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A FINITE DIFFERENCE METHOD FOR COMPUTING UNSTEADY,
INCOMPRESSIBLE, LAMINAR BOUNDARY-LAYER FLOWS

Charles Luh-Sun Farn
Vedat S. Arpaci
John A. Clark

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Roman

a  arbitrary constant

A  arbitrary positive constant, Equations (3.14)

$A_1$  point at coordinates $(X_i, \tau, U_{i,j}, \nu_{i,j}, \eta_{i,j}, \xi_{i,j}, \zeta_{i,j})$

$A_2$  point at coordinates $(X_i, \tau, \tilde{U}_{i,j}, \tilde{\nu}_{i,j}, \tilde{\eta}_{i,j}, \tilde{\xi}_{i,j}, \tilde{\zeta}_{i,j})$

b  arbitrary constant

c  arbitrary positive constant

$C_0$  maximum value of $C_i$

$C_i$  positive constant, $C_i = \Delta Y/\Delta X_i$

$C_p$  specific heat under constant pressure process

$E_r$  rounding error, Equation (3.74.a)

$E_t$  truncation error, Equation (3.30)

$\dot{E}_t$  truncation error, Equation (3.32)

$\ddot{E}_t$  truncation error, Equation (3.34)

$f()$  arbitrary function

$F()$  arbitrary function

$g()$  arbitrary function

i  any integer greater than or equal to 1

j  any integer greater than or equal to 1

$K_1$  finite positive constant

$K_1''$  finite positive constant

$L_1$  finite positive constant

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NOMENCLATURE (Continued)

$L_1'$ finite positive constant
$L_i$ finite positive constant, Equations (3.64)
$m$ positive integer, Figure 2
$m_1$ positive integer, Figure 2
$m_2$ positive integer, Figure 2
$M_n$ maximum value of $|U_{i,j}^n|$
$n$ any integer greater than or equal to 0
$N_n$ maximum value of $|\Theta_{i,j}^n|$
$Nu$ Nusselt number, $Nu = \frac{-\left(\frac{\partial \Theta}{\partial Y}\right)_{Y=0}}{\left(\Theta\right)_{Y=0} - \left(\Theta\right)_{Y=0}}$
$p$ static pressure
$P_0$ maximum value of $|\left(\frac{\partial U_0}{\partial x}\right)_i^n + (U_0 \frac{\partial U_0}{\partial x})_i^n|$
$Pr$ Prandtl number, $Pr = \rho C_p \nu / \kappa$
$q$ heat flux
$Q$ dimensionless heat flux, $Q = q/(\frac{\kappa u^3}{\nu C_p})$
$R$ local radius of curvature, Figure 1
$Re$ Reynolds number, $Re = u_c x_c / \nu$
$S$ positive integer, Figure 2
$t$ time variable
$T$ temperature
$u$ $x$-component velocity, Figure 1
$\Delta u$ amplitude of oscillation of $u$, Equation (4.1)
$U$ dimensionless form of $u$, $U = u/u_c$
NOMENCLATURE (Continued)

ΔU  dimensionless form of $\Delta u$, $\Delta U = \Delta u/u_c$

$\nu$  $y$-component velocity

$V$  dimensionless form of $v$, $V = v/u_c$

$W$  dimensionless frequency, $W = \omega/(u_c^2/v)$

$x$  space variable along the surface of the obstacle, Figure 1

$x_i$  any independent variable

$X$  dimensionless form of $x$, $X = x/(v/u_c)$

$X_i$  value of $X$ at space grid point $(i,j)$, Equations (3.16.a) and (3.16.b)

$\Delta X_i$  space interval between grid points $(i,1)$ and $(i-1,1)$, Figure 2

$y$  space variable perpendicular to the surface of the obstacle, Figure 1

$y_j$  any independent variable

$Y$  dimensionless form of $y$, $Y = y/(v/u_c)$

$Y_j$  value of $Y$ at space grid point $(i,j)$, Equations (3.16.a) and (3.16.c)

$\Delta Y$  space interval between grid point $(i,j)$ and $(i,j + 1)$, Figure 2

$\Delta Y$  and Equations (3.14)

$z$  any independent variable

$Z$  quantity defined by Equation (3.49.a)

Greek

$\alpha$  difference between $\tilde{U}$ and $U$, Equation (3.57.a)

$\alpha_n$  maximum value of $|\alpha_{1,j}^n|$, Equation (3.61.a)

$\beta$  difference between $\tilde{V}$ and $V$, Equation (3.57.b)

$\beta_n$  maximum value of $|\beta_{1,j}^n|$, Equation (3.61.b)

$\beta''_n$  maximum value of $\beta_k (k=0,1,\ldots,n)$, Equation (3.62)
NOMENCLATURE (Continued)

\( \gamma_n \) quantity defined by Equation (3.69)

\( \gamma_{ij} \) rounding error, Equation (3.75.a)

\( \delta \) some constant between 0 and 1

\( \epsilon \) instability error, Equation (4.49.b)

\( \zeta \) quantity defined by Equation (3.56.c)

\( \eta \) quantity defined by Equation (3.56.a)

\( \eta_1 \) rounding error, Equation (3.75.b)

\( \Theta \) dimensionless form of \( T \), \( \Theta = T/(u_c^2/C_p) \)

\( \Theta \) phase angle of local oscillations

\( \kappa \) thermal conductivity

\( \nu \) kinematic viscosity

\( \xi \) quantity defined by Equation (3.56.b)

\( \rho \) density

\( \sigma \) some constant between 0 and 1/2

\( \tau \) dimensionless form of \( t \), \( \tau = t/(v/u_c^2) \)

\( \tau_n \) value of \( \tau \) at the \( n \)th time level, Equation (3.17)

\( \varphi \) arbitrary function

\( \Phi \) arbitrary function

\( \psi \) function defined by Equation (3.50)

\( \omega \) frequency of oscillation in free stream, Equation (4.1)

Subscripts

\( b \) upper bound

\( c \) characteristic quantity

\( \xi \)
NOMENCLATURE (Continued)

\(i, j\) condition at coordinates \((X_i, Y_j)\)

\(o\) condition at free stream \((Y = \infty)\)

\(ob\) condition of the moving obstacle

\(s\) initial condition \((\tau = 0)\)

\(w\) condition at the wall of the obstacle \((Y = 0)\)

\(X\) partial differentiation with respect to \(X\)

\(Y\) partial differentiation with respect to \(Y\)

\(YY\) second partial differentiation with respect to \(Y\)

\(\infty\) condition at main stream

Superscripts

\((i)\) \(i\)th approximation

\((n)\) condition at the instant \(\tau_n\)

\(('\) oscillating quantity, Equations (2.14)

\((\cdot)\) condition at some point between \(A_1\) and \(A_2\)

\((\overline{\cdot})\) time-averaged quantity, Equations (2.14)

\((\hat{\cdot})\) exact solution of PDE (in distinction from the exact solution of the corresponding FDAE)

\((\tilde{\cdot})\) computer solution of FDAE (in distinction from the exact solution of the same FDAE)

Abbreviations

cps cycles per second

FDA(s) finite difference approximation(s)
NOMENCLATURE (Concluded)

FDAE(s)  finite difference approximate equation(s)

fl(B)   arithmetic operation 'B' according to the floating point number system

fps     feet per second

PDE(s)  partial differential equation(s)

|B|      absolute value of 'B'
ABSTRACT

A finite difference method for computing velocity and temperature profiles of an unsteady, incompressible, laminar boundary layer around a two-dimensional cylinder of arbitrary cross section is developed. Blowing or suction may be present on the wall of the cylinder. The thermal boundary condition at the wall may be either specified wall temperature or specified heat flux at the wall.

The governing finite difference equations are explicit, the velocity and the temperature at the next time step can be directly computed in terms of those at the current time step. Various errors associated with the finite difference method are studied. Great effort has been made to derive the stability and convergence conditions of the method. The upper bound of the local rounding errors is estimated.

Two examples, one oscillation in Blasius flow and the other impulsive start of wedge flow, are given; the numerical results are compared with the existing analytical and experimental results. It is concluded that the present method can be used for the aforementioned computation with high accuracy except at and near a singular point where the singular errors become significant.
CHAPTER I
INTRODUCTION

It is well known that the motion of a viscous, incompressible fluid can be described by the Navier-Stokes equations coupled with the equation of continuity. The general solution of these equations has not yet been available within the present knowledge of mathematics. However, by assuming the viscosity effect important in a thin fluid layer over boundaries, Prandtl (1) introduced his well-known boundary-layer equations which have simpler forms than full Navier-Stokes equations. After Prandtl, most research has been concentrated on the steady flow. It is only during the last two decades, the unsteady laminar boundary-layer flows have been treated extensively.

The unsteady laminar boundary layer theory presents many challenging problems which may be classified as the periodic and starting-ending problems. In the former case, the periodic behavior may be introduced by oscillating the obstacle or the fluid. As a result of this, the response velocities of the fluid particles in the boundary layer are also periodic with respect to the time. Many practical important problems fall into this category, such as the flow past a turbine blade or past an oscillating airfoil. The latter case includes all the flow problems in which the obstacle or the fluid starts to move from the rest with a time dependent velocity. The flow generated by an impulsive motion of an obstacle in a stationary fluid field is a special example of this category.
Generally speaking, the method of solving an unsteady, laminar boundary-layer problem will fall into one of the following six ways:

1. Similar solutions.
2. Semi-similar solutions.
4. Successive approximations.
5. Integral method.
6. Other methods.

A detailed review of the literature about the above methods will be given in Chapter II. Here we shall briefly discuss these methods.

The method of similar or semi-similar solutions reduces the number of independent variables of a two-dimensional unsteady laminar boundary-layer problem from three into one or two, respectively. Such similar or semi-similar solutions exist only when the free stream velocity of a given boundary-layer problem behaves in a certain specific way. Therefore, these methods are restricted to problems in which the obstacle has a particular geometry and moves with a particular time-dependent velocity.

The method of series solution is based on the expansion of the local velocity into a power series of some parameter or coordinate. Because of the mathematical complexity, only the first few terms of the series may be calculated. The series solution deviates appreciably from the exact solution when the value of the expansion parameter or coordinate becomes large.

The method of successive approximations is actually an iteration procedure. In this case, a first approximation is obtained by physics, and the successive approximations are then obtained by iteration. Therefore,
the convergence of the final approximation to the exact solution of the problem largely depends on the choice of the first approximation. Also, the higher approximations of this method are rather complicated algebraically.

The integral method is an approximate procedure, because of the fact that the profiles obtained by this method only satisfy the boundary conditions and the integral form of the governing equations.

None of the foregoing methods meet the requirements of generality and high accuracy. In recent years, advances in high-speed digital computer technology have made it possible to develop some numerical methods applicable to complex fluid flow problems which were beyond reach by the foregoing methods of solution. For example, De Saute and Keller (2) recently studied the transition from laminar to turbulent flow over a flat plate by finite-difference technique. Der and Raetz (3) developed a numerical method for general three-dimensional laminar boundary-layer problems. Fromm (4) computed the development of the vortices behind a two-dimensional obstacle. Also Yang (5) was able to study the hydrodynamic stability of two-dimensional unsteady incompressible laminar boundary layers by means of the numerical iteration scheme and Barakat (55) studied the transient natural convection in closed container by finite difference method.

It is the purpose of this work to develop a finite difference method to determine both unsteady velocity and temperature profiles in a laminar
boundary layer over a cylinder of arbitrary cross section moving with an arbitrary time-dependent velocity in an incompressible fluid. Suction or blowing on the surface of the solid cylinder may also be included. The wall temperature or the heat flux at the wall of the obstacle may vary arbitrarily with the time and the distance from the front stagnation point. In order to illustrate the applicability of the method, two numerical examples, one periodic and the other starting-ending type, are considered. The results are compared with those obtained by the analytical methods and, whenever possible, by experimental measurements.
CHAPTER II

REVIEW OF THE PREVIOUS WORKS

In this chapter, a brief survey of the existing literature about unsteady incompressible laminar boundary layers will be given based on the methods classified in Chapter I.

A. SIMILAR SOLUTIONS

The solutions of the velocity component \( u \) in an unsteady two-dimensional laminar boundary layer are called "similar" if the two velocity profiles at the different coordinates \( x \) and at the different instants \( t \) differ only by a scale factor. More precisely,

\[
\frac{u(x,y,t)}{u_0(x,t)} = f(\Gamma)
\]  (2.1)

where,

\[
\Gamma = \frac{y}{g(x,t)}
\]  (2.2)

The transformation defined in the equation (2.2) is called the "similarity transformation" and the function 'g' is called the "scale function." Through this transformation, the original governing PDE* for \( u \) can be reduced to an ordinary differential equation. Consequently, a considerable mathematical simplification of the problem is achieved.

*Partial Differential Equation.
Schuh (6) showed similar solutions may be obtained if the velocity at the outer edge of the boundary layer is proportional to \( t^a, e^{bt} \) or \( x/t \), where 'a' and 'b' are arbitrary constants. Yang (7) studied the similar solutions of a stagnation flow of which the free stream velocity can be expressed as \( ax/(1-bt) \).

Perhaps, the most complete work about the similar solutions of an unsteady boundary-layer flow was the one given by Hayasi (8) who studied the possible similar solutions in an unsteady quasi-two-dimensional incompressible flow. The definition of a "quasi-two-dimensional" flow can be found in Reference (9). The similar solutions found by Schuh (6) and Yang (7) are only the special cases of Hayasi's results when the flow becomes exactly two-dimensional.

A systematic study of the topics by means of the "free parameter method" is described in a recently published book by Hansen (10). The rest of this section is devoted to a brief review of this method.

Let \( \varphi \) be some physical unknown to be solved. Assume it satisfies a certain governing PDE and depends on some independent variables \( x_1, x_2, \ldots, x_i \) and \( z \). Let the boundary conditions in \( z \) be

\[
\varphi(x_1, x_2, \ldots, x_i, \delta_0) = 0 \quad (2.3.a)
\]

\[
\lim_{\delta \to \delta_1} \varphi(x_1, x_2, \ldots, x_i, \delta) = \Phi(x_1, x_2, \ldots, x_i) \quad (2.3.b)
\]

In searching for a similar solution of \( \varphi \), one usually assumes

\[
\varphi(x_1, x_2, \ldots, x_i, \delta) = \Phi(x_1, x_2, \ldots, x_i) f(\Gamma) \quad (2.4)
\]
where,
\[ \Gamma' = \Gamma(\chi_1, \chi_2, \ldots, \chi_i, \tilde{\gamma}) \]  
(2.5)

'\Gamma' is the unknown "free parameter" to be specified. By means of the transformations given by the foregoing equations, one hopes to transform the governing PDE for \( \varphi \) into an ordinary differential equation for 'f' under certain imposed conditions. If the given physical problem meets these conditions, then a similar solution of \( \varphi \) exists and vice versa.

It is also very important to examine the boundary conditions for 'f'. According to equations (2.3) through (2.5), the boundary conditions for 'f' are

\[ f(\Gamma_0) = 0 \]  
(2.6.a)
\[ \lim_{\Gamma \to \Gamma_1} f(\Gamma) = 1 \]  
(2.6.b)

where,
\[ \Gamma_0 = \Gamma(\chi_1, \chi_2, \ldots, \chi_i, \tilde{\gamma}_0) \]  
(2.7.a)
\[ \Gamma_1 = \Gamma(\chi_1, \chi_2, \ldots, \chi_i, \tilde{\gamma}_1) \]  
(2.7.b)

If a similar solution of \( \varphi \) exists, the values of \( \Gamma_0 \) and \( \Gamma_1 \) must be independent of \( x_k \) (\( k=1, \ldots, i \)). These requirements become difficult to be achieved when both \( z_0 \) and \( z_1 \) take finite values. In general, the problems of finite geometry do not have similar solutions.
B. SEMI-SIMILAR SOLUTIONS

The solution of the velocity component \( u \) in an unsteady two-di-

dimensional boundary layer is called "semi-similar" if it can be expressed

as a function of two independent variables. An example of such flow is

the incompressible boundary layer whose free stream velocity takes one

of the following forms: \( at^b, a/(1+bt), f(\lambda)t^a \), where \( \lambda = xt^{a+1} \). The

first two cases in the above example were treated by Cheng (11) and

the last one was studied by Hassan (12).

Recently, Hayasi (13) gave an extensive treatment to the semi-
similar solutions of an unsteady quasi-two-dimensional incompressible
laminar boundary-layer flow; two simplest cases, time-dependent and

space-dependent semi-similar boundary layers, were studied.

The semi-similar solutions are obtained by the "group-theoretic

method" (10). Because of the length of the background required, this

method will not be reviewed here. In short, one tries to find a one-

parameter transformation group under which the governing PDEs can be

transformed into the different variables. Therefore, by doing so, one

reduces the number of independent variables by one.

As pointed out in the Reference (10), there are two disadvantages
to employing the group theoretic method. The first is the boundary

conditions which are not taken into account until the entire analysis

is completed. The second is the uncertainty in choosing a proper trans-
formation group.
C. SERIES SOLUTIONS

For boundary-layer problems of starting-ending type, the stream function is usually expanded in ascending powers of some parameter related to $t$ or $x$. Blasius (14) employed this method to study the impulsive motion of a cylinder and the uniform acceleration of a cylinder in an incompressible fluid. In both cases, the stream function has been expanded in ascending powers of $\Lambda$ which is defined as $tu_c/x_c$, where $u_c$ and $x_c$ are the characteristic velocity and length, respectively. In the first case, only the first two terms of the series have been calculated by Blasius, the third term has been given later by Goldstein and Rosenhead (15).

Watson (16) generalized the Blasius' problems by studying the boundary-layer growth on a cylinder starting from rest with the velocity $u_{ob}$ proportional to $ta$ or $e|b|t$. For the case when $u_{ob} = e|b|t$, Watson expanded stream function in ascending powers of the parameter 'λ' which is defined as $\int_{-\infty}^{t} u_{ob}(t)dt$.

The boundary-layer problem of a semi-infinite flat plate starting initially from rest in an incompressible fluid with an arbitrary velocity $u_{ob}$ was studied by Cheng and Elliott (17) by means of the series solution method also. A parameter $\Omega$, which is defined as $x/\int_{0}^{t} u_{ob}(t)dt$, has been used in power series expansion of the stream function.

For periodic boundary-layer problems, the local velocity is expanded in ascending powers of $\Delta u_o/u_o$ which is the relative amplitude of oscillation of the free stream. For instance, Kestin, Maeder and Wang (18)
utilized this method to study the effect of a longitudinal sine wave on the boundary-layer along a flat plate. Hori (19) used the same method to investigate the boundary-layer around a cylinder of arbitrary cross section in a fluctuating main stream. Clearly, the mathematical complexity does not allow the computation of many terms in a series solution. It is doubtful, unless checked by experiments, that the series solution converges to the exact solution when the parameter involved in expansion becomes large. As pointed out by Rosenhead (21), for example, the series solutions employed in Reference (14) are not valid for values of \( \Lambda \) greater than 0.3.

The foregoing method of series solution is actually a small perturbation technique. In starting-ending problems, the time or space coordinate is used as perturbation quantity, hence, the method may be classified as the coordinate perturbation. In periodic problems, the relative amplitude of oscillation is used as the perturbation parameter, therefore, the method becomes a parameter perturbation. It should be remarked that the choice of an appropriate perturbation parameter is by no means an easy task. The details of the small perturbation method may be found in the book by Van Dyke (21).

D. SUCCESSIVE APPROXIMATIONS

The process of the successive approximations has been described in Reference (22). An example on the application of this method can be found in Reference (23). The method is based on the physical argument
that if a solid obstacle is initially at rest relative to the fluid, the boundary layer formed on the surface of the solid obstacle is thin right after the start. Consequently, the viscous term in unsteady boundary-layer equations is large, while the convective terms retain their normal values. Therefore, at early stage of the transient phenomenon, it may be assumed that the viscous term is balanced by the acceleration plus the unsteady portion of the pressure gradient. According to the foregoing argument, the first approximations $u^{(1)}$ and $v^{(1)}$ satisfy the following linear PDEs:

$$
\frac{\partial u^{(1)}}{\partial t} = \frac{\partial u_0}{\partial t} + v \frac{\partial u^{(1)}}{\partial y^2} \quad (2.8)
$$

$$
\frac{\partial u^{(1)}}{\partial x} + \frac{\partial v^{(1)}}{\partial y} = 0 \quad (2.9)
$$

with the boundary conditions $y = 0$: $u^{(1)} = v^{(1)} = 0$; $y \to \infty$: $u^{(1)} = u_0(x,t)$.

The second approximations $u^{(2)}$ and $v^{(2)}$ may then be calculated from the following equations:

$$
\frac{\partial u^{(2)}}{\partial t} - \nu \frac{\partial^2 u^{(2)}}{\partial y^2} = \frac{\partial u_0}{\partial t} + u_0 \frac{\partial u_0}{\partial x} - u^{(1)} \frac{\partial u^{(1)}}{\partial x} - v^{(1)} \frac{\partial u^{(1)}}{\partial y} \quad (2.10)
$$

$$
\frac{\partial u^{(2)}}{\partial x} + \frac{\partial v^{(2)}}{\partial y} = 0 \quad (2.11)
$$

with the boundary conditions $y = 0$: $u^{(2)} = v^{(2)} = 0$; and $y \to \infty$: $u^{(2)} = u_0(x,t)$.

The third and higher approximations are obtained by repeating the above iteration procedure. However, the complexity of algebra grows at
a staggering rate. It should be noted that the successive approximation method fails to distinguish an unsteady boundary-layer along a semi-infinite flat plate from that along an infinite flat plate. This is due to an oversimplification in the first approximation of the boundary-layer equations.

E. INTEGRAL METHOD

The methods discussed so far give local profiles which satisfy the differential form of boundary-layer equations. A high accuracy may be obtained by carrying out enough terms. When it is desired to predict approximately the distribution of the skin friction, the position of the flow separation, etc., an 'approximate' method such as the integral method becomes quite useful. Schuh (24) studied the unsteady boundary-layer problems by using this method. It was shown that in the general case of any type of velocity distribution over outer edge of the boundary-layer, regarding space and time, the integral of boundary-layer equations could be reduced to two simultaneous differential equations, one being an equation of characteristics. Two numerical examples were given: first, the growth of the boundary-layer around a circular cylinder for impulsive motion, second, the generation of the boundary-layer along a flat plate for impulsive motion and the steady acceleration. In the first example, Pohlhansen's (25) velocity profile was used, while in the second example, Hartree's (26) velocity profile was assumed. Schuh's procedure was later modified and extended by Yang (27) to problems includ-
ing heat transfer. Other works related to integral method may be found in References (28,29).

Recently, Hayasi (30) developed an approximate method for calculating unsteady quasi-two-dimensional laminar boundary-layers. The velocity profiles were assumed to be given by one-parameter family of curves. The profile parameter which was selected to be the non-dimensional shearing stress at the wall was determined by the momentum and energy-integral equations. It was found, after comparison with the exact solutions, that Pohlhansen's velocity profile gave good results for decelerating main flows, while Hartree's velocity profile was good for accelerating main flows.

F. OTHER METHODS

For a special unsteady boundary-layer problem, one may make some assumptions which are particularly suitable to the conditions of the problem under consideration. Thus the method to be developed based on these assumptions may not be used for problems of the different nature. To illustrate this point, a brief review on C. C. Lin's (31) theory of the harmonic oscillations is given next.

Lin considered an unsteady two-dimensional boundary-layer with a main stream oscillating periodically about a mean value. The mathematical difficulty of the problem was simplified by taking time-averages of the terms in boundary-layer equations in a manner analogous to that used in turbulent flows. Denoting the oscillating components by primes,
Lin assumes

\[ u_c(x,t) = \bar{u}_0(x) + u'(x,t), \quad \bar{u}'_0 = 0 \]  \hspace{1cm} (2.14.a)

\[ u(x,y,t) = \bar{u}(x,y) + u'(x,y,t), \quad \bar{u}' = 0 \]  \hspace{1cm} (2.14.b)

\[ v(x,y,t) = \bar{v}(x,y) + v'(x,y,t), \quad \bar{v}' = 0 \]  \hspace{1cm} (2.14.c)

\[ p(x,t) = \bar{p}(x) + p'(x,t), \quad \bar{p}' = 0 \]  \hspace{1cm} (2.14.d)

By restricting the oscillation of the main stream to very high frequency range, the unsteady portion of the boundary-layer equation is linearized in the form:

\[ \frac{\partial u'}{\partial t} = \frac{\partial \bar{u}_0}{\partial t} + \nu \frac{\partial^2 u'}{\partial y^2} \]  \hspace{1cm} (2.15)

After calculating the longitudinal velocity component \( u' \) from the equation above and the corresponding boundary conditions, the transverse velocity component \( v' \) is obtained from the unsteady portion of the continuity equation

\[ \frac{\partial \bar{u}'}{\partial x} + \frac{\partial \bar{v}'}{\partial y} = 0 \]  \hspace{1cm} (2.16)

The steady portion of the boundary-layer equations reduces to

\[ \bar{u} \frac{\partial \bar{u}}{\partial x} + \bar{v} \frac{\partial \bar{u}}{\partial y} = \bar{u}_0 \frac{\partial \bar{u}}{\partial x} + \nu \frac{\partial^2 \bar{u}}{\partial y^2} + \bar{u}_0' \frac{\partial \bar{u}}{\partial x} - \left( \bar{u}' \frac{\partial \bar{u}}{\partial x} + \bar{v}' \frac{\partial \bar{u}}{\partial y} \right) \]  \hspace{1cm} (2.17)

\[ \frac{\partial \bar{u}}{\partial x} + \frac{\partial \bar{v}}{\partial y} = 0 \]  \hspace{1cm} (2.18)
From the above two equations, the time-averaged velocity components \( \bar{u} \) and \( \bar{v} \) may be calculated with the corresponding boundary conditions.

Clearly, the foregoing method of solution can not be used for fluid flow problems other than those related to high frequency oscillations. Finally, it is worth noting that the same problem was later considered by Gibson (32) by the so called "splitting solution" method.
CHAPTER III

FORMULATION OF THE PROBLEM

A. DIFFERENTIAL FORM

A curvilinear orthogonal coordinate system attached to the surface of a two-dimensional cylinder is selected as shown in Figure 1.

![Diagram of curvilinear orthogonal coordinates]

**Figure 1.** Curvilinear orthogonal coordinates.

The laminar boundary-layer equations with respect to these coordinates are
\[ \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial P}{\partial x} + \frac{\partial^2 u}{\partial y^2} \]  
\[ \frac{1}{\rho} \frac{\partial P}{\partial y} = \frac{u^2}{R} \]  
\[ \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \]  
\[ \frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \frac{\kappa}{\rho c_p} \frac{\partial^2 T}{\partial y^2} \]  

where \( R(x) \) is the local radius of curvature of the wall, reckoned positive when the wall is convex outward.

The derivation of the above set of equations can be found in texts on boundary-layer theory, for example, in References (20, 22). The equations are valid within the following assumptions:

1. The fluid has constant properties.
2. The solid obstacle may be stationary or may move with a translational motion parallel to the motion of the main flow.
3. The Reynolds number \( Re \) based on the characteristic velocity and length of the problem is large enough so that the viscous effects of the flow are limited to a thin layer on the wall. The thickness of this layer is one order smaller than that of the characteristic length. Outside the velocity boundary layer, the flow is assumed to be inviscid. Also the variation of the velocity in the direction of the \( y \)-axis is one order greater than that in the direction of the \( x \)-axis.
4. The Prandtl number \( Pr \) must be order of unity or higher. This implies the thickness of the thermal boundary-layer is the same order or less than that of the velocity boundary-layer. Consequently, the
variation of the temperature in the direction of the $y$-axis is one order greater than that in the direction of the $x$-axis.

(5) The viscous dissipation is negligibly small.

(6) The magnitude of the local radius of curvature of the wall is one order greater than that of the local velocity boundary-layer thickness.

(7) The variation of the local radius of curvature along the direction of the $x$-axis is at most of the order of unity.

In general, the Navier-Stokes equations are derived relative to a space-fixed coordinate system. Consequently, the resultant boundary-layer equations are restricted to a coordinate system fixed in space. However, when the coordinate system moves with a translational motion parallel to the main stream, the uniform inertial force per unit volume due to the translational motion of the coordinate system appears in equation (3.1.a) in the form of an induced pressure gradient. The transform from the inertial force to the pressure gradient is achieved instantaneously by the acoustic waves in an incompressible medium (33). Therefore, as indicated by the second assumption, the boundary-layer equations (3.1) and (3.2) are valid for both stationary and moving obstacles.

Based on the third assumption, equation (3.1.b), it may be concluded that the magnitude of the pressure variation across the boundary-layer is second order. Therefore, hereafter the pressure is assumed to be constant across the boundary-layer, and the pressure gradient in equation (3.1.a) is same as that given by the momentum equation of the inviscid
flow at the outer edge of the boundary-layer. Thus

\[ \frac{\partial u_0}{\partial t} + u_0 \frac{\partial u_0}{\partial x} + v_0 \frac{\partial u_0}{\partial y} = - \frac{i}{s} \frac{\partial p}{\partial x} \]  \tag{3.4}

where \( u_0 \) and \( v_0 \) are the streamwise and the normal components of the velocity at the outer edge of the boundary-layer, respectively. Because of the boundary layer thickness, the first approximation \( u_0 \) and \( v_0 \) of equation (3.4) may be assumed to be the corresponding velocity components evaluated at boundaries. Since the normal component of the fluid velocity vanishes at boundaries, equation (3.4) reduces to

\[ \frac{\partial u_0}{\partial t} + u_0 \frac{\partial u_0}{\partial x} = - \frac{i}{s} \frac{\partial p}{\partial x} \]  \tag{3.5}

For boundary-layer problems with blowing or suction, equation (3.5) is approximately true provided that the magnitude of blowing or suction velocity is smaller in order than that of the free stream velocity \( u_0 \). Accordingly, equations (3.1) become

\[ \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \frac{\partial u_0}{\partial t} + u_0 \frac{\partial u_0}{\partial x} + v \frac{\partial u_0}{\partial y^2} \]  \tag{3.6}

Equations (3.6), (3.2) and (3.3) are the governing differential equations for dependent variables \( u, v \) and \( T \) in a laminar boundary-layer flow. Their boundary conditions are

for \( x = 0, y > 0 \) and \( t > 0 \); \( u = u_0(0,t) \) \hspace{1cm} (3.7.a)

\( T = T_0(0,t) \) \hspace{1cm} (3.7.b)

for \( x \geq 0, y = 0 \) and \( t > 0 \); \( u = 0 \) \hspace{1cm} (3.7.c)
\[ v = v_w(x,t) \quad (3.7.d) \]
\[ T = T_w(x,t) \quad (3.7.e) \]

for \( x \geq 0, \ y \rightarrow \infty \) and \( t > 0; \ u = u_0(x,t) \quad (3.7.f) \]
\[ T = T_0(x,t) \quad (3.7.g) \]

where \( u_0 \) and \( T_0 \) are the velocity and the temperature of the free stream, respectively, while \( v_w \) is the blowing or suction velocity normal to the wall and \( T_w \) is the wall temperature. When the blowing or suction is absent, \( v_w \) is equal to zero. The thermal boundary condition given by equation (3.7.e) specifies the wall temperature. In case when the heat flux at the wall is specified, this boundary condition becomes then

\[-k \left( \frac{\partial T}{\partial y} \right)_{y=0} = q_w(x,t) \quad (3.7.h)\]

where \( q_w(x,t) \) is the heat flux at the wall, reckoned positive when heat flows from the wall to the fluid.

The initial condition of the problem is

at \( t = 0; \ u = u_0(x,y) \quad (3.8.a) \]
\[ v = v_0(x,y) \quad (3.8.b) \]
\[ T = T_0(x,y) \quad (3.8.c) \]

where \( u_0, v_0 \) and \( T_0 \) are the initial distributions of velocity components \( u, v \) and temperature \( T \), respectively. However, only \( u_0 \) and \( T_0 \) need to be specified according to the physical situation of the problem; once \( u_0 \) is given, \( v_0 \) can be readily obtained by integrating equation (3.2).

Next, the above governing differential equations will be nondimension-
alized. Defining the following dimensionless quantities:

\[ U = \frac{u}{u_c}, \quad V = \frac{v}{u_c}, \quad \theta = \frac{T - T_c}{(u_c^2 C_p)}, \quad Pr = \frac{\nu S C_p}{K}, \]

\[ X = \frac{x}{u_c}, \quad Y = \frac{y}{u_c}, \quad \tau = \frac{t}{(u_c)^2}, \quad Q = \frac{q}{(K/\mu C_p)}, \]

where \( u_c \) denotes some constant characteristic velocity, equations (3.6), (3.3) and (3.2) take the following forms, respectively:

\[ \frac{\partial U}{\partial \tau} + U \frac{\partial U}{\partial X} + V \frac{\partial U}{\partial Y} = \frac{\partial U_0}{\partial \tau} + U_0 \frac{\partial U_0}{\partial X} + \frac{\partial^2 U}{\partial Y^2} \]  \( \text{(3.9)} \)

\[ \frac{\partial \theta}{\partial \tau} + U \frac{\partial \theta}{\partial X} + V \frac{\partial \theta}{\partial Y} = \frac{1}{Pr} \frac{\partial^2 \theta}{\partial Y^2} \]  \( \text{(3.10)} \)

\[ \frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} = 0 \]  \( \text{(3.11)} \)

The dimensionless boundary conditions are:

for \( X = 0, \ Y > 0 \) and \( \tau > 0; \ U = U_0(0, \tau) \)  \( \text{(3.12.a)} \)

\[ \theta = \theta_0(0, \tau) \]  \( \text{(3.12.b)} \)

for \( X \geq 0, \ Y = 0 \) and \( \tau > 0; \ U = 0 \)  \( \text{(3.12.c)} \)

\[ V = V_w(X, \tau) \]  \( \text{(3.12.d)} \)

\[ \theta = \theta_w(X, \tau) \]  \( \text{(3.12.e)} \)

for \( X \geq 0, \ Y \to \infty \) and \( \tau > 0; \ U = U_0(X, \tau) \)  \( \text{(3.12.f)} \)

\[ \theta = \theta_0(X, \tau) \]  \( \text{(3.12.g)} \)

If the heat flux at the wall is specified, the thermal boundary condition given by equation (3.12.e) becomes

\[ -\left( \frac{\partial \theta}{\partial Y} \right)_{Y=0} = Q_w(X, \tau) \]  \( \text{(3.12.h)} \)
The dimensionless initial condition is

\[ \tau = 0; \quad U = U_0(X,Y) \quad (3.13.a) \]
\[ V = V_0(X,Y) \quad (3.13.b) \]
\[ \theta = \theta_0(X,Y) \quad (3.13.c) \]

B. FINITE DIFFERENCE FORM

1. System of Space Grid Points

To obtain the solutions to a set of partial differential equations by the finite difference approximations, it is necessary to establish a system of space grid points in the domain of interest. The space grid used in this work is shown in Figure 2. The interval \( \Delta Y \) between two neighboring grid points along a line where \( X = X_i \) is assumed constant. However, this constant varies with the value of \( i \) as follows:

\[ \Delta Y = A, \quad \text{for} \ 1 \leq i \leq m_2 \quad (3.14.a) \]
\[ \Delta Y = 2A, \quad \text{for} \ m_2 < i \leq m \quad (3.14.b) \]

where \( i, m \) and \( m_2 \) are positive integers, and 'A', any positive constant. Likewise, the interval \( \Delta X_i \) between two points \((i-1,1)\) and \((i,1)\) on the X-axis also depends on the value of \( i \). Thus

\[ \Delta X_i = f_n(i), \quad \text{for} \ 2 \leq i \leq m \quad (3.15) \]

where \( f_n(i) \) is any real function of \( i \), its value is positive and increases monotonously with \( i \).

All grids are smaller in the neighborhood of the stagnation point.
Figure 2. System of space grid points.
and become larger in the downstream region. The reason of choosing such nonuniform grid size will be explained in Section (B-8) of this chapter.

Figure 2 also shows a "stair" shaped domain of interest. This is specially designed for boundary-layer problems.

The space grid point \((1,1)\) corresponds to the origin of the coordinate system described in Part (A) of this chapter. The relations between the coordinate point \((X_1,Y_1)\) and the space grid point \((i,j)\) are given as follows:

\[
X_1 = 0, \quad Y_1 = 0 \quad (3.16.a)
\]

\[
X_i = \sum_{k=2}^{i} \Delta X_k, \quad \text{for } 2 \leq i \leq m \quad (3.16.b)
\]

\[
Y_j = (j-1)\Delta Y, \quad \text{for } \begin{cases} 2 \leq j \leq S+1 & \text{if } 1 \leq i < m_1 \\ 2 \leq j \leq 2S+1 & \text{if } m_1 \leq i \leq m \end{cases} \quad (3.16.c)
\]

where \(i, j, m, m_1\) and \(S\) are all positive integers. Special attention is paid to the following two points: firstly, \(\Delta Y\) in equation \((3.16.c)\) takes two different values according to equations \((3.14)\); secondly, for a given set of \(\Delta X_i\) and \(\Delta Y\), the integers \(m, m_1, m_2\) and \(S\) are so chosen that the corresponding "stair" shaped space domain covers the entire boundary layer flow.

Besides the space variables \(X\) and \(Y\), time variable \(\tau\) is also involved in the study of an unsteady boundary-layer problem. Hence, in addition to the space grid, a uniform positive time step \(\Delta \tau\) will be specified. The relation between the time variable \(\tau_n\) and the time step \(\Delta \tau\) is
\[ T_{n} = n \cdot A \cdot U \]

where \( n \) is a positive integer or zero.

Based on the foregoing space and time intervals, the correspondence between the coordinate point \((X_i, Y_j, \tau_n)\) and the grid point \((i, j, n)\) is one to one. Thereafter, any physical property \( \phi \) at the coordinate \((X_i, Y_j, \tau_n)\) is uniquely described by \( \Phi_{i, j}^{n} \).

2. Approximation of Derivatives by Finite Differences

Let \( \phi(X, Y, \tau) \) be any function with continuous and bounded partial derivatives of sufficient high orders. By Taylor's series with remainder, it yields

\[ \Phi_{i, j+1}^{n} = \Phi_{i, j}^{n} + \Delta Y \left( \frac{\partial \phi}{\partial Y} \right)_{i, j}^{n} + \frac{1}{2} (\Delta Y)^2 \left( \frac{\partial^2 \phi}{\partial Y^2} \right)_{i, j+\delta}^{n} \]  

(3.18)

\[ \Phi_{i, j-1}^{n} = \Phi_{i, j}^{n} - \Delta Y \left( \frac{\partial \phi}{\partial Y} \right)_{i, j}^{n} + \frac{1}{2} (\Delta Y)^2 \left( \frac{\partial^2 \phi}{\partial Y^2} \right)_{i, j-\delta}^{n} \]  

(3.19)

where \( \delta \) is some number between 0 and 1, and it generally takes different values in the different terms unless specified. Dividing the foregoing equations by \( \Delta Y \), after some arrangements, they become

\[ \left( \frac{\partial \phi}{\partial Y} \right)_{i, j}^{n} \left( \frac{\partial \phi}{\partial Y} \right)_{i, j+\delta}^{n} = \frac{\Phi_{i, j+1}^{n} - \Phi_{i, j}^{n}}{\Delta Y} - \frac{1}{2} (\Delta Y)^2 \left( \frac{\partial^2 \phi}{\partial Y^2} \right)_{i, j+\delta}^{n} \]  

(3.20)

\[ \left( \frac{\partial \phi}{\partial Y} \right)_{i, j}^{n} \left( \frac{\partial \phi}{\partial Y} \right)_{i, j-\delta}^{n} = \frac{\Phi_{i, j-1}^{n} - \Phi_{i, j}^{n}}{\Delta Y} + \frac{1}{2} (\Delta Y)^2 \left( \frac{\partial^2 \phi}{\partial Y^2} \right)_{i, j-\delta}^{n} \]  

(3.21)

If Taylor's expansions of \( \phi(X, Y, \tau) \) in equations (3.18) and (3.19)
are carried out including the fourth term, they become

\[
\Phi_{i,j+1}^n = \Phi_{i,j}^n + \Delta Y \left( \frac{\partial^2 \Phi}{\partial Y^2} \right)_{i,j}^n + \frac{1}{2} \left( \Delta Y \right)^2 \left( \frac{\partial^3 \Phi}{\partial Y^3} \right)_{i,j}^n + \frac{1}{6} \left( \Delta Y \right)^3 \left( \frac{\partial^4 \Phi}{\partial Y^4} \right)_{i,j}^n \tag{3.22}
\]

\[
\Phi_{i,j-1}^n = \Phi_{i,j}^n - \Delta Y \left( \frac{\partial \Phi}{\partial Y} \right)_{i,j}^n + \frac{1}{2} \left( \Delta Y \right)^2 \left( \frac{\partial^2 \Phi}{\partial Y^2} \right)_{i,j}^n - \frac{1}{6} \left( \Delta Y \right)^3 \left( \frac{\partial^3 \Phi}{\partial Y^3} \right)_{i,j}^n \tag{3.23}
\]

Subtracting equation (3.23) from equation (3.22), then dividing the result with \( \Delta Y \), gives

\[
\left( \frac{\partial \Phi}{\partial Y} \right)_{i,j}^n = \frac{\Phi_{i,j+1}^n - \Phi_{i,j-1}^n}{2 \Delta Y} - \frac{\left( \Delta Y \right)^2}{12} \left( \frac{\partial^3 \Phi}{\partial Y^3} \right)_{i,j}^n + \frac{\left( \Delta Y \right)^3}{6} \left( \frac{\partial^4 \Phi}{\partial Y^4} \right)_{i,j}^n \tag{3.24}
\]

The first terms on the right-hand side of equations (3.20), (3.21) and (3.24) are called forward, backward and central FDAs* for the derivative \( (\partial \Phi/\partial Y)_{i,j}^n \), respectively. The difference between this derivative and its corresponding FDA is known as the truncation error of FDA. The truncation error of the central FDA involves the higher ordered derivatives of \( \Phi \) with respect to \( Y \) than that of the forward or backward FDA.

In general, the truncation error of any FDA approaches zero as the interval \( \Delta Y \) approaches zero. In this limiting case, FDA becomes the exact representation of the corresponding derivative.

Finally, it is easy to show that the second partial derivative of \( \Phi \) with respect to \( Y \) at \( (X_i, Y_j, \tau_n) \) can be represented by the following formula:

\[
\left( \frac{\partial^2 \Phi}{\partial Y^2} \right)_{i,j}^n = \frac{\Phi_{i+1,j}^n - 2\Phi_{i,j}^n + \Phi_{i-1,j}^n}{(\Delta Y)^2} - \frac{\left( \Delta Y \right)^2}{24} \left[ \left( \frac{\partial^3 \Phi}{\partial Y^3} \right)_{i,j}^n + \left( \frac{\partial^4 \Phi}{\partial Y^4} \right)_{i,j}^n \right] \tag{3.25}
\]

*Finite Difference Approximations.
3. Finite Difference Form of Boundary-Layer Equations

As is well-known, the boundary-layer equation given by (3.9) is valid only if there is no reverse flow within the boundary layer. Otherwise, the boundary layer becomes so thick that the third assumption in Part (A) of this chapter no longer holds. Therefore, the streamwise velocity $U$ will be restricted to zero or positive values. Consequently, problems involving instantaneous local separation are excluded from the present investigation.

Now equations (3.9) and (3.10) are considered at the coordinate $(x_i, y_j, \tau_n)$ and equation (3.11) at $(x_i, y_{j-1/2}, \tau_{n+1})$ as follows:

$$
\left( \frac{\partial U}{\partial t} \right)_{x_i,j} + \frac{U}{\partial x} \frac{\partial U}{\partial x} + \frac{v}{\partial y} \frac{\partial U}{\partial y} - \frac{\partial^2 U}{\partial y^2} = \left( \frac{\partial U}{\partial t} \right)_{i,j} + \left( \frac{V}{\partial x} \right)_{i,j} \tag{3.26}
$$

$$
\left( \frac{\partial \theta}{\partial t} \right)_{x_i,j} + \frac{U}{\partial x} \frac{\partial \theta}{\partial x} + \frac{v}{\partial y} \frac{\partial \theta}{\partial y} - \frac{1}{Pr} \left( \frac{\partial^2 \theta}{\partial y^2} \right)_{i,j} = 0 \tag{3.27}
$$

$$
\left( \frac{\partial U}{\partial x} \right)_{x_i,j-1/2} + \left( \frac{\partial V}{\partial y} \right)_{x_i,j-1/2} = 0 \tag{3.28}
$$

Note the use of forward, backward and central exact finite difference representations for the derivatives in the first, second and third terms on the left-hand side of equation (3.26), respectively. Furthermore, the last term on the left-hand side of the equation is represented by the formula given by equation (3.25), then PDE stated by equation (3.26) becomes

$$
\left( \frac{U_{i,j+1} - U_{i,j}}{\Delta t} \right) + \frac{U}{\partial x} \left( \frac{U_{i,j+1} - U_{i,j}}{\Delta x_i} \right) + \frac{v}{\partial y} \left( \frac{U_{i,j+1} - U_{i,j}}{\Delta y} \right) = \left( \frac{U_{i,j+1} - 2U_{i,j} + U_{i,j-1}}{\Delta Y^2} \right) \tag{3.29}
$$
where
\[
E_t = \frac{1}{2} \left\{ \Delta U \left( \frac{\partial U}{\partial t} \right)_{i,j} + \Delta X \left( \frac{\partial U}{\partial X} \right)_{i,j} \right\} + \frac{(\Delta Y)^2}{6} V_{i,j} \left[ \left( \frac{\partial U}{\partial Y} \right)_{i,j} + \left( \frac{\partial U}{\partial Y} \right)_{i,j} \right] - \frac{(\Delta Y)^2}{12} \left[ \left( \frac{\partial U}{\partial Y} \right)_{i,j} + \left( \frac{\partial U}{\partial Y} \right)_{i,j} \right] \right\} 
\]  
(3.30)

Similarly, PDE given by equation (3.27) becomes
\[
\frac{\partial \bar{\theta}_{i,j}}{\partial t} + U_{i,j} \left( \frac{\partial \bar{\theta}_{i,j}}{\partial X} \right) + V_{i,j} \left( \frac{\partial \bar{\theta}_{i,j}}{\partial Y} \right) - \left( \frac{\partial k_{i,j}}{\partial \bar{\theta}_{i,j}} + \frac{\partial \bar{\theta}_{i,j}}{\partial \bar{\theta}_{i,j}} \right) \right\} = E_t 
\]  
(3.31)

where
\[
E_t = \frac{1}{2} \left\{ \Delta U \left( \frac{\partial \theta}{\partial t} \right)_{i,j} + \Delta X \left( \frac{\partial \theta}{\partial X} \right)_{i,j} \right\} + \frac{(\Delta Y)^2}{6} V_{i,j} \left[ \left( \frac{\partial \theta}{\partial Y} \right)_{i,j} + \left( \frac{\partial \theta}{\partial Y} \right)_{i,j} \right] - \frac{1}{Pr} \frac{(\Delta Y)^2}{12} \left[ \left( \frac{\partial \theta}{\partial Y} \right)_{i,j} + \left( \frac{\partial \theta}{\partial Y} \right)_{i,j} \right] \right\} 
\]  
(3.32)

It should be remarked that the first two terms on the right-hand side of equation (3.29) remain in differential form. These terms can easily be calculated once the functional form of the free stream velocity \( U_0 \) is known. In boundary-layer problems, \( U_0 \) is available from the corresponding inviscid flow problems.

Next equation (3.28) is expressed in finite difference form. An exact central finite difference representation will be used for the second term of the equation. The first term of the equation, namely, the derivative of \( U \) with respect to \( X \) at the grid point \((i,j-1/2, n+1)\) is first expressed by the mathematical average of the same derivatives at points \((i,j-1, n+1)\) and \((i,j, n+1)\), then each of these derivatives is replaced by
the backward finite difference form. The complete equation becomes then

\[
\frac{U_{i+1,j}^{n+1} - U_{i,j}^{n+1} + U_{i-1,j}^{n+1} - U_{i,j}^{n+1}}{2(\Delta X_i)} + \frac{V_{i,j+1}^{n+1} - V_{i,j}^{n+1}}{\Delta Y} = \bar{E}_t
\]  

(3.33)

where

\[
\bar{E}_t = -\frac{1}{4} \left\{ \Delta X_i \left[ \left( \frac{\partial^2 U}{\partial X^2} \right)_{i-\frac{1}{2},j} + \left( \frac{\partial^2 U}{\partial X^2} \right)_{i+\frac{1}{2},j-1} \right] + \Delta Y \left[ \left( \frac{\partial^2 U}{\partial X \partial Y} \right)_{i,j+1} - \left( \frac{\partial^2 U}{\partial X \partial Y} \right)_{i,j-1} \right] \right\}
\]

(3.34)

and \( \delta \) is some number between 0 and 1/2, its value may vary from term to term.

The above treatment of the continuity equation was first used by Wu (34). Its main advantage is that the normal velocity \( V_{1,j}^{n+1} \) for entire space domain can be directly calculated according to the order \( j = 2 \), \( 3,4, \ldots \) provided that the streamwise velocity \( U_{i,j}^{n+1} \) is known for entire space domain and the boundary condition \( V_{1,j}^{n+1} \) is given for all values of 'i'. In addition, \( \partial V/\partial Y \) term in the continuity equation is replaced by a central FDA which is a better approximation than a forward or backward FDA.

4. Numerical Computation Procedure

Equations (3.29), (3.31) and (3.33) are the exact finite difference representations of PDEs given by equations (3.26), (3.27) and (3.28), respectively. Hence, no truncation error has been introduced in these equations. Suppose now the space intervals \( \Delta Y, \Delta X_i \) and the time step \( \Delta \tau \) in equations (3.29), (3.31) and (3.33) are chosen to be sufficiently
small such that the terms $E_t$, and $\ddot{E}_t$ and $E_t$ in these equations become negligible. Hence, they can be approximated by the following PDAEs*, respectively:

\[
U_{i,j}^{n+1} = U_{i,j}^n + \Delta t \left( \frac{\partial U_i^n}{\partial t} + \frac{\partial U_i^n}{\partial X_i} - U_{i,j}^n \frac{U_{i,j}^n - U_{i,j+1}^n}{\Delta X_i} - U_{i,j}^n \frac{U_{i,j+1}^n - U_{i,j-1}^n}{2 \Delta Y} \right) \\
+ \left( \frac{U_{i,j}^n - 2 U_{i,j+1}^n + U_{i,j-1}^n}{(\Delta Y)^2} \right) 
\]  
(3.35)

\[
\theta_{i,j}^{n+1} = \theta_{i,j}^n + \Delta t \left( -U_{i,j}^n \frac{\partial \theta_{i,j}^n}{\partial X_i} - U_{i,j}^n \frac{\theta_{i,j}^n - \theta_{i,j+1}^n}{\Delta X_i} \\
+ \left( \frac{\theta_{i,j}^n - 2 \theta_{i,j+1}^n + \theta_{i,j+2}^n}{Pr (\Delta Y)^2} \right) \right) 
\]  
(3.36)

\[
V_{i,j}^{n+1} = V_{i,j}^{n+1} - \frac{\Delta Y}{2 (\Delta X_i)} \left( U_{i,j+1}^{n+1} - U_{i,j-1}^{n+1} + U_{i,j+1}^{n+1} - U_{i,j-1}^{n+1} \right) 
\]  
(3.37)

The truncation errors of PDAEs (3.35), (3.36) and (3.37) are $\Delta t \cdot E_t$, $\Delta t \cdot \dot{E}_t$ and $\Delta Y \cdot \ddot{E}_t$ respectively, where $E_t$, $\dot{E}_t$ and $\ddot{E}_t$ are given in equations (3.30), (3.32) and (3.34), correspondingly.

Directly interpreted from equation (3.12), the boundary conditions are

for $8+1 \geq j \geq 2$ and $n > 0$; $U_{1,j} = (U_0)_{1}^{n+1}$  
(3.38.a)

$\theta_{1,j} = (\theta_0)_{1}^{n+1}$  
(3.38.b)

for $m \geq 1 \geq 1$ and $n > 0$; $U_{i,1} = 0$  
(3.38.c)

$V_{1,1} = (V_0)_{1}^{n+1}$  
(3.38.d)

*Finite Difference Approximate Equations.
\[ \Theta_{i,j}^{n+1} = (\Theta_{n})_{i,j}^{n+1} \tag{3.38.e} \]
\[ U_{i,k}^{n+1} = (U_{o})_{i}^{n+1} \tag{3.38.f} \]
\[ \Theta_{i,k}^{n+1} = (\Theta_{o})_{i}^{n+1} \tag{3.38.g} \]
\[ \frac{-3\Theta_{i,i}^{n+1} + 4\Theta_{i-1,i}^{n+1} - \Theta_{i+1,i}^{n+1}}{2(\Delta Y)} = (\Theta_{w})_{i}^{n+1} \tag{3.38.h} \]

where \( k = S + 1 \), for \( m_1 > i \geq 1 \); \( k = 2S + 1 \), for \( m \geq i \geq m_1 \).

The initial condition, according to equations (3.13), is

\[ U_{i,j}^{o} = (U_{s})_{i,j} \tag{3.39.a} \]
\[ V_{i,j}^{o} = (V_{s})_{i,j} \tag{3.39.b} \]
\[ \Theta_{i,j}^{o} = (\Theta_{s})_{i,j} \tag{3.39.c} \]

where \((i,j)\) covers all the space grid points in the domain of interest.

FDAE given by equation (3.36) is called "explicit" because of the fact that the temperature variable \( \Theta \) at time step \((n+1)\) can be directly computed from this equation once the values of \( \Theta \), \( U \) and \( V \) at time step \( n \) are known and the boundary conditions of \( \Theta \) are specified. Suppose, instead of a forward FDAE, a backward FDAE were used for the first term on the left-hand side of equation (3.27), then the corresponding FDAE of equation (3.27) would be

\[ \Theta_{i,j}^{n} = \Theta_{i,j}^{n-1} + \Delta t \left[ -U_{i,j}^{n} \left( \frac{\Theta_{i,j}^{n} - \Theta_{i+1,j}^{n}}{\Delta X_{i}} \right) - V_{i,j}^{n} \left( \frac{\Theta_{i,j}^{n} - \Theta_{i,j+1}^{n}}{\Delta Y_{i}} \right) - \frac{\Theta_{i,j}^{n} - \Theta_{i,j-1}^{n}}{\Delta X_{i}} \right] \]
which is equivalent to

\[ \Theta_{i,j}^{n+1} = \Theta_{i,j}^n + \Delta t \left[ -U_{i,j}^n \left( \frac{\Theta_{i,j}^n - \Theta_{i,j}^{n+1}}{\Delta X_i} \right) - V_{i,j}^n \left( \frac{\Theta_{i,j}^n - \Theta_{i,j}^{n+1}}{2 \cdot \Delta Y} \right) + \frac{\Theta_{i,j}^n - 2 \Theta_{i,j}^{n+1} + \Theta_{i,j}^{n+2}}{\Pr (\Delta Y)^2} \right] \]

In this case, even the values of U and V at time step (n+1) are known, a large set of simultaneous algebraic equations must be solved in order to compute the temperature variable \( \Theta \) at time step (n+1) in terms of the values of \( \Theta \) at time step n. This kind of FDAE is called "implicit". In general, an iteration procedure is necessary to solve a large set of simultaneous algebraic equations on a digital computer. Therefore, it is clear that one advantage to choose an explicit system of FDAEs is to avoid time consuming iteration procedure. Equation (3.35) coupled with equation (3.37) is also an explicit FDAE.

Equations (3.35) through (3.39) are actually used in the numerical computation. The procedure of the numerical computation may be outlined as follows:

1. Determine the numerical values of \( m, m_1, m_2 \) and \( S \).
2. Choose a proper set of \( \Delta t, \Delta Y \) and \( \Delta X_i \).
3. Set \( n = 0 \), and compute \( U_{i,j}^n, V_{i,j}^n \) and \( Q_{i,j}^n \) according to the initial conditions given by equations (3.39) for all possible grid points \((i,j)\).
4. Set \( i = 2 \).
5. Compute \( U_{i,j}^{n+1} \) and \( \Theta_{i,j}^{n+1} \) according to equations (3.35) and (3.36) respectively for all possible values of 'j' except the two values \( j = 1 \) and \( j = S+1 \) (for \( m_1 > i > 2 \)) or \( j = 2S + 1 \) (for \( m > i > m_1 \)).
(6) Compute $V_{1,j}^{n+1}$ according to equation (3.37) and the boundary condition given by equation (3.38.d) for all the values of 'j'.

(7) Increase the value of 'i' by one.

(8) Repeat steps (5) through (7) until $i = m$ is included.

(9) Increase the value of 'n' by one.

(10) Repeat steps (4) through (9) until 'n' takes any desirable value.

In the fifth step, the stability conditions of FDAEs (3.35) and (3.36) should be examined before proceeding to the computation of $U_{1,j}^{n+1}$ and $\theta_{1,j}^{n+1}$. If they are satisfied, the computation can be continued as outlined above. Otherwise, equations are unstable and the computation should be stopped. In this case, a new set of $\Delta r$, $\Delta y$ and $\Delta x_i$ should be chosen such that the stability conditions are satisfied. A general definition of the "stability" of a FDE is given in the next section. The stability conditions of FDAEs (3.35), (3.36) and (3.37) are discussed in Section(B-6) of this chapter.

5. General Error Analysis

To justify the use of a finite difference method, various errors associated with the method must be examined first. For a detailed discussion on the errors involved in the approximation of finite difference method to steady, three-dimensional, compressible laminar boundary-layers, Raetz (35) may be referred to. Here these errors are classified as the truncation, instability, singular, rounding and machine errors.
The truncation error of a FDAE is defined as the difference between the FDAE and its corresponding PDE. The existence of this error introduces the problems of "consistency" and "convergence".

A FDAE is said to be consistent with its corresponding PDE if its truncation error vanishes when all the space intervals and the time step approach zero. In our case, FDAEs given by equations (3.35) through (3.37) satisfy this requirement.

A FDAE is said to be convergent if its solution approaches the solution of its corresponding PDE as all the space intervals and the time step approach to zero. Note that the consistency of a FDAE does not necessarily imply its convergence; the solutions of both FDAE and its corresponding PDE may not be unique and even may not exist. The convergence conditions of FDAEs (3.35) through (3.37) are studied in Section (B-7) of this chapter.

The instability errors are those due to the growth or amplification of any local errors (such as rounding errors) as the computation advances. They are directly related to the "stability" of the FDAE. A FDAE is called stable if its solution remains uniformly bounded as the computation advances indefinitely under a set of fixed space intervals. Thus, for a stable FDAE, if any local error is introduced in its solution at some-time level 'n' of the computation procedure, it must either remain bounded or decrease as the value of 'n' advances. Hence, if the decay of the local errors in a stable FDAE is faster than the generation of the local rounding errors, the computer solution of this FDAE will closely approxi-
mate its exact solution. On the other hand, when the instability exists, a small error tends to grow as the computation progresses and finally leads to a meaningless computer solution. The stability conditions of FDAEs (3.35) through (3.37) are discussed in Section (B-6) of this chapter.

The singular errors are due to the existence of mathematical singularities. At and near a singular point, the velocity or temperature changes so rapidly that the otherwise valid FDAs of their derivatives become inadequate. An example of this kind of error will be given in Section (B-6) of this chapter.

The rounding errors are those due to rounding the numbers. IBM 7090 digital computer, by which all numerical computations are carried out, has thirty-six binary bits in every single word. This gives an accuracy of eight decimal digits if a floating-point number system is used in the computer. Any number has decimal digits more than eight will therefore be rounded in the computer, that introduces the rounding error. Likewise, any arithmetic operation on the computer which leads to a number of more than eight decimal digits also introduces the rounding error. An estimate of the upper bounds of the local rounding errors introduced in the numerical computations for $U_{i,j}^{n+1}$, $\theta_{i,j}^{n+1}$ and $V_{i,j}^{n+1}$ according to FDAEs (3.35), (3.36) and (3.37), respectively, are given in Section (B-9) of this chapter.

The machine errors are those due to the incorrect functioning of the computer machine. The probability that such errors occur is very low.
6. Stability Analysis and Test

Here the stability theory is briefly reviewed first. In literature (36,37), Von Neumann's theory has been often used to study the stability conditions for the governing FDAEs of the nonlinear fluid flow problems. Strictly speaking, Von Neumann's stability theory, which may be found in References (38,39,40,41,42) is only valid for initial-value problems with linear, homogeneous governing differential equations. The theory predicts the necessary and sufficient conditions for the stability of a multi-level FDAE which approximates a differential equation of the above category.

John (43) studied the stability conditions of the following explicit FDAE:

$$\Phi_{l}^{n+1} = \sum_{k=-N}^{N} F_k(x,t,\Delta t) \cdot \Phi_{l+k}^{n} + \Delta t \cdot (\partial_x \Phi)^n_{l}$$  \hspace{1cm} (3.40)$$

which corresponds to the differential equation,

$$\frac{\partial \Phi}{\partial t} = a_0(x,t)\frac{\partial \Phi}{\partial x^2} + a_1(x,t)\frac{\partial \Phi}{\partial x} + a_2(x,t,\Phi)$$  \hspace{1cm} (3.41)$$

where $a_0 > 0$ and the functions $a_0, \partial a_0/\partial x, \partial^2 a_0/\partial x^2, a_1, \partial a_1/\partial x, a_2$ are bounded and uniformly continuous in all their arguments. Furthermore, $a_2$ satisfies a uniform Lipshitz condition in $\Phi$, that is

$$\left| a_2(x,t,\Phi_2) - a_2(x,t,\Phi_1) \right| \leq C \left| \Phi_2 - \Phi_1 \right|$$

for any values of $\Phi_2$ and $\Phi_1$.

Next the stability conditions for FDAEs (3.35), (3.36) and (3.37) are derived. First, these equations are rearranged in the following form:
\[
U_{i,j}^{n+1} = \left(1 - U_{i,j}^{n} \frac{(\Delta T)}{(\Delta X_i)} - 2 \frac{(\Delta T)}{(\Delta Y)^2} \right) U_{i,j}^{n} + \left( \frac{(\Delta T)}{(\Delta Y)^2} - V_{i,j}^{n} \frac{(\Delta T)}{Z(\Delta Y)} \right) U_{i,j}^{n} \\
+ \left( V_{i,j}^{n} \frac{(\Delta T)}{Z(\Delta Y)} + \frac{(\Delta T)}{(\Delta Y)^2} \right) U_{i-1,j}^{n} + \left( \frac{U_{i,j}^{n} (\Delta T)}{(\Delta X_i)} \right) U_{i-1,j}^{n} \\
+ \Delta T \left( \frac{\partial U_{i,j}^{n}}{\partial X_i} + \left( U_0 \frac{\partial U_{i,j}^{n}}{\partial X} \right)_{i,j} \right)
\]  

\[(3.42)\]

\[
\Theta_{i,j}^{n+1} = \left(1 - U_{i,j}^{n} \frac{(\Delta T)}{(\Delta X_i)} - 2 \frac{(\Delta T)}{\Pr(\Delta Y)^2} \right) \Theta_{i,j}^{n} + \left( \frac{(\Delta T)}{\Pr(\Delta Y)^2} - V_{i,j}^{n} \frac{(\Delta T)}{Z(\Delta Y)} \right) \Theta_{i,j}^{n} \\
+ \left( V_{i,j}^{n} \frac{(\Delta T)}{\Pr(\Delta Y)^2} \right) \Theta_{i-1,j}^{n} + \left( \frac{U_{i,j}^{n} (\Delta T)}{(\Delta X_i)} \right) \Theta_{i-1,j}^{n}
\]  

\[(3.43)\]

\[
V_{i,j}^{n+1} = V_{i,j}^{n} - \frac{(\Delta Y)}{2(\Delta X_i)} \left( U_{i,j}^{n} - U_{i-1,j}^{n} + U_{i,j}^{n} - U_{i-1,j}^{n} \right)
\]  

\[(3.44)\]

Then, it is assumed that

1. \(\Delta T\), \(\Delta Y\) and \(\Delta X_i\) are all positive quantities.

2. A finite time domain \(\tau_b\) is assumed. Thus, \(\tau_b \geq n\Delta t \geq 0\). Hence \(n \to \infty\) implies the limit \(\Delta t \to 0\). However, the converse is not true.

3. A finite space domain \(X_b\) by \(Y_b\) is considered. Furthermore, the space intervals \(\Delta Y\) and \(\Delta X_i\) are fixed and take finite but non-zero values during the numerical computation.

4. Flow separation is excluded. That is \(U_{i,j}^{n} \geq 0\).

5. The initial values \((U_0)_{i,j}\) and \((\Theta_0)_{i,j}\) are bounded by \(M_0\) and \(N_0\) respectively. Thus, according to equations (3.39), \(|U_{i,j}^{0}| \leq M_0\) and \(|\Theta_{i,j}^{0}| \leq N_0\) for all the space grid points \((i,j)\) under consideration.

6. The boundary values \((V_{w,i}^{n+1})\) given by equation (3.38.d) is
uniformly bounded by \((V_w)_b\). Thus, \(|V_{i,1}^{n+1}| \leq (V_w)_b\) for all possible values of 'i' and at any time level 'n'.

(7) The terms \((\partial U_0/\partial \tau)_i^n + (U_0 \partial U_0/\partial X)_i^n\) on the right-hand side of equation (3.35) represent dimensionless pressure gradient in the boundary layer. It is assumed that this pressure gradient remains finite in the interested space and time domains. Let its maximum value be \(P_o\), then

\[
\left| \left( \frac{\partial U_0}{\partial \tau} \right)_i^n + \left( U_0 \frac{\partial U_0}{\partial X} \right)_i^n \right| \leq P_o
\]

for all possible values of 'i' and at any time level 'n'.

Based on the above assumptions, the stability conditions for FDAEs (3.42), (3.43) and (3.44) are derived in the following theorem.

**Theorem:** The sufficient conditions for the uniform boundedness of \(|U_{i,j}^{n+1}|\) given by FDAE (3.42) and \(|\varphi_{i,j}^{n+1}|\) given by FDAE (3.43) are respectively,

\[
\begin{align*}
\frac{1}{U_{i,i}^n} + \frac{2}{(\Delta \tau)^2} & \geq \Delta \tau \\
\frac{2}{|V_{i,j}^n|} & \geq \Delta Y
\end{align*}
\]

(3.45.a)  (3.45.b)

and

\[
\begin{align*}
\frac{1}{U_{i,i}^n} + \frac{2}{Pr(\Delta \tau)^2} & \geq \Delta \tau \\
\frac{2}{Pr|V_{i,j}^n|} & \geq \Delta Y
\end{align*}
\]

(3.46.a)  (3.46.b)

Furthermore, if \(|U_{i,j}|\) is uniformly bounded, then \(|V_{i,j}^{n+1}|\) given by FDAE (3.44) is also uniformly bounded.
Proof: First, the uniform boundedness of $|U_{i,j}^{n+1}|$ is proved. According to FDAE (3.42),

$$
|U_{i,j}^{n+1}| \leq |1 - U_{i,j}^n| \frac{(\Delta t)}{\Delta x_i} - \frac{2(\Delta t)}{(\Delta y)^2} |U_{i,j}^n| + \left| \frac{(\Delta t)}{(\Delta y)^2} - V_{i,j}^n \right| \frac{(\Delta t)}{2(\Delta y)} |U_{i,j}^n| \\
+ |V_{i,j}^n| \frac{(\Delta t)}{2(\Delta y)} + \frac{(\Delta t)|}{(\Delta y)^2} |U_{i,j}^n| + |U_{i,j}^n| \frac{(\Delta t)}{(\Delta x_i)} |U_{i,j}^n| \\
+ |\Delta t| \left| \frac{\partial U_{i,j}^n}{\partial x_i} \right| + \left( U_{i,j}^n \frac{\partial U_{i,j}^n}{\partial y} \right)
$$

Let the maximum of $|U_{i,j}|$ be $M_n$ for all the space grid points $(i,j)$ under consideration. Hence, by the assumptions (1), (4), (7) and stability conditions (3.45), the above inequality can be written as

$$
|U_{i,j}^{n+1}| \leq \left(1 - U_{i,j}^n\right) \frac{(\Delta t)}{\Delta x_i} - \frac{2(\Delta t)}{(\Delta y)^2} M_n + \left( \frac{(\Delta t)}{(\Delta y)^2} - V_{i,j}^n \right) \frac{(\Delta t)}{2(\Delta y)} M_n \\
+ \left( V_{i,j}^n \frac{(\Delta t)}{2(\Delta y)} + \frac{(\Delta t)|}{(\Delta y)^2} \right) M_n + \left( U_{i,j}^n \frac{(\Delta t)}{(\Delta x_i)} \right) M_n + \Delta t \cdot P_0
$$

or

$$
|U_{i,j}^{n+1}| \leq M_n + \Delta t \cdot P_0
$$

Since $M_{n+1}$ is the maximum value of $|U_{i,j}^{n+1}|$ for all the space grid points $(i,j)$ under consideration, therefore,

$$
M_{n+1} \leq M_n + \Delta t \cdot P_0
$$

Successive application of the foregoing inequality until $n = 0$ gives

$$
M_{n+1} \leq M_0 + (n \Delta t) P_0
$$

or

$$
|U_{i,j}^{n+1}| \leq M_0 + (n \Delta t) P_0
$$
The second assumption implies \( n\Delta t \leq \tau_b \), therefore,

\[
|U_{i,j}^{n+1}| \leq M_0 + \tau_b \cdot P_0 \tag{3.47}
\]

Since \( M_0, \tau_b \) and \( P_0 \) are all finite and are independent of \( n \), hence, it is concluded that \( |U_{i,j}^{n+1}| \) is uniformly bounded. The uniform boundedness of \( |\phi_{i,j}^{n+1}| \) under the stability conditions given by equation (3.46) can be proved in a similar way.

Next, the uniform boundedness of \( |V_{i,j}^{n+1}| \) is proved from FDAE (3.44) as follows:

\[
|V_{i,j}^{n+1}| \leq |V_{i,j}^{n}| + \frac{\Delta Y}{2(\Delta x_i)} \cdot |U_{i,j}^{n+1} - U_{i,j}^{n}| + |U_{i,j}^{n+1} - U_{i,j}^{n+1}|
\]

\[
\leq |V_{i,j}^{n}| + \frac{\Delta Y}{2(\Delta x_i)} \cdot \left( |U_{i,j}^{n+1} - U_{i,j}^{n}| + |U_{i,j}^{n} - U_{i,j}^{n+1}| \right)
\]

The fourth assumption and inequality (3.47) imply

\[
|U_{i,j}^{n+1} - U_{i,j+1}^{n+1}| \leq \max \text{ of } U_{i,j+1}^{n+1} \text{ and } U_{i+1,j}^{n+1} \leq M_0 + \tau_b \cdot P_0
\]

\[
|U_{i,j}^{n+1} - U_{i,j}^{n}| \leq \max \text{ of } U_{i,j}^{n+1} \text{ and } U_{i,j}^{n+1} \leq M_0 + \tau_b \cdot P_0
\]

It follows then

\[
|V_{i,j}^{n+1}| \leq |V_{i,j}^{n+1}| + \frac{\Delta Y}{\Delta x_i} (M_0 + \tau_b \cdot P_0)
\]

Repeatedly applying the above inequality until \( j = 2 \) and also making use of the assumption (6), the following expression is obtained:

\[
|V_{i,j}^{n+1}| \leq (V_{w})_b + (j-1)\left(\frac{\Delta Y}{\Delta x_i}\right)(M_0 + \tau_b \cdot P_0)
\]
Furthermore noting from Figure 2 that \( j \leq 2s + 1 \), the above inequality may be rearranged in the form

\[
|V_{i,j}^{n+1}| \leq (V_w)_b + 2S \left( \frac{\Delta Y}{\Delta X_i} \right) (M_0 + T_b \cdot P_0)
\]

(3.48)

If \( \Delta Y/\Delta X_i \) is denoted by \( C_i \), the assumption (3) implies that \( C_i \) is a finite number. Then in terms of the maximum value \( C_o \) of \( C_i \) corresponding to all possible values of 'i', equation (4.48) becomes

\[
|V_{i,j}^{n+1}| \leq (V_w)_b + 2S \cdot C_o (M_0 + T_b \cdot P_0)
\]

Since \( (V_w)_b, S, C_o, M_o, T_b \) and \( P_0 \) are all finite and are independent of \( n \), therefore, \( |V_{i,j}^{n+1}| \) is uniformly bounded and the theorem is proved.

A test on the computer has been carried out for Blasius flow. A standard space domain shown in Figure 2 is used. Its dimensions are given by system no.1 in Table 1. The result shows how the size of time step \( \Delta \tau \) affects the amplification of the local errors. At first, Blasius solution is read into the computer as the initial condition. Then the numerical computation is carried out according to the procedure described in Section (B-4). Time step \( \Delta \tau \) is chosen to be 200. After 600 time steps, the numerical solutions of the local velocity become steady.

Actually, the criterion employed to determine the steadiness is

\[
\left| \frac{U_{i,j}^{600} - U_{i,j}^{599}}{U_{i,j}^{600}} \right| \leq 5.0 \times 10^{-5}
\]

for all the grid points \((i,j)\) in the above space domain. At this stage, 'n' is set to be 0, a new quantity \( Z_{i,j} \) is defined as follows,
\[ Z_{i,j} = U_{i,j}^0 \] 

(3.49.a)

Then a -5% error is artificially introduced in \( U_{3,12}^0 \) which takes a numerical value of \( 0.8115774 \) before the introduction of the error. Now denoting the instantaneous local errors by \( \epsilon_{i,j}^n \), it yields

\[ \epsilon_{i,j}^n = U_{i,j}^n - Z_{i,j} \] 

(3.49.b)

where the initial value of \( \epsilon_{i,j}^n \) are

\[ \epsilon_{3,12}^0 = -0.05 \times 0.8115774 = -4.0578870 \times 10^{-2} \]

and

\[ \epsilon_{i,j}^0 = 0 \]

for all the space grid points \((i,j)\) except the point \((3,12)\).

During the test, the stability condition given by equation (3.45.b) is assured, however, that given by equation (3.45.a) is not imposed.

First \( U_{i,j}^n \) is computed according to FDAEs (3.35) and (3.37) for \( \Delta \tau = 200, 250, 300 \) and 350. Then local error \( \epsilon_{3,12}^n \) is computed from equation (3.49.b) correspondingly. Its numerical results are shown in Figure 3. For cases \( \Delta \tau = 200 \) and 250, \( |\epsilon_{3,12}^n| \) becomes smaller and smaller, and finally vanishes as 'n' increases indefinitely. This implies, according to the definition of \( \epsilon_{i,j}^n \) given by equation (3.49.b), that \( U_{3,12}^n \) is uniformly bounded and approaches its steady value \( Z_{3,12} \) as 'n' increases. When \( \Delta \tau = 300, |\epsilon_{3,12}^n| \) first decays, and finally remains constant with a numerical value
Figure 3. Effects of the size of $\Delta \tau$ on the instability errors.
TABLE 1
SYSTEMS OF SPACE DOMAIN

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*ΔY = A for m₂ ≥ i ≥ 1, ΔY = 2A for m ≥ i > m₂.
**Series of ΔX₁ are listed in Table 2.

of 0.5 x 10⁻³ as 'n' increases. Hence, $U^n_{3,12}$ is still uniformly bounded but deviates from its steady value $Z^n_{3,12}$ by an amount of ± 0.5 x 10⁻³.

In the last case that $Δτ = 350$, $|e^n_{3,12}|$ is amplified indefinitely as 'n' increases. In fact, it becomes so large that the accumulator of the computer overflows before 'n' increases to 50. Consequently, $U^n_{3,12}$ becomes unbounded and is meaningless in the limit as $n \to \infty$.

It is important to note that, for $U^{n+1}_{3,12}$ to be uniformly bounded, the stability condition given by equation (3.45.a) requires $Δτ \leq 266$. This estimate of the limiting value of $Δτ$ agrees very well with the results obtained from the above test on the computer.

7. Convergence Analysis and Test

If a FDAE is consistent with its corresponding PDE, the proof of its convergence is reduced to the proofs of the uniqueness and the ex-
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istance of their solutions. For a general initial-value problem, such
a theory has not been available yet. However, for a properly posed
initial-value problem, if the governing PDEs are linear and homogeneous,
it is known \((39,41)\) that the corresponding FDEs are convergent if and
only if they are consistent and stable; here a problem is characterized
as "properly posed" if a solution of governing PDE exists and depends
uniquely and continuously on the initial values. John \((43)\) has treated
the convergence of the solutions of FDE \((3.40)\). The result that con-
sistency and stability imply convergence also holds in this case. Douglas
\((44)\) has studied the sufficient convergence conditions for the explicit
FDE which approximates the following nonlinear parabolic PDE,
\[
\frac{\partial \phi}{\partial t} = \mathcal{F}(x,t, \phi, \frac{\partial \phi}{\partial x}, \frac{\partial^2 \phi}{\partial x^2}).
\]
The convergence conditions for the implicit FDE which corresponds to
the following nonlinear PDE,
\[
\frac{\partial^2 \phi}{\partial x^2} = \mathcal{F}(x,t, \phi, \frac{\partial \phi}{\partial x}, \frac{\partial \phi}{\partial t})
\]
were investigated by Douglas \((45)\) and Lees \((46)\).

In the rest of the section, the sufficient conditions are derived
under which the solutions of FDEs \((3.35), (3.36)\) and \((3.37)\) respectively
converge to the solutions of equations \((3.29), (3.31)\) and \((3.33)\) as \(\Delta t, \Delta Y\)
and \(\Delta X_i\) approach zero. It is important to note that equations \((3.29),
(3.31)\) and \((3.33)\) are the exact finite difference representations of
PDEs \((3.26), (3.27)\) and \((3.28)\), respectively.
Now let $\tilde{U}_{i,j}^{n+1}$, $\tilde{V}_{i,j}^{n+1}$ be the solutions corresponding to the exact
finite difference equations (3.29) and (3.33) and $U_{i,j}^{n+1}$ and $V_{i,j}^{n+1}$ be the
solutions of FDEs (3.35) and (3.37). Then FDEs (3.35) and (3.37) are
called convergent if

$$\lim_{\Delta\tau, \Delta\gamma, \Delta X_i \to 0} \left| \tilde{U}_{i,j}^{n+1} - U_{i,j}^{n+1} \right| = 0$$

and

$$\lim_{\Delta\tau, \Delta\gamma, \Delta X_i \to 0} \left| \tilde{V}_{i,j}^{n+1} - V_{i,j}^{n+1} \right| = 0$$

for all space grid points $(i,j)$ and at any time level '$n'.

It is important to note that $\tilde{U}$ and $\tilde{V}$ are also the solutions of PDEs
(3.6) and (3.2). For a problem of physical significance, there is no
loss of generality by the assumption that the solution is a well-behaved
function. Therefore, it is assumed that

(1) The solutions $\tilde{U}$ and $\tilde{V}$ are uniformly bounded and their partial
derivatives with respect to $\tau$, $X$ and $Y$ exist and are also bounded uniformly.

Particularly,

$$\left| \frac{\partial^2 \tilde{U}}{\partial \tau^2} \right| \leq 2K_1, \quad \left| \tilde{U} \frac{\partial \tilde{U}}{\partial X^2} \right| \leq 2K_2, \quad \left| \frac{\partial \tilde{U}}{\partial \gamma} \right| \leq 6K_3,$$

$$\left| \frac{\partial \tilde{U}}{\partial \gamma^2} \right| \leq 12K_3'' \quad \left| \frac{\partial \tilde{U}}{\partial X^2} \right| \leq 2K_4, \quad \left| \frac{\partial \tilde{U}}{\partial X \partial \gamma} \right| \leq 2K_5,$$

$$\left| \tilde{U} \frac{\partial \tilde{V}}{\partial Y} \right| \leq 24K_6 \quad \left| \frac{\partial \tilde{U}}{\partial X} \right| \leq L_1, \quad \left| \frac{\partial \tilde{V}}{\partial Y} \right| \leq L_2.$$

(2) The following two limits exist and are uniformly bounded:
\[
\lim_{\Delta X_i \to 0} \left| \frac{U_{i,j}^n - U_{i,j}^{n-1}}{\Delta X_i} \right| = \left| \left( \frac{\partial U}{\partial X} \right)_{i,j} \right| \leq L_1^n
\]
\[
\lim_{\Delta Y \to 0} \left| \frac{U_{i,j}^n - U_{i+1,j}^{n-1}}{2(\Delta Y)} \right| = \left| \left( \frac{\partial U}{\partial Y} \right)_{i,j} \right| \leq L_2^n
\]

for all possible grid points \((n,i,j)\).

Based on these assumptions and the assumptions made in the previous section (except that \(\Delta Y\) and \(\Delta X_i\) are now allowed to approach zero), the convergence conditions for FDAEs (3.35) and (3.37) are stated in the following theorem.

**Theorem:** The stability conditions given by equations (3.45) are the sufficient conditions for the convergence of FDAEs (3.35) and (3.37).

**Proof:** Rewriting PDE (3.6) in the form,

\[
\frac{\partial U}{\partial t} = \psi(X, \tau, \bar{U}, \bar{V}, \bar{U}_X, \bar{U}_Y, \bar{U}_{YY})
\]

and

\[
\psi = \frac{\partial U_0}{\partial t} + U_0 \frac{\partial U}{\partial X} + \bar{U} \bar{U}_X - \bar{V} \bar{U}_Y + \bar{U}_{YY},
\]

(3.50)

where

\[
\bar{U}_X = \frac{\partial \bar{U}}{\partial X}, \quad \bar{U}_Y = \frac{\partial \bar{U}}{\partial Y}, \quad \bar{U}_{YY} = \frac{\partial^2 \bar{U}}{\partial Y^2},
\]

the equations (3.29) and (3.33), which are the exact finite difference representations of PDEs (3.6) and (3.2), can respectively be transformed into the following inequality forms:
\[ \tilde{U}_{i,j}^{n+1} \leq \tilde{U}_{i,j}^n + \Delta t \cdot \psi(X_i, T_n, \tilde{U}_{i,j}^n, \tilde{V}_{i,j}^n, \eta_{i,j}^n, \xi_{i,j}^n, \zeta_{i,j}^n) + \Delta t \left( K_1(\Delta t) + K_2(\Delta X_i) + (K_3 + K_3')(\Delta Y)^2 \right) \]  
(3.51)

\[ \tilde{V}_{i,j}^{n+1} \leq \tilde{V}_{i,j}^{n+1} - \frac{(\Delta Y)}{2(\Delta X_i)} \left( \tilde{U}_{i,j}^{n+1} - \tilde{U}_{i,j+1}^{n+1} + \tilde{U}_{i,j+1}^{n+1} - \tilde{U}_{i,j-1}^{n+1} \right) + \Delta Y \left( K_4(\Delta X_i) + K_5(\Delta Y) + K_6(\Delta Y)^2 \right) \]  
(3.52)

where

\[ \eta_{i,j}^n \equiv \frac{\tilde{U}_{i,j}^n - \tilde{U}_{i,j-1}^n}{\Delta X_i} \]  
(3.53.a)

\[ \xi_{i,j}^n \equiv \frac{\tilde{U}_{i,j+1}^n - \tilde{U}_{i,j-1}^n}{2(\Delta Y)} \]  
(3.53.b)

\[ \zeta_{i,j}^n \equiv \frac{\tilde{U}_{i,j+1}^n - 2\tilde{U}_{i,j}^n + \tilde{U}_{i,j-1}^n}{(\Delta Y)^2} \]  
(3.53.c)

Here it should be remarked that the first assumption was used during the transformation from equations (3.29) and (3.33) to inequalities (3.51) and (3.52) respectively.

Similarly, FDAEs (3.35) and (3.37) can be written as follows:

\[ U_{i,j}^{n+1} = U_{i,j}^n + \Delta t \cdot \psi(X_i, T_n, U_{i,j}^n, V_{i,j}^n, \eta_{i,j}^n, \xi_{i,j}^n, \zeta_{i,j}^n) \]  
(3.54)

\[ V_{i,j}^{n+1} = V_{i,j}^{n+1} - \frac{(\Delta Y)}{2(\Delta X_i)} \left( U_{i,j}^{n+1} - U_{i,j-1}^{n+1} + U_{i,j+1}^{n+1} - U_{i,j-1}^{n+1} \right) \]  
(3.55)

where

\[ \eta_{i,j}^n \equiv \frac{U_{i,j}^n - U_{i,j-1}^n}{\Delta X_i} \]  
(3.56.a)
\[
\xi^{n}_{i,j} \equiv \frac{U_{i,j,n}^{n} - U_{i,j}^{n}}{2(\Delta Y)} \\
\xi^{n}_{i,j} \equiv \frac{U_{i,j,n}^{n} - 2U_{i,j}^{n} + U_{i,j}^{n-1}}{(\Delta Y)^2}
\]

Defining \(x^{n+1}_{i,j}\) and \(y^{n+1}_{i,j}\) as
\[
\alpha^{n+1}_{i,j} \equiv \tilde{U}_{i,j}^{n+1} - U_{i,j}^{n+1},
\]
\[
\beta^{n+1}_{i,j} \equiv \tilde{V}_{i,j}^{n+1} - V_{i,j}^{n+1},
\]

and subtracting equation (3.54) from inequality (3.51) yield
\[
\alpha^{n+1}_{i,j} \leq \alpha^{n}_{i,j} + \Delta t \left( \psi_{2} - \psi_{1} \right) + \Delta t \left( K_{1} (\Delta \gamma) + K_{2} (\Delta \beta) + (K_{3} + K_{n}^{2}) (\Delta Y)^{2} \right)
\]

where \(\psi_{2}\) and \(\psi_{1}\) are the function \(\psi\) evaluated at the points \(A_{2}, (X_{n}, \tau_{n}, U_{i,j}^{n}, V_{i,j}^{n}, \eta_{i,j}^{n}, \xi_{i,j}^{n}, \tilde{\eta}_{i,j}^{n}, \tilde{\xi}_{i,j}^{n})\), and \(A_{1}, (X_{1}, \tau_{n}, U_{i,j}^{n}, V_{i,j}^{n}, \eta_{i,j}^{n}, \xi_{i,j}^{n}, \tilde{\eta}_{i,j}^{n}, \tilde{\xi}_{i,j}^{n})\), respectively.

Using the mean values, the following equation is obtained:
\[
\psi_{2} - \psi_{1} = \frac{\partial \psi}{\partial \bar{U}} \left( \tilde{U}_{i,j}^{n} - U_{i,j}^{n} \right) + \frac{\partial \psi}{\partial \bar{V}} \left( \tilde{V}_{i,j}^{n} - V_{i,j}^{n} \right) + \frac{\partial \psi}{\partial \tilde{\eta}} \left( \tilde{\eta}_{i,j}^{n} - \eta_{i,j}^{n} \right) + \frac{\partial \psi}{\partial \tilde{\xi}} \left( \tilde{\xi}_{i,j}^{n} - \xi_{i,j}^{n} \right)
\]

\[
= - \frac{\partial \psi}{\partial \bar{U}} \left( \tilde{U}_{i,j}^{n} - U_{i,j}^{n} \right) - \frac{\partial \psi}{\partial \bar{V}} \left( \tilde{V}_{i,j}^{n} - V_{i,j}^{n} \right) + \frac{\partial \psi}{\partial \tilde{\eta}} \left( \tilde{\eta}_{i,j}^{n} - \eta_{i,j}^{n} \right) + \frac{\partial \psi}{\partial \tilde{\xi}} \left( \tilde{\xi}_{i,j}^{n} - \xi_{i,j}^{n} \right)
\]
where all the quantities in the brackets ( ) are evaluated at some point between $A_1$ and $A_2$. It follows, by the definitions given in equations (3.53), (3.56) and (3.57), that

$$
\Psi_2 - \Psi_1 = -\left(\bar{U}_G \right) \frac{\partial^n}{\partial x^n} + \left(\bar{V} \right) \left(\frac{\partial^n}{\partial y^n} \right) \left( \frac{\alpha^n_{i,j} - \alpha^n_{i,j+1}}{2(\Delta x)} \right) + \left( \frac{\alpha^n_{i,j} - 2\alpha^n_{i,j} + \alpha^n_{i,j-1}}{(\Delta y)^2} \right)
$$

(3.59)

Inserting equation (3.59) into (3.58) yields

$$
\alpha^n_{i,j} = \left( 1 - \frac{\bar{U}}{(\Delta x)^2} \right) \alpha^n_{i,j} + \left( \frac{\bar{T}}{(\Delta y)^2} \right) \alpha^n_{i,j+1} + \left( \frac{\bar{V}}{2(\Delta y)} \right) \alpha^n_{i,j-1} - \Delta T \left( \bar{U}_G \right) \alpha^n_{i,j} + \Delta T \left( K_1 \bar{U} + K_2 \bar{U}_G + K_3 + K_4 \bar{V} \right)
$$

(3.60)

If $\alpha_n$ and $\beta_n$ are the lowest upper bounds of $\alpha^n_{i,j}$ and $\beta^n_{i,j}$, respectively, then,

$$
\text{Max.} |\alpha^n_{i,j}| = \alpha_n
$$

(3.61.a)

and

$$
\text{Max.} |\beta^n_{i,j}| = \beta_n
$$

(3.61.b)

for all the grid points $(i,j)$ under consideration. Furthermore, let

$$
\text{Max.} \beta^n_k = \beta''_n
$$

(3.62)

for $k = 0, 1, 2, \ldots, n$. 
Now if
\[
\frac{1}{(\Delta X_i)} + \frac{2}{(\Delta Y)^2} \geq \Delta \tau, \tag{3.63.a}
\]
\[
\frac{2}{|\bar{V}|} \geq \Delta Y, \tag{3.63.b}
\]
then, by the definitions given in equations (3.61) and (3.62), inequality (3.60) can be expressed in the following form:
\[
|n_{i,j}^{n+1}| \leq \left[1 + \mathcal{L}_1(\Delta \tau)\right] |n_i^n| + \Delta \tau \left(\mathcal{L}_2 \bar{V}_n'' + K_1(\Delta \tau) + K_2(\Delta X_i) + (K_3 + K_3')(\Delta Y)^2\right)
\]
where
\[
\mathcal{L}_1 = \text{Max.} |\bar{U}_x| \tag{3.64.a}
\]
\[
\mathcal{L}_2 = \text{Max.} |\bar{U}_y| \tag{3.64.b}
\]
Since $|\bar{U}_x|$ takes a value between $|\tilde{\eta}_{i,j}^n|$ and $|\tilde{n}_{i,j}^n|$, from equation (3.53.a) and the first assumption made in this section, it may easily be seen that the maximum of $|\tilde{n}_{i,j}^n|$ is uniformly bounded whether $\Delta X_i$ approaches to zero or not. Similarly, from equation (3.56.a) and the second assumption, the maximum of $|\eta_{i,j}^n|$ remains uniformly bounded in the limit as $\Delta X_i$ and $\Delta Y$ approach zero. When $\Delta X_i$ and $\Delta Y$ are finite, the conditions (3.63) imply that $|U_{i,j}^n|$ is uniformly bounded, and $|\tilde{n}_{i,j}^n|$ still remains uniformly bounded. Consequently, $\mathcal{L}_1$ is a finite positive constant. The same is true with $\mathcal{L}_2$.

Since $|n_{i,j}^{n+1}|$ has a lowest upper bound $\alpha_{n+1}$, hence,
\[
|n_{i,j}^{n+1}| \leq \left[1 + \mathcal{L}_1(\Delta \tau)\right] |n_i^n| + \Delta \tau \left(\mathcal{L}_2 \bar{V}_n'' + K_1(\Delta \tau) + K_2(\Delta X_i) + (K_3 + K_3')(\Delta Y)^2\right) \tag{3.65}
\]
Applying the above inequality successively until \( n = 0 \) gives

\[
\mathcal{X}_{n+1} \leq \mathcal{X} \cdot e^{\tau_n T_b} + \mathcal{L}_b e^{\tau_n T_b} \left( \mathcal{L}_2 \beta'' + K_1 (\Delta t) + h_1 (\Delta x) + (k_3 t k_2 (\Delta x))^2 \right)
\] (3.66)

The detailed derivation of this inequality is given in Appendix I.

Since

\[
\alpha_0 = M_{\text{max}} \left| U_{t,i,j} - U_{t,i+1,j} \right| = M_{\text{max}} \left| U_{t,i,j} - U_{t,i+1,j} \right| = 0
\]

in the limit as \( \Delta t, \Delta x, \) and \( \Delta y \) approach zero, inequality (3.66) becomes

\[
\mathcal{X}_{n+1} \leq \left( \mathcal{L}_2 T_b \right) \beta''
\] (3.67)

Next the inequality for \( \beta_{n+1}^{t,j} \) which was defined in equation (3.57.b) is established. Subtracting equation (3.55) from inequality (3.52) results in

\[
\mathcal{L}_{i+1,j}^{n+1} \leq \beta_{i+1,j}^{n+1} - \frac{(\Delta y) (U_{i+1,j} - U_{i,j})}{\Delta x_i} \left( U_{i+1,j} - U_{i,j} \right) + \frac{(\Delta y) (U_{i+1,j} - U_{i,j})}{\Delta x_i} \left( U_{i+1,j} - U_{i,j} \right)
\]

Then repeated use of this inequality until \( j = 2 \) gives

\[
\beta_{i,j}^{n+1} \leq \beta_{i,j}^{n+1} - \frac{(\Delta y) \sum_{k=2}^{j} \left( (U_{i+1,j} - U_{i+1,k}) - (U_{i+1,j} - U_{i+1,k}) \right) + (\Delta y) (U_{i+1,j} - U_{i+1,k})}{\Delta x_i} \left( U_{i+1,j} - U_{i+1,k} \right)
\]

or,

\[
\left| \beta_{i,j}^{n+1} \right| \leq \left| \beta_{i,j}^{n+1} \right| + (\Delta y) \left( Y_{n+1} + K_4 (\Delta x_i) + K_5 (\Delta y) + K_6 (\Delta y) \right)
\] (3.68)

where

\[
Y_{n+1} = \text{Max} \left| \frac{U_{i,j}^{n+1} - U_{i,j}^{n+1}}{\Delta x_i} \right|
\] (3.69)
for all the space grid points \((i,j)\) under consideration.

Since
\[
\beta_{i,j}^{n+1} = \tilde{V}_{i,j}^{n+1} - V_{i,j}^{n+1} = (V_w)_i^{n+1} - (V_w)_i^{n+1} = 0
\]
and
\[
(\Delta t) \Delta Y = Y_j \leq Y_b
\]
therefore, in the limit as \(\Delta t, \Delta x\) and \(\Delta Y\) approach zero, inequality (3.68) becomes
\[
|\beta_{i,j}^{n+1}| \leq Y_b Y_{n+1}
\]
or
\[
\lim_{\Delta t, \Delta x, \Delta Y \to 0} \beta_{i,j}^{n+1} \leq Y_b Y_{n+1}
\]
(3.70)
where
\[
\lim_{\Delta t, \Delta x, \Delta Y \to 0} Y_{n+1} \equiv \max \left| \frac{\partial (\tilde{U})}{\partial X} \right|_{j,i,j} - \left| \frac{\partial (\tilde{U})}{\partial X} \right|_{i,j}
\]
(3.71)
Applying mathematical induction to inequalities (3.67), (3.70)
and equation (3.71), and the use of
\[
\beta_o'' = \beta_o = \max |\beta_o| = \max |\tilde{V}_{i,j} - V_{i,j}| = \max |(V)_i,j - (V)_i,j| = 0,
\]
readily yields
\[
\lim_{\Delta t, \Delta x, \Delta Y \to 0} \left\{ \alpha_{n+1} = 0 \right\} \beta_{n+1} = 0
\]
at any time level 'n'.

According to equations (3.61) and (3.57), it may therefore be concluded that
\[
\lim_{\Delta t, \Delta x, \Delta Y \to 0} \left\{ \left| \tilde{U}_{i,j}^{n+1} - U_{i,j}^{n+1} \right| = 0 \right\}
\]
and
\[
\lim_{\Delta t, \Delta x, \Delta Y \to 0} \left\{ \left| \tilde{V}_{i,j}^{n+1} - V_{i,j}^{n+1} \right| = 0 \right\}
\]
for any point \((n,i,j)\).

The above proof of the convergence of FDAEs (3.35) and (3.37) is based on the assumptions given by inequalities (3.63). Hence, these assumptions are the sufficient conditions for the convergence of FDAEs (3.35) and (3.37). It should be noted that \((U)\) in inequality (3.63.a) takes some value between \(\tilde{U}_{i,j}^n\) and \(U_{i,j}^n\), while \((V)\) in inequality (3.63.b) takes some value between \(\tilde{V}_{i,j}^n\) and \(V_{i,j}^n\). Since the solution of any convergent FDAE can be made considerably close to the solution of corresponding PDE by choosing \(\Delta \tau\), \(\Delta X_i\) and \(\Delta Y\) sufficiently small, it may be assumed then

\[
\tilde{U}_{i,j}^n \approx U_{i,j}^n
\]

and

\[
\tilde{V}_{i,j}^n \approx V_{i,j}^n
\]

Accordingly, the convergence conditions given by inequalities (3.63) may be approximately written as

\[
\left\{\begin{array}{c}
\frac{1}{U_{i,j}^n} \geq \Delta \tau \\
\frac{2}{(\Delta X_i)^2} + \frac{2}{(\Delta Y)^2} \geq \Delta Y
\end{array}\right. \tag{3.72.a,b}
\]

which are identical to those given by inequalities (3.45). Therefore, the theorem is proved.

Similarly, it may be proved that the stability conditions given by inequalities (3.46) are the approximate sufficient conditions for the convergence of FDAE (3.36). This proof is omitted here.
Figure 4 shows how the solutions of FDAEs (3.35) and (3.37) approach the solutions of corresponding PDEs as the space grid size is refined. The solid curve in Figure 4 represents the Blasius solution $U$ versus the similarity parameter $Y/\sqrt{X}$. The broken curve no. 1 is the numerical solution $Z_{0,j}$ (for $j = 1, 2, \ldots, 41$) which was computed in the previous section and corresponds to the local velocity profile at $X = 19604.05$. The relatively large deviation of $Z_{0,j}$ from the Blasius solution is due to the local truncation errors. The broken curve no. 2 is the numerical solution of the local velocity profile at the same location after the size of $\Delta X_1$ has been refined. The deviation of this numerical solution from the Blasius solution is much smaller than that of the previous case. It is reasonable to assume that as the grid size is further reduced, the numerical solution of the Blasius flow converges to the Blasius solution.

8. Singular Errors

During the computation of the numerical solution for the broken curve no. 2 described in the previous section, a standard space domain shown in Figure 2 is adopted. Its dimensions are specified by system no. 2 in Table 1. The time step $\Delta \tau$ is taken to be 200, and the initial values of $U$ and $V$ are assumed to be Blasius solution. The steady state has been reached after 600 time steps. The numerical results of the steady local velocity profiles at five different locations are shown in Figure 5. At $X = 55386.61$, the numerical solution agrees very well
Figure 4. Effects of the size of $\Delta X_1$ on the convergence of solutions of FDAEs (3.35) and (3.37) for Blasius flow.
Figure 5. Effects of singular errors on numerical solutions of the local velocity profiles in Blasius flow.
with Blasius solution. However, as the values of $X$ decreases, the deviation of the numerical solution from the Blasius solution increases. Figure 6 shows that the above conclusion also holds for numerical solution of the local skin friction. The numerical computation of the local skin friction, $(\partial U / \partial Y)_{Y=0}$, is based on the following three-point Lagrangian interpolation formula (47):

$$\left(\frac{\partial U}{\partial Y}\right)_{Y=0} \approx \left(\frac{U_{i+1} - U_{i-1}}{2(\Delta Y)}\right)$$

Hereafter this formula is used for all numerical computations of the local skin friction and the local temperature gradient.

There are two reasons for the deviations mentioned above. Firstly, Blasius solution is not the exact solution of PDEs (3.26) and (3.28) at and near the origin $X = 0$. Secondly, under the current system of the grid size, FDEs (3.35) and (3.37) are not valid near $X = 0$ where a singularity exists. From the second reason, it is clear that the existence of singularity causes an unacceptable local truncation error. Complete elimination of these singular errors is impossible. However, by choosing a system of nonuniform grid size as described in Section (B-l) of this chapter, the singular errors due to the singularity at $X = 0$ can best be compensated.

A close examination of Figures 5 and 6 shows that, under the present space grid size the numerical solution of Blasius flow agrees quite well
Figure 6. Effects of singular errors on numerical solution of the local skin friction in Blasius flow.
with Blasius solution for \( X \geq 1.96 \times 10^4 \).* However, for \( X < 1.9 \times 10^4 \), the deviation from Blasius solution becomes considerable because of the presence of the singular errors. Therefore, in the examples to be given in Part (A) of Chapter IV, the numerical results in the domain that \( X < 1.96 \times 10^4 \) are not used.

9. Local Rounding Errors

The procedure of the numerical computation described in Section (B-4) of this chapter has been carried out on IBM 7090. Since IBM 7090 has only thirty-six binary bits in a single word, the aforementioned rounding errors are inevitable during the numerical computation. The only concern is that whether the rate of generation of the local rounding errors is higher than the rate of decay of the local errors or not. If it is, then the numerical solution to be obtained from the computer is meaningless even though FDAEs are stable and convergent. Thus becomes necessary to estimate the upper bounds of the local rounding errors during the numerical computations of \( U_{i,j}^{n+1} \), \( \Theta_{i,j}^{n+1} \) and \( V_{i,j}^{n+1} \) according to FDAEs (3.35), (3.36) and (3.37), respectively. However, before this estimate can be made, the maximum possible rounding error introduced in a single floating-point arithmetical operation must first be studied.

In the floating-point computation, each number \( z \) is represented

---

*If the main stream velocity is assumed to be 20 fps and air to be the medium of flow, the point \( X = 1.96 \times 10^4 \) corresponds to a location about 1.9 inches from the leading edge of the plate.
by an ordered pair 'a' and 'b' such that

\[ z = a \cdot 2^b, \]

where 'b' is called the exponent and takes an integer value, positive or negative; 'a' is called the fractional part and satisfies

\[ 0.5 \leq |a| \leq 1. \]

In IBM 7090, the word bits are so allocated that 'b' has eight binary bits and 'a' has twenty-seven binary bits. This gives an accuracy of eight decimal digits for 'a' or, accordingly, for z. Meanwhile 'b' is restricted by

\[ |b| \leq 127. \]

The above restriction on 'b' implies that IBM 7090 is only capable to take a number whose absolute value is smaller than \(2^{127}\).

Let \(fl(B)\) denote the computer result of the arithmetic operation 'B' according to the floating-point number system described in the previous paragraph. It is not difficult then to derive the following identities (46):

\[ fl\{x_i \pm y_j\} = (x_i \pm y_j)(1 + E_r) \]  
(3.73.a)

\[ fl\{x_i + x_j + \cdots + x_k\} = x_i(1 + \gamma_i) + x_j(1 + \gamma_j) + \cdots + x_k(1 + \gamma_k) \]  
(3.73.b)
\[
\begin{align*}
\mathcal{B}\{X_1 \cdot X_2\} &= X_1 \cdot X_2 (1 + E_r) & (3.73.c) \\
\mathcal{B}\{X_1 / X_2\} &= (X_1 / X_2) (1 + E_r) & (3.73.a) \\
\mathcal{B}\left(\frac{X_i \cdots X_{i+1}}{y_i \cdots y_{j-1}}\right) &= \left(\frac{X_i \cdots X_{i+1}}{y_i \cdots y_{j-1}}\right) (1 + \gamma_{i,j}) & (3.73.e)
\end{align*}
\]

where

\[
|E_r| \leq 2^{-27} & (3.74.a) \\
(1 - 2^{-27})^{i-1} \leq 1 + \gamma_i \leq (1 + 2^{-27})^{i-1} & (3.74.b) \\
(1 - 2^{-27})^{i+1-k} \leq 1 + \gamma_{k} \leq (1 + 2^{-27})^{i+1-k}, \text{ for } k = 2, 3, \ldots, i & (3.74.c)
\]

Since the number $2^{-27}$ is much smaller than unity, all the second and higher order power terms of $2^{-27}$ may be neglected in all binomial expansions in inequalities (3.74). Consequently, $\gamma_{i,j}$, $\eta_i$ and $\eta_k$ can be expressed as follows:

\[
\begin{align*}
\gamma_{i,j} &= (i+j-1)E_r & (3.75.a) \\
\eta_i &= (i-1)E_r \\
\eta_k &= (i+1-k)E_r, \text{ for } k = 2, 3, \ldots, i & (3.75.b)
\end{align*}
\]

Similarly, it is easy to verify that

\[
(a + c''E_r)(b + b''E_r) = a \cdot b + (|a| b'' + |b| a'')E_r & (3.76)
\]
where \(a^n\) and \(b^n\) are arbitrary constants.

The upper bounds of the local rounding errors introduced during the numerical computations of \(U_{i,j}^{n+1}, \Theta_{i,j}^{n+1}\) and \(V_{i,j}^{n+1}\) may now be estimated. Let \(U_{i,j}^{n+1}, \Theta_{i,j}^{n+1}\) and \(V_{i,j}^{n+1}\) be the exact solutions of FDAEs (3.35), (3.36) and (3.37), respectively, and the corresponding computer solutions be \(\tilde{U}_{i,j}^{n+1}, \tilde{\Theta}_{i,j}^{n+1}\) and \(\tilde{V}_{i,j}^{n+1}\). Therefore, according to FDAEs (3.35), (3.36) and (3.37),

\[
\tilde{U}_{i,j}^{n+1} = \begin{cases} \tilde{U}_{i,j}^n + \Delta T \left( (\frac{\partial U_i}{\partial t})^n_i + (U_0 \frac{\partial U_i}{\partial X})^n_i - \tilde{U}_{i,j}^n \left( \frac{\tilde{U}_{i,j}^n - \tilde{U}_{i,j}^n}{\Delta X_i} \right) \\ - \tilde{V}_{i,j}^n \left( \frac{\tilde{U}_{i,j}^n - \tilde{U}_{i,j}^n}{2 \Delta Y_i} \right) + \left( \frac{\tilde{U}_{i,j}^n - \tilde{U}_{i,j}^n}{\Delta Y_i} \right) \right) \end{cases} \tag{3.77}
\]

\[
\tilde{\Theta}_{i,j}^{n+1} = \begin{cases} \tilde{\Theta}_{i,j}^n + \Delta T \left( - \tilde{V}_{i,j}^n \left( \frac{\tilde{\Theta}_{i,j}^n - \tilde{\Theta}_{i,j}^n}{\Delta X_i} \right) - \tilde{V}_{i,j}^n \left( \frac{\tilde{\Theta}_{i,j}^n - \tilde{\Theta}_{i,j}^n}{2 \Delta Y_i} \right) \\ + \frac{\tilde{\Theta}_{i,j}^n - \tilde{\Theta}_{i,j}^n}{\Delta Y_i} \right) \right) \end{cases} \tag{3.78}
\]

\[
\tilde{V}_{i,j}^{n+1} = \begin{cases} \tilde{V}_{i,j}^n - \frac{\Delta Y_i}{2 \Delta X_i} \left( \tilde{U}_{i,j}^n - \tilde{U}_{i,j}^n + \tilde{U}_{i,j}^n - \tilde{U}_{i,j}^n \right) \end{cases} \tag{3.79}
\]

In above equations, we have assumed that the numerical values of \(\Delta T, \Delta X_i, \Delta Y_i, \text{Pr}, (\partial U_0/\partial t)_i^n\) and \((U_0 \partial U_0/\partial X)_i^n\) can exactly be read into the computer and no rounding for these numbers is needed.

The local rounding errors of \(U_{i,j}^{n+1}, \Theta_{i,j}^{n+1}\) and \(V_{i,j}^{n+1}\) are defined as the differences \((\tilde{U}_{i,j}^{n+1} - U_{i,j}^{n+1}), (\tilde{\Theta}_{i,j}^{n+1} - \Theta_{i,j}^{n+1})\) and \((\tilde{V}_{i,j}^{n+1} - V_{i,j}^{n+1})\) respectively provided that \(U_{i,j}^{n+1}, \Theta_{i,j}^{n+1}\) and \(V_{i,j}^{n+1}\) for all the space grid points \((i,j)\) can exactly be represented on the computer. In other words, assuming \(\tilde{U}_{i,j}^n = U_{i,j}^n, \tilde{\Theta}_{i,j}^n = \Theta_{i,j}^n\) and \(\tilde{V}_{i,j}^n = V_{i,j}^n\)
for all the space grid points \((i,j)\), it is desired to study the upper bounds of \(|\ddot{U}_{i,j}^{n+1} - U_{i,j}^{n+1}|, |\ddot{\Theta}_{i,j}^{n+1} - \Theta_{i,j}^{n+1}|\) and \(|\ddot{V}_{i,j}^{n+1} - V_{i,j}^{n+1}|\).

According to equations (3.77), (3.35) and the floating-point arith-
matic operation rules given by equations (3.73), (3.75) and (3.76), it may be estimated that

\[
\left| \ddot{U}_{i,j}^{n+1} - U_{i,j}^{n+1} \right| \leq (2^{27}) \left\{ U_{i,j}^n + \Delta t \left[ 6 \left( \frac{\partial U_{i,j}^n}{\partial x_i} \right) + \left( \frac{\partial U_{i,j}^n}{\partial y_j} \right) \right] + 8 \left| U_{i,j}^{n} \left( \frac{U_{i,j}^n - U_{i,j}^{n-1}}{\Delta x_i} \right) \right| \right.
\]
\[
+ 8 \left| V_{i,j}^{n} \left( \frac{U_{i,j}^{n+1} - U_{i,j}^{n-1}}{2 \cdot \Delta y_j} \right) \right| + \left( 7 \left| U_{i,j}^{n+1} + 16 U_{i,j}^n + 6 U_{i,j}^{n-1} \right| \right) \right\} \quad (3.80)
\]

Similarly,

\[
\left| \ddot{\Theta}_{i,j}^{n+1} - \Theta_{i,j}^{n+1} \right| \leq (2^{27}) \left\{ \Theta_{i,j}^{n} + \Delta t \left[ 7 \left| U_{i,j}^{n} \left( \frac{\Theta_{i,j}^{n} - \Theta_{i,j}^{n-1}}{\Delta x_i} \right) \right| + 8 \left| V_{i,j}^{n} \left( \frac{\Theta_{i,j}^{n} - \Theta_{i,j}^{n-1}}{2 \cdot \Delta y_j} \right) \right| \right. \right.
\]
\[
\left. + \frac{8 \left| \Theta_{i,j}^{n+1} + 18 \left| \Theta_{i,j}^{n} \right| + 7 \left| \Theta_{i,j}^{n+1} \right| \right|}{Pr \cdot (\Delta Y)^2} \right\} \quad (3.81)
\]

\[
\left| \ddot{V}_{i,j}^{n+1} - V_{i,j}^{n+1} \right| \leq (2^{27}) \left\{ \left( \frac{j-1}{2} \right) \left( V_{w,j}^{n+1} \right) + \frac{(\Delta Y)}{2 \cdot \Delta x_i} \sum_{k=2}^{\frac{j}{2} - k} \left[ (7 + j - k) \left( U_{i,k}^{n+1} + U_{i,k}^{n+1} \right) \right. \right.
\]
\[
\left. + (k + j - k) U_{i,k}^{n+1} + H_{i,k}^{n+1} + H_{i,k}^{n+1} + H_{i,k}^{n+1} + H_{i,k}^{n+1} \right] \right\} \quad (3.82)
\]

where \(H_{i,j}^{n}\) is defined as the quantity in the brackets \{ \} on the right-
hand side of equation (3.80).
CHAPTER IV
NUMERICAL EXAMPLES

A. OSCILLATIONS IN BLASIUS FLOW

1. Statement of the Problem

Here a fluctuating boundary layer on a stationary semi-infinite flat plate is considered. The fluctuations in the boundary layer are introduced by an unsteady main stream which oscillates periodically about a constant mean value. Let this mean value of the main stream velocity be \( u_\infty \), and the corresponding amplitude and the frequency of the oscillation be \( \Delta u_o \) and \( \omega \), respectively. Hence, the free stream velocity \( u_o \) of the problem takes the following form:

\[
U_o = u_\infty + \Delta u_o \sin(\omega t) \tag{4.1}
\]

Selecting \( u_\infty \) as the characteristic velocity of the problem, equation (4.1) may be rearranged in the dimensionless form,

\[
U_0 = 1 + \bar{U}_o \sin(W t) \tag{4.2}
\]

where \( \Delta U_o \) and \( W \), defined as \( \Delta u_o/u_\infty \) and \( \omega/(u_\infty^2/\nu) \), respectively, are the corresponding dimensionless amplitude and frequency of the free stream oscillation.

With \( U_o \) specified in equation (4.2), the governing FDAEs of the problem can easily be obtained from equations (3.35) and (3.37). The boundary conditions are given by equations (3.38.a), (3.38.c), (3.38.d)
and (3.38.f). Since neither blowing nor suction is present in the flow, the value of \( V_w \) in equation (3.38.d) is taken to be zero. The initial conditions are specified in equations (3.39.a) and (3.39.b) where \( U_g \) and \( V_s \) are assumed to be the steady Blasius solution.

The analytical solutions of the above problem were studied by many authors. In general, they can be classified into three groups, namely, high-frequency, intermediate-frequency and low-frequency solutions, according to the characteristic frequency parameter \( X.W \).

For high-frequency case \((X.W \geq 10)\), the problem was investigated by Lin (31) with a method described in Part (F) of Chapter II. According to Lin's result, the amplitude and the phase lag of the local oscillations depend only on the combined variable \( Y\sqrt{W/2} \) and is independent of \( X \). Hence, a single curve is sufficient to represent these quantities for all frequencies.

In the intermediate-frequency range \((10 > X.W. > 1)\), Hill and Stenning (49) studied the problem analytically and experimentally. The analytical method which they used is actually the method of successive approximations described in Part (D) of Chapter II. Lin's (31) results were taken as the solutions of the first order approximation, and the solutions of the second order approximation were computed. They found, in addition to variable \( Y\sqrt{W/2} \), that the amplitude and the phase lag of the local oscillations also vary with parameter \( X.W \).
For low-frequency case \((1 \geq X.W)\), the analytical investigations were carried out by Lighthill (33) and Nickerson (50). Lighthill linearized the governing PDEs by using the series solution method which was discussed in Part (C) of Chapter II. Only first two terms in the series expansions of the local velocity were considered, hence, the validity of the results are restricted to the small oscillations. The small oscillation in Blasius flow was also studied by Ghosh (51) for all frequency ranges. Again, only first two terms in the series expansions of the local velocity were calculated. As pointed out by Schlichting (22), the first two terms in the series solution are not sufficient to reveal the net effect of the free stream oscillation on the local velocities in the boundary layer.

2. Numerical Solution for High-Frequency Case

In this example, a standard space domain as shown in Figure 2 is adopted. The dimension of the space domain is specified by system no. 2 in Table 1. According to this specification, the maximum value of \(X\) is about \(8.22 \times 10^4\). Again the numerical solutions in the domain that \(X \leq 1.96 \times 10^4\) are not used, because of the significance of the singular errors introduced in this domain. The dimensionless frequency \(\dot{w}\) and amplitude \(\Delta U_o\) are chosen to be \(\pi \times 10^{-4}\) and 0.05, respectively.

*If the mean value of the main stream velocity, \(u_o\), is chosen to be 20 fps and air is the medium of the flow, \(W = \pi \times 10^9\) corresponds to a frequency of 125 cps.
Therefore, the characteristic frequency parameter $X.W$ in this example varies from 6.15 to 25.8 which covers most high-frequency range and the upper half of the intermediate-frequency range.

The time step $\Delta \tau$ is selected to be 200. According to the frequency specified in the previous paragraph, 100 time steps are needed to complete each cycle of the oscillation. The flow becomes approximately steady periodic after five cycles of oscillation from the steady Blasius flow, or equivalently, after 500 time steps. In fact,

$$\left| \frac{U_{400+i,j}^{400+n} - U_{400+i,j}^{300+n}}{U_{400+i,j}^{400+n}} \right| \leq 6.5 \times 10^{-3}$$

is satisfied for all grid points $(i,j)$ in space domain and for any $n$ between 0 and 100. The above numerical computations are carried out on IBM 7090 according to the program given in Appendix II-A. The total machine execution time is about twenty-five minutes.

The numerical solutions shown in Figures 7 and 8 are drawn during the computation of the local velocities at the sixth cycle. In order to draw the necessary information about the local velocity at every degree of the phase angle, the time step $\Delta \tau$ used in this cycle is changed to 55.5555556 and the computer program is modified slightly.

Figure 7 shows the numerical solutions of phase angle $\Theta$ of the local velocity component $U$ for $X.W = 25.8$ and 9.0. A comparison is made with Lin's (31) high-frequency solutions. The numerical solutions agree well with Lin's solutions for $X.W = 25.8$. For $X.W = 9.0$, which corresponds to a frequency a little lower than the lower limit of the high-frequency
Figure 7. Phase of high-frequency oscillations in Blastus flow.

High-Freq. Solution by C.C. Lin (31)

$\Delta XW = 9.0$ Numerical Solution ($\Delta U_0 = 0.05$)

- $XW = 25.8$
Figure 8. Amplitude of high-frequency oscillations in Blasius flow.
region, a slight deviation appears between the solutions, as expected. The same conclusion can be extended to the local amplitude ratio \(\Delta U/\Delta U_0\) according to Figure 8.

3. Numerical Solution for Intermediate-Frequency Case

The dimension of the space domain used in this example is specified by system no. 3 in Table 1. The numerical results bounded by the lines \(X = 2.1 \times 10^4\) and \(X = 10.2 \times 10^4\) will later be used for discussion. The frequency \(W\) and the amplitude \(\Delta U_0\) are chosen to be \(5.8177641 \times 10^{-5}\)* and 0.05, respectively. The corresponding characteristic frequency parameter \(XW\) varies from 1.22 to 5.94 which covers the lower half of the intermediate-frequency region.

The time step \(\Delta t\) is chosen to be 300, 360 time steps are needed for one cycle of the oscillation. In the fifth cycle, the flow is almost steady periodic, in fact,

\[
\left| \frac{U_{i,j}^{1440+n} - U_{i,j}^{1080+n}}{U_{i,j}^{1440+n}} \right| \leq 4.5 \times 10^{-3}
\]

for all grid points \((i,j)\) in the space domain and for any \(n\) between 0 and 360. The numerical results shown in Figures 9 through 16 are drawn during the computations of \(U\) and \(V\) in this cycle. From the first cycle to the fourth cycle, the program given in Appendix II-A is used. This

---

*If \(u_\infty\) is taken to be 20 fps and air is the medium of the flow, \(W = 5.817764 \times 10^{-5}\) corresponds to a frequency of 23.2 cps.
program is slightly modified in the fifth cycle in order to print out all the information needed for Figures 9 through 16. The total machine execution time is about ninety minutes.

Figures 9 and 10 show the numerical solutions of the phase angle $\Theta$ and the amplitude ratio $\Delta U/\Delta U_0$, respectively, for the characteristic frequency parameter $X.W = 5.06$. A comparison is made with Hill and Stenning's (49) intermediate-frequency solutions and experimental measurements. Both the numerical and the intermediate-frequency solutions slightly deviate from experimental measurements.

The numerical solutions together with the intermediate-frequency solutions and the experimental measurements of $\Theta$ and $\Delta U/\Delta U_0$ for $X.W = 2.45$ are shown in the Figures 11 and 12, respectively. As far as amplitude ratio $\Delta U/\Delta U_0$ is concerned, Figure 12 shows the agreement of the numerical and intermediate-frequency solutions with experimental data. However, for phase angle $\Theta$, Figure 11 shows numerical solutions agree with experimental data better than the intermediate-frequency solutions. For smaller values of the characteristic frequency parameter, say $X.W = 1.40$, Figure 13 shows that the deviation of the intermediate-frequency solution from the experimental data becomes more severe, while the agreement with the numerical solution is still excellent.

Figure 14 shows that the numerical solutions of the maximum phase lag have the same trend as the experimental measurements for the complete intermediate-frequency range, while the intermediate-frequency
Figure 9. Phase of intermediate-frequency oscillations in Blasius flow ($xW = 5.06$).

Numerical Solution ($\Delta U_0 = 0.05, xW = 5.06$)

Interm.-Freq. Solution (By Hill and Stenning (49))

Measurement
Figure 10. Amplitude of intermediate-frequency oscillations in Blasius flow ($X^* = 5.06$).

Numerical Solution ($\Delta U_0 = 0.05, X^* = 5.06$)

Interim.-Freq. Solution

By Hill and Stenning (49)

($\Delta U_0 = 0.09, X^* = 4.98$)

Measurement

$\nabla U_0 / \nabla U_0^0$ vs. $Y / W/2$
Figure 11. Phase of intermediate-frequency oscillations in Blasius flow ($W = 2.45$).
Figure 12. Amplitude of intermediate-frequency oscillations in Blasius flow ($XW = 2.45$).
Figure 13. Phase of intermediate-frequency oscillations in Blasius flow ($XW = 1.40$).
Figure 14. Maximum lagging phase angle of intermediate-frequency oscillations in Blasius flow.
Figure 15. Time-averaged velocity profiles of high and intermediate-frequency oscillations in Blasius flow.
Figure 16. Local velocity fluctuations of intermediate-frequency oscillations in Blasius flow ($W = 5.818 \times 10^{-5}$).
solutions start to deviate from them at $X.W = 3.0$. Since Hill and Stenning's intermediate-frequency solution is merely an improved Lin's high-frequency solution by a second order successive approximation, the deviation from the corresponding experimental data at small values of the characteristic frequency parameter is not surprising.

It is worth noting that the existing literature (18,31) observes the net streaming effects introduced due to the presence of the oscillation in Blasius flow is of order of $(\Delta U_o)^2$. Since $\Delta U_o$ used here is equal to 0.05, therefore, the difference between the time-averaged local velocity $\tilde{U}$ and the steady Blasius velocity profile must be negligibly small. The numerical solution shown in Figure 15 confirms this point. Furthermore, Figure 16 shows that the instantaneous local velocity computed by the numerical method oscillates sinusoidally about its time-averaged value.

B. IMPULSIVE START OF WEDGE FLOW

1. Numerical Solution of the Transient Velocity Profile

The boundary-layer flow along a wedge of $45^\circ$ half-angle is considered. The wedge is initially at rest relative to a stationary fluid and suddenly moves with a constant speed along its center-line. After the motion has started, the free stream velocity $u_o$ relative to a moving coordinate system described in Part (A) of Chapter III takes the steady value,

$$ u_o(x) = u_{oo} \left( \frac{x}{x_c} \right)^{\frac{1}{3}} $$

(4.3)
where \( x_c \) is an arbitrary characteristic length and \( u_\infty \) is the velocity of the fluid particle on the center-line of the wedge at distance \( x_c \) from the stagnation point of the wedge as shown in Figure 17.

![Figure 17. Wedge Flow](image)

Selecting \( x_c = \varphi / u_c \) and arbitrarily setting \( u_\infty = u_c / 45 \), the dimensionless free stream velocity becomes

\[
U_0(x) = \frac{x^{1/3}}{45}
\]  

(4.4)

The governing FDEs and the corresponding boundary and initial conditions of the problem can be obtained in the same way as described in Section (A-1) of this chapter, provided that \( U_s \) and \( V_s \) are replaced by \( U_0 \) and \( O \), respectively.

A standard space domain is adopted, its dimensions are specified by system no. 4 in Table 1. The time step \( \Delta \tau \) is chosen to be 50, 100 and 110 for \( n = 1 \) to 100, 101 to 800 and 801 to 1050, respectively, at \( n = 1050 \), the flow is considered to be steady, in fact,
\[
\left| \frac{U_{1050}^{i,j} - U_{1049}^{i,j}}{U_{1050}^{i,j}} \right| \leq 1.4 \times 10^{-4}
\]

is satisfied for all the grid points \((i,j)\) in the space domain. The numerical computation of this example is carried out according to the program given in Appendix II-B. The total machine execution time is about forty-five minutes.

The numerical solutions of the transient local velocity profile and local skin friction are shown in Figures 18 and 19, respectively. Neither analytical nor experimental solutions are available for these profiles. It is worth noting that the flow becomes fully developed in a very short period, say \(\tau = 10.25 \times 10^4\). Figure 20 shows the numerical solutions of the fully developed velocity profile at two different locations. For comparison, Hartree's (26) analytical solution versus the similarity parameter \(\Gamma\) is also plotted in this figure. The agreement is excellent.

2. Numerical Solution of the Transient Temperature Profile

In this example, the velocity field is assumed to be fully developed, the temperature of the fluid and the wedge is assumed to be uniform and equal to zero initially. The transient temperature profiles due to sudden raise of the surface temperature of the wedge from 0 to 1 will be studied.

*If air is considered as the medium of the flow and the characteristic velocity \(u_c\) is assumed to be 10 fps, then the dimensionless time \(\tau = 10.25 \times 10^4\) corresponds to a physical time of 0.164 second.
Figure 18. Development of the local velocity profile in wedge flow.
Figure 19: Development of the local skin friction in wedge flow.
Figure 20. Numerical solutions of the fully developed velocity profile in wedge flow.
The governing FDE is given in the equation (3.36) where Prandtl number Pr is chosen to be 0.75. The boundary conditions are specified by equations (3.38.b), (3.38.e) and (3.38.g) where \( \Theta_o = 0 \) and \( \Theta_w = 1 \). The initial condition is given by equation (3.39.c) where \( \Theta_s = 0 \). The velocity components \( U_{i,j}^n \) and \( V_{i,j}^n \) in equation (3.36) are assumed to be the steady numerical solutions of the previous example. The space domain is chosen to be the same as that used in the previous example.

The numerical computation of \( \Theta \) has been carried out for 1400 time steps. For the first 200 time steps, \( \Delta \tau \) is chosen to be 50; while for the last 1200 time steps, \( \Delta \tau \) is equal to 88. At \( n = 1400 \), the temperature profile is considered to be fully developed and the instantaneous local temperature satisfies the following relation:

\[
\left| \frac{\Theta_{i,j}^{1400} - \Theta_{i,j}^{39\frac{q}{q}}}{\Theta_{i,j}^{1400}} \right| \leq 5.0 \times 10^{-4}
\]

for all the space grid points \((i,j)\). The computer program given in Appendix II-C is used for the numerical computation of this example. Approximately a total of thirty-nine minute machine time is required for the entire calculation.

Figures 21 and 22 show the development of the local temperature profile and the local Nusselt number \( Nu \) which is defined as

\[
\frac{-(\partial \Theta/\partial Y)_{Y=0}}{\Theta_w - \Theta_o}
\]

In this case, no analytical solution is available for comparison.
Figure 21. Development of the local temperature profile in wedge flow.
Figure 22. Development of the local Nusselt number in wedge flow.
CHAPTER V
CONCLUSION

In the preceding chapters, a finite difference method for computing velocity and temperature profiles of an unsteady, incompressible, laminar boundary layer around a two-dimensional cylinder of arbitrary cross section is developed. Blowing or suction may be present on the wall of the cylinder. The thermal boundary condition at the wall may be either specified wall temperature or specified heat flux at the wall. Two examples, one oscillation in Blasius flow and the other impulsive start of wedge flow are given; the numerical results are compared with the existing analytical and experimental results. From these, the following conclusions are drawn.

(1) The method can be used to compute, with high accuracy, the velocity and temperature profiles of an unsteady boundary-layer flow of the foregoing category up to separation point.

(2) The method can easily be extended to determine the velocity and temperature profiles at the separation point, where a temporary reverse flow may occur, provided that a forward FDA, instead of a backward FDA, is used for the derivative in the second term on the left-hand side of equation (3.26).

(3) At and near a singular point, the numerical solutions given by the method are doubtful because of the singular errors.

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(4) For periodic flow problems, if the period of oscillation is very long or the frequency is very low, a large number of time steps are needed to complete one cycle of oscillation because of the limitation on $\Delta \tau$ imposed by the stability criterion. Therefore, it may not be advantageous to study oscillating flows of very low frequency by the present method.

(5) For starting-ending flow problems, which are difficult to investigate by analytical methods, the present method is indispensable. The method can also be used to study the starting-ending flow problems for large values of time. In this case, the problem becomes pure 'ending' type and its solution approaches the steady solution of the corresponding problem.
CHAPTER VI

HYDRODYNAMIC STABILITY

An attempt is made to simulate Schubauer and Skramstad's experiment by the present numerical method. Disturbances are introduced at a distance $X_a$ away from the leading edge, and they are investigated as they travel downstream. Since the thickness of boundary layer is quite uniform at the downstream of Blasius flow, a rectangular space domain bounded by straight lines $X = X_a$, $X_b$ and $Y = 0, Y_b$ as shown in Figure 23 is chosen. Space grids $\Delta X$ and $\Delta Y$ are selected to be uniform. For small disturbances, it is assumed that the disturbed velocity at $X_a$ is equal to the velocity of the base flow plus the disturbances introduced by the vibration of a ribbon at $X_a$. Thus

$$V_{x=X_a} = (V_B)_{x=X_a} - F(Y) \sin(W \tau)$$  \hspace{1cm} (6.1)

where subscript 'B' denotes the Blasius solution, function $F(Y)$ the effect of disturbances in the $Y$-direction and $W$ the dimensionless frequency of the vibration of ribbon. It readily follows from the equation of continuity that

$$U_{x=X_a} = (U_B)_{x=X_a} + (\Delta U)_{x=X_a} \sin(W \tau)$$  \hspace{1cm} (6.2a)

and

$$(\Delta U)_{x=X_a} = C \left( \frac{dF}{dY} \right)$$  \hspace{1cm} (6.2b)
Figure 23. Rectangular space domain.
where \( C \) is a constant proportional to the width of the vibrating ribbon.

Following Schubauer and Skramstad (52) the distribution function \( F(Y) \) and its derivative \( dF/dy \) are assumed to have the forms as indicated in Figure 24. This figure is used as boundary condition at \( X = X_a \). The rest of boundary conditions are

\[
Y = 0: \quad U = V = 0 \quad (6.3a)
\]

\[
Y = Y_b: \quad U = 1 \quad (6.3b)
\]

where \( Y_b \) is chosen to be reasonably greater than the maximum thickness of the boundary layer in the region considered. Blasius solution is used as initial condition. According to the boundary and initial conditions mentioned above, the disturbances at downstream of \( X_a \) can be calculated by the present numerical method.

Two cases are considered here. \( W \) is chosen to be \( 8.0 \times 10^{-5} \) in both cases. For the first case,

\[
X_a = 1.95 \times 10^5, \quad X_b = 2.65 \times 10^5, \quad Y_b = 7.50 \times 10^3, \\
\Delta X = 5000, \quad \Delta Y = 100, \quad \Delta \tau = 785.39816;
\]

and the distribution of \( \Delta U \) at \( X_a \) is given in Table 3. According to References (52,53,54), the foregoing case is in the unstable region. However, the numerical results drawn from the computation of \( U \) and \( V \) in the eleventh cycle of oscillation show, as indicated by Table 4, that the disturbances are rapidly damped out when they travel downstream. For
### TABLE 3
DISTRIBUTION OF DISTURBANCES AT $x_a$

<table>
<thead>
<tr>
<th>$j^*$</th>
<th>$\Delta U$</th>
<th>$j$</th>
<th>$\Delta U$</th>
<th>$j$</th>
<th>$\Delta U$</th>
<th>$j$</th>
<th>$\Delta U$</th>
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<td>59</td>
<td>-0.009</td>
</tr>
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<td>-0.009</td>
<td>76</td>
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</table>

$Y = (j-1)\Delta Y$.

![Figure 24. Function $F(Y)$ and $dF/dY$.](image-url)
TABLE 4

DAMP OF DISTURBANCES INTRODUCED AT $X = 1.95 \times 10^5$

| $Y$ | $|\Delta U|_{X=1.95 \times 10^5}$ | $|\Delta U|_{X=2.15 \times 10^5}$ | $|\Delta U|_{X=2.40 \times 10^5}$ | $|\Delta U|_{X=2.65 \times 10^5}$ |
|-----|---------------------------------|---------------------------------|---------------------------------|---------------------------------|
| 100 | $1.00 \times 10^{-2}$           | $1.09 \times 10^{-4}$           | $1.15 \times 10^{-5}$           | $4.50 \times 10^{-6}$           |
| 700 | $4.00 \times 10^{-2}$           | $1.05 \times 10^{-3}$           | $4.55 \times 10^{-4}$           | $1.60 \times 10^{-4}$           |

the second case,

$$X_a = 4.00 \times 10^5, \quad X_b = 5.00 \times 10^5, \quad Y_b = 9.00 \times 10^3,$$

$$\Delta X = 5000, \quad \Delta Y = 120, \quad \Delta \tau = 785.39816;$$

and the same distribution of $\Delta U$ at $X_a$ used in the first case is adopted here. According to References (52,53,54) the flow should be unstable. However the numerical results corresponding to the fourteenth cycle of oscillations (Table 5) indicate the stability of the flow.

TABLE 5

DAMP OF DISTURBANCES INTRODUCED AT $X = 4.00 \times 10^5$

| $Y$ | $|\Delta U|_{X=4.00 \times 10^5}$ | $|\Delta U|_{X=4.30 \times 10^5}$ | $|\Delta U|_{X=4.65 \times 10^5}$ |
|-----|---------------------------------|---------------------------------|---------------------------------|
| 120 | $1.00 \times 10^{-2}$           | $0.30 \times 10^{-5}$           | $0.23 \times 10^{-5}$           |
| 600 | $5.00 \times 10^{-2}$           | $1.20 \times 10^{-4}$           | $4.45 \times 10^{-5}$           |

Since Prandtl's approximations are certainly valid in describing the boundary layer flows, why the boundary layer equations yield stability
for apparently unstable cases is an interesting question to answer. To
explore the point further, the classical small perturbation theory of
stability is applied to the boundary layer formulation
\[
\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u}{\partial y^2}, \tag{6.4}
\]
\[
\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0. \tag{6.5}
\]
Thus introducing
\[
u = \bar{u} + u', \quad v = v'
\] (6.6)
into equations (6.4) and (6.5), and neglecting the higher order terms
gives
\[
\frac{\partial^2 \bar{u}}{\partial y^2} = 0, \quad \text{base flow} \tag{6.7}
\]
\[
\frac{\partial u'}{\partial t} + u \frac{\partial u'}{\partial x} + v' \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u'}{\partial y^2}, \quad \text{perturbation} \tag{6.8}
\]
clearly, a base flow that satisfies equation (6.7) and the boundary
conditions
\[
\bar{u}(0) = 0, \quad \bar{u}(\infty) = U_\infty \tag{6.9}
\]
is not possible. Therefore equation (6.8) cannot be made a characteristic-
value problem whose first characteristic-value is the critical Reynolds
number. Thus the comparison of the inviscous Orr-Sommerfeld equations
based on the Navier-Stokes and boundary layer equations and expressed in
terms of the usual nomenclature
\begin{align}
(\bar{u} - c)(\partial^2 \alpha^2) \phi - (D^2 \bar{u}) \phi &= 0 , \\
(\bar{u} - c) D^2 \phi - (D^2 \bar{u}) \phi &= 0 ,
\end{align}

(6.10)

(6.11)

reveals the importance of the missing terms

\begin{equation}
- (\bar{u} - c) \alpha^2 \phi
\end{equation}

(6.12)

in the latter equation. These terms are related to the terms

\begin{equation}
\frac{\partial v'}{\partial t} + \bar{u} \frac{\partial v'}{\partial x}
\end{equation}

(6.13)

of the y-momentum.

It may then be concluded that, although the vertical velocity component is a small quantity of order 1/Re, it produces a momentum transfer across the boundary layer where both the disturbance quantities and the mean flow properties vary rapidly. Thus the net effect on the transport process turns out to be much larger than the magnitude of the vertical velocity which produces the process.
APPENDIX I

DERIVATION OF INEQUALITY (3.66)

Rewriting inequality (3.65) in the form,
\[ \alpha_{k+1} \leq \left(1 + z_1 \Delta T \right) \alpha_k + \Delta T \left( z_2 p_n' + K_1 \Delta \tau + K_2 \Delta \xi_1 + (K_3 + K_3') \Delta Y \right) \]

and rearranging it by the help of equation (3.62) gives
\[ \alpha_{k+1} \leq \left(1 + z_1 \Delta T \right) \alpha_k + \Delta T \left( z_2 p_n' + K_1 \Delta \tau + K_2 \Delta \xi_1 + (K_3 + K_3') \Delta Y \right) \]  \hspace{1cm} (I.1)

for any \( k \leq n \).

Applying inequality (I.1) successively for \( k = n, n-1, \ldots, 0 \), yields

\[ \alpha_{n+1} \leq \left(1 + z_1 \Delta T \right) \alpha_0 + \Delta T \left( \left(1 + z_1 \Delta T \right)^n + \cdots + 1 \right) \cdot \left( z_2 p_n' + K_1 \Delta \tau + K_2 \Delta \xi_1 + (K_3 + K_3') \Delta Y \right) \]

or
\[ \alpha_{n+1} \leq \left(1 + z_1 \Delta T \right)^{n+1} \alpha_0 + \left(1 + z_1 \Delta T \right)^n \cdot \left( z_2 p_n' + K_1 \Delta \tau + K_2 \Delta \xi_1 + (K_3 + K_3') \Delta Y \right) \]  \hspace{1cm} (I.2)

Since the time domain \( T_b \) is finite,

\[ 0 \leq (n+1) \Delta T \leq T_b \]  \hspace{1cm} (I.3)

for any value of \( n \). It follows then

\[ \left(1 + z_1 \Delta T \right)^{n+1} \leq \left(1 + \frac{T_b}{n+1} \right)^{n+1} \leq e^{\frac{T_b}{n+1}} \]  \hspace{1cm} (I.4)
Finally, by the use of inequalities (I.3) and (I.4), inequality (I.2) may be rearranged to give

\[ \alpha_{n+1} \leq \alpha_0 \epsilon_{i-1} r_b + \tau_i \epsilon_{i-1} \epsilon_{i-1} \left( \frac{\Delta \lambda}{\tau_i} + k_1 \Delta \tau + k_2 \Delta \Delta \right) + (k_3 + k_4) (\Delta \gamma)^2 \]

which is the statement of inequality (3.66).
APPENDIX II

COMPUTER PROGRAMS
A. CALCULATION OF VELOCITY OF OSCILLATION IN BLASIUS FLOW

$ COMPILE MAD, PUNCH OBJECT

READ FORMAT VA,NA,MB,N1,N2,T1,T2,T3,T4,Y1,Y2,DT,E,W,DX(2)...DX(38)
VECTOR VALUES VA=$815/5F12.1/(10F6.2)*$
PAUSE NO. 1

BACK
SET HIGH DENSITY TAPE 9
REWIND TAPE 9
READ BCD TAPE 9, VR, TN1(1)...TN(7)
VECTOR VALUES VR=$7C6*$
WHENEVER TN(3).NE.18CDEFG*$
PRINT COMMENT $WRONG TAPE$*$
PRINT ON LINE FORMAT $WRONG TAPE MOUNTED PLEASE CORRECT**$
SKIP6.$
UNLOAD TAPE 9
PAUSE NO. 77777$*$
TRANSFER TO BACK
END OF CONDITIONAL
EXECUTE SKIP..(T1,T2,9)
READ BINARY TAPE 9,N, U(1,1)...U(38,41), V(1,1)...V(38,41)
AA(1)=0.$
AB(1)=0.$

NA
N=N+1
SINA=SIN.(N*DT)
COSA=COS.(N*(N-1)*DT)
UDU=E*K*COSA
DY=1
DYP=DX.P2
DYM=2./DY
DYP2=2./DYP
THROUGH S1,FOR J=2,1,J.G.41
AA(J)=1.+E*SINA
THROUGH S2, FOR I=2,1,J.G.6
THROUGH S21, FOR J=2,1,J.G.20
WHENEVER .ABS.V(I,J).G.DYM.OR.DT.G.1./U(I,J)/DX(I)+DYP2$1.
1 OR.U(I,J)=0.$
PRINT RESULTS 1,J,V(I,J), V(I,J), DT,DX(I), DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I,J)-DT*(U(I,J)-U(I,J-1))/DX(I)+V(I,J)*1
1U(I,J+1)-U(I,J-1)/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1)+DYP)+D0+U(I,J)
V(I,J)=V(I,J-1)-(AB(J-1)-AA(J-1)+AB(J)-AA(J))/DYM/DX(I)

S21
END OF CONDITIONAL
THROUGH S2,FOR J=2,1,J.G.20
U(I-1,J)=AA(J)

S2
AA(J)=AB(J)
THROUGH S3,FOR I=7,1,J.G.15
THROUGH S31,FOR J=2,1,J.G.40
1 OR.U(I,J)=0.$
PRINT RESULTS 1,J,N, U(I,J), V(I,J), DT,DX(I), DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I,J)-DT*(U(I,J)*(U(I,J)-U(I-1,J)))/DX(I)+V(I,J)*
1 U(I,J-1)-U(I,J-1))/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1))/DYP)
2 *DT+UDD
   V(I,J)=V(I,J-1)-(AB(J-1)-AA(J-1)+AB(J)-AA(J))/DYD/DX(I)
   END OF CONDITIONAL
   THROUGH S3, FOR J=2,1,J,G,40
   U(I-1,J)=AA(J)
   AA(J)=AB(J)
   DY=Y2
   DYP=DY,P,2
   DYD=2./DY
   DYM=2.*DY
   DYP2=2./DYP
   THROUGH S4, FOR J=1,1,J,G,21
   AC(J)=AA(2*J-1)
   U(15,J)=U(15,2*J-1)
   THROUGH S5, FOR J=22,1,J,G,40
   AC(J)=AA(41)
   U(15,J)=U(15,21)
I=16
   THROUGH S6, FOR J=2,1,J,G,40
   WHENEVER ABS.V(I,J).G.DYD.OR.DT.G.1./U(I,J)/DX(I)+DYP2)
   1 .OR.U(I,J).L.0.
   PRINT RESULTS I,J,N,U(I,J),V(I,J),DT,DX(I),DY
   OTHERWISE
   AB(J)=U(I,J)-DT*(U(I,J)*F(U(I,J)-U(I-1,J))/DX(I)+V(I,J)*
1 U(I,J+1)-U(I,J+1))/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1))/DYP)
2 *DT+UDD
   V(I,J)=V(I,J-1)-(AB(J-1)-AC(J-1)+AB(J)-AC(J))/DYD/DX(I)
   END OF CONDITIONAL
   THROUGH S7, FOR J=2,1,J,G,40
   U(I-1,J)=AA(J)
   AA(J)=AB(J)
   THROUGH S8, FOR I=17,1,I,G,38
   THROUGH S81, FOR J=2,1,J,G,40
   WHENEVER ABS.V(I,J).G.DYD.OR.DT.G.1./U(I,J)/DX(I)+DYP2)
   1 .OR.U(I,J).L.0.
   PRINT RESULTS I,J,N,U(I,J),V(I,J),DT,DX(I),DY
   OTHERWISE
   AB(J)=U(I,J)-DT*(U(I,J)*F(U(I,J)-U(I-1,J))/DX(I)+V(I,J)*
1 U(I,J+1)-U(I,J+1))/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1))/DYP)
2 *DT+UDD
   V(I,J)=V(I,J-1)-(AB(J-1)-AC(J-1)+AB(J)-AC(J))/DYD/DX(I)
   END OF CONDITIONAL
   THROUGH S9, FOR J=2,1,J,G,40
   U(I-1,J)=AA(J)
   AA(J)=AB(J)
   THROUGH S9, FOR J=2,1,J,G,40
   U(I,J)=AA(J)
   THROUGH S10, FOR I=2,1,I,G,6
   U(I,21)=AA(41)
   THROUGH S11, FOR I=7,1,I,G,38
   U(I,41)=AA(41)
   WHENEVER N.E.N1
   PRINT RESULTS N,U(I,1).....U(I,41)
   WHENEVER M.A.E.0
   EXECUTE SKIP.(T3,T4,9)
   OTHERWISE
HACKSPACE RECORD OF TAPE 9
END OF CONDITIONAL
WRITE BINARY TAPE 9,N,U(1,1)\ldots U(38,41),V(1,1)\ldots V(38,41)
END OF FILE TAPE 9
TRANSFER TO SE
OR WHENEVER N/N INTEGER MB
TRANSFER TO NA
OR WHENEVER N/N INTEGER MB
MB=N/N INTEGER
PRINT RESULTS N,U(25,1)\ldots U(27,41),U(36,1)\ldots U(38,41)
TRANSFER TO NA
END OF CONDITIONAL
DIMENSION U(1558,V8),V(1558,V8),AA(41),AB(41),AC(41),DX(38),TN(7)
VECTOR VALUES V8=2,1,41
INTEGER I,J,N,N1,N2,TN,MA,MB,T1,T2,T3,T4
UNLOAD TAPE 9
END OF PROGRAM
B. CALCULATION OF VELOCITY OF IMPULSIVE START OF WEDGE FLOW

$ COMPILE MAC, PUNCH OBJECT

READ FORMAT VA, MA, MB, N1, N2, T1, T2, T3, T4, Y1, Y2, DT, DX(2) ... DX(30)
VECCTR VALUES VA=$B15, 3F12.I/(1OF6.2) **$
PAUSE NO. 1
BACK
SET HIGH DENSITY TAPE 9
REWIND TAPE 9
READ BCD TAPE 9, VR, TN(1) ... TN(7)
VECCTR VALUES VR=$C6 **$
WHENEVER TN(3) NE. $BCDEFG$
PRINT COMMENT $WRONG TAPE$
PRINT ON LINE FORMAT WRONG
VECCTR VALUES WRONG=$H$WRONG TAPE MOUNTED PLEASE CORRECT**$
SKIP6.*
UNLOAD TAPE 9
PAUSE NO. 7777
TRANSFER TO BACK
END OF CONDITIONAL
EXECUTE SKIP..(T1, T2, 9)
READ BINARY TAPE 9, N, U(I, 1) ... U(30, 45), V(I, 1) ... V(30, 45),
1 UDU(2) ... UDU(30)
NA
N=N1
ZERO.(AA(1) ... AA(22), AB(1))
DY=Y1
DYP=CY.P.2
DYD=Z.2.DY
NMY=Z.2.DY
DYP2=2.2.DY
I=2
THROUGH S11, FOR J=2, 1, G.22
WHENEVER .ABS.V(I, J).G.DYD.OR.DTG.1./((U(I, J)/DX(I)+DYP2)
1. OR.U(I, J).L.0.
PRINT RESULTS I, J, N, U(I, J), V(I, J), DT, DX(I), DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I, J)-DT*(U(I, J)+U(I, J)-U(I-1, J))/DX(I)+V(I, J)*
1 U(I, J)+U(I, J-1))/DYM-2.*U(I, J)+U(I, J-1))/DYP
2 +DT*UDU(1)
V(I, J)=V(I, J-1)+(-AB(J-1)-AA(J-1)+AB(J)-AA(J))/DYD/DXI(I)
END OF CONDITIONAL
THROUGH S1, FCR=J=2, 1, G.22
U(I-1, J)=AA(J)
S1
AA(J)=AB(J)
SPRAY.(U(2, 23), AA(23) ... AA(45))
THROUGH S2, FCR=I=3, 1, G.7
THROUGH S21, FOR J=2, 1, G.46
WHENEVER .ABS.V(I, J).G.DYD.OR.DTG.1./((U(I, J)/DX(I)+DYP2)
1. OR.U(I, J).L.0.
PRINT RESULTS I, J, N, U(I, J), V(I, J), DT, DX(I), DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I, J)-DT*(U(I, J)+U(I, J)-U(I-1, J))/DX(I)+V(I, J)*
1 U(I, J)+U(I, J-1))/DYM-2.*U(I, J)+U(I, J-1))/DYP
2 +DT*UDU(1)
V(I, J)=V(I, J-1)+(-AB(J-1)-AA(J-1)+AB(J)-AA(J))/DYD/DXI(I)
END CF CONDITIONAL
THROUGH S2, FOR J=-1,2,1,J,G.44
U(I-1,J)=AA(J)
AA(J)=AB(J)
CYP=CY.*2
LYD=2./DY
CYM=2.*DY
CYP2=2.*DYP
AA(45)=U(7,45)
THROUGH S3, FOR J=-1,1,J,G.23
AC(J)=AA(2*J-1)
U(I,J)=U(I,J-1)
SPRAY.(U(I,J),AC(24),...AC(45),U(7,24),...U(7,45))
I=8
THROUGH S5, FOR J=-2,1,J,G.44
WHENEVER ABS(V(I,J).G.DYD.UR.DT.G.1./U(I,J)/DX(I)+DYP2)
1+DR.(U(I,J),L.O.,
PRINT RESULTS I,J,N,U(I,J),V(I,J),DT,DX(I),DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I,J)-DT*(U(I,J)*U(I,J)-U(I-1,J))/DX(I)/V(I,J)*
1(U(I,J+1)-U(I,J-1))/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1))/DYP
2+CT*U(I,J)
V(I,J)=V(I,J-1)-(AB(J-1)-AC(J-1)+AB(J)-AC(J))/DYD/DX(I)
END CF CONDITIONAL
THROUGH S6, FOR J=-2,1,J,G.44
U(I-1,J)=AA(J)
AA(J)=AB(J)
THROUGH S7, FOR I=-9,1,1,I,G.30
THROUGH S71, FOR J=-2,1,J,G.44
WHENEVER ABS(V(I,J).G.DYD.UR.DT.G.1./U(I,J)/DX(I)+DYP2)
1+DR.(U(I,J),L.O.,
PRINT RESULTS I,J,N,U(I,J),V(I,J),DT,DX(I),DY
TRANSFER TO SE
OTHERWISE
AB(J)=U(I,J)-DT*(U(I,J)*U(I,J)-U(I-1,J))/DX(I)/V(I,J)*
1(U(I,J+1)-U(I,J-1))/DYM-(U(I,J+1)-2.*U(I,J)+U(I,J-1))/DYP
2+DT*U(I,J)
V(I,J)=V(I,J-1)-(AB(J-1)-AA(J-1)+AB(J)-AA(J))/DYD/DX(I)
END CF CONDITIONAL
THROUGH S7, FOR J=-2,1,J,G.44
U(I-1,J)=AA(J)
AA(J)=AB(J)
THROUGH S8, FOR J=-2,1,J,G.44
U(30,J)=AA(J)
WHENEVER N.E.N1
PRINT RESULTS N,U(1,1) ... U(30,45)
WHENEVER MA.E.O
EXECUTE SKIP.(T3,T4,9)
OTHERWISE
BACKSPACE RECORD OF TAPE 9
END CF CONDITIONAL
WRITE BINARY TAPE 9,N,U(1,1) ... U(30,45),V(1,1) ... V(30,45),
1 UDU(2)... UDU(30)
END CF FILE TAPE 9
LR WHENEVER H/NZ,E.MB
TRANSFER TO NA
CR WHENEVER H/NZ,G.MB
#B=H/NZ
PRINT RESULTS N, U(25, 1), ..., U(25, 45), U(30, 1), ..., U(30, 45)
TRANSFER TC NA
EAC CF CONDITIONAL
CIPERSION U(1350, VB), V(1350, VB), DX(30), AA(45), AB(45), AC(45), TN(7),
L = U(30)
VECTRK VALUES VB = 2, 1, 45
INTEGER I, J, N, NL, N2, IN, MA, MB, T1, T2, T3, T4
CALGAD TAPE 9
END CF PROGRAM
C. CALCULATION OF TEMPERATURE OF WEDGE FLOW AFTER SUDDEN DROP OF WALL TEMPERATURE

$ COMPilee MAD, PUNCH OBJECT

FTRAP.
READ FORMAT VA,MA,MB,N1,N2,T1,T2,T3,T4,Q1,PR,Y1,Y2,DT,DX(2)
1...DX(30)
VECTOR VALUES VA=$B15,5F6.1/(10F6.2)**$;
PAUSE NO. 1
BACK
SET HIGH DENSITY TAPE 9
REWORK TAPE 9
READ BCD TAPE 9,VR,TN(1)...TN(7)
VECTOR VALUES VR=$766**$
WHenever TN(3).NE.88CD0FG$;
PRINT COMMENT $WRONG TAPE$;
PRINT ON LINE FORMAT WRONG
VECTOR VALUES WRONG=$SHWRONG TAPE MOUNTED PLEASE CORRECT**$;
SKIP6.
UNLOAD TAPE 9
PAUSE NO. 77777
TRANSFER TO BACK
END OF CONDITIONAL
EXECUTE SKIP,(T1,T2,9)
READ BINARY TAPE 9,N,U(1,1)....U(30,45),V(1,1)....V(30,45),
1 Q(1,1)....Q(30,45)
N=N+1
SPRAY,TQ1,AA(2)....AA(22))
DY=Y1
DYP=DY,P,2*PR
JYM=2,*DY
I=2
THROUGH $11$, FOR J=2,1,JG.22
AB(J)=Q(I,J)-DT*(U(I,J)*(Q(I,J)-Q(I-1,J))/DX(I)+V(I,J)*)
1 U(I,J)-Q(I-1,J-1))/DYM-(Q(I,J+1)-2.*Q(I,J)+Q(I,J-1))/DYP)
THROUGH $1$, FOR J=2,1,JG.22
Q(I-1,J)=AA(J)
$1$
AA(J)=AB(J)
SPRAY,TQ1,AA(23)....AA(45))
THROUGH $21$, FOR I=3,1,IG.7
THROUGH $21$, FOR J=2,1,JG.44
S21
AB(J)=Q(I,J)-DT*(U(I,J)*(Q(I,J)-Q(I-1,J))/DX(I)+V(I,J)*)
1 Q(I,J-1)-Q(I,J-1)/DYM-(Q(I,J+1)-2.*Q(I,J)+Q(I,J-1))/DYP)
THROUGH $21$, FOR J=7,1,JG.44
Q(I-1,J-1)=AA(J)
$2$
AA(J)=AB(J)
DY=Y2
DYP=DY,P,2*PR
DYM=2,*DY
THROUGH $31$, FOR J=1,1,JG.23
Q(T,J)=Q(T,2,J-1)
SPRAY,(Q(T,23),Q(T,24)....Q(T,45))
THROUGH $41$, FOR I=8,1,IG.30
THROUGH $41$, FOR J=2,1,JG.44
$41$
AB(J)=Q(I,J)-DT*(U(I,J)*(Q(I,J)-Q(I-1,J))/DX(I)+V(I,J)*)
1 Q(I,J+1)-Q(I,J-1)/DYM-(Q(I,J+1)-2.*Q(I,J)+Q(I,J-1))/DYP)
THROUGH $41$, FOR J=2,1,JG.44
Q(I-1,J)=AA(J)
S4  AA(J)=AA(J)
    THROUGH S5, FOR J=2,1,J,G.44
S5  U(30,J)=AA(J)
    WHENEVER N.E.,N1
    PRINT RESULTS N,Q(2,1),...Q(30,45)
    WHENEVER MA.E.,O
    EXECUTE SKIP.(T3,T4,9)
    OTHERWISE
    BACKSPACE RECORD OF TAPE 9
    END OF CONDITIONAL
    WRITE BINARY TAPE 9,N,U(1,1),...U(30,45),V(1,1),...V(30,45)
1  ,Q(1,1),...Q(30,45)
    END OF FILE TAPE 9
    TRANSFER TO SE
    OR WHENEVER N/N2.E.,MB
    TRANSFER TO NA
    OR WHENEVER N/N2.G.,MB
    MA=N/N2
    PRINT RESULTS N,Q(25,1),...Q(25,45),Q(30,1),...Q(30,45)
    TRANSFER TO NA
    END OF CONDITIONAL
    DIMENSION U(1350,VB),V(1350,VB),Q(1350,VB),DX(30),AA(45),AB(45),
1  TN(T)
    VECTOR VALUES VB=2,1,45
    INTEGER J,J,N,N1,N2,TN,MA,MB,T1,T2,T3,T4
SE  UNLOAD TAPE 9
    END OF PROGRAM
REFERENCES


