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Report of Project MICHIGAN

**DYNAMIC BALANCING OF SCANNER DRUM  
AND GYROSCOPIC EFFECT  
DURING MANEUVERS OF AIRCRAFT**

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Progress and results described in reports are continually reassessed by Project MICHIGAN. Comments and suggestions from readers are invited.

Robert L. Hess  
Technical Director  
Project MICHIGAN



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# DYNAMIC BALANCING OF SCANNER DRUM AND GYROSCOPIC EFFECT DURING MANEUVERS OF AIRCRAFT

## ABSTRACT

This report presents the equations for determining the dynamic balancing and gyroscopic effect of a rotating scanner drum assembly of the type used in the Project MICHIGAN wide-angle scanner. The parameters used in the numerical example are those of the wide-angle scanner, and the theoretical calculations of the dynamic balancing agree excellently with the actual mechanical dynamic balancing of the drum assembly. The calculations also indicate that gyroscopic forces in the drum are negligible even during maximum rate of turn and maximum rate of climb of the aircraft carrying the scanner.

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## 1 INTRODUCTION

This report presents the equations for calculating, from a purely analytical viewpoint, the physical changes needed to balance a rotating mechanism. The Project MICHIGAN wide-angle scanner drum assembly was used in the numerical examples. The results obtained from a mathematical analysis were compared with the results of conventional mechanical measurements and a close agreement was found, indicating that such an analysis could be useful in reducing the amount of trial-and-error work required for dynamic balancing.

This report also discusses the gyroscopic effects on the balanced scanner during maneuvers of the transporting aircraft. It is shown that the gyroscopic forces which would be encountered in a rotating scanner during the most extreme maneuvers of conventional reconnaissance aircraft are negligible compared to the weight and centrifugal forces in the scanner.

## 2 EQUATIONS for the DYNAMIC BALANCING of a SCANNER DRUM

### 2.1. GENERAL THEORY

Equilibrium, whether static or dynamic, depends on the forces acting upon the system in question. A system is in equilibrium when

$$\sum \vec{F} = 0$$

and

$$\sum \vec{M} = 0$$

If the x-axis is the axis of rotation (Figure 1), the system is in equilibrium when

$$\sum F_x = F_x' + F_x'' = 0$$

$$\sum F_y = \iiint_V \rho \omega^2 y dV - \iiint_V \rho g dV + F_y' + F_y'' = 0$$

$$\sum F_z = \iiint_V \rho \omega^2 z dV + F_z' + F_z'' = 0$$

$$\begin{aligned} \sum M_x = & \iiint_V \rho \omega^2 zy dV - \iiint_V \rho \omega^2 yz dV + \iiint_V \rho g z dV \\ & + F_y' c + F_y'' f - F_z' b - F_z'' e = 0 \end{aligned}$$

$$\sum M_y = \iiint_V \rho \omega^2 zx dV - F_z' a + F_z'' d = 0$$

and

$$\sum M_z = \iiint_V \rho \omega^2 yx dV - \iiint_V \rho g x dV - F_x' b - F_x'' e - F_y' a + F_y'' d = 0$$

where

$$\rho = \text{density} \left( \frac{\text{pound-seconds}^2}{\text{inch}^4} \right)$$

$\omega$  = angular velocity (radians/second)

$g$  = acceleration of gravity (inches/second<sup>2</sup>)

$F', F''$  = bearing forces (pounds)

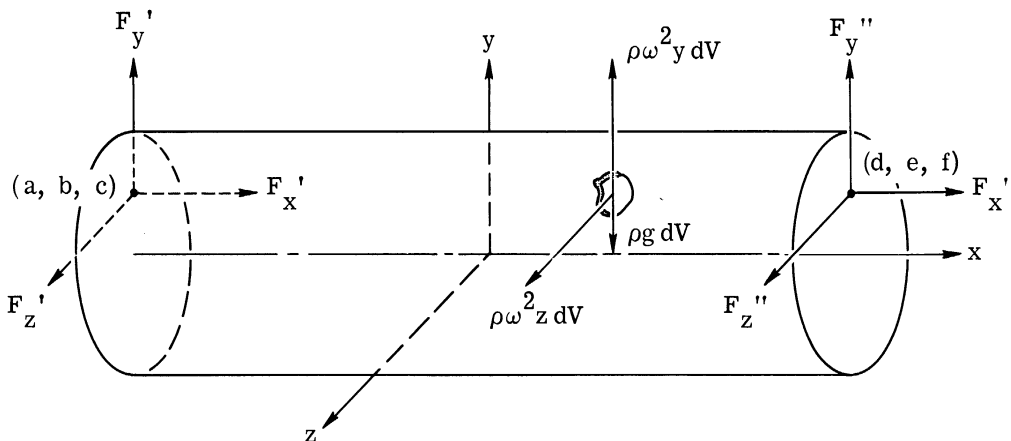


FIGURE 1. FORCES ACTING UPON A SCANNER DRUM



If the origin is located at the centroid, the system is in "static balance" and

$$\iiint_V \rho g x \, dV = \iiint_V \rho g y \, dV = \iiint_V \rho g z \, dV = 0$$

There are no x- or y-components of the bearing forces in the balanced drum, since "balance" implies that there is no bearing force other than that required to support the weight of the drum. Hence, for the dynamically balanced drum (neglecting gravity), where the axes are located at the centroid of the system and the x-axis is the axis of rotation,

$$F'_x = F''_x = F'_y = F''_y = F'_z = F''_z = 0$$

$$\iiint_V \rho z x \, dV = 0$$

and

$$\iiint_V \rho y x \, dV = 0$$

The integrals

$$\iiint_V \rho z x \, dV \equiv I_{zx}$$

and

$$\iiint_V \rho y x \, dV \equiv I_{yx}$$

are known as the "products of inertia" of the system.

The drum assembly of the wide-angle scanner is an extremely complicated mechanism geometrically, and as a result it would be very difficult to evaluate the products of inertia for the entire drum at one time. However, if a body  $B_1$  of density  $\rho_1$  has a hollow region  $B_2$  in its interior (Figure 2), then the products of inertia are

$$I_{ij} = \iiint_{B_1} \rho_1 ij \, dV - \iiint_{B_2} \rho_1 ij \, dV$$

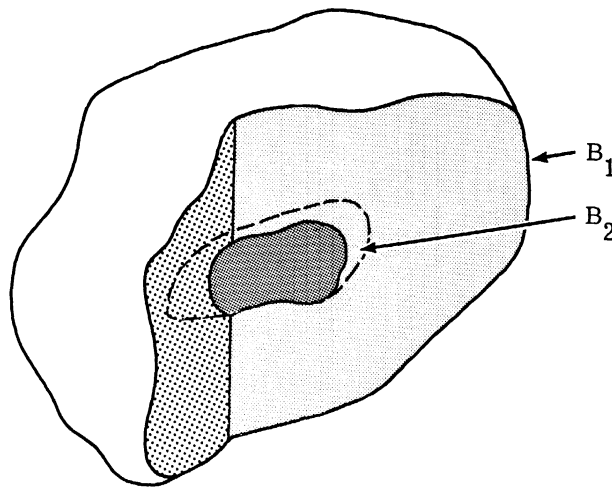


FIGURE 2. BODY WITH HOLLOW PORTION

Hence, as far as the products of inertia are concerned, a body of irregular shape may be thought of as a combination of regularly shaped bodies.

One further consideration may be made. If a body has a cross section which is independent of  $x$ , the products of inertia become

$$I_{yx} = \rho V \bar{y} \bar{x}$$

and

$$I_{zx} = \rho V \bar{z} \bar{x}$$

where  $\bar{x}$ ,  $\bar{y}$ , and  $\bar{z}$  are the coordinates of the centroid of the body. In many cases it is simpler to calculate the product of inertia in this fashion.

### 2.2. APPLICATION OF THEORY TO SCANNER DRUM ASSEMBLY

A scanner drum assembly is shown in Figure 3. Essentially, the drum is constructed by taking an aluminum cylinder and drilling two holes (of equal radii) through it, one axially and one normal to its longitudinal axis. The resulting member is cut along a  $45^\circ$  plane (Figure 4). The two halves are then separated sufficiently to allow an elliptical mirror and a mounting ring to be inserted in the  $45^\circ$  cut. Screws are placed longitudinally into the drum to hold the parts together (Figure 3). Finally, balancing is accomplished by mounting two  $90^\circ$  rings (whose thickness will be determined by the analysis) onto the drum shown in Figure 3.

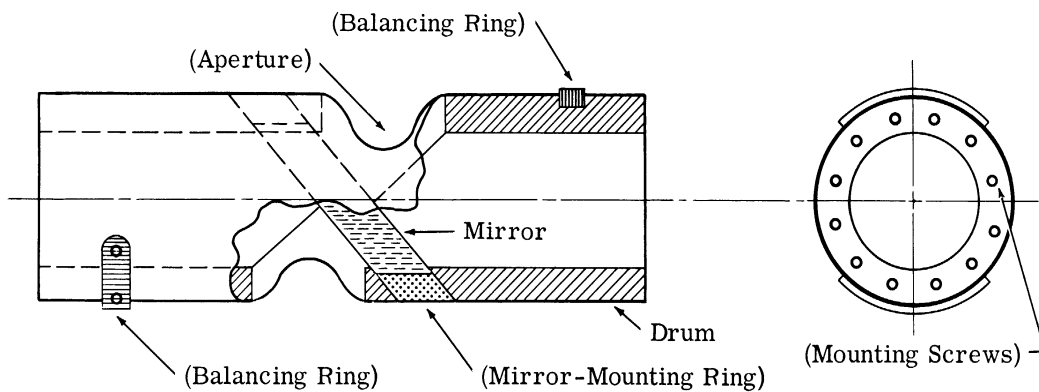


FIGURE 3. SCANNER DRUM ASSEMBLY

The  $yx$  products of inertia of several bodies follow. (The  $zx$  products all cancel or go to zero due to the symmetry of the drum assembly.)

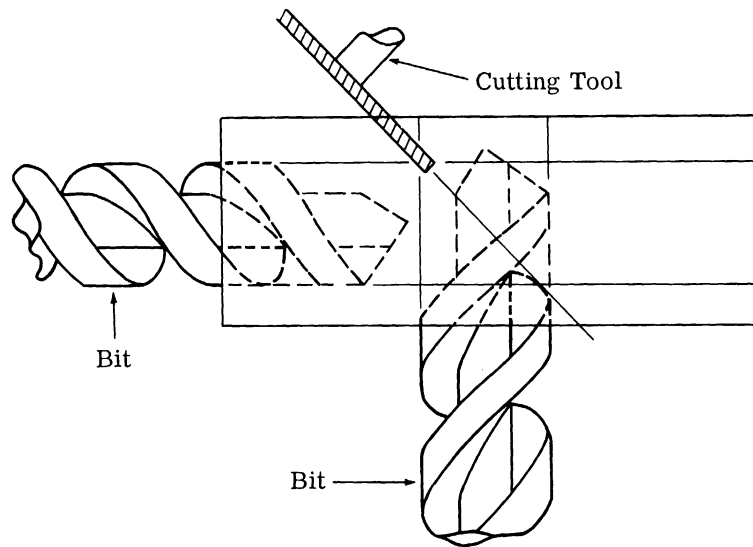


FIGURE 4. CONSTRUCTION OF SCANNER DRUM

- (A) Cylinder. The longitudinal axis of the cylinder is parallel to the x-axis, and the centers of the two ends have the coordinates  $(x_0, y_0, z_0)$  and  $(x_0 + L, y_0, z_0)$ . The length of cylinder is  $L$  and its radius is  $r$  (Figure 5).

$$I_{yx} = \rho V \bar{y} \bar{x} = \rho \pi r^2 L (x_0 + L/2) (y_0)$$

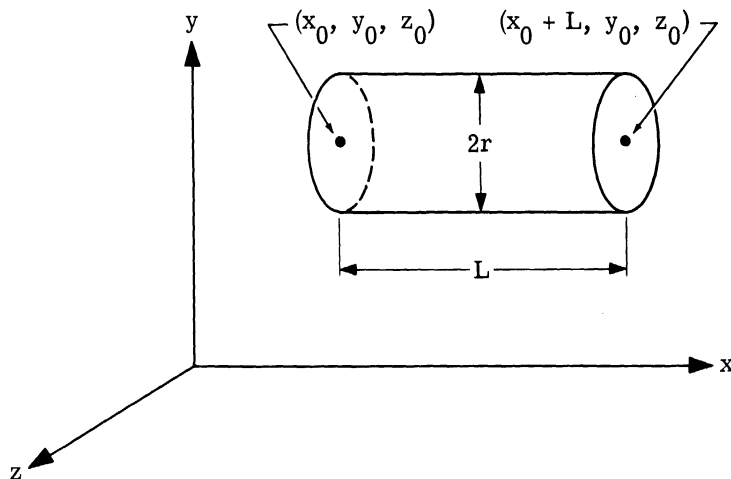


FIGURE 5. CYLINDER

- (B) Ring (1). The ring is centered at the origin but makes a 45° angle with the yz-plane (Figure 6). Its inside radius is  $r_i$ , its outside radius is  $r_o$ , and its width is  $2a$ .

$$I_{yx} = \int_0^{2\pi} \int_{r_i}^{r_o} \int_{-a-r\cos\theta}^{a-r\cos\theta} \rho(x)(r\cos\theta) r dx dr d\theta = -\frac{\pi a}{2} (r_o^4 - r_i^4) \rho$$

- (C) Ring (2). The ring is symmetric about the xy-plane and parallel to the yz-plane (Figure 7). Its inside radius is  $r_i$ , its outside radius is  $r_o$ , its width is  $2b$ , and  $\bar{x} = \ell$ . The ring subtends an arc of  $2\alpha$ .

$$I_{yx} = \int_{-\alpha}^{\alpha} \int_{r_i}^{r_o} \int_{\ell-b}^{\ell+b} \rho(x) (r\cos\theta) r dx dr d\theta = \frac{4}{3} b \ell (r_o^3 - r_i^3) (\sin\alpha) \rho$$

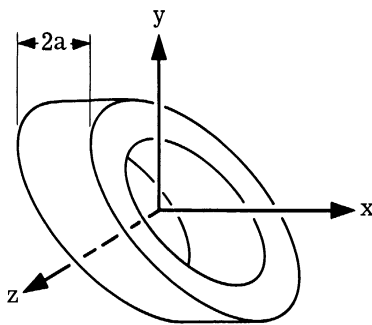


FIGURE 6. 45° RING

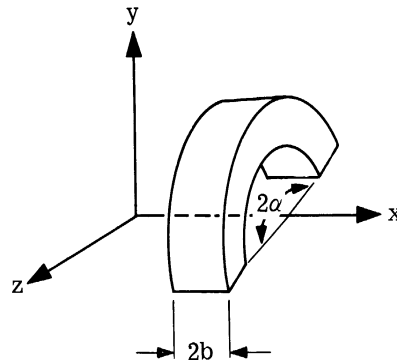


FIGURE 7. RING (2)

- (D) Aperture Opening. The aperture opening is the intersection of three cylinders (Figures 3, 4, and 8). The aperture radius is  $r_i$ , and the outside radius of the tube is  $r_o$ , and  $\bar{x} = a$ .

$$I_{yx} = \int_{-r_i}^{r_i} \int_{-\sqrt{r_o^2-z^2}}^{\sqrt{r_o^2-z^2}} \int_{-\sqrt{r_i^2-z^2}}^{\sqrt{r_i^2-z^2}} \rho(x+a)(y) dx dy dz = \frac{a}{2} (r_o^2 - r_i^2) (\pi r_i^2) \rho$$

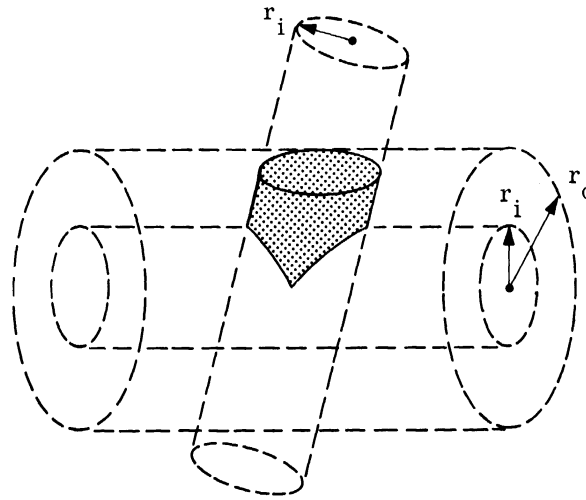


FIGURE 8. APERTURE OPENING

Each part of the scanner drum assembly in Figure 3 may be classed as one of the above bodies. All that remains is to put the proper numbers into the product of inertia expressions, add them up, and set the total equal to zero. The only unknown will be either  $\ell$ ,  $b$ ,  $\alpha$ ,  $r_o$ , or  $r_i$  for the balancing ring.

For example, the balancing of the Project MICHIGAN wide-angle scanner drum assembly may be calculated as follows.

- (a) Apertures (quantity = 2)—aluminum:  $a = 9/16\sqrt{2}$  inch,  $r_o = 3 \frac{15}{16}$  inches,  $r_i = 3 \frac{1}{16}$  inches.

$$\begin{aligned} \text{Total } I_{yx} &= -2 \left[ \frac{a}{2} (r_o^2 - r_i^2) (\pi r_i^2) \right] \rho \\ &= -71.79(2.62 \times 10^{-4}) = -0.0188 \text{ inch-pound-second}^2 \end{aligned}$$

- (b) Drum:

$$\text{Total } I_{yx} = 0$$

- (c)  $45^\circ$  cut in drum—aluminum:  $a = 9/16\sqrt{2}$  inch,  $r_o = 3 \frac{15}{16}$  inches,  $r_i = 3 \frac{1}{16}$  inches.

$$I_{yx} = \frac{\pi a}{2} (r_o^4 - r_i^4) \rho = 95.24(2.62 \times 10^{-4}) = 0.0250 \text{ inch-pound-second}^2$$

- (d) Mirror-mounting ring—Invar:  $a = 9/16\sqrt{2}$  inch,  $r_o = 3\ 15/16$  inches,  $r_i = 3\ 3/16$  inches.

$$I_{yx} = -\frac{\pi a}{2} (r_o^4 - r_i^4) \rho = -85.70 (7.63 \times 10^{-4}) = -0.0654 \text{ inch-pound-second}^2$$

- (e) Mirror—quartz:  $a = 9/16\sqrt{2}$  inch,  $r_o = 3\ 3/16$  inches,  $r_i = 0$ .

$$I_{yx} = -\frac{\pi a}{2} (r_o^4 - r_i^4) \rho = -64.51 (2.07 \times 10^{-4}) = -0.0134 \text{ inch-pound-second}^2$$

- (f) Slot for balancing rings (quantity = 2)—aluminum:  $b = 1/2$  inch,  $\alpha = \pi/4$ ,  $\ell = 4$  inches,  $r_o = 3\ 15/16$  inches,  $r_i = 3\ 3/16$  inches.

$$\text{Total } I_{yx} = -2 \left[ \frac{4}{3} b \ell (r_o^3 - r_i^3) \sin \alpha \right] \rho = -21.24 (2.62 \times 10^{-4}) = -0.0041 \text{ inch-pound-second}^2$$

- (g) Mounting screw holes— aluminum and Invar:

$$\text{Total } I_{yx} = 26.29(2.62 \times 10^{-4}) + 4.80(7.63 \times 10^{-4}) = 0.0106 \text{ inch-pound-second}^2$$

- (h) Mounting screws—steel:

$$\text{Total } I_{yx} = -10.40(7.50 \times 10^{-4}) = -0.0078 \text{ inch-pound-second}^2$$

- (i) Balancing rings— GE Hevimet:  $b = 1/2$  inch,  $\alpha = \pi/4$ ,  $\ell = 4$  inches,  $r_o = r_i$ ,  $r_i = 3\ 13/16$  inches.

$$\begin{aligned} \text{Total } I_{yx} &= 2 \left[ \frac{4}{3} b \ell (r_o^3 - r_i^3) \sin \alpha \right] \rho = (3.77r_o^3 - 209.36)(16.00 \times 10^{-4}) \\ &= 0.0060r_o^3 - 0.3350 \text{ inch-pound-second}^2 \end{aligned}$$

Since the total product of inertia of the balanced drum should be zero,

$$\sum I_{yx} = 0$$

Adding up all the parts of the drum assembly, the result is

$$0.0060r_o^3 - 0.4089 = 0$$

or

$$r_o = (67.78)^{1/3} = 4.0775 \text{ inches}$$

Hence, the thickness of the ring is

$$r_o = r_i \quad 4.0775 - 3.8125 = 0.2650 \text{ inch}$$

The thickness of the balancing weights used on the actual scanner assembly was determined experimentally to be slightly over 0.250 inch. Hence, the theoretical results agree very closely with the experimental results.

### 3 GYROSCOPIC EFFECTS on SCANNER DRUM during MANEUVERS of AIRCRAFT

The scanner is carried by an aircraft whose line of flight is parallel to the axis of rotation of the scanner drum. During maneuvers of the aircraft, the rotating drum assembly experiences a "precession" or rotation about an axis normal to the axis of rotation. A gyroscopic couple is produced in the scanner drum as shown in Figure 9. This couple produces forces in the bearings which support the drum.

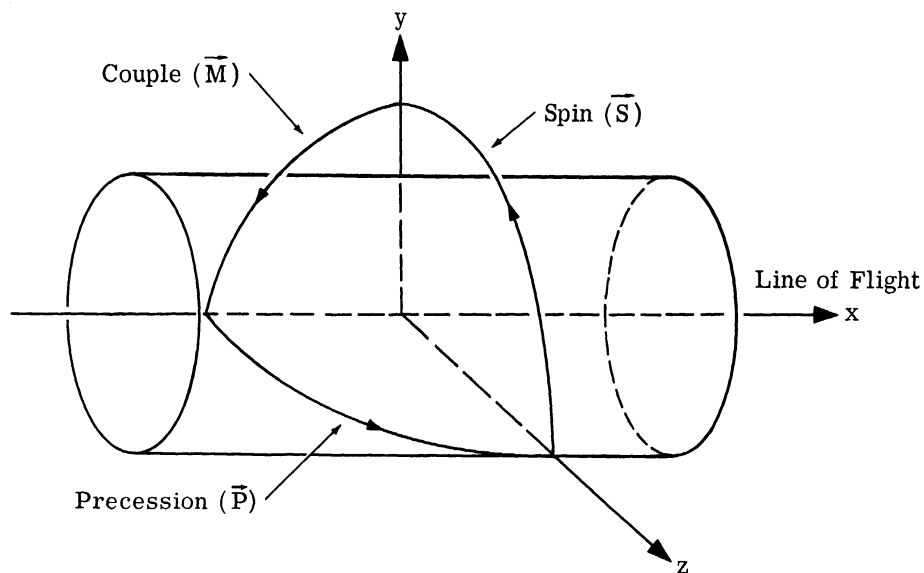


FIGURE 9. GYROSCOPIC EFFECT DURING TURN OF AIRCRAFT

The equation for the gyroscopic couple is

$$\vec{M} = I_{xx} (\vec{P} \times \vec{S}) = \begin{cases} I_{xx} PS\vec{k} & \text{(turn)} \\ I_{xx} PS\vec{j} & \text{(dive)} \end{cases}$$

where  $\vec{M}$  = couple (inch-pounds)

$I_{xx}$  = moment of inertia about x-axis (inch-pound-seconds<sup>2</sup>)

$\vec{P}$  = precession vector (radians/second)

$\vec{S}$  = spin vector (radians/second)

$\vec{j}, \vec{k}$  = unit vectors in y, z directions, respectively

Again, the wide-angle scanner drum is used as an example. The moments of inertia of the components (see Section 2.2 for descriptions) may be evaluated as follows:

(A) Cylinder

$$I_{xx} = \frac{\pi L}{2} (r_o^4 - r_i^4) \rho$$

(B) Ring (1)

$$I_{xx} = \pi a (r_o^4 - r_i^4) \rho$$

(C) Ring (2)

$$I_{xx} = ab (r_o^4 - r_i^4) \rho$$

(D) Aperture Opening

$$\text{Volume} = V = \frac{4}{3} \left\{ r^2 R [K(k) + E(k)] - R^3 [K(k) - E(k)] - 2r^3 \right\}$$

where  $k = r_i/r_o$  and  $E(k)$  and  $K(k)$  are elliptic functions.

$$I_{xx} \cong \rho V \left( \frac{r_o + r_i}{2} \right)^2$$

The calculations based on the dimensions of the wide-angle scanner follow.

(a) Apertures (quantity = 2) — aluminum:

$$\text{Total } I_{xx} = -2\rho V \left( \frac{r_o + r_i}{2} \right)^2 = -2 (2.62 \times 10^{-4}) (25.78) \left( \frac{49}{4} \right) = -0.1654 \text{ inch-pound-second}^2$$

(b) Drum — aluminum:

$$I_{xx} = \frac{\pi L}{2} (r_o^4 - r_i^4) \rho = (2482)(2.62 \times 10^{-4}) = 0.6504 \text{ inch-pound-second}^2$$



(c) 45° cut in drum — aluminum:

$$I_{xx} = -\pi a (r_o^4 - r_i^4) \rho = -(190.5)(2.62 \times 10^{-4}) = -0.0499 \text{ inch-pound-second}^2$$

(d) Mirror-mounting ring — Invar:

$$I_{xx} = \pi a (r_o^4 - r_i^4) \rho = (171.4)(7.63 \times 10^{-4}) = 0.1308 \text{ inch-pound-second}^2$$

(e) Mirror — quartz:

$$I_{xx} = \pi a (r_o^4 - r_i^4) \rho = (129.0)(2.07 \times 10^{-4}) = 0.0268 \text{ inch-pound-second}^2$$

(f) Slots for balancing rings (quantity = 2) — aluminum:

$$\text{Total } I_{xx} = -2 \left[ \alpha b (r_o^4 - r_i^4) \rho \right] = -(10.80)(2.62 \times 10^{-4}) = -0.0028 \text{ inch-pound-second}^2$$

(g) Mounting screw holes — aluminum and Invar:

$$\text{Total } I_{xx} = -0.0047 \text{ inch-pound-second}^2$$

(h) Mounting screws — steel:

$$\text{Total } I_{xx} = 0.0089 \text{ inch-pound-second}^2$$

(i) Balancing rings — GE Hevimet:

$$\text{Total } I_{xx} = 2 \left[ \alpha b (r_o^4 - r_i^4) \rho \right] = (35.10)(16.00 \times 10^{-4}) = 0.0561 \text{ inch-pound-second}^2$$

The total moment of inertia is the sum of the moments of inertia of the parts.

Hence,

$$\text{Total } I_{xx} = \sum I_{xx} = 0.6502 \text{ inch-pound-second}^2$$

Since the drum rotates at 5000 rpm, the magnitude of the gyroscopic couple is

$$|\vec{M}| = I_{xx} \text{PS} = (0.6502)(P) \frac{5000 \cdot 2\pi}{60} = 342.6 \text{ P inch-pounds}$$

If the scanner is carried by an aircraft cruising at 150 mph, the maximum rate of precession about any axis is  $360^\circ$  per minute. Therefore, the maximum gyroscopic couple is

$$|\vec{M}|_{\text{maximum}} = 342.6 |P|_{\text{maximum}} = (342.6) \left( \frac{2\pi}{60} \right) = 35.88 \text{ inch-pounds}$$

Since the distance between the bearings is 1 foot, the maximum bearing force is about 3 pounds.

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illus.

(Rept. no. 2900-250-T)

(Contract DA-36-039 SC-78801)

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