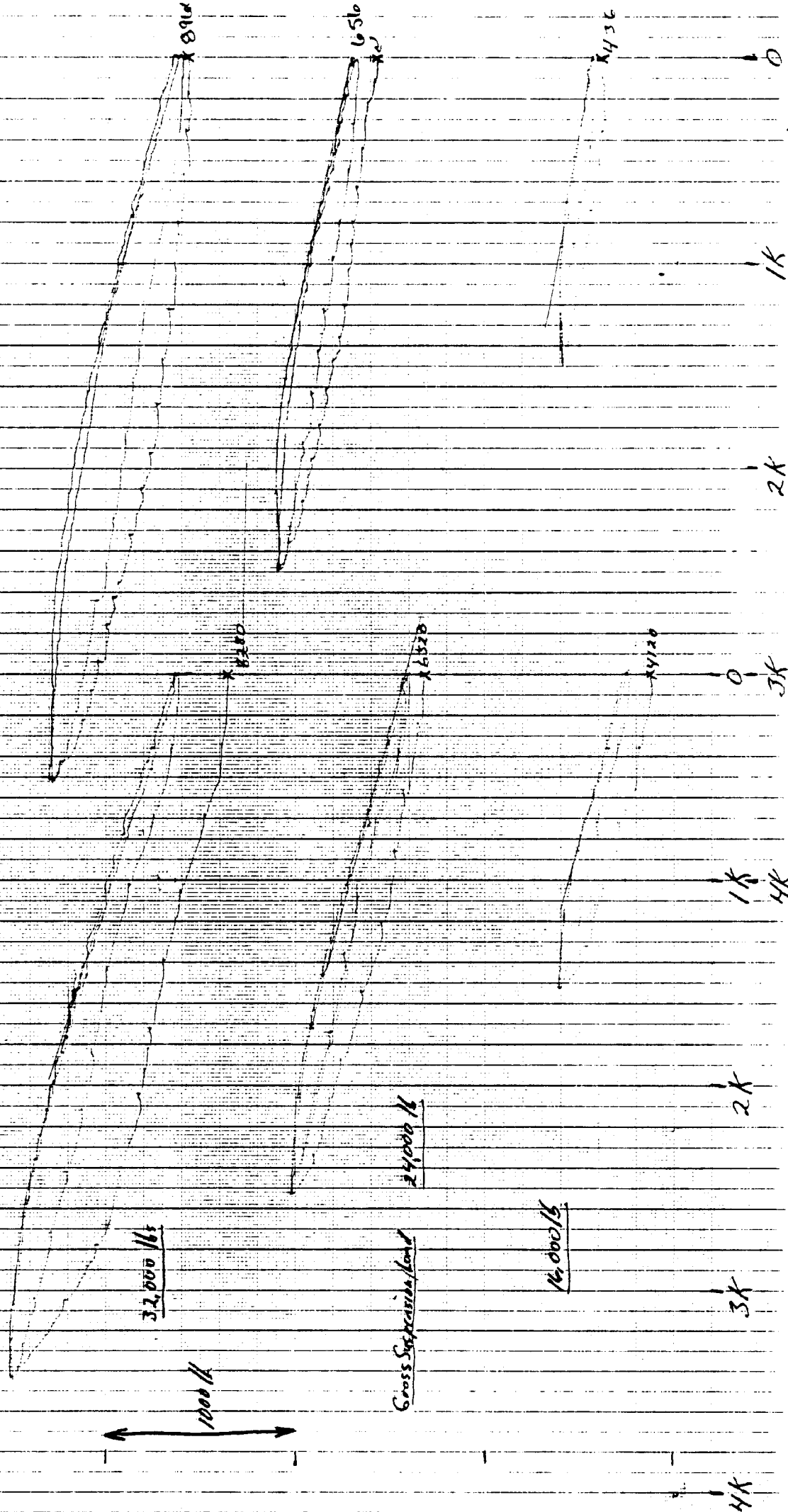
Freightliner - 4 Springs

In turn to load transfer due to Brake Force

Right Side

Left Side

Load transfer from leading to trailing wheel



Brake force per wheel set, pounds. (Applied to all four simultaneous ly.)

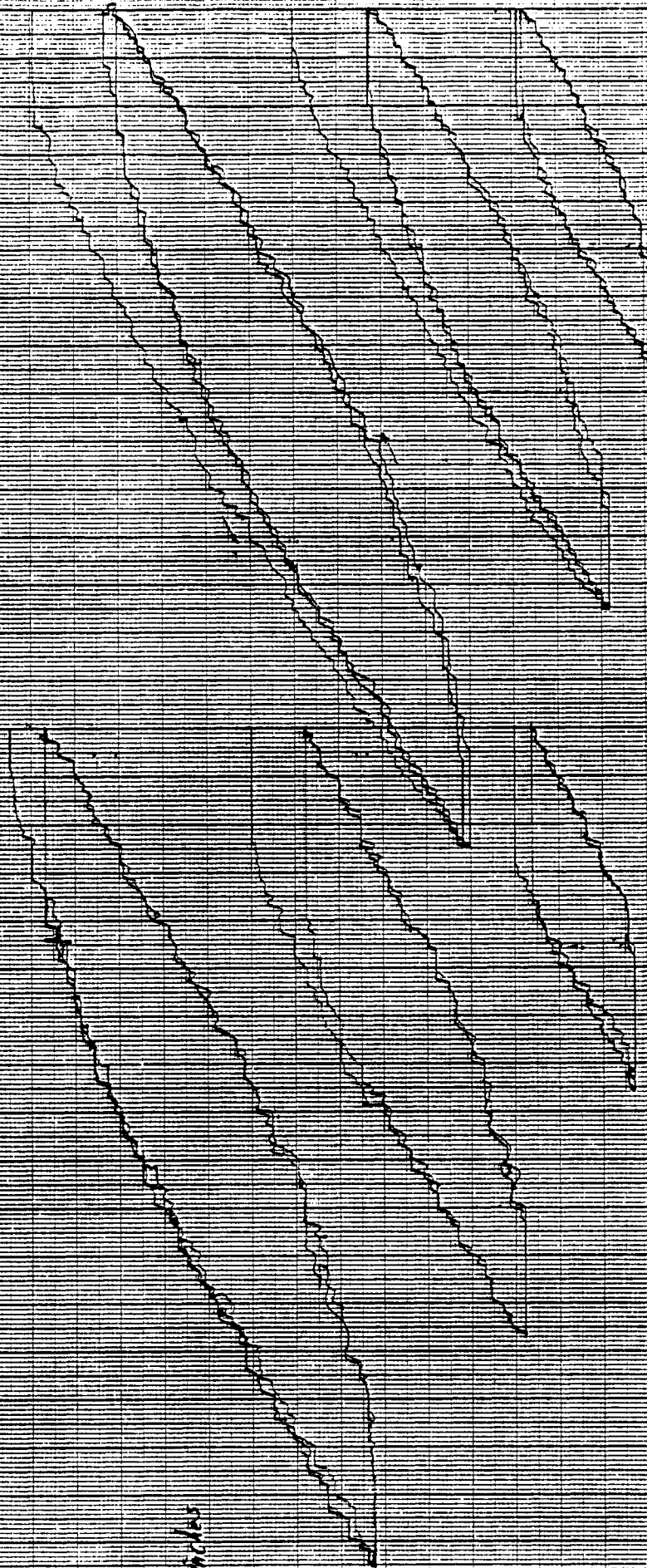
Longitudinal Motion of ARK center, inches (← Forward)

Freight line - 4 Spring  
Longitudinal Motion of ARK Center  
To Brake Force, Trucks, Pk

LEFT SIDE

RIGHT SIDE

1/2 inches



1K

2K

3K

1K

2K

3K

1K

2K

3K

1K

2K

3K

1K

Brake force per wheel set, pounds (all four applied simultaneously)



Freightliner, 4 Spins

Longitudinal Oscillation of Axle Center  
to Brake Force, Leading Axle

Left Side

Right Side

0.1  
Inches

Longitudinal Motion of Axle Center, inches (→ rearward)

5K

3K

2K

1K

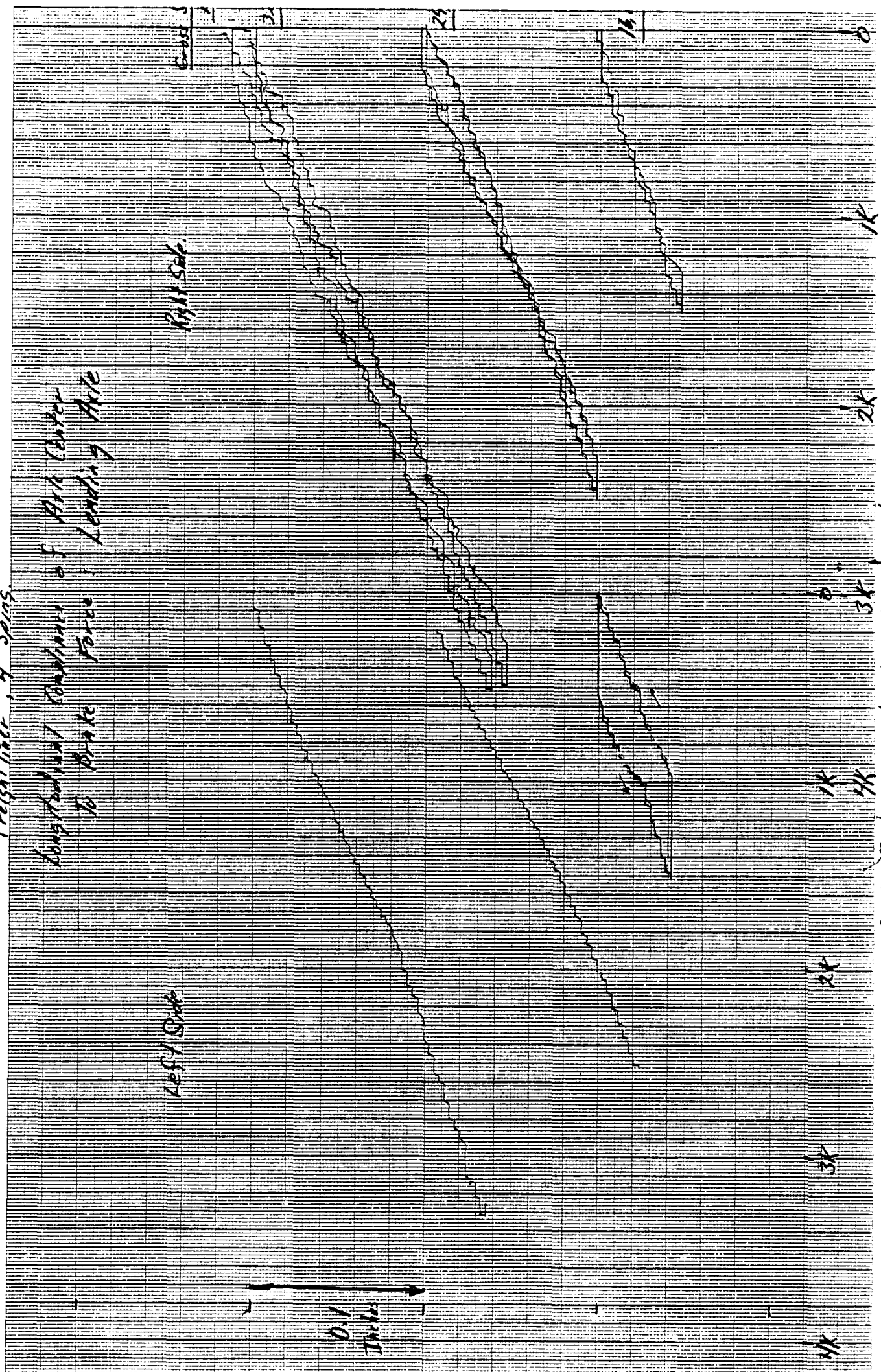
0

2K

1K

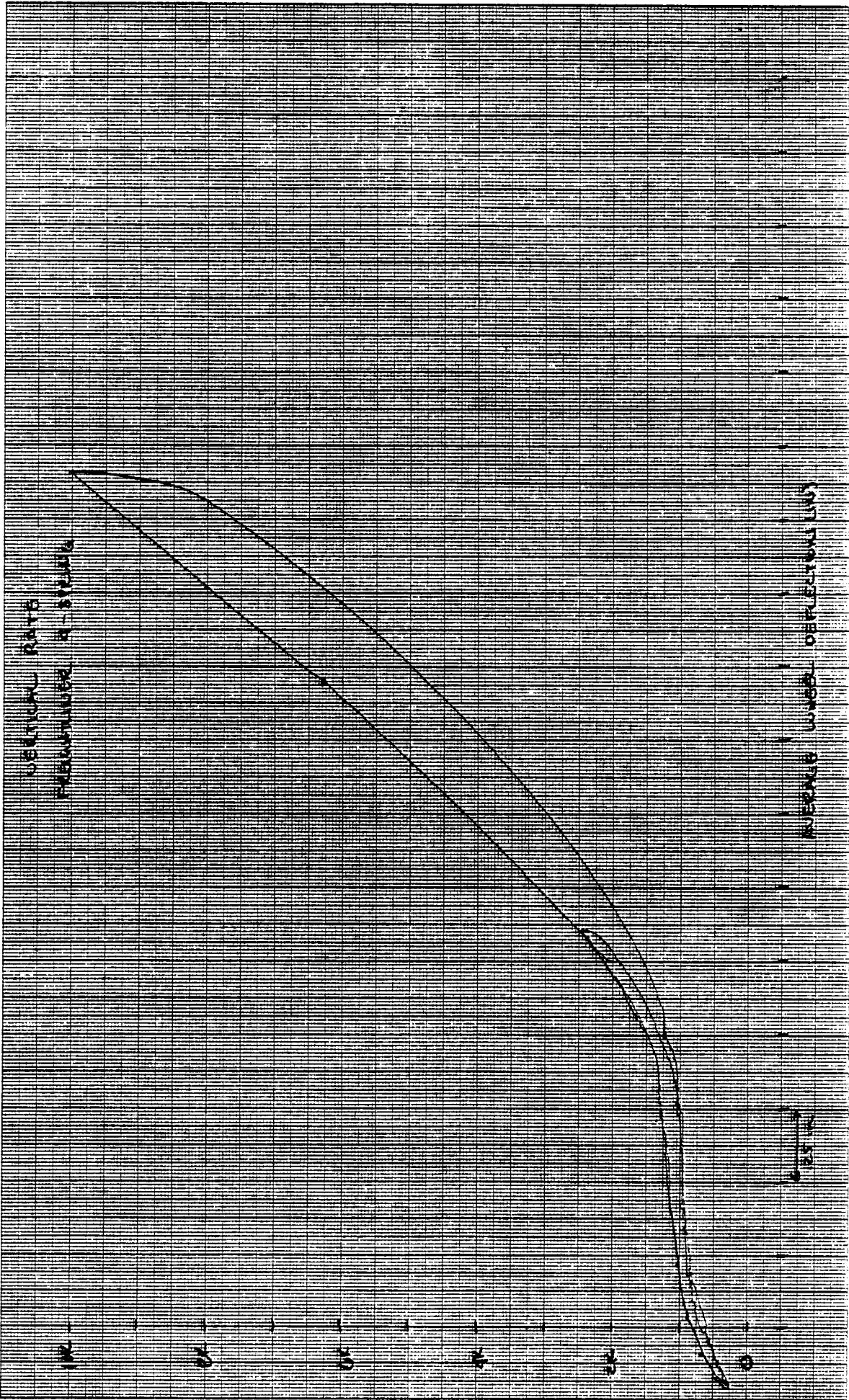
0

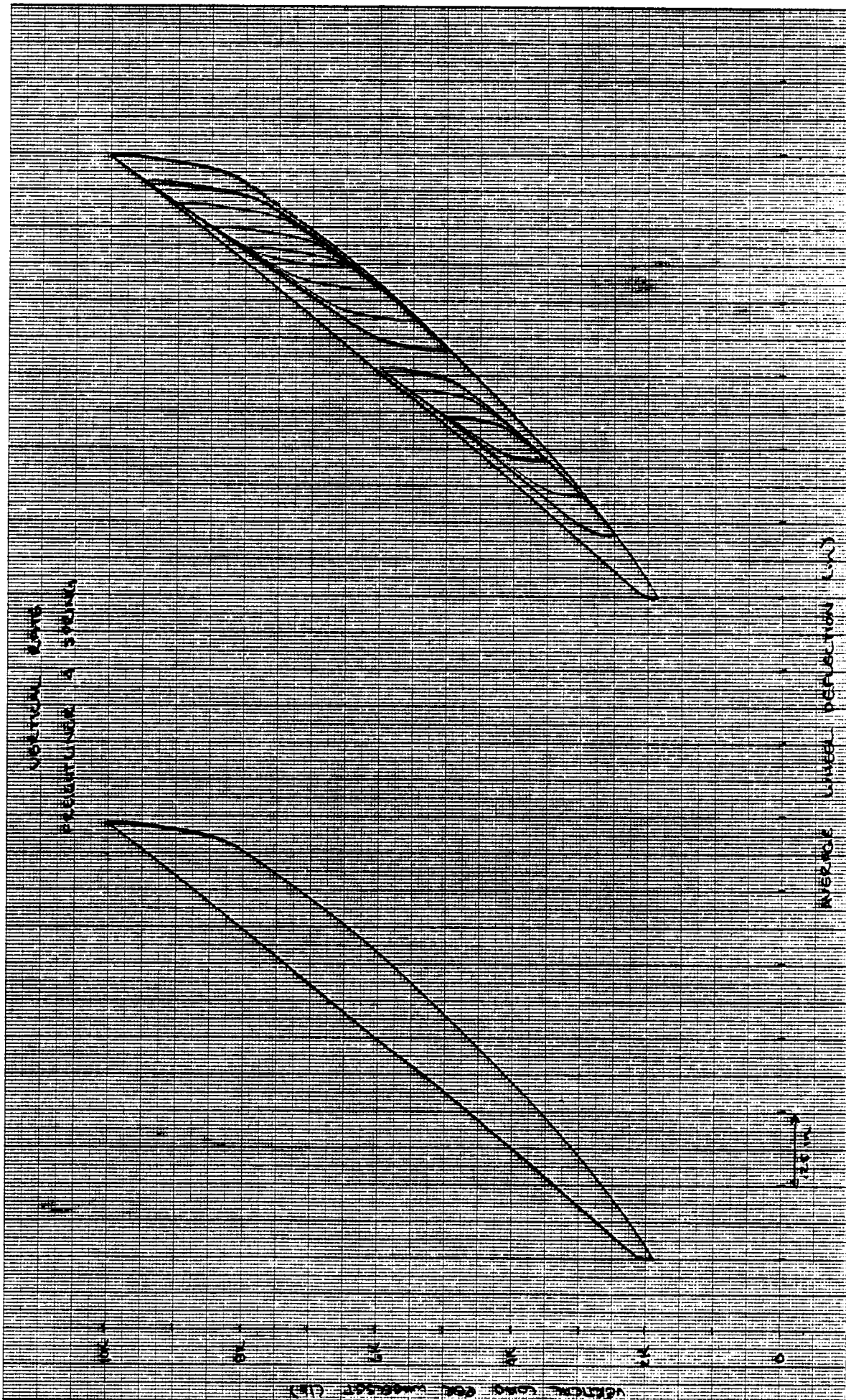
Brake Force per wheel set, pounds



Suspension: FREIGHTLINER 4 SPRING

		<u>32000</u>	Susp Load <u>24000</u>	<u>16000</u>
Vertical Rate (lb/in)		5400	5400	5400
Coulomb Friction (lb)		850	800	600
ROLL RATE	F	90K	85K	75K
(in./lb/deg)	R	90K	80K	70K
ROLL CENTER HT	F	12.3	12.9	13.2
(in below top of frame)	R	10.2	10.5	10.8
ROLL STEER	F			
(DEG / DEG)	R			
ALIGNING MOMENT COMPLIANCE STEER	F			
(deg/in lb)	R			
LATERAL FORCE COMPLIANCE	F			
(deg/lb)	R			
INTERAXLE LOAD TRANSFER (BANK)				
(lb/lb)				



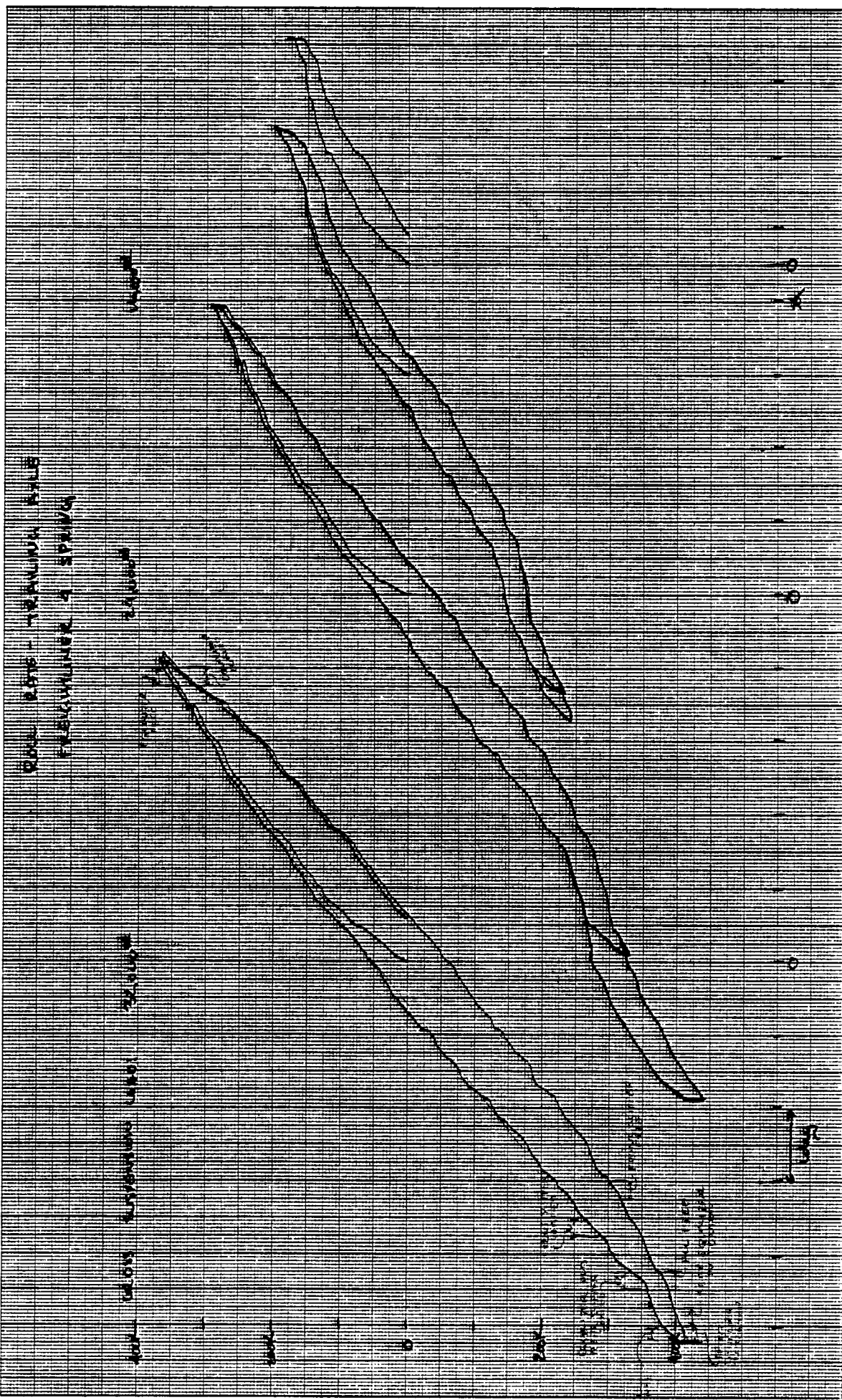


100 mm







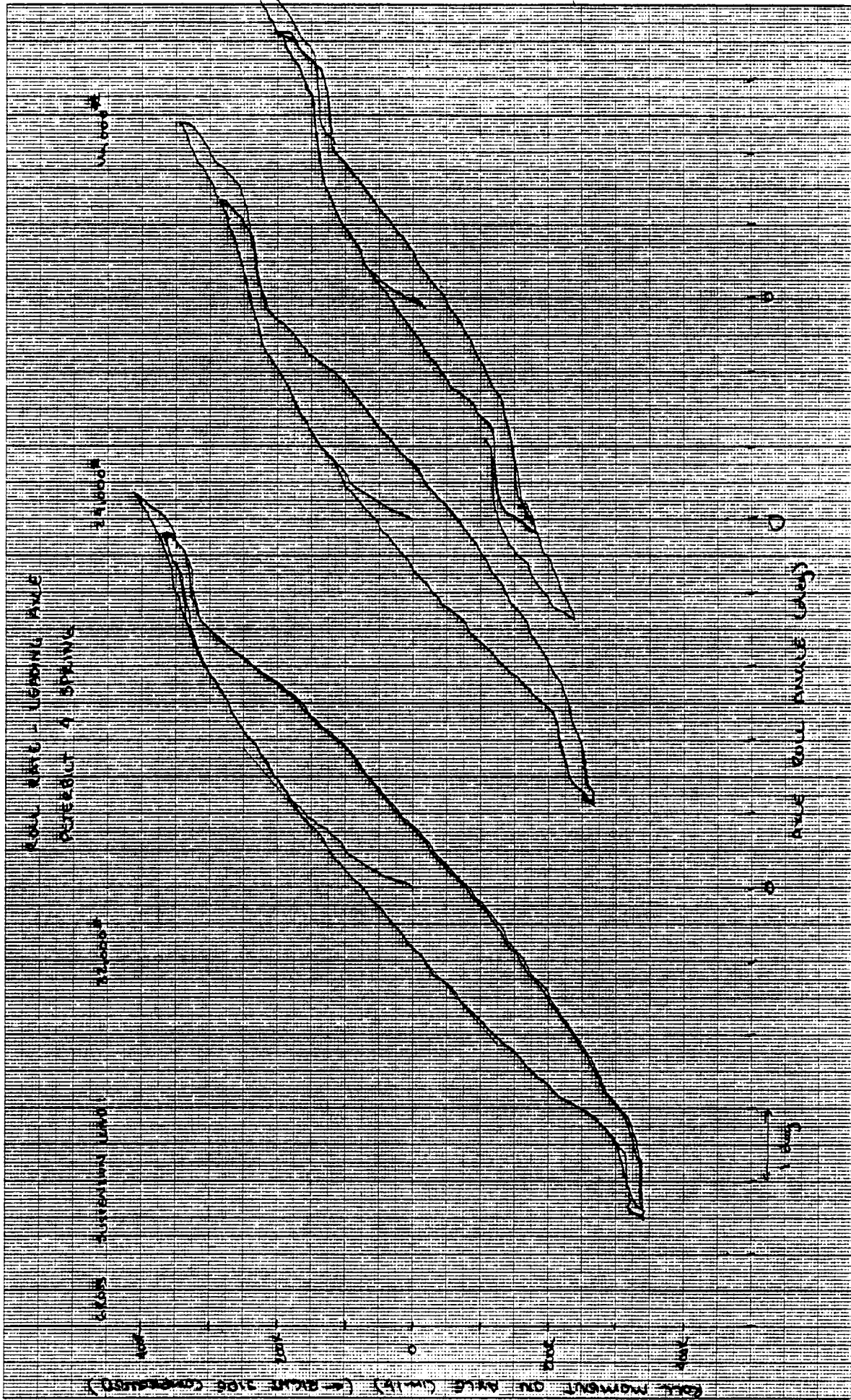


0000 2000 4000 6000 8000  
 10000 12000 14000 16000 18000  
 20000 22000 24000 26000 28000  
 30000 32000 34000 36000 38000  
 40000 42000 44000 46000 48000  
 50000 52000 54000 56000 58000  
 60000 62000 64000 66000 68000  
 70000 72000 74000 76000 78000  
 80000 82000 84000 86000 88000  
 90000 92000 94000 96000 98000  
 100000 102000 104000 106000 108000  
 110000 112000 114000 116000 118000  
 120000 122000 124000 126000 128000  
 130000 132000 134000 136000 138000  
 140000 142000 144000 146000 148000  
 150000 152000 154000 156000 158000  
 160000 162000 164000 166000 168000  
 170000 172000 174000 176000 178000  
 180000 182000 184000 186000 188000  
 190000 192000 194000 196000 198000  
 200000 202000 204000 206000 208000  
 210000 212000 214000 216000 218000  
 220000 222000 224000 226000 228000  
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 240000 242000 244000 246000 248000  
 250000 252000 254000 256000 258000  
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 390000 392000 394000 396000 398000  
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Suspension: PETERBILT 4-SPRINK

	Susp Load		
	<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>tr</del>			
Vertical Rate (lb/in)	7600	7600	7600
Coulomb Friction (lb)	1450	1250	800
Roll RATE (in/lb/deg)	95K	90K	75K
	R	80K	70K
Roll CENTER HT (in below top of frame)	10.5	10.5	10.8
	R	13.7	14.3
Roll STEER (deg/deg)	F		
	R		
ALIGNING MOMENT COMPLIANCE STEER (deg/in/lb)			R
LATERAL FORCE COMPLIANCE (deg/lb)			R
INTERAXLE LOAD TRANSFER (BYRANK) (lb/lb)			











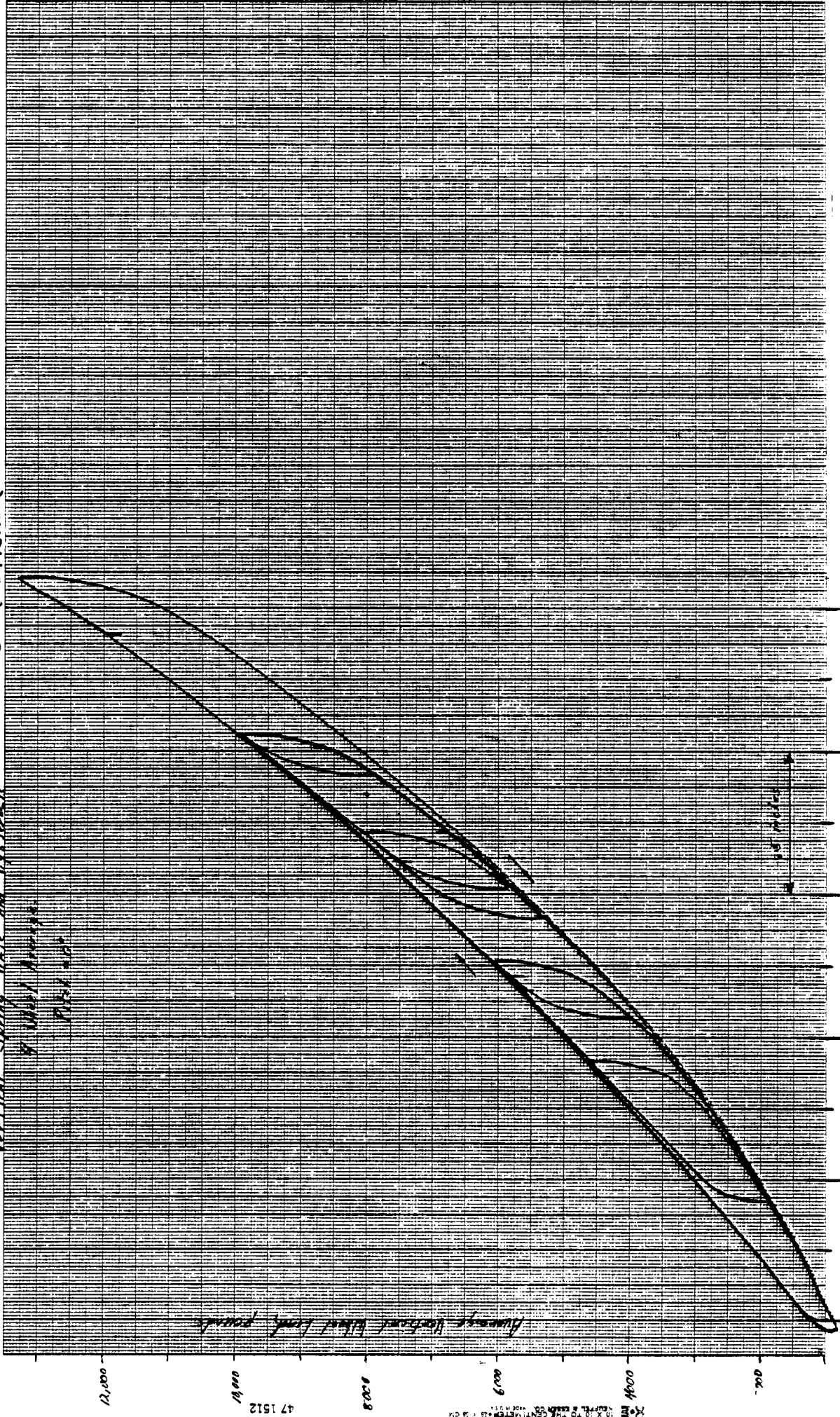
Suspension: white 4-Spring

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
Vertical Rate (lb/in)		4700	4400	4000
Coulomb Friction (lb)		850	800	600
Roll RATE	F	77.4K	73.8K	64.8K
(in. lb / deg)	R	68.4K	64.8K	54K
Roll CENTER HT	F	7.79	8.14	8.31
(in below top of trans)	R	7.60	8.11	9.44
Roll STEER	F	.09	.08	.01
(DEG / DEG)	R	.01	.03	.04
ALIGNING MOMENT COMPLIANCE STEER F (deg / in lb)	R		← N/A →	
LATERAL FORCE COMPLIANCE F (deg / lb)	R	$10^{-5}$	nil	nil
INTERAXLE LOAD TRANSFER (PERCENT) L		.31	.32	.26
(lb / lb) + → Trailing to rear	R	.35	.34	.34
@ 0° Pitch				

WHITE 4-SPRING

Vertical Spring Rate and Hysteresis

8 Wheel Average  
Pilot 200



Average Horizontal Wheel Load, pounds

Average Vertical Deflection, inches

2,000

1,000

47 1512

800

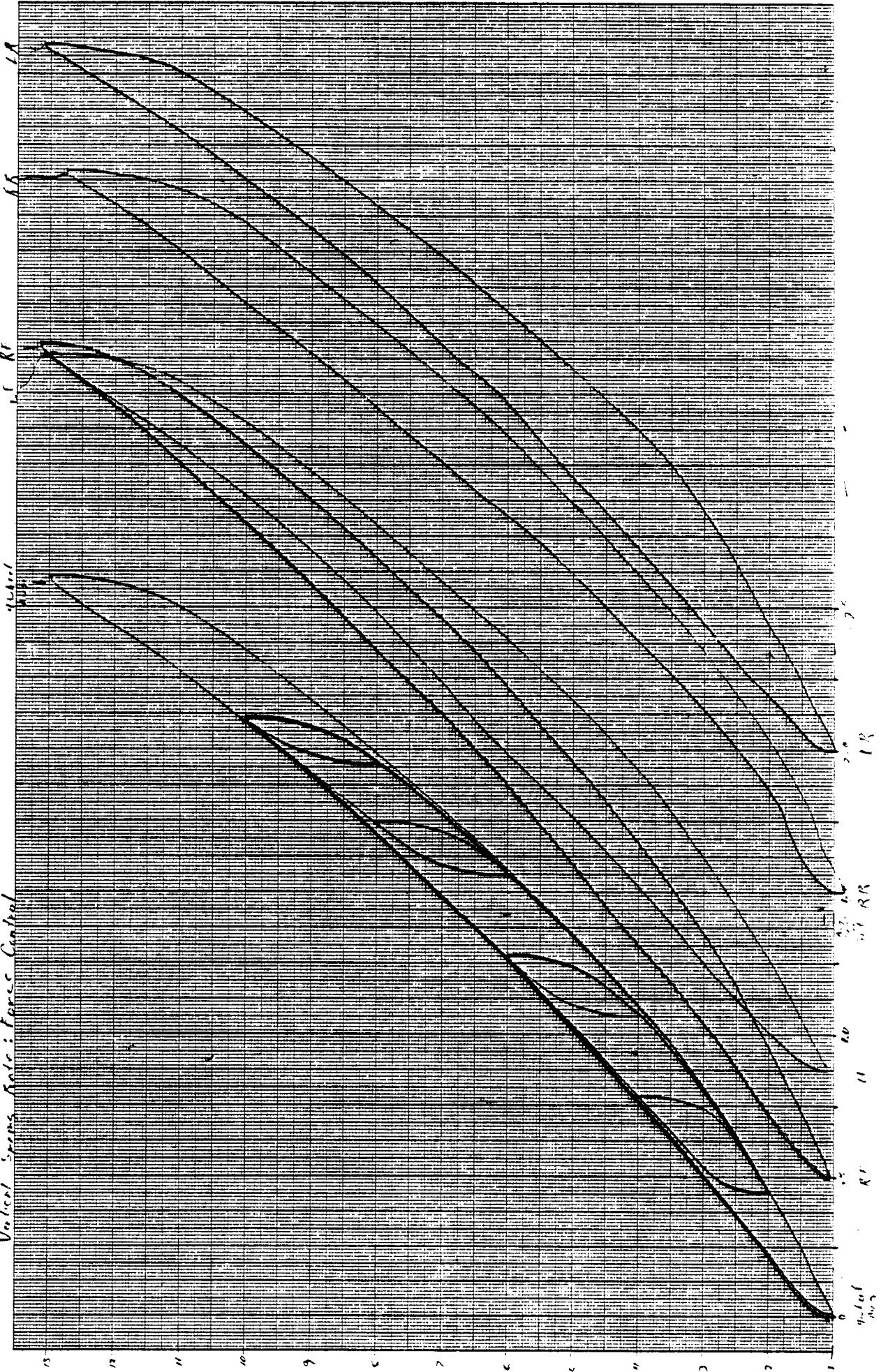
600

400

200

10 X 10 TO THE CENTIMETER 25 X 25 CM  
EUPHONIA INSTRUMENT CO. WASHINGTON, D. C.

Vertical Spacing Ratio: Force Control

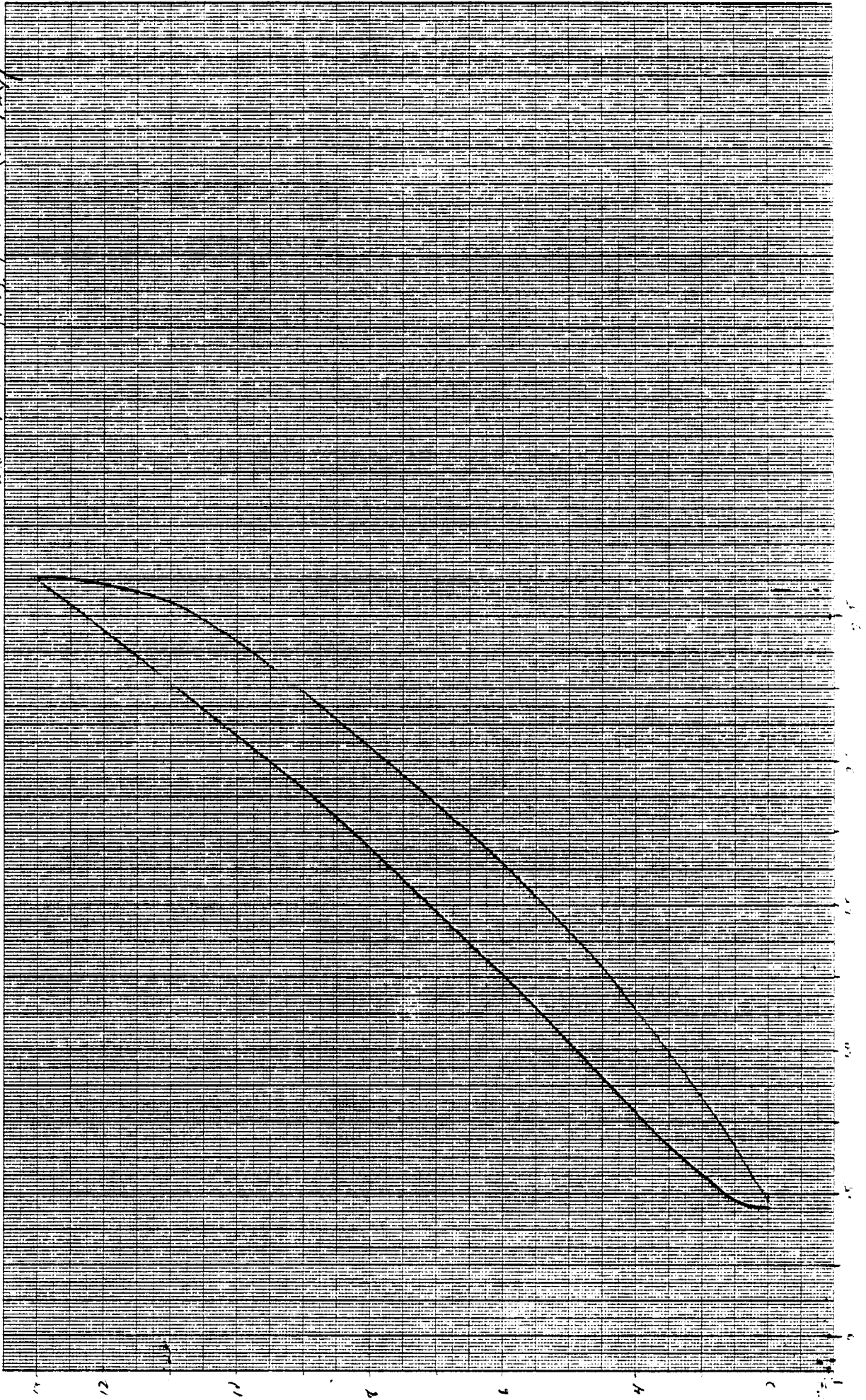


47 1512

$\frac{1}{10}$   
 $\frac{1}{10}$   
 $\frac{1}{10}$

10 X 10 TO THE CENTIMETER A 3 CM  
K. E. KAPPEL & BUSH CO. MADE IN U.S.A.

Vertical Spinning Pos. 7-20 (500 Pa)

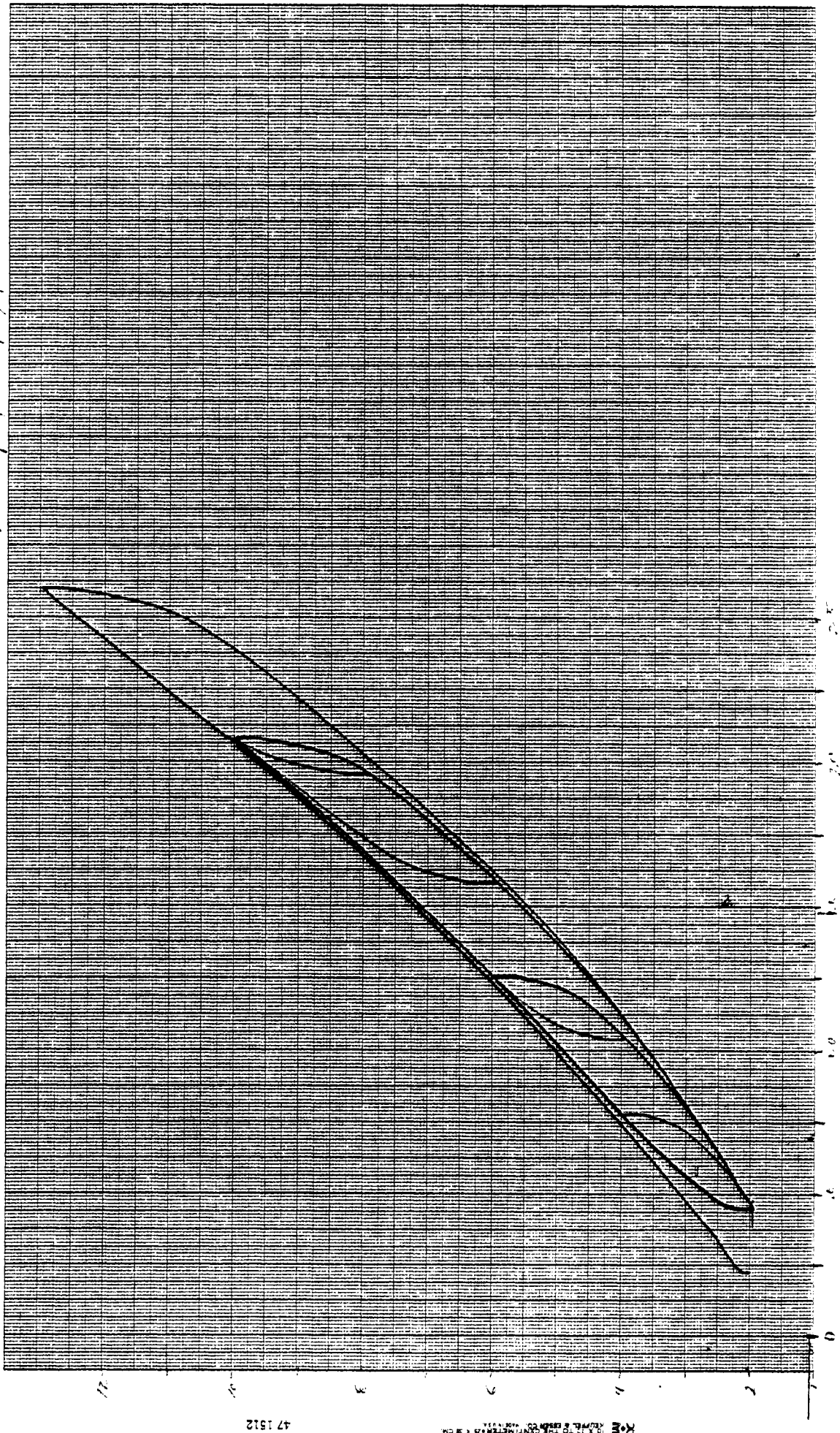


47 1512

K: 10.0, 12.0 TO THE CENTIMETER = 0.39 CM



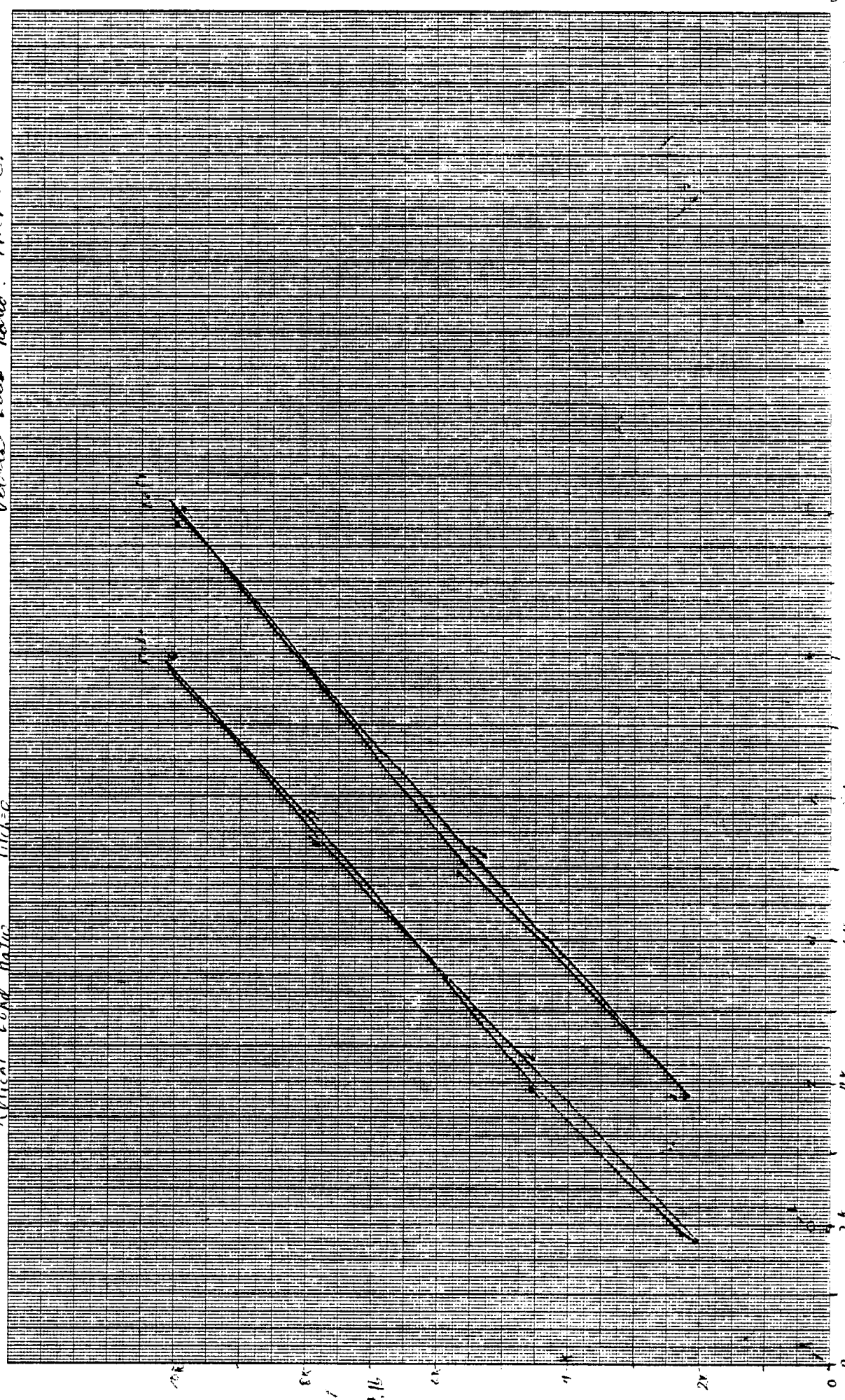
Unburnt Spray Rate /  $W_2$ , P-20





Vertical Load Ratio: Pitch = 0°

Vertical Load Ratio: Pitch = 0°



47 1512

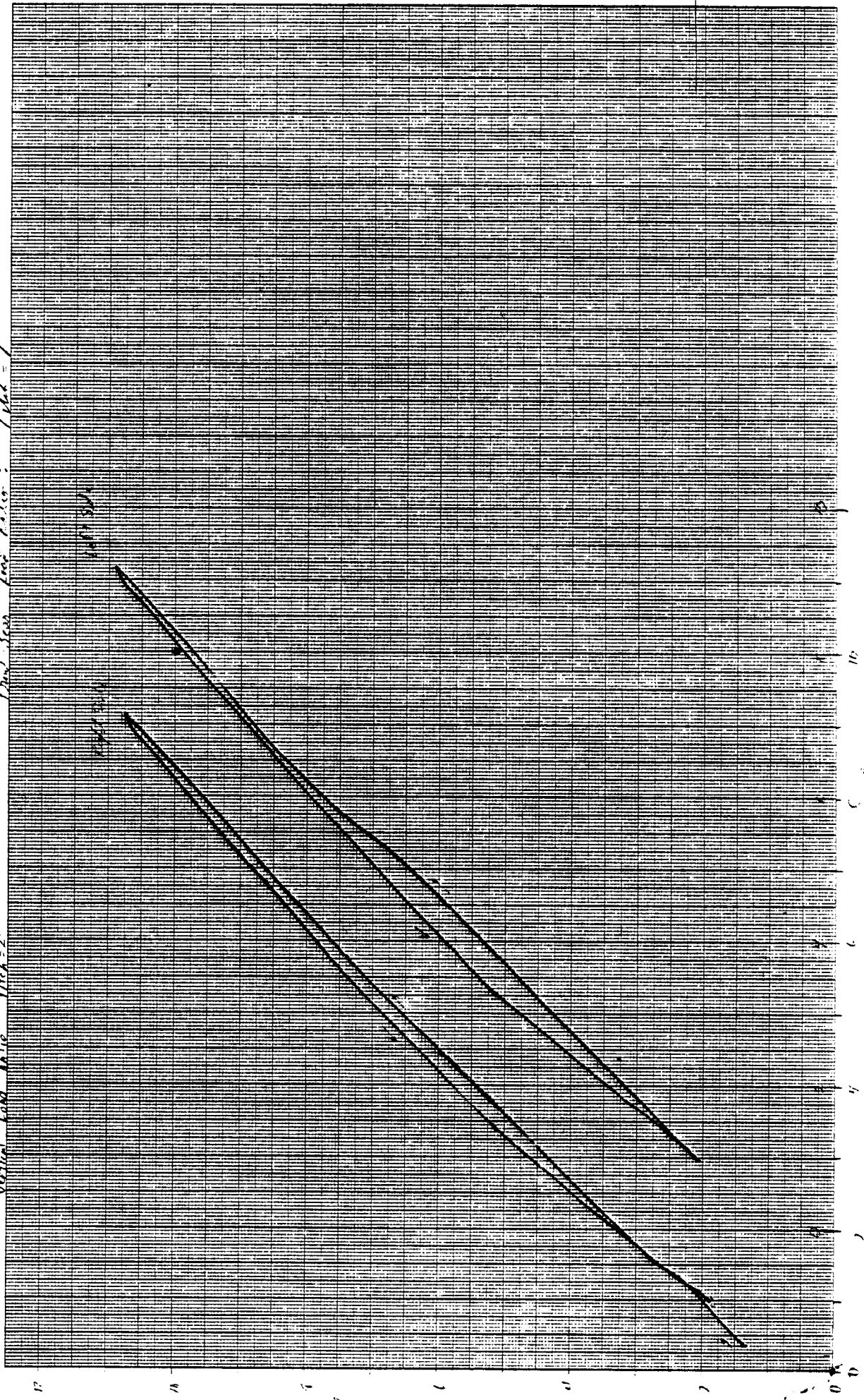
Vertical Load Ratio

Scale: 1 cm = 10 kN, 1 cm = 20

Vertical Load, kN

2

Vertical Load Rate  $\text{Site} = 2^{\circ}$  Final Year Load Rate:  $\text{P} = 1^{\circ}$



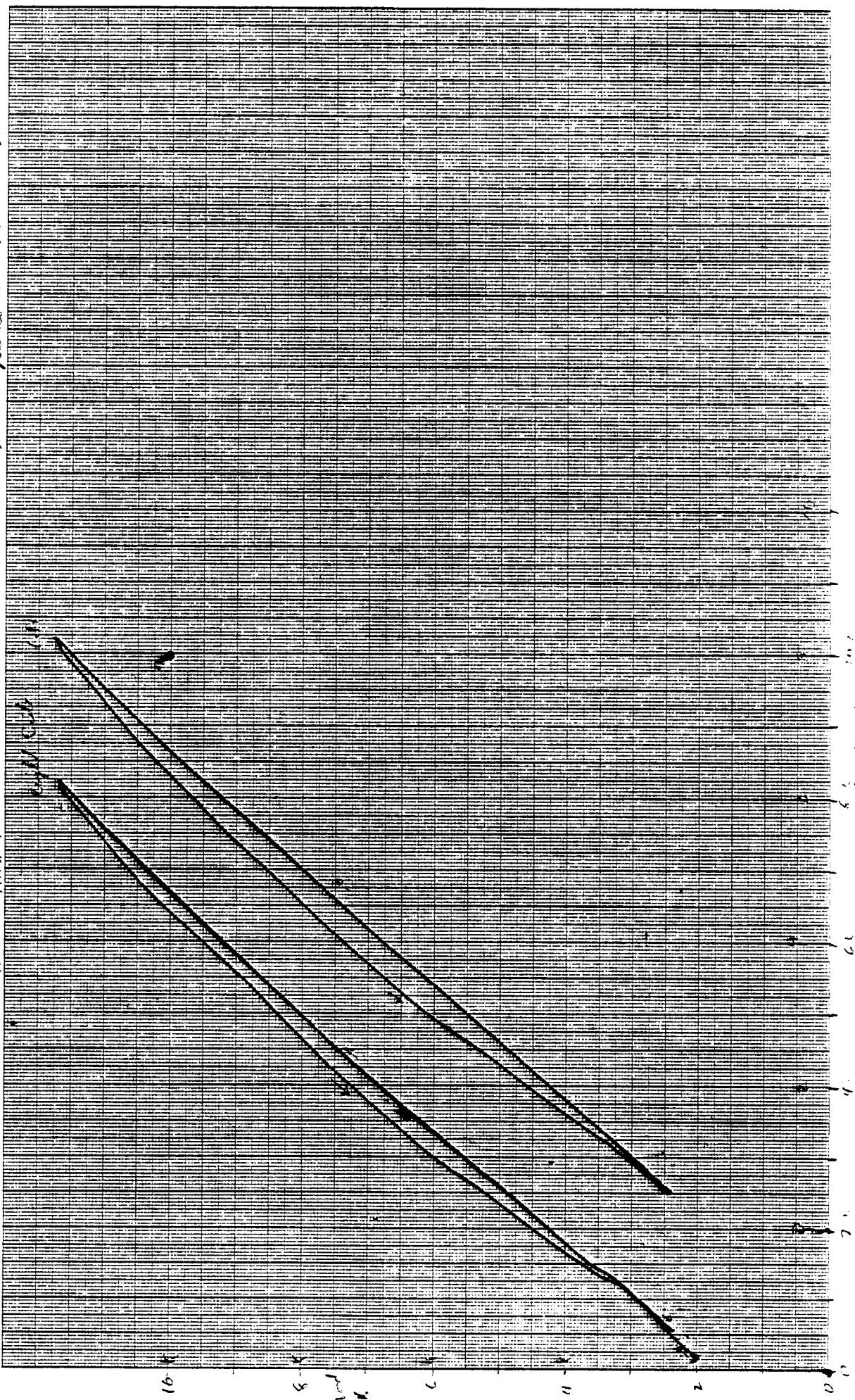
47 1512

For use  
of the  
1973-74

KEPPEL & BUSH CO. MADE IN U.S.A.  
10 X 10 TO THE CENTIMETERS & 10 CM.

Vertical Load Ratio: Field = 2°

Vertical Load Ratio: Plot = 5°



47 1512

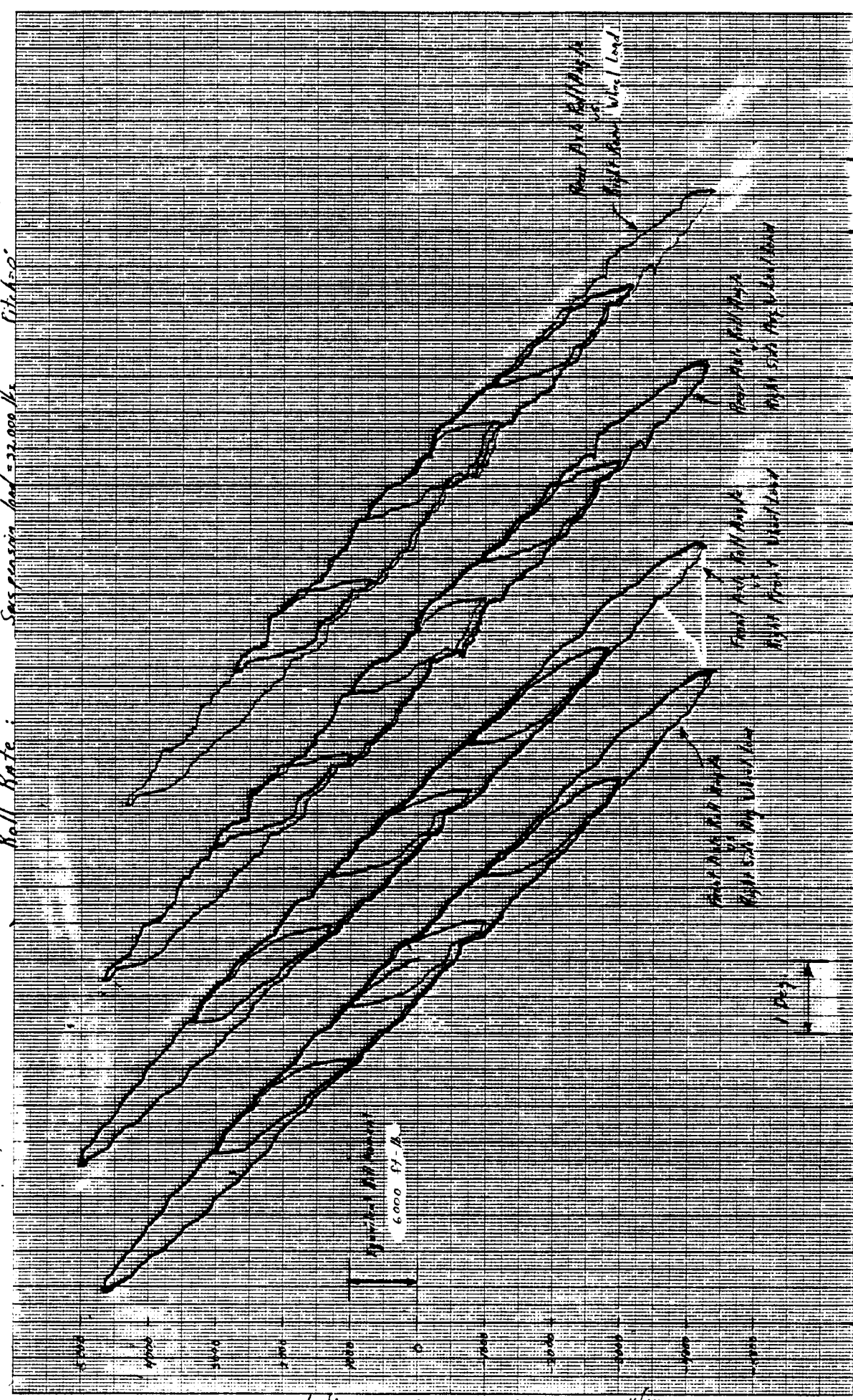
• 11 X 11 TO THE CENTIMETER X 2 CM

Vertical Load Ratio

Vertical Load Ratio



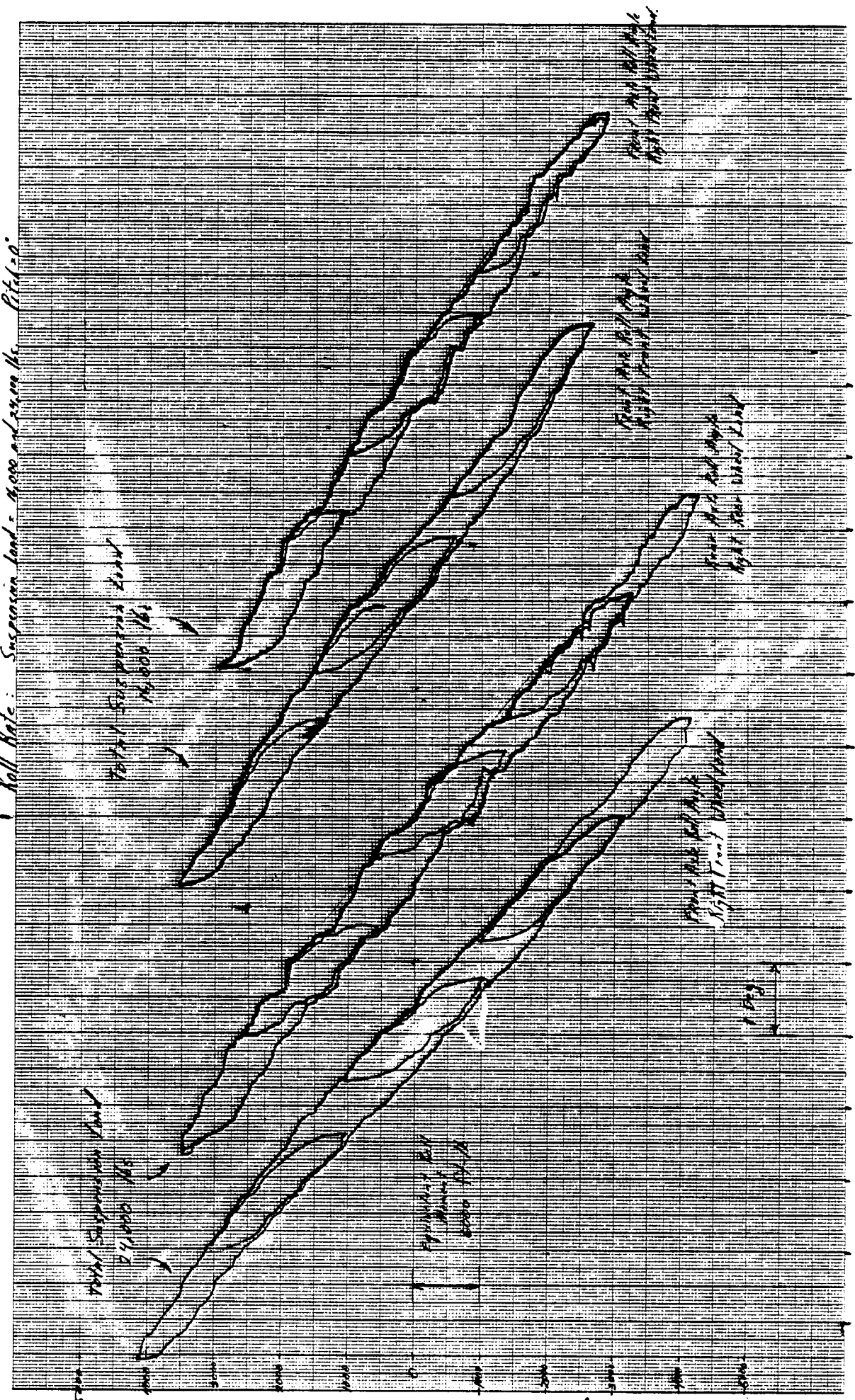
Roll Rate: Suspension load = 22,000 lbs. Pitch = 0°



Right Side Differential Wheel Load, pounds

Roll Angle

Roll Angle: Suspension load - Mass and drum the  $P_{td} = 0$



21,000 lb  
20,000 lb  
19,000 lb

18,000 lb  
17,000 lb  
16,000 lb

15,000 lb  
14,000 lb  
13,000 lb

12,000 lb  
11,000 lb  
10,000 lb

9,000 lb  
8,000 lb  
7,000 lb

6,000 lb  
5,000 lb  
4,000 lb

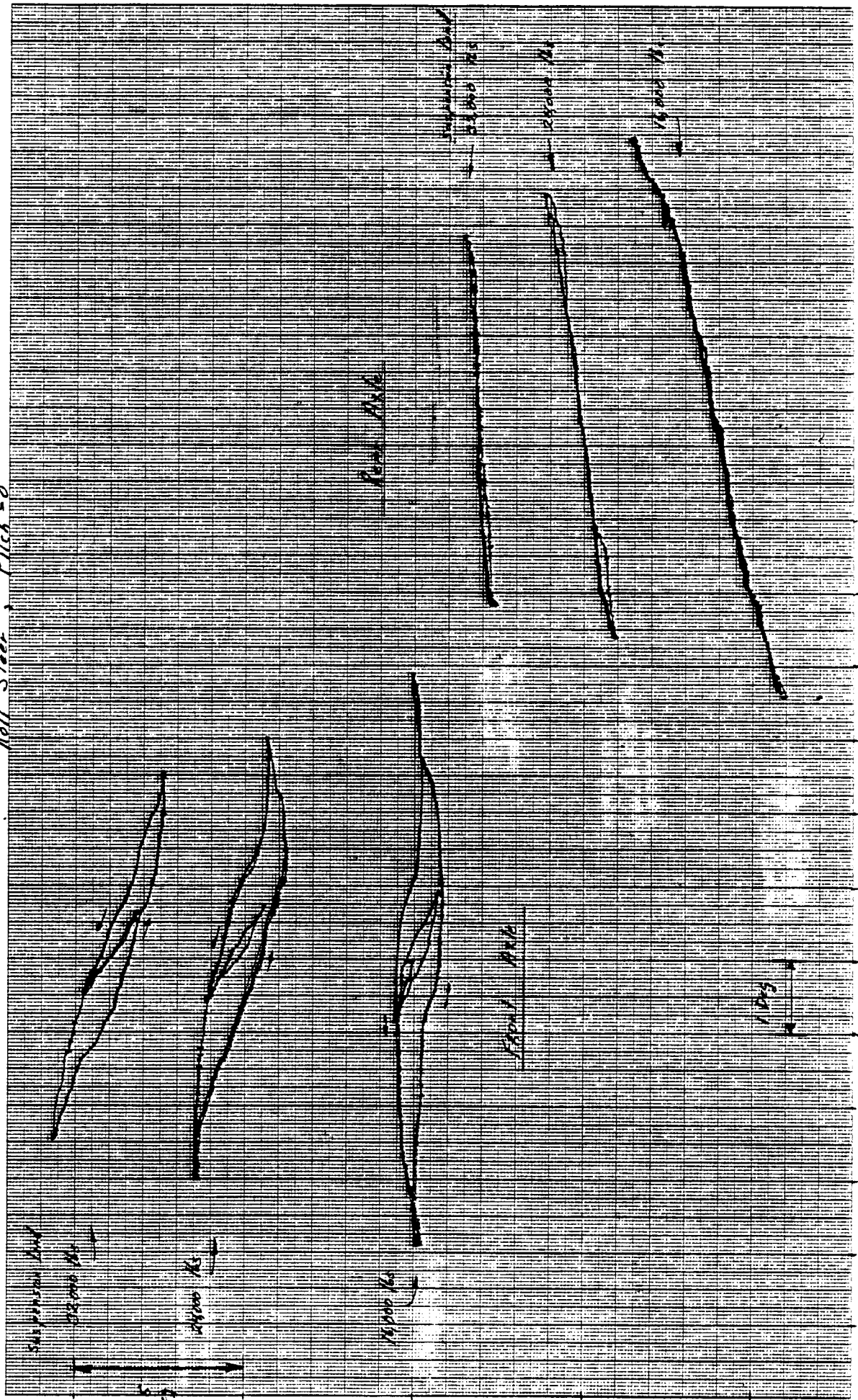
3,000 lb  
2,000 lb  
1,000 lb  
0

Roll Angle

Right Side Differential Wheel Load, pounds

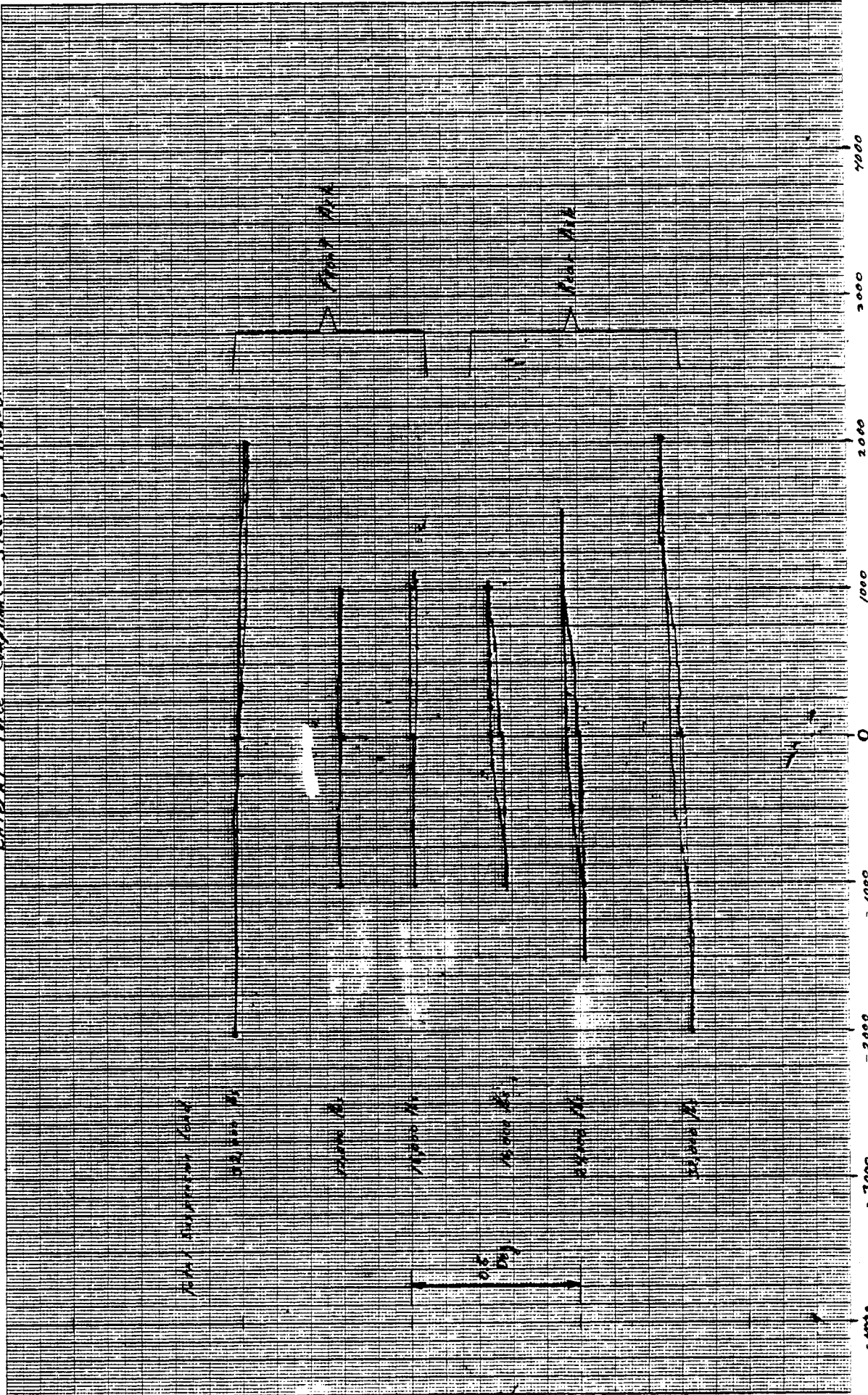


Roll Steer, Pitch = 0°

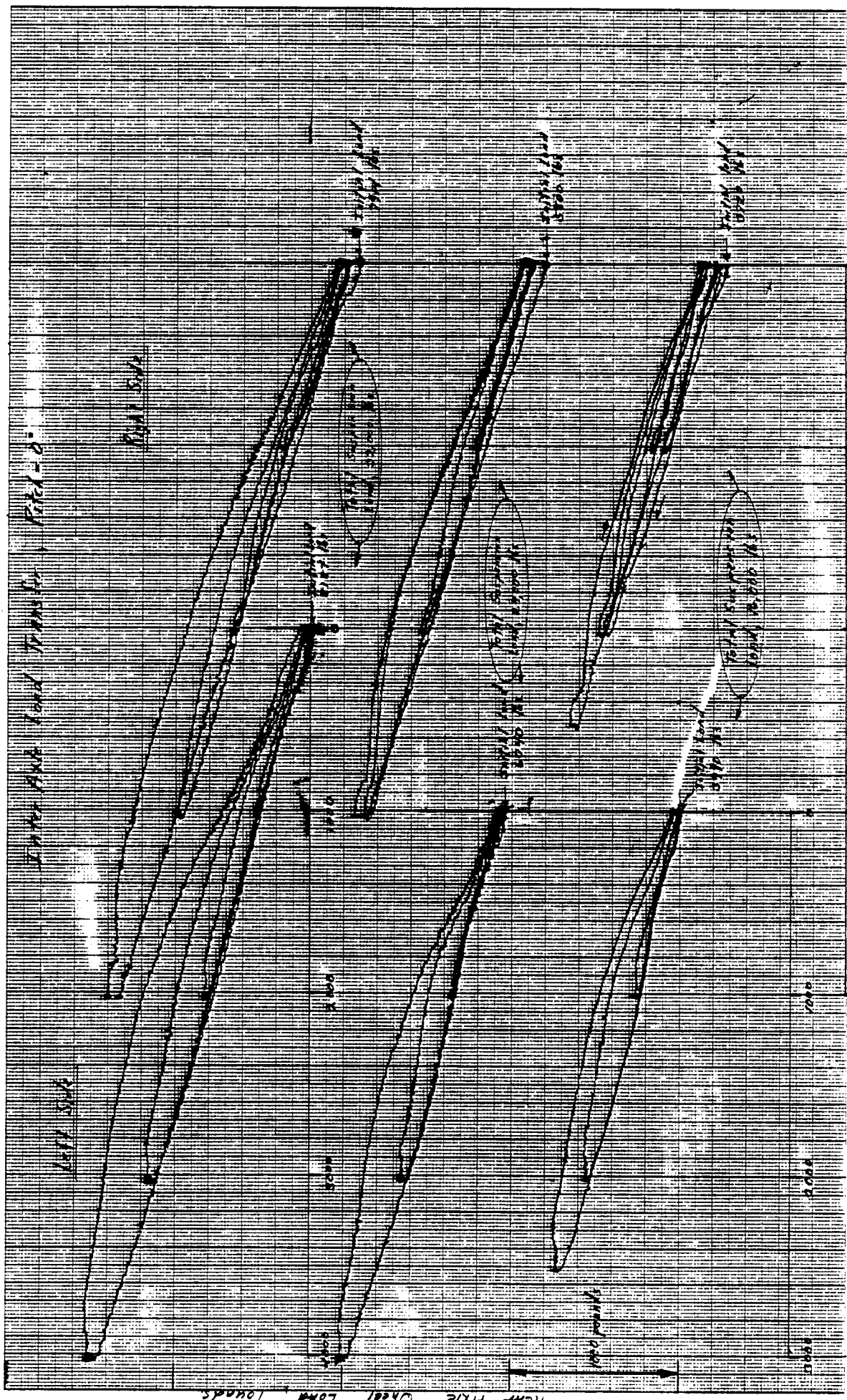


Roll Angle (Right Spring Compressed: →)

Lateral Force Compliance Steer, Ptk-0.

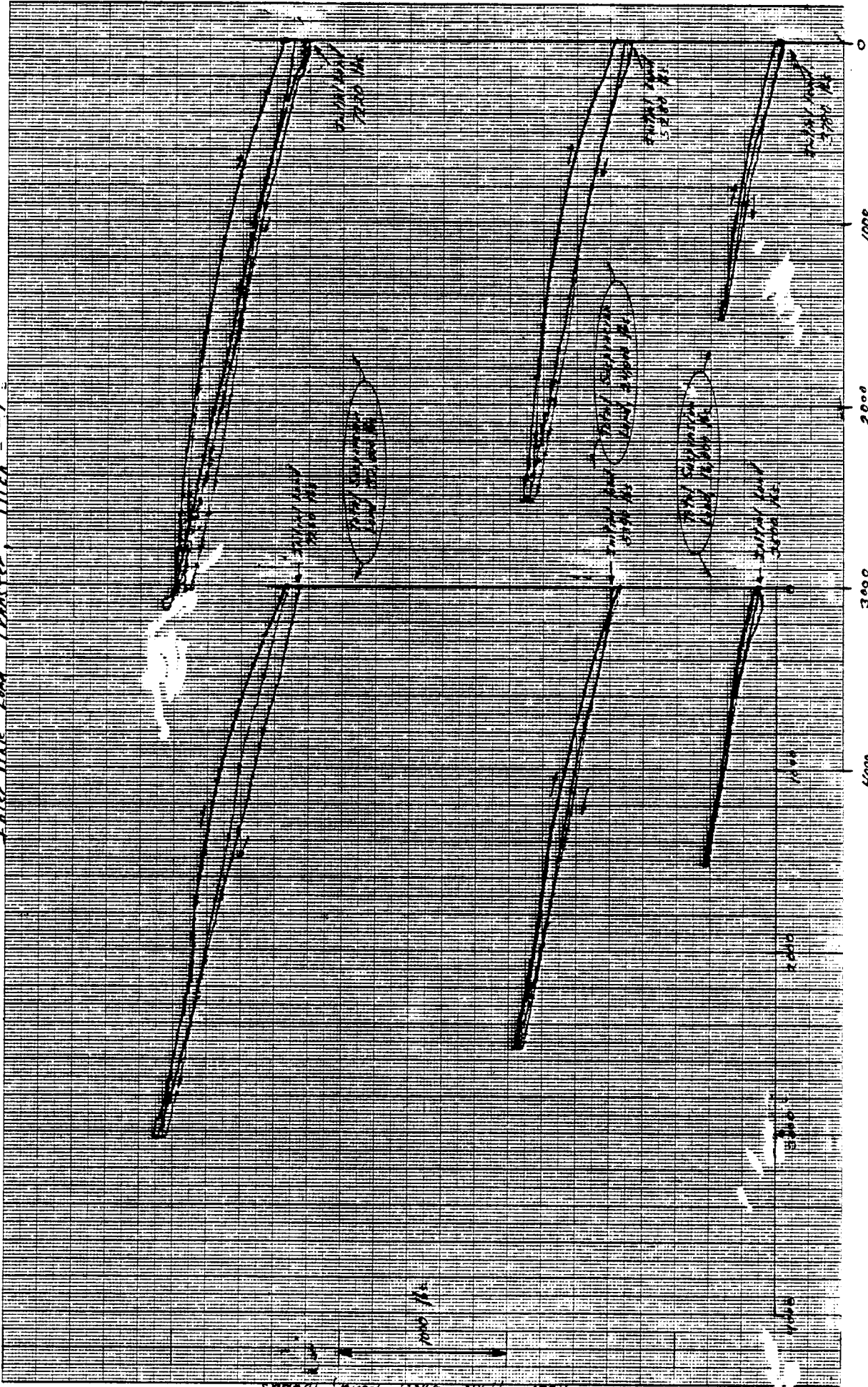


Steer Angle (positive direction is wheels turned toward right)





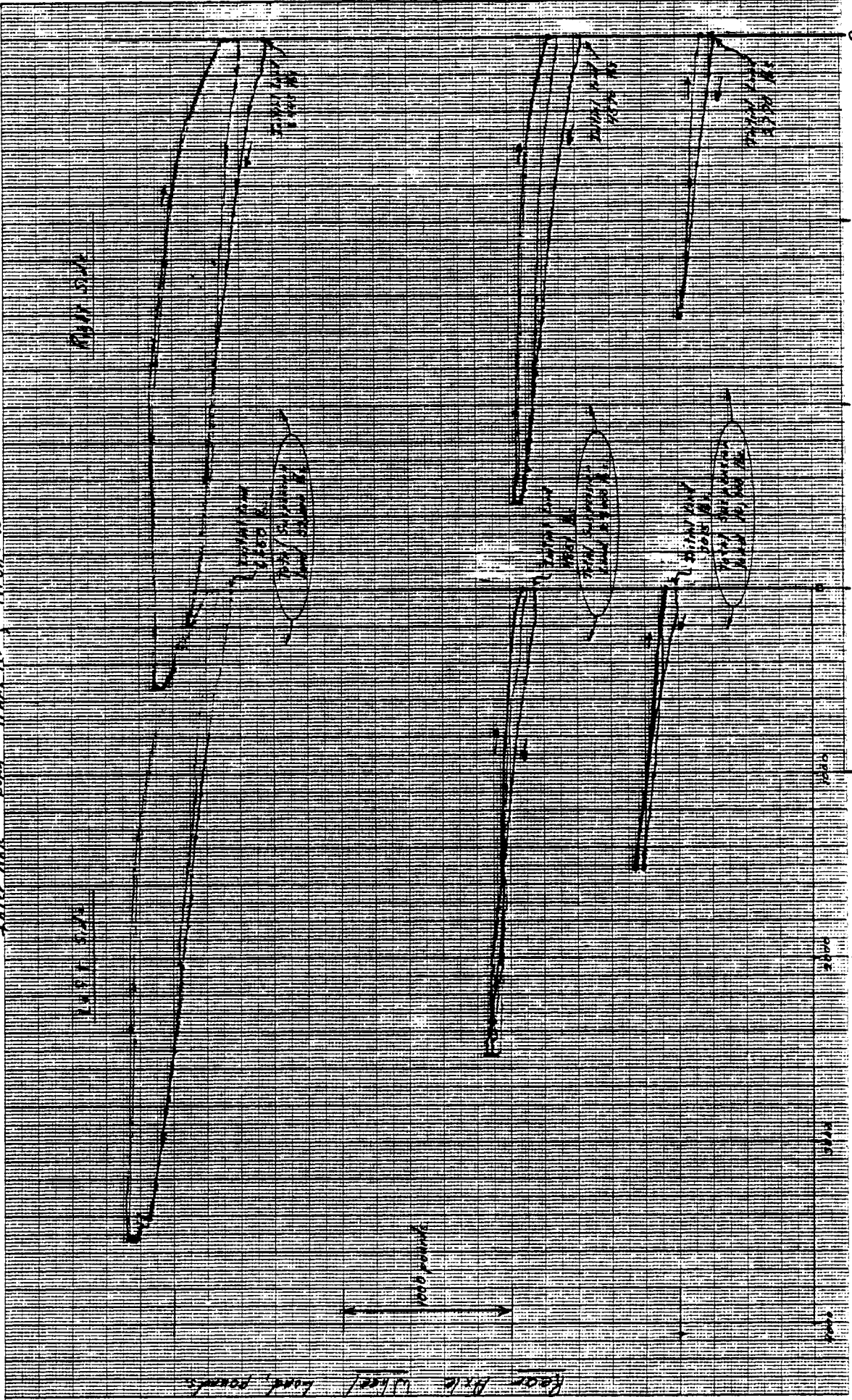
Intr Axle load Transfer, Pited = -1°



Applied Brake Force Per Wheel, Pounds (Applied to all wheels)

Rear Axle Wheel Load, Pounds

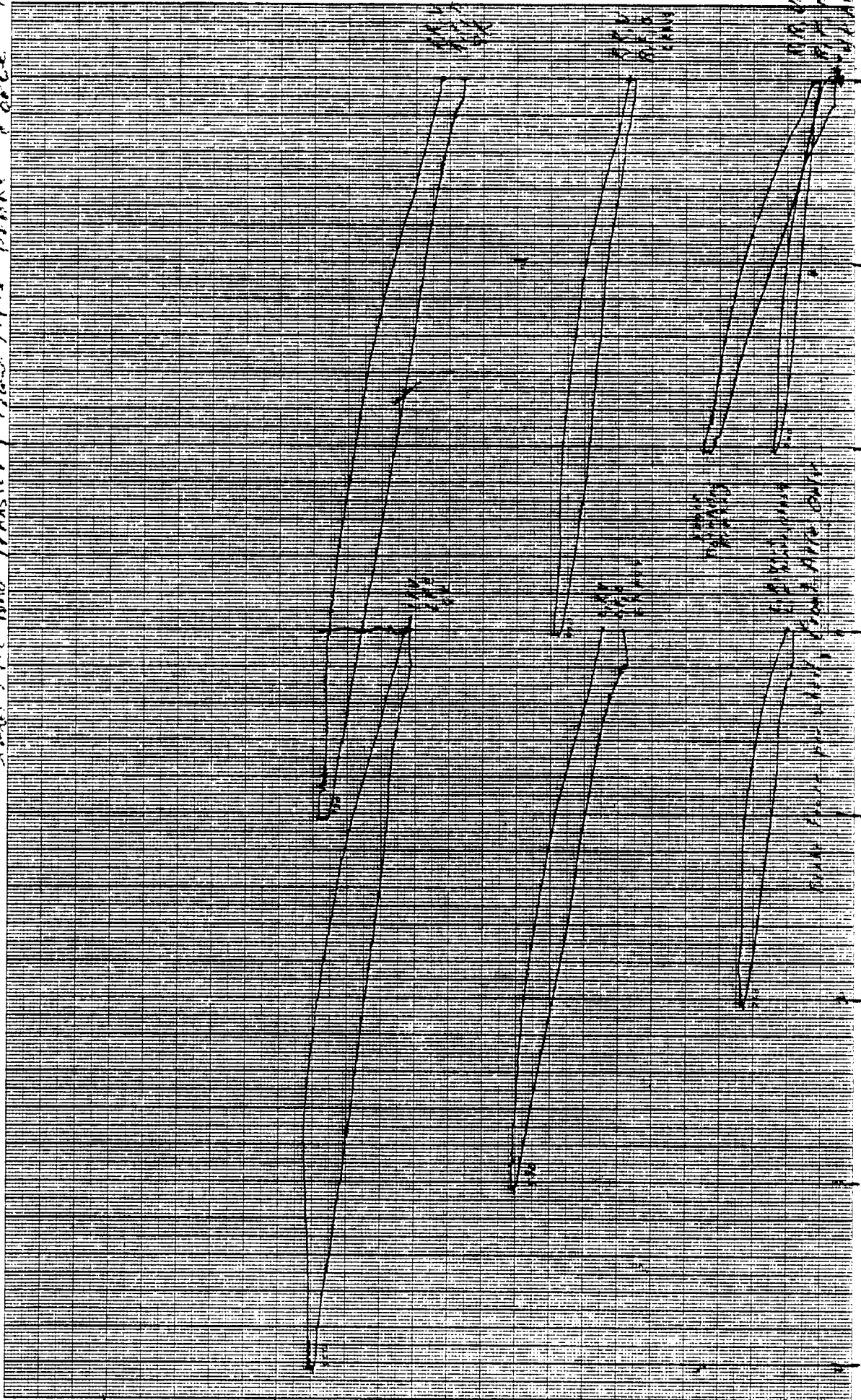
Trailer Axle Load Transfer Pitch=2°



Applied Brake Force per Wheel, pounds. (Applied to all wheels)



Load Myie load Transfer, Small Myie peak Force P=0



47 1512

Unlabeled  
100

KOE 10 X 10 TO THE CENTIMETER 4 X 3 CM  
EQUIP. & REGR. CO. - NORTH B.V.

1K

Myie force

4

3

2

1

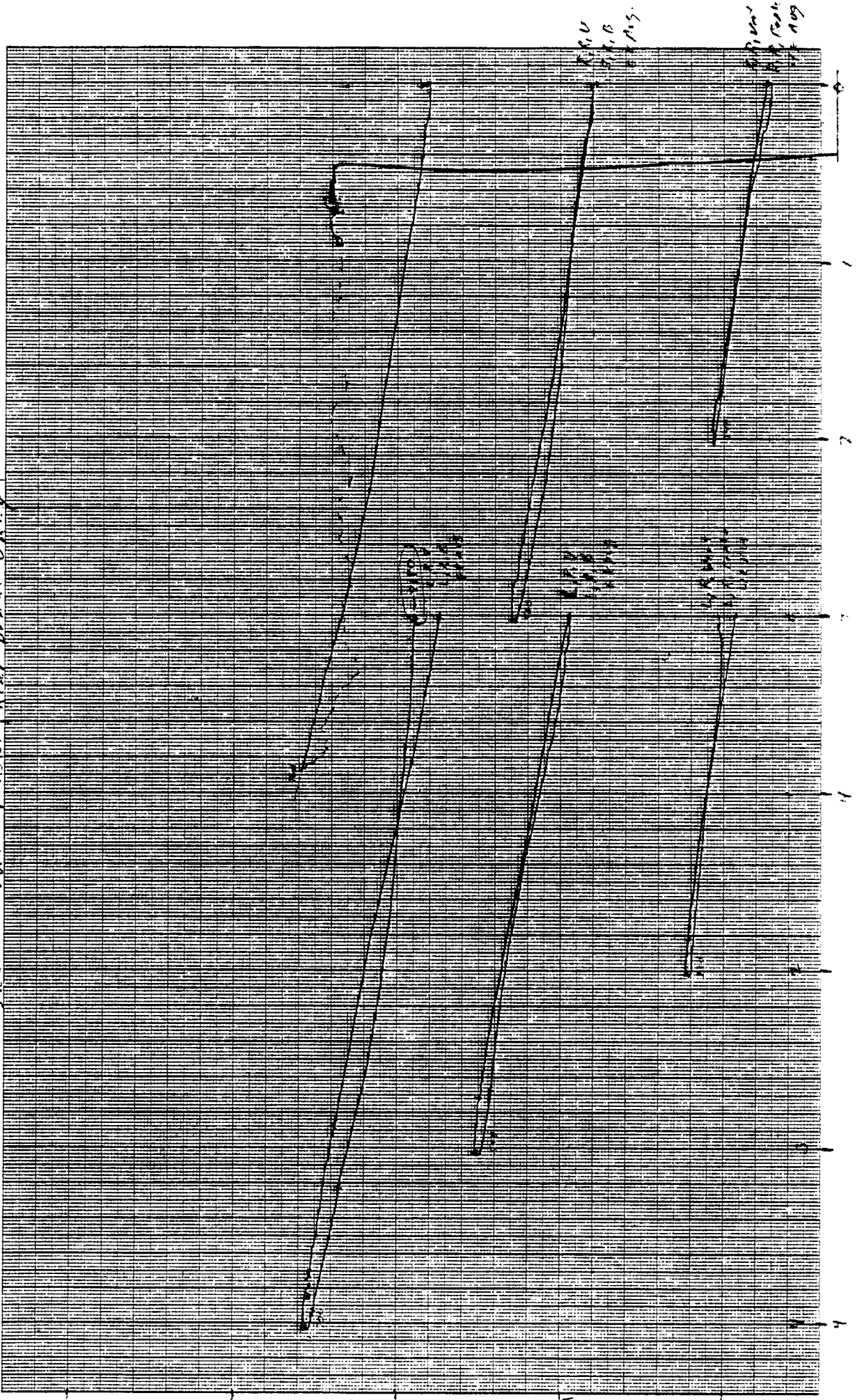
0

Small Myie peak Force P=0

Small Myie peak Force P=0.5

Small Myie peak Force P=0.5

Taker Aisle Load Transfers Rear Brakes Only 80



K10  
K11  
K12

K13  
K14  
K15

Suspension: FREIGHTLINER AIR SUSPENSION

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
Vertical Rate (lb/in)		3500	2400	1700
Coulomb Friction (lb)		625	500	500
Roll RATE	F	105K	85K	80K
(in./lb/deg)	R	45K	85K	85K

Roll CENTER HT F  
 (in below top of frame) R

← NA →

Roll STEER F  
 (DEG / DEG) R

\_\_\_\_\_

\_\_\_\_\_

ALIGNING MOMENT COMPLIANCE STEER F  
 (deg/in lb) R

\_\_\_\_\_  $4 \times 10^{-6}$  \_\_\_\_\_

\_\_\_\_\_  $4 \times 10^{-6}$  \_\_\_\_\_

LATERAL FORCE COMPLIANCE F  
 (deg/lb) R

\_\_\_\_\_  $4 \times 10^{-6}$  \_\_\_\_\_

\_\_\_\_\_  $3 \times 10^{-6}$  \_\_\_\_\_

INTERAXLE LOAD TRANSFER (BY MARK) \_\_\_\_\_  
 (lb/lb) \_\_\_\_\_

\_\_\_\_\_ nil \_\_\_\_\_

Average Vertical Wheel Load, pounds relative to nominal wheel load

24  
21  
0  
21  
24

Load set, 1000 lbs  
to wheel level  
Normal Air Spring Press. 20

24  
21  
0  
21  
24

24  
21  
0  
21  
24

Frigit Inc Air Suspension  
Vertical Deflection  
Ritch-o



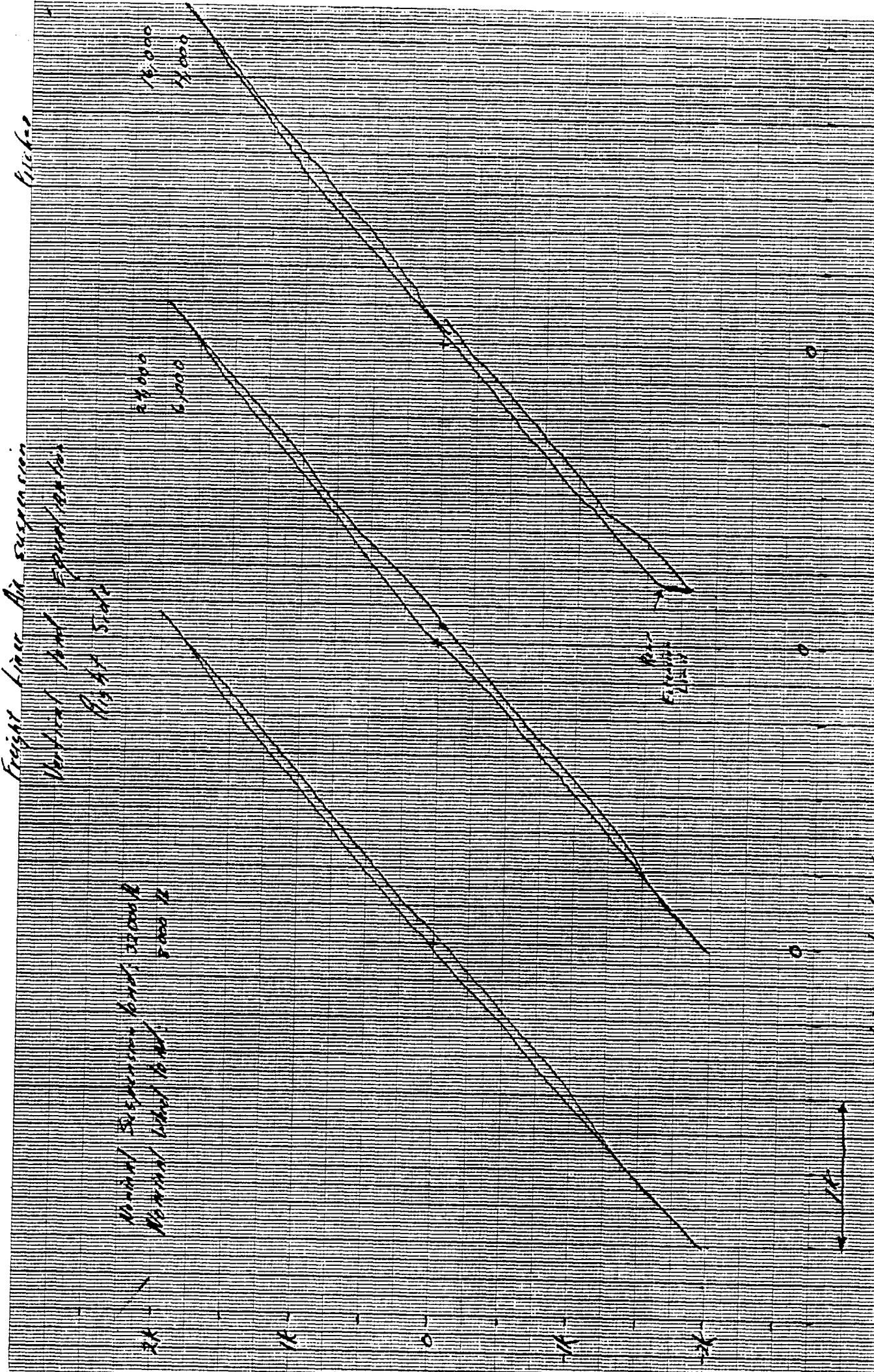
Vertical Deflection, inches relative

Trailing Left Side  
Frigit Inc  
Air Spring

1.25



Vertical wheel load, leading axle, pounds relative to nominal wheel load

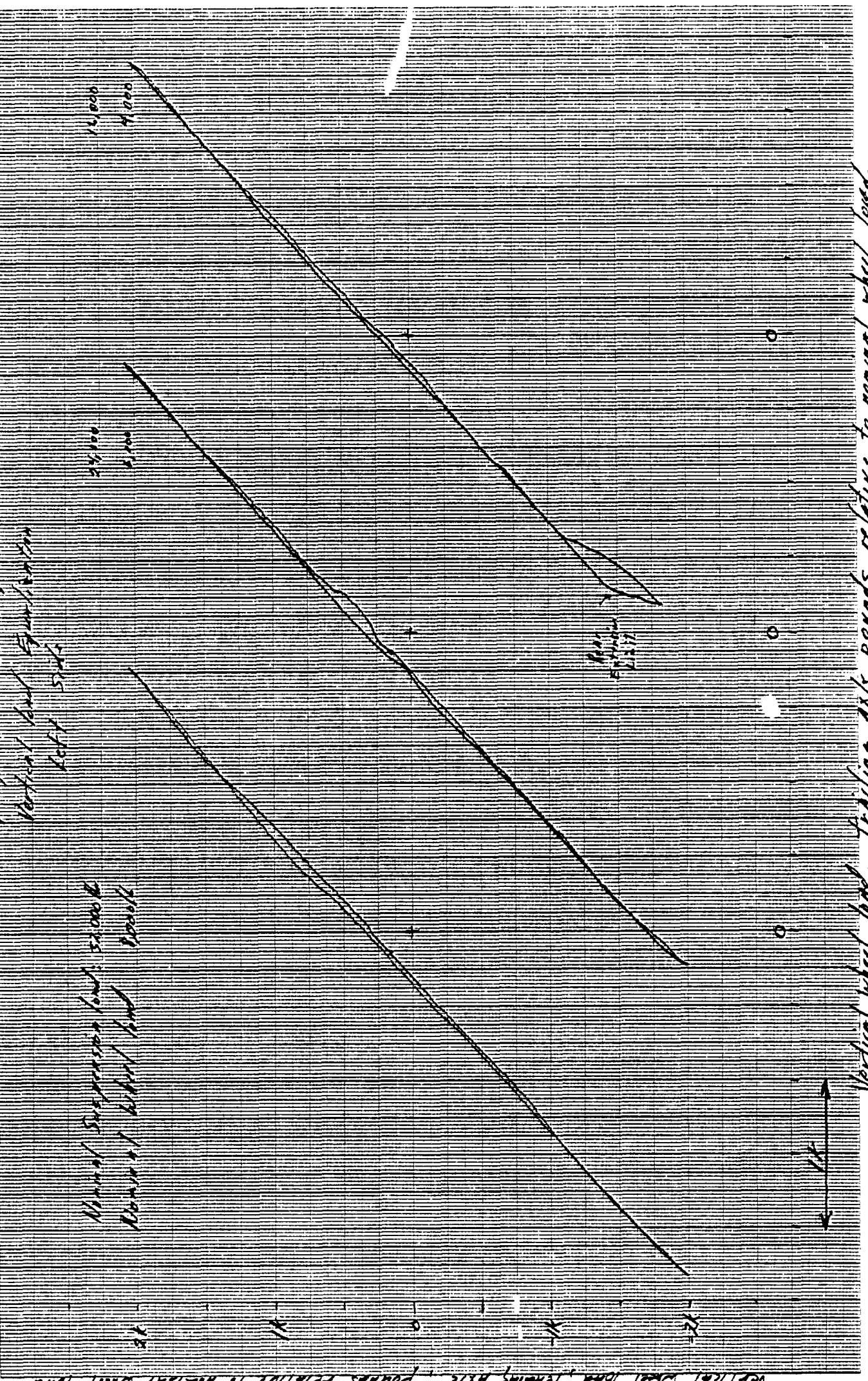


Vertical wheel load and trailing axle, pounds relative to nominal wheel load



Pitch=2

Flightline Air Suspension



Vertical wheel load, leading axle, pounds relative to nominal wheel load

Vertical wheel load, trailing axle, pounds relative to nominal wheel load

Average wheel load, pounds relative to nominal wheel load

24  
21  
0  
21  
24

Nominal Suspension load 22,000  
Nominal Wheel load 8,000  
Applied Air Sides Pressure 70

24,000  
6,000  
30

16,000  
4,000  
35

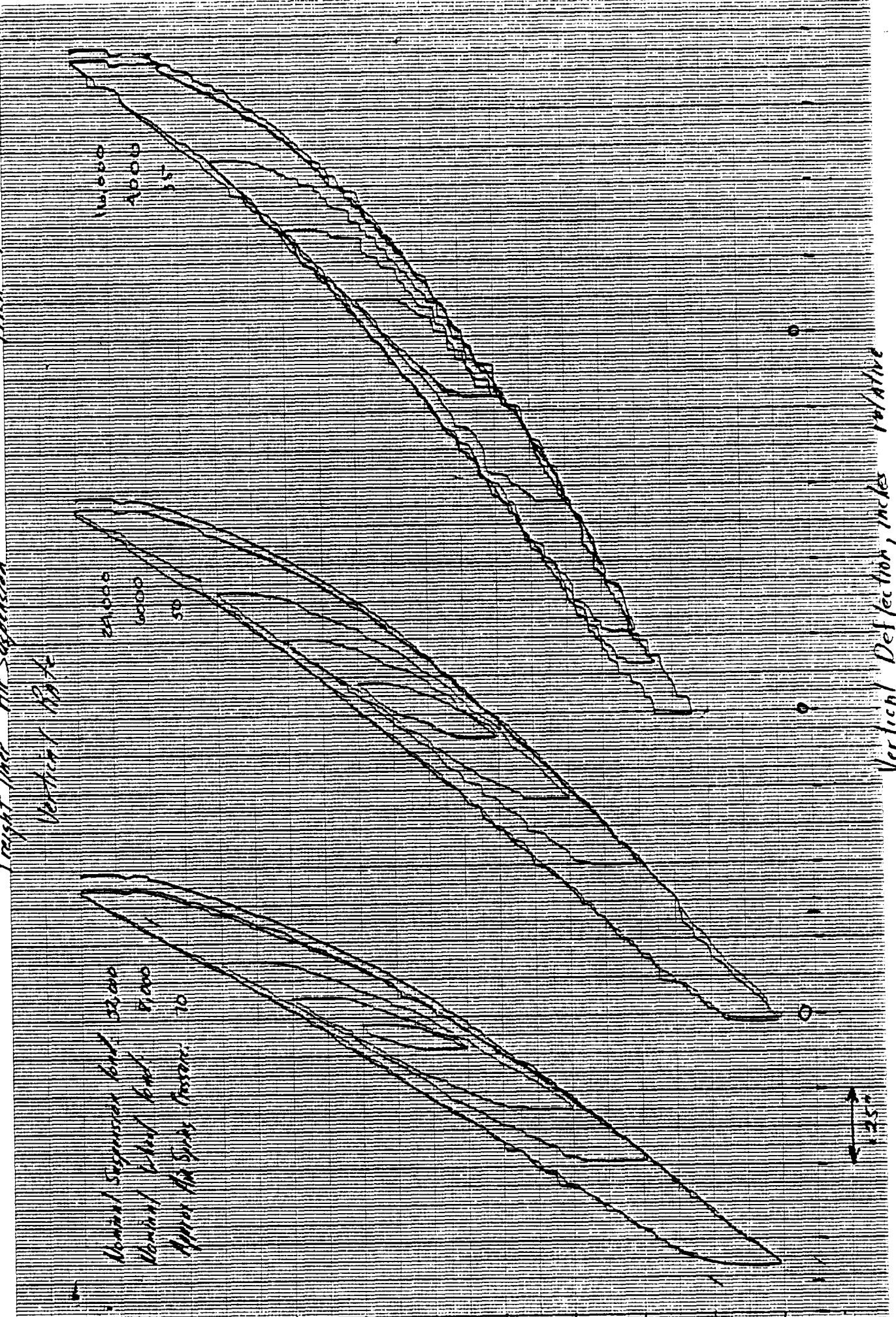
Pitch = 1°

Freight liner Air Suspension

Vertical Rate

1.25"

Vertical Deflection, inches relative



Pitch = -1°

Freight Line Air Suspension

Load Equalization - Left Side

Minimum Suspension Load  
Nominal Wheel Load

16,000  
4,000

24,000  
6,000

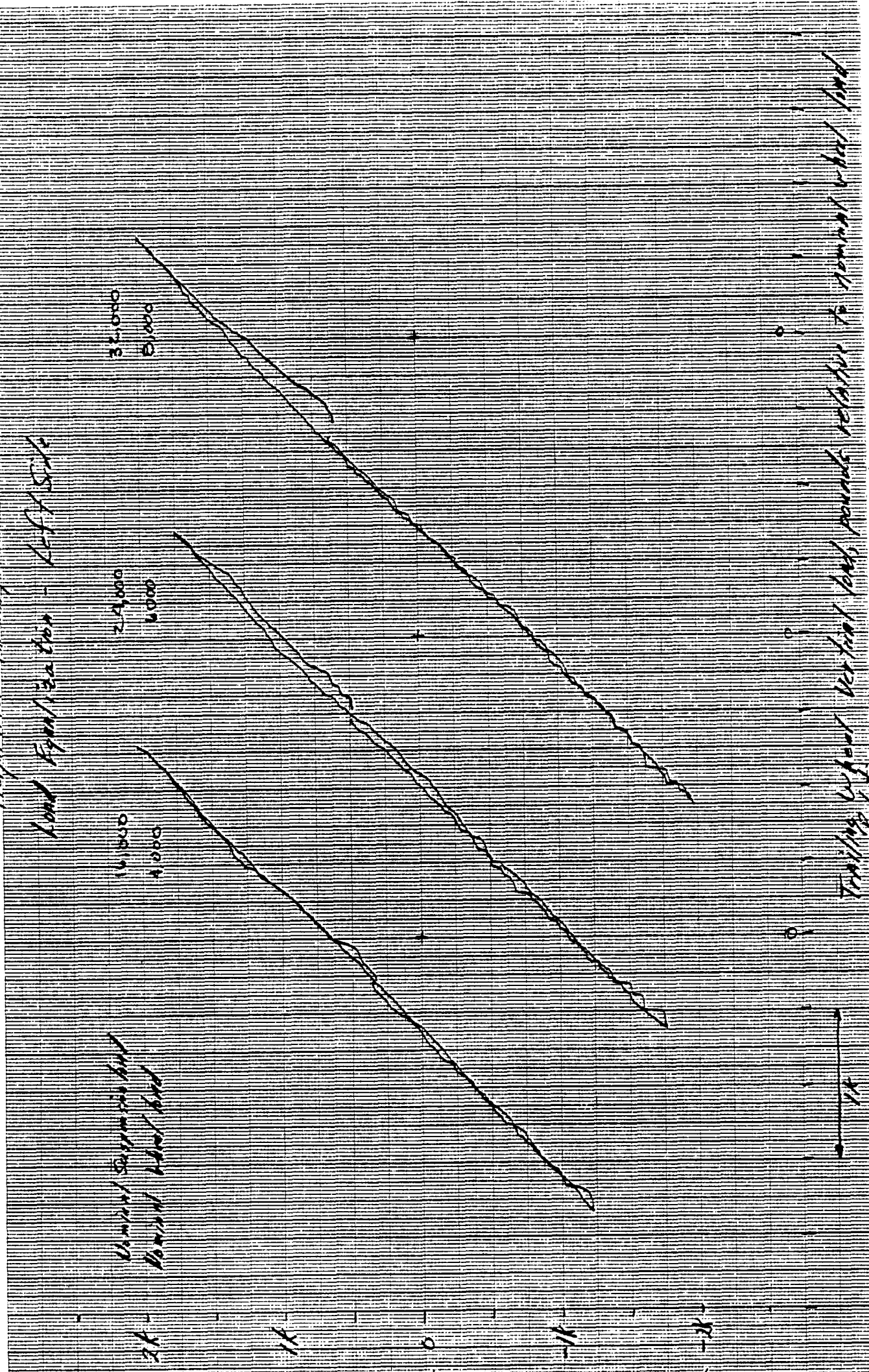
32,000  
8,000

Leading Wheel Vertical Load, pounds relative to nominal wheel load

Trailing Wheel Vertical Load, pounds relative to nominal wheel load

1X

1X



Pt. A - 1°

Freight line - No Suspension

Load equalization - Right side

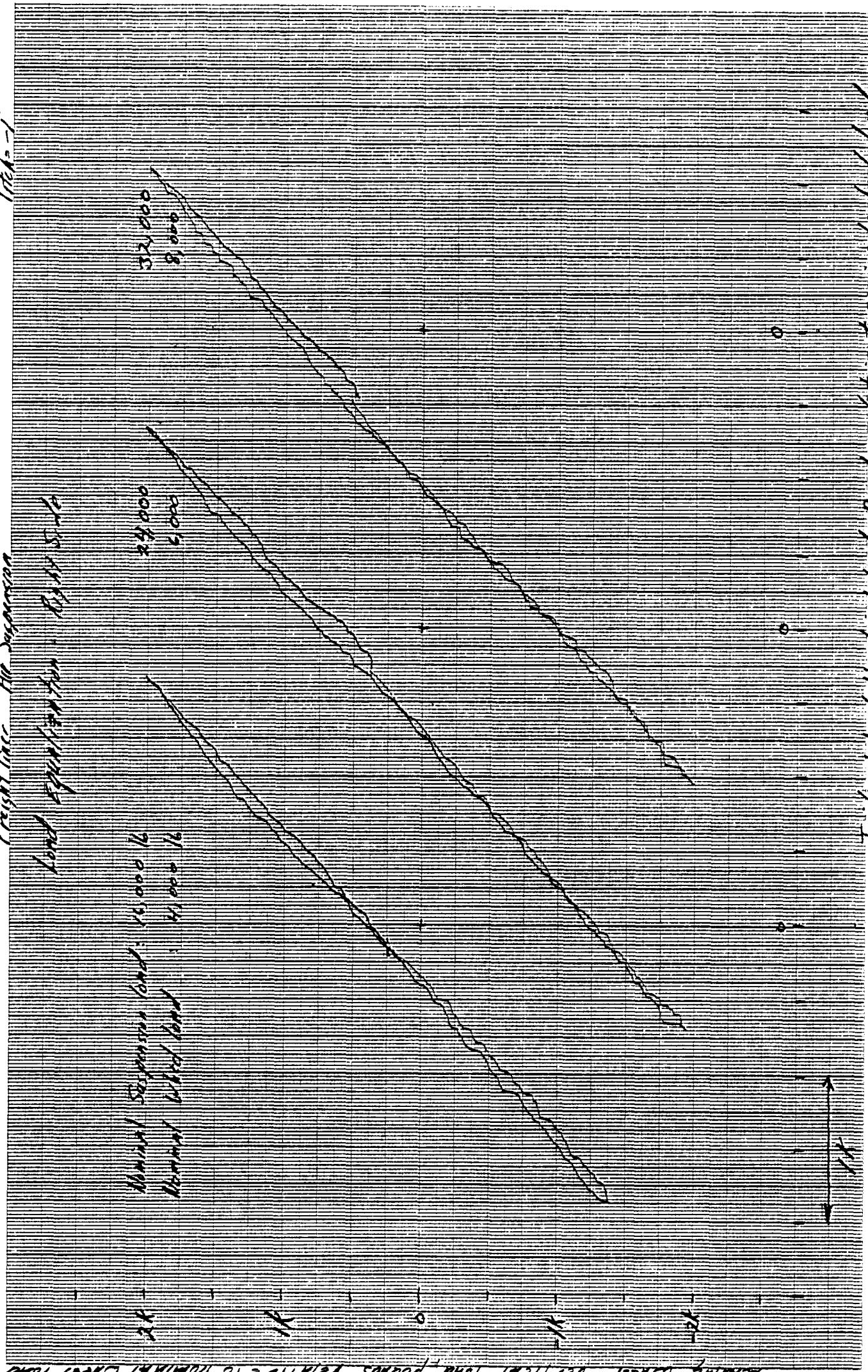
Minimal Suspension load: 15,000 lb  
Minimal Wheel load: 4,000 lb

32,000  
8,000

24,000  
6,000

Leading wheel vertical load, pounds relative to nominal wheel load

Trailing wheel vertical load, pounds relative to nominal wheel load





Freight line Air Suspension

Roll Rate at 30,000 ft suspension level

Roll Moment on Axle m-lb → Right Side Compressed

1000

1000

0

1000

1000

Landing Rate

Trailing Rate

ISOLATED

CONNECTED

ISOLATED

CONNECTED



Axle Roll Angle, deg



PITCH=0

FREIGHTLINER AIR SUSPENSION

ROLL RATE @ 24,000 IN SUSPENSION LOAD

LEADING AXLE

TRAILING AXLES

ROLL MOMENT ON AXLES (in-lb) (← RIGHT SIDE COMPRESSOR)

1000

2000

0

1000

2000

LEADING

TRAILING

TRAILING

TRAILING

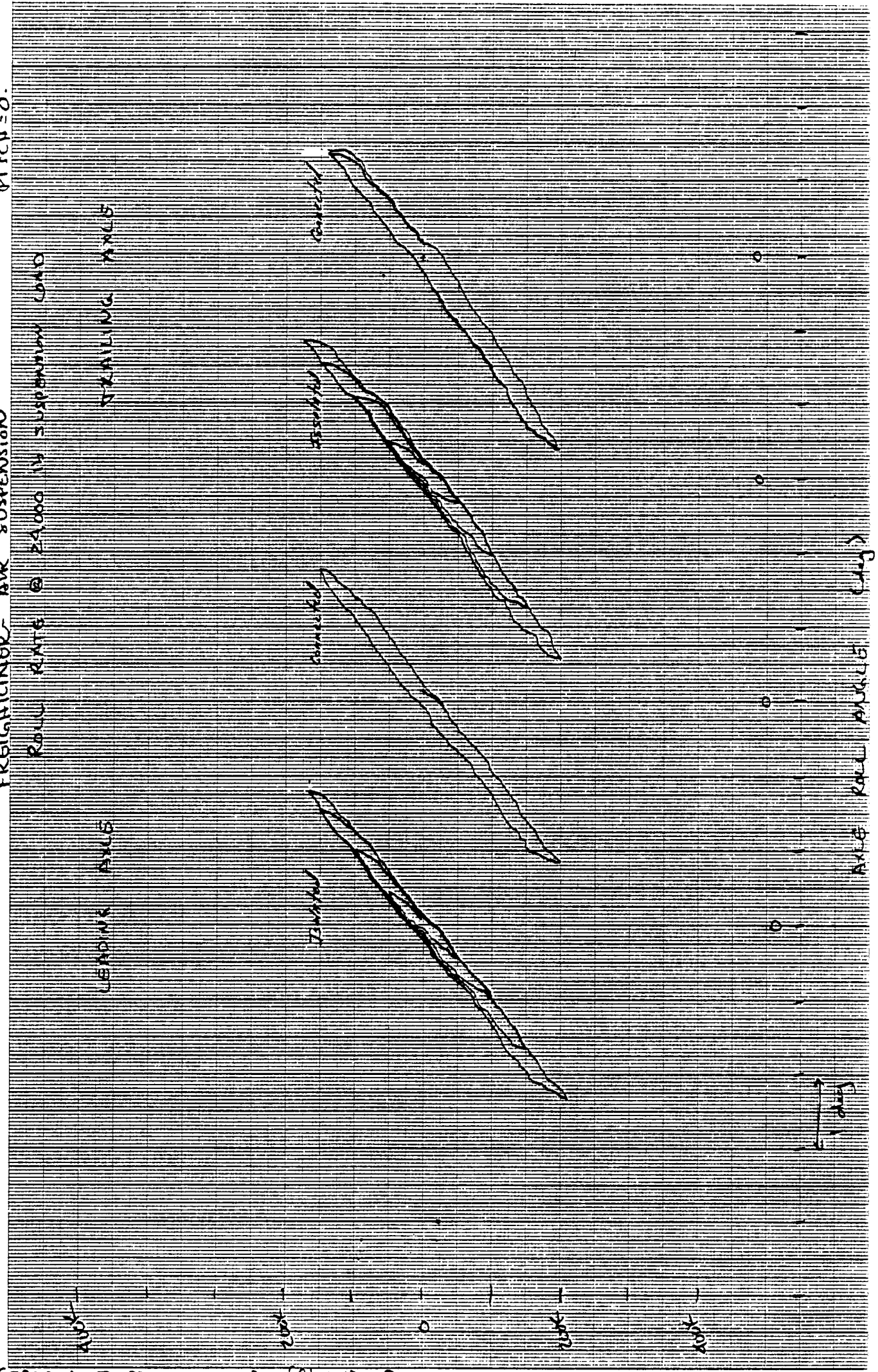
AXLE ROLL ANGLE (deg)

0

0

0

0

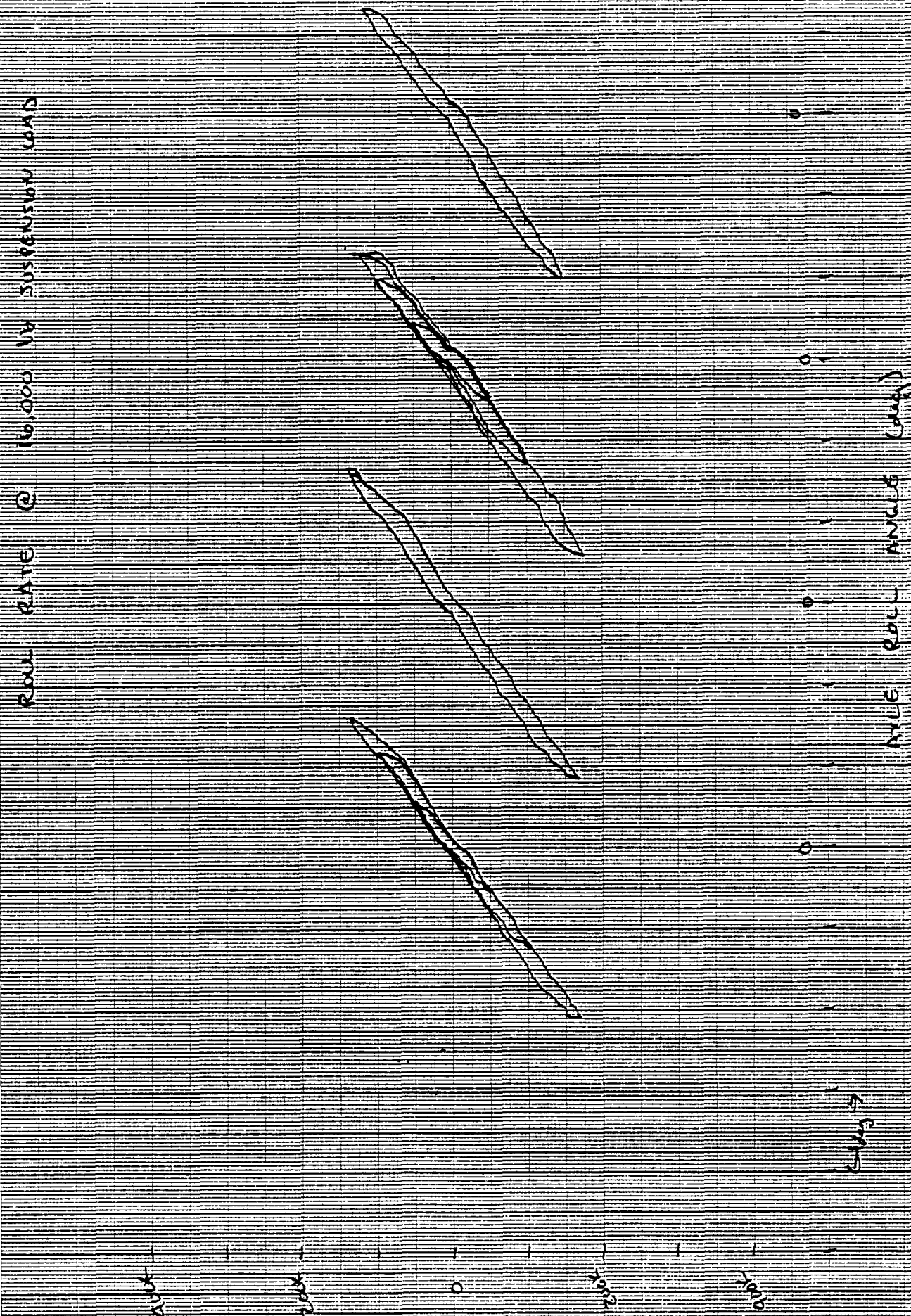


WITCH = 0.

FREIGHTNER - AIR SUSPENSION

ROLL RATE @ 16,000 LB SUSPENSION LOAD

ROLL MOMENT ON AXLE (IN-LB) (← RIGHT SIDE COMPRESSED)



Roll Rate

AXLE ROLL ANGLE (deg)

Freight liner Air Suspension

Roll Street

Mounted at 22K  
22K and 10K  
per 1/2 control  
springs reacting from controlled  
ride height

Pitch=0

Leaning Axle



Twisting Axle



Axle Steer angle, deg (steered toward right →)

5  
4  
3  
2  
1  
0  
1  
2  
3  
4  
5

Axle Roll Angle deg (right side compressed)

1 DEG

FREIGHTLINER - AIR SUSPENSION

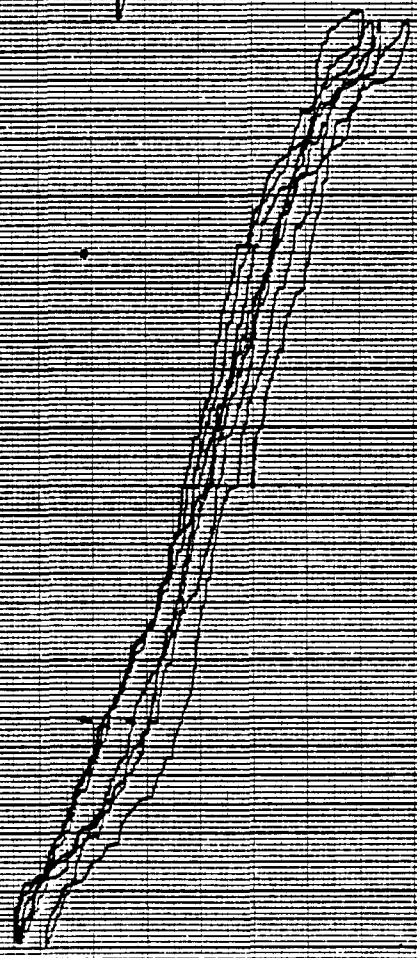
1/14/70

ALIGNING MOMENT COMPENSATED STEER

Measured at 32,000 lbs  
At 16,000 Same behavior  
At 24,000 lbs to constant  
steering geometry from  
connected with trucks

Trailing Axis

Leading Axis



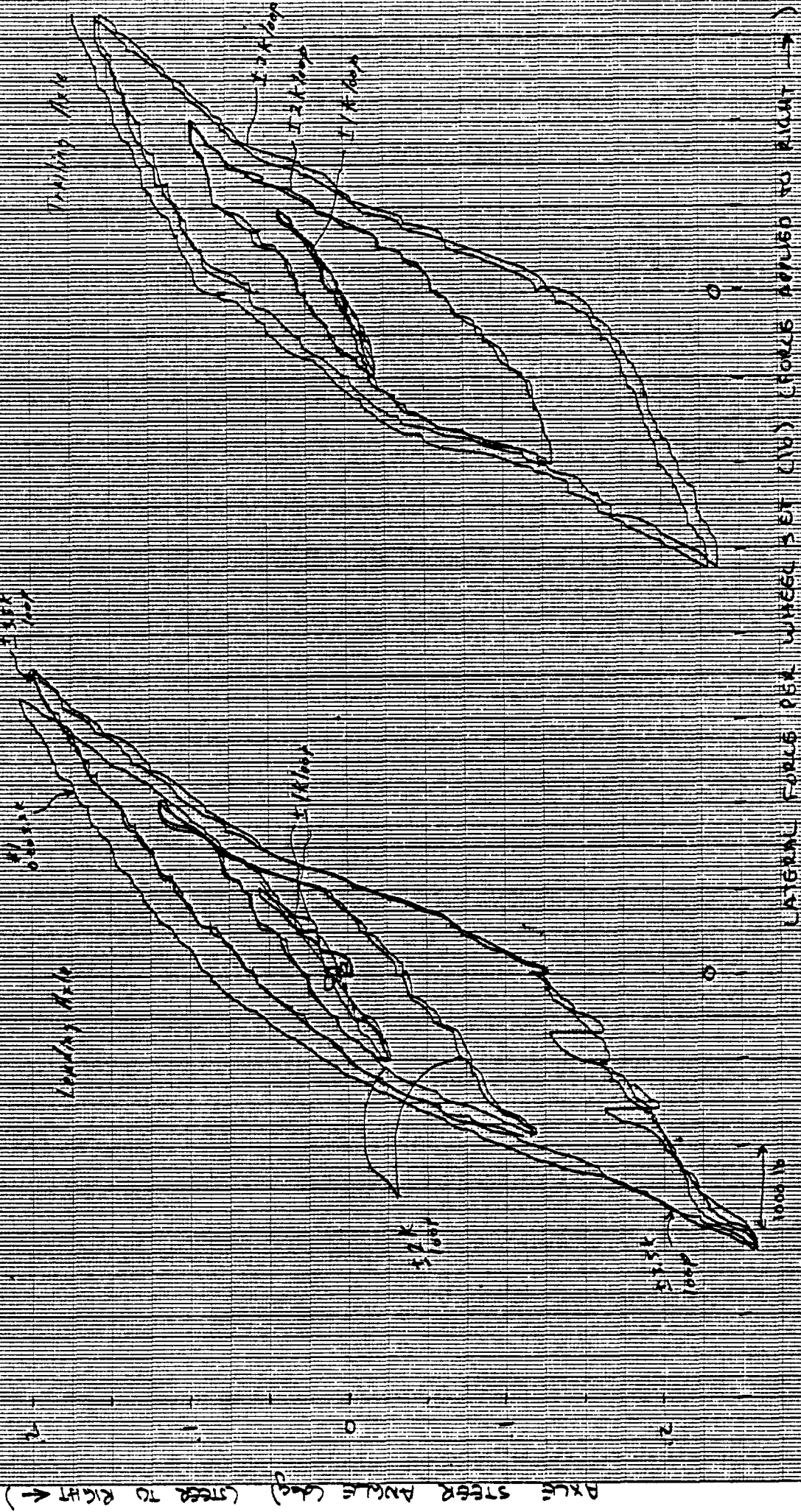
WHEEL SETS

ALIGNING MOMENT PER WHEEL SET (IN LB)  
CHANGED TO ALL WHEEL SETS SIMULTANEOUSLY



LATERAL FORCE COMPLIANCE STEER  
 FREIGHTLINER - AIR SUSPENSION

33,000 LBS. TEST LOAD



AXLE STEER ANGLE (deg) (STEER TO RIGHT ←)

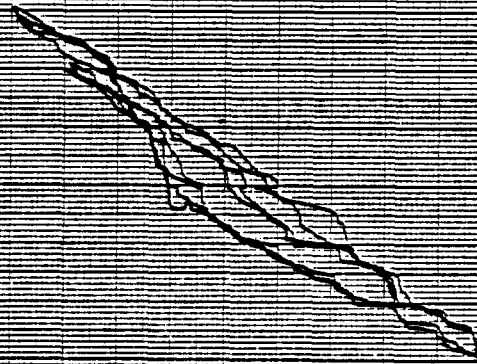
LATERAL FORCE PER WHEEL SET (LB) (FORCE APPLIED TO RIGHT →)  
 (APPLIED TO ALL WHEEL SETS SIMULTANEOUSLY)



FREIGHTLNER - AIR SUSPENSION

UNTIL FORCE COMPLIANCE STEER  
10,000 LB SUSPENSION LOAD

AXLE STEER ANGLE (deg) (← STEER TO RIGHT)



1000 LB

LATERAL FORCE PER WHEEL SET (LB) (FORCES APPLIED TO RIGHT →)  
(APPLIED TO ALL WHEEL SETS SIMULTANEOUSLY)

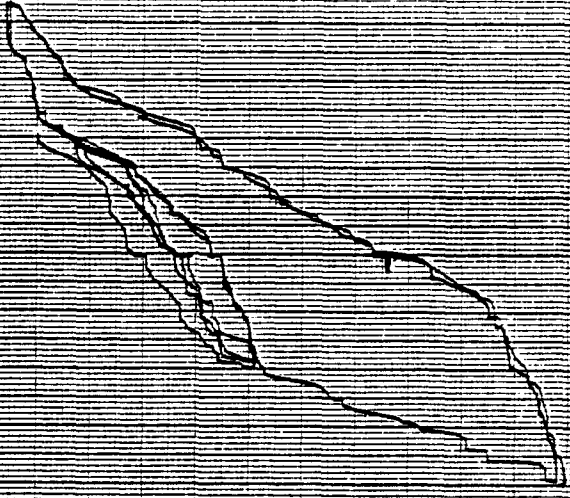
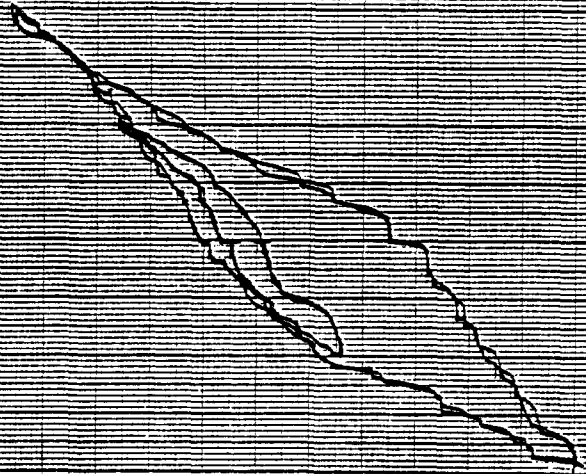
FRIGHTLINER - AIR SUSPENSION

LATERAL FORCE COMPLIANCE STEER  
GROSS SUSPENSION LOAD 24,000 LB

AXLE STEER ANGLE (deg) (← STEER TO RIGHT)

Leading Edge

Trailing Edge

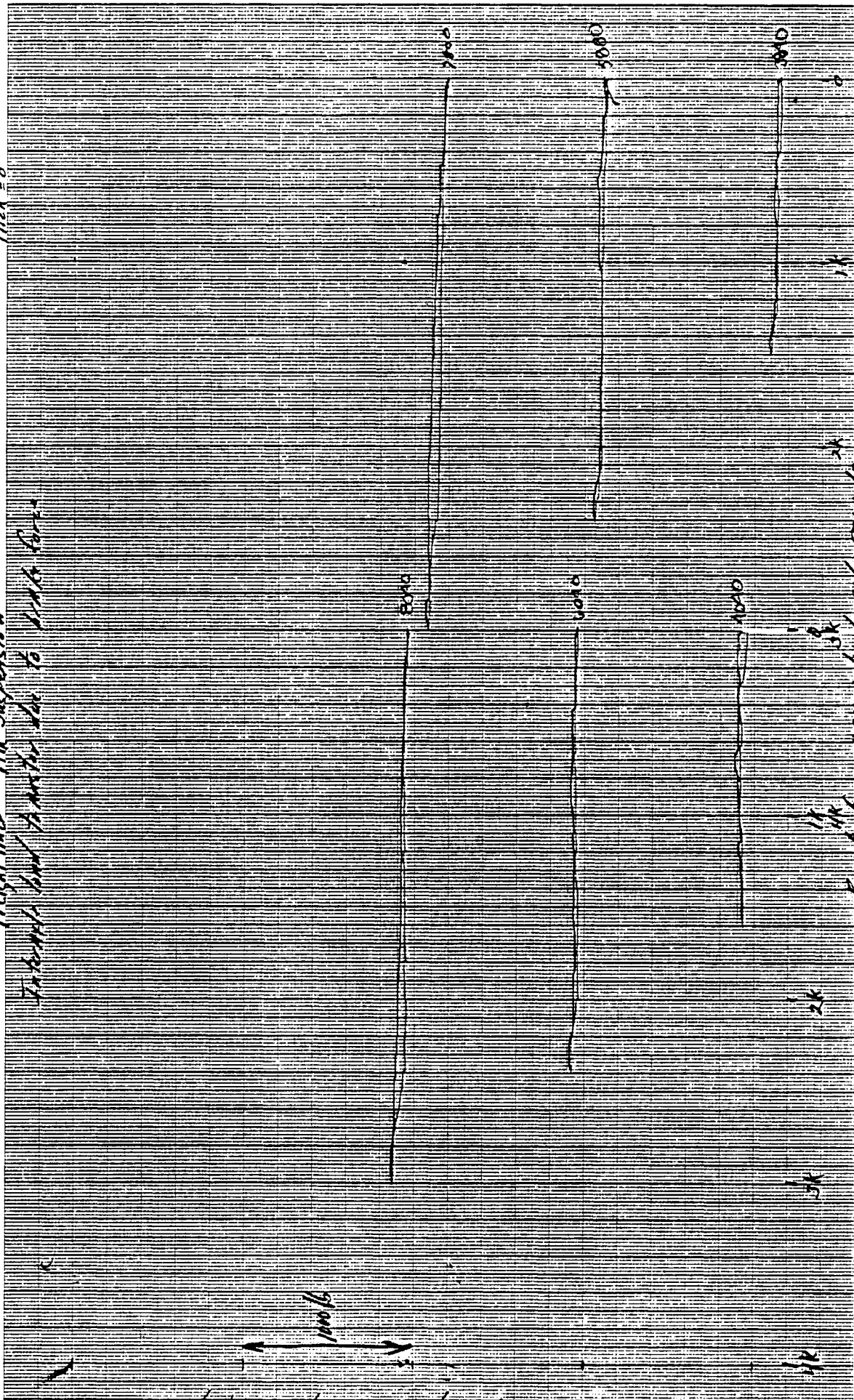


1000 LB

LATERAL FORCE PER WHEEL SET (16) (FIXED TO RIGHT →)  
(APPLIED TO ALL WHEEL SETS SIMULTANEOUSLY)

Load Transfer from Landing to Trolley wheel, pounds

1000



1000  
1000  
1000

1000  
1000  
1000

1000  
1000  
1000

Brake force per wheel set, pounds  
Applied to all four simultaneously

Fright line Air Suspension  
Internal load transfer due to wheel deflection  
Pth = 0

Axle longitudinal displacement, inches

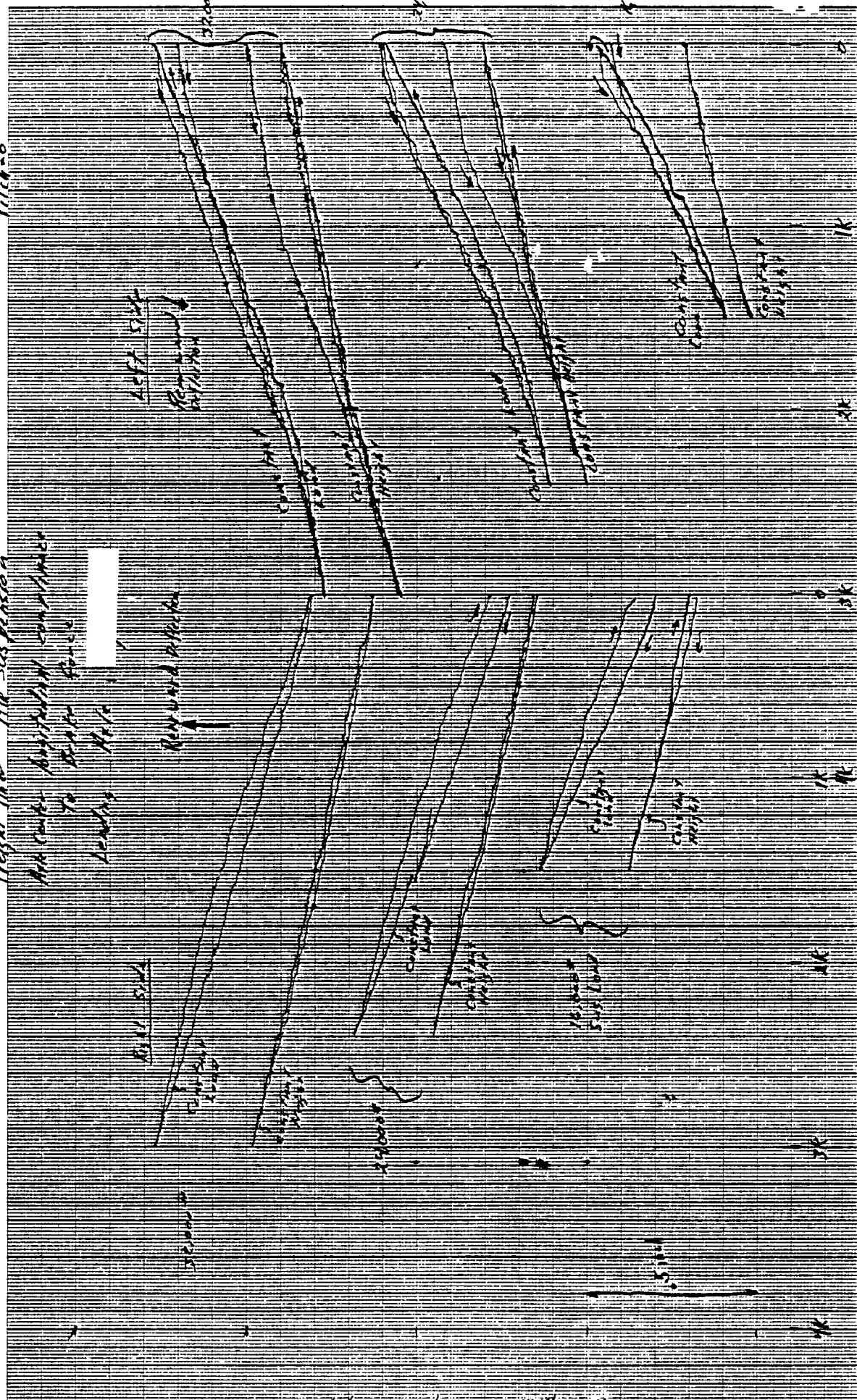
Freight car All Suspension

All center distributed constant  
to front force

Leading axle

Rebound motion

Pit 1.0



Bank force per wheel set, pounds



11/14/00

# FREIGHTLINER - AIR SUSPENSION

## AXLE CENTER LONGITUDINAL COMPLIANCE TO BRAKE FORCE

### TRAILING AXLE

BRAKE

RESTRICTION TO FORWARD

STRAIN LIMIT

32000

CONSTRAINT

24000

CONSTRAINT

400

16000

3K

2K

1K

0

2K

1K

0

RESTRICTION TO FORWARD

STRAIN LIMIT

32000

CONSTRAINT

24000

CONSTRAINT

CONSTRAINT

CONSTRAINT

CONSTRAINT

AXLE LONGITUDINAL DISPLACEMENT (IN)

BRAKE FORCE PER WHEEL SET (LB)

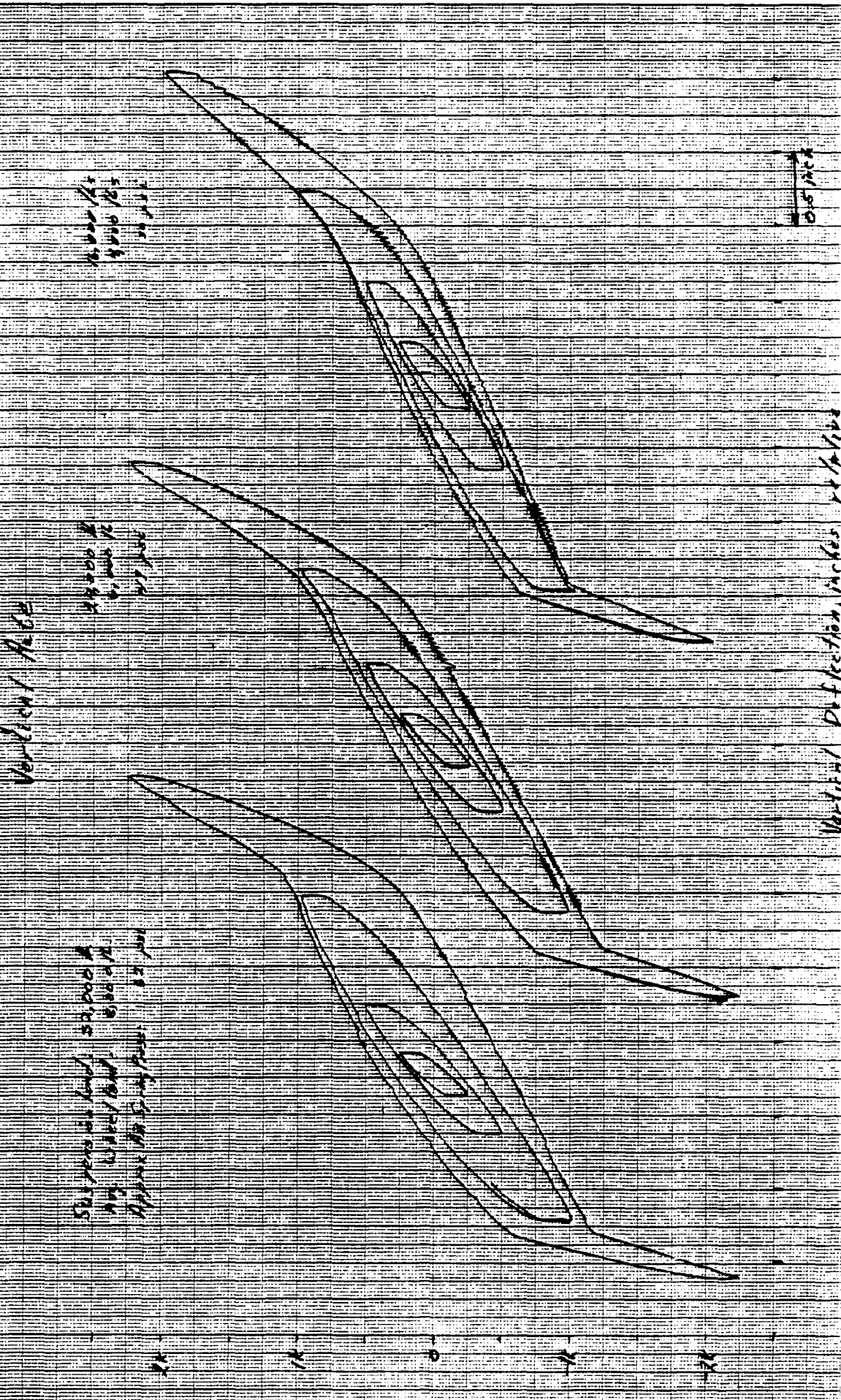


Suspension: NEWAY ARD 234

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>tr</del>				
Vertical Rate (lb/in)		675	625	500
Coulomb's Friction (lb)		475	375	300
Roll RATE (in./lb/deg)	F	35K	33K	25K
	R	35K	32K	26K
Roll CENTER HT (in below top of frame)	F		8.7	
	R		9.1	
Roll STEER (DEG / DEG)	F		.23	
	R		.22	
ALIGNING MOMENT COMPLIANCE STEER (deg/in lb)	F		$5.5 \times 10^{-6}$	
	R		$6.5 \times 10^{-6}$	
LATERAL FORCE COMPLIANCE (deg/lb)	F		$1.7 \times 10^{-6}$	
	R		$1.5 \times 10^{-6}$	
INTERAXLE LOAD TRANSFER (BYRINK) (lb/lb)			nil	

Pt. 1.0

Mary ARD 234

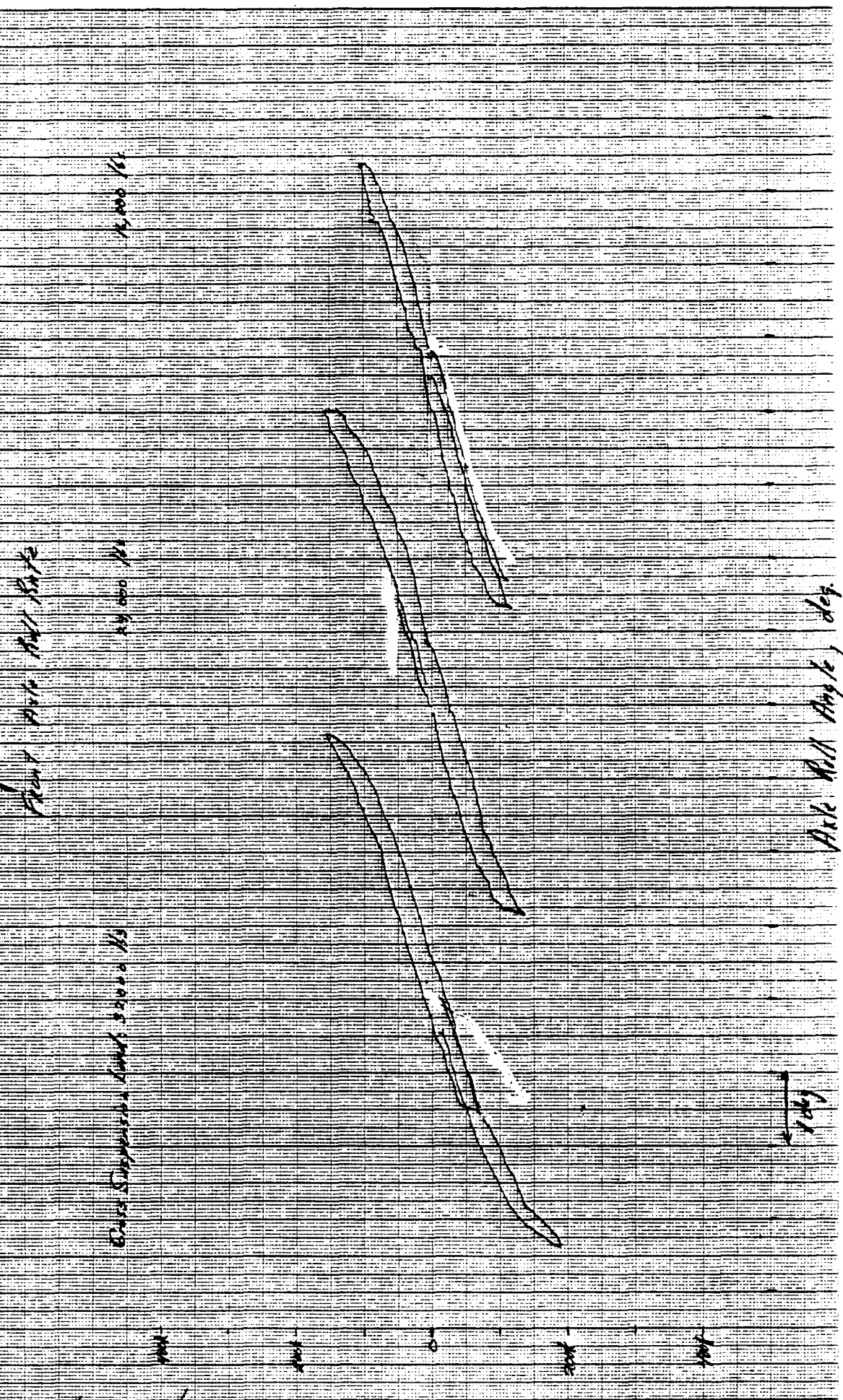


Average Vertical Load per wheel, pounds relative to nominal load

Vertical Deflection, inches relative to nominal

Fit 6 = 0

Henry APD 234



Roll Moment on Axle, in-lb ( → Right Side Compressed )

Axle Roll Angle, deg

Nuway ARD 234

Photo

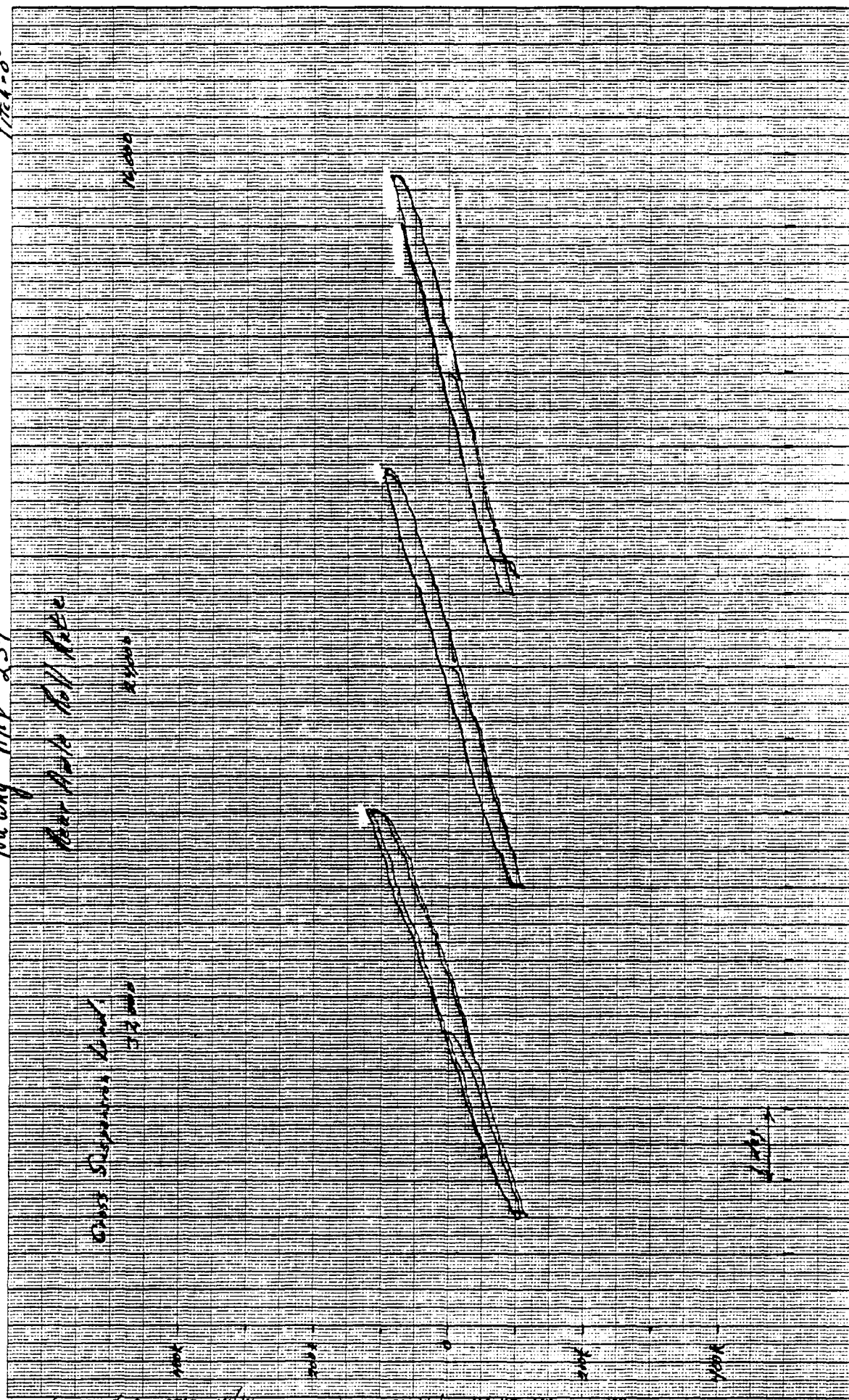
Roll Angle Roll Rate

Roll Rate Roll Rate  
17.5000

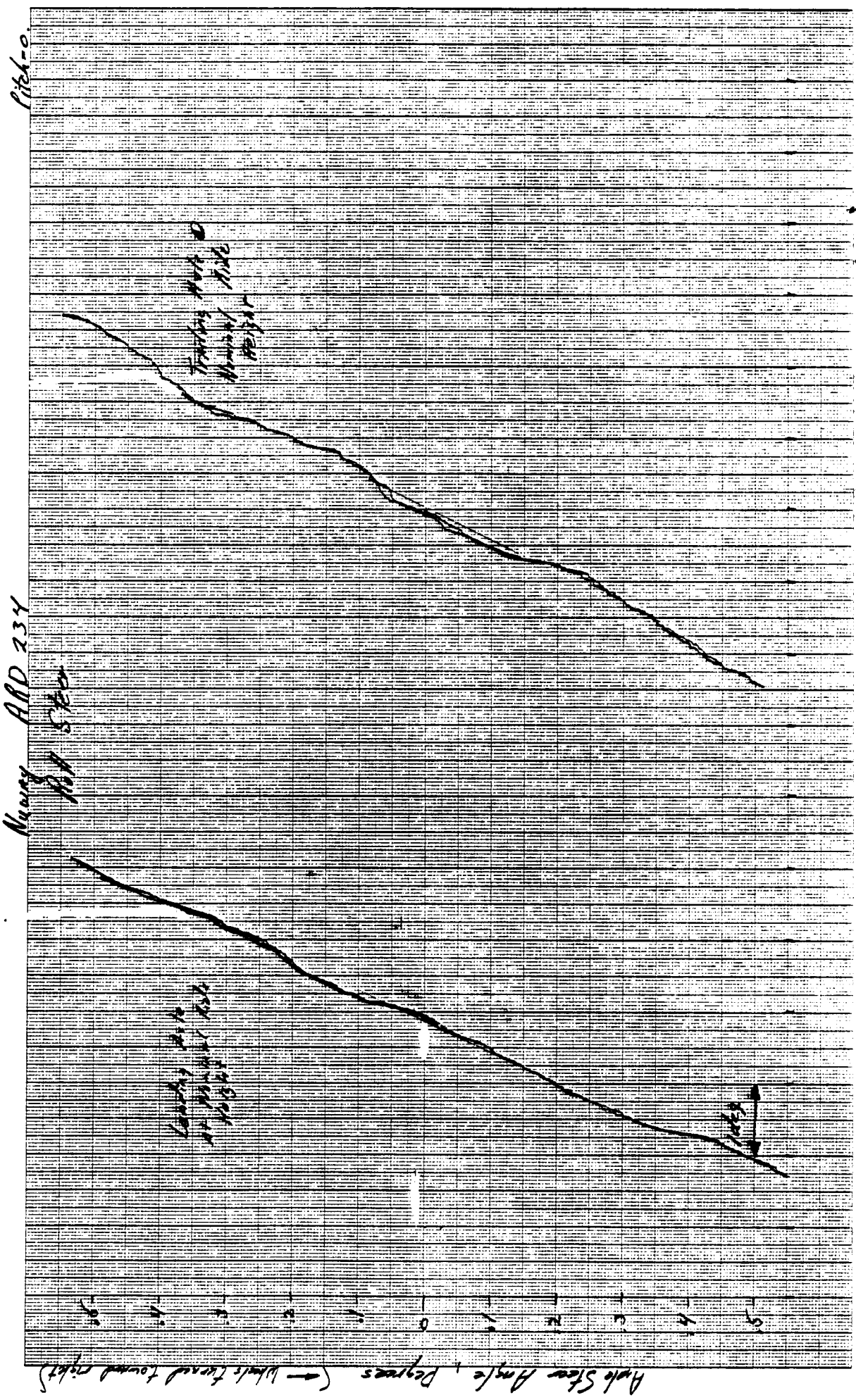
17.5000

17.5000

Roll Moment on Aft, a-16 (Right Side Compressed) →



Roll Rate, deg.



Roll Angle, Degrees (Whole turned toward right)

Roll Angle, degrees (Right Side Compressed)



Angle Steer Angle, Degrees (→ when turned toward right)

Newby ARD 234

ARD 00

Aligning Moment Compensates Steer

Leading Side at  
Nominal Side Right

Trailing Side at  
Nominal Side Right

Aligning Moment Applied to each wheelset (Applied to all four), M.A. (positive →)

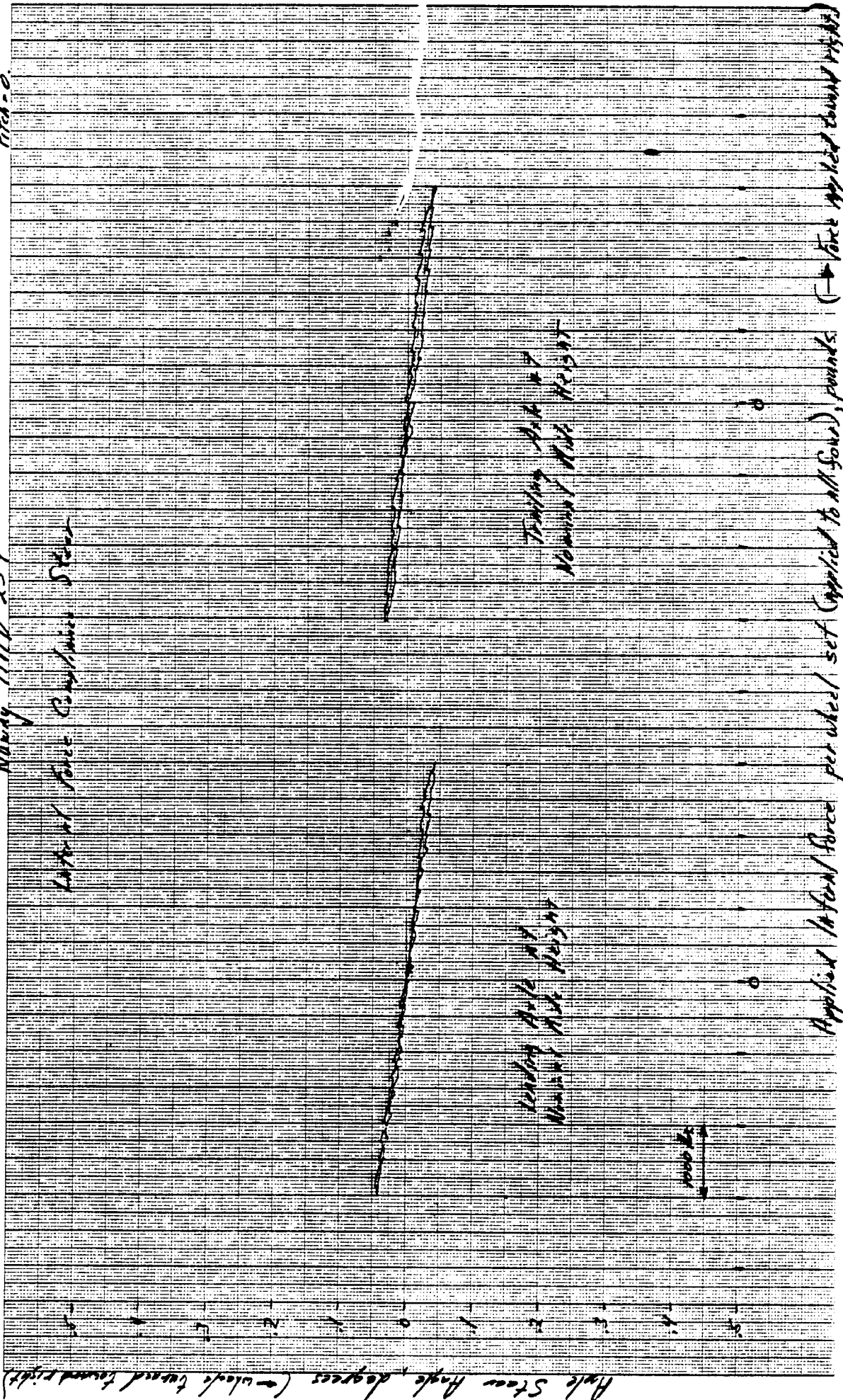
4000 lb-ft

Pitch - 0.

Navy APR 234

External Force Condition Steer

Axle Steer Angle, degrees (wheels turned toward right)

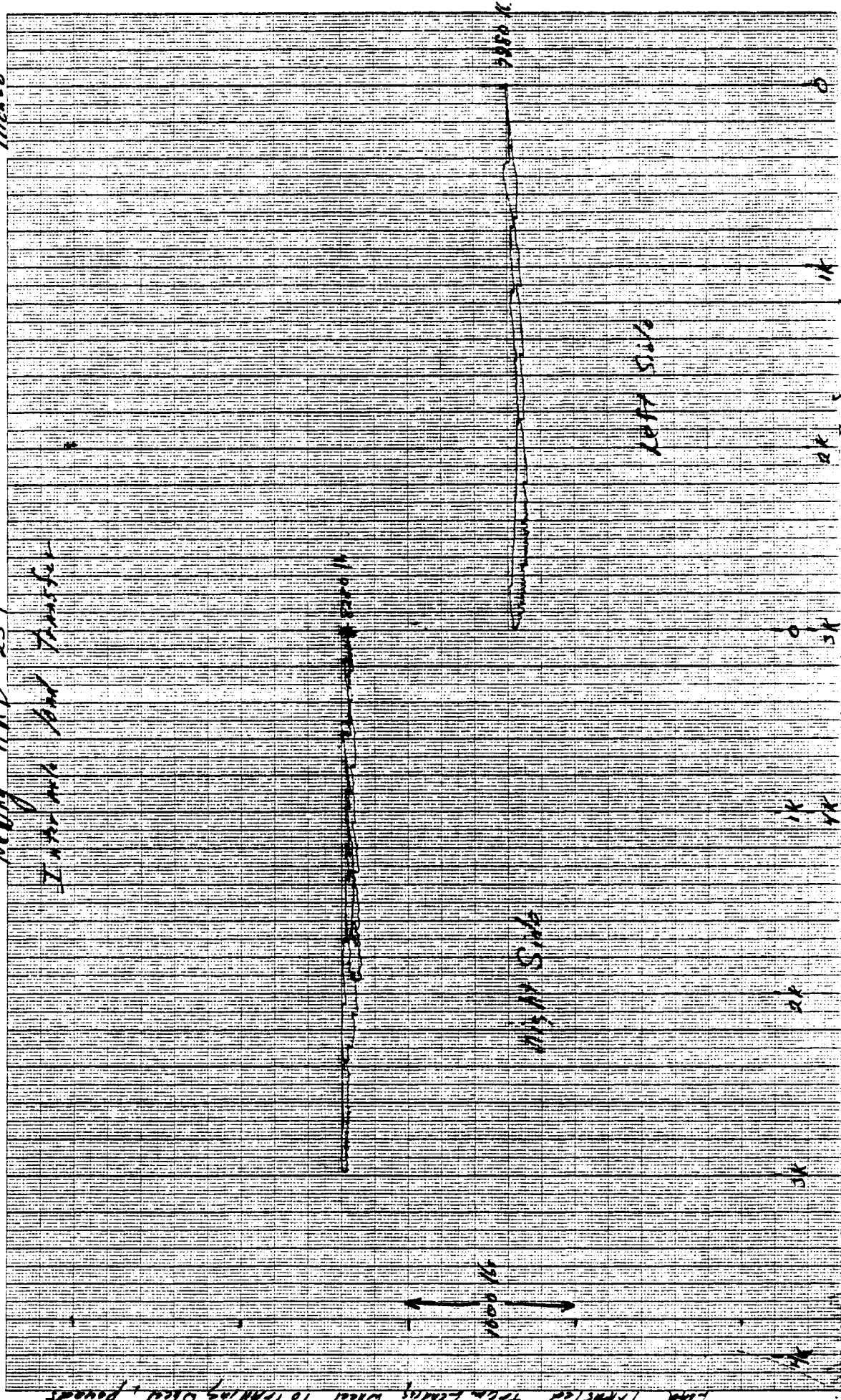


NO. 1111 A.R.D. 234

Pitch 0°

Inter axle load transfer

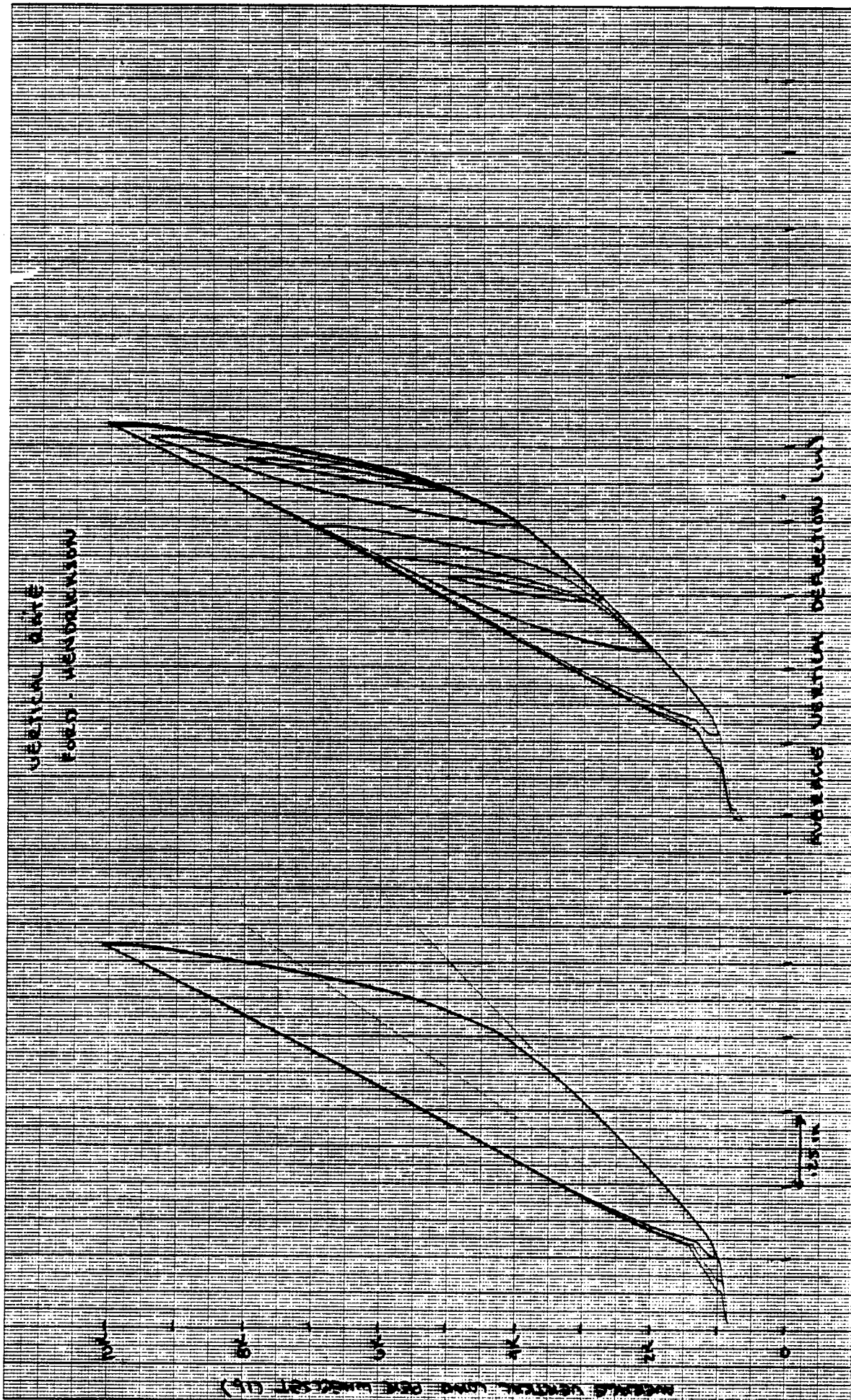
Load transferred from leading wheel to trailing wheel, pounds



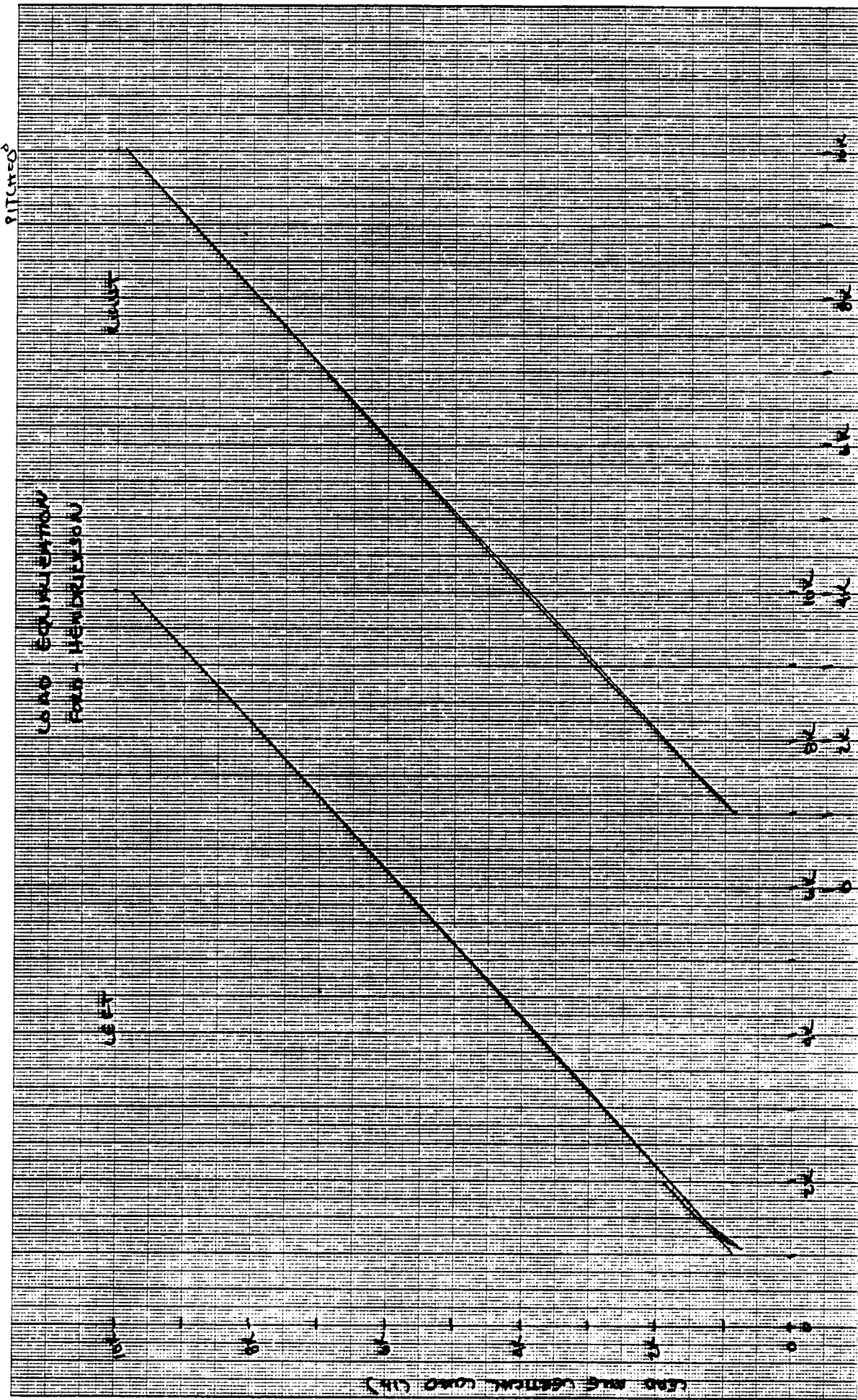
Applied Brake Force per Wheel Set (Applied to All four) pounds.

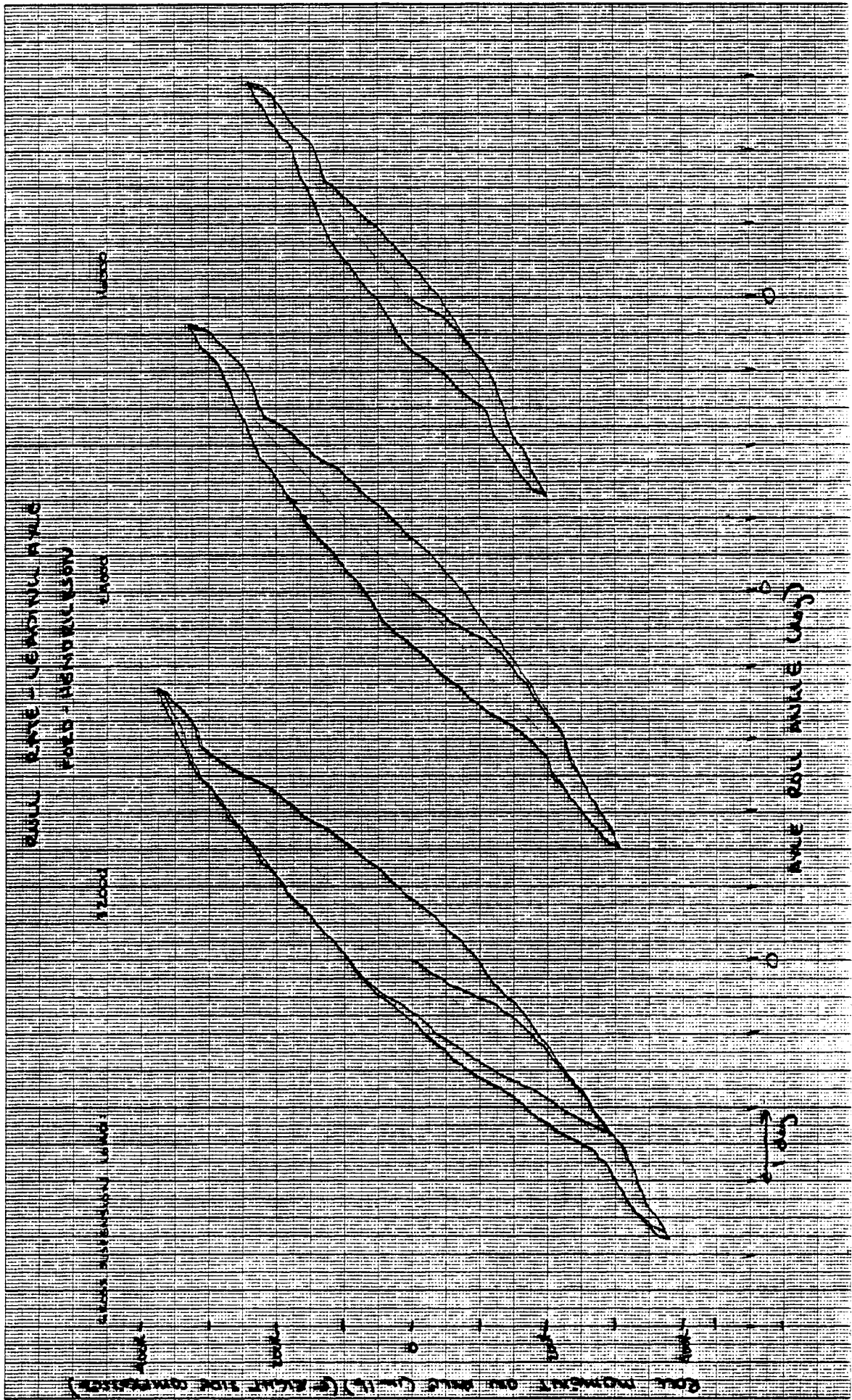
Suspension: Hendrickson Walking Beam - Steel Spring

		Susp Load (lb)		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
Vertical Rate (lb/in)		6280	6280	6280
Coulomb Friction		2500	1850	1150
Roll RATE	F	115K	100K	85K
(in/lb/deg)	R	110K	95K	85K
Roll CENTER HT	F	17.3	15.8	15.8
(in below top of beam)	R	17.8	17.4	16.7
Roll STEER	F	.16	.23	.24
(DEG / DEG)	R	.19	.26	.26
ALIGNING MOMENT COMPLIANCE STEER	F	$7 \times 10^{-6}$	$6 \times 10^{-6}$	$6 \times 10^{-6}$
(deg/in/lb)	R	$1 \times 10^{-5}$	$1 \times 10^{-5}$	$1 \times 10^{-5}$
LATERAL FORCE COMPLIANCE	F	_____ nil _____		
(deg/lb)	R			
INTERAXLE LOAD TRANSFER (BRANK)	F	_____ nil _____		
(lb/lb)	R	for 0 - -2° pitch		

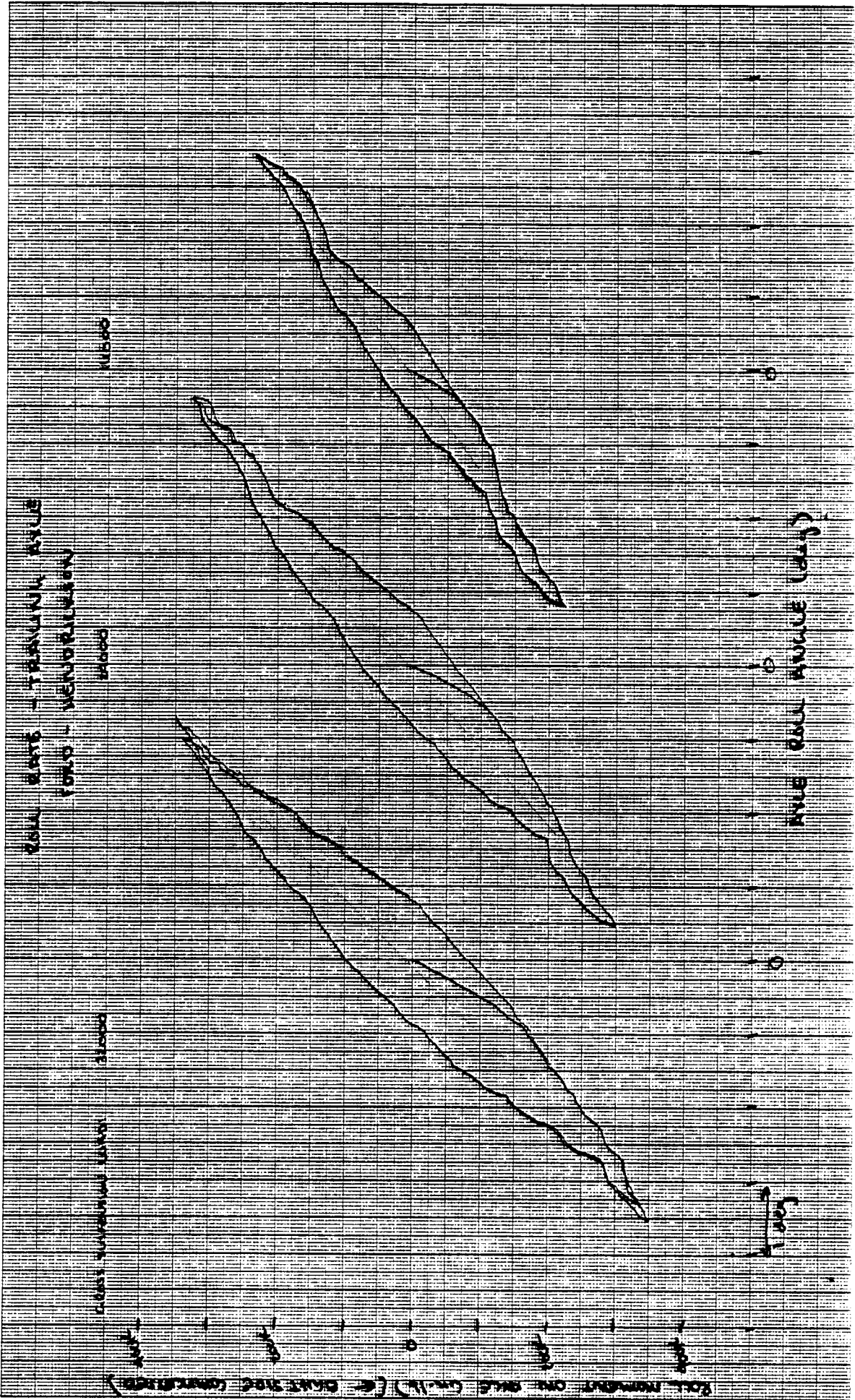


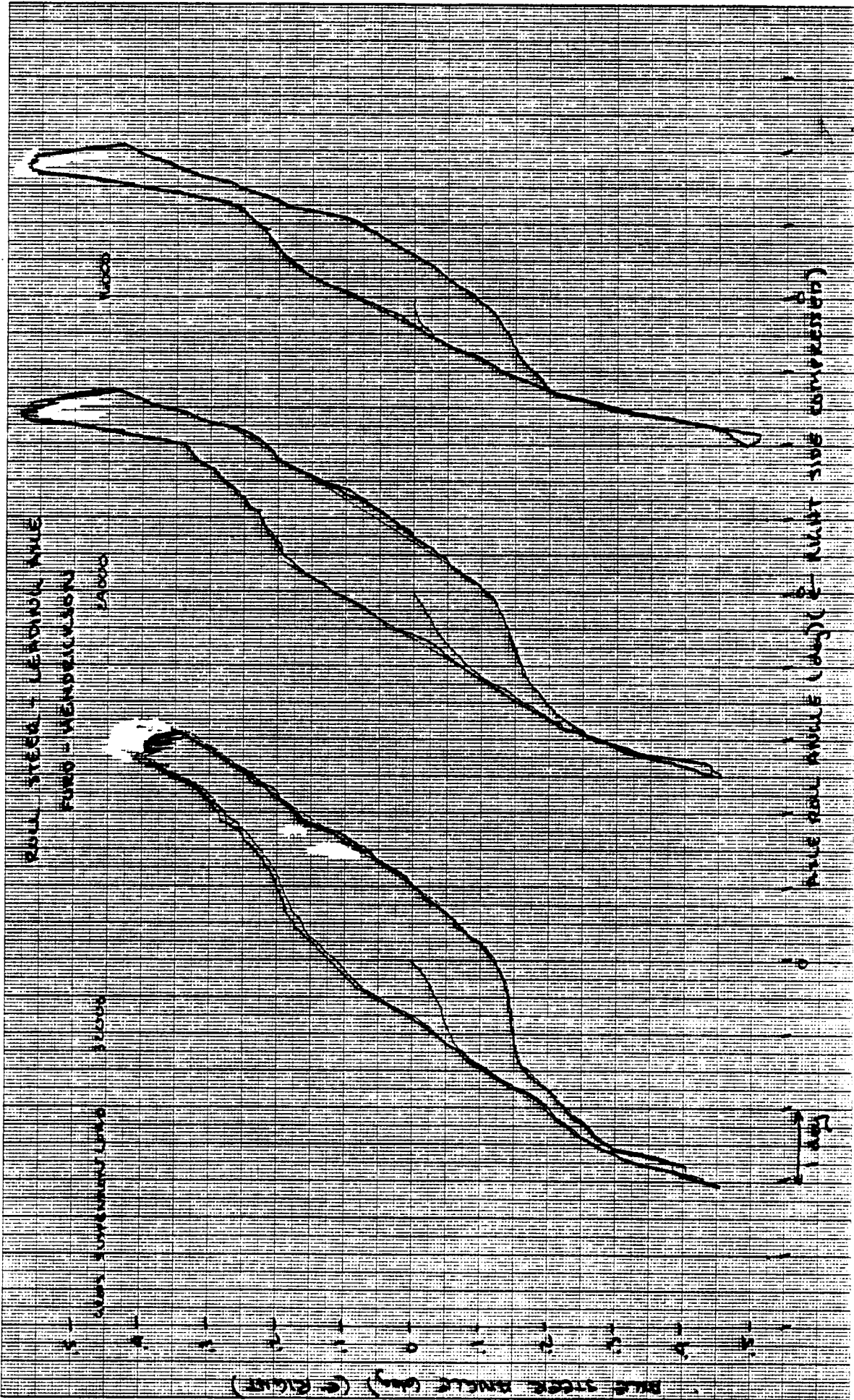






ROLL MOMENT ON PALE (L-12) (RIGHT SIDE COMPASS)





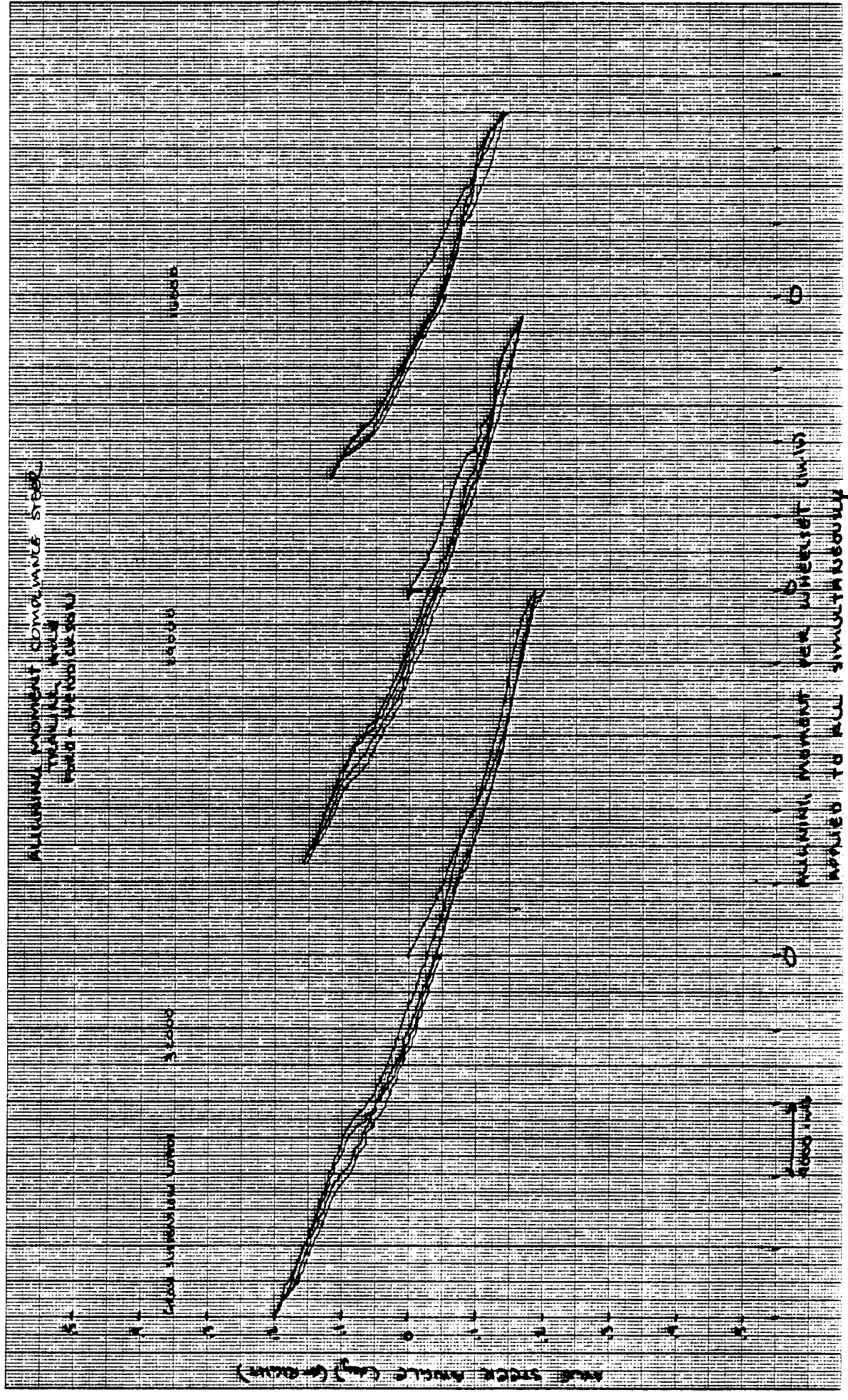
ROLL ANGLE - LEADING WHEEL (← RIGHT)



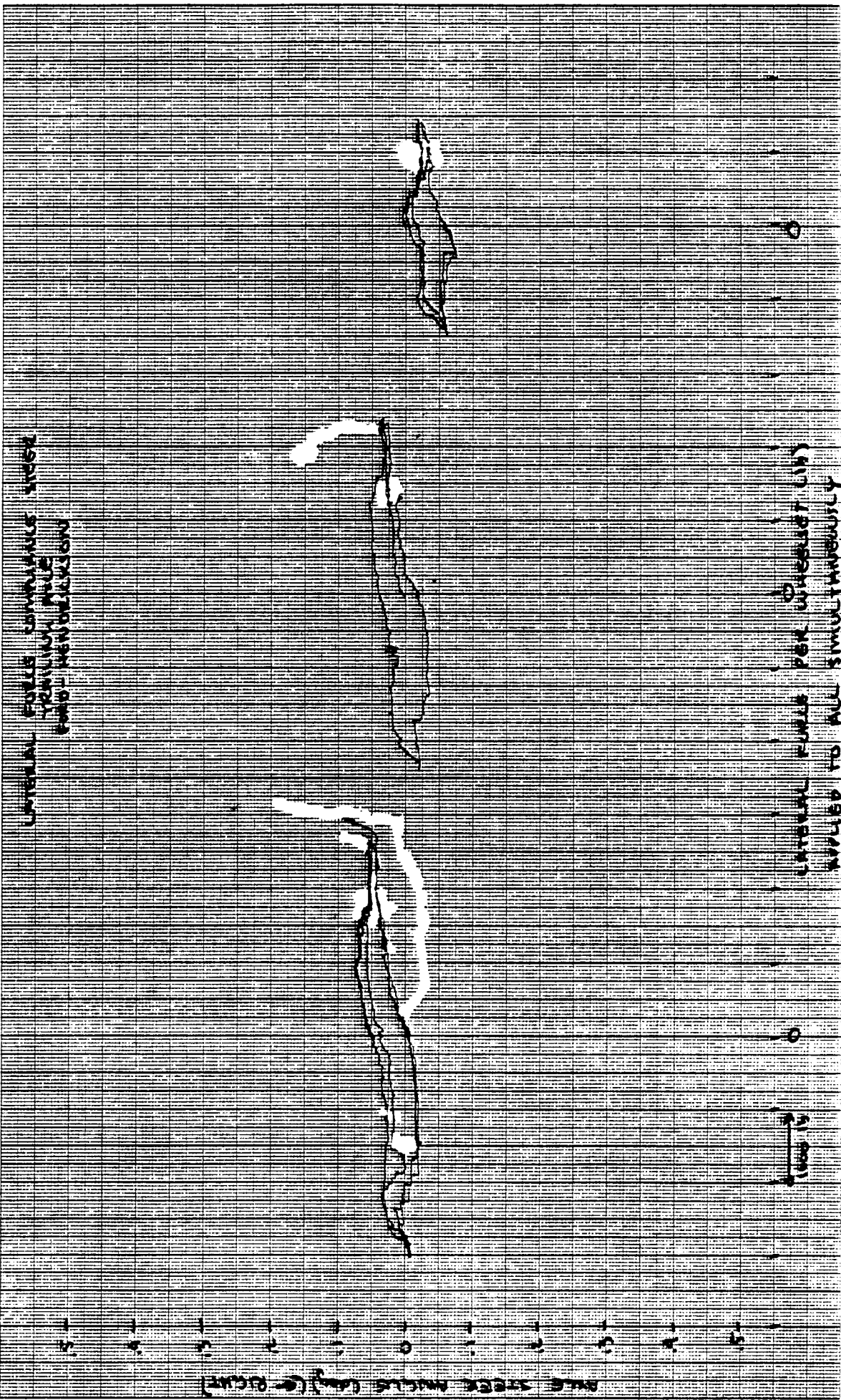














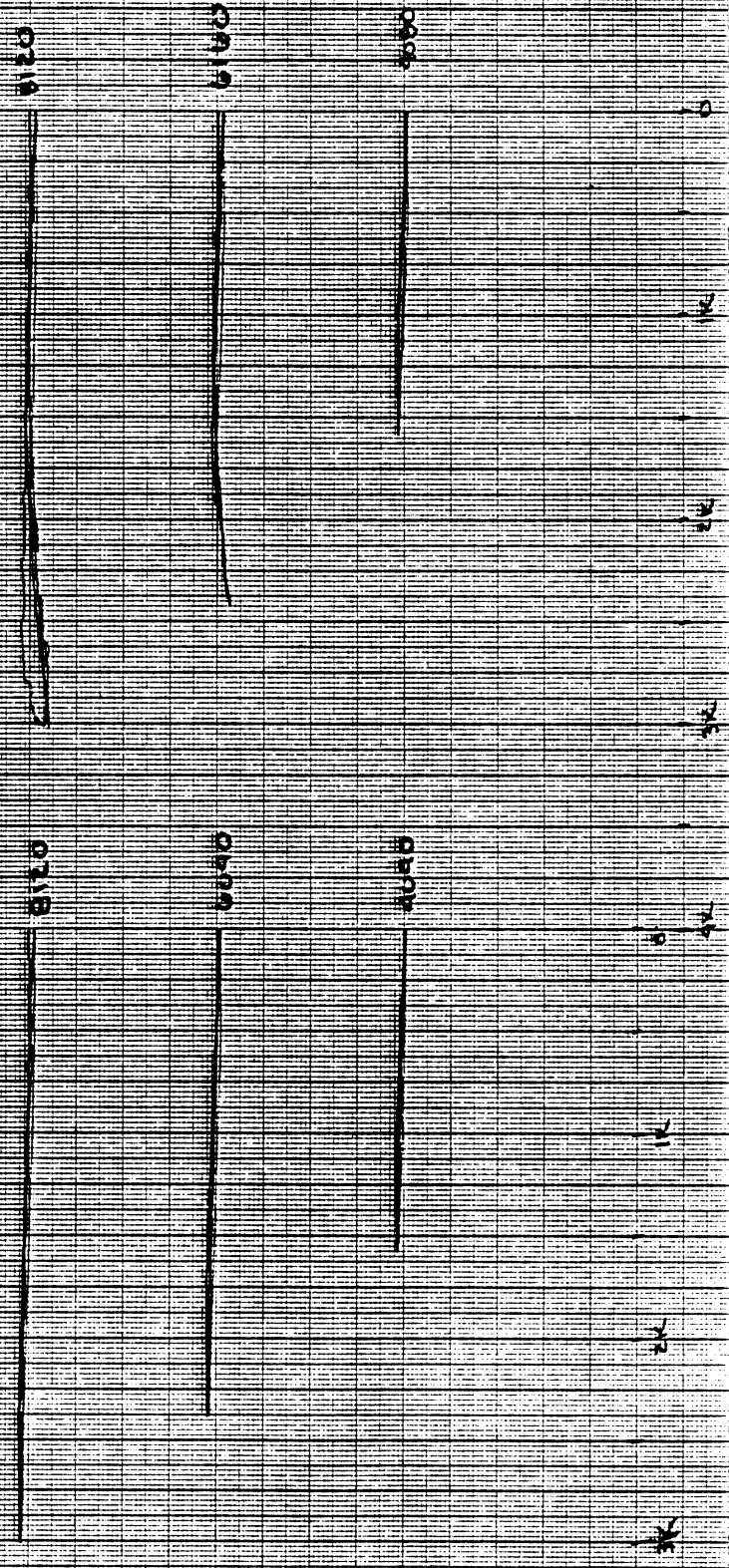
0.00000

INTERAXLE LOAD TRANSFER DUE TO BRAKE FORCE  
FUEL MEMBERSHIP

Right

LOAD TRANSFER FROM (AXON) TO TRAILING AXLE (BY)

10000



BRAKE FORCE PER WHEELSET (10)  
APPLIED TO ALL SIMULTANEOUSLY



PITCH-1

INTEGRAL LOAD TRANSFER DUE TO BLAKE PITCH  
FORWARD - HORIZONTAL

LEFT

DRAYS

DRAYS

DRAYS

DRAYS

DRAYS

DRAYS

LOAD TRANSFERRED FROM DRAYS TO TRACKS

1.5000

AK

AK

AK

AK

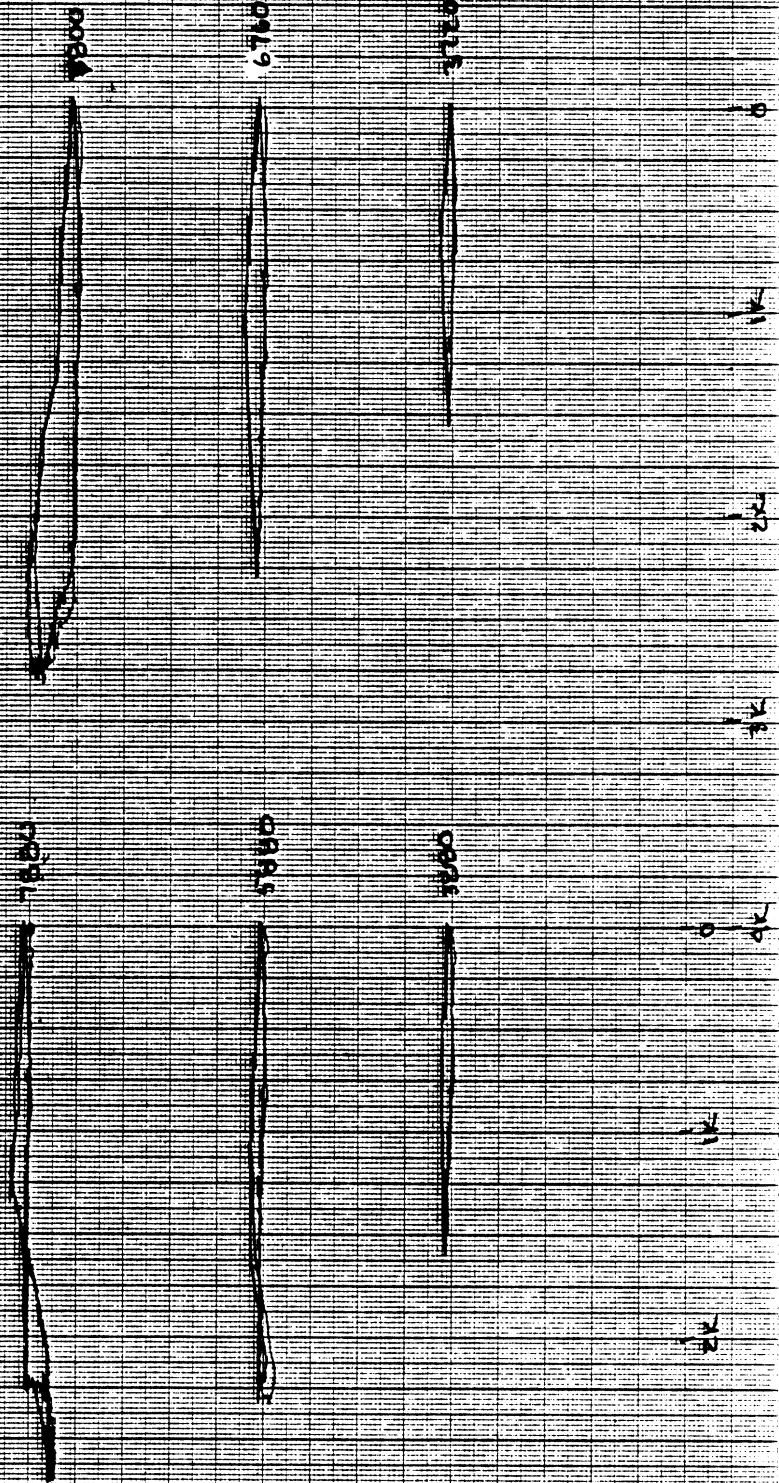
AK

DRAYS FORCE PER WHEELSET (1/2)  
APPLIED TO ALL SIMULTANEOUSLY

PITCH = -2°

INTERAXIAL LOAD TRANSFER DUE TO BRAKE FORCE  
FORces - MECHANISMS

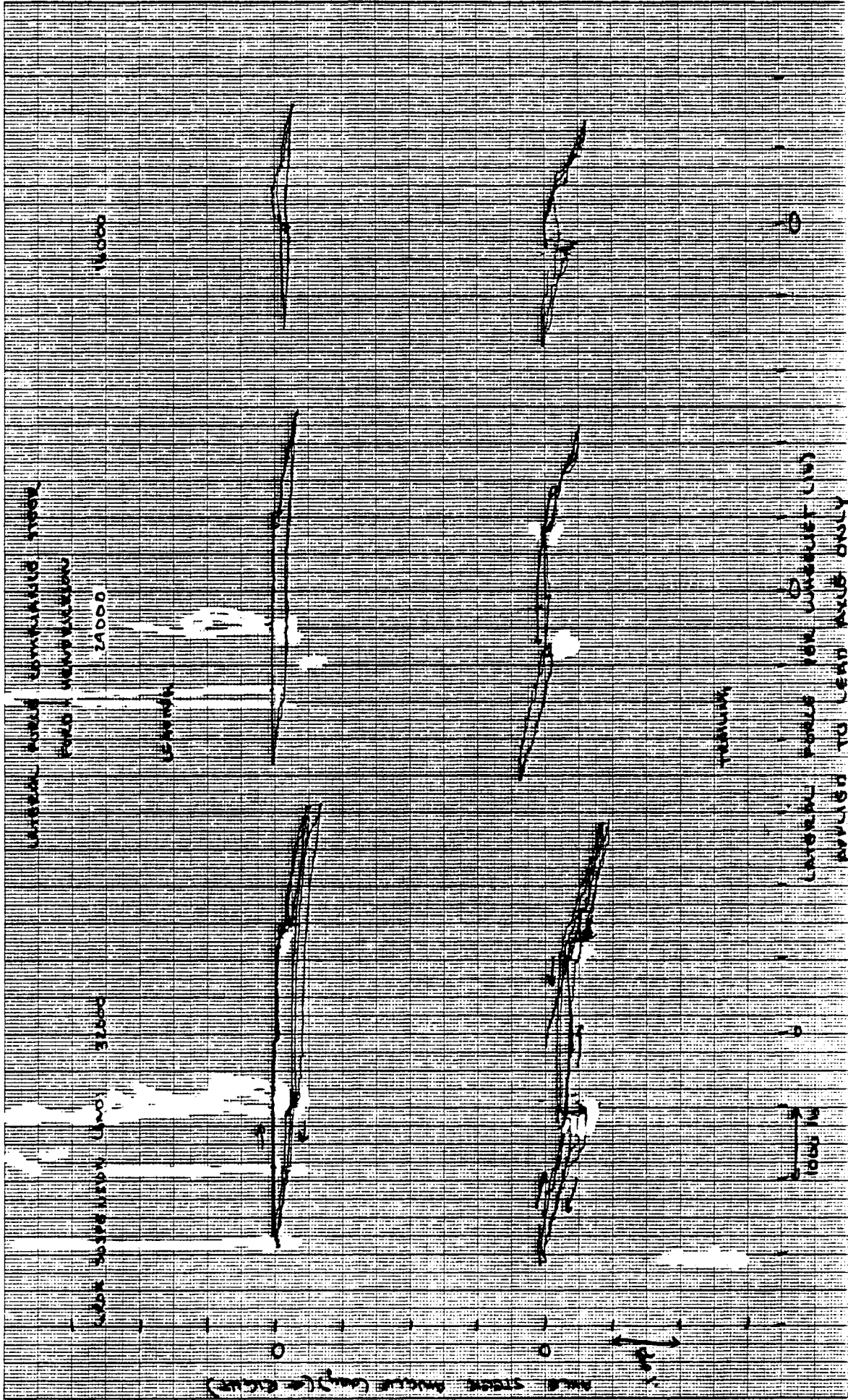
LOAD TRANSFER FROM FRONT TO REAR (DECELT)



BRAKE FORCE PER WHEELS (lb)  
APPLIED TO ALL SIMULTANEOUSLY



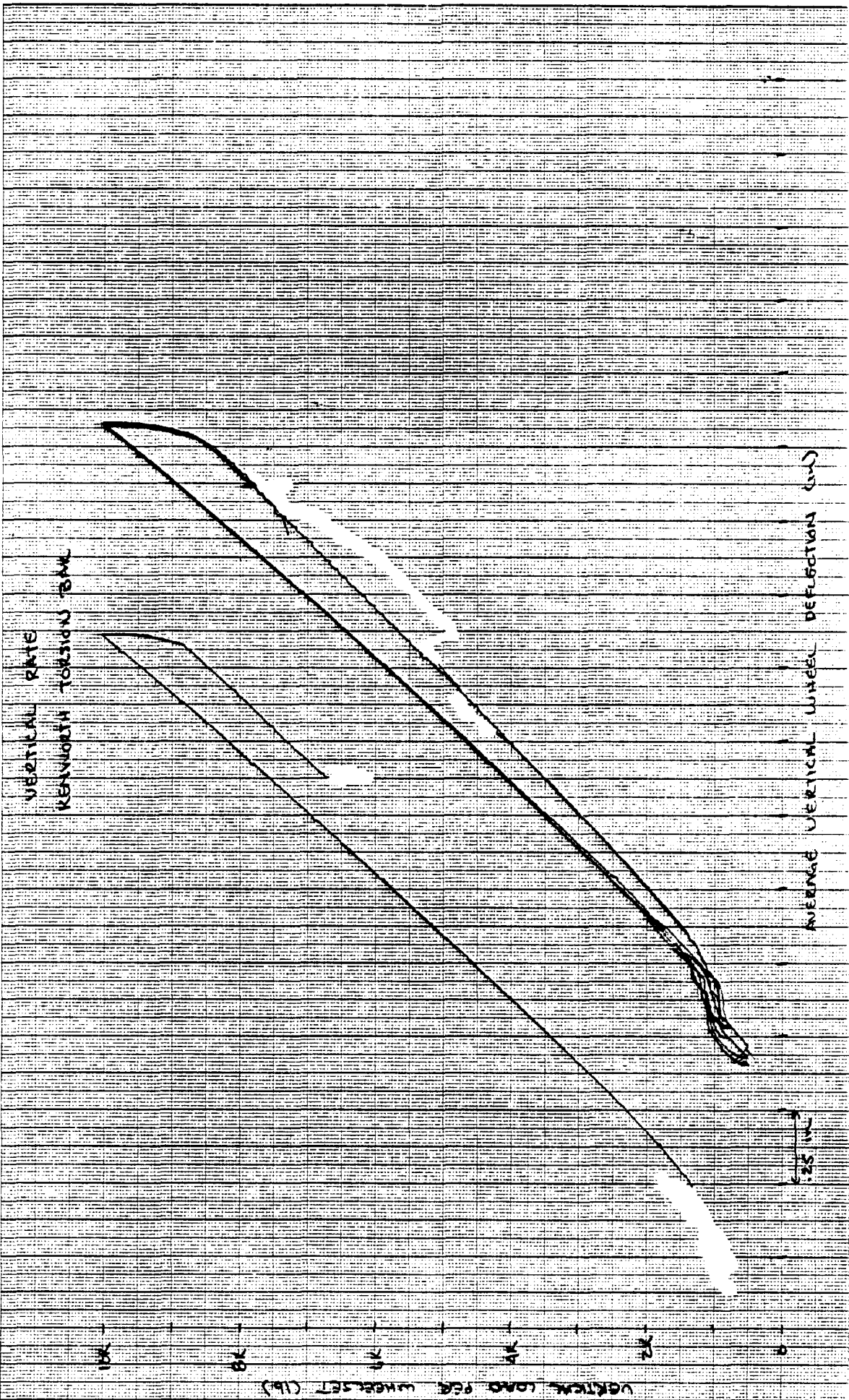


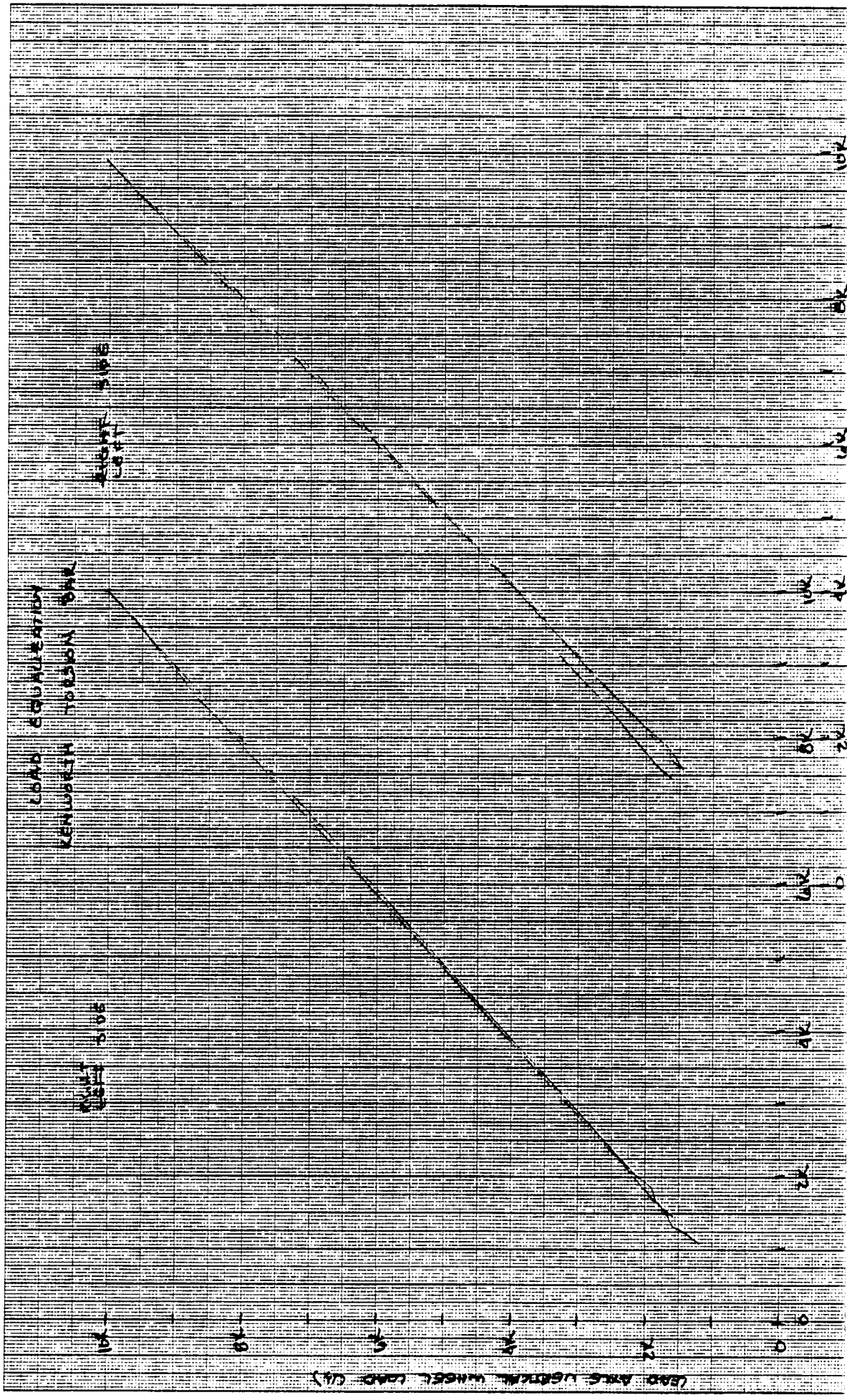


Suspension: KENWORTH TORSION BAR

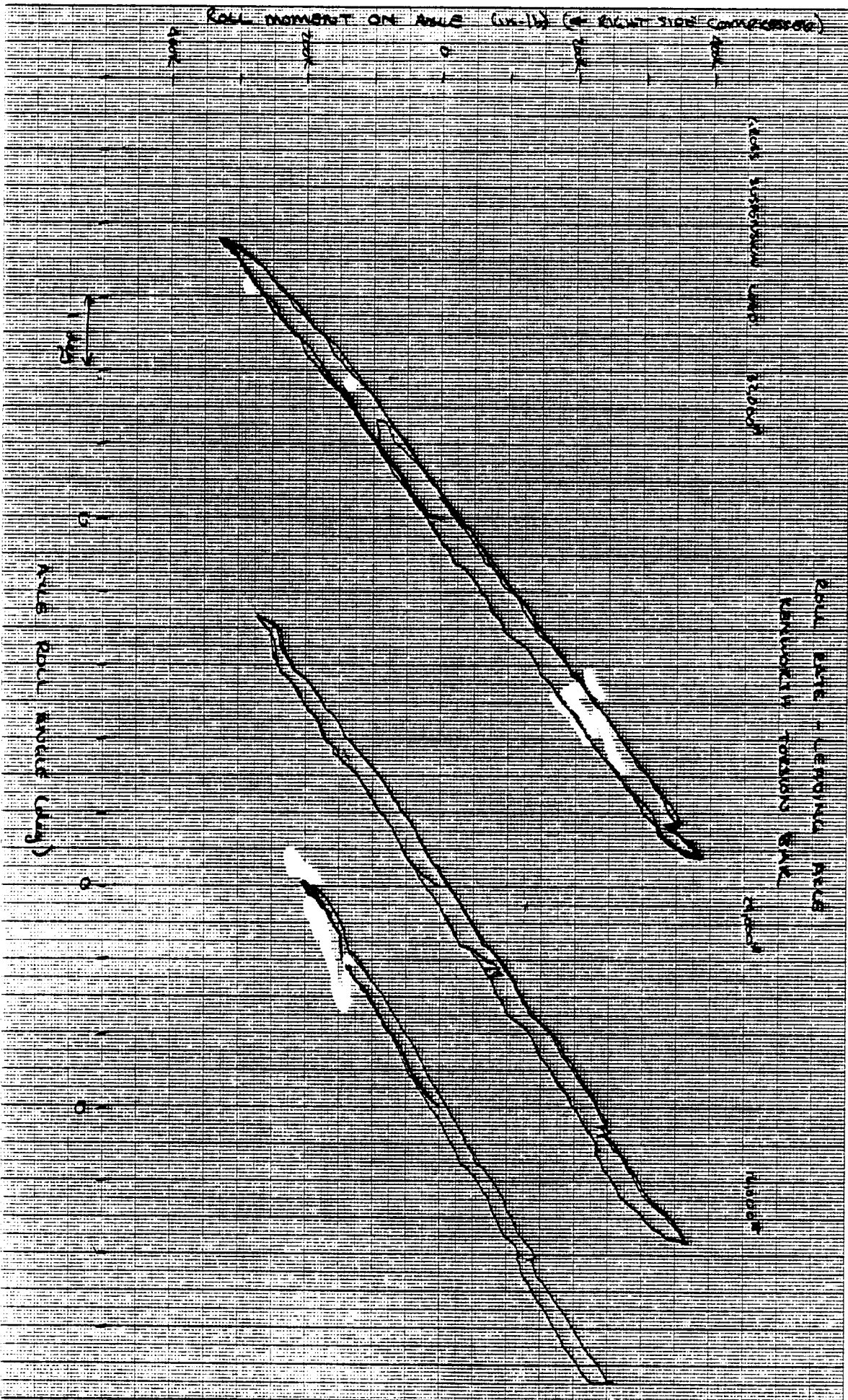
		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>tr</del>				
Vertical Rate (lb/in)		5000	4000	4800
Coulomb Friction		500	400	250
Roll RATE	F	80K	75K	65K
(in. lb / deg)	R	45K	35K	35K
Roll CENTER HT	F	13.4	13.0	NA
(in below top of frame)	R	15.5	15.9	16.5
Roll STEER	F	_____	nil	_____
(deg / deg)	R	- .015	- .01	- .02
ALIGNING MOMENT COMPLIANCE STEER	F	$5 \times 10^{-6}$	$5.2 \times 10^{-6}$	$5.2 \times 10^{-6}$
(deg / in lb)	R	$7.5 \times 10^{-6}$	$6 \times 10^{-6}$	$7 \times 10^{-6}$
LATERAL FORCE COMPLIANCE	F	_____	nil	_____
(deg / lb)	R		(lateral bush is substantial)	
INTERAXLE LOAD TRANSFER (BRKBL)				
(lb / lb)			NA.	

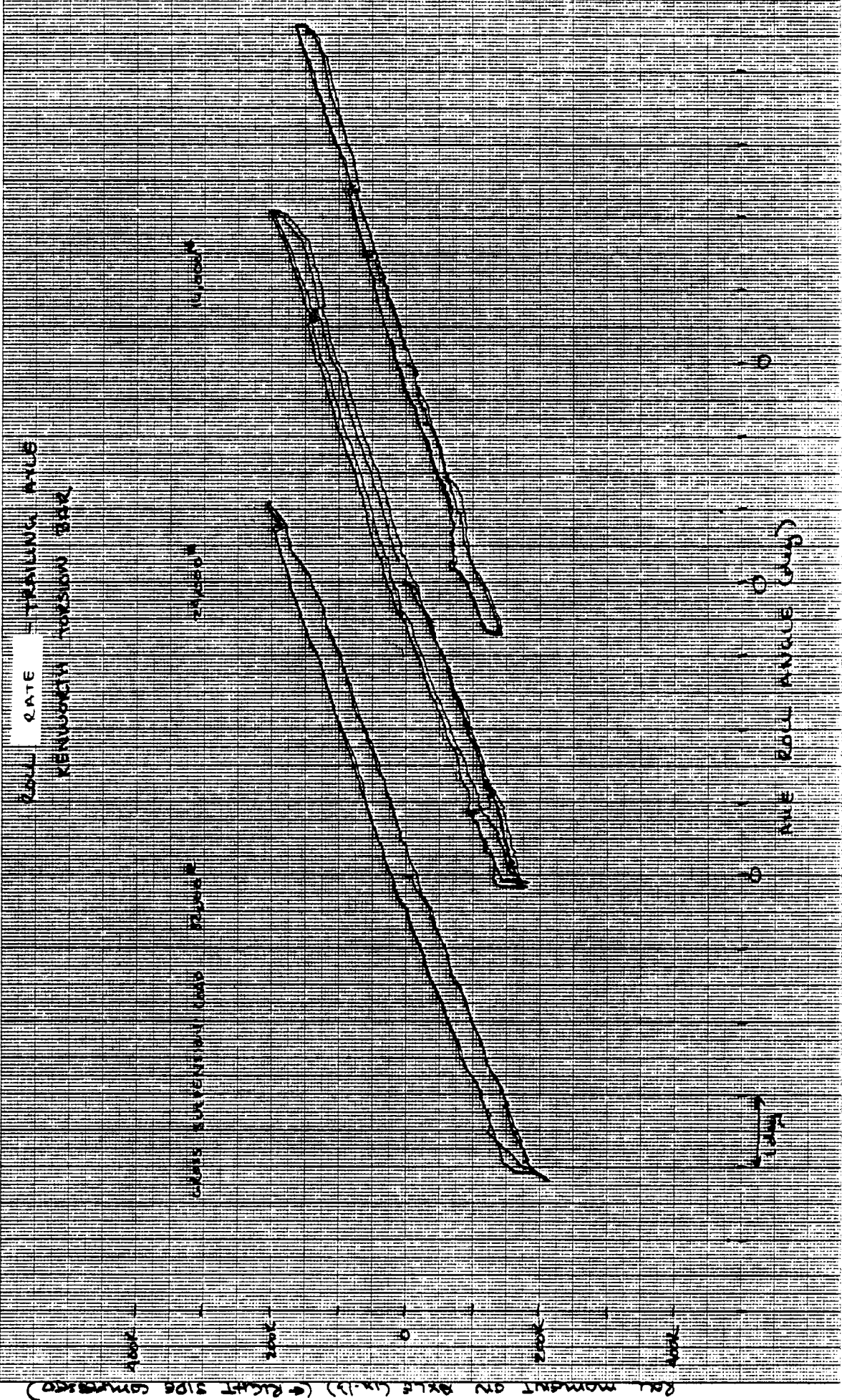




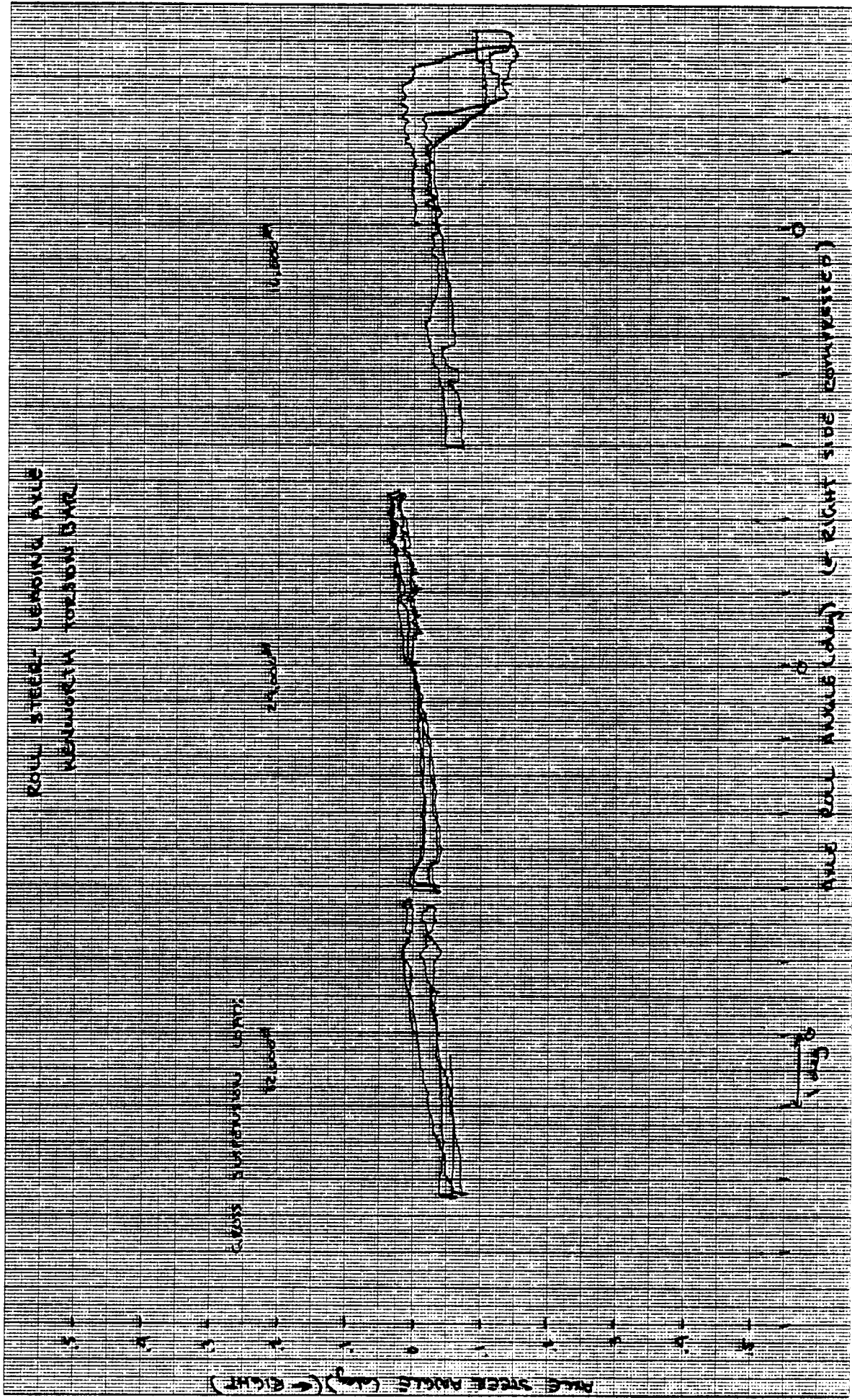


TRAILING WHEEL LOAD (lb)

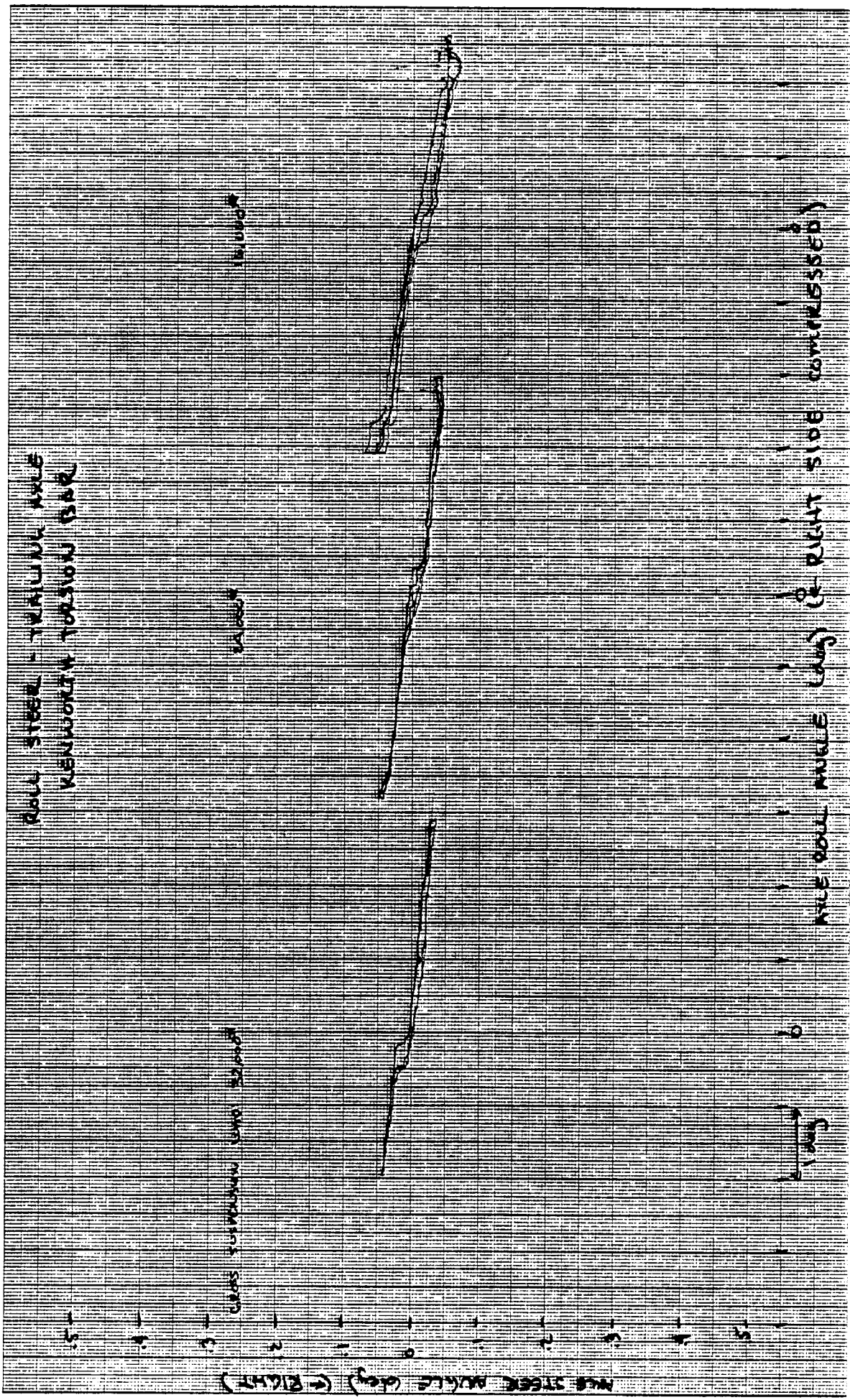




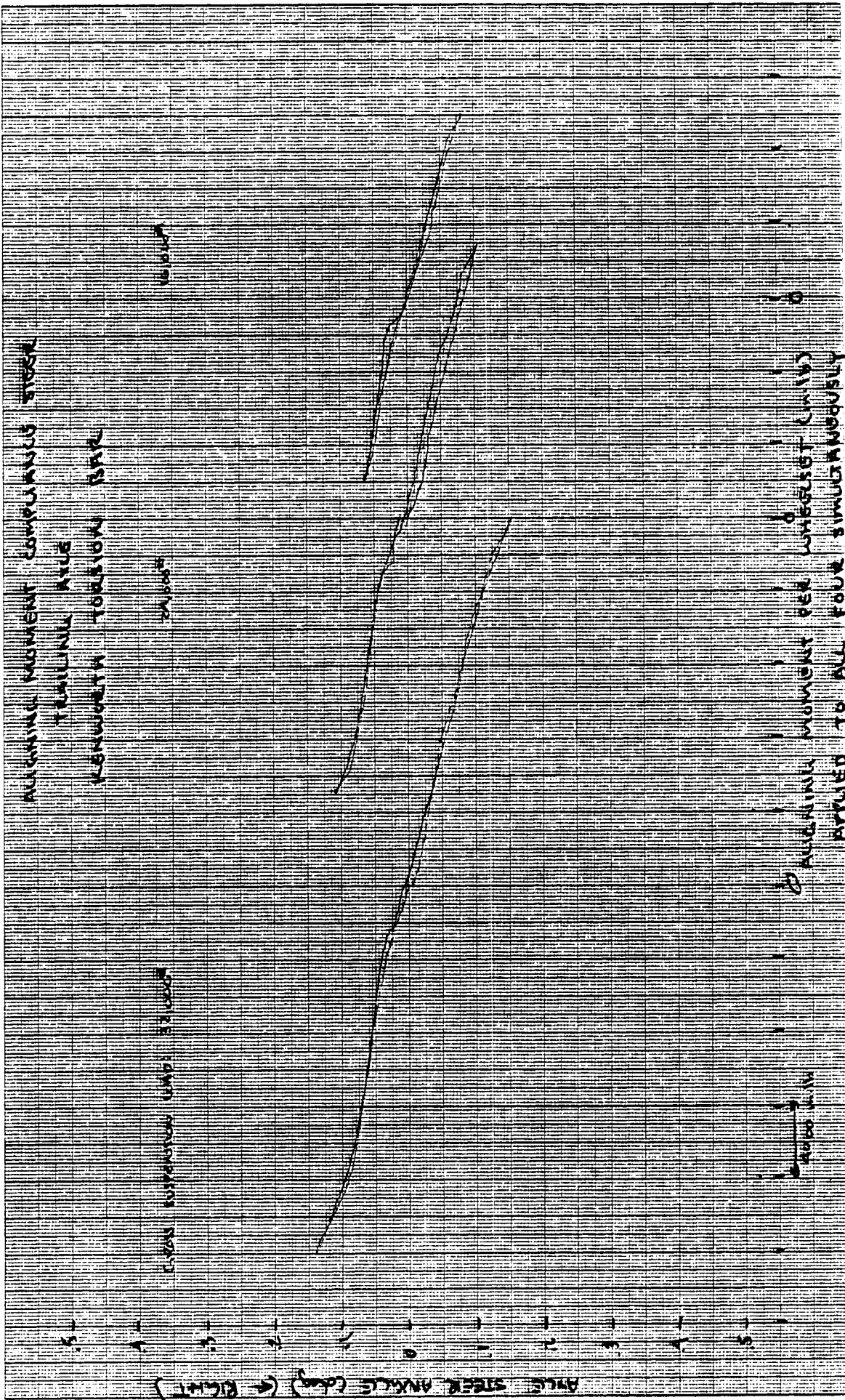




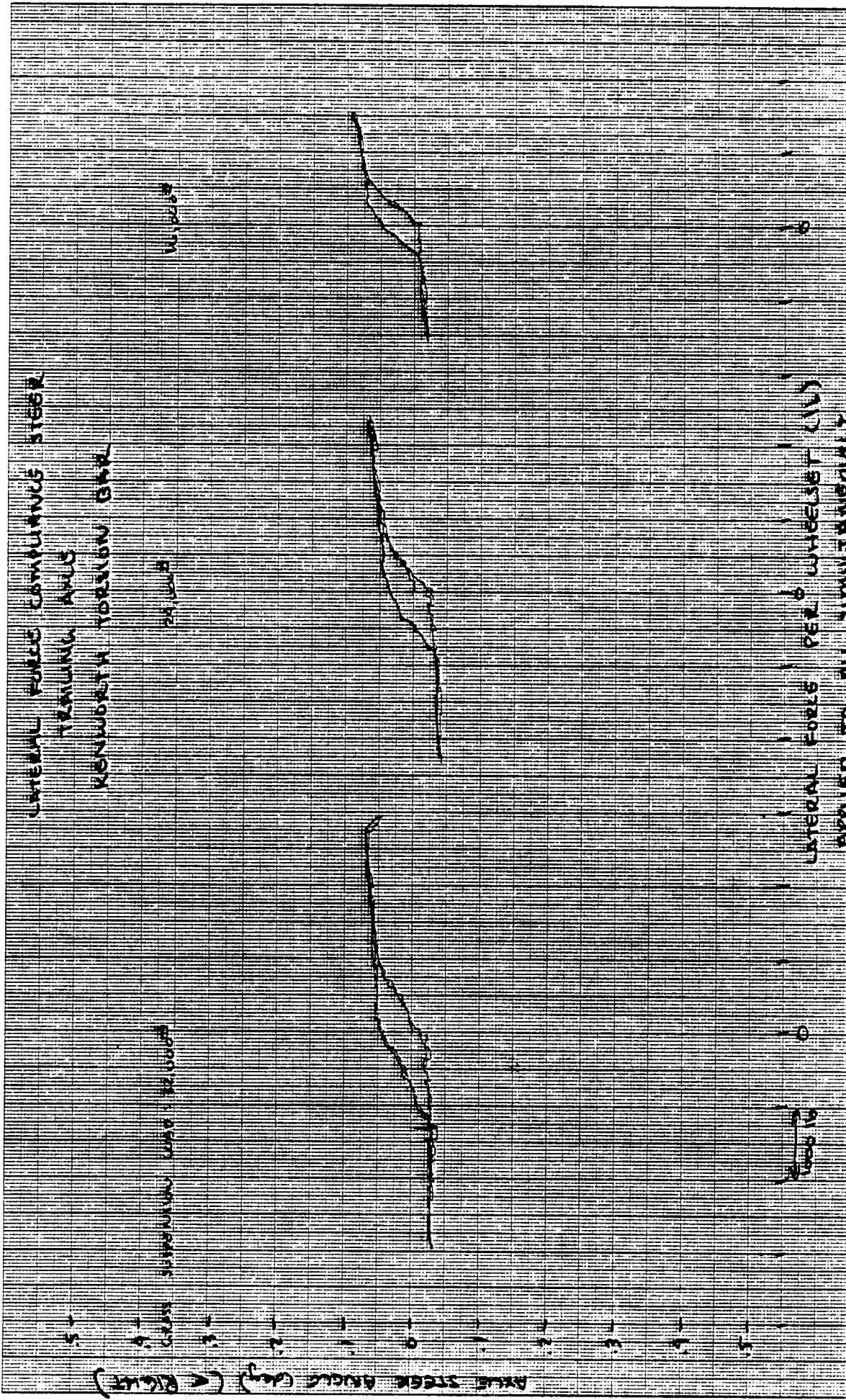




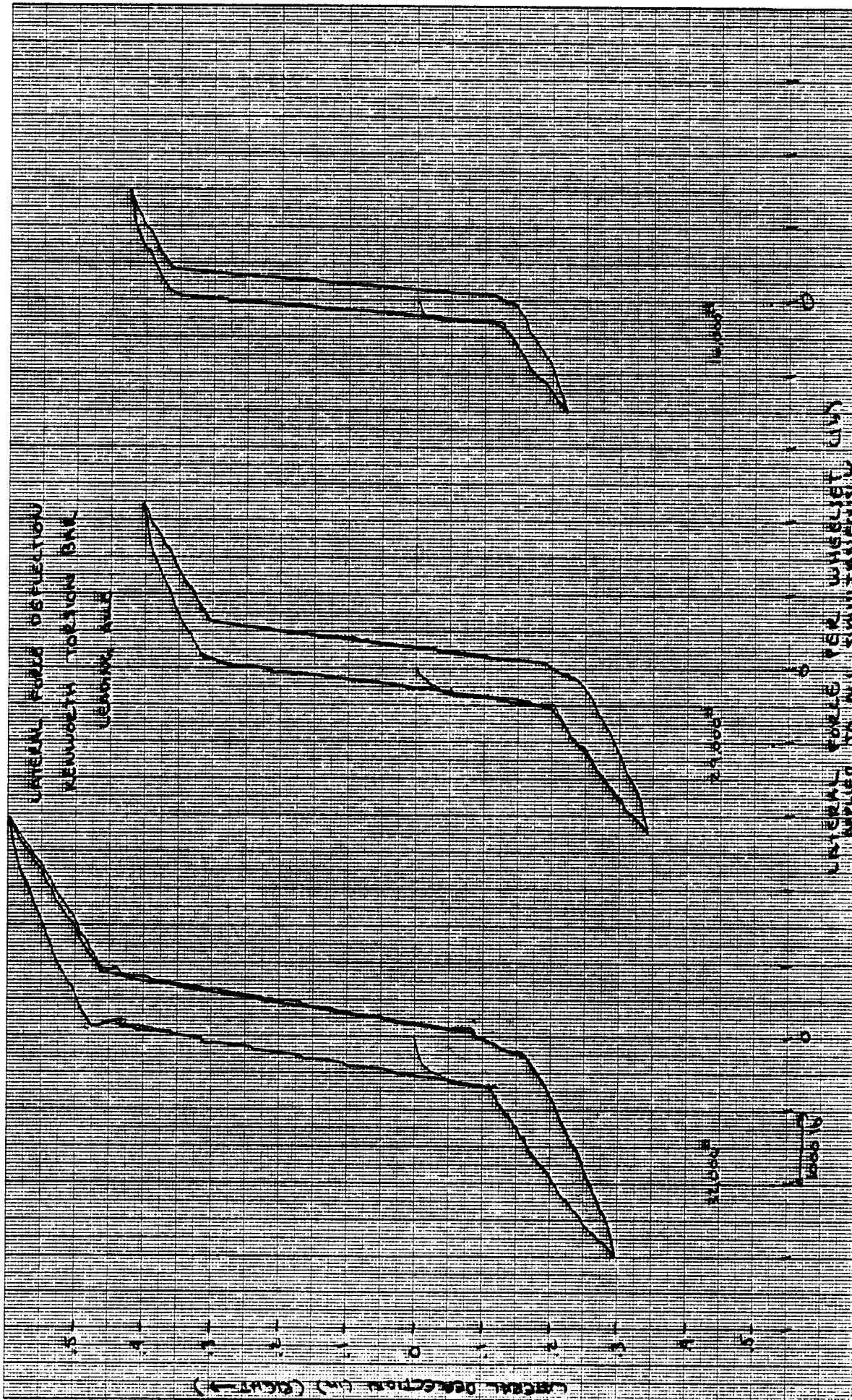








AXLE STEER ANGLE (deg) (← RIGHT)



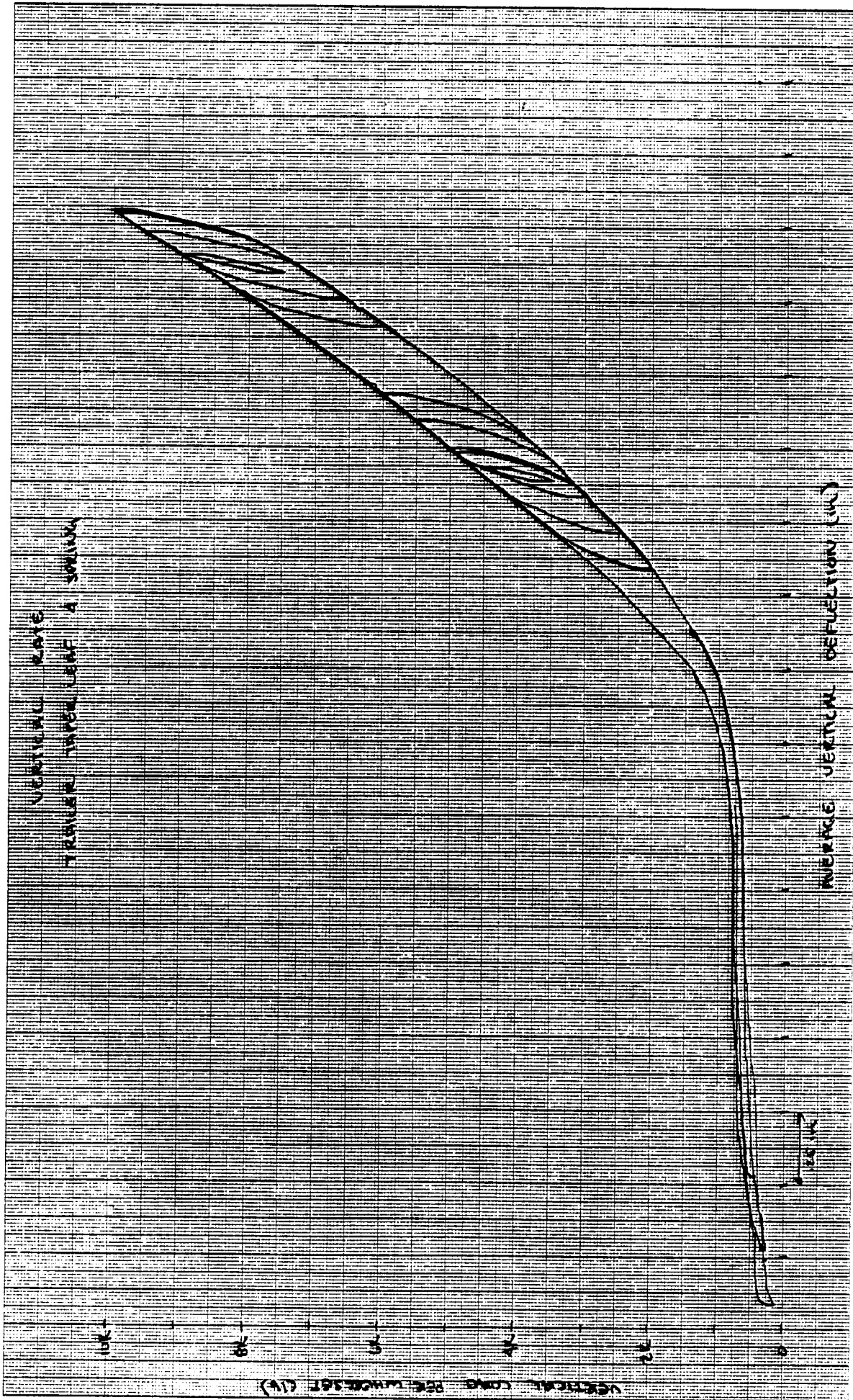






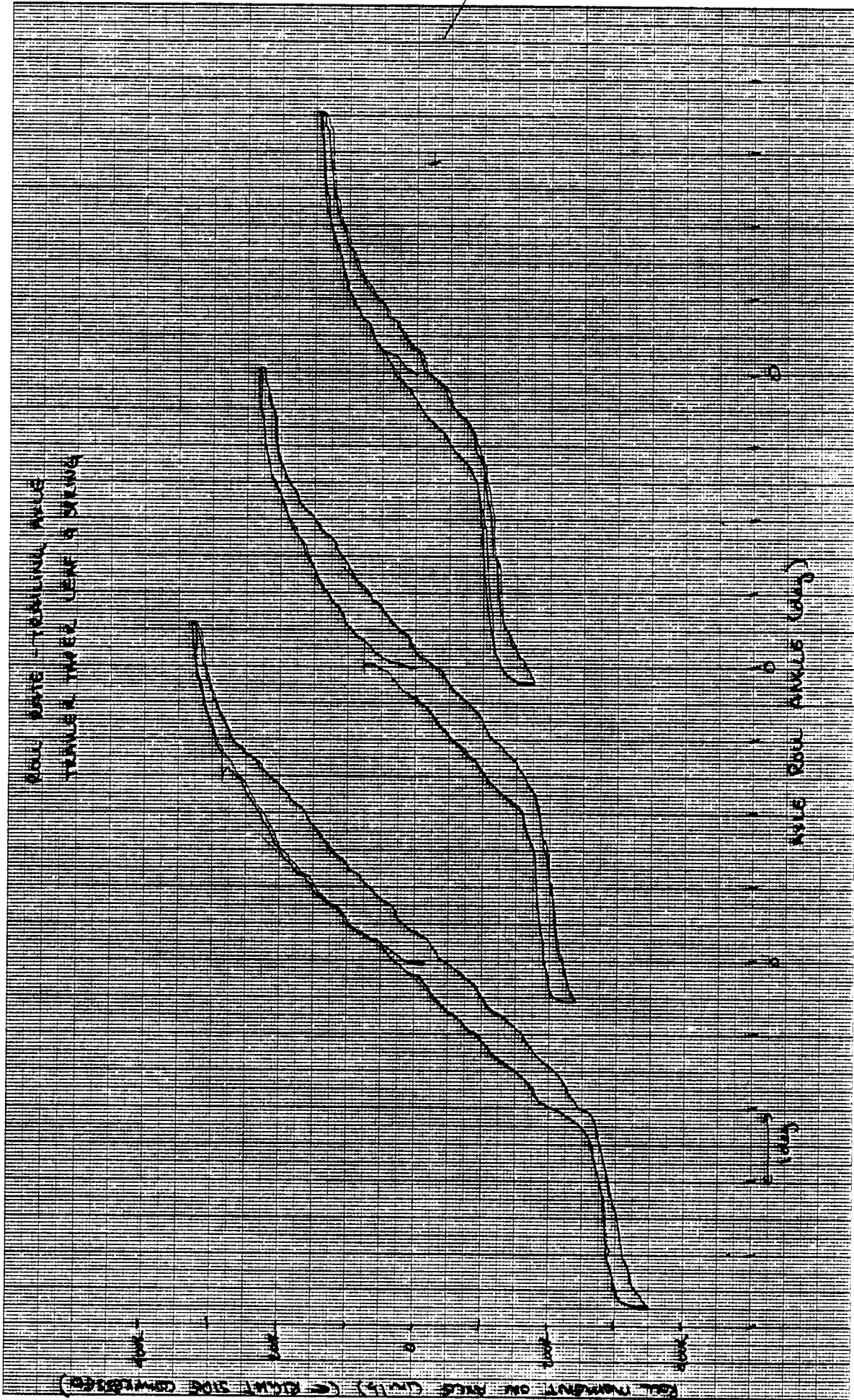
Suspension: Taper Leaf 4 Spring - Hutchens Trailer

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>R</del>				
Vertical Rate (lb/in)		6200	5800	5400
Coulomb Friction (lb)		750	700	550
Roll RATE	F	110K	100K	80K
(in./lb/deg)	R	120K	97K	75K
Roll CENTER HT	F	15.2	15.8	16.3
(in below top of beam)	R	15.9	15.6	14.7
Roll STEER	F	_____	NA	_____
(deg / deg)	R			
ALIGNING MOMENT COMPLIANCE STEER F				
(deg / in lb)	R	_____	NA	_____
LATERAL FORCE COMPLIANCE F				
(deg / lb)	R	_____	NA	_____
INTERAXLE LOAD TRANSFER (BRKIN)				
(lb / lb)		_____	NA	_____



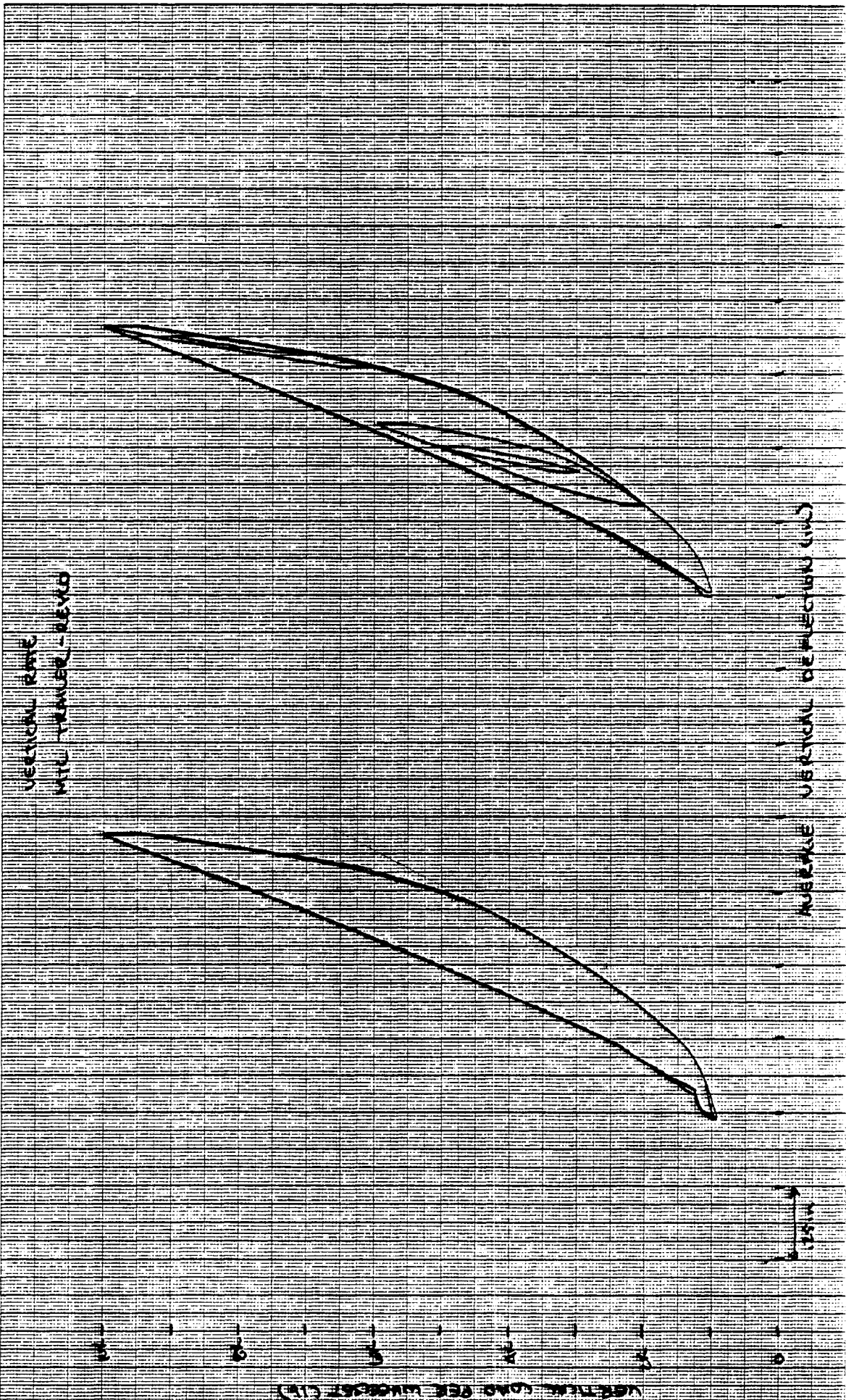




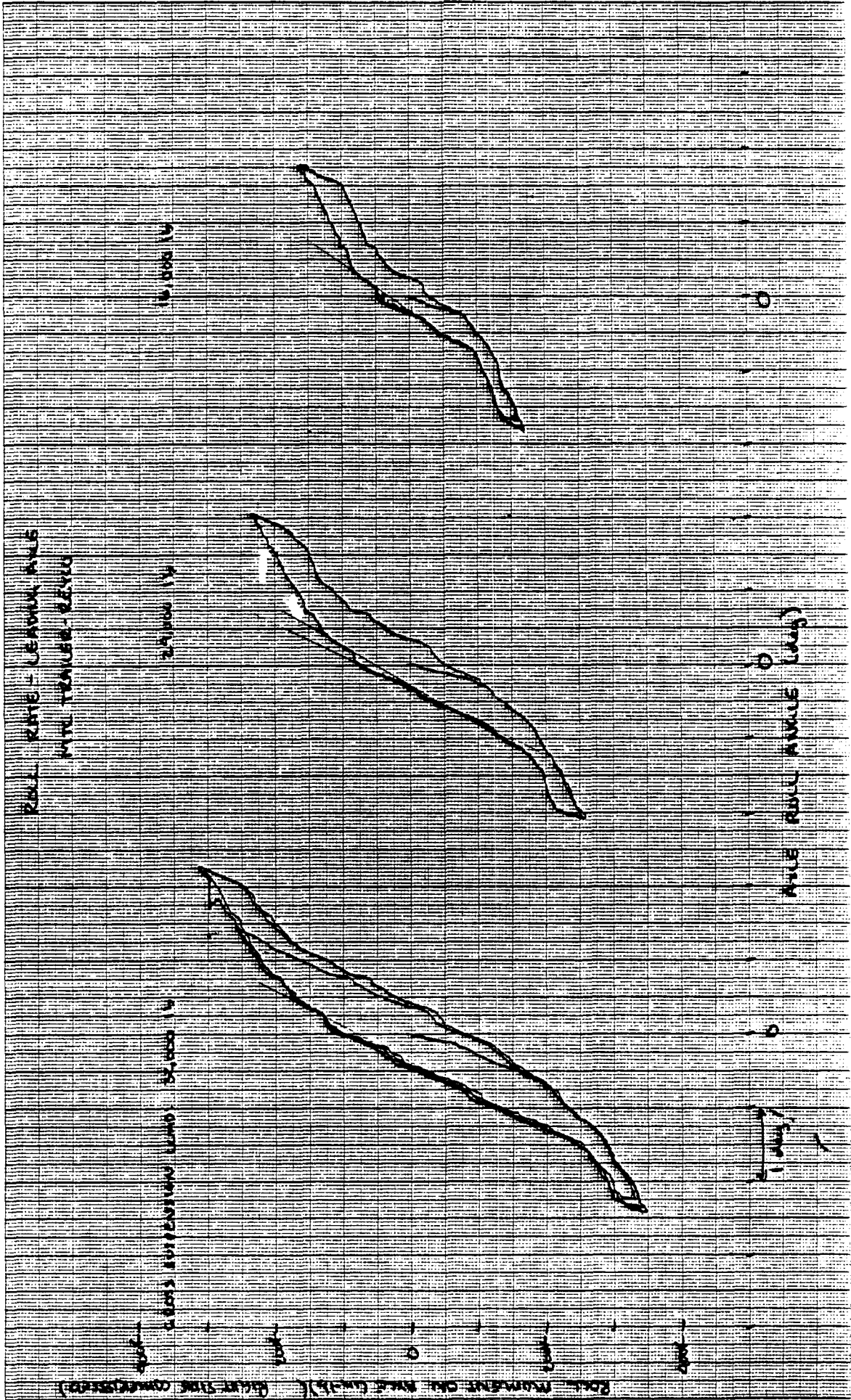


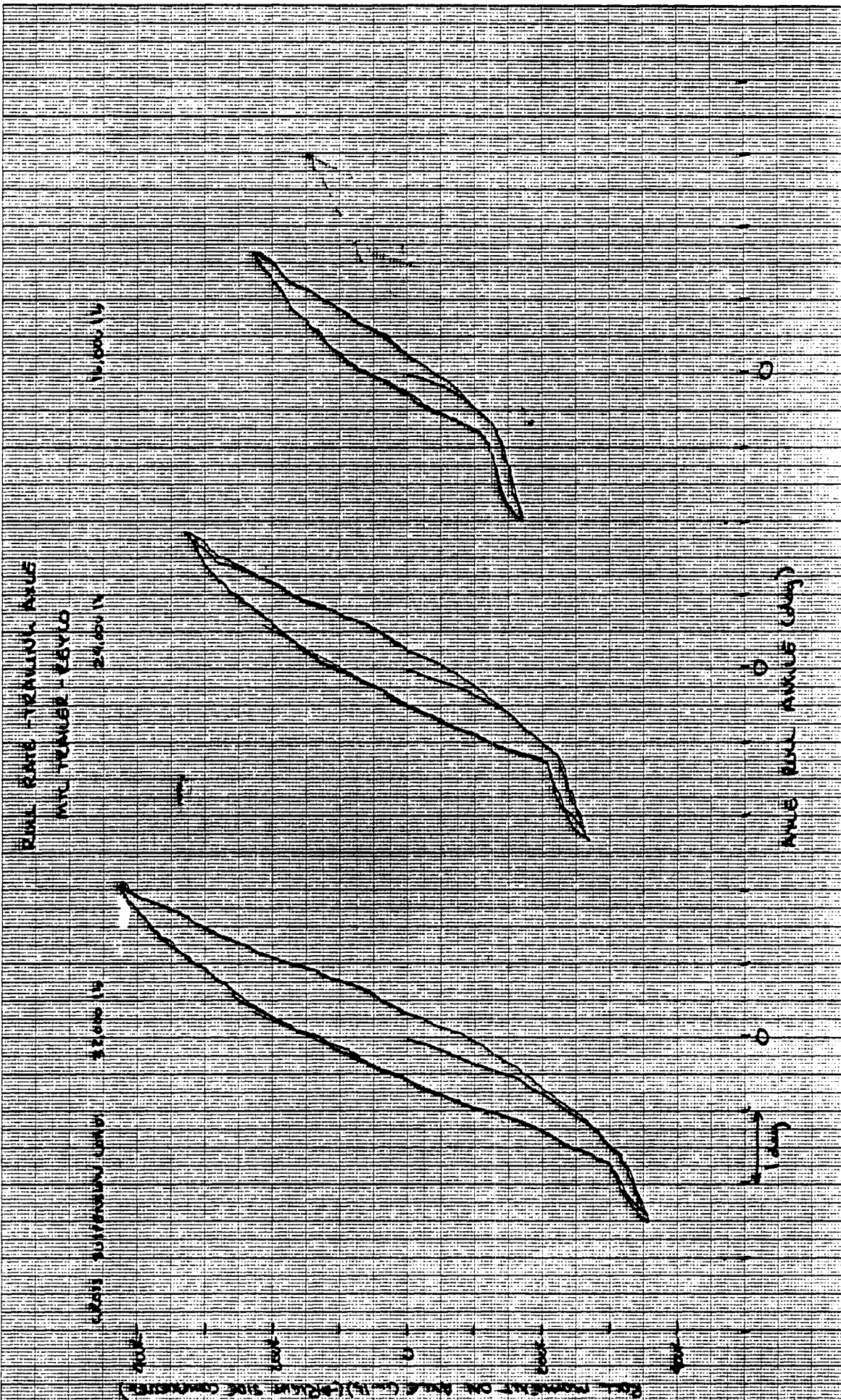
Suspension: Reyco 4 Spring - King Trailer

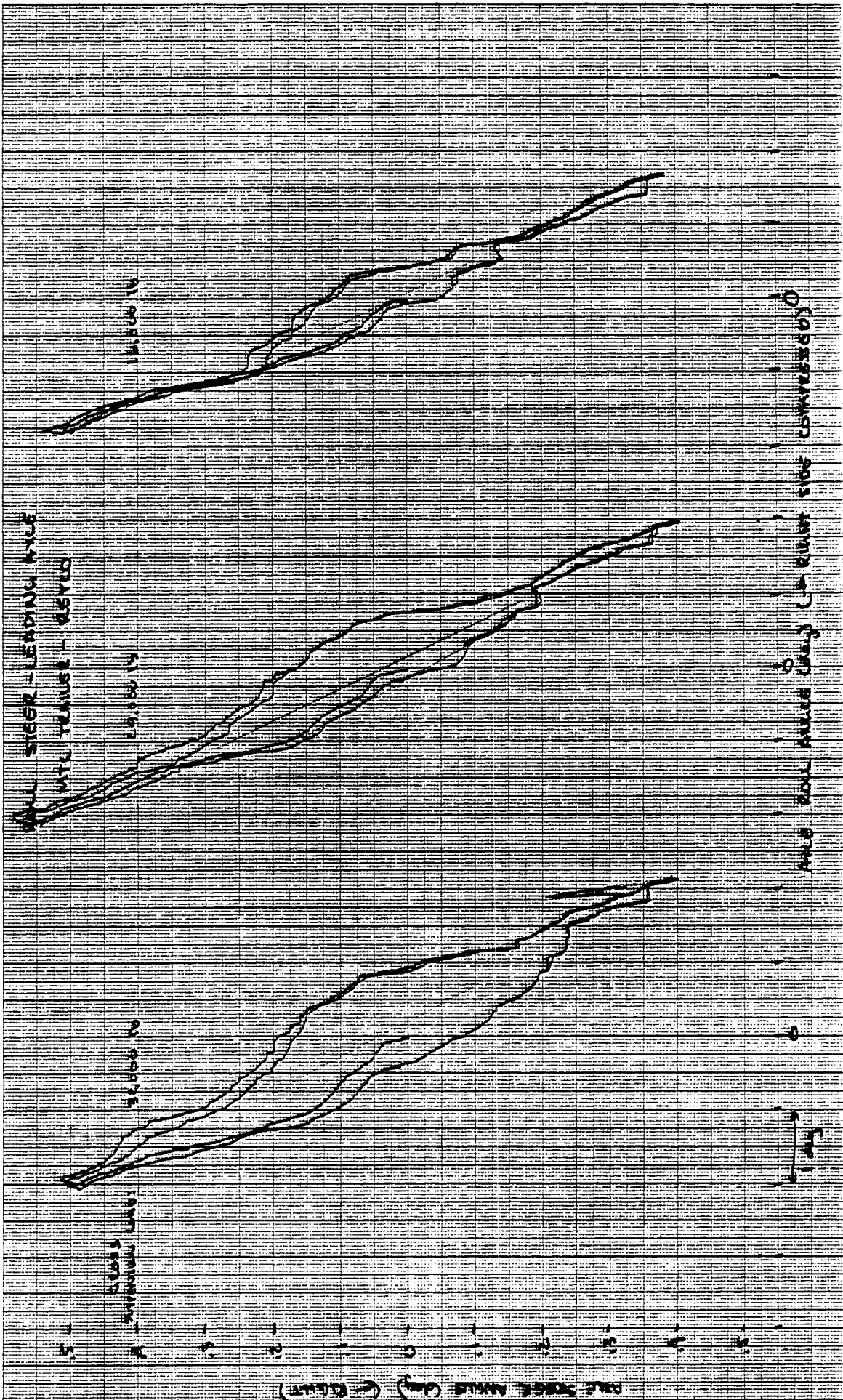
		<u>32000</u>	<u>24000</u>	<u>16000</u>
Susp Load				
Vertical Rate (lb/in)		11,800	10,800	10000
Coulumb Friction (lb)		1400	1250	850
Roll RATE	F	200K	180K	150K
(in./lb/deg)	R	250K	200K	165K
Roll CENTER HT	F	NA		
(in below top of frame)	R	NA		
Roll STEER	F	.21	.23	.23
(DEG / DEG)	R	.23	.23	.23
ALIGNING MOMENT COMPLIANCE STEER	F	$3 \times 10^{-6}$	$2.7 \times 10^{-6}$	$3.5 \times 10^{-6}$
	R	$5.5 \times 10^{-6}$	$6 \times 10^{-6}$	$6.5 \times 10^{-6}$
LATERAL FORCE COMPLIANCE	F	nil		
(deg/lb)	R	nil		
INTERAXLE LOAD TRANSFER (BYWIND)		NA		
(lb / lb)		NA		

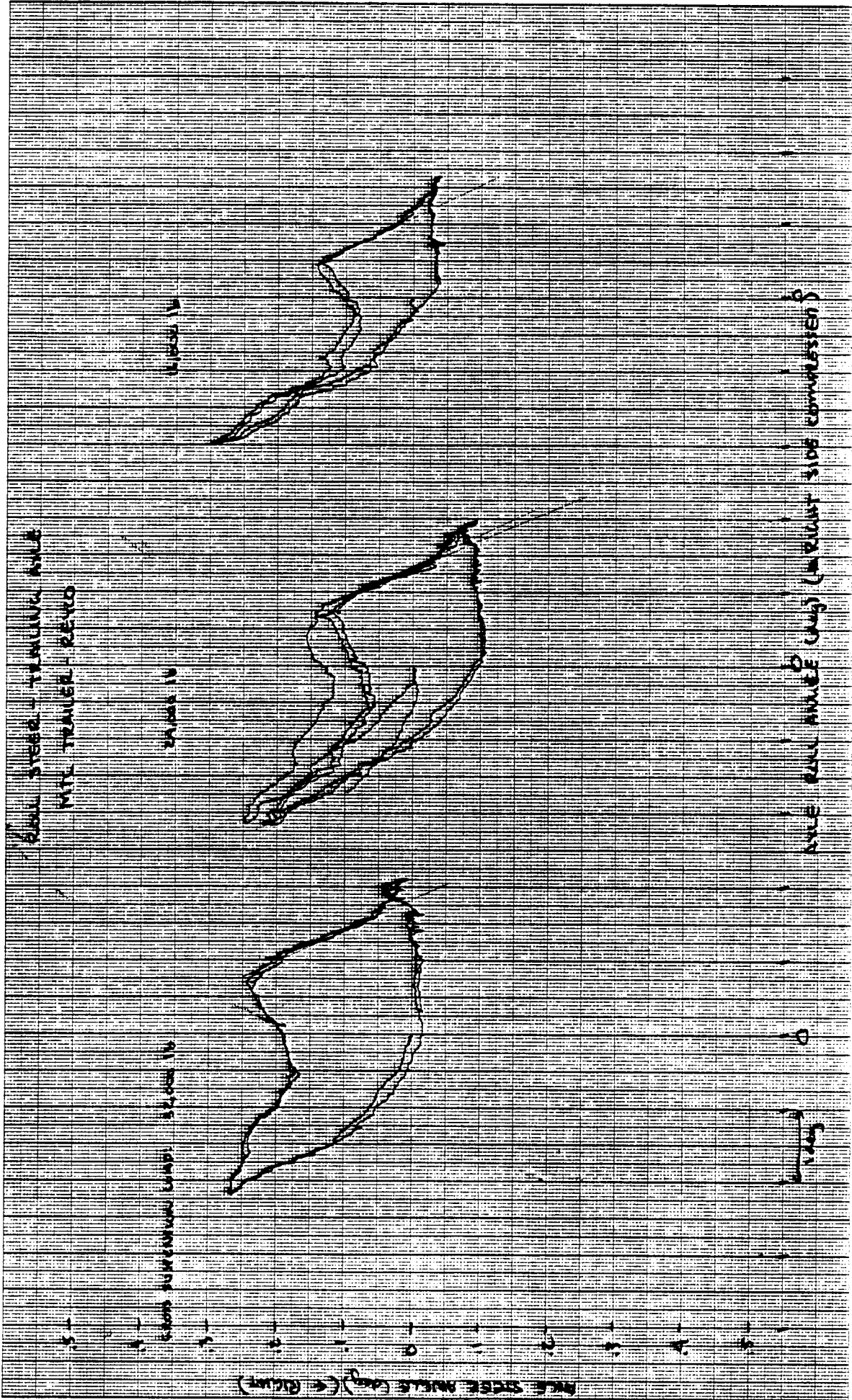




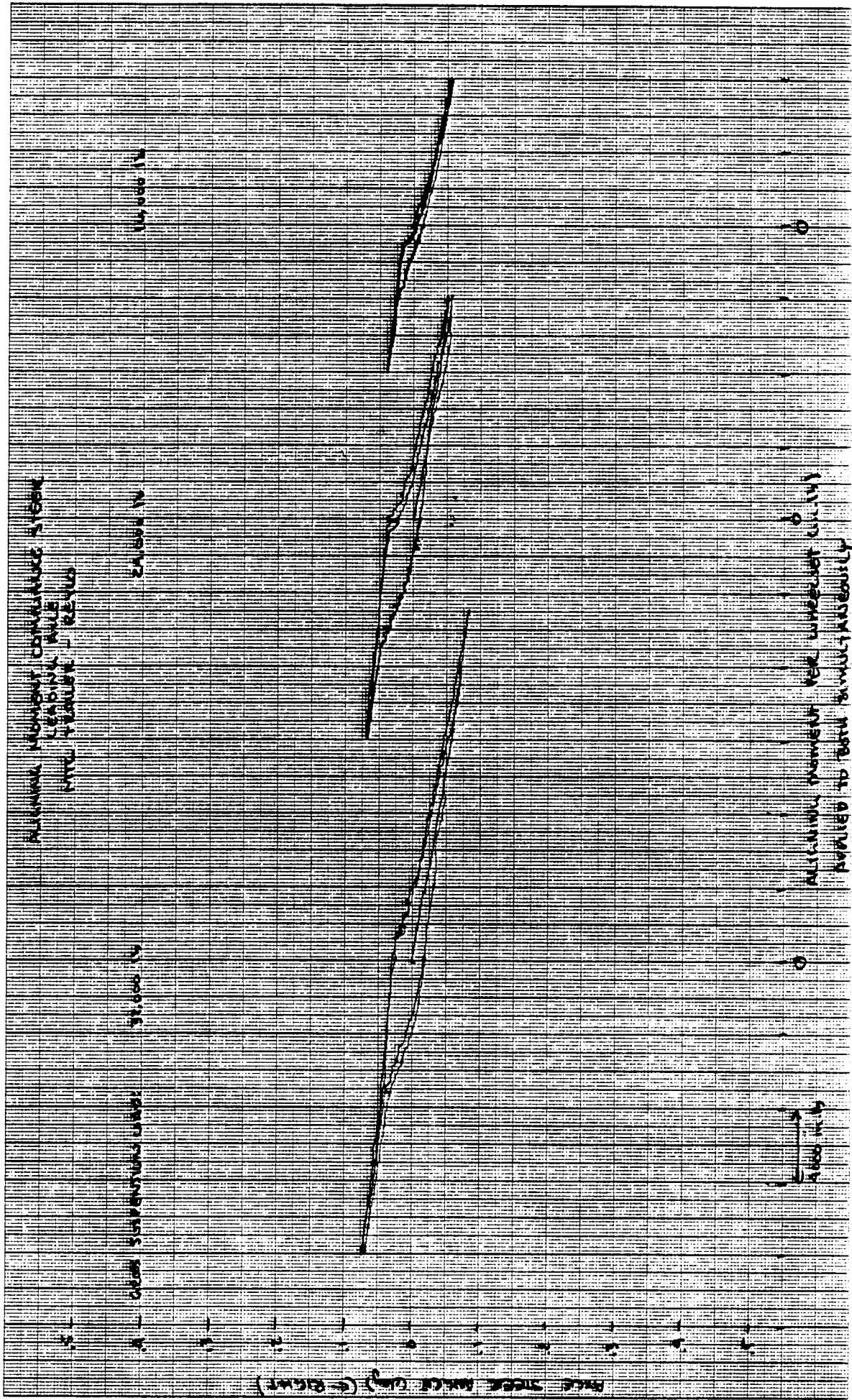


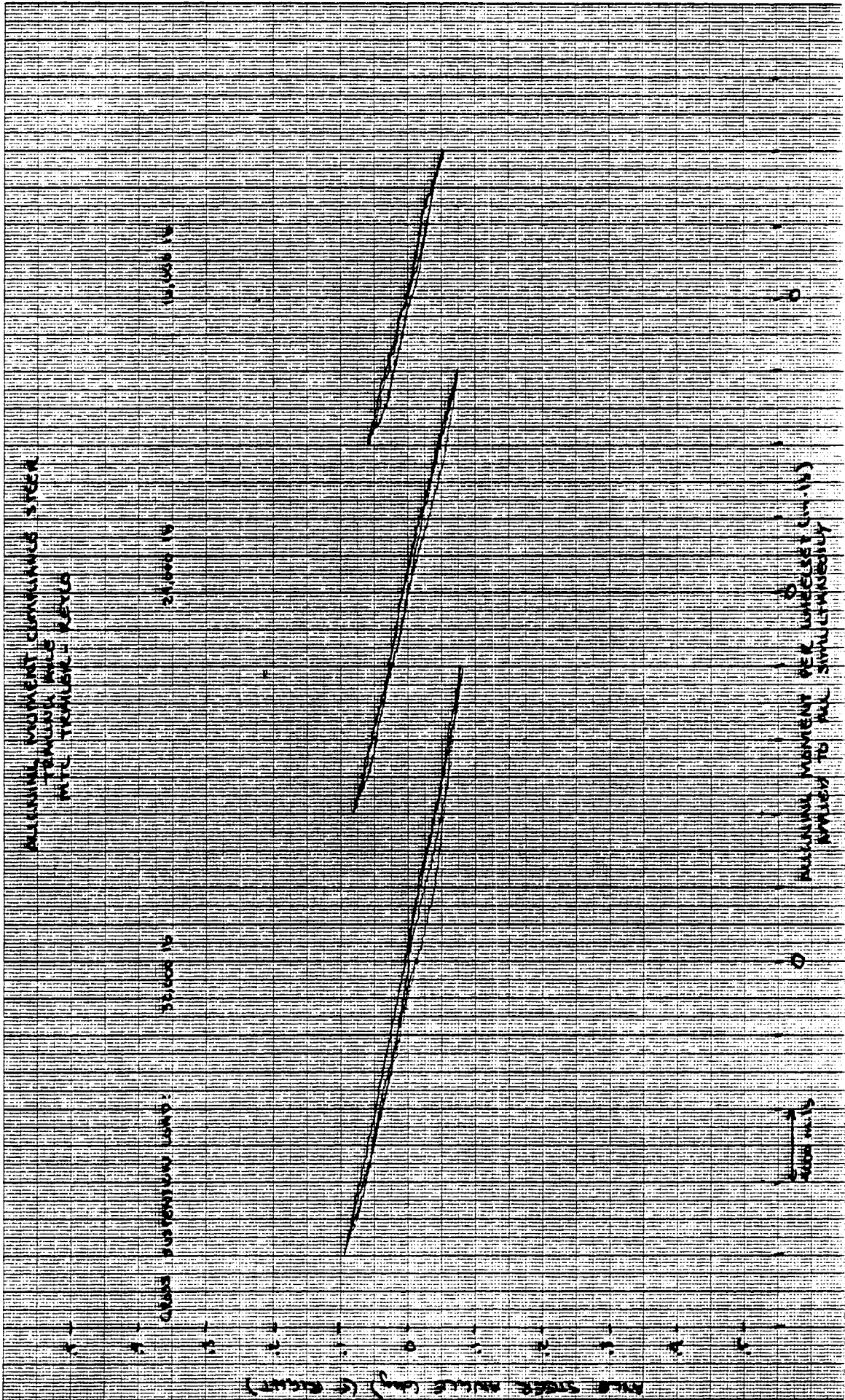




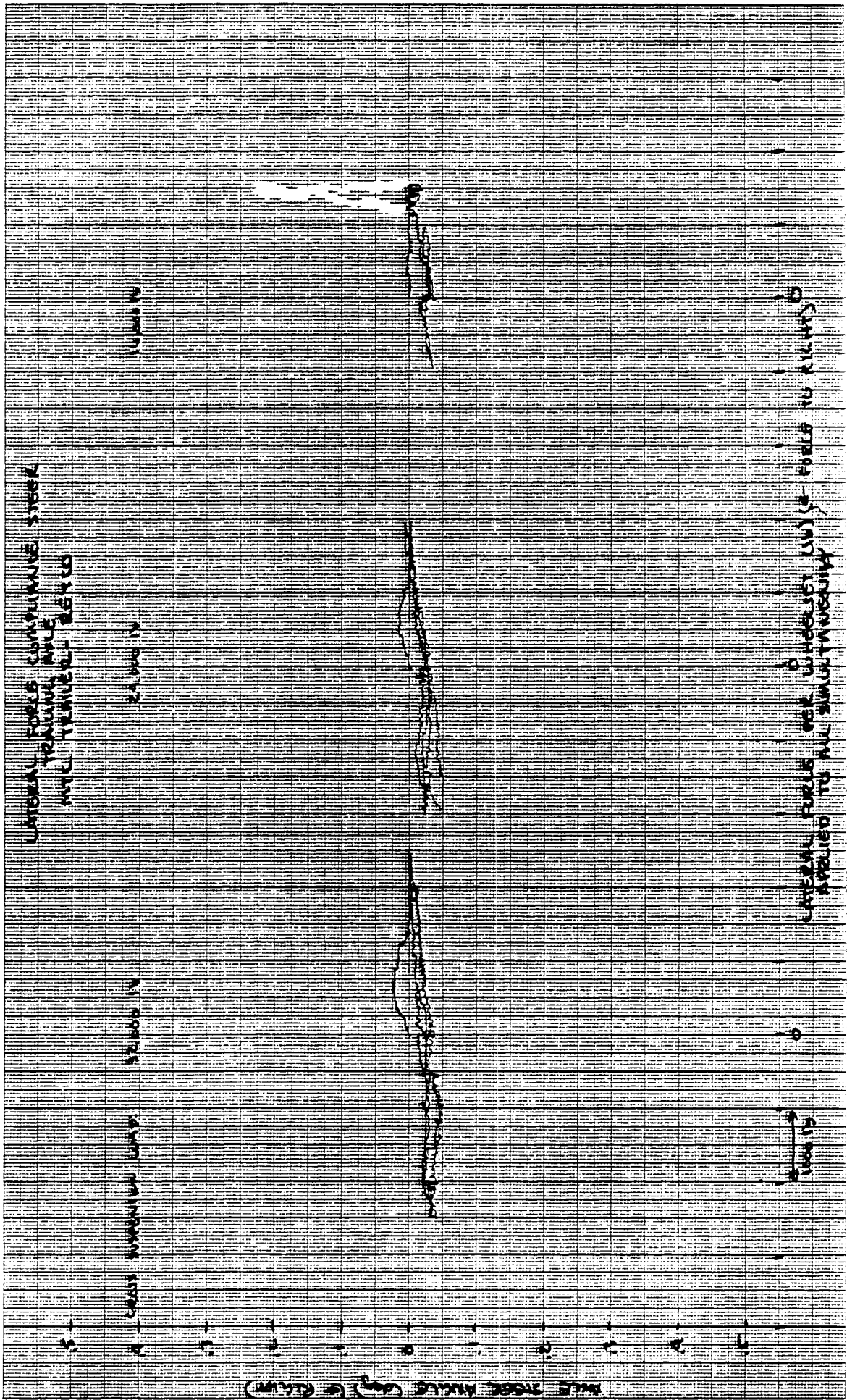








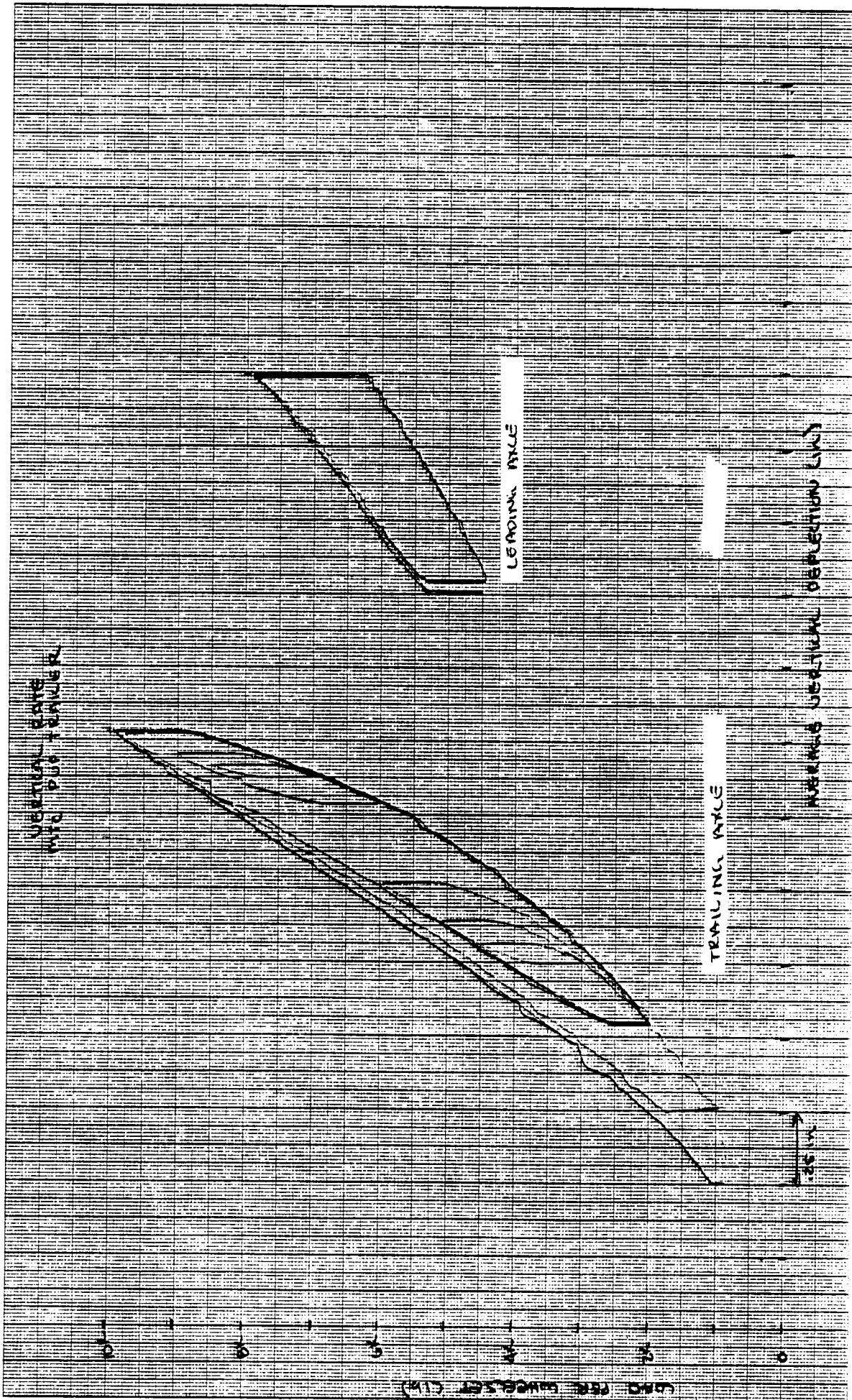


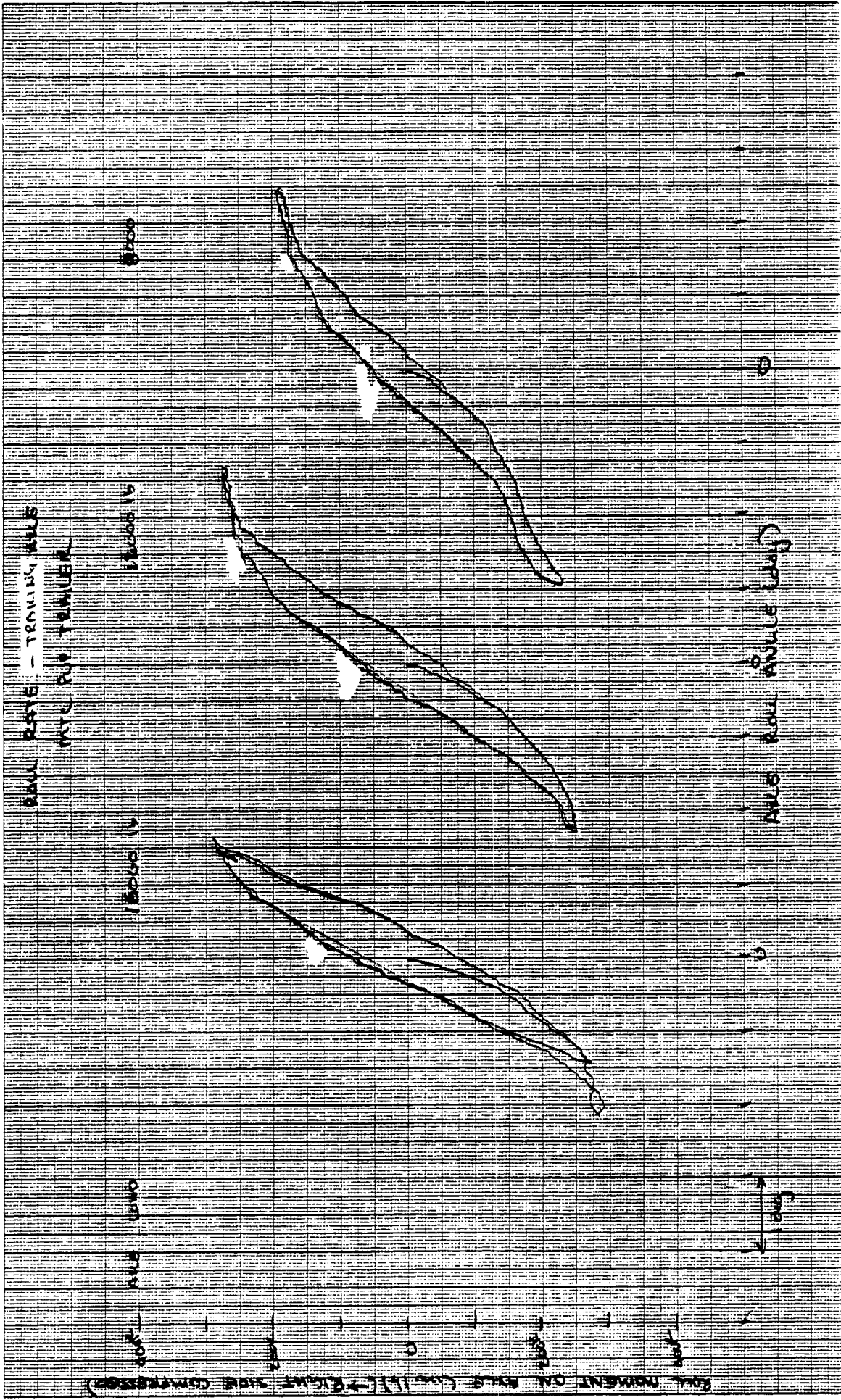


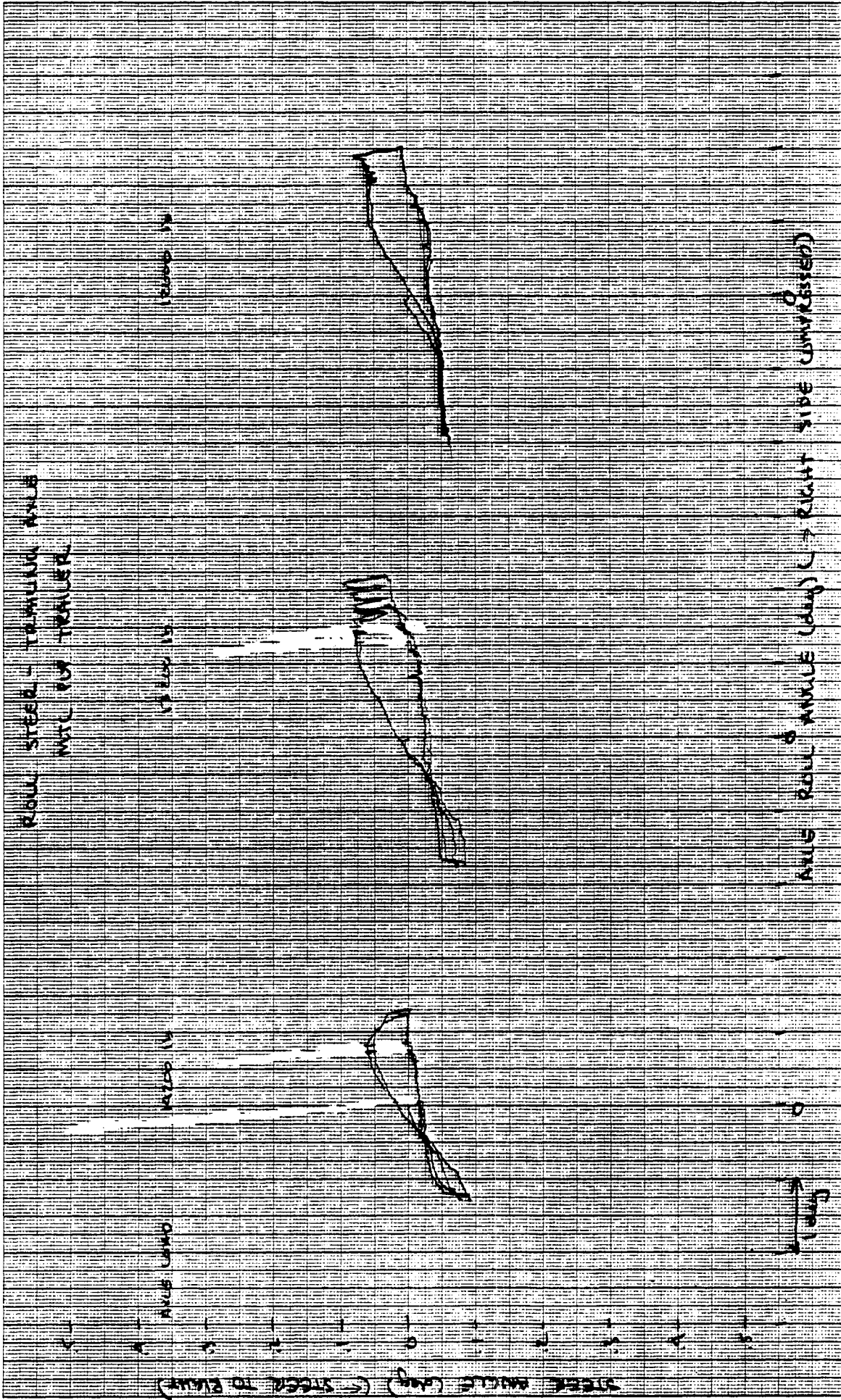


Suspension: Reyco Single - King Trailer

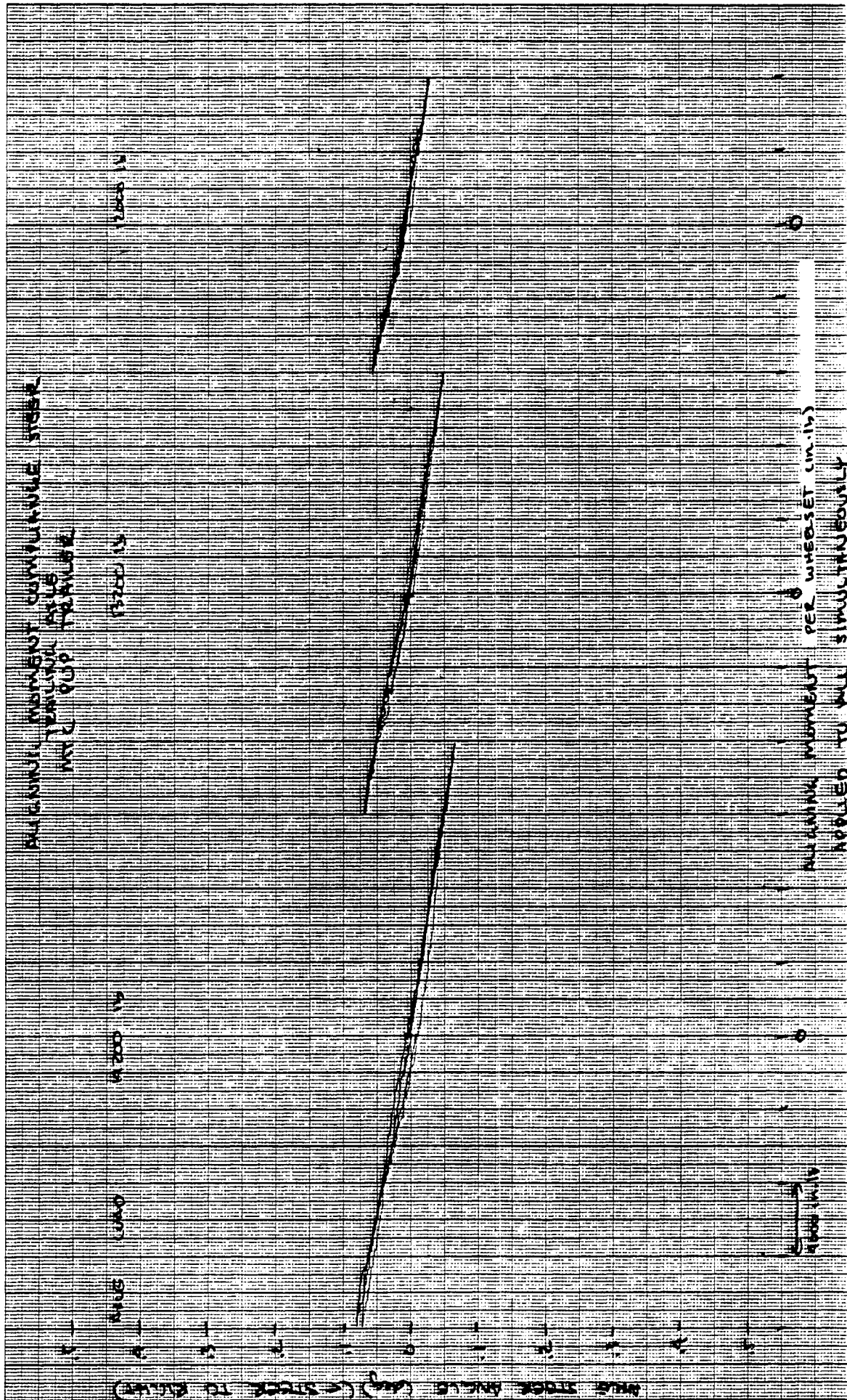
		Susp Load		
		<u>16000</u>	<u>12000</u>	<u>8000</u>
Vertical Rate (lb/in)		6800	6600	6000
Coulomb Friction (lb)		1250	1200	825
Roll RATE (in./lb/deg)	F	200K	160K	120
	R			
Roll CENTER HT (in below top of beam)	F	NA		
	R			
Roll STEER (deg/deg)	F	-0.24	-0.25	-0.26
	R			
ALIGNING MOMENT COMPLIANCE STEER (deg/in/lb)	F	$8 \times 10^{-6}$	$1 \times 10^{-5}$	$1 \times 10^{-5}$
	R			
LATERAL FORCE COMPLIANCE (deg/lb)	F	$6 \times 10^{-5}$	$8 \times 10^{-5}$	$1.2 \times 10^{-4}$
	R			
INTERAXLE LOAD TRANSFER (BYWIND) (lb/lb)				

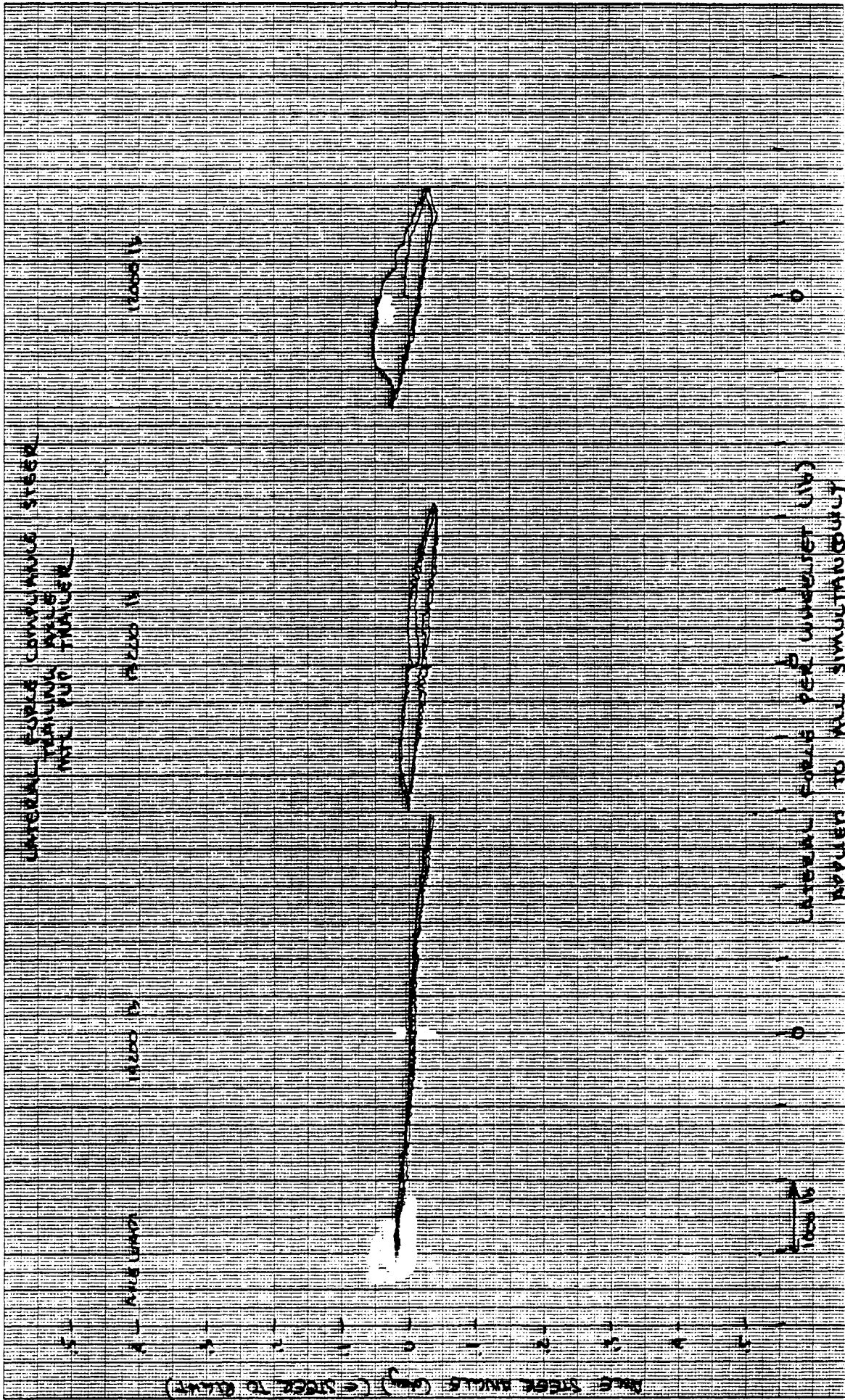










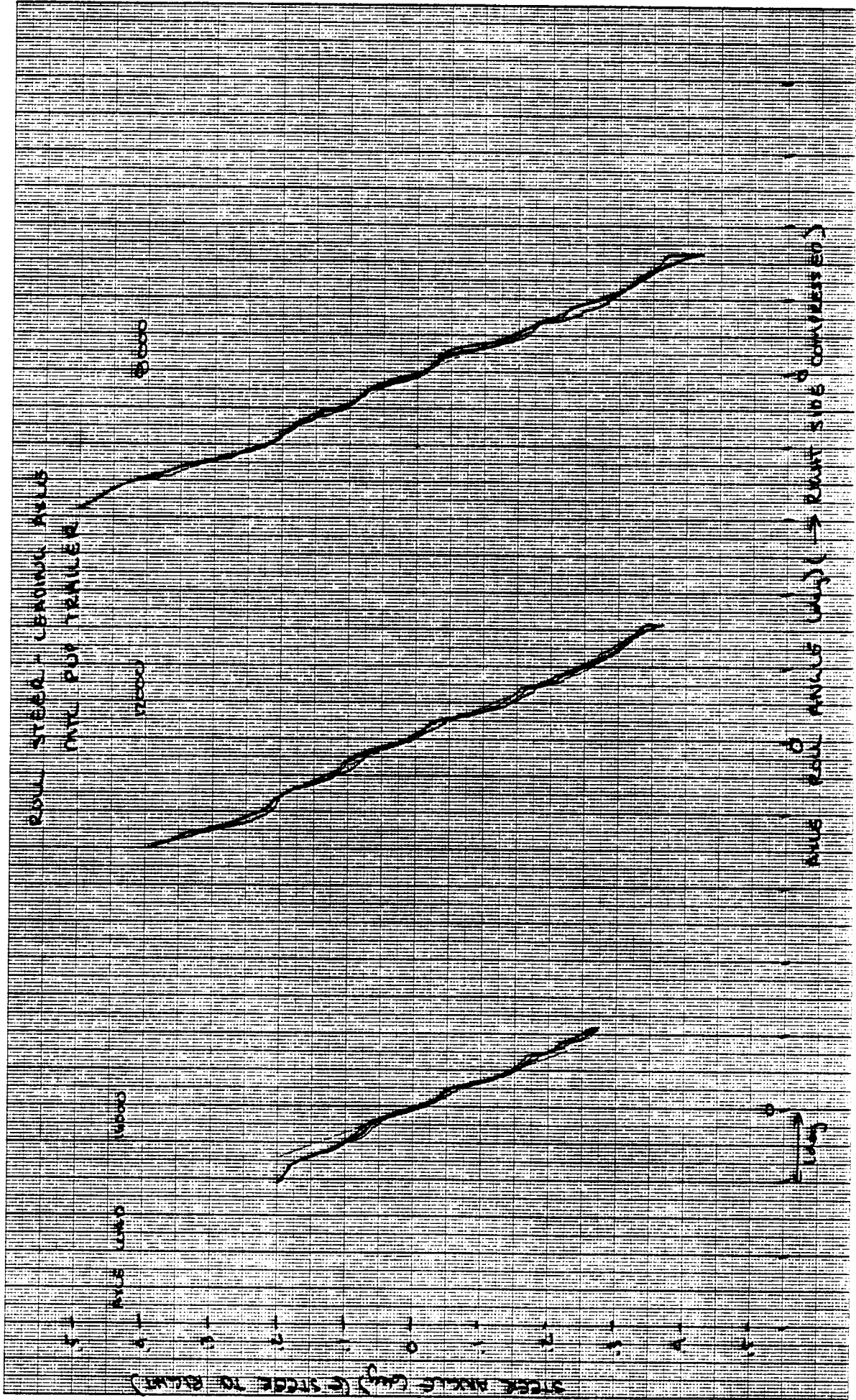


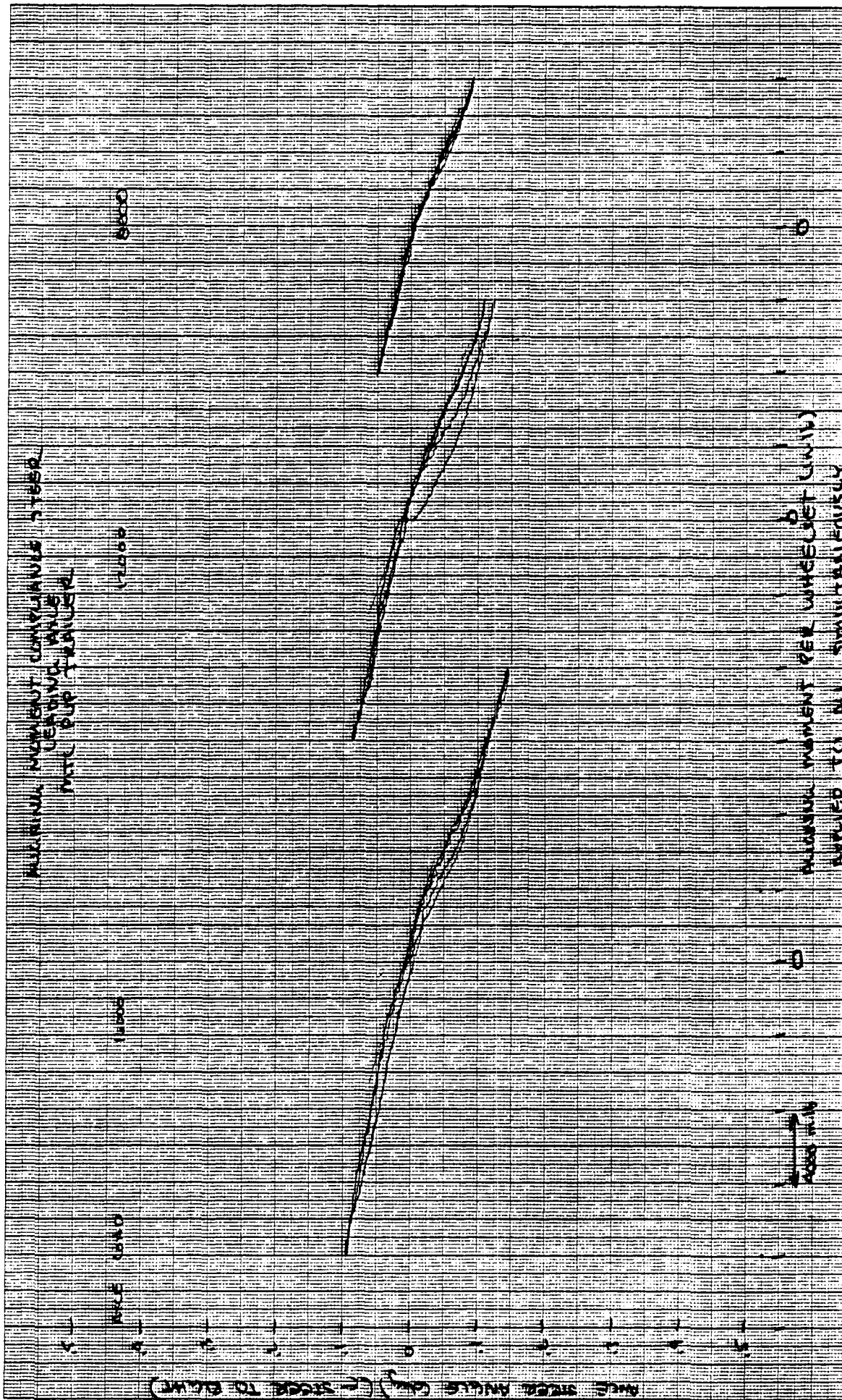
Suspension: Air Lift - King Trailer

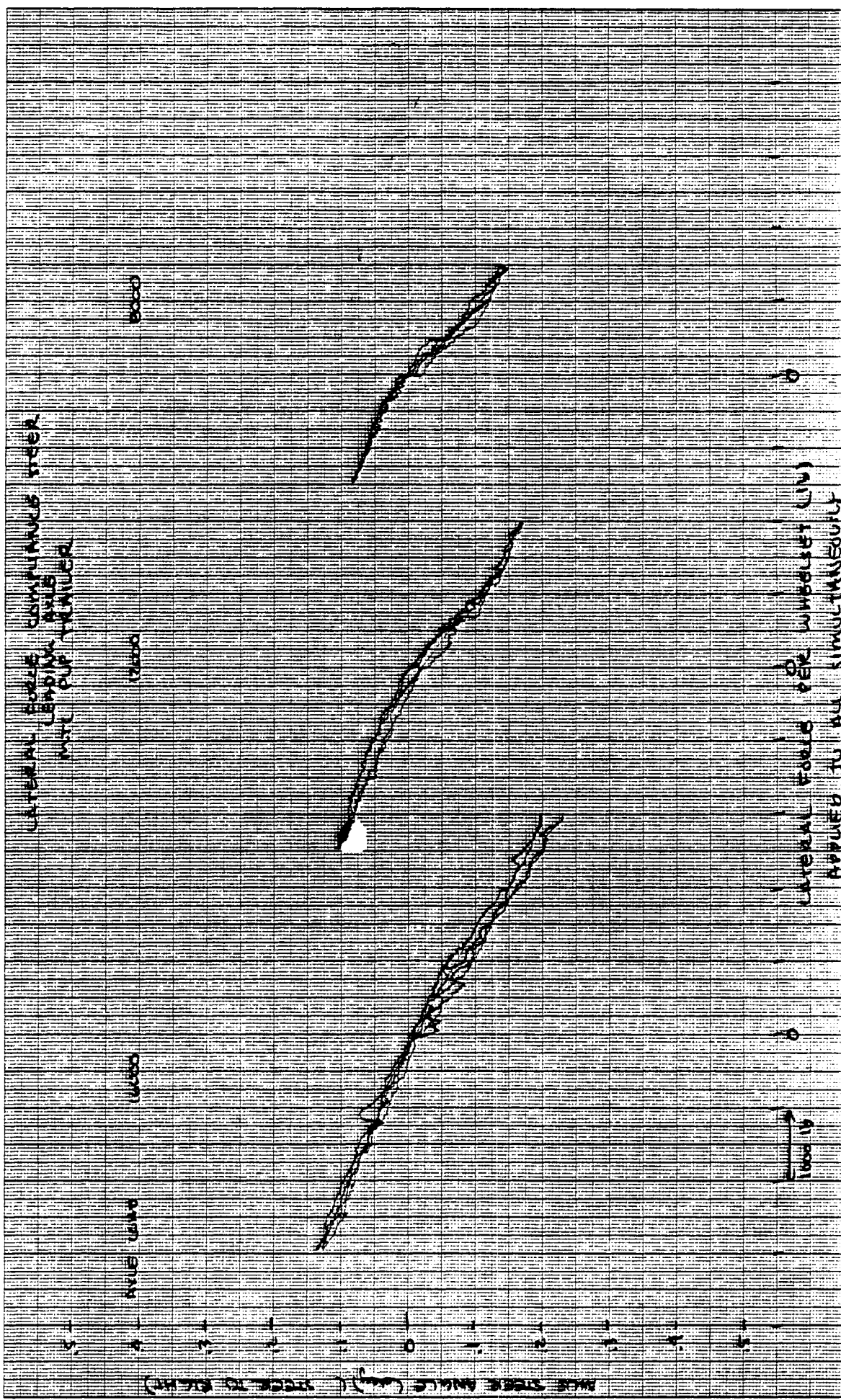
			Susp Load		
			14200	13200	12000
Vertical Rate (lb/in)			3400		
Roll Rate (in./lb/deg)	F	150K	135K	130K	
	R				
Roll Center HT (in below top of frame)	F		NA		
	R				
Roll Steer (deg/deg)	F	.02	.02	.02	
	R				
Aligning Moment Compliance Steer (deg/in lb)	F	$4 \times 10^{-6}$	$5 \times 10^{-6}$	$5 \times 10^{-6}$	
	R				
Lateral Force Compliance (deg/lb)	F	$5 \times 10^{-5}$	$1 \times 10^{-4}$	$2.5 \times 10^{-4}$	
	R				
Interaxle Load Transfer (GRAIN)					
(lb/lb)					











LATERAL FORCE COMPLIANCE CURVE  
WITH LEMMA'S  
MIL CUP TAILER

LATERAL FORCE PER WIMBLET (lb)  
APPLIED TO ALL SIMULTANEOUSLY

LATERAL FORCE ANGLE (deg) (POOR TO BEST)

41.0001

Suspension: Neway Air Suspension - Trailer

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
Vertical Rate (lb/in)		1900	1400	1000
Coulumb Friction (lb)		400	425	400
Roll RATE (in/lb/deg)	F	80K	75K	65K
	R	95K	75K	70K

Roll CENTER HT (in below top of frame)	F		24.1	
	R		25.0	

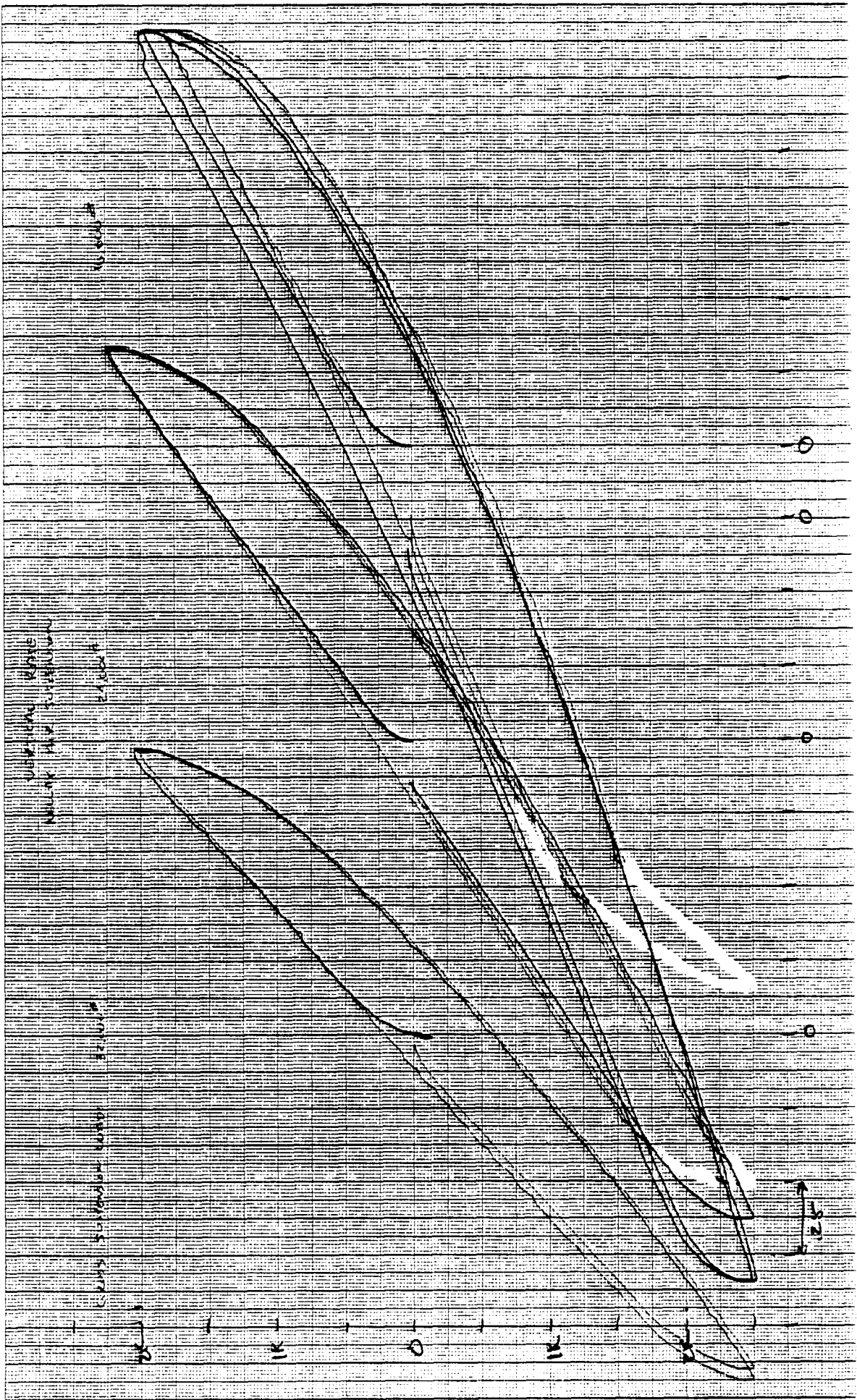
Roll STEER (DEG / DEG)	F		NA	
	R			

ALIGNING MOMENT COMPLIANCE STEER (deg/in/lb)	F		NA	
	R			

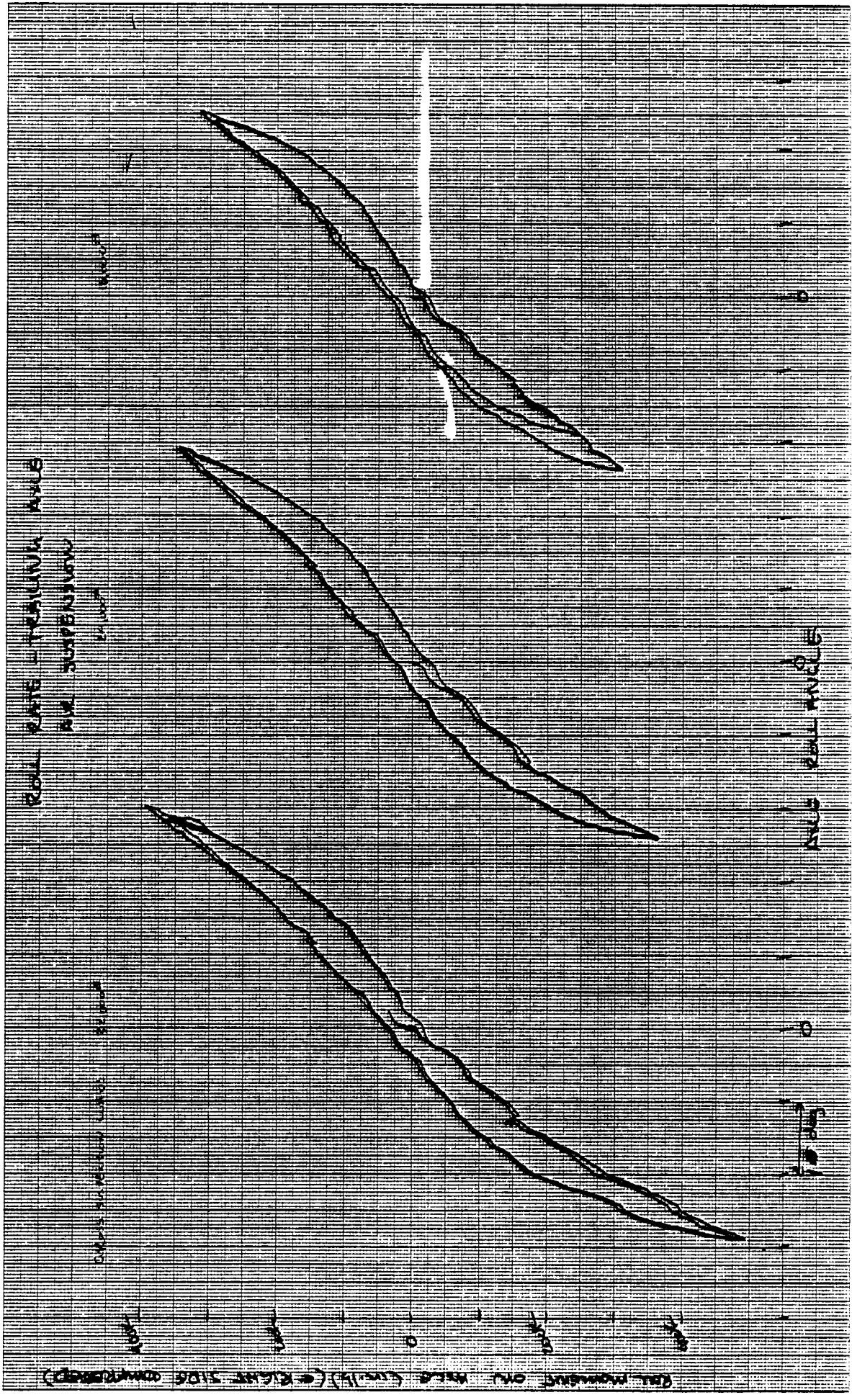
LATERAL FORCE COMPLIANCE (deg/lb)	F		NA	
	R			

INTERAXLE LOAD TRANSFER (BRANK) (lb/lb)			NA	
-----------------------------------------	--	--	----	--







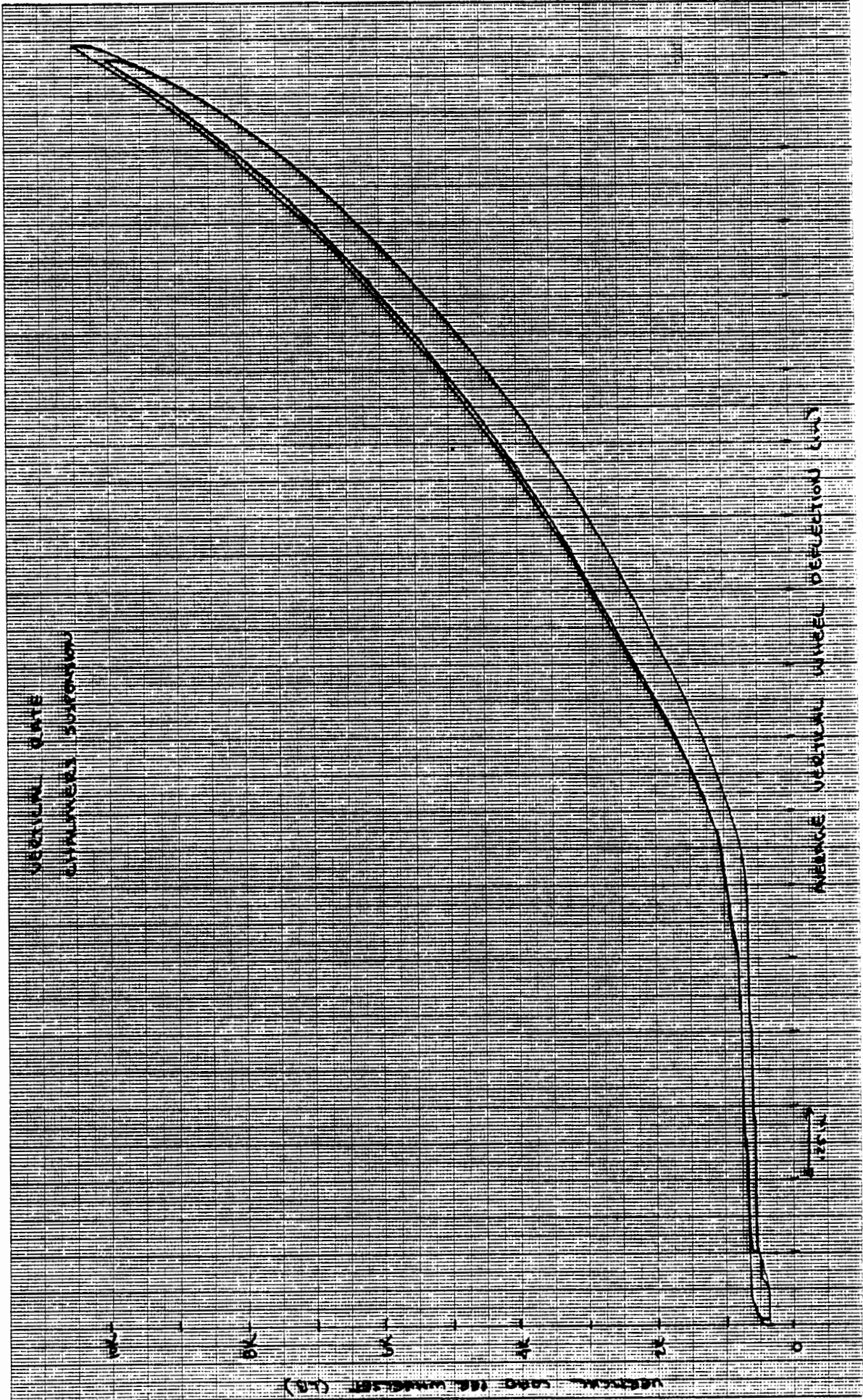


ROLL RATE - TORSION AXIS (Right Side compressed)

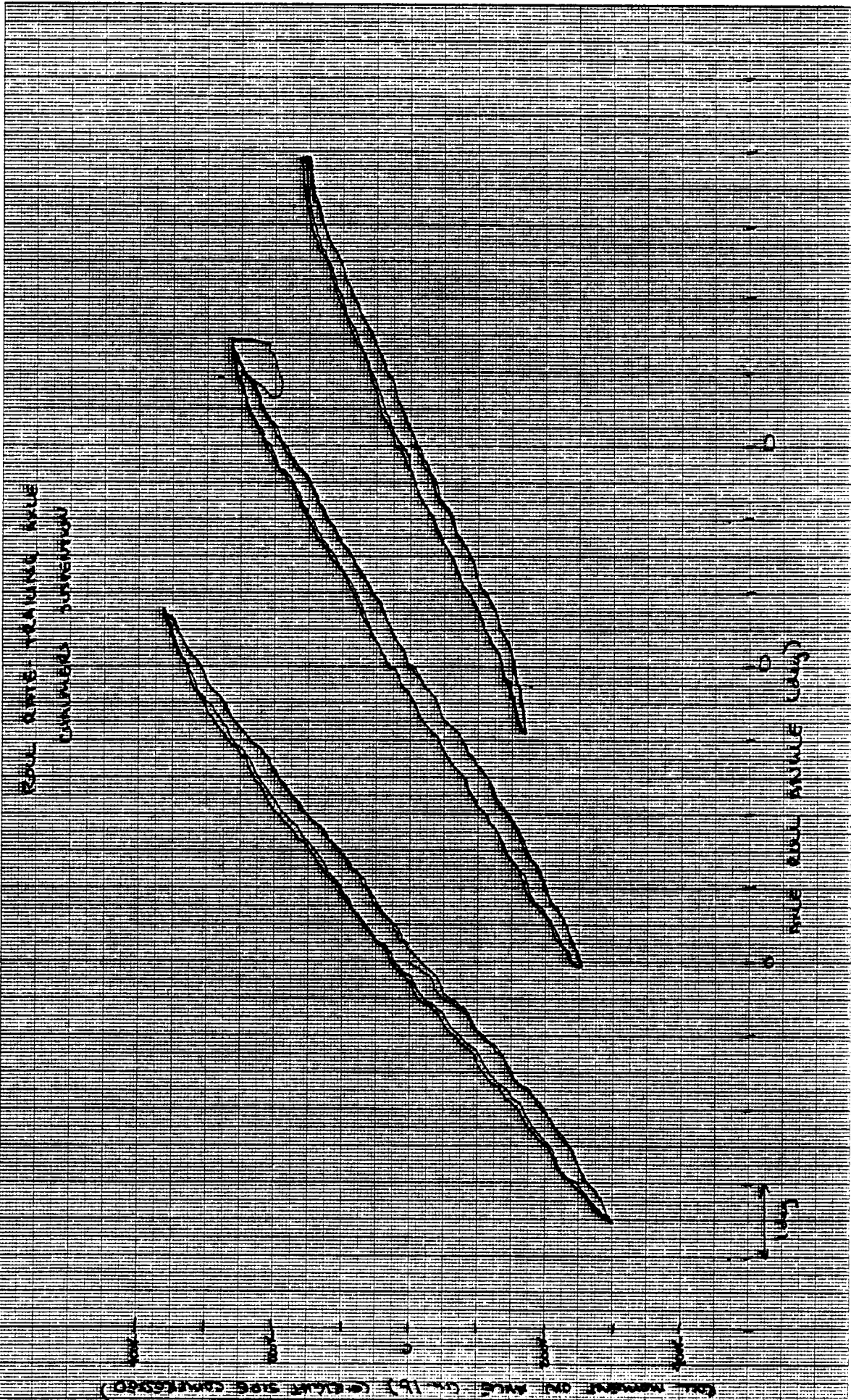
Suspension: Chalmers Rubber Block Walking Beam

		Susp Load		
		<u>32000</u>	<u>24000</u>	<u>16000</u>
<del>W</del>				
Vertical Rate (lb/in)		5200	3800	3000
Coulomb Friction (lb)		375	325	300
Roll RATE	F	92K	67K	52K
(in/lb/deg)	R	90K	65K	47K
<hr/>				
Roll CENTER HT	F	14.9	16.1	16.4
(in below top of frame)	R	15.4	15.9	16.5
<hr/>				
Roll STEER	F			
(DEG / DEG)	R			
<hr/>				
ALIGNING MOMENT COMPLIANCE STEER F				
(deg/in/lb)	R			
<hr/>				
LATERAL FORCE COMPLIANCE F				
(deg/lb)	R			
<hr/>				
INTERAXLE LOAD TRANSFER (BRIDGE)				
(lb/lb)				

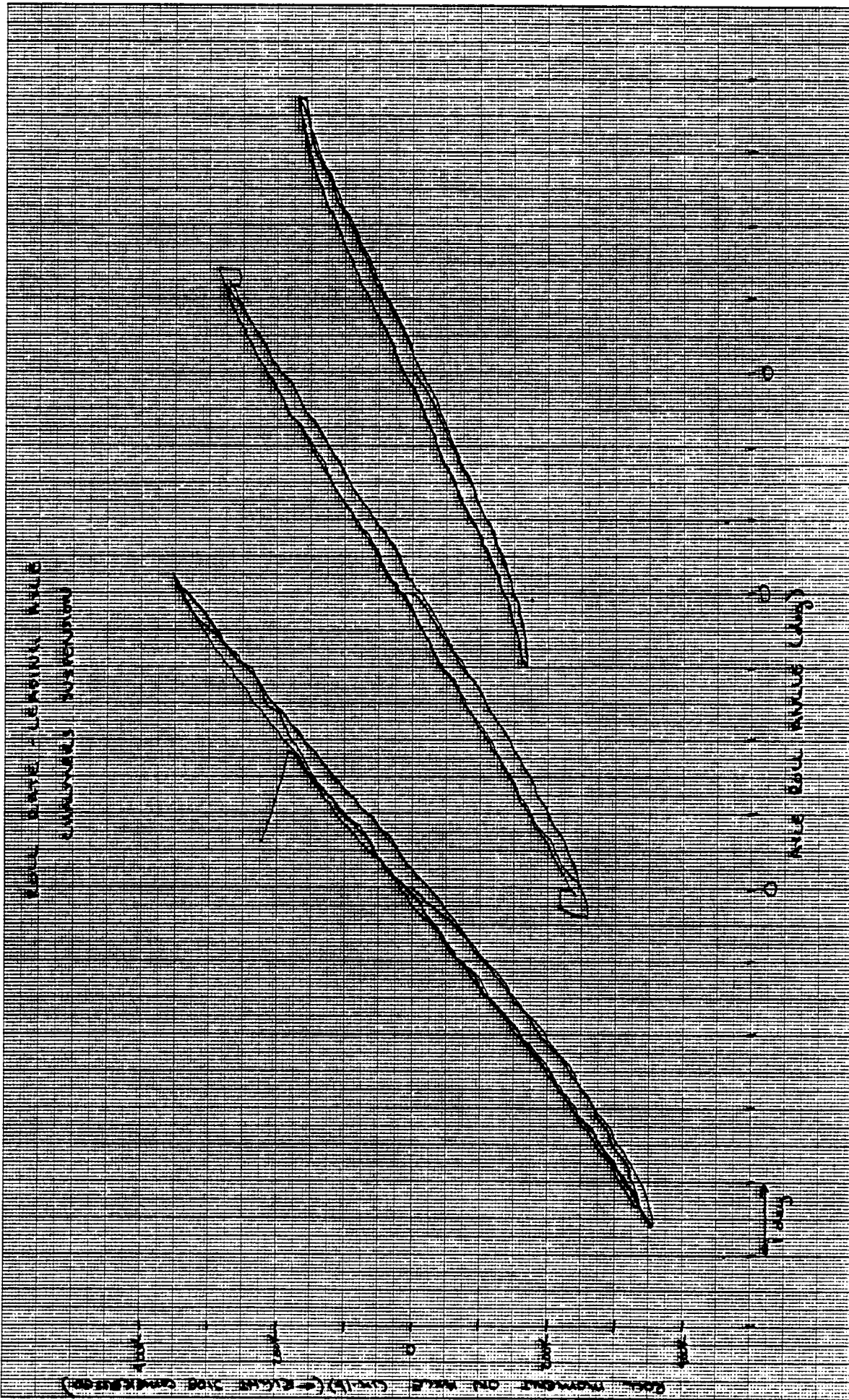




ROLL RATE, YAWING RATE  
CHARACTERS SUPERIMPOSED



ROLL RATE ON AXIS (L) (SCALE 950 CM/SEC)



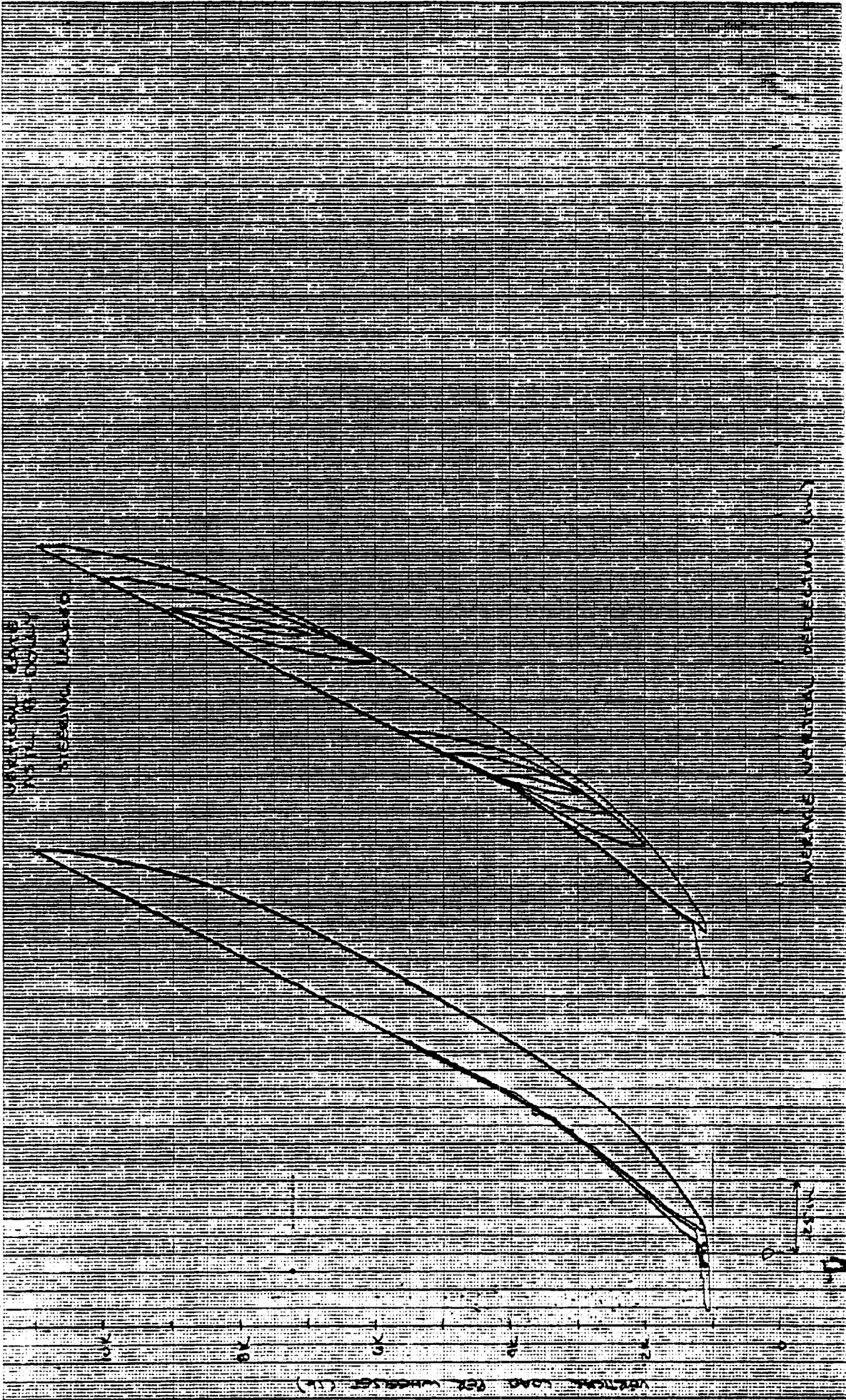
BALL RATE (PER MIN) AXLE ANGLE  
 AXLE BALL ANGLE (deg)





Suspension:

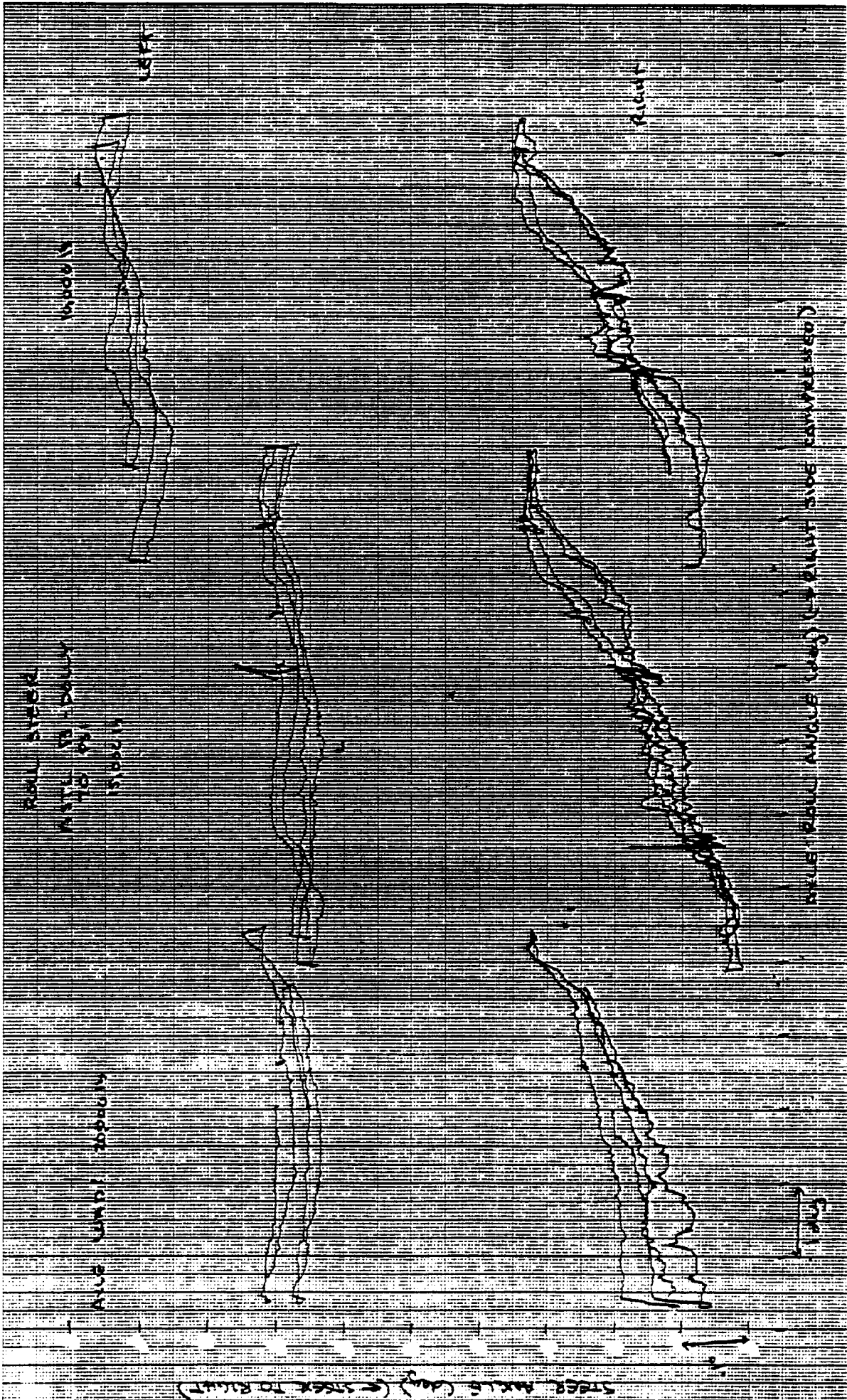
		Susp Load		
		<u>20000</u>	<u>15000</u>	<u>10000</u>
Vertical Rate (lb/in)		8000	8000	8000
Roll RATE (in./lb/deg)	F R	105K	87.5K	60K
Roll CENTER HT (in below top of frame)	F R	13.4	14.0	14.0
Roll STEER (deg/deg)	L R	_____	nil	_____
		.09	.05	.08
ALIGNING MOMENT COMPLIANCE STEER (deg/in lb)	F R	_____	nil	_____
LATERAL FORCE COMPLIANCE (deg/lb)	F R	_____	nil	_____
INTERAXLE LOAD TRANSFER (STEER) (lb/lb)				











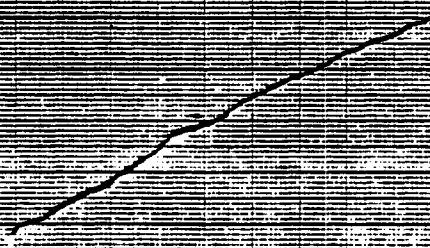
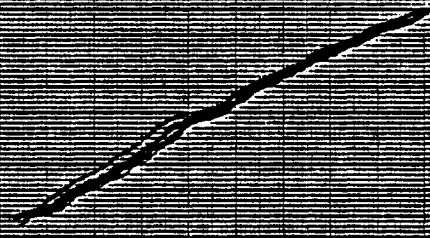
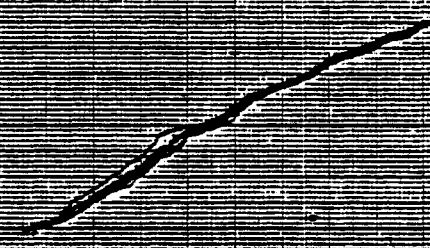
ROLL CENTER  
ASTC BUBBLE

15.0000

15.0000

15.0000

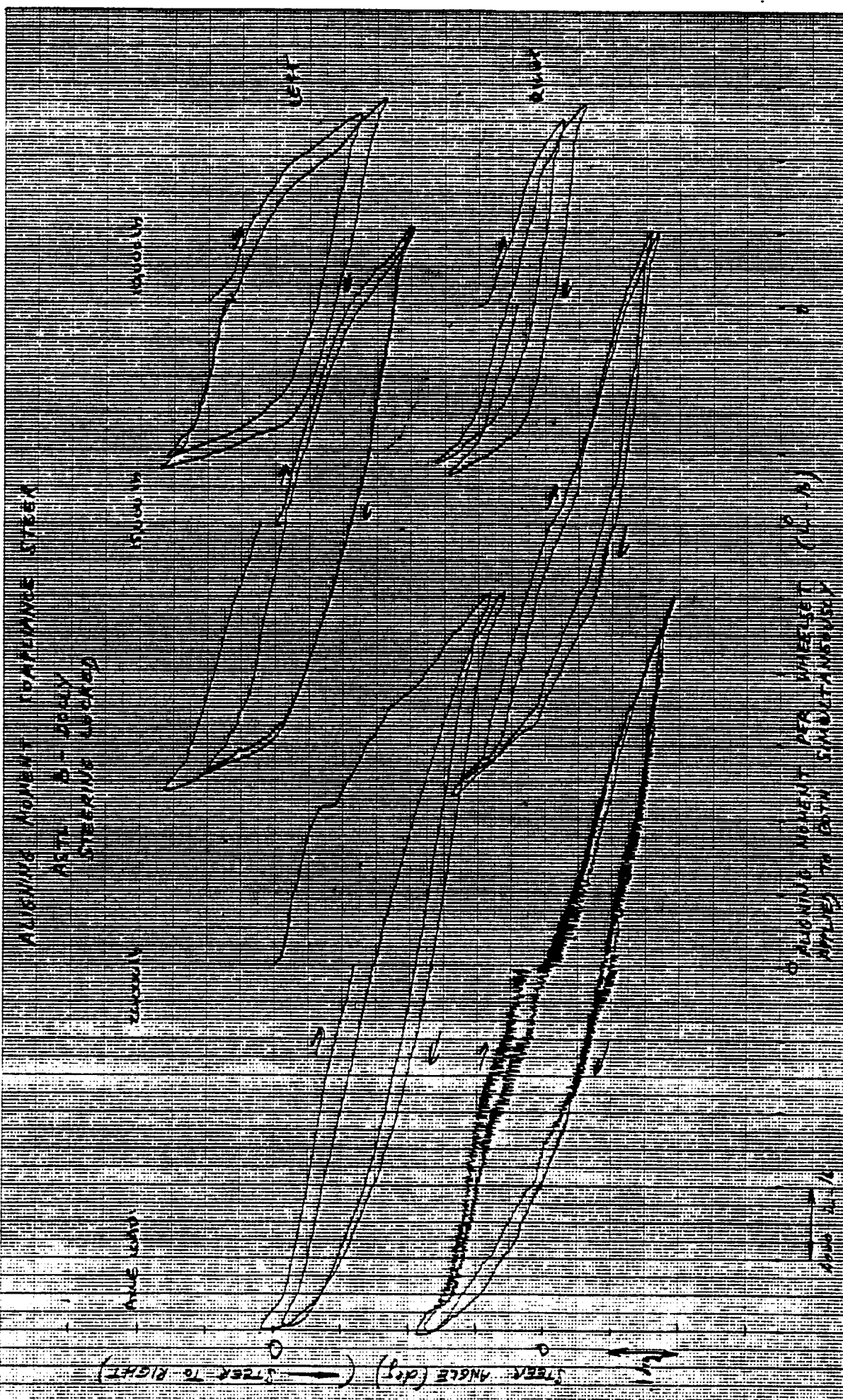
15.0000



ROLL ANGLE (DEG)

(← RIGHT SIDE COMPRESSED)

LATERAL DEFLECTION (CM) 2.5g BELOW TILT TO FORWARD (←RIGHT)









STEEL PARTICLE (40%) (← STEEL TO RIGHT)

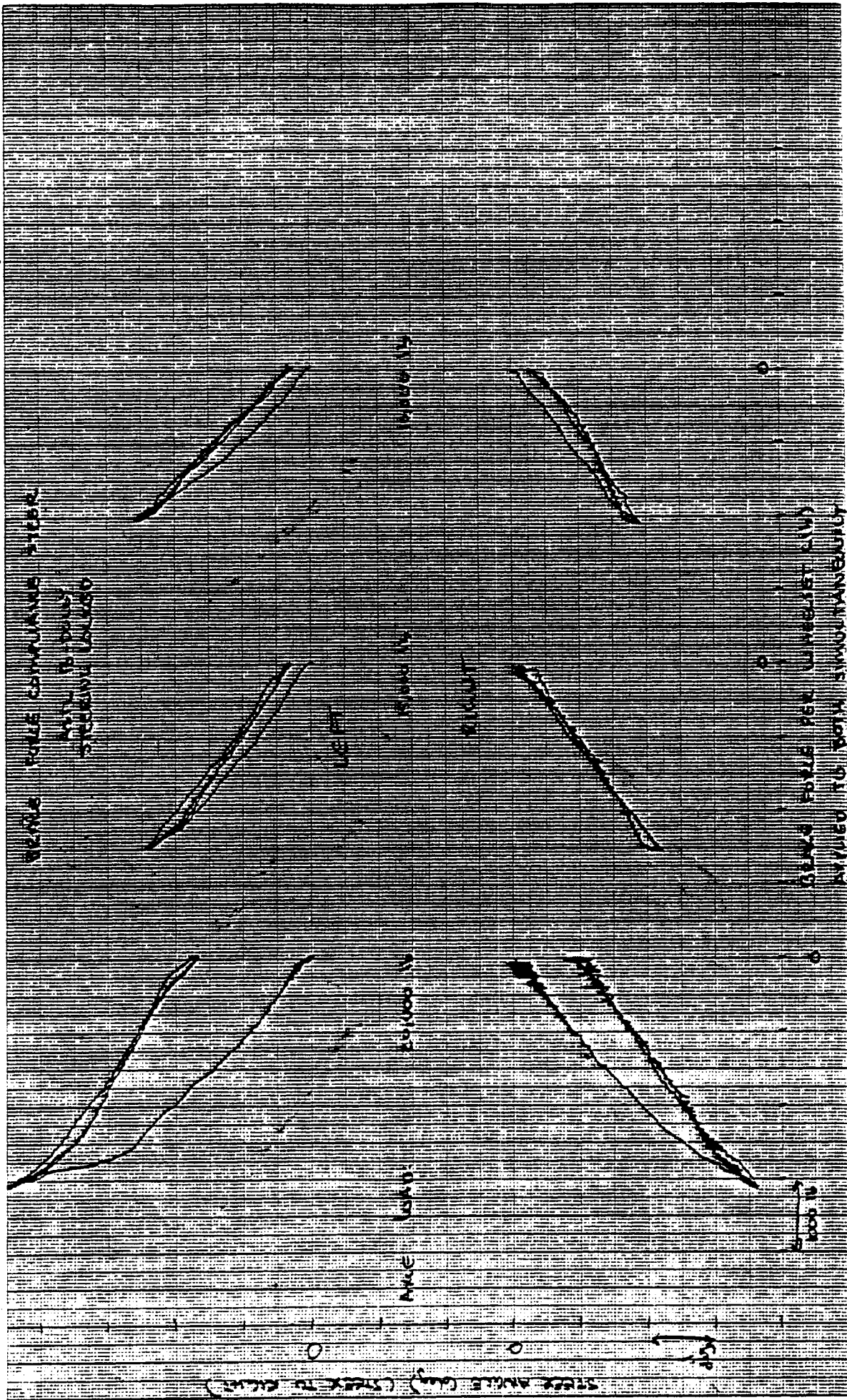


LATERAL FORCE COMPLIANCE STEEL  
WITH PROBABLY  
SOME PLASTICITY

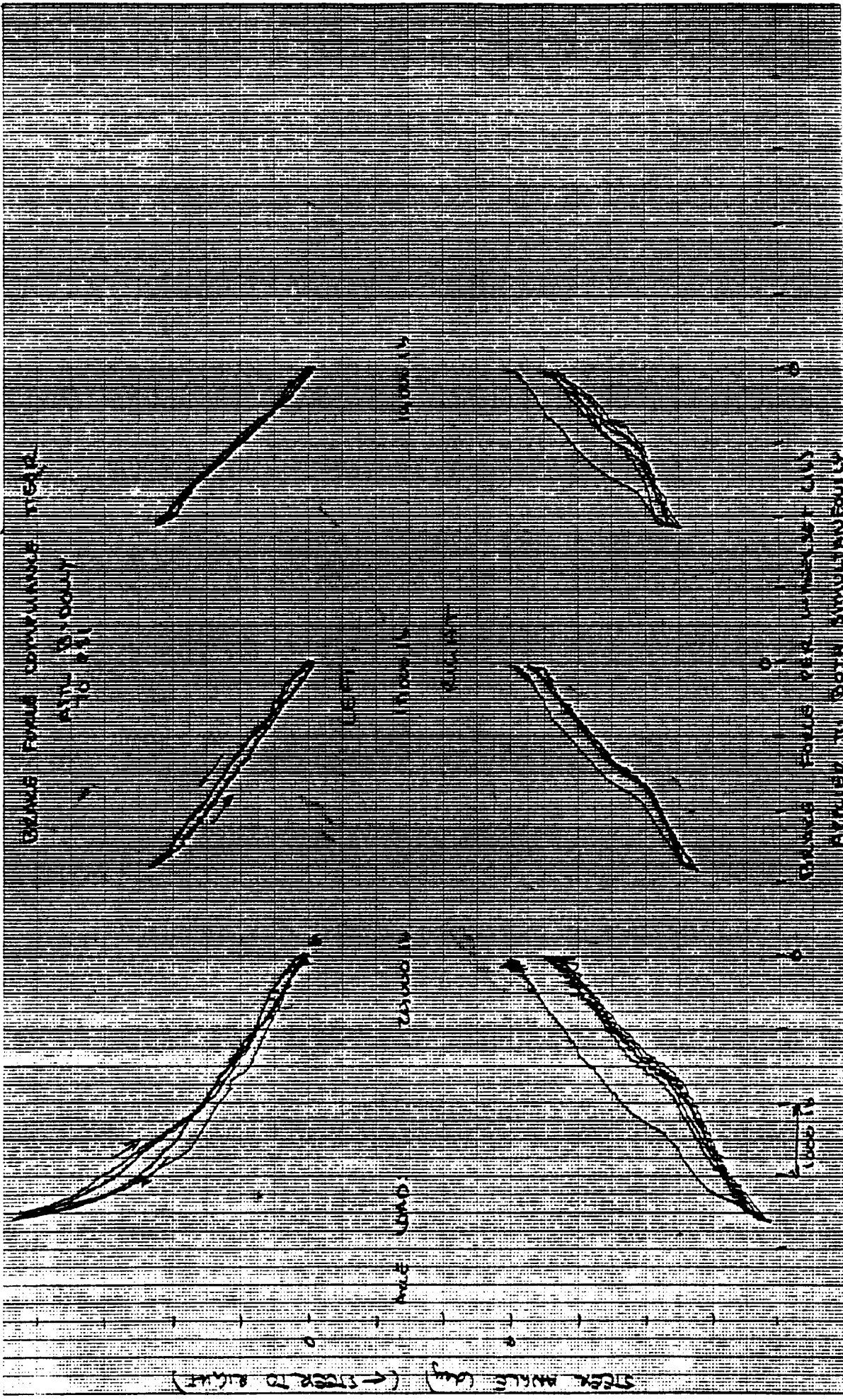
LATERAL FORCE PER UNIT SET (15)  
APPLIED TO AREA SIMULTANEOUSLY

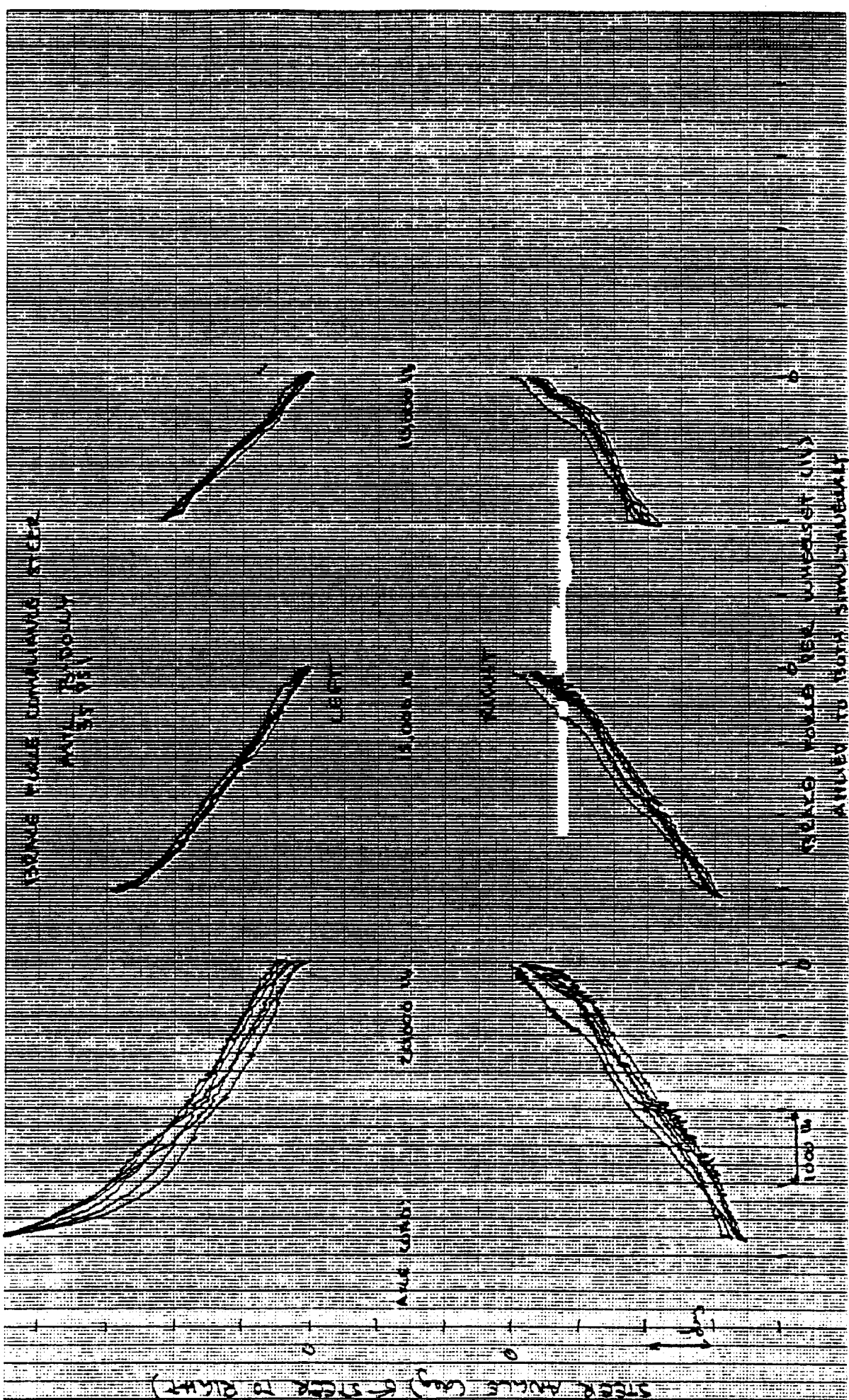
1/2 IN













#### 4.0 REFERENCES

1. A Test Facility for the Measurement of Heavy Vehicle Suspension Parameters. Winkler, Hagan, SAE Paper No. 800906.
2. Vehicle Dynamics Terminology. SAE J670e Handbook Supplement. 1976.
3. Measurement and Representation of the Mechanical Properties of Truck Leaf Springs. Fancher, Ervin, MacAdam, Winkler. SAE Paper No. 800905.
4. Parametric Analysis of Heavy Duty Truck Dynamic Stability. Winkler, C.B., Fancher, P.S., MacAdam, C.C. Final Report to NHTSA. Contract No. DTNH22-80-C-07344. University of Michigan Transportation Research Institute, UMTRI-83-13. March 1983.