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COLLEGE OF ENGINEERING
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Final Report

PIPELINE CALCULATIONS FOR GAS-LIQUID SYSTEMS

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SUMMARY AND CONCLUSIONS

This study has considered the transporting of light hydro-carbon liquids by dissolving them in high pressure natural gases. The presence of the higher molecular weight material increases the deviation of the gas mixture from ideal gas behavior and increases the gas density in the pipeline but with only minor viscosity increases. Thus, the addition of liquids like propane to natural gases increases the flow capacity of the line and thereby reduces the cost of transportation.

A computer program was written to solve the pipeline flow equation, to calculate the horsepower of compression of the gas, and to determine the cost of building and operating the line. This program was capable of finding the minimum cost of transportation for a given pipe strength, line diameter, gas composition, and flow rate.

The phase relationships for gas-liquid systems were studied. The minimum pressure to maintain the mixtures in a single phase was found for various systems of natural gases and light hydro-carbon liquids. The pipeline pressure was always maintained at pressures above this minimum value in the design calculations.

The transportation costs were calculated for natural gas and nine mixtures containing condensable liquids for a series of conditions. The parameters studied were

Composition and corresponding minimum pressure

Steel strength: 65,000 and 100,000 psi

Pipe diameter: 16, 24, and 30 inch

Pressure: 700 to some 3600 psia

Flow rate: 200-1500 million cu. ft./day.

The important factors used in the study and the costs of transportation of the mixtures (single phase) in cents/100 miles/Mcf are given in Tables 5, 6A, and 6B, attached.

It was found that the transportation cost decreased with increasing flow rate, increasing pressure level to some 2500 psia, and increasing molecular weight of the gas mixture up to gravities of about 0.9. The costs of transporting propane, butane, and condensates are in the range of 8 to 15 cents per barrel per 1000 miles. The cost of separating the condensates from the natural gas at the market terminal has not been included in these figures. Costs for specific mixtures are attached in Table 9. The above costs for hauling liquids in large quantities appear to be from 35-50% of the tariff for product pipelines.

The study indicates that further work is justified in considering the transportation of light hydrocarbons in large quantities along with natural gas in long distance pipelines.

TABLE 5
Economic and Other Factors Used in Study

Length of line (L_t) 1000 miles
 Flowing temperature (T_f) 60°F
 Pipe strength (S) 65,000 psi and 100,000 psi
 Pipe diameter (OD) 16 inch, 24 inch and 30 inch (OD)
 Pipe roughness EE = 250 micro-inches
 Bend Index 200° per mile
 Longitudinal joint factor for pipe (E) 1.0
 Pipe design factor (F) 0.72
 Compressor efficiency, EFF, 80%
 Pipe cost (Y) 65,000 psi strength, \$265 per ton
 100,000 psi strength, \$384 per ton
 Cost of laying pipe (N) \$1200 per inch OD per mile
 Communication system cost (H) \$3000 per mile
 Compression station cost ($X_1 + X$) \$270,000 + \$165 per horsepower
 Cost of fuel (CF) 20 cents/Mcf
 Fuel consumption (FHPHR) 8.7×10^{-3} Mcf per horsepower - hour
 Cost of labor and maintenance for pipeline (CLML) \$850/mile/yr
 Cost of labor and maintenance for compressor stations (CLMS)
 \$19/hp/yr
 Gas loss (LG) = 0.005 fraction of flow at 20 cents/Mcf
 Administration expenses (AD) $\frac{1}{3650}$ cents/100 miles/Mcf
 Annual investment charge, 15% per year
 Line assumed to flow at 100% capacity 365 days/yr

TABLE 6A

Cost of Transporting Fluids
cents/100 miles/Mcf

Mixture	Based on 1000 mile pipeline, 65,000 psi steel						30" diameter					
	16" diameter	24" diameter	Min. Pipe Flow Min. Pipe Flow Min.	Thick- Rate Press. Thick- Rate Press. Min. Pipe Flow Min.	Mol. wt.							
	Min. Cost cents /100 miles /Mcf	Pipe Thick- ness in.	Flow MMcf psia /day	Min. Cost cents /100 miles /Mcf	Rate psia /day	MMcf psia /100 miles in.	Cost cents /Mcf	Thick- ness /day	Rate psia /day	MMcf psia /100 miles in.	Mol. wt.	
Natural gas	1.99 1.61	.19 .31	150 250	700 1100	1.37 1.17	.29 .39	400 600	700 1100	1.16 .997	.36 .445	600 900	700 1100
1.45	.376	.31	300	1560	1.06	.516	800	1560	.904	.585	1200	1560
1.338	.477	.400	400	2100	.994	.626	900	2100	.858	.754	1500	2100
10% Propane	1.984	.194	150	700	1.374	.273	400	700	1.148	.356	800	700
1.581	.304	.250	250	1100	1.148	.404	600	1100	.979	.443	900	1100
20% Propane	1.329	.389	350	1560	.972	.502	800	1560	.833	.592	1300	1560
1.27	.447	.400	400	1960	.938	.590	900	1960	.807	.708	1500	1960
30% Propane	1.23	.39	400	1540	.893	.505	900	1540	.765	.60	1500	1540
1.20	.437	.400	400	1940	.877	.598	1000	1940	.761	.696	1500	1940
5% Propane and Butane	1.388	.404	350	1600	1.017	.514	800	1600	.873	.609	1300	1600
5% Butane	1.32	.458	400	2000	.977	.633	1000	2000	.84	.724	1500	2000
5% Butane	1.509	.365	300	1300	1.104	.46	700	1300	.942	.525	1100	1300
	1.39	.416	350	1700	1.02	.53	800	1700	.88	.627	1300	1700
10% Butane	1.338	.455	400	1880	.982	.58	900	1880	.846	.701	1500	1880
1.308	.514	.450	450	2280	.97	.707	1100	2280	.846	.806	1500	2280
15% Butane	1.293	.453	400	2000	.953	.632	1000	2000	.82	.725	1500	2000
1.30	.515	.400	400	2400	.957	.731	1100	2400	.877	.853	1500	2400
Kurata S-2	1.383	.555	400	2660	.985	.787	1100	2660	.941	.909	1500	2660
	.642	.450	450	3060	1.034	.890	1100	3060	1.018	1.037	1300	3060
Kurata S-4	1.353	.579	450	2630	.9937	.789	1100	2630	.8875	.926	1500	2550
	.649	.450	450	3030	1.070	.896	1100	3030	1.03	1.036	1300	3030
												29.15

TABLE 6B
Cost of Transporting Fluids
 cents/100 miles/Mcf

Based on 1000 mile pipeline, 100,000 psi steel													
Mixture	16" diameter					30" diameter							
	Min. Cost cents /100 miles /McF	Pipe Thick- ness in.	Flow Min. /day	Rate Press. MMcf psia	Min. Cost cents /100 miles /McF	Pipe Thick- ness in.	Flow Min. /day	Rate Press. MMcf psia	Min. Cost cents /100 miles /McF	Pipe Thick- ness in.			
Natural Gas	1.56 1.36 1.26	.191 .264 .306	250 350 400	1100 1560 2100	1.12 1.00 .94	.261 .327 .402	600 800 900	1100 1560 2100	.96 .86 .82	.281 .377 .479	900 1200 1400	1100 1560 2100	17.39
10% Propane	1.93 1.52	.124 .186	150 250	700 1100	1.32 1.10	.189 .259	400 600	700 1100	1.11 .94	.234 .308	700 1000	700 1100	20.06
20% Propane	1.26 1.20	.251 .287	350 400	1560 1960	.92 .88	.322 .395	800 1000	1560 1960	.79 .74	.391 .467	1400 1600	1560 1960	22.73
5% Propane and 5% Butane	1.32 1.25	.260 .297	350 400	1600 2000	.97 .92	.332 .402	800 1000	1600 2000	.831 .802	.403 .460	1400 1400	1600 2000	20.8
10% Butane	1.26 1.22	.292 .331	400 450	1880 2280	.93 .91	.373 .450	900 1100	1880 2280	.80 .80	.447 .510	1500 1500	1880 2280	21.47
Kurata S-2	1.24 1.28	.371 .406	450 450	2660 3060	.92 .96	.504 .590	1100 1300	2660 3060	.81 .90	.610 .680	1700 1500	2660 3060	24.96
Kurata S-4	1.26 1.33	.371 .409	450 450	2630 3030	.93 .99	.504 .572	1100 1100	2630 3030	.84 .91	.596 .680	1500 1500	2630 3030	29.15

TABLE 9

Cost of Hauling Liquids

Composition	Dia.	Min. Press. psia	Cost Mixture	Gents/1000 miles	Total Gas Rate	Liq. Rate	Gas Cost \$1000/day	Liq. Cost \$1000/day	Cents/bbl /1000 miles
				Gas	MMcf /day	1000 bbls /day	Mixture \$1000/day	Gas \$1000/day	
<u>65,000 psi steel</u>									
30% Propane	24	1540 2100	8.93	9.94	900	616	189	80.4	65.6 14.8
20% Propane	24	1960 2100	9.38	9.94	900	704	131	84.4	70.0 14.4
5% Butane	24	1300	11.04	11.04	700	650	37.8	77.4	71.8 5.6
S-2	24	2660 2100	9.85	9.94	1100	934	143	108.3	92.9 15.4
<u>100,000 psi steel</u>									
10% Propane	24	1100 1560	11.0	11.2	600	528	48.1	66.0	59.1 6.9
20% Propane	24	1560 1560	9.2	10.0	800	625	116	73.6	62.5 11.1
S-2	24	2660	9.2	9.6	1100	934	144	101	89.1 11.1
S-4	24	2630	9.3	9.6	1100	860	206	102.3	82.5 19.8

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PIPELINE CALCULATIONS FOR GAS-LIQUID SYSTEMS

Introduction

The purpose of this project is to investigate the economics of carrying liquids along with natural gases in a single phase in high pressure pipelines. To carry out the investigation, the physical properties of the gas-liquid mixtures, pipeline flow calculations, compression calculations, and economical factors are combined into a single relationship. This relationship was programmed for the computer to permit a quick evaluation of the parameters over a wide range of conditions.

The basis of the study is the fact that liquefiable hydrocarbons when added to natural gas increase the density of the system, both because of the increased molecular weight of the mixture and also because the compressibility factor is lower for the mixture due to its lower pseudocritical temperature. At the same time, the increased molecular weight of the mixture does not increase the viscosity of the gas significantly. This combination of properties was thought to be such that the cost of transporting gases containing propane, butane, and raw natural gasoline could be less than for natural gas itself. To carry out the study, it is necessary to determine the conditions at which various systems are in a single phase. The density in the form of the compressibility factor and the viscosity are needed for such systems at the pressures and temperatures of the pipelines.

The pipeline flow calculations for this study are based on the American Gas Association Institute of Gas Technology report¹. It was deemed necessary to ascertain that the procedure was the equivalent of Weymouth's equation with Moody friction factors², and this was found to be true for the "partially turbulent case" adopted.

In any comparative study, it is necessary to establish a standard. After several trials, it appeared best to compare the cost of transporting natural gas a distance of 1000 miles with included liquids dissolved in a single phase fluid with the cost of transporting the gas alone. For each mixture and pipe diameter, the cheapest transportation cost was sought before comparing the two costs. From these two costs, one can find the cost of hauling the normally liquid constituent and compare it with liquid pipeline rates.

The various phases of the study will be described in turn starting with the phase behavior of gas-liquid systems.

Phase Behavior of Hydrocarbon Systems

Mixtures of natural gas and light hydrocarbons may be transported as a single phase provided the proper temperatures and pressures are maintained. For pipelines buried in the ground, the temperature range is limited to from about 30-150°F or from ground temperature to the compressor outlet temperature. The objective of this study is to find the pressures suitable for pipeline transportation over which gas-liquid mixtures will remain in single phase at temperatures of 40-100°F. As will be illustrated in a general discussion of phase behavior, there are two pressure regions at which mixtures such as methane and propane will remain in single phase. The first is the low pressure region wherein increased pressure causes condensation of liquid to occur. The second region--that of interest in this study--is the high pressure region where two phases occur upon lowering the pressure. The latter is sometimes referred to as the retrograde region since phase changes occur due to an opposite change in pressure to that normally found at low pressure. This behavior will be made clear in a general discussion of the behavior of mixtures.

Behavior of a Binary Mixture

Figure 1 shows the behavior of a binary system such as methane-propane. The curve AC_1 is the vapor pressure curve of methane and HC_3 is the vapor pressure curve of propane. At pressures above $AC_1C_2C_3$, all mixtures of methane and propane are in a single phase. Diagram BDC_2EFG represents the border curve between the single phase region outside and the two phase region inside for a specific mixture of 80% methane and 20% propane (molal basis). The areas of interest in the single phase region are pressures above the curves DC_2EF or any pressure at temperatures above F. The problem is to find a convenient way of calculating the single phase region for any mixtures of known composition at pipeline temperatures.

Although Figure 1 shows the behavior of only one mixture of methane and propane, the two phase region for all mixtures lies within the area of $AC_1C_2C_3H$ and outside this region all mixtures are in single phase. Figures 2a and 2b show quantitatively the location of the border curves for the methane-propane system. It may be seen that the various mixtures are similar in behavior and so only the 80% methane mixture will be discussed. When viewed from the interest of pipeline flow at 40-100°F, one needs to consider a series of mixtures of increasing propane content. Figures 3a and 3b give experimental data for the methane-butane system.

Studies were made by the writer on phase behavior of hydrocarbon systems in glass windowed cells some 25-30 years ago^{2,3,4}. Consider the behavior of mixture A, Figure 1. At the temperature and pressure of I, it is in a single phase. Upon dropping the pressure isothermally to D, it remains in single phase until reaching pressure D at which pressure bubbles of vapor appear. The curve BDC_2 is the bubble point curve for mixture A. Further pressure reductions below D cause various percentages of vapor to form as indicated qualitatively on Figure 1. For quantitative percentages of liquid, see Kurata phase diagrams in this report.

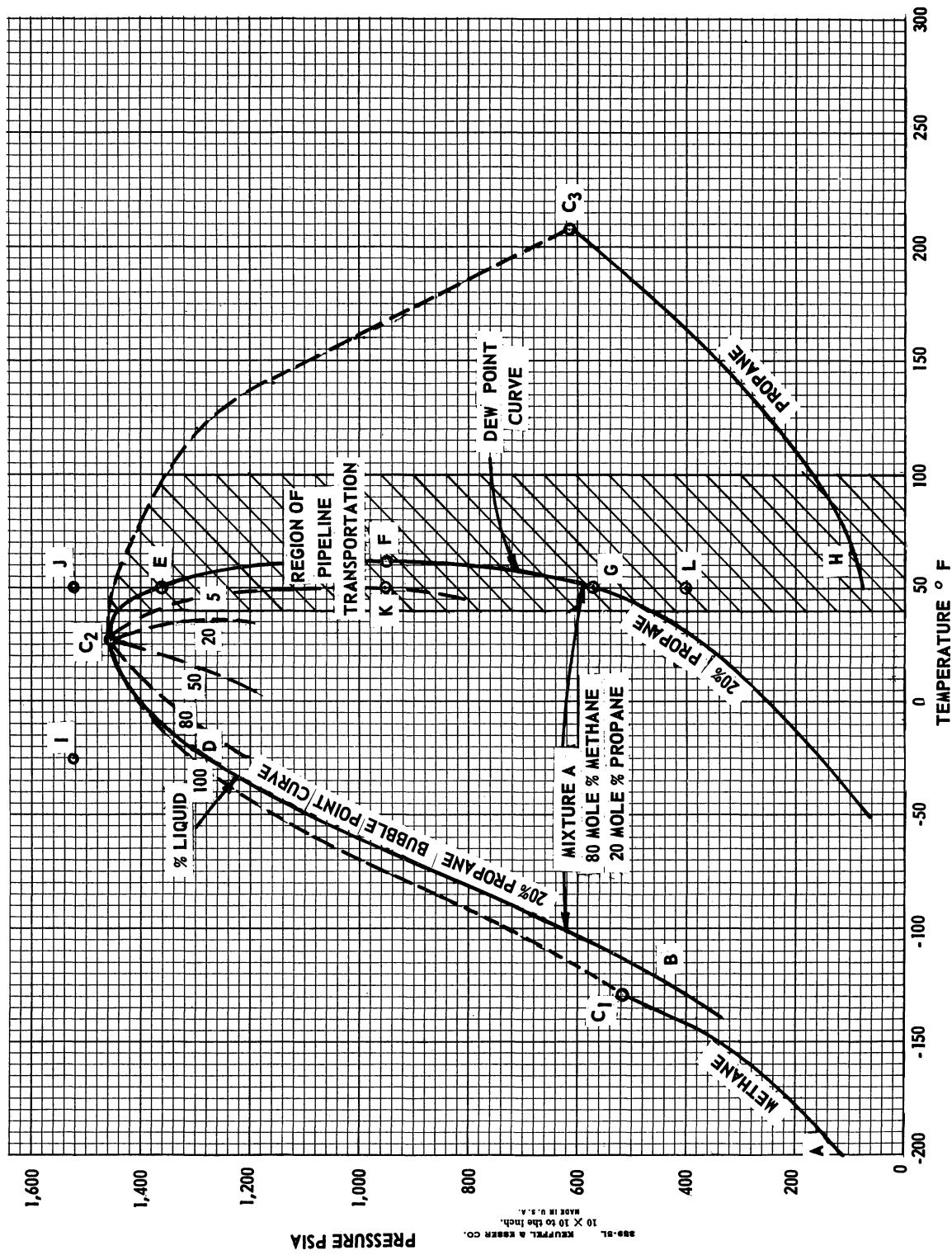


Figure 1. Illustration of Phase Behavior for Binary System of Methane-Propane

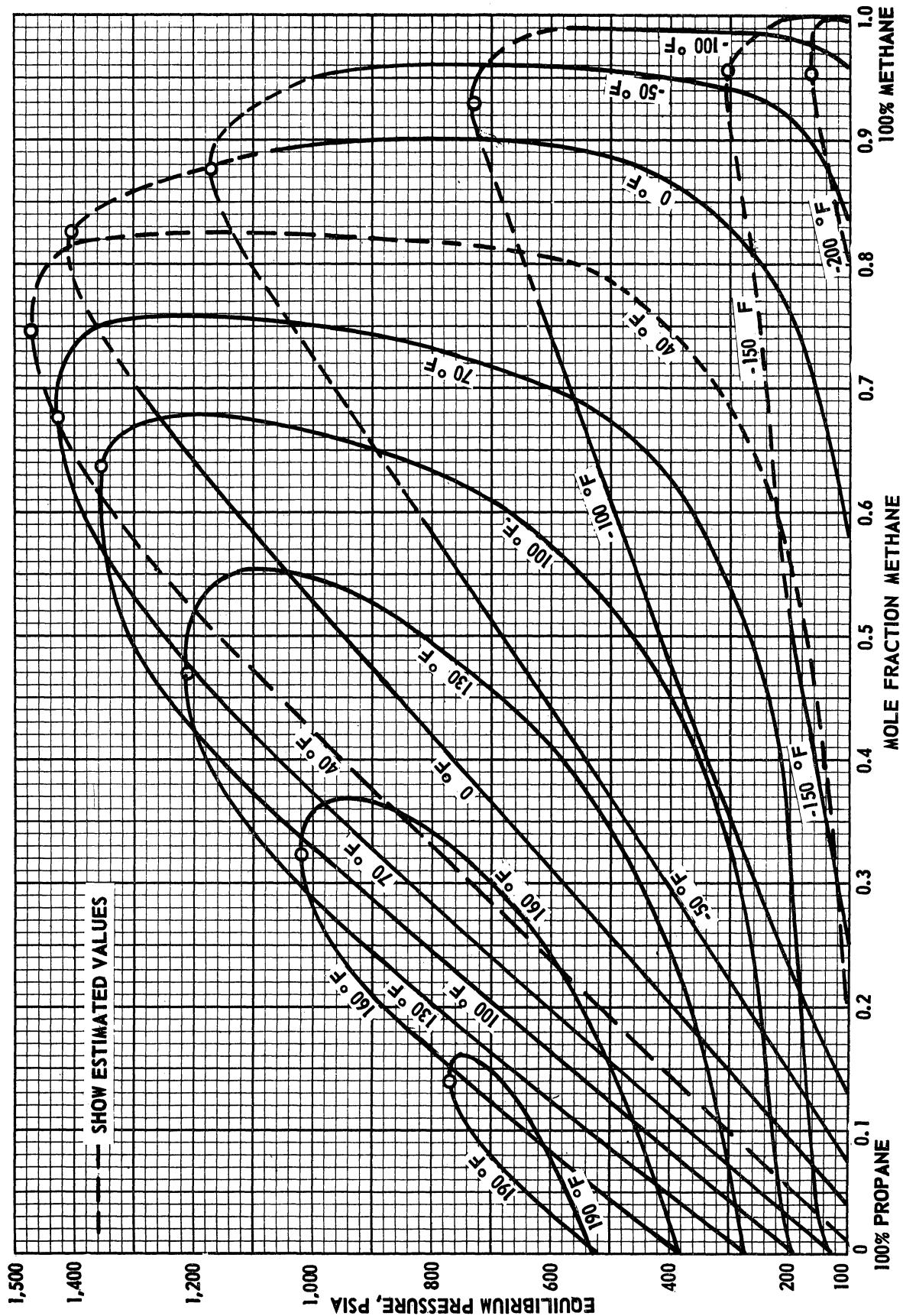


Figure 2a. Bubble and Dew Point Curves for Methane-Propane System.

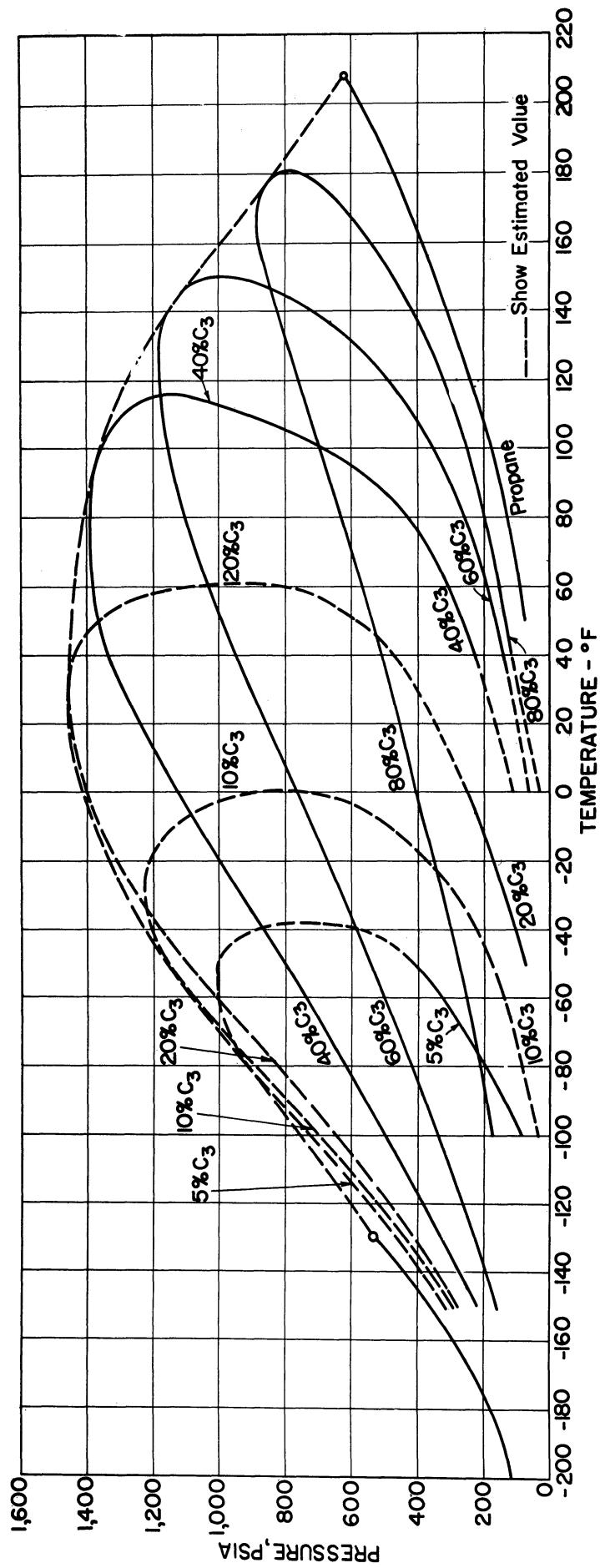


Figure 2b. Phase Behavior of Methane-Propane System

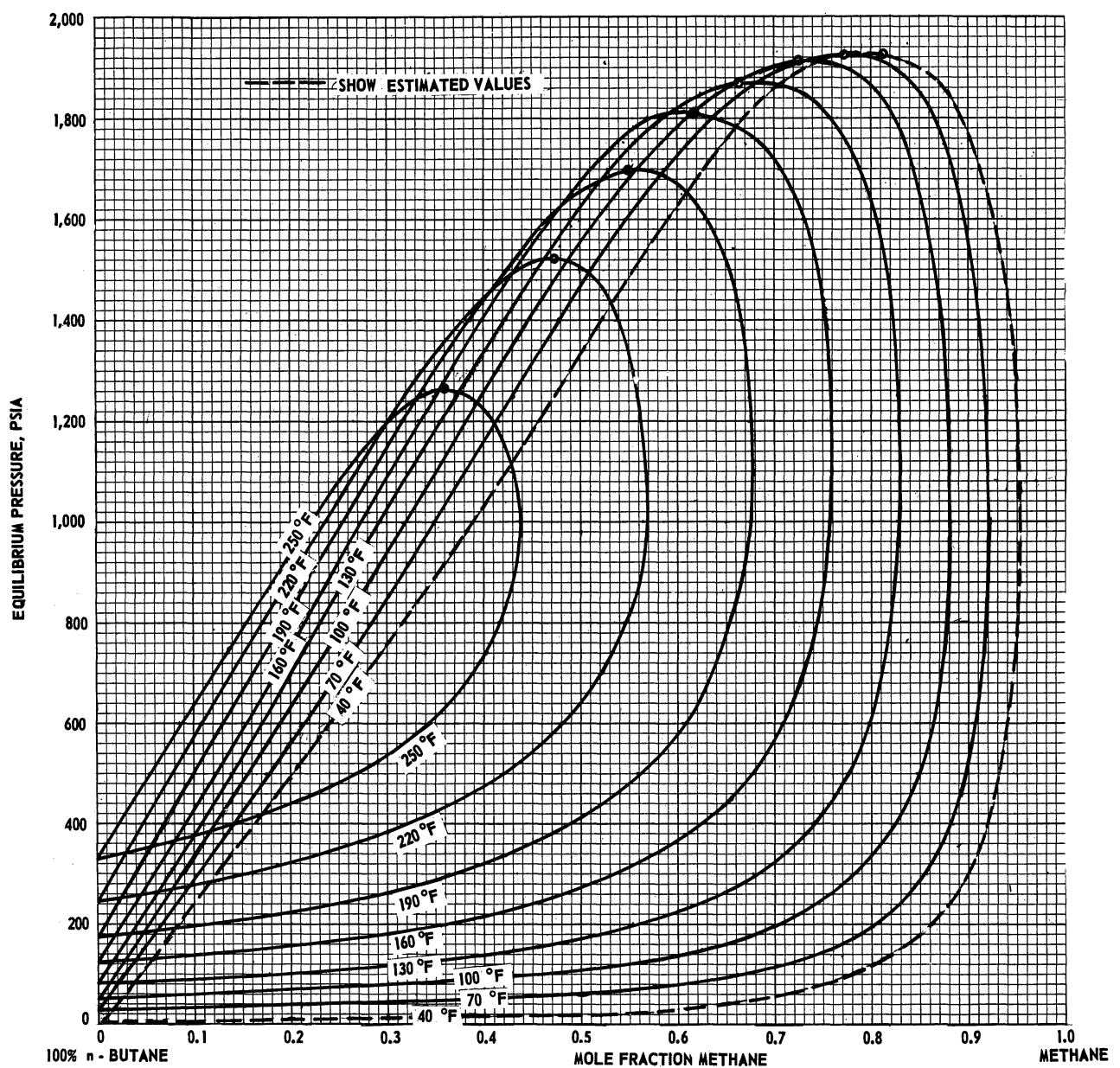


Figure 3a. Bubble and Dew Point Curves for Methane-n-Butane System

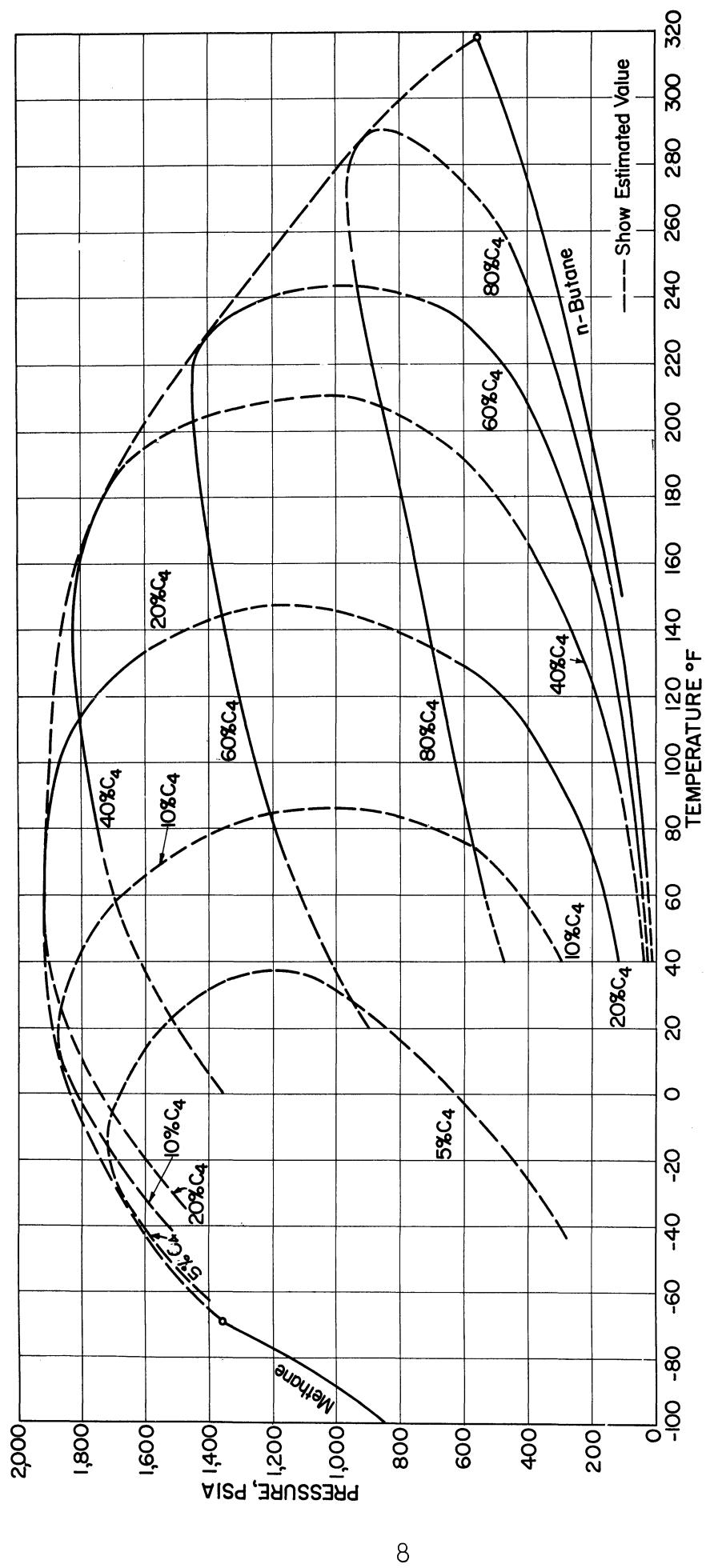


Figure 3b. Phase Behavior of Methane-n-Butane System

Mixture A at J is in a single phase and upon dropping the pressure to E, droplets of liquid or dew form beginning at pressure E. The curve C_2 EF is the dew point curve for mixture A, sometimes known as the upper or retrograde dew point curve. As the pressure is dropped below E, liquid forms reaching its maximum volume at K. Further pressure reductions below K cause the liquid to vaporize. When the pressure reaches G, all of the liquid has vaporized and point G is the lower or normal dew point. When compressing the mixture from L to G, dew forms in the manner usual for pipelines.

Should one have selected a single pressure just above C_2 and allowed the pressure to drop, it would have been observed that at C_2 the mixture would change from a single phase to about 50% by volume liquid and 50% vapor. Point C_2 is the critical temperature and pressure for mixture A; it is the convergence of the bubble point, dew point, and various percentage liquid lines.

Region of Interest for Pipeline Flow

Any mixture of natural gas and liquid hydrocarbons has a phase diagram generally similar to mixture A. Figure 4 is presented to relate the region of interest relative to the phase behavior of a mixture. In the past, pipelines have been operated at temperatures above C of Figure 4 to keep the gas in a single phase. A mixture carefully stripped of heavy hydrocarbons, pentanes and heavier, can carry a considerable amount of liquefiable propane or butane at temperatures of 40°F and above. For these mixtures whose maximum two phase temperature (criterion therm point C of Figure 4) lies below the pipeline temperature, the currently used procedures may be followed in that there is no lower or higher limit of pressure which will cause condensation.

This study proposes to handle in pipelines those mixtures for which point C is within or above the pipeline temperature range of 40-100°F. Therefore, it will be necessary for such

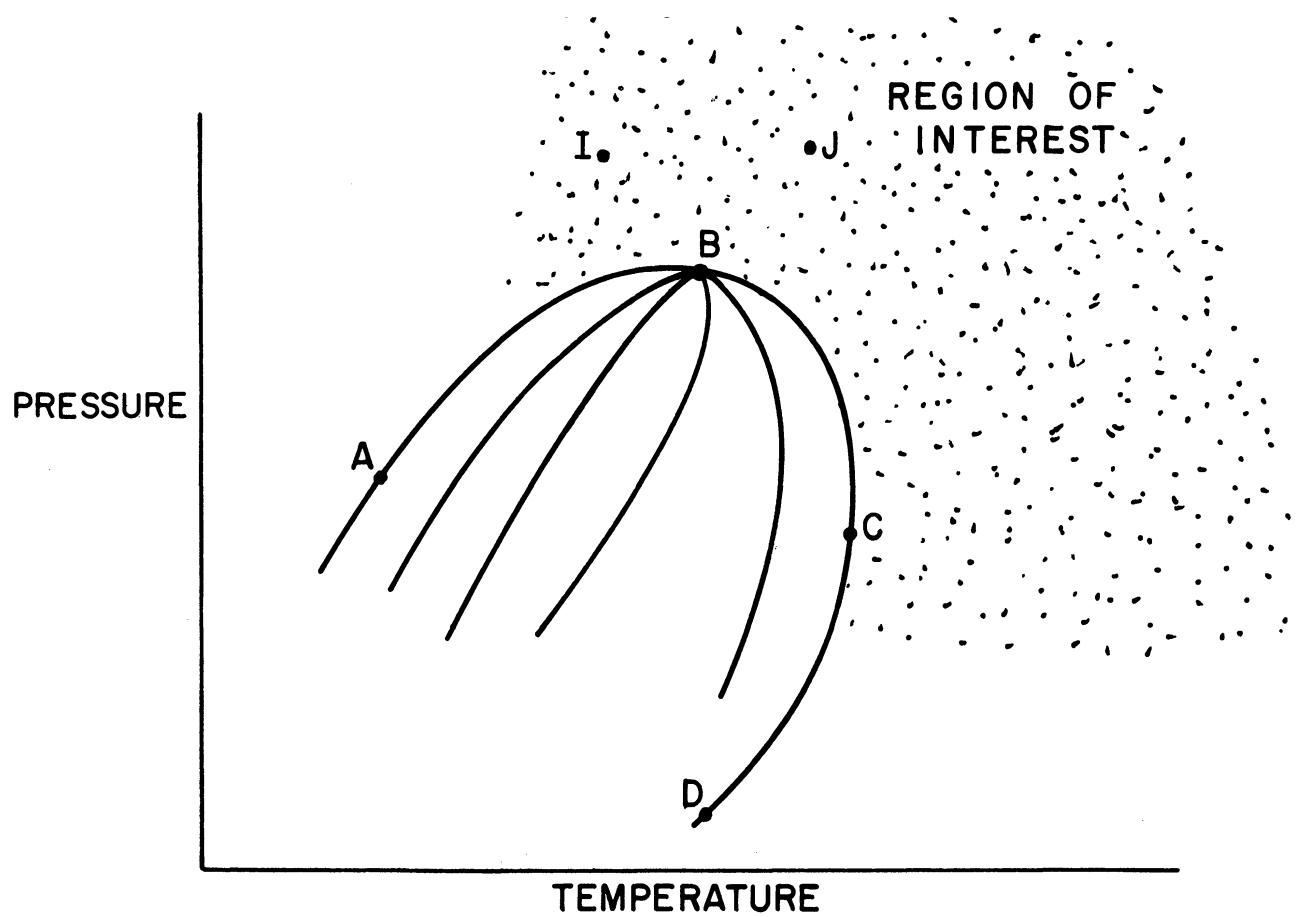


Figure 4. Region of Interest for Pipeline Transportation of Gas-Liquid Systems

mixtures to limit the lower pressure at which they may be transported to the conditions corresponding to pressures of ABC of Figure 4. It would seem that the procedure desired is a way of finding the maximum pressure in the temperature range of pipeline flow used here as 40-100°F at which two phases occur. This maximum two phase pressure will then become the minimum pipeline pressure for single phase operation. Thus for methane-propane mixtures, the minimum pressure in pipelines for mixtures rich in propane, one can use the critical loci, curve $C_1 C_2 C_3$ of Figure 1. The plan is to use a minimum pressure in the pipeline 100 psi above the single phase pressure to give some degree of safety.

It does not matter with respect to phase behavior whether the mixture is at condition I or J on Figure 1. For pipeline transportation, it can move over a temperature range including crossing the critical at these pressures and remain in single phase. It is true that the densities of the mixtures in this region are high compared to present values in gas pipelines, and that the density changes rapidly with both temperature and pressure. It is believed that centrifugal compressors should be considered for such fluids in that they are as much like a liquid as a gas.

The terms liquid and gas become relatively useless when describing single phase fluids in the region of I and J in Figures 1 and 4. Fluid at I might be called a "compressed liquid", but the nature of the liquid at D, Figure 1, is so unusual, i.e., density, compressibility, etc., that use of the term "liquid" may be misleading. Likewise, application of the term "gas" to fluid at J would not be meaningful. The description of these single phase fluids can be made only by indicating their density and compressibility.

Mixtures Considered in Study

The liquids which need transportation are propane, butanes, natural gasolines or condensates, and crude oil. Initially,

it was intended to consider transporting all of these liquids. However, pipeline calculations have been made only for propane, butanes, natural gasoline, and propane-butane mixtures. Table 1 lists the mixtures which have been studied.

One consideration is the volume of liquids which a pipeline can handle for various concentrations. When vaporized, propane represents 35.8 cubic feet at 60°F and 1 atmosphere per gallon and n-butane 30.77 cubic feet per gallon. A pipeline with a 500 million cubic feet per day capacity will carry 1,580,000 gallons (37,700 bbls) of propane per day at a concentration of 10 mole % propane. These quantities of liquids are large. The State of Louisiana in 1964 produced a total of 350,000 bbls per day of all natural gas liquids, while Texas produced 750,000 bbls per day. The barrels condensate per million cubic feet of gas included all the propane as listed in Table 1.

Figure 5 is a plot of barrels of liquid per day versus quantity of gas with lines for various mixtures of gas and liquids. It is helpful in arriving at the quantities of liquid being considered for various concentrations and gas flow rates. This figure does not include the propane present in the natural gas as condensate.

Permissible Pressures

For each mixture on Table 1, one needs to know the minimum pressure permitted when transporting the fluid in a single phase. For a 0.6 gravity natural gas and the 10% propane-90% natural gas mixture, the 40°F minimum pipeline temperature is above the liquefaction temperature at any pressure. For such mixtures, the minimum pressure put in Table 1 is 600 psia, but this is an arbitrary value to make the lowest pipeline pressure at 700 psia the minimum permitted pressure plus 100 psia safety factor used for all mixtures to avoid condensation in liquid rich mixtures.

For those mixtures which form two phases at the lowest contemplated pipeline temperature, used here as 40°F, one needs

TABLE 1

Compositions and Properties of Mixtures

	1 Gas 90% Propane 10%	2 Gas 80% Propane 20%	3 Gas 70% Propane 30%	4 Gas 60% Propane 50%	5 Gas 90% Propane 5%	6 Gas 95% Butane 5%	7 Gas 90% Butane 10%	8 Gas 85% Butane 15%	9 Mixture S-2 (Kurata)	10 Mixture S-4 (Kurata)
Natural gas				Mole Percentage Composition						
methane	.927	.8343	.7416	.6499	.8343	.88115	.8343	.78795	.78800	.72800
ethane	.05	.0450	.0400	.0350	.0450	.04700	.0450	.04250	.05900	.05460
propane	.023	.1207	.2184	.3161	.0707	.02185	.0207	.01955	.03146	.03020
i butane					.0200	.02000	.0400	.06000	.01064	.01228
n-butane					.0300	.03000	.0600	.09000	.01596	.01842
pentanes									.04250	.06880
hexanes									.02525	.04382
heptanes ⁺									.02140	.03750
N ₂									.00579	.00538
Total	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00000	.99900
mol. wt.	17.39	20.06	22.73	25.4	20.76	19.42	21.47	23.51	24.96	29.15
gravity	0.600	0.692	0.778	0.875	0.716	0.671	0.742	0.811	0.861	1.005
bbls condensate per million cu. ft.	15.30	80.25	145.6	210.2	86.7	54.2	93.9	131.6	130.5	187.6
Limiting single phase pressure 40-100° F psia	600	600	1460	1440	1500 (est)	1200	1780	1900	2560	2530
Pseudo T _c , P _c , psia	361	391.5 668	421.5 661	453 667	396 663	381 667	400 660	420 654	426 649	460.5 634

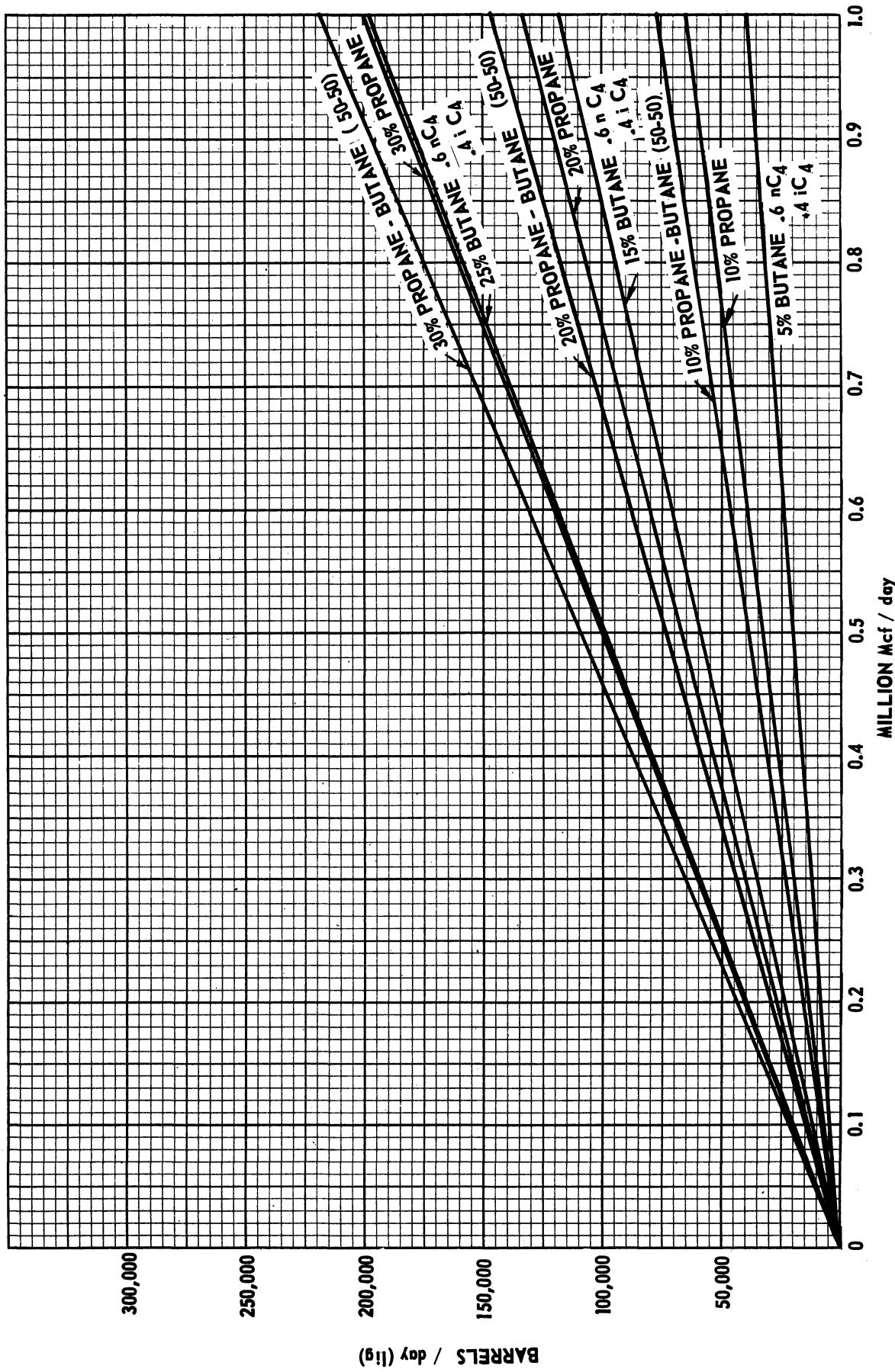


Figure 5. Relationship of Barrels of Liquid Per Day as a Function of Quantity of Gas as Mixture

the upper dew point pressure or bubble point pressure should the critical temperature be above 40°F. Calculation of the dew or bubble point pressure could be made for various mixtures using equilibrium constants². However, in this region, the convergence pressure for the equilibrium constants is very sensitive to composition and machine computation with an established program such as the NGPA and Oklahoma State University are devising would be needed. However, even such a program cannot be trusted at critical conditions. A rather fruitless study was made of a correlation of the single phase pressures with molal average boiling point and molecular weight for the mixtures under consideration. From the lack of correlation found in this study, it was decided to rely on experimental phase data with minor adjustments for the mixtures shown in Table 1. References to experimental data for specific systems are given in Table 2.

For the natural gas, it was decided to use a 0.6 gravity gas with the composition shown on Table 1. The compositions for streams 2-8 are given on the table after adding the indicated amount of liquids. For the last mixtures, 9 and 10, the compositions are those used for determining the phase behavior.

It should be noted that the mixtures in Table 1 are not the pure methane-propane and methane-butane systems, but have intermediate constituents present in the natural gas.

The differences in the behavior of mixtures 2, 3 and 4 as compared to the methane-propane mixtures should be considered. Figure 6 shows the critical loci of the methane-propane-pentane system. From the figure, it is observed that adding an intermediate constituent, propane, lowers the critical pressure for any methane-pentane system. In like manner, ethane will lower not only the critical pressure at a given temperature but also the bubble and dew point curves for a specific mixture. Reference can be made to the methane-ethane-propane systems of Price and Kobayashi, shown in the Handbook, p. 579². At 1100 psia and 50°F, ethane at 3% does not reduce the propane content of a dew point gas. At 800 and 1000 psia and 50°F, the propane content is 1 or 2% less for a dew point when some 5% of ethane is added to the methane-propane system.

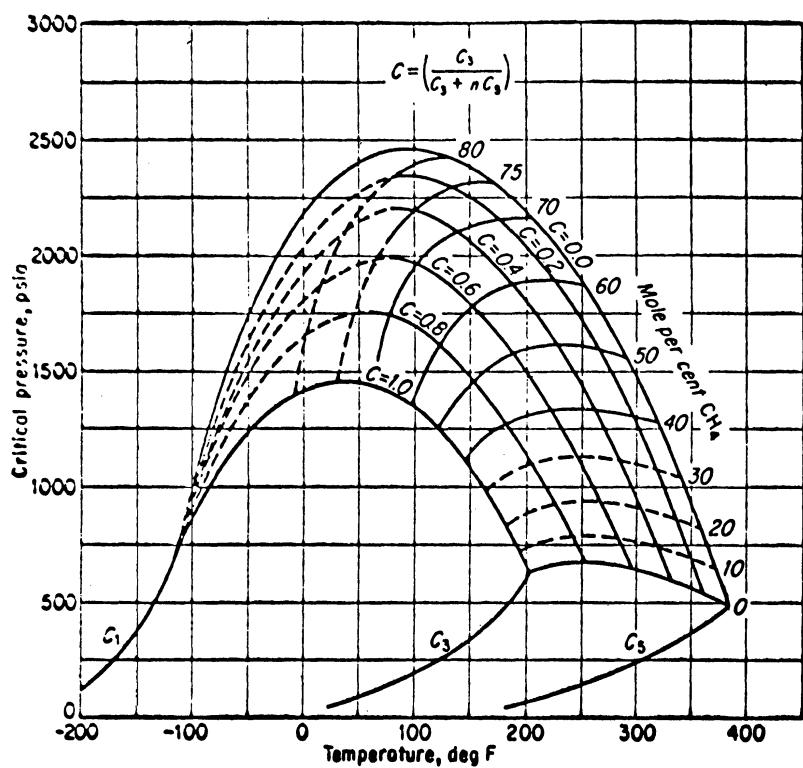


Figure 6. Critical Loci of Methane-Propane-Pentane System

TABLE 2
Phase Behavior Data Sources for
Light Hydrocarbon Systems

System	Temp. (°F)	Two Phase Press. Range (psia)	Reference
Methane-Ethane	0 40	300-900 500-900	5
Methane-Propane	40 100 160	511-1469 180-1355 380-890	6
Methane-n-Butane	70 100 160	819-1924 339-1904 435-1810	7
Methane-Isobutane	100 160	100-1680 150-1400	8
Methane-n-Pentane	100 160	313-2443 720-2300	9
Methane-Propane- n-Pentane	100 160	1350-2375 700-2150	10 11
Methane-Ethane-Propane	0 50	100-1300 100-1200	12
Ethane-Propane	0 50	100-400 100-200	12
Propane-n-Butane	160	55-172	13
Methane-Butane-Decane	100-460 160 100-460 40	400-10,000 400-5000 200-10,000 1000-4000	14 15 16
Gas Condensates	0-200	600-2700	4

It may be concluded that the differences in minimum pipeline pressures selected from binary systems are not in error enough to influence the cost of pipeline transportation significantly. Dr. Fred Kurata measured for his doctorate thesis⁴ gas-natural gasoline mixtures for which the phase behavior are of interest. Two of these mixtures S-2 and S-4 were selected for this study. Figures 7 to 8 give the phase diagrams from the thesis. At 40°F, mixture S-2 is below the critical temperature of 55°F. In this case, a higher pressure is required to hold the mixture in a single phase at 100°F than at 40°F. Mixture S-4 has the full pipeline temperature range below its critical temperature of 109°F.

The phase diagrams for nine other mixtures studied by Kurata are given in Appendix D along with the compositions. To obtain the critical points for other mixtures, the method of Kurata^{4,2} is suggested since it is based on natural gas-liquid systems.

The minimum pressure listed on Table 1 is either the 600 psia when no phase changes occur, or the highest bubble or dew point value found over the temperature range of 40-100°F. As indicated previously, the ethane in the natural gas was neglected. Likewise, the propane in the gas was neglected in finding the single phase pressure for mixtures 6, 7, and 8.

Initially, it was intended to use butanes as 40 mole % isobutane and 60 mole % normal butane. The composition is indicated in this manner on Table 1 and was used in finding pseudo-critical conditions for viscosity and density to be discussed later. Otherwise, the phase behavior is based on methane-n-butane properties. Isobutane generally would provide some safety factor. The 5% propane-5% butane, mixture No. 5, has not been measured experimentally. The value given is an estimate.

The Density of Single Phase Hydrocarbon Fluids

The method of expressing the density for natural gases is the gas law, including a compressibility factor (Z)

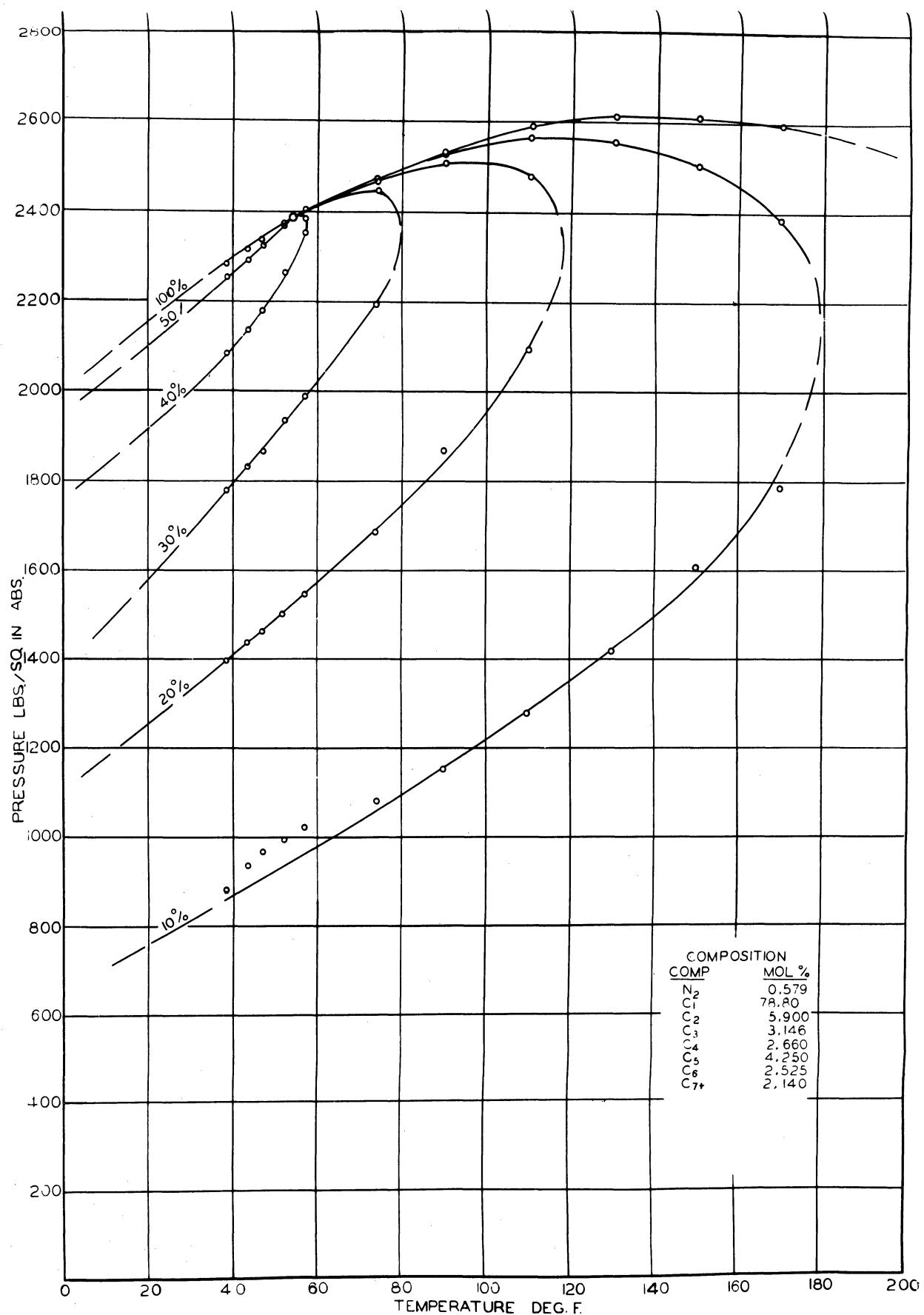


Figure 7. Phase Diagram for Mixture S-2 (Kurata)²⁸

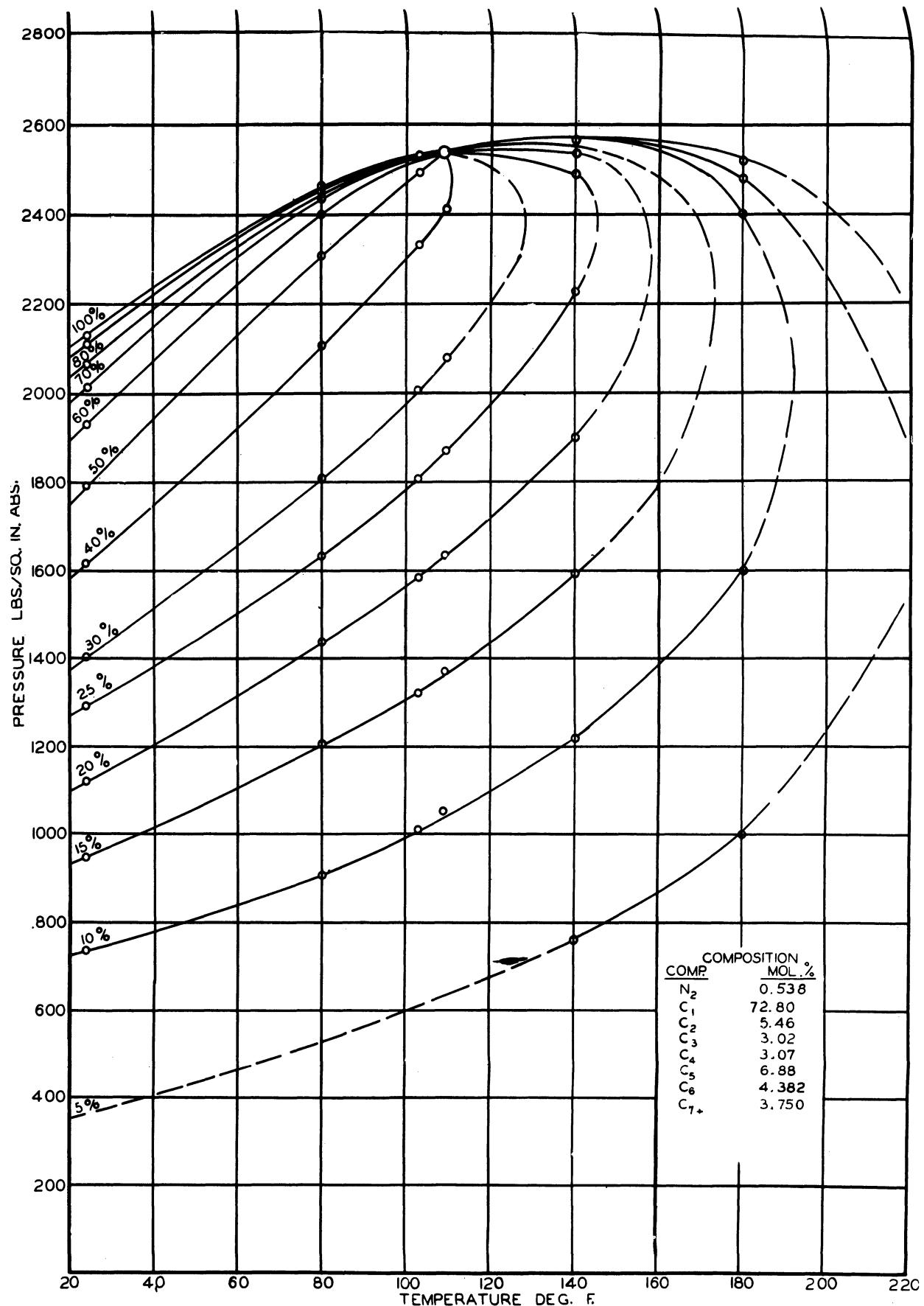


Figure 8. Phase Diagram for Mixture S-4 (Kurata)²⁸

$$PV = ZnRT$$

where P = pressure, psia

V = volume, cu. ft./lb. mole

Z = compressibility factor

n = number of pound moles

R = gas constant, 10.73 for these units

T = $^{\circ}$ Rankine, $^{\circ}$ F + 460.

A generalized chart of compressibility factor versus reduced pressure (pressure over critical pressure) and lines of reduced temperature (absolute temperature over critical temperature) has been devised for natural gases, Figure 4-16, p. 106². This chart has been put in form for computing by Sarem¹⁷ and is generally satisfactory, excepting for values of the compressibility factor below 0.50 where errors can become large. At reduced temperatures of 1.15 and below and at pressures of 600 to 2000 psia, the correlation is not reliable.

A program (ZFAC) has been prepared using Sarem's method to give the compressibility factor for any temperature and pressure when the gas composition is known. Should the composition be represented only by the gas gravity (G), the program uses the pseudocriticals corresponding to the published relationship of these values with G.

The subroutine for computing Z at any temperature (Tr 1.05-2.95) and pressure (Pr 0.1-14.95) requires the composition of the gas or its gravity. The critical properties and molecular weight of N₂, CO₂, CH₄-C₇H₁₆, iC₄H₁₀, and iC₅H₁₂ are already in the program. Three added constituents may be specified along with their T_c, P_c and molecular weights. Initially, the mole fractions of all constituents and properties of the three added constituents are set at zero.

This subroutine is incorporated into the pipeline flow program, and it provides the Z when called upon. An example of the use of this program for predicting Z 's is given in Appendix F, along with a comparison of computer values with those read from the chart.

The AGA supercompressibility factors were not used in this study, and presumably would be an alternate way of expressing density.

For mixtures S-2 and S-4, the thesis⁴ gives the saturated densities with only a nominal precision. Figure 9 shows the measured saturated densities along with values at 3000 and 4000 psia calculated from the compressibility chart.

Viscosity

The pipeline flow formula requires the viscosity of the fluid as a function of temperature and pressure. In general, for simplified cases a constant flowing temperature will be used in the pipeline. Therefore, it is more appropriate to say that the viscosity is needed as a function of pressure at a given temperature (60°F) and for a given composition.

The viscosity of methane, some natural gases, and of the methane-butane and methane-propane system have been determined. These viscosities also have been correlated as a function of reduced temperature and pressure with a reasonable degree of success^{18,19}. Table 3 lists the sources of data which are of interest. Figure 10 gives the reported methane-propane viscosities.

It was decided to use the correlation of Bicher and Katz for the viscosity of the fluids. Figures 11 and 12 give the correlation based on reduced temperature, reduced pressure, and molecular weight. The work of Carr et al¹⁹ indicated that Bicher's data had some inaccuracies over some ranges of pressure, since Bicher apparently had some turbulence in his rolling ball viscosimeter. The correlation of Carr et al probably would have been better but would have been more difficult to use. In view of the indirect effect of viscosity on the friction factor, the work of Bicher seemed satisfactory.

The viscosity correlations as shown in Figures 11 and 12 were programmed by submitting the points on the curves as data

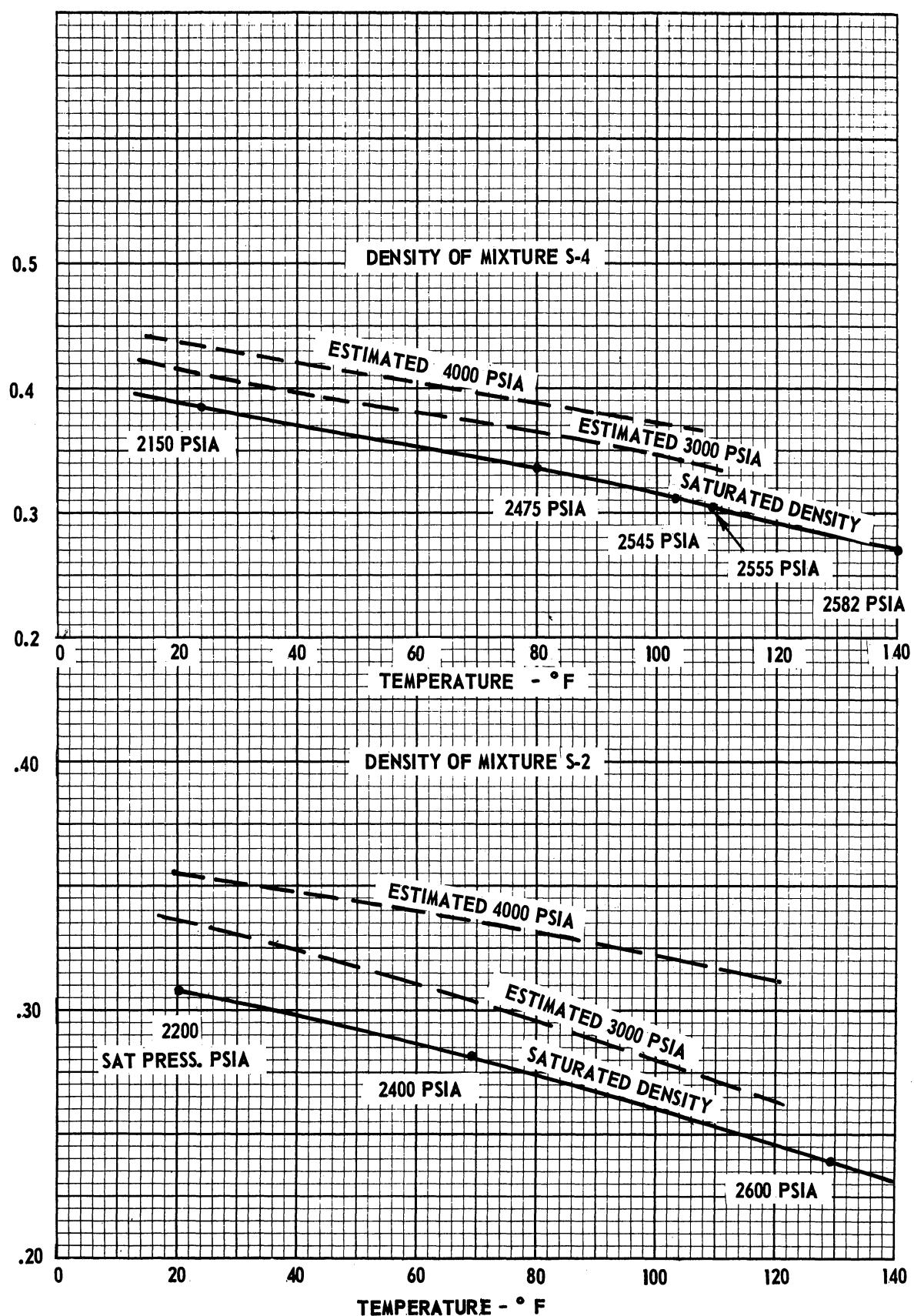


Figure 9. Density of Gas S-Natural Gasoline Mixture (S-2 and S-4)

TABLE 3
Viscosity Data

System	Temp. (°F)	Pressure (psia)	Source
Methane	0-150	14.7-4000	18,21
Methane-Propane	0-150	14.7-4000	18
Propane	0-150	14.7-4000	18,22
Methane-n-Butane	70-160	14.7-4000	23
Ethane	59-150	100-4000	22
Pentane	77-150	100-3000	24
Natural Gases		100-4000	21

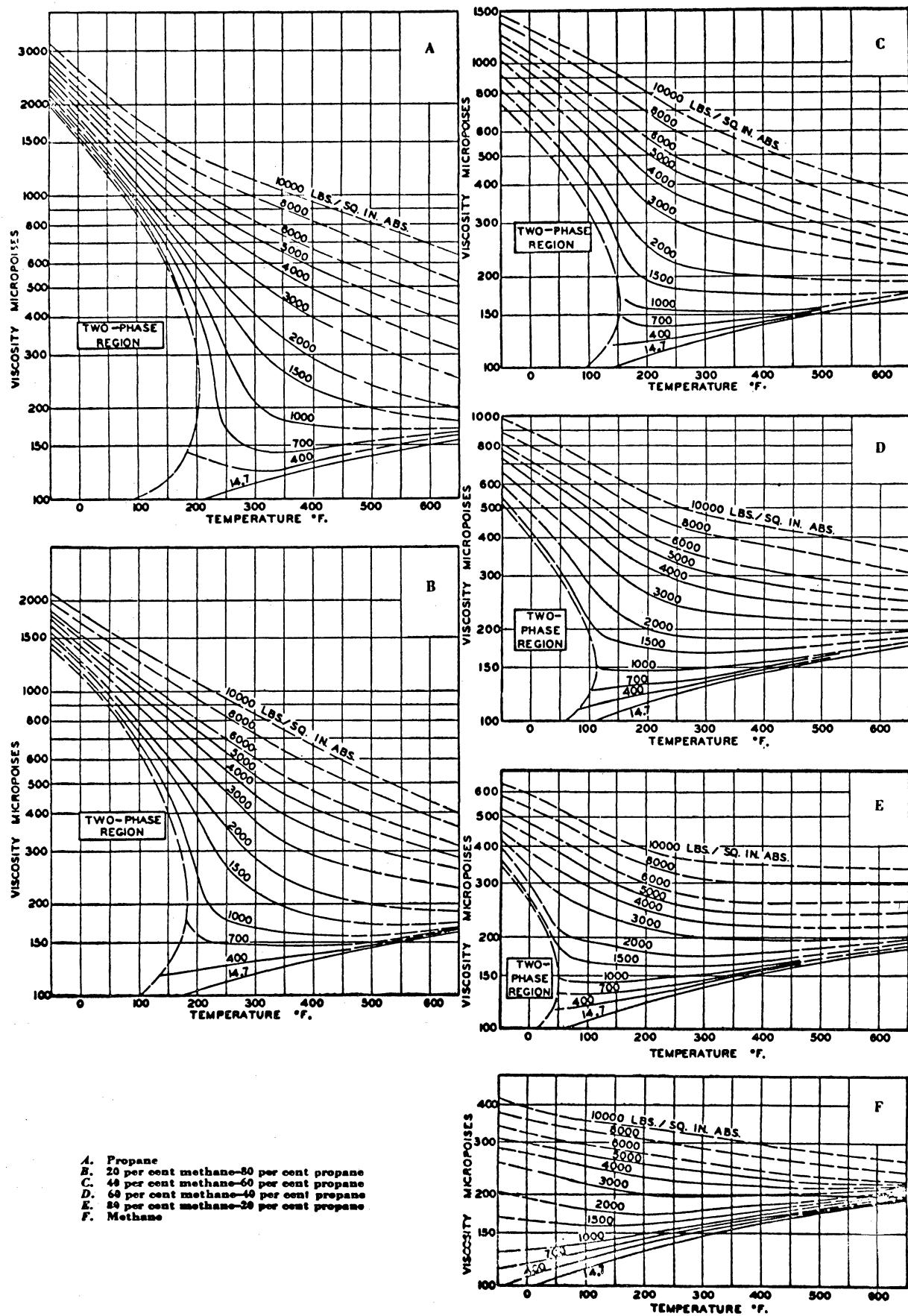


Figure 10. Viscosity of Propane, Methane and Mixtures

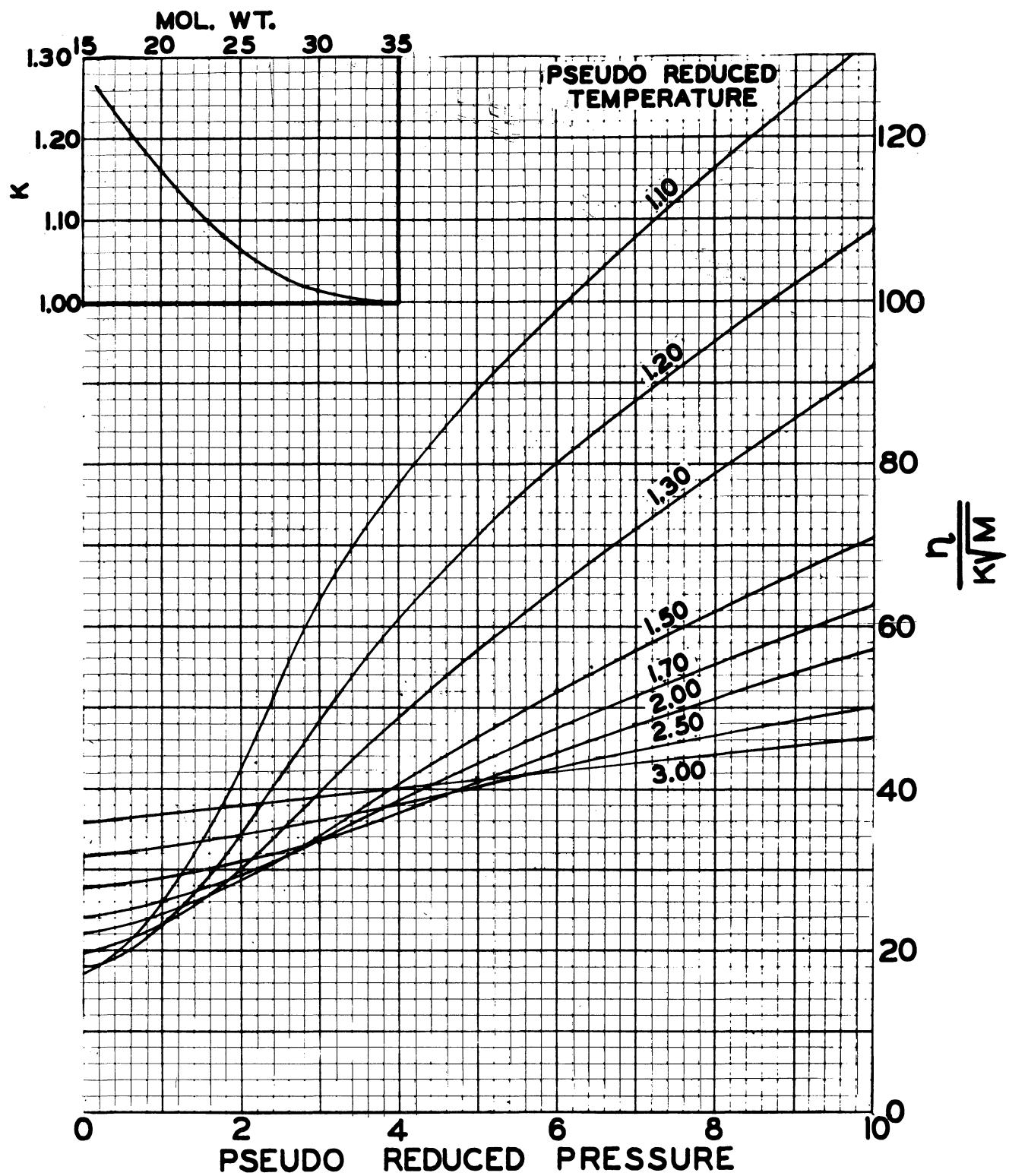


Figure 11. Viscosity of Paraffin Hydrocarbons (Correlation) at High Reduced Temperatures

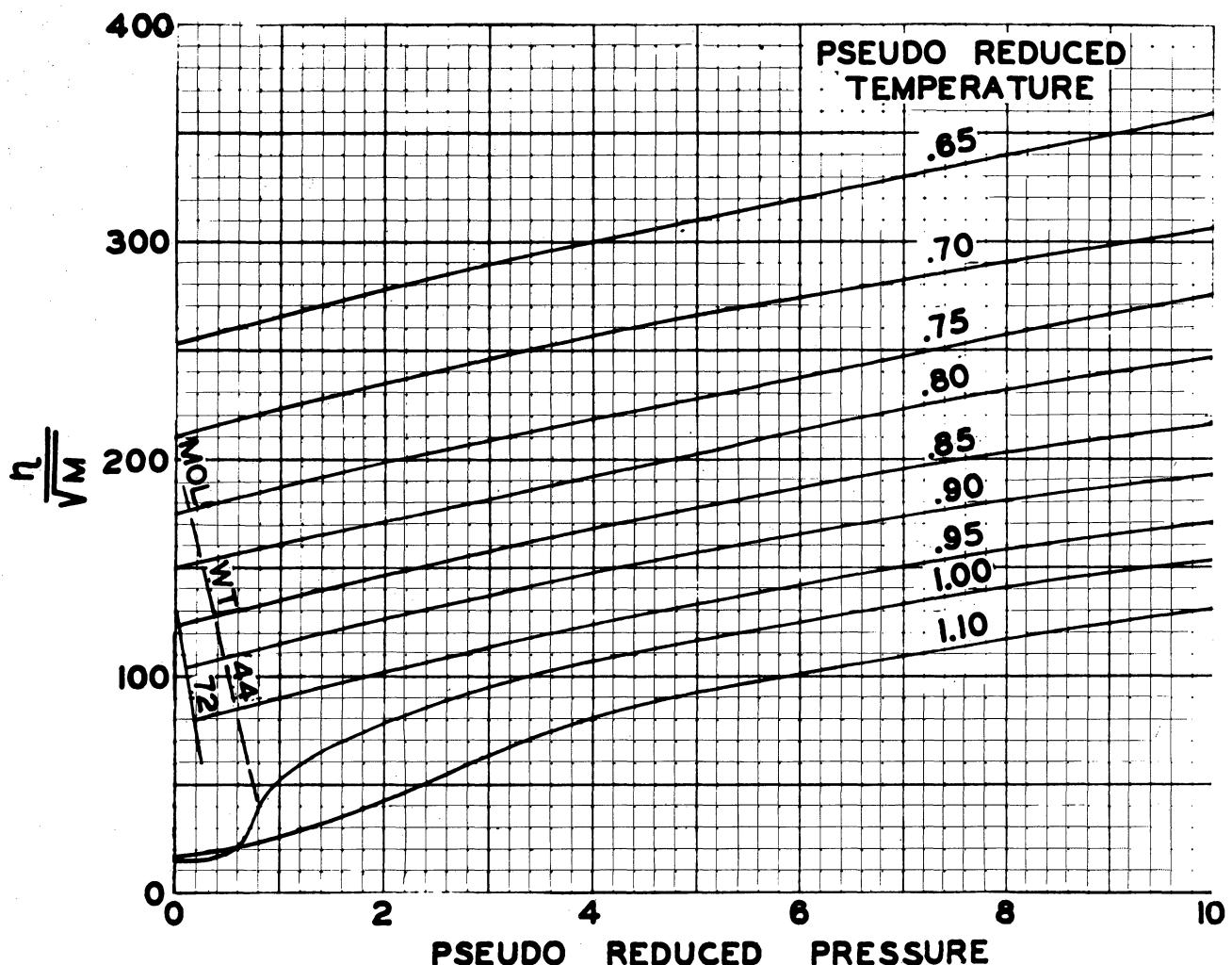


Figure 12. Viscosity of Paraffin Hydrocarbons at Low Reduced Temperatures

and providing an interpolation procedure. Appendix G lists the viscosity data used by the computer.

Pipeline Flow Calculations

Upon discussing with the Advisory Committee the appropriate equation for making pipeline calculations, a copy of the report "Computation of Flow and Natural Gas Transmission Lines" by the Institute of Gas Technology with A. E. Uhl and Committee NBL3 as the author (1964) was supplied to the project¹. It was agreed that this method of computing pipeline flow would be satisfactory from the standpoint of the sponsor companies. Therefore, an effort was made to see how this formulation of pipeline flow fitted into the usual Moody friction factor concept for flow of fluids in pipes. If possible, the flow formulas used in the report would be used but with an understanding of the friction factor which the procedure follows.

It was found that the partially turbulent flow calculation procedure was that of using the Moody friction factors with a roughness corresponding to smooth pipe. Added factors for degrees of bending and pipeline flow efficiency are included. Otherwise, the procedure is exactly that which would be used in the normal Weymouth type equation which employs the Moody friction factor. In the fully turbulent case, a further simplification is made in the report which does not appear necessary in this project. Accordingly, the pipeline flow formula is presented in the manner which is believed to be compatible with the IGT report and the flow calculations with which the writer is already familiar.

Flow Formula

It is proposed to calculate flow in natural gas transmission lines by the following equation¹:

$$Q_b = 77.5 F_{tb} F_{pb} F_{gr} F_{tf} F_{pv} F_d F_t F_f F_{fe} \left(\frac{P_1^2 - P_2^2}{L} \right)^{0.5} \quad (1)$$

where Q_b = gas flow rate - cu. ft./day at P_b and T_b

P = pressure - psia

P_1 = inlet pressure

P_2 = outlet pressure

L = total length of pipe between compressor stations excluding any equivalent length due to bends in pipe since such is included in F_f . Equivalent length due to valves, gates, etc. are to be included.

$L = L_t / NOS$, miles

L_t = total length of line, miles

NOS = number of stations

F_{tb} = base temperature factor $\frac{T_b}{520}$. A base temperature factor of 60°F will be used. $F_{tb} = 1.0$ (2)

F_{pb} = base pressure factor $\frac{14.73}{P_b}$. A base pressure of 14.73 will be used.

$$F_{pb} = 1.0 \quad (3)$$

F_{gr} = gas gravity factor = $\sqrt{\frac{0.6}{G}}$ (4)

G = molecular weight/29.0

F_{tf} = flowing temperature factor = $\sqrt{\frac{520}{T_f}}$ (5)

T_f = flowing temperature - $^{\circ}\text{R}$

F_{pv} = supercompressibility factor = $\sqrt{\frac{1}{Z}}$ (6)

F_d = line diameter factor = $D^{2.5}$

D = inside diameter of the pipe in inches (7)

F_t = transmission factor = $\frac{1}{\sqrt{f_F}} = \frac{2}{\sqrt{f_M}}$ (8)

f_F = fanning friction factor (9)

f_M = Moody friction factor

F_f = drag factor = function (Bend Index). Graph of F_f versus Bend Index is as shown in Figure 13 (reproduced from Figure IV-1, p. 56, IGT report, 1964).

F_{fe} = flow efficiency factor. Normally use $F_{fe} = 1.0$ unless there are data on cleanliness, etc. to reduce the value. (11)

For the above restrictions the equation reduces to

$$Q_b = 77.5 F_{gr} F_{tf} F_{pv} F_d F_t F_f \left(\frac{P_1^2 - P_2^2}{L} \right)^{0.5} \quad (12)$$

Calculation of Supercompressibility Factor

In evaluating F_{pv} , a pressure and temperature are needed.

The pressure to be used is the P_{average} (P_{avg}) defined as follows:

$$P_{\text{avg}} = \frac{2}{3} \left[P_1 + P_2 - \frac{P_1 P_2}{P_1 + P_2} \right] \quad (13)$$

Calculation of Moody Friction Factor and Transmission Factor (F_t)

The Moody friction factor has been plotted against Reynolds number with lines of constant wall roughness (Moody, 7-25), (Katz, Figure 7-3, p. 303)². The Reynolds number for a gas flowing in pipe is given by the following equation:

$$Re = 4.775 \times 10^{-4} \left(\frac{Q_b}{\mu} \right) \left(\frac{G}{D} \right) \left(\frac{P_b}{T_b} \right) \quad (14)$$

where Re = Reynolds number, dimensionless

Q_b = gas flow - cu. ft./day at P_b and T_b

μ = fluid viscosity - $\frac{\text{lb}}{\text{ft sec}}$

D = inside diameter of pipe - inches

P_b = base pressure = psia

T_b = base temperature - $^{\circ}\text{R}$

For $P_b = 14.73$ psia and $T_b = 520^{\circ}\text{R}$ of this report

$$\text{Re} = 1.3526 \times 10^{-5} \frac{Q_b G}{D} \quad (15)$$

The friction factor will be found for a pipe roughness of 250 micro-inches. Colebrook's relationship, Handbook², p. 302, for evaluating the Moody friction factor in the form used by the AGA becomes

$$F_t = \frac{2}{\sqrt{f_M}} = 4 \log \frac{D}{EE} + 2.28 - 4 \log \left(1 + 9.34 \frac{D/EE}{\text{Re} \sqrt{f_M}} \right) \quad (16)$$

Drag Factor (F_f)

This factor is taken from Figure 13 corresponding to 200 degrees of bends per mile for plastic lined pipe and becomes 0.936.

Work of Compression

In obtaining the work of compression, one solves the basic flow equation which for horizontal flow and neglecting kinetic energy changes become² (p.315, Equation (7-66))

$$A = \int_{P_1}^{P_2} V dP \quad (17)$$

where A = work of compression

V = volume

P = pressure

During compression, the temperature rises and the relationship between volume and pressure is dependent upon the thermodynamic properties of the fluid, the heat transferred from the fluid during compression, and the flow inefficiencies which occur in the compressor.

It has been customary to employ the ideal gas relationship for adiabatic and reversible compression in finding the relationship

*Where EE = pipe roughness, inches

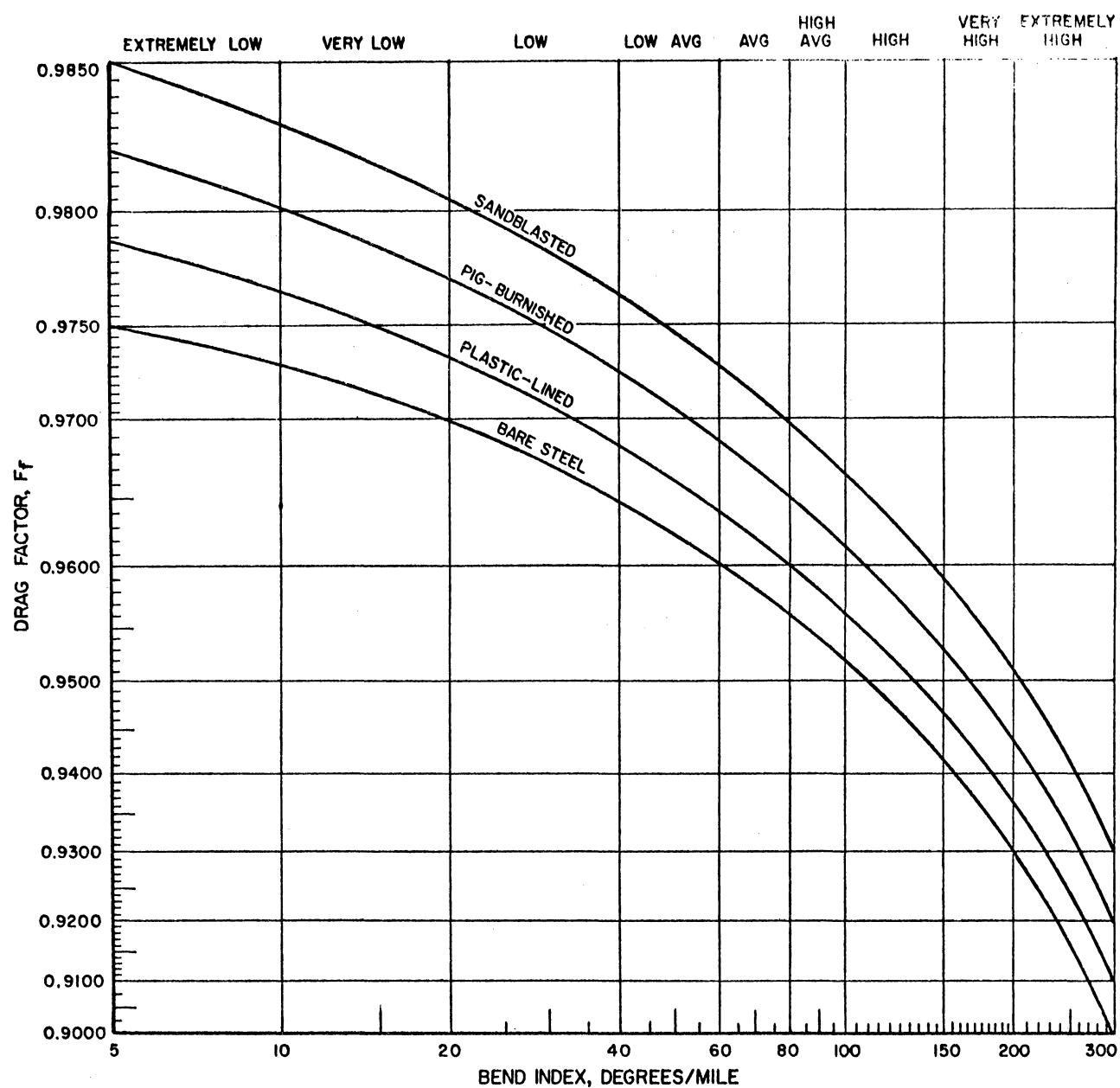


Fig. IV-1.-DRAG FACTOR AS A FUNCTION OF PIPE TYPE AND BEND INDEX
 (For typical lines constructed of 40-foot pipe joints welded in the field,
 and involving a valve setting approximately every 10 miles)

Fig. 13. Reproduced from Reference 1, Uhl et al.

between V and P, namely

$$PV^k = P_1 V_1^k = \text{constant} \quad (18)$$

where $k = \text{ratio of specific heats, } C_p/C_v$.

When using this relationship between P and V and still retaining the units of ft. lbs./lb. for A, one obtains

$$A = \frac{k}{k-1} \frac{53.241 T_f}{G} \left[\left(\frac{P_1}{P_2} \right)^{\frac{k-1}{k}} - 1 \right] \quad (19)$$

$G = \text{gas gravity, molecular weight}/29.0$

$T_f = \text{inlet temperature, } {}^\circ\text{R}$

Subscript 1 refers to the outlet pressure of the compressor or maximum pressure in the pipeline.

Subscript 2 refers to the inlet pressure to the compressor or minimum pressure in the pipeline.

Retaining the ideal gas relationship but converting units of A to horsepower per million cubic feet of Gas (14.73 psia and 60°F) per day, one obtains

$$A = 0.088 \frac{k T_1}{k-1} \left[\left(\frac{P_1}{P_2} \right)^{\frac{k-1}{k}} - 1 \right] \quad (20)$$

The question arises as to the most suitable method of converting this formula based on ideal gases to one which will handle actual gases, knowing that the deviation from ideal gases can be of a considerable magnitude. Also, at what temperature should the value of k be evaluated?

One way to evaluate the work is to invoke a parallel equation for work of compression based on enthalpy increase during adiabatic reversible compression. Equation (7-79), p. 316 of Reference 2 with units of A given above shows the equation to be

$$A = 0.0432 \Delta H \quad (21)$$

where ΔH is increase in enthalpy during pressure rise along a constant entropy line.

Enthalpy - Entropy diagrams have been prepared for natural gases at various gas gravities, such as 0.6, 0.7, 0.8, 0.9, and 1.0. By use of such charts and selected conditions, one can obtain the work of compression.

Three methods for correcting the deviation of the gas from ideal conditions have been used in modifying Equation (20). None of these equations are exact; they are expected to find the same result as integrating the VdP term with the actual volume used at each P. This would require multiplying each ideal volume by the Z which corresponds not only to the pressure involved but to the temperature to which the gas has risen at each increment of pressure rise. The three versions are

$$A_1 = 0.0854 \left(\frac{k}{k-1} \right) \frac{T_f Z_2}{\text{EFF}} \left[\left(\frac{P_1}{P_2} \right)^{\frac{k-1}{k}} - 1 \right] \frac{14.73}{14.65} \quad (22)$$

where $\frac{14.73}{14.65}$ is a conversion of the constant to correspond to 14.73 as P_b .

Z_2 = compressibility factor corresponding to inlet conditions.

EFF = compressor efficiency, taken as 0.80 for centrifugal compressors.

$\frac{P_1}{P_2}$ = compression ratio.

T_f = inlet gas temperature, $^{\circ}\text{R}$.

k = ratio of specific heat at constant pressure to that at constant volume.

This equation was recommended by the Natural Gas Pipeline Company of America.

$$A_2 = 0.0853 \left(\frac{k}{k-1} \right) \frac{T_f}{\text{EFF}} \left[\left(\frac{P_1}{P_2} \right)^{\frac{(k-1)Z}{k}} - 1 \right] \frac{14.73}{14.65} \quad (23)$$

The equation is from the Handbook², p. 316.

$$A_3 = \frac{44.64}{\text{EFF}} \left(\frac{T_f}{520} \right) \left(\frac{k}{k-1} \right) \left(\frac{Z_1 + Z_2}{2} \right) \left[\left(\frac{P_1}{P_2} \right)^{\frac{k-1}{k}} - 1 \right] \quad (24)$$

This equation was supplied by the Columbia Gas System Service Corporation.

From this study, see Table 4, it was agreed to use Equation (22) for computing the work of compression. The equation uses the compressibility factor at the entrance to the compressors as a direct factor on the ideal gas quantity.

Economics of Single Phase Pipeline Flow for Mixtures of Natural Gas and Liquid Hydrocarbons

The purpose of this study is to devise a method of calculation for predicting the conditions at which natural gas and liquid hydrocarbons may be combined as a single phase system and transported economically in pipelines. It is known that the light hydrocarbons can be maintained in single phase with natural gases under specified temperatures and pressures. The ability to carry such light hydrocarbons over long distances along with natural gas may become economical if pressures higher than those normally used in pipelines are available. The use of higher strength steels would be advantageous in this connection. This survey proposes to set up a master economic program of pipeline flow in which the prime variables are investigated systematically to find the minimum cost of transportation.

During the course of the study, three programs have been written for the economic calculations. At the time the first

TABLE 4
Comparison of Work of Compression Calculations

Inlet at 1500 psia, 60°F (520°R)
 $k (= \frac{C_p}{C_v})$ at Inlet Conditions

Gas Gravity = 0.6		$\frac{P_1}{P_2}$		A_{chart}		A_1	A_2	Z_2 , exponent	$\left(\frac{Z_2 + Z_1}{2} \right)$	A_3	T_1	ideal °R	chart °R
				100	95	90	85	80	Z_2 , factor				
1.35	12.3	12.9	13.7	14.6	15.4	16.6	17.5	19.0	18.8	12.6	12.25	13.2	553
1.45	14.0	14.7	15.6	16.6	17.5	18.0	19.3	20.0	20.05	15.85	15.25	16.8	564
1.55	16.2	17.0	18.0	19.3	19.0	19.0	19.0	19.0	18.05	18.8	18.05	20.0	570
Gas Gravity = 0.8													
1.35	10.6	13.3	11.2	11.8	12.5	13.8	15.8	18.6	17.8	13.3	10.4	12.5	548
1.45	13.4	16.8	15.2	13.8	15.8	17.6	18.6	18.6	17.8	14.9	11.85	16.0	555
1.55	14.9	15.7	16.5	17.6	17.6	17.6	17.6	17.6	17.6	14.1	14.1	19.3	563
Gas Gravity = 1.0													
1.35	6.7	7.1	7.4	7.9	8.4	9.1	9.8	9.8	9.8	8.65	5.59	9.9	544
1.45	7.25	7.6	8.0	8.5	9.1	9.8	10.5	10.5	10.5	10.85	6.88	12.87	550
1.55	7.85	8.25	8.7	9.25	9.8	10.5	11.2	11.2	11.2	12.9	8.12	15.6	555

* Z_1 refers to value taken at T_1 , "chart". NOTE: Subscript 2 refers to inlet to compressor and subscript 1 to outlet.

progress report was written, the calculations were directed toward finding the size of pipe and number of compression stations to give the minimum cost for a fixed flow rate, composition and strength of pipe. It included a Lagrangian method of undetermined multiplier to predict the compression ratio for the minimum cost before proceeding to calculate the minimum cost. The maximum pressure in the pipeline was fixed.

The second method fixed the composition, pipe strength, diameter of the pipe, and maximum pressure in the pipeline as 1.65 times the minimum permissible pressure for the composition.

The third method--which is the one used in reporting the results herein--fixes

Composition
Minimum pipeline pressure
Strength of steel
Pipe diameter.

For a series of flow rates, the cost of transporting gas is found for an increasing number of compression stations (decreasing compression ratio) until the cheapest cost is found for that flow rate. The flow rate is increased in increments, and the procedure is repeated until the cost increases over the previous minimum. Calculations are made for 16, 24, and 30 inch OD pipelines.

The economic computer program has been devised to compute the cost of transporting a thousand cubic feet (Mcf) of natural gas 100 miles when using a pipeline 1000 miles long. The cost includes both amortization of the investment and operating expenses.

The basic calculations made fix the following items:

- a. Composition
- b. Strength of steel
- c. Pipe diameter
- d. Flow rate

- e. Number of compressor stations
- f. Minimum pressure or inlet pressure to compressors (P_2).

For a given case, the pressure drop between stations is computed by Equation (12), and then the horsepower is computed by Equation (22). For this particular case, the cost of transportation can be found.

The pipeline investment is obtained by finding the tons of steel and laying cost. The pipe wall thickness is calculated by Equation (A1) using the maximum pressure calculated in connection with pipeline flow. From the wall thickness, diameter, and cost per ton of steel, the pipeline and communication system investment is computed (IINVL).

$$IINVL = (YW + N \times OD + H) \frac{10^7}{QB} \quad (25)$$

Y = \$/ton of steel

W = tons of steel per mile (see Appendix A)

N = cost of laying pipe, \$/mile/in. OD = \$1200

OD = outside diameter of pipe in inches

H = cost of communication system, \$/mile = \$3000

QB = cubic feet of gas flowing per day

IINVL = cents/100 miles/Mcf/day.

The station investment (IINVS) is obtained from

$$IINVS = \left[X + \frac{X_1 10^6}{A \times QB} \right] 10 \frac{A}{L} \quad (26)$$

IINVS = cents/100 miles/Mcf/day

X_1 = fixed station cost = \$270,000

X = cost per horsepower = \$165

A = horsepower per station per million cubic feet per day

NOS = number of stations for 1000 mile line, $\frac{L_t}{L}$

L_t = total length of pipeline, miles (used as 1000)

L = length of line between stations, miles

QB = flow rate in cubic feet per day.

The transportation charge due to investment is taken as 15% per year. Therefore the cost of transportation due to investment when flowing gas 365 days per year becomes

$$CMMAM = [IINVL + IINVS] \frac{0.15}{365} \quad (27)$$

CMMAM = transportation charge or cost due to investment in cents/100 miles/Mcf transported.

The operating charges are computed as follows by adding the fuel cost, compressor operating charge, the gas loss, the line maintenance, and the administration expense:

$$\begin{aligned} CMMOP &= \left[(\text{FPHPR}) 24 \times 365 \times CF + CLMS \right] 10 \frac{A}{L} \frac{FOOP}{365} \\ &\quad + \left[LG \times GASCST \times \frac{10^3}{LT} + \frac{CLML \times 10^6}{QB \times 365} \right] 10 + \frac{AD^*}{3650} \end{aligned} \quad (28)$$

CMMOP = transportation cost for operations in cents/100 miles/Mcf

FPHPR = Mcf per horsepower - hour = 8.7×10^{-3}

A = horsepower per station per million cubic feet per day

QB = flow rate of gas, cubic feet per day

CF = fuel cost for compressors, \$0.20/Mcf

CLMS = maintenance charge of stations, \$/hp/yr = \$19

FOOP = fraction of time in operation = 1.0

NOS = number of stations, used as $\frac{1000}{L}$

GASCST = charge for gas lost, \$/Mcf, used as \$0.20 = CF

LG = gas loss in passing through line, fraction, used as 0.005

LT = total length of line = 1000 miles

CLML = labor and maintenance cost for line, \$/mile/yr, used as \$850

AD = administration expenses, \$/mile/yr/MMcf/day.

The total cost of transportation then becomes the cost due to investment plus that due to operations.

$$CY = CMMAM + CMMOP \quad (29)$$

CY = cents/100 miles/Mcf.

* After all calculations had been made, it was found that the correct expression should have been AD/36.5.

TABLE 5
Economic and Other Factors Used in Study

Length of line (L_t) 1000 miles
 Flowing temperature (T_f) 60°F
 Pipe strength (S) 65,000 psi and 100,000 psi
 Pipe diameter (OD) 16 inch, 24 inch and 30 inch (OD)
 Pipe roughness EE = 250 micro-inches
 Bend Index 200° per mile
 Longitudinal joint factor for pipe (E) 1.0
 Pipe design factor (F) 0.72
 Compressor efficiency, EFF, 80%
 Pipe cost (Y) 65,000 psi strength, \$265 per ton
 100,000 psi strength, \$384 per ton
 Cost of laying pipe (N) \$1200 per inch OD per mile
 Communication system cost (H) \$3000 per mile
 Compression station cost ($x_1 + X$) \$270,000 + \$165 per horsepower
 Cost of fuel (CF) 20 cents/Mcf
 Fuel consumption (FHPHR) 8.7×10^{-3} Mcf per horsepower - hour
 Cost of labor and maintenance for pipeline (CLML) \$850/mile/yr
 Cost of labor and maintenance for compressor stations (CLMS)
 \$19/hp/yr
 Gas loss (LG) = 0.005 fraction of flow at 20 cents/Mcf
 Administration expenses (AD) $\frac{1}{3650}$ cents/100 miles/Mcf
 Annual investment charge, 15% per year
 Line assumed to flow at 100% capacity 365 days/yr

Calculation of Transportation Costs

With the computer program given in Appendix B, the costs of transporting the 10 mixtures given in Table 1 were computed as shown by example in Appendix C. For each mixture, the minimum single phase pressure was determined as listed in Table 1. To this pressure, 100 psi was added to find the minimum pipeline pressure. Calculations were made for this minimum pipeline pressure and for a pressure 400 psi higher.

The variables used in addition to pressure when surveying the cost of transporting the various mixtures were

Pipe strength
Pipe diameter
Number of stations in 1000 miles
Flow rate.

For each composition, strength of steel, minimum pressure, and pipe diameter, a series of calculations are made with increasing number of stations at each of a series of flow rates. As shown on Figure 14, for a fixed composition, strength of steel, pipe diameter, and minimum pipeline pressure, the cost is found at each of a series of flow rates for an increasing number of compression stations until the cost goes through a minimum value. Figure 14 is for the 80% natural gas-20% propane mixture. The starting minimum number or compressor stations is set by including it as data in the computer program, and the costs are calculated for it and the next larger number of stations in sequence until the cost rises. At this point, the flow is incremented, and the procedure is repeated as long as the minimum cost decreases for the new flow rate.

The minimum cost at each flow rate of Figure 14 is plotted versus flow rate on Figure 15. All three pipe sizes are included on this one plot. As would be expected, the larger pipe sizes give lower costs of transportation. Figure 15 for the 80% natural

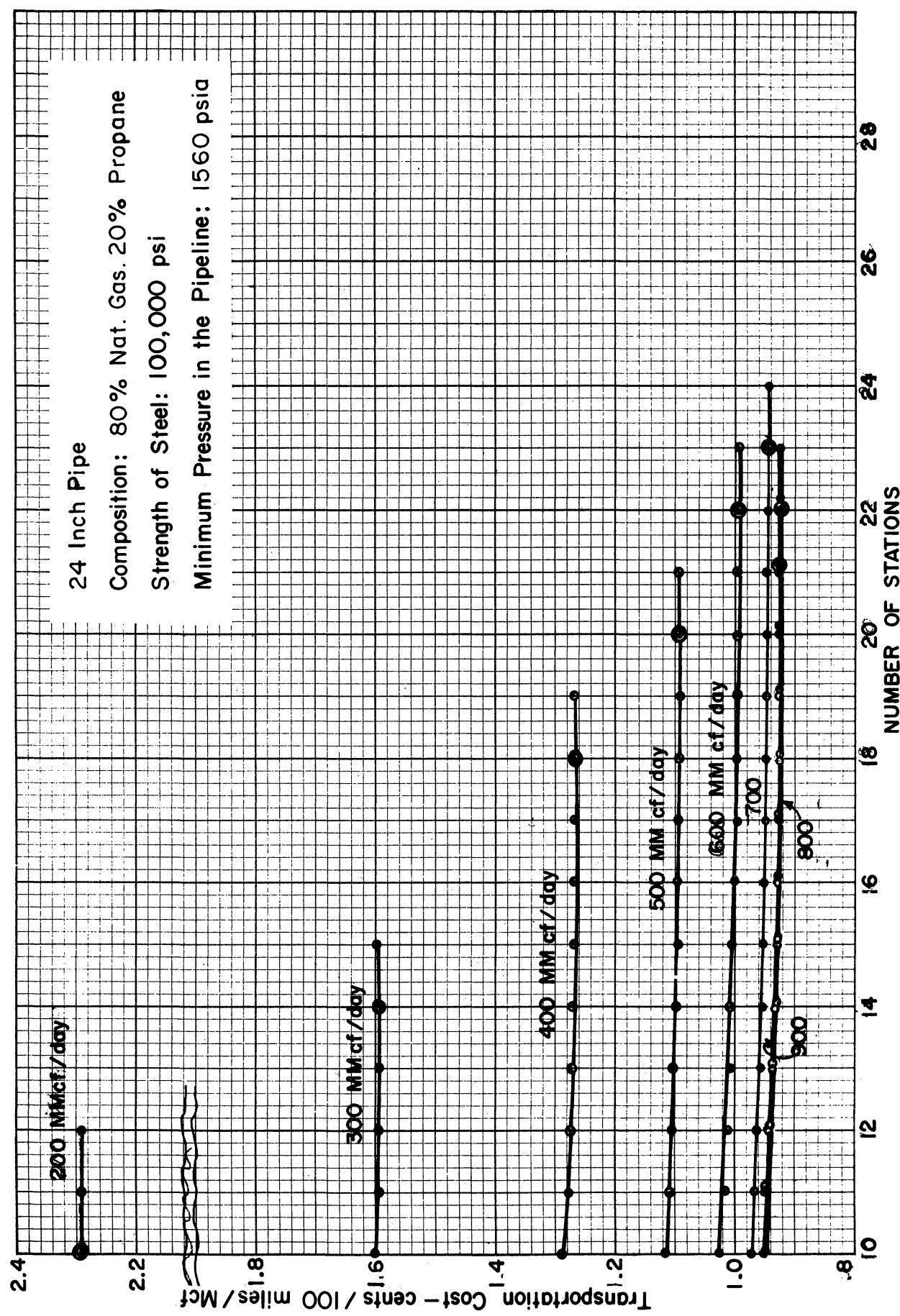


Figure 14. Cost of Transporting Gas as a Function of Number of Stations

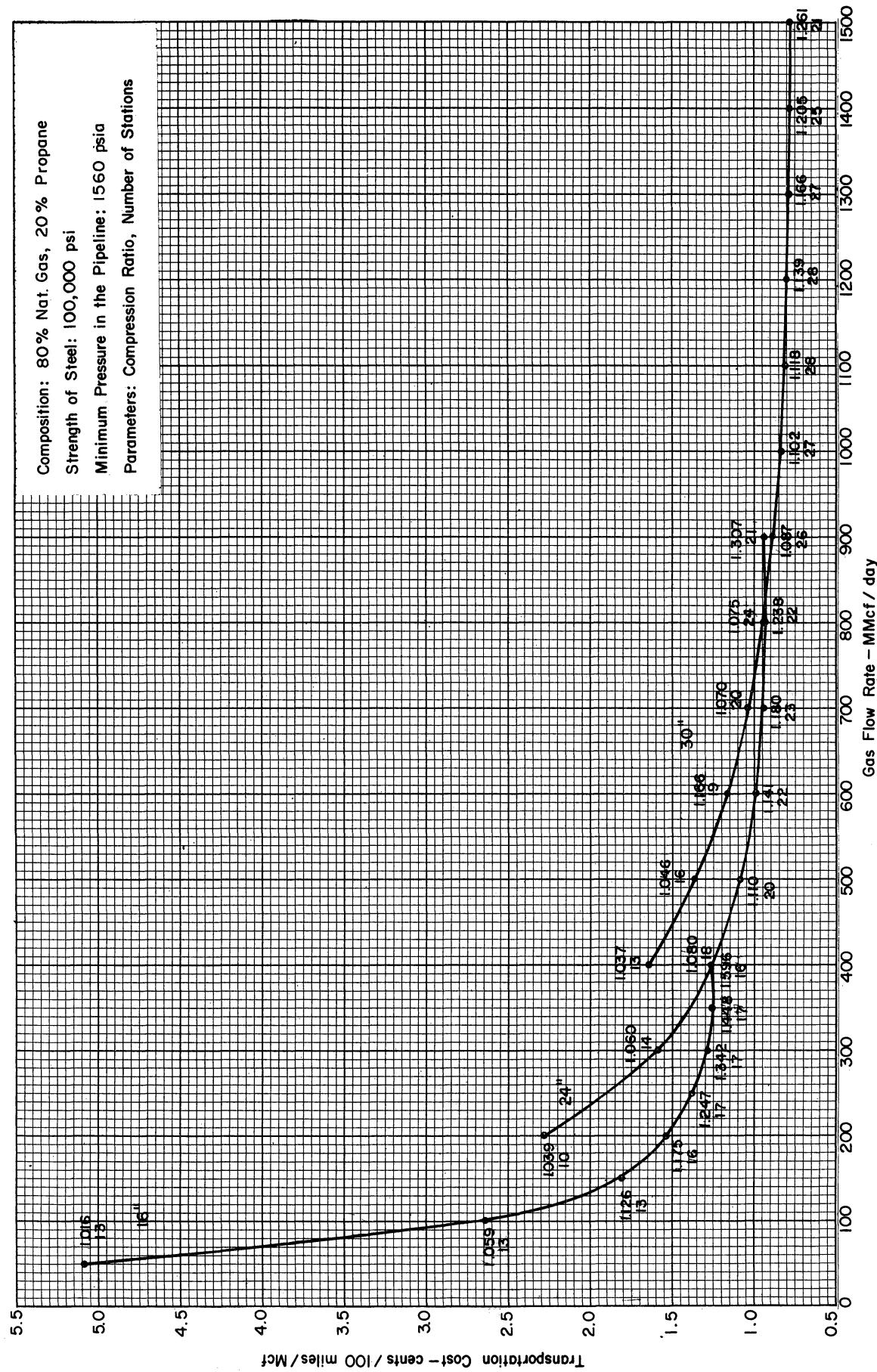


Figure 15. Cost of Transporting Gas as a Function of Flow Rate, 80% Natural Gas—20% Propane

gas-20% propane with 100,000 psi steel is included in the report. The remainder of these plots is included as Appendix E.

The original calculations for the 65,000 psi steel cases were based on \$228 per ton rather than the \$265 agreed upon as the proper cost. Rather than redoing the entire calculations, a program was written to compute only the new cost based on the horsepower per million cubic feet, pipe thickness, number of stations, and flow rate already determined. These computations were made for the 65,000 psi cases using five different number of stations adjacent to the minimum point previously found.

The 100,000 psi steel cases were run correctly as shown in the example print-out in Appendix B.

The minimum cost for each case is listed on Table 6, 66 values for the 65,000 psi steel and 45 values for the 100,000 psi steel since only 7 compositions were run in the latter case.

These costs for transporting various streams composed of natural gas and condensates are used to predict the cost of transporting the liquid portion of the gas stream.

The strength of the steel is a variable in the computer program, but only the 65,000 psi steel at \$265 per ton and the 100,000 psi steel at \$384 per ton were investigated. At these prices, the lines designed to use 100,000 psi steel gave cheaper transportation costs than the 65,000 psi steel.

The flow rate is a key variable in determining the cost of transporting the mixtures. Figure 16 is a plot of cost of transportation versus flow rate, with an identification of the pipe diameter and minimum pipeline pressure all for the 100,000 psi steel. The relationship is much as one would expect, a decreasing cost with higher flow rate.

The effect of pressure on the cost of transportation is indicated on Figures 17 and 18; the flow rate is identified for each point, all for the 100,000 psi steel. It is difficult to separate flow rate and pressure, but it appears that the cost of transportation decreases with pressure up to about 2500-3000

TABLE 6A

Cost of Transporting Fluids
cents/100 miles/Mcf

Mixture	Based on 1000 mile pipeline, 65,000 psi steel						30" diameter					
	Min.	16" diameter	24" diameter	Min.	Pipe Flow Min.	Min.	Pipe Flow Min.	Min.	Pipe Flow Min.	Min.	Pipe Flow Min.	Mol.
	Cost cents /100 miles /Mcf	Pipe Thick- ness in.	Rate psia /day	Cost cents /100 miles /Mcf	Rate psia /day	Thickness in.	Cost cents /100 miles /Mcf	Rate psia /day	Thickness in.	Cost cents /100 miles /Mcf	Rate psia /day	wt. psia
Natural Gas	1.99	.19	150	700	1.37	.29	400	700	1.16	.36	600	700
	1.61	.31	250	1100	1.17	.39	600	1100	.997	.445	900	1100
	1.45	.376	300	1560	1.06	.516	800	1560	.904	.585	1200	1560
	1.338	.477	400	2100	.994	.626	900	2100	.858	.754	1500	2100
10% Propane	1.984	.194	150	700	1.374	.273	400	700	1.148	.356	800	700
	1.581	.304	250	1100	1.148	.404	600	1100	.979	.443	900	1100
20% Propane	1.329	.389	350	1560	.972	.502	800	1560	.833	.592	1300	1560
	1.27	.447	400	1960	.938	.590	900	1960	.807	.708	1500	1960
30% Propane	1.23	.39	400	1540	.893	.505	900	1540	.765	.60	1500	1540
	1.20	.437	400	1940	.877	.598	1000	1940	.761	.696	1500	1940
5% Propane and Butane	1.388	.404	350	1600	1.017	.514	800	1600	.873	.609	1300	1600
5% Butane	1.32	.458	400	2000	.977	.633	1000	2000	.84	.724	1500	2000
5% Butane	1.509	.365	300	1300	1.104	.46	700	1300	.942	.525	1100	1300
	1.39	.416	350	1700	1.02	.53	800	1700	.88	.627	1300	1700
10% Butane	1.338	.455	400	1880	.982	.58	900	1880	.846	.701	1500	1880
	1.308	.514	450	2280	.97	.707	1100	2280	.846	.806	1500	2280
15% Butane	1.293	.453	400	2000	.953	.632	1000	2000	.82	.725	1500	2000
	1.30	.515	400	2400	.957	.731	1100	2400	.877	.853	1500	2400
Kurata S-2	1.343	.555	400	2660	.985	.787	1100	2660	.941	.909	1500	2660
	1.383	.642	450	3060	1.034	.890	1100	3060	1.018	1.037	1300	3060
Kurata S-4	1.353	.579	450	2630	.9937	.789	1100	2630	.8875	.926	1500	2630
	1.443	.649	450	3030	1.070	.896	1100	3030	1.03	1.036	1300	3030

TABLE 6B

Cost of Transporting Fluids
cents/100 miles/Mcf

Mixture	Based on 1000 mile pipeline, 100,000 psi steel						30" diameter						
	16" diameter	24" diameter	Pipe Min.	Flow Min.	Min.	Flow Min.	Pipe Min.	Flow Min.	Min.	Cost	Thick-	Rate Press.	Mol.
	Min. Cost cents/100 miles/Mcf	Pipe Thick- ness in.	Flow Rate Press. MMcf psia	Min. Cost cents/100 miles/Mcf	Pipe Thick- ness in.	Flow Rate Press. MMcf psia	Min. Cost cents/100 miles/Mcf	Pipe Thick- ness in.	Cost cents/100 miles/Mcf	Thick- ness in.	Rate Press. MMcf psia	wt.	
Natural Gas	1.56 1.36 1.26	.191 .264 .306	250 350 400	1100 1560 2100	1.12 1.00 .94	.261 .327 .402	600 800 900	1100 1560 2100	.96 .86 .82	.281 .377 .479	900 1200 1400	1100 1560 2100	17.39
10% Propane	1.93 1.52	.124 .186	150 250	700 1100	1.32 1.10	.189 .259	400 600	700 1100	1.11 .94	.234 .308	700 1000	700 1100	20.06
20% Propane	1.26 1.20	.251 .287	350 400	1560 1960	.92 .88	.322 .395	800 1000	1560 1960	.79 .74	.391 .467	1400 1600	1560 1960	22.73
5% Propane and 5% Butane	1.32 1.25	.260 .297	350 400	1600 2000	.97 .92	.332 .402	800 1000	1600 2000	.831 .802	.403 .460	1400 1400	1600 2000	20.8
10% Butane	1.26 1.22	.292 .331	400 450	1880 2280	.93 .91	.373 .450	900 1100	1880 2280	.80 .80	.447 .510	1500 1500	1880 2280	21.47
Kurata S-2	1.24 1.28	.371 .406	450 450	2660 3060	.92 .96	.504 .590	1100 1300	2660 3060	.81 .90	.610 .680	1700 21500	2660 3060	24.96
Kurata S-4	1.26 1.33	.371 .409	450 450	2630 3030	.93 .99	.504 .572	1100 1100	2630 3030	.84 .91	.596 .680	1500 1500	2630 3030	29.15

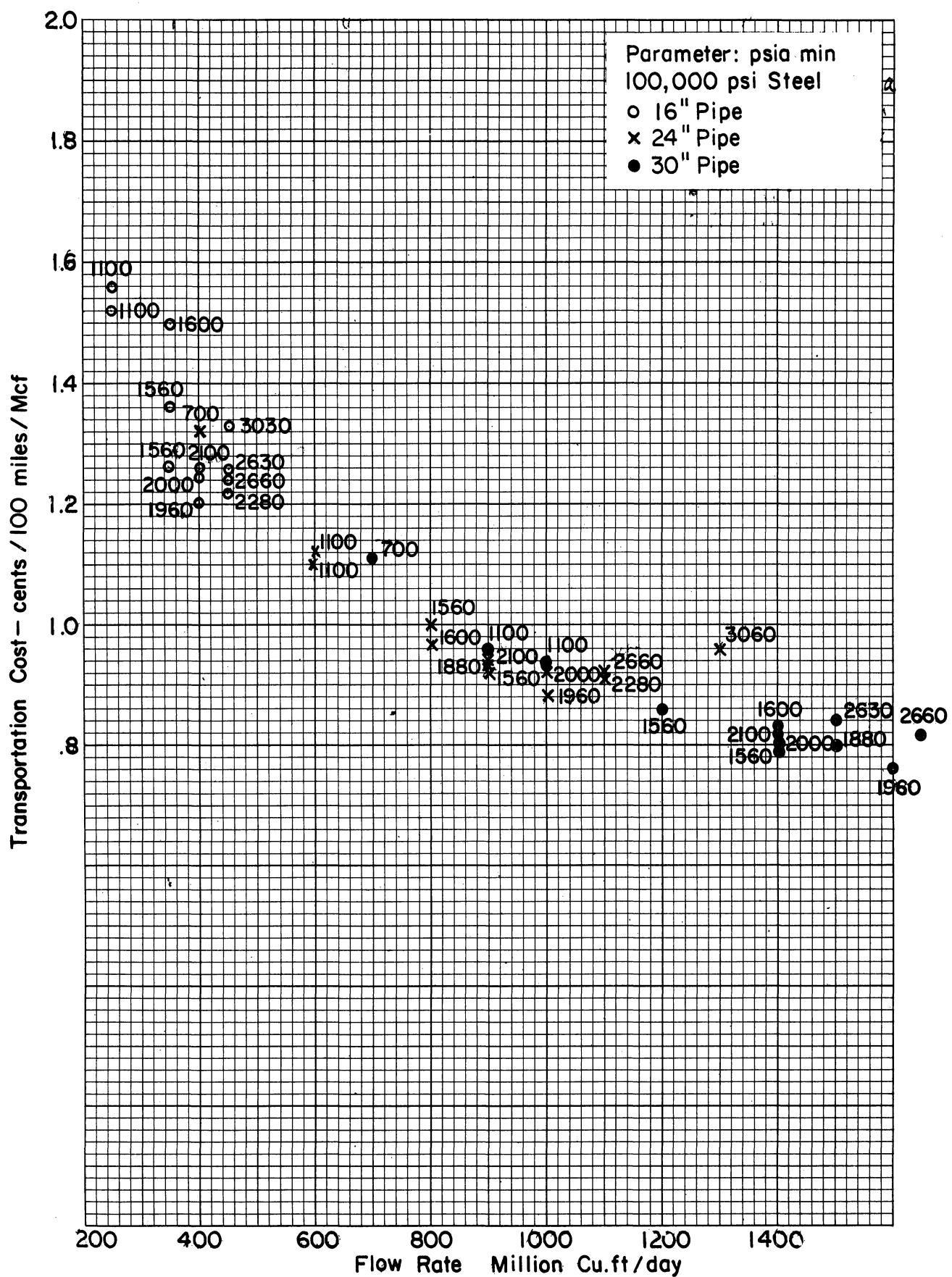


Figure 16. Effect of Flow Rate on Cost

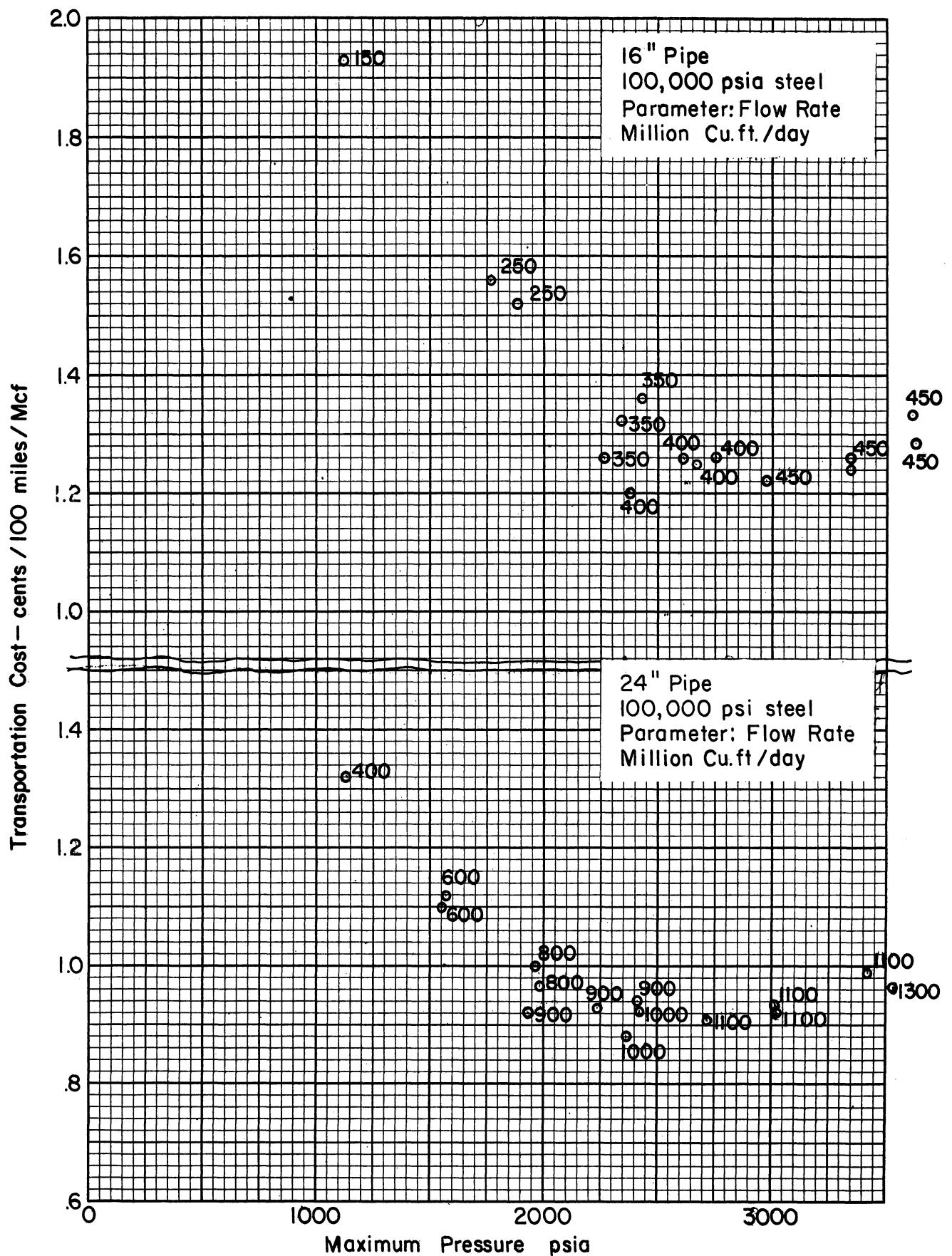


Figure 17. Effect of Pressure on Cost of Transportation

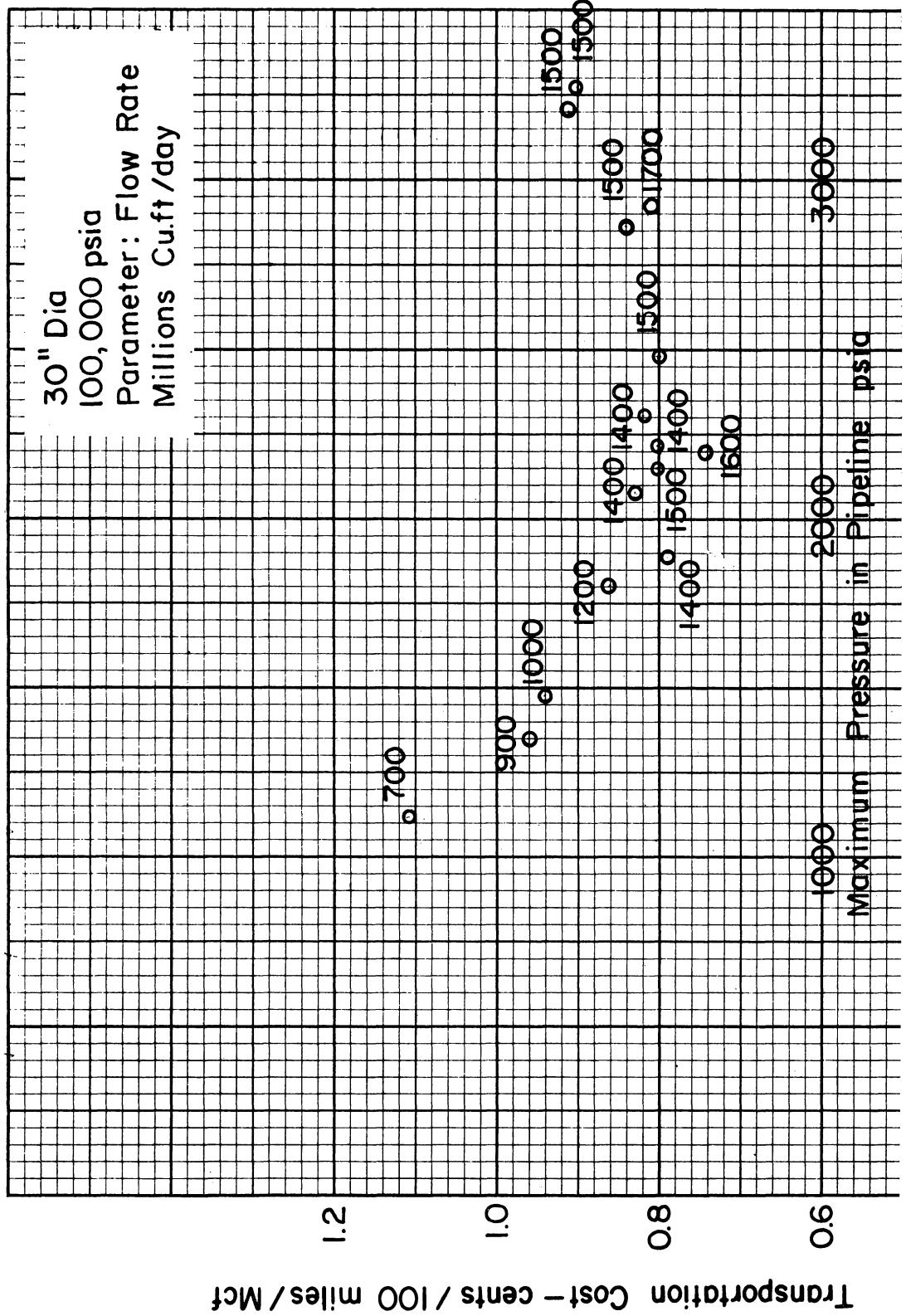


Figure 18. Effect of Pressure on Cost of Transportation

psia. It should be noted that the maximum pipeline pressure is plotted on these figures. The maximum pressure was found by multiplying the minimum pressures in Table 6 by the compression ratio listed on Figure 15. A similar plot using minimum pressures for the 65,000 psi steel with the incorrect cost of steel as distributed in the progress report gave a similar impression.

The effect of molecular weight on cost is given on Figures 19 and 20. Again, it is difficult to be sure the effect of flow rate is not obscuring the independent effect of molecular weight. Generally, increases in molecular weight are accompanied by a decrease in cost. That is, the addition of propane or butane gives a lower overall cost of transportation per unit of fluid. The higher molecular weights represented by the Kurata mixtures give rising costs, but they could be influenced by the unusually high operating pressure needed to keep the mixtures in single phase.

Effect of Temperature

These calculations have all been made at 60°F. The nature of the compressibility factors is such that there would be less effect on the density by adding higher molecular weight hydrocarbons at higher temperatures and greater effect on density at lower temperatures. Thus, the effect of increased pipeline capacity observed in winter when the temperatures are lower would be accentuated for gases carrying condensates.

Cost of Hauling Liquids

The objective is to find the cost of transporting the liquid constituent. To do this, it appears logical to set some cost for transporting the gas alone. Then it is necessary to set some total flow rate and divide the composition into the gas portion and the liquid portion. By knowing the transportation cost of the mixture and that of the gas alone, one can find the cost of transporting the liquid.

One method of proceeding is to use a standard cost for transporting gas. The value of 1.1 cents/Mcf/100 miles or

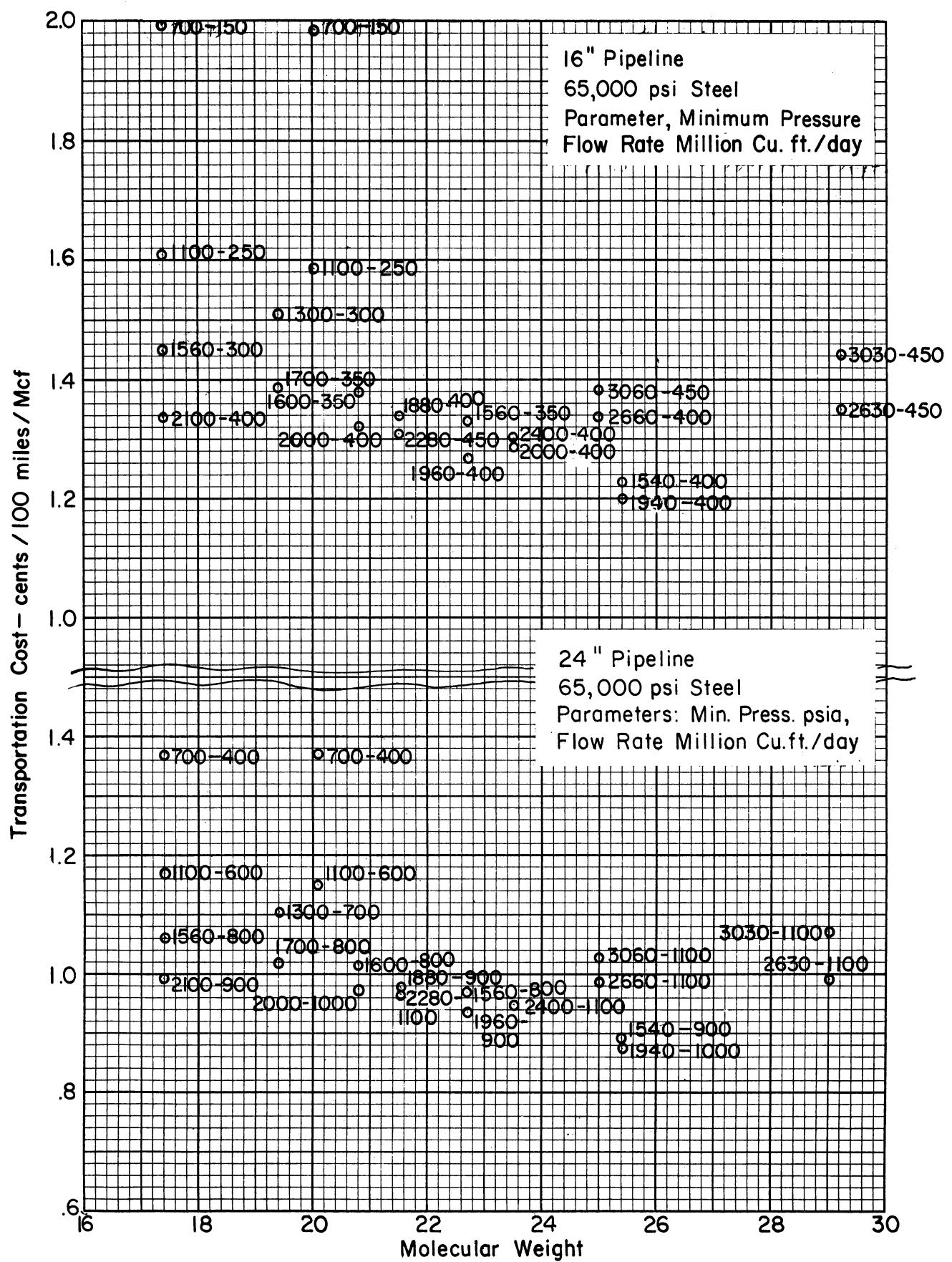


Figure 19. Effect of Molecular Weight on Cost of Transportation (65,000 psi steel)

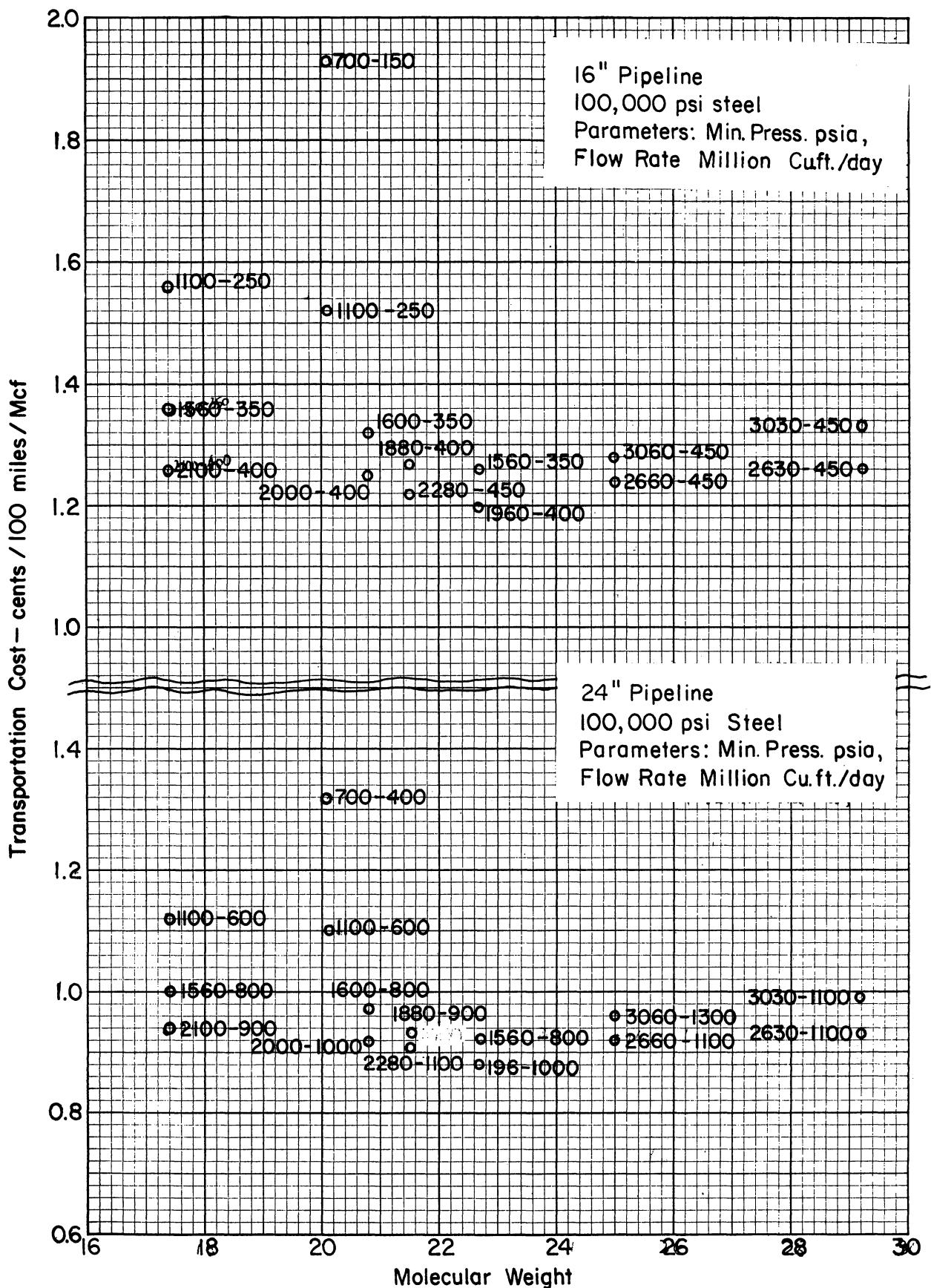


Figure 20. Effect of Molecular Weight on Cost of Transportation (100,000 psi steel)

11 cents/Mcf/100 miles is used as the best value at 500-600 million cu. ft./day and is the minimum value considered. At a flow rate of 900 million cu. ft./day, the cost of transporting gas is down to 0.95 cents/Mcf/100 miles.

Table 7 gives two example calculations of the cost of hauling liquids. The propane cost of 13.4 cents per barrel per 1000 miles for Case I is considerably below that of Colonial Pipeline Company tariff for products shown in Table 8, but of course there is some cost of separating the liquids from the gas at terminal. Case II is for condensate where the mixture costs more to haul than does natural gas because a 16" line is used. Here, the condensate is still cheaper than product pipeline tariffs.

In examining Tables 6A and 6B, one can find for 24" or 30" diameter pipelines several cases where the flow rates are the same for gas and for specific mixtures. Taking such pairs of conditions, the cost of hauling liquids is computed for several mixtures on Table 9. These calculations show very favorable rates for transporting liquids when extremely large quantities are involved. These results are ample justification for further pursuit of these concepts for hauling liquids, including investigation of separation costs.

Conclusions

1. The addition of the light hydrocarbons propane, butane, and condensates to natural gas flowing in pipelines increases the total flow capacity of the line and reduces the cost of transportation.

2. The pressures required to keep these liquids in single phase are above present pipeline pressures, but not high enough to increase the cost of transporting the gas mixtures.

3. The cost of hauling liquids, dissolved in natural gas without consideration of separation costs, is some 35-50% of the tariffs posted by the Colonial Pipeline Company for liquid products

4. A slight advantage was found for using 100,000 psi steel at \$384 per ton over 65,000 psi steel at \$265 per ton.

5. Operating pressures up to some 2500 psi in pipelines appear to decrease the cost of transporting single phase natural gas liquid mixtures.

6. This study should be a sufficient basis for further consideration of transporting liquid along with natural gas, whenever large quantities of liquid need to be transported over long distances.

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TABLE 7

Calculation of Cost of Transporting Liquids
in Gas Pipeline for 1000 Miles

Case I. 20% Propane-80% Natural Gas

65,000 psi steel

24 inch pipe

Minimum pressure 1960 psia

Flow rate 900 million cu. ft./day
(131,000 bbls/day propane)

Cost of transportation = 0.938 cents/100 miles/Mcf

Using a cost for natural gas of 0.95 cents/100 miles/Mcf
(9.5 cents/1000 miles) and dividing stream into
 $0.7816 \times 900,000 = 704,000$ Mcf of gas and 131,000 bbls $(0.2184 \times \frac{900,000,000}{35.8 \times 42})$ of propane.Cost per day for 900 million at $0.938 \times \frac{10}{100} = \$84,500$ Cost per day for 704,000 Mcf at $0.95 \times \frac{10}{100} = \frac{66,900}{\$17,600}$

In this case the propane is hauled 1000 miles for

 $\frac{17,600 \times 100}{131,000} = 13.4$ cents/bbl.Case II Kurata S-4 Gas Condensate

100,000 psi steel

Cost of hauling 1.26 cents/Mcf/100 miles

Flow rate 450 million cu. ft./day

16 inch pipe

Minimum pressure 2630 psia

84,300 bbls condensate/day

352 million cubic feet of natural gas

Cost of hauling mixture 1000 miles $450,000 \times 1.26 \times \frac{10}{100} = \$56,700$ Cost of hauling natural gas $352,000 \times 1.1 \times \frac{10}{100} = \frac{38,700}{\$18,000}$ The cost of hauling the condensate is $\frac{18,000 \times 100}{84,300} = 21.4$ cents/bbls/1000 miles.

TABLE 8
Published Tariff for Product Transportation
by Colonial Pipeline Company

Entrance to Pipeline	Exit from Pipeline	Estimated Miles	Cost cents /bbl	Cents /bbl 1000 miles
Beaumont (Texas)	Philadelphia (Pa.)	1380	31.75	23.0
Lake Charles (Louisiana)	Philadelphia (Pa.)	1320	31.25	23.7
Pasadena (Texas)	Philadelphia (Pa.)	1450	33.50	23.01
Beaumont (Texas)	Chattanooga (Hamilton County)	860	24.65	28.7
Beaumont (Texas)	Knoxville (Tenn.)	960	25.75	26.8
Beaumont (Texas)	Nashville-Davidson (Davidson County)	990	26.05	26.3

TABLE 9
Cost of Hauling Liquids

Composition	Dia.	Min. Press. psia	Cost cents/1000 miles Mixture	Cost cents/1000 miles Gas	Total Rate MMcf /day	Gas Rate MMcf /day	Liq. Rate 1000 bbls /day	Cost Mixture \$1000 /day	Gas Cost \$1000 /day	Liq. Cost \$1000 /day	Costs/bbl /1000 miles
<u>65,000 psi steel</u>											
30% Propane	24	1540 2100	8.93	9.94	900	616	189	80.4	65.6	14.8	7.8
20% Propane	24	1960 2100	9.38	9.94	900	704	131	84.4	70.0	14.4	11.0
5% Butane	24	1300	11.04	11.04	700	650	37.8	77.4	71.8	5.6	14.8
S-2	24	2660 2100	9.85	9.94	1100	934	143	108.3	92.9	15.4	10.8
<u>100,000 psi steel</u>											
10% Propane	24	1100 1560	11.0	11.2	600	528	48.1	66.0	59.1	6.9	14.3
20% Propane	24	1560 1560	9.2	10.0	800	625	116	73.6	62.5	11.1	9.6
S-2	24	2660	9.2	9.6	1100	934	144	101	89.1	11.1	7.7
S-4	24	2630	9.3	9.6	1100	860	206	102.3	82.5	19.8	9.6

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APPENDIX A
FORMULA FOR COST OF PIPE

The thickness of the pipe can be calculated by the following formula (Eqn. 841.1, ASME code B31.8, 1963)²⁵

$$THK = \frac{(P_1)(OD)}{(2S)(F)(E)} \quad (A-1)$$

The cost of pipe is given by

$$\begin{aligned} YW &= Y \times 5280 \times \frac{\pi(OD^2 - (OD - 2THK))^2}{4 \times 144} \times \frac{62.4 \times 7.86}{2000} \quad (A-2) \\ &= 7.08 (OD^2 - (OD + 4THK^2 - 2(OD)(2THK)))(Y) \end{aligned}$$

$$YW = 28.2 THK(OD-THK)(Y) \quad (A-3)$$

- Y = Cost of pipe material, \$/ton
- OD = Outside diameter of pipe, in.
- ID = Inside diameter of pipe, in.
- P₁ = Maximum pressure in pipe, psig
- ρ_s = Density of steel 490, lb/ft³
- S = Minimum specified yield strength of steel, psi
- F = Construction type design factor as given in Table 841.11, ASME code B31.8, 1963 (Ref. 25) (taken as 0.72)
- E = Longitudinal joint factor as given in Table 831.12, ASME code (taken as 1.0)
- T = Temperature derating factor for project (taken as 1.0)
- YW = Cost of pipe, \$/mile

APPENDIX B
COMPUTER PROGRAM AND NOMENCLATURE

The computer program written in MAD solves in sequence the flow equation for pressure drop in the pipeline, the horsepower requirement for compression of the gas, and economical equations for obtaining the cost of transportation of the gas. Table B-1 is a print-out of the program as used in the pipeline flow calculations reported herein. It is the third program developed. The program was written to make it as general as possible and not all of the possible cases have been used in the example problem. The hand calculation follows in Appendix C. The nomenclature follows the program.

It is believed that a reading of the program along with the nomenclature will provide an understanding of the calculations. Some explanations are given where they are thought to be necessary.

The program reads all the data. It stores the information about the composition and initializes all the counters for diameter of the pipe, the strength of steel, minimum specified flow rate, maximum pressure in the pipeline and the initial number of stations. The diameter of the pipe and the strength of steel are fixed parameters. A provision is made to allow for different initial and final flow rates, and number of stations. The initial flow rate corresponding to the diameter selected is read in as data. The switch (COMPSW) is turned "on" or "off" by making its values 1 or 0. By having it off the program computes the composition of the gas based on addition of propane and butane in barrels/day. If the switch is "on" the program expects the compositions of gas specified as part of the data.

The next step is to compute the pseudo critical properties of the mixture and the molecular weight. The program allows for N₂, CO₂, and methane to heptanes in the mixture. The gas gravity is computed by dividing the molecular weight of the mixture by 29, the molecular weight of the air. From gas gravity the gas gravity factor is computed.

The next step is to have the machine know what the minimum pressure in the pipeline is going to be. If this pressure is already known then the value of MMWT should be set higher than the molecular weight of the mixture. As we do not expect any mixture to exceed a molecular weight of 40, this number can be safely used. The program also has provisions for computing the minimum pressure as a function of the molecular weight at 40° or 60°F. In order for the computer to calculate the minimum pressure the value of MMWT should be set less than the expected molecular weight. In case nothing is read in the data for MMWT, it will be taken as zero making it safe automatically. This feature was put in the program when a correlation of minimum pressure with molecular weight was being developed. Whenever the interpolation is required as in the calculation of viscosity, compressibility factor and ratio of specific heats, the program makes use of subroutine TAB. A comment "unsuccessful interpolation" is printed whenever some error in interpolation occurs. The TAB subroutine follows the program.

The maximum pressure in the pipeline is computed by adding 100 psia to the minimum pressure (for safety when handling condensate) and multiplying by the assumed compression ratio. The initial value for the starting number of stations is furnished as input data.

The length between the compressor stations is computed. The thickness of the pipe, the inside diameter of the pipe and the line diameter factor are computed. Since both the maximum and the minimum pressure in the pipeline are known the average pressure is computed. From this value the reduced pressure is calculated; the reduced temperature is computed from the temperature of the line supplied as part of the data.

Based on the reduced temperature and pressure the compressibility factor of the single phase fluid is computed by Sarem's¹⁷ method. A separate subroutine (ZFAC) has been prepared for this purpose, Appendix F.

The compressibility factor calculation by Sarem's method is good only when reduced temperature is greater than 1.1. The program has the facility to compute compressibility factors for reduced temperatures less than 1.1 provided the experimental data is available. In the case of the methane-propane system the data has been obtained from Sage and Lacey⁶. The data are stored as a two-dimensional array. Multiple interpolation with the help of subroutine TAB has been used to obtain Z values at any reduced temperature and pressure. At any stage if there is any difficulty in obtaining interpolation a comment is printed out indicating that effect. At the same time based on molecular weight and the reduced temperature and pressure, the viscosity of the mixture is computed by the method of Bicher and Katz.¹⁸ The points on the curves of the viscosity correlation are supplied as part of the data in this program and interpolation techniques are used (see Appendix G).

In order to compute the transmission factor (F_t) the value for the Moody friction factor must be known. The Moody friction factor chart is well satisfied by Colebrook's relationship² as given by Eq. (16) both in the fully turbulent region and in the transition zone. If the value for FTB is set as 1, it indicates fully turbulent region, and if the value is zero it is taken as the transition region. In most of our calculations we are in the transition region. Therefore, the value of FTB has been set as zero. The Moody friction factor requires a trial and error calculation and therefore it has been prepared as an external function and named FNGFRI. This external function computes the value of transmission factor ($= 2/\sqrt{FM}$) directly.

Since the gas gravity factor, temperature flowing factor, supercompressibility factor, transmission factor, drag factor (read from Fig. 13 and fed as part of the data), flow efficiency factor (known by experience) and line diameter factor are known the program is ready to calculate the pressure drop. The compression ratio is computed using the minimum pressure, P2, and the pressure drop. If the difference between the assumed and calculated values of the compression ratio is greater than 1%, the calculated compression ratio is

taken as the assumed value and the calculation repeated until there is agreement within 1%.

Based on this compression ratio, the new maximum pressure in the pipeline, new thickness of the pipe and new internal diameter of the pipe are calculated. A new flow rate is also computed. This will be automatically within 1% of the flow rate for which the entire calculations are based.

Once the compression ratio is fixed the horsepower required is computed with the help of Eq. (22). A new value of compressibility factor based on inlet conditions at the compressor is computed and used in the formula.

At this stage the program is ready to compute all the costs required. The operating cost, ammortization cost, initial line and station investment are computed based on the flow rate existing in the line at that particular section of the line. The fuel consumed at each station is subtracted from the flow rate existing in the line.*

When the calculations are completed for the minimum flow rate and number of stations, the number of stations is incremented by one and the calculations repeated until the transportation cost increases for that flow rate. The minimum cost is stored separately for each flow rate. The flow rate is increased in appropriate increments (set in data) until the minimum cost corresponding to each flow rate starts rising. The absolute minimum cost corresponding to a certain diameter of the line and flow rate is stored separately and is printed out at the end when all the calculations pertaining to that line have been completed.

For each flow rate the program makes a plot of transportation cost (sum of operating cost and ammortization cost) as a function of number of stations. Upon completion of the calculations for each diameter of the line it also makes a plot of the minimum cost of transportation as a function of flow rate. These plots are very handy for a quick survey of the answers obtained.

It may be pointed out that any information fed to the computer in one set of data remains unaltered even when that set of data has been processed and a new set is getting processed unless and until those values get changed in the program or are changed by the new set of data. As an example if the composition of propane is 0.05% for one mixture and is zero for the other mixture, then it will be necessary to make the value of the composition of the propane as 0, otherwise it will be counted as 0.05.

*The initial line and station investment and the transportation cost are based on the average value of gas flowing which amounts to saying that the cost of the fuel should include a transportation cost to the station at which it is used.

Explanation of the Input Data

The computer program was written to be very flexible in processing data for various types of conditions. This flexibility is controlled by various control variables which are defined in the nomenclature to make the understanding and use of the program simpler. A brief description of the input variables are given below with a detailed account of how to use the control variables. Reference may be made to Table B-3 for an example print-out of the input data.

KCP, MWT1

These represent the points taken from a graph of $K(C_p/C_v)$ as a function of molecular weight. For any value of molecular weight of the mixture the subroutine TAB finds the corresponding value of K.

PP, VFM, VFMI, COM1, MP

These are used to calculate the specific volume and then the compressibility factor at any pressure between 200 and 5000 psia and any temperature between 40 to 100°F for methane-propane system and having reduced temperature less than 1.1. PP represents the array for the pressure, VFM the specific volumes at 40°F starting at 0.2 mole fraction methane to 0.9 mole fraction methane. Similarly VFMI represents the specific volumes at 100°F. COM1 specifies the mole fraction methane. MP represents the number of pressures used which in turn become the number of columns in the two-dimensional array of VFM and VFMI.

V, V1, PRD, NP, MT, KS

These are used to calculate the viscosity at any reduced temperature, pressure and molecular weight for the mixtures under study. V represents the points taken from curves starting at reduced temperature of 3 to the ones at 0.65. V1 represents the array for reduced temperatures. PRD gives the array for reduced pressures, and NP for the number of reduced pressures. MT and KS represent the graph of KI (correction factor for viscosity) as a function of molecular weight.

TN2, TCO2, ..., TNC₆H₄, PN2, PCO2, ..., PNC₆H₄, MN2, MC02, ... MNC₆H₄

These are data for the critical temperatures and pressures and the molecular weights for N₂, CO₂, CH₄, C₆H₁₄, iC₄H₁₀, and iC₅H₁₂.

SR1, SR2, ..., SR6, Y

These represent the strength of steels and their corresponding costs in \$/ton. In case only one strength of steel is to be processed set SR2 to SR6 as zero.

BUTI, DBUT, BUTF AND PROPI, DPROP, PROPF

If information is desired for the addition of butane and/or propane to the gas in barrels/day the variables above may be used. These variables represent the quantity of butane or propane added in barrels/day. The initial and final values will be set as zero in case there is no addition as barrels/day. DBUT and DPROP are the incremental additions of butane and propane used in the iteration if a series of calculations are desired. Additions are normally added by specifying the desired values of the gas composition.

NOSF

This represents the maximum value for the number of stations. Depending on what is considered a minimum length between compressor stations, the value of NOSF should be set.

CRI AND DCR

These represent the initial value for the compression ratio. If the maximum allowable compression ratio (CRMAX) has been set as 1.65 CRI may be set as 1.2. DCR allows for changes in CRI during the processing of the data, but normally it should be set as zero.

COMPSW

If we have mixture of known composition and no additional propane or butane (PROPI, BUTI) is added then this should be set as 1. Otherwise it may be set as zero.

QBI, DQB, QBF

These represent the initial, incremental, and final flow rates for the flow rate iteration loop. Judgment should be used in setting up these values. QBF may be set higher than normally expected as the program is going to stop increasing the flow rate once the cost starts increasing.

MMWT

Normally we should set its value greater than the expected molecular weight. In case we have data for the minimum pressure as a function of molecular weight at 40° or 60°F then MMWT may be set as zero.

OD11, OD12, OD13

These represent the outside diameters of the pipe chosen for investigation. In case we went to process only one diameter, OD11 should be set equal to the desired diameter and OD12 and OD13 should be set as zero.

CRMAX

This represents the maximum allowable compression ratio.

NN

This represents the minimum number of stations which will be processed irrespective of whether the minimum cost has occurred or not. This is usually set at 5.

FFF

Minimum number of flow rates which will be processed. If only one flow rate is to be processed set FF equal to 1. Normally FF should be set as 3 since if the second flow rate gives a higher cost than the first, the program will set the flow rate one step less than the first flow rate and process that condition. It will repeat the process until it finds a suitable flow rate. This helps set the initial flow rate when poor judgment is used in selecting it.

FF

This is a drag factor. Its value is read from the graph of drag factor as a function of bend index.

NST1, NST2, DQB2, DQB4, QBMX, QBMX2

In certain mixtures it will be wasteful to process the costs starting from the same number of stations for all the flow rates. To avoid this, the program has the facility to change the initial value for the number of sta-

tions and the flow rate increment. NST1 and DQB2 represent the initial number of stations and the incremental flow rate to be used when the flow rate becomes equal to QBMX. Similarly NST2 and DQB4 represent the initial number of stations and the incremental flow rate when the flow rate becomes equal to QBMX2. QBMX2 should be set higher than QBMX. In case no change is needed then their values should be set higher than the expected final flow rates (QBF1, QBF12, QBF13).

Three external functions available from the Computer Library of The University of Michigan Computing Center were used in the program. They were TAB, SETPLT, and ZERO. A brief description of how to use the external functions is included.

TABLE B-1

PRINT-OUT OF COMPUTER PROGRAM FOR COST OF TRANSPORTING GAS IN CENTS/100 MILES/Mcf

```

$COMPILE MAD, EXECUTE, LUMP, PRINT OBJECT, PUNCH OBJECT      PIPELINE
MAD (01 MAY 1965 VERSION) PROGRAM LISTING ..... .
                                         7/11/724      7/12/65      7/4 9:4 AM

DIMENSION NOSS(40), CYN(40) , NOSTA(40) , *C1
DIMENSION KCP(30), MWTL(30) , QBF1(30) , CY3(30) , CY6(30) , *C2
INTEGER NOS12, NOS13 , NOSTA , NN , FFF , NST1, NST2 , *C3
DIMENSION PP(50), VFM(330, C11), VFM1(360, DIM1), CUM1(25) , *C4
INTEGER CP, MP, MN, MK, NO , *C5
VECTOR VALUES DIM1= 2, 1, 0 , *C6
DIMENSION DI(60), Y(30) , UDI(30) , MW(60), VP(60) , VP1(60) , *C7
INTEGER I, J, II, JJ, III, JJJ, KK, DF, ND, XX, YY , *C8
INTEGER LL, TT, MN, NOS, NUSI, DNSF, UDT , *C9
INTEGER Q, BU, FTB, NUST, COMPSh, ZSW , *C10
INTEGER T, TG, NP, NF , *C11
DIMENSION V1(40), V(40), DIM, MT(20), KS(20), PRD(20) , *C12
VECTOR VALUES DIM= 2,1, 0 , *C13
ZERO.( FN2, FC02, FCH4, FC2H6, FC3H8, FIG4HO, FNC4HU, *C14
1   FIG5H2, FNC5H2, FNC6H4, FX, FY, FZ, TX, TY, TZ, PX, PY, *C14
2   PZ, MX, NY, MZ ) , *C15
ZERO.( FNC7H6 ) , *C15
PRINT COMMENT $1 PIPELINE CALCULATIONS $1
READ AND PRINT DATA *C16
DIM1(2)= MP *C17
DIM(2)= NP *C18
FCH4S=FCH4 *C19
FC2H6S= FC2H6 *C20
FC3H8S= FC3H8 *C21
FIG4HS= FIG4HO *C22
FNC4HS= FNC4HU *C23
KK=C *C24
KKK=0 *C25
KKKK= 0 *C26
INVI= 1.E36 *C27
CY1= 1.E36 *C28
III=C *C29
THROUGH S14, FOR VALUES OF GD= DD11, DC12, DD13 *C30
WHENEVER UD .E. 0., TRANSFER TO S14 , *C31
PRINT COMMENT $0 NEW DIAMETER GF_PIPL$ *C32
PRINT RESULTS-$0 *C33
III=III+L *C34
WHENEVER UD .E. 0D12 *C35
GBI= QBI12 *C36
QBF1= QBF12 *C37
NOSI= NUS12 *C38
END OF CONDITIONAL *C39
WHENEVER UD .E. 0D13 *C40
QBI= QBI13 *C41
QBF1= QBF13 *C42
NOSI= NUS13 *C43
END OF CONDITIONAL *C44
JJ= 0 *C45
THROUGH S1, FOR VALUES OF S= SR1, SR2, SR3, SR4, SR5, SR6 , *C46
ST4 =S *C47
WHENEVER S.E. 0., TRANSFLR TU S1 *C48
PRINT COMMENT $0 NEW STRENGTH OF STEEL$ *C49
PRINT RESULTS S *C50

```

TABLE B-1 (Continued)

```

KK= 0          C2
KKK= 0          02
JJJ= JJJ+1      02
Q= 0           PRINT COMMENT $C
      THROUGH S3, FOR QB= QBI, DCB, QB.G. QBFL1
      NEW FLOW RATE AND COMPOSITION $
QB$= QB
Q= Q+1
WHENEVER QB.E. 1, TRANSFER TO QBFLW
WHENEVER NOSTL. 4
NOSI= 4
TRANSFER TO QBFLW
END OF CONDITIONAL
NOSI= NOST1(Q-1)
QBFL1(Q)= QB
WHENEVER QB.G. QBMX
NOSI= NOST1
DQB= DQB2
END OF CONDITIONAL
WHENEVER QB.G. QBMX2
NOSI= NOST2
DQB= DQB4
END OF CONDITIONAL
BU= 0
THROUGH S3, FOR BUT= BUT1, DBUT, BUT.G. BUTF
BU= BU+1
I=0
THROUGH S3, FOR PROP= PRU1, DPRUD, PROP.G. PRUPP
I= I+1

NEW GAS FLOW RATE AND COMPOSITION OF MIXTURE
WHENEVER COMPSEN .E. 1, TRANSFER TO S11
QB= QBS
FCH4= FCH4S
FC2H6= FC2H6S
FC3H8= FC3H8S
FIGC4HO= FIGC4HS
FNC4HO= FNC4HS
PRPDF= PROP* 5.61*PRPRHO*62.4* 379.* 14.7/(44.C9* 14.73)
IBUTFD= 0.3* BUT* NBUTRH* 379.* 14.7*62.4*5.61/(58.12*14.73)
NBUTFD= 0.7* BUT* NBUTRH* 379.* 14.7*62.4*5.61/(58.12*14.73)
QBF= QB+ PRPDF + IBUTFD* NBUTFD
FCH4= FCH4* QB/ QBF
FC2H6= FC2H6* QB/ QBF
FC3H8=(FC3H8 * QB+ PRPDF) 1/ QBF
FIGC4HO = IBUTFD/QBF
FNC4HO= NBUTFD /QBF
QB= QBF
S11

NATGAS= FX
PRINT RESULTS NATGAS, FC3H3, FIGC4HO, FNC4HO, QBF
PRINT RESULTS FCH4, FC2H6, FNC5H2, FNC6H4, FNC7H6, FN2

CALCULATION OF PSEUDO CRITICAL TEMPERATURE, PRESSURE
AND MOLECULAR WEIGHT
PSUM= FX*PX+ FY*PY + FZ*PZ
TSUM= FX*TX+ FY*TY + FZ*TZ

```

TABLE B-1 (Continued)

```

MSUM= I-X*MX+ FY*MY + FZ*MZ          *1.5
TC= TSUM+ FN2*TN2+ FC02* TCD2+      *1.6
      FC4* TCH4+ FC2H6* TC2H6+
1   FC3H3* T3H8+ FIC4H* TIC4H+      *1.6
2   + FNC5H2* TNC5H2+ FNC6H4*      *1.6
      FNC4H0* TNC4H0+ FIC5H2* TIC5H2
PC= PSUM+ FN2*PN2+ FC02* PC02+      *1.6
      FC4H* PCH4+ FC2H6* PC2H6+
1   FC3H8* PC3H8+ FIC4H0*PIC4H0+      *1.7
2   + FNC5H2* PNC5H2+ FNC6H4*      *1.7
      FNC2H6* PC5H2* PIC5H2
MWT=MSUM+ FN2*MN2+ FC02* MCU2+      *1.7
1   FC3H8*MC3H8+ FIC4H0*MIC4H0+      *1.7
2   +FNC5H2*INC5H2+FNCGH4*MNCGH4*      *1.7
      FNC4H0*FNC4H0+ FIC5H2*MIC5H2
TC= TC+ FNC7H6* TNC7H6
PC= PC+ FNC7H6* PNC7H6
MWT= MWT + FNC7H6* MNCT7H6
G=MWT/29
PRINT RESULTSQB,TC,PC,G,MWT
WHENEVER MWT.L.MMWTF, TRANSFER TO GO
WHENEVER TF.E.500.
END OF CONDITIONAL.
PMIN=TAB.(MMWT,MMWLL),VR(1),1,1,2,MF,SW)
WHENEVER TF.E.520.
PMIN=TAB.(MMWT,MMWLL),VP(1),1,1,2,MF,SW)
END OF CONDITIONAL.
WHENEVER SW.E.2., PRINT COMMENT $0 UNSUCCESSFUL INTERPOLATION
$0
FGR=SQRT.(0.6/G)
FTF=SQRT.(520./TF)
K= TAB.*MWT, MMWT11), KCP(1), 1, 1, 2, MF, SW)
WHENEVER SW.E. 2., PRINT COMMENT $0 INTERPOLATION UNSUCCESSFUL $
J=0
II=0
THROUGH S12,FOR VALUES OF P1= (PMIN+1CG.)*CR
II=II+1
P22= P1/CR
THROUGH S19, FOR NUS= NOSI, DNOS, NUS.G. NOSF
PRINT COMMENT $0 NEW N. U. S.
NO= 0
L= LT/ NOS
P2=P1/CR
CR4= CR
CR3= CR
CR= CRI
CR4= CR
CR3= CR
J=J+1
II=0
THROUGH S12,FOR VALUES OF P1= (PMIN+1CG.)*CR
II=II+1
P22= P1/CR
THROUGH S19, FOR NUS= NOSI, DNOS, NUS.G. NOSF
PRINT COMMENT $0 NEW N. U. S.
NO= 0
L= LT/ NOS
P2=P1/CR
NO= NO+1
THK=P1*OD/(2.*F*EE*S)
D= 0D- 2.*THK
FD= D.P. 2.5
WHENEVER P2.*L. PMIN+90., TRANSFER TO S12
PAVG=2.*(P1+P2-P1*P2/(P1+P2))/3.
TR= TF/TC
PR= PAVG/PC
WHENEVER TR .L. 1.1
THROUGH BETA, FOR C.P=1, 1, CP.G. 7
WHENEVER FCH4 .L. COM1(CP)
VOL1= TAB.(PAVG, PP(1), VFM(CP-1,1 ),1,1,2, MP, SW)
VOL2= TAB.(PAVG, PP(1), VFM(CP ,1 ),1,1,2, MP, SW)
WHENEVER SW .E. 2., PRINT COMMENT $0 INTERPOLATION UNSUCCESSFUL $
TRANSFER TO OUT
END OF CONDITIONAL.
CONTINUE
VOL= VOL1+ (VOL2- VOL1)*(FCCH4 - COM1(CP-1) )/(COM1(CP)
1 -COM1(CP-1))

```

TABLE B-1 (Continued.)

```

01   *158
01   01   C7
01   *159   01   C7
01   *16C   01   C7
01   *161   01   C8
01   *162   02   C8
01   *163   02   C8
01   *164   C2   C8
01   *165   02   C8
01   *166   02   C8
01   *167   01   C8
01   *168   01   C7
01   *169   C1   C7
01   *17C   C1   C7
01   *171   01   C7
01   *172   01   C7
01   *173   01   C7
01   *174   01   C7
01   *175   01   C7
01   *176   01   C7
01   *177   01   C7
01   *178   01   C6
01   *179   01   C6
01   *180   01   C6
01   *181   01   C6
01   *182   01   C6
01   *183   01   C6
01   *184   01   C8
01   *185   01   C8
01   *185   01   C8
01   *186   01   C8
01   *187   C1   C8
01   *188   01   C8
01   *189   01   C8
01   *190   01   C8
01   *191   01   C7
01   *192   01   C7
01   *193   01   C7
01   *194   01   C7
01   *195   01   C7
01   *196   01   C7
01   *197   01   C7
01   *198   01   C7
01   *199   02   C7
01   *200   02   C7
01   *201   02   C7
01   *202   02   C7
01   *203   02   C7
01   *204   02   C7
01   *205   02   C7
01   *206   02   C7
01   *207   02   C7
01   *208   01   C7
01   *209   C1   C7
01   *210   C1   C7
01   *211   *212   C7
01   *213   *214   C7
01   *214   *215   C7

01   VOL= VOL* PAVG/( 10.73* TF)
01   PRINT RESULTS VOL, VOL1, VOL2 ,21
01   THROUGH GAMA, FOR CP=1, 1, CP.G. 7
01   WHENEVER FCH4 .L.
01   VOL3= TAB*(PAVG, PP(1),VFM)(CP-1,1 )1,1,2, MP, SW
01   VOL4= TAB*(PAVG, PP(1),VFM)(CP ,1)1,1,2, MP, SW
01   WHENEVER SW .E. 2., PRINT COMMENT $C INTERPOLATION UNSUCCESSFUL$
01   TRANSFER TO OUT1
01   END CF CONDITIONAL
01   CONTINUE
01   VOL5=VOL3+ (VOL4- VOL3)*(FCH4 - COM1(CP-1) )/(CCM1(CP))
01   -CCM1(CP-1)
01   Z2= VOL5*PAVG/( 10.73* TF)
01   PRINT RESULTS VOL5, VOL3, VOL4, Z2
01   Z= (Z2-Z1)* (TF-500.)/(560.-500.) + Z1
01   PRINT RESULTS Z
01   WHENEVER ZSW .E. 1, TRANSFER TO AHP.
01   TRANSFER TO FPVS
01   END OF CONDITIONAL
01   ZFAC*(TF,PAVG,TC,PC,G,Z)
01   FPV=SQRT*(1./Z)
01   FPVS
01   THROUGH TEES,FORT=1,1,T.G.TG
01   WHENEVER TR.GE. V1(T)
01   T= T-1
01   VISKM1= TAB.(PR,PRD(1),V(T,1),1,1,NP,SW)
01   VISKM2= TAB.(PR,PRD(1),V(T+1,1),1,1,NP,SW)
01   VISKM=(V1(T)-TR)/(V1(T)-V1(T+1))* (VISKM2-VISKM1)
01   KI=TAB.(MWT,MT(1),KS(1),1,1,2, MF, SW)
01   WHENEVER SW.E.2., PRINT COMMENT $4 UNSUCCESSFUL INTERPOLATION
01   $1
01   WHENEVER TR.L. 1., KI= 1.
01   MU= KI*SQRT (MWT)*VISKM*6.7197E-8
01   TRANSFER TO XP
01   END OF CONDITIONAL
01   CONTINUE
01   FTB=0
01   FNFFRI*(FTB,D,EE,QB,G,MU,FT)
01   K6= 77.5*FGF*FTF*FPV*FT*FF*FFE*FD
01   P1= SQRT. (P2*P2+ QB*QB*/L/(K6*K6))
01   CR= P1/P2
01   PRINT RESULTS P1, P2, K6, CR4
01   WHENEVER .ABS. (CR4-CR)/CR .G. .01
01   WHENEVER NO.G. 6 .OR. CR.G. CRMMAX
01   NOS= NOS+1
01   L= LT/NOS
01   WHENEVER L.L. 20., TRANSFER TO S3
01   CR4= CR
01   CR= CRI
01   P2= P22
01   P1= P2*CR
01   TRANSFER TO LLT
01   END OF CONDITIONAL
01   WHENEVER CR-CR4 .G. 3.
01   CR4= CR
01   TRANSFER TO PP2
01   END OF CONDITIONAL
01   THK= P1* OD/ (2.* F*E* S )
01   OD= 'OD- 2.* THK
01   FD= 'C.P. 2.5
01   DELTP= SQRT.((P1*P1- P2*P2)/L)
01   .OR. NO.G. 10, TRANSFER TO S9

```

TABLE B-1 (Continued)

```

QB3= 77.5* FGR* FT* FPV* FT* FF* FFE* FD* DELTP *216
QB= QB3 *217

COSTS
    CRPW= (K-1.)/K *218
    P20= P2 *219
    WHENEVER TR .L. 1.1 *220
    ZSW =1 *221
    PAVG= P2 *222
    TRANSFER TO ZRTE *223
    END OF CONDITIONAL *224
    ZFAC. (TF, P2 , TC, PC, G, Z) *225
    A=(1.0854*TK*TZ*(CR.*CRP*1.)*(CRP*(K-1.)) *14.73/14.65 *226
    ZSW =0 *227
    P2= P20 *228
    PRINT RESULTS CR,A,Z, K, NOS, L, LT, PAVG *229
    COSTS
    YWT *230
    YW=2.8*(D+THK)*THK*Y(JJJ)
    CMMOP=0. *231
    CMMAM=0. *232
    IINVL= 0. *233
    IINVS= 0. *234
    THROUGH S7, FOR NUST=1,1,NOST.G.NOS *235
    CMMOP1=((365.*24.*FHPHR*CF+CLMS)*A*FOOP/(L*365. ) *236
    1 +(LG*GASST*1E3/LT+CLML*1.E6/(QB*365. ) *236
    1 )*10.+AD3650. *236
    CMMAM1=((YW+N*0D+H)*B*1.E6/QB+ALPHA*(X+X1*1.E6/(A* *237
    1 QB))*A/L)*FOOP/365. )*10. *237
    1 IINVL=(YW+N*0D+H)/(QB/1.E7) *238
    1 IINVS1=(X+X1*1.E6/(A*QB))*10.*A/L *238
    1 LSG=(24.*FHPHR*A*QB*1.E-3+(LG*QB)/NOS) *239
    QB=QB-LSG *240
    CMMOP=CMMOP+CMMOP 1 *241
    CMMAM=CMMAM+CMMAM 1 *242
    IINVS= IINVS+ IINVS1 *243
    IINVL= IINVL+ IINVL1 *244
    CONTINUE *245
    IINVL= IINVL/NOS *246
    IINV= IINVL+ IINVS *247
    CMMOP=CMMOP/NOS *248
    CY=CMMOP+CMMAM *249
    IINVS= IINVS/NOS *250
    TINV=IINV *QB3*1.E-7*L *251
    TOPCS=CY*QB3*1.E-7*L *252
    PRINT RESULTS P1, PAVG, CR, S, OD, D, QB, QB3 *253
    PRINT RESULTS IINVL,IINVS,CMMOP,CMMAM *254
    PRINT RESULTS IINV,CY,THK,YW *255
    KK=KK+1 *256
    NOS(KK)= NOS *257
    CYN(KK)= CY *258
    WHENEVER KK.E.1 *259
    CY1=CY *260
    KK=KK+1 *261
    NOS(KK)= NOS *262
    CYN(KK)= CY *263
    WHENEVER KK.E.1 *264
    CY1=CY *264

```

TABLE B-1 (Continued)

```

NOSTA(Q)= NOS
TRANSFER TO INN
END OF CONDITIONAL
WHENEVER CY1.GE.CY
  INV1= INV
CY1=CY
CY3(Q)= CY1
PROP2=PROP
CR2=CR
P12=P1
NOS2=NOS
BUT2=BUT
D04=CD
D2=C
ST2=S
QB4= QB
QBF4= QB$1
TINV2 = TINV
TPCST2= TOPCST
WHENEVER NOS.E. NOSF, TRANSFER TO M0S
  WHENEVER KK.E. 1, TRANSFER TO S1C
  OTHERWISE
    WHENEVER KK.L. NN, TRANSFER TO S1C
    PRINT COMMENT $0 MINIMUM COST AT THE END OF N. C. $.
    PRINT RESULTS QBF4, FIC4HO, FIC4HU
    PRINT RESULTS NATGAS, FC3H8, CR2, P12, NOS2, ST2, CD4, D2
    PRINT RESULTS FCH4, FC2H6, FNC5H2, FNC6H4, FNC7H6, FN2
    PRINT RESULTS TINV1, CY1, TPCST2, TINV2
    PRINT COMMENT $1 PLOT OF TRANSPORTATION CCST VS NOS$
    PRINT RESULTS UD, S, P2, QBF4
    SETPLT(L, NOS(1), CYN(1), KK, $$, 33, LABEL)
    PRINT COMMENT $0 NUMBER OF STATIONS$
KKK= KKK+1
WHENEVER KKK .E. 1
  TINV6= INV1
  CY6= CY1
  CY61(Q)= CY3(Q)
  PROP6= PROP2
  CR6= CR2
  P16= P12
  NOS6= NOS2
  BUT6= BUT2
  D06= D04
  D6= D2
  ST6= ST2
  QB6= QB4
  QBF6= QB$3
  TINV6= TINV2
  TPCST6= TPCST2
KK= 0
TRANSFER TO S12
END OF CONDITIONAL
WHENEVER CY6 .GE. CY1
  TRANSFER TO S9
  OTHERWISE
    KK= 0
    CY61(Q)= CY3(Q)
    TRANSFER TO S9
    END OF CONDITIONAL
    END OF CONDITIONAL
*265   01  07
*266   C1  07
*267   C1  07
*268   C1  07
*269   01  07
*270   C1  07
*271   C1  07
*272   C1  07
*273   C1  07
*274   C1  07
*275   C1  07
*276   C1  07
*277   C1  07
*278   C1  07
*279   C1  07
*280   C1  07
*281   C1  07
*282   C1  07
*283   C1  07
*284   C1  07
*285   C1  07
*286   C1  07
*287   C1  07
*288   01  07
*289   C1  07
*290   C1  07
*291   C1  07
*292   01  07
*293   C1  07
*294   C1  07
*295   C1  07
*296   C1  07
*297   01  07
*298   01  07
*299   02  07
*300   C2  07
*301   C2  07
*302   C2  07
*303   02  07
*304   C2  07
*305   02  07
*306   02  07
*307   02  07
*308   02  07
*309   C2  07
*310   02  07
*311   02  07
*312   02  07
*313   02  07
*314   C2  07
*315   02  07
*316   C2  07
*317   C1  07
*318   02  07
*319   02  07
*320   C2  07
*321   02  07
*322   C2  07
*323   C2  07
*324   C1  07

```

TABLE B-1 (Continued)

```

S10      QB= QBFL(Q)          07
        QB = QBFL(Q)          06
        KKK= KKK+1            05
        KK= 0                  05
        QB= QBFL(Q)          05
        WHENEVER KKKK.E. 1     05
        CY8= CY6               05
        TRANSFER TO S3         01
        END OF CONDITIONAL    01
        WHENEVER CY6.G. CY8   05
        TRANSFER TO S9         01
        OTHERWISE              05
        CY8= CY6               01
        END OF CONDITIONAL    01
        CONTINUE               05
S3       CONTINUE             02
        WHENEVER Q.L. FFF      02
        QBI= QBI-DQB           01
        TRANSFER TO Q01         01
        END OF CONDITIONAL    01
        PRINT COMMENT $6 MINIMUM INITIAL INVESTMENT(CENTS/100 MILES/M
1       INVESTMENT(DOLLARS) $          02
        PRINT COMMENT $0 TOTAL OPERATING COST(DOLLARS/DAY) AND TOTAL
1       CF/DAY) AND TRANSPORTATION COST(CENTS/100 MILES/MCF) $          02
        PRINT RESULTS NATGAS, FC3H8, CR6, P16, NOS6, ST6, QD6, D6      02
        PRINT RESULTS QB6, QBF6, *FNC4H0, FIC4H0, *FNC4H1, FNC4H2      02
        PRINT RESULTS FCH4, FC2H6, FNC5H2, FNC6H4, FNC7H6, FN2          02
        PRINT RESULTS TINV6, CY6, TPCST6, TINV6                         02
        WHENEVER Q.L. 3, TRANSFER TO S2                                     02
        PRINT COMMENT $1 PLOT OF TRANSPORTATION COST VS FLGM FLOW RA
1       TE                   02
        PRINT RESULTS QD, S, P2                                         02
        SEPLT(1, QBFL(1), CY6(1),Q, $$, 33, LABEL)                      02
        PRINT COMMENT $0 FLOW RATE, CUBIC FEET/DAYS$                     02
CONTINUE
S2       ZERO.(QB6, QBF6, CR6, P16, NOS6, ST6, QD6, D6, IINV6,          02
        KK= 0                  02
        KKKK= 0                02
        CR= CR3+ DCR           02
        CONTINUE               02
        S=ST4                 01
        CONTINUE               01
S14      VECTOR VALUES LABEL = $ TR. COST(CENTS/100 MILES/MCF) $          01
        TRANSFER TO START          01
        END OF PROGRAM           01

```

TABLE B-1 (Continued)

TAB

1/20/62

SINGLE TABLE INTERPOLATION

PURPOSE GIVEN THE VALUE OF AN INDEPENDENT ARGUMENT X, PERFORM A KTH ORDER INTERPOLATION ON A TABLE OF (X(I),Y(I)) VALUES FOR THE CORRESPONDING DEPENDENT ARGUMENT Y.

CALLING SEQUENCES

MAD Y = TAB.(X,XT,YT,M1,M2,K,N,SW)
FORTRAN Y = TAB(X,XT,YT,M1,M2,K,N,SW)
UMAP CALL TAB
 PAR X
 PAR XT
 PAR YT
 PAR M1
 PAR M2
 PAR K
 PAR N
 PAR SW
 NORMAL RETURN - Y IN THE ACCUMULATOR

ARGUMENTS

X INDEPENDENT FLOATING POINT ARGUMENT X FOR WHICH THE CORRESPONDING VALUE Y IS DESIRED.
XT NAME OF THE FIRST ENTRY IN THE TABLE OF FLOATING POINT INDEPENDENT VARIABLES, X(I).
YT NAME OF THE FIRST ENTRY IN THE TABLE OF FLOATING POINT DEPENDENT VARIABLES, Y(I).
M1 INTEGRAL NUMBER OF STORAGE LOCATION STEPS BETWEEN EACH ENTRY OF THE INDEPENDENT VARIABLE TABLE. NORMALLY M1 = 1 WHEN THE VARIABLES ARE STORED IN SEQUENTIAL LOCATIONS.
M2 INTEGRAL NUMBER OF LOCATIONS BETWEEN EACH ENTRY OF THE DEPENDENT VARIABLE TABLE. NORMALLY M2 = 1.
K INTEGRAL ORDER OF INTERPOLATION DESIRED, K .LE. 5.
N INTEGRAL NUMBER OF ENTRIES IN THE INDEPENDENT VARIABLE TABLE (NUMBER OF PAIRS (X(I),Y(I))).
SW FLOATING POINT COMPUTATION SWITCH
 SW = 1.0 NORMAL RETURN, INTERPOLATION SUCCESSFUL.
 SW = 2.0 AC OR MQ OVERFLOW OR UNDERFLOW OR DIVIDE CHECK -- ERROR RETURN.
Y FLOATING POINT DEPENDENT VARIABLE, THE INTERPOLATED VALUE FOR THE INDEPENDENT VARIABLE X.

CODING INFORMATION

STORAGE REQUIRED

TAB 307
ERASABLE 22

TABLE B-1 (Continued)

SETPLT
USTPLT

1/20/62

SET UP FOR PLOT SUBROUTINE
(IT MAKES PLOT PAINLESS)

PURPOSE -- THIS SUBROUTINE IS DESIGNED TO BE USED WITH THE PLOT SUBROUTINE (WHICH IS ON LIBRARY TAPE). THE PLOT SUBROUTINE PRODUCES GRAPHS OF THE QUANTITIES GIVEN IT BY THE USER. (FOR A DETAILED EXPLANATION, SEE THE PLOT WRITEUP) IT IS A POWERFUL AND VERSATILE TOOL, BUT IS, AS A RESULT, RATHER COMPLICATED AND CLUMSY TO USE. IT REQUIRES THAT THE USER MAKE 4 ENTRIES TO THE SUBROUTINE WITH A TOTAL OF 16 ARGUMENTS, AND IN ORDER TO DETERMINE THE VALUES FOR THESE ARGUMENTS (SUCH AS THE NUMBER OF HORIZONTAL LINES, NUMBER OF SPACES BETWEEN HORIZONTAL LINES, ETC.) THE USER MUST DO CONSIDERABLE PRECALCULATION. THE USER MUST ALSO KNOW THE RANGE OF ANSWERS IN ADVANCE SO HE CAN SET THE MAXIMUM AND MINIMUM VALUES FOR THE ABSCISSA AND FOR THE ORDINATE. THIS IS ALL WORK THAT CAN BE DONE BY THE COMPUTER, AND SETPLT IS A SUBROUTINE THAT DOES IT.

FEATURES -- SETPLT INSPECTS THE DATA TO BE PLOTTED, CALCULATES THE ARGUMENTS, AND THEN EXECUTES PLOT SUCH THAT --

1. ALL POINTS TO BE PLOTTED LIE IN THE RANGE OF THE GRAPH.
2. GRIDWORK IS SQUARE.
3. NUMERIC LABELS ON ABSCISSA AND ORDINATE GRID LINES ARE "NICE" VALUES.
4. GRAPH IS APPROXIMATELY SQUARE.
5. IF THE POINTS TO BE PLOTTED HAVE ABSCISSA AND/OR ORDINATE VALUES WHOSE MAGNITUDE IS GREATER THAN 10.P.7, THE NUMERIC LABELS TO THE GRID LINES ARE MODIFIED BY A SCALE FACTOR, AND A HEADING IS PRINTED OUT INFORMING THE USER OF THE SIZE OF THE SCALE FACTOR.
6. IF THE SIZE OF THE GRAPH IS INDETERMINATE IN EITHER THE Y(VERTICAL) AND/OR THE X(HORIZONTAL) DIRECTION (IE. A HORIZONTAL OR VERTICAL LINE, OR A POINT), AN APPROPRIATE COMMENT IS PRINTED OUT AND THE MAXIMUM AND MINIMUM VALUES OF THE APPROPRIATE AXES ARE ADJUSTED SO THAT THE VALUES MAY BE GRAPHED.

RESTRICTIONS-- ALL POINTS (X,Y) WHICH ARE TO BE PLOTTED MUST BE OBTAINED AND STORED IN TABLES BEFORE EXECUTING SETPLT.

CALLING SEQUENCES -- THERE ARE TWO CALLING SEQUENCES AVAILABLE, A REGULAR AND AN ALTERNATE ONE.

REGULAR CALLING SEQUENCE -- USER EXECUTES ONLY SETPLT (OR USTPLT). USER DOES NOT EXECUTE PLOT.

```
MAD-      EXECUTE SETPLT.(L,XLOC,YLOC,NUM,BCD,NCHAR,LABEL)
FORTRAN- CALL SETPLT(L,XLOC,YLOC,NUM,BCD,NCHAR,NHABCD...)
UMAP-    CALL      USTPLT
          PAR      L
          .
          .
          .
          PAR      LABEL
```

OR ANY EQUIVALENT UMAP SUBROUTINE CALL

TABLE B-1 (Continued)

ALTERNATE CALLING SEQUENCE -- THIS IS FOR USERS WHO WANT TO USE "OMIT" TO CHANGE THE GRAPH BEFORE IT IS PRINTED, WHO WANT TO PRINT MORE THAN ONE COPY OF THE GRAPH, WHO WANT TO USE DIFFERENT PLOTTING CHARACTERS FOR DIFFERENT PARTS OF THE DATA, OR, IN GENERAL, WHO WANT TO TAKE ADVANTAGE OF SOME OF THE SPECIAL FEATURES OF PLOT (FOR DETAILS ON THESE SPECIAL FEATURES, SEE THE PLOT WRITEUP) WHEN USING THIS ALTERNATE CALLING SEQUENCE, USER EXECUTES SETPLT, AND THEN MUST EXECUTE PLOT3 AND PLOT4 (OR FPLOT4) HIMSELF.

```
MAD- EXECUTE SETPLT.(L,XLOC,YLOC,NUM)
FORTRAN- CALL SETPLT(L,XLOC,YLOC,NUM)
UMAP- (FOR THIS ALTERNATE CALLING SEQUENCE EITHER THE NAME SETPLT
      OR USTPLT MAY BE USED)
      CALL USTPLT,L,XLOC,YLOC,NUM
      OR EQUIVALENT SUBROUTINE CALL
```

ARGUMENTS --

L	=NONZERO IF MAX GRAPH LENGTH IS TO BE ONE PAGE =ZERO OTHERWISE. (IN THIS CASE , LENGTH .LE. 2 PAGES)
XLOC	=LOCATION OF FIRST VALUE OF X OR POINTS (X,Y) TO BE PLOTTED (IN TABLE OF X VALUES)
YLOC	=LOCATION OF FIRST VALUE OF Y OF POINTS (X,Y) TO BE PLOTTED (IN TABLE OF Y VALUES) (THESE TWO TABLES MUST BE STORED BACKWARDS IN STORAGE, AS MAD AND FORTRAN DO. THE VALUES OF X AND Y STORED IN THESE MUST BE FLOATING- POINT VALUES)
NUM	=NUMBER OF POINTS TO BE PLOTTED (EITHER MAD, UMAP, OR FORTRAN INTEGER)
BCD	=LEFT-ADJUSTED BCD(HOLLERITH) PLOTTING CHARACTER.
NCHAR	=NUMBER OF BCD CHARACTERS (INCLUDING BLANKS) IN THE LABEL ARRAY.
LABEL	=NAME OF ARRAY CONTAINING THE STRING OF BCD CHARACTERS TO BE PRINTED AT LEFT EDGE OF OUTPUT PAGE (LABEL FOR ORDINATE). MUST BE STORED BACKWARD WHEN USING MAD (USING VECTOR VALUES STATEMENT), OR FORWARD WHEN USING UMAP (USING BCD OR BCI BLOCK). MUST BE STORED 6 CHARACTERS TO THE WORD (C6).
NHABCD...	FOR FORTRAN USERS, THE STRING OF CHARACTERS FOR THE ORDINATE LABEL APPEARS DIRECTLY IN THE CALLING SEQUENCE. THE N PRECEDING THE H (SPECIFYING THE HOLLERITH STRING) SHOULD BE THE SAME AS THE VALUE OF NCHAR.

EXAMPLES-- SEE NEXT PAGE**CODING INFORMATION--**

STORAGE USED	SUBROUTINES USED
SETPLT 701	.PRINT
ERASABLE 1750,IF L=NONZERO	PLOT1
4064,IF L=ZERO	PLOT2
(FIRST 25 LOCATIONS OF ERASABLE ARE NOT USED)	PLOT3 PLOT4 FPLOT4 ELOG
STORAGE REQUIRED	.01301
SETPLT 701	
ERASABLE 4064	

TABLE B-1 (Continued)

SAMPLE PROBLEM

THIS PROBLEM IS THE FIRST EXAMPLE PROBLEM AT THE END OF THE PLOT WRITEUP, REWRITTEN TO USE SETPLT. IT IS SUGGESTED THAT THE READER COMPARE THEM. BOTH MAD AND FORTRAN VERSIONS ARE GIVEN.

\$COMPILE MAD, PUNCH OBJECT	PLMAD000
R	
R PROGRAM TO ILLUSTRATE PLOTTING MULTIPLE POINTS WITH MAD	
R	
FIRST	DIMENSION X(100), Y(100)
	INTEGER N
	READ FORMAT ENTR, N
	READ FORMAT DATA, X(1)...X(N)
	READ FORMAT DATA, Y(1)...Y(N)
	PRINT FORMAT TITLE
	EXECUTE SETPLT,(1,X(1),Y(1),N,\$\$\$,32,ORD)
	PRINT FORMAT ABS
	TRANSFER TO FIRST
R	
R	FORMAT STATEMENTS
R	
	VECTOR VALUES ENTR = \$I10+\$
	VECTOR VALUES DATA = \$7F10.4+\$
	VECTOR VALUES TITLE = \$1H1,S54,15H PLOT OF X VS Y /1H +\$
	VECTOR VALUES ABS = \$1H0,S55,14H THE ABSCISSA X +\$
	VECTOR VALUES ORD = \$ THE ORDINATE Y \$
	END OF PROGRAM
\$DATA	

\$COMPILE FORTRAN, PRINT OBJECT, PUNCH OBJECT	PLFTRO00
C	
C	PROGRAM TO ILLUSTRATE PLOTTING MULTIPLE POINTS WITH FORTRAN
C	
1	DIMENSION X(100),Y(100)
1	READ INPUT TAPE 7,100, N
1	READ INPUT TAPE 7,101,(X(I), I=1,N)
1	READ INPUT TAPE 7,101,(Y(I), I=1,N)
1	WRITE OUTPUT TAPE 6,102
1	CALL SETPLT(1,X(1),Y(1),N,1H*,32,32H
X Y)	THE ORDINATE
1	WRITE OUTPUT TAPE 6,103
1	GO TO 1
C	
C	FORMAT STATEMENTS
C	
100	FORMAT (I10)
101	FORMAT (7F10.9)
102	FORMAT (1H1,54(1H),15H PLOT OF X VS Y /1H)
103	FORMAT (1H0,55(1H),14H THE ABSCISSA X)
\$ DATA	

TABLE B-1 (Concluded)

MOD NR 16
10-21-64 p 1 of 6
UMES WRITEUPS
SEPTEMBER 1964

ZERO - SPRAY

STORE CONSTANT

PURPOSE ZERO STORES ZERO
 SPRAY STORES ARBITRARY CONSTANT.

CALLING SEQUENCES

MAD EXECUTE ZERO.(L1,L2,....,LN)
UMAP CALL ZERO
 L1
 L2
 •
 •
 •
 LN

ARGUMENTS

THE LI ARE STANDARD ARGUMENT LIST ELEMENTS OF THE FORM
MAD A,,D OR A
UMAP BLK A,,D OR PAR A
SPRAY IS CALLED EXACTLY AS ZERO EXCEPT THAT THE FIRST ARGUMENT
IS A SINGLE CONSTANT (IN MAD) OR THE LOCATION OF A SINGLE CONSTANT
(IN UMAP) WHICH IS TO BE STORED INSTEAD OF ZERO.

PROGRAMMING INFORMATION

LOCATIONS REQUIRED
ZERO - SPRAY 34
ERASABLE 0

TABLE B-2

NOMENCLATURE FOR COMPUTER PROGRAM AND FORMULATIONS

Symbol	Designation	Units
AD	Administration expenses	\$/mile/year/MMcf/day
ALPHA (α)	Fixed charges on compression, fraction	none
A	Brake horsepower	hp/MMcf/day at 60°F and 14.73 psia
B	Fixed charges on pipe, fraction of investment	none
BU	Counter for butane addition	none
BUTI2	Butane condensate at minimum cost for each flow rate	bbl/day
BUTI6	Butane condensate at minimum cost for the line	bbl/day
BUTF	Final value of butane condensate	bbl/day
BUTI	Initial value of butane condensate	bbl/day
BUT	Butane condensate	bbl/day
CF	Fuel Cost	\$/Mcf at 60°F and 14.73 psia
86		\$/mile/yr
CLML	Cost of labor and maintenance for line	\$/hp/yr
CLMS	Cost of labor and maintenance for stations	cents/100 miles/Mcf
CMMAM	Average amortization costs of pipeline	cents/100 miles/Mcf
CMMAM1	Amortization cost at each station	cents/100 miles/Mcf
CMMOP	Average operating costs of pipeline	cents/100 miles/Mcf
CMMOP1	Operating cost at each station	cents/100 miles/Mcf
COM1	Fraction methane (used when the TR is less than 1.1 for methane-propane system)	none
COMPSTW	Switch for either having program determine the composition of the mixture ("off" i.e. = 0) or when the composition is already specified ("on" i.e. = 1)	none
CP	Used as counter for COM1	none
CR	Compression ratio, P_1/P_2	none
CR2	Compression ratio corresponding to minimum yearly transportation cost for certain flow rate	none
CR3	Storage for initial value of compression ratio	none

TABLE B-2 (Continued)

Symbol	Designation	Units
CR4	Storage for initial value of compression ratio	none
CR6	Compression ratio corresponding to minimum yearly transportation cost for certain diameter of pipe	none
CRI	Initial value for compression ratio	none
CRMAX	Maximum value for compression ratio	none
CRPW	(K-1)/K	none
CY	Total transportation cost	cents/100 miles/Mcf
CY1	Minimum yearly transportation cost for certain flow rate	cents/100 miles/Mcf
CY3	Minimum yearly transportation cost for certain flow rate	cents/100 miles/Mcf
CY61	Minimum yearly transportation cost for certain diameter of pipe	cents/100 miles/Mcf
CY6	Minimum yearly transportation cost for certain diameter of pipe	cents/100 miles/Mcf
CY8	Storage for CY6	cents/100 miles/Mcf
CYN	Total transportation cost—used for plotting CY vs. NOS	cents/100 miles/Mcf
D2	Inside diameter of the pipe corresponding to minimum transportation cost for certain flow rate	in.
D6	Inside diameter of the pipe corresponding to minimum transportation cost for certain diameter of the pipe	in.
DBUT	Increment to butane condensate	bbl/day
DCR	Increment in compression ratio	none
DELTIP	$\sqrt{P_1^2 - P_2^2/L}$	--
D	Inside diameter of pipe	in.
DNOS	Increment to number of stations	none
DPROP	Increment to propane addition	bbl/day
DQB2	Increment to flow rate when it exceeds QBMX	ft^3/day
DQB4	Increment to flow rate when it exceeds QBMX2	ft^3/day
DQB	Increment to flow rate	ft^3/day
EE	Pipe roughness	in.
EFF	Efficiency of compressor stations	none
E	Longitudinal joint factor	none
FC2H6	Mole fraction C ₂ H ₆	none
FC2H6S	Storage for FC ₂ H ₆	none

TABLE B-2 (Continued)

Symbol	Designation	Units
FC3H8	Mole fraction C ₃ H ₈	none
FC3H8S	Storage for FC ₃ H ₈	none
FCH ₄	Mole fraction CH ₄	none
FCH ₄ S	Storage for FCH ₄	none
FCO ₂	Mole fraction CO ₂	none
FD	Line diameter factor	in. ² •5
FFE	Flow efficiency factor	none
FF	Drag factor	none
FFF	Number for processing minimum number of flow rates for one diameter of pipe. Ordinarily FFF = 1	none
FGR	Gas gravity factor	none
FHPHR	Fuel use	Mcf/hp hr
FIC4HO	Mole fraction IC ₄ H ₁₀	none
FIC4HS	Storage for FIC ₄ H ₄	none
FIC5H ₂	Mole fraction IC ₅ H ₁₂	none
FM	Moody friction factor	none
FM2	Mole fraction N ₂	none
FNC4HO	Mole fraction nC ₄ H ₁₀	none
FNC4HS	Storage for FNCH ₄	none
FNC5H ₂	Mole fraction nC ₅ H ₁₂	none
FNC6H ₄	Mole fraction nC ₆ H ₁₄	none
FNC7H ₆	Fraction normal heptane	none
FOOP	Fraction of year in operation	none
FPV	Supercompressibility factor	none
F	Pipeline design factor (average = 0.72)	none
FTB	Switch for fully turbulent and (1) and transition zone (0). Set FTB = 1 when fully turbulent and FTB = 0 when otherwise	none
FTF	Flowing temperature factor	none
FT	Transmission factor	none
FX	Mole fraction unknown component X, e.g., natural gas	none
FY	Mole fraction unknown component Y, e.g., natural gas	none

TABLE B-2 (Continued.)

Symbol	Designation	Units
FZ	Mole fraction unknown component Z, e.g., natural gas	none
GASCST	Cost of gas	\$/Mcf at 14.73 psia and 60°F
G	Gas gravity	none
H	Communication cost	\$/mile
I BUTTFD	Isobutane condensate (gas added)	ft ³ /day
I BUTTRH	Density of isobutane (liquid)	g/cc
III	Counter for outside diameter	none
I	Counter for compositions	none
IINV1	Minimum initial investment at each station	cents/100 miles/Mcf/day
IINV6	Minimum average initial investment of pipeline corresponding to certain diameter of line	cents/100 miles/Mcf/day
89	Average initial investment for the pipeline	cents/100 miles/Mcf/day
IINVLL	Initial investment for the pipeline	cents/100 miles/Mcf/day
IINVVS	Initial investment for stations	cents/100 miles/Mcf/day
IINVSL	Initial investment for stations	cents/100 miles/Mcf/day
IINV	Total initial investment for line and stations	cents/100 miles/Mcf/day
II	Counter for various maximum pressures in the pipe (P_1)	none
JJJ	Counter for stresses in pipe	none
J	Counter for compression ratio	none
K6	77.5 x FGR x FTF x FPV x FT x FF x FFE x FD	none
K	Ratio of heat capacity at constant pressure to constant volume	none
KCP	Used for correlation of ratio of specific heats (C_p/C_v) as a function of molecular weight	none
KI	Correction factor to viscosity as a function of molecular weight	none
KKKK	Counter used for keeping track of flow rate and the minimum transportation cost	none
KKK	Counter used for keeping track of flow rate	none
KK	Counter for keeping track of NOS	none
KS	Used as KI for correlation of KI as a function of molecular weight	none
LG	Loss of gas in pipeline, fraction	none

TABLE B-2 (Continued)

Symbol	Designation	Units
L	Length between compression stations	miles
LT	Total length of pipeline	miles
MC2H6	Molecular weight C ₂ H ₆	lb/lb mole
MC3H8	Molecular weight C ₃ H ₈	lb/lb mole
MCH ₄	Molecular weight CH ₄	lb/lb mole
MCO ₂	Molecular weight CO ₂	lb/lb mole
MF	Number of points used in correlation of KI as a function of molecular weight	none
MIC ₄ HO	Molecular weight iC ₄ H ₁₀	lb/lb mole
MIC ₅ H ₁₂	Molecular weight iC ₅ H ₁₂	lb/lb mole
MMWT	For molecular weight comparison. If PMIN is known set MMWT greater than predicted molecular weight of the mixture, e.g., say 40. If PMIN is to be determined at 40° or 60°F from correlation of PMIN as a function of molecular weight, set PMIN = 0	lb/lb mole
MN2	Molecular weight N ₂	lb/lb mole
MNC ₄ HO	Molecular weight NC ₄ H ₁₀	lb/lb mole
MNC ₅ H ₁₂	Molecular weight NC ₅ H ₁₂	lb/lb mole
MNC ₆ H ₁₄	Molecular weight NC ₆ H ₁₄	lb/lb mole
MNC ₇ H ₁₆	Molecular weight of normal heptane	lb/lb mole
MP	Number of points used in correlation of volume as a function of pressure for determining Z	none
MSUM	Sum of molecular weight for X,Y,Z	--
MT	Used as molecular weight array in correlation of KI as a function of molecular weight	lb/lb mole lb/ft sec
MU	Viscosity of gas	lb/lb mole
MW	Used as molecular weight array in correlation of equilibrium pressure as a function of molecular weight	lb/lb mole
MWT	Molecular weight of mixture	lb/lb mole
MWT1	Used as molecular weight array in correlation of K(C _P /C _V) as a function of molecular weight	lb/lb mole

TABLE B-2 (Continued)

Symbol	Designation	Units
MX	Molecular weight of X	lb/lb mole
MY	Molecular weight of Y	lb/lb mole
MZ	Molecular weight of Z	lb/lb mole
NATGAS	Fraction of natural gas	none
NBUTFD	Normal butane condensate (gas added)	ft ³ /day
NBUTRH	Density of normal butane (liquid)	g/cc
NN	The minimum number for which the number of stations will be processed irrespective of where the minimum cost lies. For example see case of natural gas at 65,000 psi and 16 in. pipeline	none
NO	Counter for number of trials involved in checking of compression ratio	none
NOS12	Number of stations to start computation when OD = OD12	none
NOS13	Number of stations to start computation when OD = OD13	none
NOS	Number of stations	none
NOSI	Initial value for number of stations	none
NOSF	Final value for number of stations	none
NOS2	Number of stations for minimum yearly transportation cost for certain flow rate	none
NOS6	Number of stations corresponding to minimum transportation cost for certain diameter of the line	none
NOSS	Array (NOS) used for plot of cost as a function of number of stations	none
NOSTA	Storage for starting number of stations	none
NOST	Used for iteration of number of stations	none
N	Pipe laying cost as function of diameter	\$/in. diam/mile
NP	Number of points used for correlation of viscosity as a function of reduced pressure and temperature	none
NST1	Number of stations when flow rate is greater than QBMX	none
NST2	Number of stations when flow rate is greater than QBMX2	none
OD	Outside diameter of pipe	in.
OD11	First value in the iteration loop of the outside diameter of the pipe	in.

TABLE B-2 (Continued)

Symbol	Designation	Units
OD12	Second value in the iteration loop of the outside diameter of the pipe	in.
OD13	Third value in the iteration loop of the outside diameter of the pipe	in.
OD4	Outside diameter of the pipe giving minimum transportation cost for certain flow rate	in.
OD6	Outside diameter of the pipe giving minimum transportation cost for certain diameter of the line	in.
P12	Maximum line pressure corresponding to minimum yearly transportation cost at certain flow rate	psia
P16	Maximum pressure in the pipeline corresponding to minimum transportation cost for certain diameter of the line	psia
P20	Pressure at end of line section (minimum) — minimum pressure in the pipeline	psia
92	Pressure at end of line section (minimum)	psia
P1	Maximum line pressure	psia
P2	Pressure at end of line section (minimum)	psia
P AVG	Average flowing pressure	psia
PC2H6	Critical pressure for C ₂ H ₆	psia
PC3H8	Critical pressure for C ₃ H ₈	psia
PCH ₄	Critical pressure for CH ₄	psia
PCO ₂	Critical pressure for CO ₂	psia
PIC ₄ HO	Critical pressure for 1C ₄ H ₁₀	psia
PMIN	Maximum point on bubble or dew point curve for a mixture in the temperature range of interest	psia
PNC ₄ HO	Critical pressure for nC ₄ H ₁₀	psia
PIC ₅ H ₂	Critical pressure for 1C ₅ H ₁₂	psia
PNC ₅ H ₂	Critical pressure for nC ₅ H ₁₂	psia
PNC ₆ H ₄	Critical pressure for 1C ₆ H ₁₄	psia
PNC ₇ H ₆	Critical pressure for normal heptane	psia
PN2	Critical pressure for N ₂	psia
PP	Array corresponding to pressure in correlation of volume as a function of pressure	psia

TABLE B-2 (Continued)

Symbol	Designation	Units
PRD	Array corresponding to reduced pressure in correlation of viscosity as a function of reduced pressure	none
PC	Critical pressure of mixture	psia
PROP	Quantity of propane added	bbl/day
PROPI	Initial value to quantity of propane	bbl/day
PROPF	Final value to quantity of propane added	bbl/day
PROP2	Quantity of added propane giving minimum yearly transportation cost	bbl/day
PROP6	Quantity of propane added corresponding to the minimum transportation cost	bbl/day
PRPFD	Quantity of propane added (gas)	ft ³ /day
PRPRHO	Density of propane (liquid)	g/cc
PR	Reduced pressure	none
PSUM	Sum of critical pressures corresponding to constituents X,Y,Z	psia
PX	Critical pressure for X	psia
PY	Critical pressure for Y	psia
PZ	Critical pressure for Z	psia
Q	Counter for variation of flow rate	none
QB	Gas flow rate for natural gas at the entrance	ft ³ /day at 14.73 and 60°F
QB12	Starting flow rate for OD12 (second diameter)	ft ³ /day
QB13	Starting flow rate for OD13 (third diameter)	ft ³ /day
QB3	The flow rate for which the cost has been computed	ft ³ /day
QB4	Minimum cost corresponding to certain flow rate	ft ³ /day
QB6	Minimum cost corresponding to certain diameter of the line	ft ³ /day
QBFR	Total gas flow rate after propane or butane addition. Whenever there is no addition QB = QBF	ft ³ /day at 14.73 and 60°F
QBFL2	Final flow rate corresponding to OD12 (second diameter of the pipe)	ft ³ /day
QBFL3	Final flow rate corresponding to OD13 (third flow rate)	ft ³ /day
QBFL1	Final flow rate	ft ³ /day
QBFL6	Flow rate corresponding to the minimum transportation cost	ft ³ /day

TABLE B-2 (Continued)

Symbol	Designation	Units
QBFL	Array used for plotting minimum transportation cost (CY61) as a function of flow rate	ft^3/day
QBI	Starting flow rate	ft^3/day
QBMX	Flow rate at which the starting value for the number of stations can be changed (variable name NST1)	ft^3/day
QBMX2	Flow rate at which the starting value for the number of stations can be changed (variable name NST2). QBMX2 should be higher than QBMX.	ft^3/day
QBS	Storage for QB (flow rate)	ft^3/day
SR1	First strength of steel	psi
SR2	Second strength of steel. Set at zero when only one strength of steel is to be processed.	psi
SR3	Third strength of steel. Set at zero when only one strength of steel is to be processed.	psi
SR4	Fourth strength of steel. Set at zero when only one strength of steel is to be processed.	psi
SR5	Fifth strength of steel. Set at zero when only one strength of steel is to be processed.	psi
SR6	Sixth strength of steel. Set at zero when only one strength of steel is to be processed.	psi
ST4	Strength of steel corresponding to the minimum transportation cost for a certain flow rate	psi
ST6	Strength of steel corresponding to the minimum transportation cost for certain diameter of the line	psi
SW	Switch for interpolation. SW = 1, correct interpolation. SW = 2, incorrect interpolation.	psi
S	Stress in pipe	psia
ST2	Strength in pipe corresponding to minimum yearly transportation cost	psia
TC	Critical temperature of mixture	$^{\circ}\text{R}$ or $^{\circ}\text{K}$
TF	Flowing temperature of mixture	$^{\circ}\text{R}$ or $^{\circ}\text{K}$
TN2	Critical temperature of N_2	$^{\circ}\text{R}$ or $^{\circ}\text{K}$
TCH4	Critical temperature of methane	$^{\circ}\text{R}$ or $^{\circ}\text{K}$

TABLE B-2 (Continued)

Symbol	Designation	Units
TG	Number of reduced temperatures for which the viscosity correlation is available (Figs. 11 and 12)	°R or °K
TCO2	Critical temperature of CO ₂	°R or °K
TC2H6	Critical temperature of C ₂ H ₆	°R or °K
TC3H8	Critical temperature of C ₃ H ₈	°R or °K
TIC4HO	Critical temperature of iC ₄ H ₁₀	°R or °K
TINV2	Minimum total investment corresponding to certain flow rate	cents/100 miles/Mcf/day
TINV6	Minimum total investment corresponding to certain diameter of the line	cents/100 miles/Mcf/day
TINV	Total investment. Sum of initial investments due to line and stations	cents/100 miles/Mcf/day
TN2	Critical temperature of nitrogen	°R
TNC4HO	Critical temperature of nC ₄ H ₁₀	°R or °K
TIC5H2	Critical temperature of iC ₅ H ₁₂	°R or °K
TNC5H2	Critical temperature of nC ₅ H ₁₂	°R or °K
TNC7H6	Critical temperature of normal heptane	°R
TOPCST	Total transportation cost for operation	\$/day
TPCST2	Total transportation cost for operation corresponding to certain flow rate	\$/day
TPCST6	Total transportation cost for operation corresponding to certain diameter of the line	\$/day
TR	Reduced temperature	°R
T	Counter corresponding to reduced temperatures for viscosity correlation (Figs. 11 and 12)	none
TNC6H4	Critical temperature of nC ₆ H ₁₄	°R or °K
TX	Critical temperature of X	°R or °K
TY	Critical temperature of Y	°R or °K
TZ	Critical temperature of Z	°R or °K
TSUM	Sum of critical temperature corresponding to X,Y,Z above	°R or °K
THK	Thickness of pipe	in.

TABLE B-2 (Concluded)

Symbol	Designation	Units
V1	Array corresponding to reduced temperature in the correlation of viscosity as a function of molecular weight, TR, and TP (Figs. 11 and 12)	--
VFM1	Array corresponding to volume as a function of pressure used for multiple interpolation for a temperature of 100°F	--
VFM	Array corresponding to volume as a function of pressure used for multiple interpolation for a temperature of 40°F	--
VISKML	Viscosity/($K_1 \times \sqrt{\text{molecular weight}}$) used for multiple interpolation	--
VISKM2	Viscosity/($K_1 \times \sqrt{\text{molecular weight}}$) used for multiple interpolation	--
VISKM	Viscosity/($K_1 \times \sqrt{\text{molecular weight}}$) used for multiple interpolation	--
VOLL	Volume used for multiple interpolation	ft^3
VOL2	Volume used for multiple interpolation	ft^3
VOL3	Volume used for multiple interpolation	ft^3
VOL4	Volume used for multiple interpolation	ft^3
VOL5	Volume used for multiple interpolation	ft^3
VOL	Volume used for multiple interpolation	ft^3
VP	Array used for maximum point on the dew point, bubble point curve at 40°F (pressure as a function of molecular weight)	psia
VPL	Array used for maximum point on the dew point, bubble point curve at 60°F (pressure as a function of molecular weight)	psia
V	Array corresponding to viscosity as a function of molecular weight, reduced temperature, and pressure	psia
X	Cost of compressor horsepower	\$/hp
X1	Fixed cost of compressor horsepower	\$/hp
Y	Cost of pipe	\$/ton
YW	Cost of pipe	\$/mile
Z	Compressibility factor	none
Z1	Compressibility factor at 40°F for TR less than 1.1 and corresponding to methane-propane system	none
Z2	Compressibility factor at 100°F for TR less than 1.1 and corresponding to methane-propane system	none
ZSW	Switch used for skipping over computation of viscosity and trial and error calculation for the compression ratio	none

TABLE B-3

PRINT-OUT OF DATA AND COMPUTED RESULTS OF SAMPLE PROBLEM
 (80% natural gas, 20% propane at 1560 psia minimum pressure, 100,000 psi steel, 24 in. dia line, 600 million cu ft/day flow rate)

PIPELINE CALCULATIONS

KCP(1)= 1.06, 1.08, 1.1, 1.12, 1.14, 1.16, 1.18, 1.2, 1.22, 1.24,

1.26, 1.28

MWT(1)= 78.4, 60.3, 49.3, 41.7, 36.2, 31.8, 28.3, 25.2, 22.7

20.5, 18.5, 16.7

MP= 17

PP(1)= 200., 400., 600., 800., 1000., 1250., 1500., 1750., 2000.,
 2250., 2500., 2750., 3000., 3500., 4000., 4500., 5000.

VFM(1)= 8.37, 2.23, 1.287, 1.262, 1.26, 1.212, 1.241,
 1.238, 1.227, 1.217, 1.213, 1.206, 1.198, 1.184, 1.174, 1.164, 1.154,
 13.06, 5.26, 1.917, 1.25, 1.251, 1.208, 1.219, 1.213, 1.201, 1.192,
 1.182, 1.173, 1.153, 1.142, 1.125, 1.111, 1.C96,
 17.77, 5.63, 2.963, 1.8C9, 1.266, 1.236, 1.215, 1.2, 1.185,
 1.174, 1.162, 1.153, 1.142, 1.125, 1.111, 1.096, 1.C84,
 22.42, 7.32, 4., 2.577, 1.74, 1.289, 1.234, 1.185, 1.167,
 1.15, 1.139, 1.123, 1.1, 1.083, 1.068, 1.054,
 24.14, 9.02, 5.04, 3.34, 2.341, 1.559, 1.317, 1.258, 1.222, 1.186,
 1.162, 1.142, 1.12, 1.091, 1.067, 1.046, 1.029,
 24.40, 10.71, 6.23, 4.11, 2.941, 2.02, 1.525, 1.4C2, 1.311, 1.252,
 1.205, 1.167, 1.138, 1.C45, 1.06, 1.033, 1.01,

TABLE B-3 (Continued)

24.75, 11.71, 7.11, 4.87, 3.54, 2.49, 1.868, 1.632, 1.459
 1.353, 1.27, 1.214, 1.173, 1.109, 1.062, 1.027, 0.9974
 VFM1(1)= 24.86, 7.26, 3.5, 1.465, 1.442, 1.414, 1.389, 1.366,
 1.353, 1.338, 1.325, 1.313, 1.303, 1.283, 1.260, 1.25, 1.236,
 25.8, 9.2, 4.59, 2.5, 1.482, 1.431, 1.399, 1.364, 1.345, 1.326,
 1.31, 1.292, 1.279, 1.255, 1.234, 1.217, 1.202,
 26.61, 9.75, 5.6, 3.54, 2.42, 1.533, 1.456, 1.401, 1.363, 1.333, 1.307,
 1.286, 1.266, 1.235, 1.213, 1.191, 1.173,
 27.29, 12.11, 6.45, 4.32, 3.09, 2.21, 1.615, 1.497, 1.424, 1.373,
 1.3333, 1.3, 1.273, 1.23, 1.197, 1.17, 1.147,
 27.88, 12.81, 7.71, 5., 3.67, 2.63, 1.946, 1.704, 1.55, 1.453, 1.388,
 1.34, 1.301, 1.241, 1.198, 1.162, 1.134,
 28.38, 13.33, 8.29, 5.78, 4.28, 3.06, 2.445, 2.033, 1.767, 1.594,
 1.494, 1.415, 1.356, 1.268, 1.208, 1.161, 1.125,
 28.8, 13.75, 8.73, 6.24, 4.75, 3.6, 2.861, 2.366, 2.039, 1.811,
 1.652, 1.531, 1.441, 1.317, 1.234, 1.172, 1.127
 COM1(1)= 0.2, .3, .4, .5, .6, .7, .8, .9
 V(1)= 36.9, 38., 39.1, 40., 41.1, 42.1, 43.2, 44.1, 45.4, 46.1,
 32.4, 34.1, 36., 38.1, 40., 42.3, 44.9, 46.4, 48.2, 50.,
 29., 31., 33.9, 37., 40.8, 44.2, 48., 51., 54.1, 57.,

TABLE B-3 (Continued)

26., 29.2, 34., 38.3, 43.2, 47.8, 51.8, 55.2, 59., 62.3,
24.2, 29., 34.1, 40.2, 46.2, 52., 57., 62., 66.3, 70.8,
23., 30., 39.8, 48.8, 57.2, 64.8, 72., 79., 85.9, 92.,
23., 34., 48.1, 60.6, 71.3, 80., 87.8, 95.1, 102.1, 108.3,
26., 42.2, 63., 77.8, 89., 98.4, 108., 116., 124.1, 130.3
52., 78., 95., 101.6, 116., 124., 132., 140., 148., 152. ,
90., 102., 113., 123., 132., 142., 151., 159., 163., 171.,
115., 126., 139., 149., 156., 163., 172., 180., 186., 192.,
135., 147., 158., 168., 178., 186., 195., 202., 210., 215.,
160., 170., 181., 192., 202., 212., 222., 231., 240., 248.,
186., 200., 210., 219., 228., 238., 247., 256., 265., 273.,
222., 233., 246., 256., 265., 273., 282., 290., 300., 305.,
265., 278., 290., 300., 310., 320., 330., 340., 350., 360.

MW(1)= 22.74, 28.1, 33.38

VP1(1)= 1400., 1340., 1050.

MF= 3 , TG= 16

V1(1)= 3., 2.5, 2., 1.7, 1.5, 1.3, 1.2, 1.1, 1. ,
.95, .90, .85, .75, .7, .65

MT(1)= 15., 18., 21., 24., 27., 30., 33., 35.

KS(1)= 1.3, 1.205, 1.14, 1.08, 1.04, 1.008, 1.004, 1.

TABLE B-3 (Continued)

PRD(1)= 1., 2., 3., 4., 5., 6., 7., 8., 9., 10.

NP=10

TN2= 226.9, TC02= 277.9, TCH4 = 343.3, TC2H6 = 549.8, TC3H8 = 660.

TIC4HO = 734.7, TNC4HO = 765.3, TIC5H2 = 829.8, TNC5H2= 845.6

TNC6H4 = 914.1

PN2= 492., PCO2= 730., PCH4 = 673.1, PC2H6 = 708.3, PC3H8 = 617.4

PIC4HO = 529.1, PNC4HO = 550.7, PIC5H2 = 483., PNC5H2= 489.5

PNC6H4 = 439.7

MN2= 28.02, MC02= 32., MCH4 = 16.04, MC2H6 = 30.07, MC3H8 = 44.09

MIC4HO = 58.12, MNC4HO = 58.12, MIC5H2 = 72.15, MNC5H2= 72.15

MNC6H4 = 84.16

SR1= 65.E3, SR2=1.E5, SR3= 0., SR4= 0., SR5= 0., SR6= 0.

Y(1)= 228., 254., 265., 384., 467., 535.

Y (1)= 254., 384., 0., 0., 0., 0.

FF= .936, FFE= 1., F= .72, E= 1.0 , EFF= 0.8

X= 165., N=1200., H= 3000., ALPHA= .15, CF= 0.2, CLNL= 850.

CLMS=19., LG= 0.005, B= 0.15, FHPHR= 8.7E-3, FLUP=1.

PRPRHO= 0.51, IBUTRH= 0.563, NBUTRH= 0.583, BUTI= 0., DBLT= 1.E4

BUTF= 0., EE= 250.E-6

GASOST= 0.20 , X1=2.7E5 , AD= 1.

TABLE B-3 (Continued)

LT= 1000.

NOSI= 6, DNGS= 1, NOSF= 25

CRI= 1.4 , DCR= .1

TF= 520., COMPSW=1

PROPI= 0. , DPRDP=2E4, PROPF= 0.

QBI=3.0E8, DQB=0.5E8, QBF1= 6.E8

QBI= 1.5E8, QB12= 4.0E8, QB13= 5.0E8

NOSI= 7 , NOS12= 6, NOS13= 6

QBF12= 1.6E9, QBF13= 1.7 E9

PMIN= 600. , MMWT= 30.

OD11= 16., OD12= 24., OD13= 30.

CRMAX= 1.65, NN= 5,

CRI= 1.2

SR1= 65.E3,

SR2= 1.E5 , Y(2)= 384.

TNC7H6=972.36 , PNC7H6= 397.71 , MNC7H6= 100.2

MMWT= 30.

NOSF= 35

NOS12= 10, NOS13= 14

FF= .936 , FFF= 3

TABLE B-3 (Continued)

QBMX= 5.E9 , QBMX2= 5.E9
 DQB2= 2.E8, DQB4= 3.E8
 DQB2= 1.E8, DQB4= 2.E8
 SR1= 65.E3, Y(1)= 2E5., SR2= 1.E5, Y(2)= 3E4.
 SR1= 1.E5, Y(1)= 3E4.
 FFF= 1 , NN= 5
 SR2= 0.
 DQB2= 1.E8, DQB4= 2.E8
 FX= 0.8, MX= 17.387, TX= 361.35, PX= 673.58, FC3H8= 0.2
 PMIN= 1460.
 GD11= 24., GD12=0., GD13=0.
 NOS1= 18
 QBI= 6.0E8, DQB1=1.0E8, QBF1= 6.0E8 * End of Input Data Print-out
 NEW DIAMETER OF PIPE
 0D = 24.C00000
 NEW STRENGTH OF STEEL
 S = 1.00000E 05
 NEW FLOW RATE AND COMPOSITION
 NATGAS = .8C0CC0,
 QBF = 6.00000E 08
 FCH4 = .C00000,
 FNC7H6 = .C00C0C,
 QB = 6.00000E 08,
 MWT = 22.72760C
 FC3H8 = .200000,
 FC2H6 = .CCC0C0,
 FN2 = .CCCC00,
 TC = 422.039993,
 PC = 662.343987,
 C = 783710
 F1C4H0 = .CCC00C,
 FNC5H2 = .000000C,
 FNC6H4 = .0C0000
 FNC4HC = .000000
 FNC6H4 = .0C0000

TABLE B-3 (Continued)

NEW N.	U.	S.	P2 =	1559.999985,	K6 =	4.710186E 06,	CR4 =	1.200000
P1 =	1826.218842,		P2 =	1559.999985,	K6 =	4.715852E 06,	CR4 =	1.170653
P1 =	1825.626C22,		P2 =	1559.999985,	K6 =	4.715852E 06,	CR4 =	1.170653
CR =	1.170273,		A =	5.260765,	Z =	*591005,	K =	1.219763
NOS =	18,		L =	55.555555,	LT =	1C00.000000,	PAVG =	1696.597672
P1 =	1825.626C22,		PAVG =	1696.597672,	CR =	1.170273,	S =	1.000000E 25
OD =	24.0000C0,		D =	23.391458,	QB =	5.853216E 08,	Q83 =	6.000127E 28
IINVL =	1852.817642,		IINVS =	238.198824,	CMMOP =	*13E381,	CMMAM =	*859322
IINV =	2091.016449,		CY =	*9977C3,	THK =	*3C4271,	YW =	7.8C7482E 04
TOPCST =	5.986341E 04,		TINV =	1.254636E 08,	A =	5.26C765,	Z =	*591005
P2 =	1559.999985,		TR =	1.232111,	PR =	2.561505,	FT =	21.451289
NEW N. U. S.								
P1 =	1812.609360,		P2 =	1559.999985,	K6 =	4.715923E 06,	CR4 =	1.170653
CR =	1.161929,		A =	5.018107,	Z =	*591005,	K =	1.219763
NOS =	19,		L =	52.631579,	LT =	1C00.000000,	PAVG =	1696.286377
P1 =	1812.609360,		PAVG =	1696.286377,	CR =	1.161929,	S =	1.000000E 05
OD =	24.0000C0,		D =	23.395797,	QB =	5.855017E 08,	Q83 =	6.002783E 08
IINVL =	1842.919586,		IINVS =	243.795176,	CMMOP =	*13E977,	CMMAM =	*857554
IINV =	2086.714752,		CY =	*996531,	THK =	*3C2102,	YW =	7.752524E 04
TOPCST =	5.981961E 04,		TINV =	1.252610E 08,	A =	5.018107,	Z =	*591005
P2 =	1559.999985,		TR =	1.232111,	PR =	2.561035,	FT =	21.451382
NEW N. U. S.								
P1 =	1800.674896,		P2 =	1559.999985,	K6 =	4.717453E 06,	CR4 =	1.170653
P1 =	1800.544296,		P2 =	1559.999985,	K6 =	4.718825E 06,	CR4 =	1.154279
CR =	1.154195,		A =	4.791913,	Z =	*591005,	K =	1.219763
NOS =	20,		L =	5C.000000,	LT =	1C00.000000,	PAVG =	1683.21C083
P1 =	1800.544296,		PAVG =	1683.210083,	CR =	1.154195,	S =	1.000000E 05
OD =	24.0000C0,		D =	23.399818,	QB =	5.85173C E 08,	Q83 =	6.0000C28E 08
IINVL =	1835.323303,		IINVS =	249.211349,	CMMOP =	*139462,	CMMAM =	*856658
IINV =	2084.534637,		CY =	*996120,	THK =	*3C0091,	YW =	7.7C1575E 04

TABLE B-3 (Continued)

TOPCST =	5.976749E 04,	TINV =	1.25C727E C8,	A =	4.791913,	Z =	•591C55
P2 =	1559.999985,	TR =	1.232111,	PR =	2.541293,	FT =	21.455258
NEW N. O. S.							
P1 =	1789.821671,	P2 =	1555.999985,	K6 =	4.71884CE 06,	CR4 =	1.154279
CR =	1.147322,	A =	4.589842,	Z =	•591005,	K =	1.219763
NOS =	21,	L =	47.619047,	LT =	1C00.000000,	P AVG =	1683.141785
P1 =	1789.821671,	P AVG =	1683.141785,	CR =	1.147322,	S =	1.5CCCE 05
OD =	24.000000,	D =	23.403393,	QB =	5.85327CE 08,	Q63 =	6.72291E 08
INVL =	1827.149841,	IINVS =	254.642143,	CMMOP =	•139965,	C MNAN =	•855531
INV =	2081.791962,	CY =	•995496,	THK =	•298304,	Y W =	7.656288E C4
TOPCST =	5.975257E 04,	TINV =	1.249552E C8,	A =	4.589842,	Z =	•591005
P2 =	1559.999985,	TR =	1.232111,	PR =	2.541190,	FT =	21.455278
NEW N. O. S.							
P1 =	1779.913513,	P2 =	1560.000000,	K6 =	4.720047E 06,	CR4 =	1.154279
P1 =	1779.8117871,	P2 =	1560.000000,	K6 =	4.721141E 06,	CR4 =	1.149975
CR =	1.140909,	A =	4.400415,	Z =	•591005,	K =	1.219763
NOS =	22,	L =	45.454545,	LT =	1C00.000000,	P AVG =	1672.37CE 07
P1 =	1779.8117871,	P AVG =	1672.370087,	CR =	1.140909,	S =	1.5CCCE 05
OD =	24.000000,	D =	23.406727,	QB =	5.85C543E 08,	Q63 =	6.72291E 08
INVL =	1820.839523,	IINVS =	255.936623,	CMMOP =	•140379,	C MNAN =	•355113
INV =	2080.776123,	CY =	•995493,	THK =	•296636,	Y W =	7.614C3CE C4
TOPCST =	5.972975E 04,	TINV =	1.24847CE C8,	A =	4.40C415,	Z =	•591005
P2 =	1560.000000,	TR =	1.232111,	PR =	2.524927,	FT =	21.456490
NEW N. O. S.							
P1 =	1770.827179,	P2 =	1560.000000,	K6 =	4.721152E 06,	CR4 =	1.149975
CR =	1.135146,	A =	4.229429,	Z =	•591005,	K =	1.219763
NOS =	23,	L =	43.478261,	LT =	1C00.000000,	P AVG =	1672.32C221
P1 =	1770.827179,	P AVG =	1672.320221,	CR =	1.135146,	S =	1.5CCCE 05
OD =	24.000000,	D =	23.409727,	QB =	5.851828E 08,	Q63 =	6.71921E 08
INVL =	1813.987427,	IINVS =	265.2373C9,	CMMOP =	•140818,	C MNAN =	•354476

TABLE B-3 (Continued)

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APPENDIX C
EXAMPLE HAND CALCULATION

ILLUSTRATIVE EXAMPLE PROBLEM BY HAND CALCULATION

Problem

Compute the minimum cost of the pipeline system and cost of transporting 600 million cu ft per day of a mixture of 80% natural gas and 20% propane (molal basis) over a distance of 1000 miles at 60°F through 24-in. OD pipe having a strength of 100,000 psi. Select the minimum pressure as 100 psi above the dew point at 60°F. This is the first of a series of calculations to be done on the computer at increasing flow rates to find the minimum cost of some optimum flow rate. The number of compressor stations is a variable in this calculation and will include one more and one less station than the optimum.

Solution

MINIMUM PRESSURE

The minimum pressure for 80% natural gas and 20% propane in the range 40° to 100°F is 1460 psia and therefore the minimum pressure in the pipeline is set at 1560 psia. This information is given in the program as data.

HORSEPOWER AND MAXIMUM LINE PRESSURE

By use of the flow equation (Eq. 12) the pressure drop in the line is computed to obtain the maximum pressure in the line. From the inlet and outlet pressures the horsepower will be computed. The steps solving the flow equation are given.

1. Calculation of Gas Gravity Factor (FGR)

The molecular weight of the gas from Table 1 is 17.4.

$$\text{Molecular Weight} = 0.8 \times 17.4 + 0.2 \times 44.09 = 22.73 \quad (22.728*) \quad (\text{C-1})$$

*The values in parentheses are those on the computer print-out for this example.

$$\text{Gas gravity, } G = \frac{22.73}{29.0} = 0.784 \text{ (0.7837)} \quad (\text{C-2})$$

$$\text{FGR} = \sqrt{\frac{0.6}{0.784}} = \underline{\underline{0.874}} \quad (\text{C-3})$$

Temperature flowing factor (FTF)

$$\text{FTF} = \sqrt{\frac{\text{TF}}{520}} = 1 \quad (\text{C-4})$$

2. Trial and Error Calculations for Compression Ratio

From computer results we know that optimum value for number of stations is 22. So we shall make computations for 21 stations. NOS = 21 (including the one at the entrance).

$$\text{Let assumed compression ratio be } = 1.16 \quad (\text{C-5})$$

$$\text{Maximum pressure in pipeline } P_1 = 1.16 \times 1560 = 1810 \text{ psia} \quad (\text{C-6})$$

3. Length Between Compressor Stations

$$L = \frac{1000}{21} = 47.6 \text{ (47.62) miles} \quad (\text{C-7})$$

4. Thickness and ID of the Pipe

$$\text{THK} = \frac{P_1 \times \text{OD}}{2 \times F \times E \times S} = \frac{1810 \times 24}{2 \times 0.72 \times 1 \times 100,000} = 0.301 \text{ in.} \quad (\text{C-8})$$

$$D(\text{ID}) = 24 - 2 \times 0.301 = 23.398 \text{ in.} \quad (\text{C-9})$$

$$FD = \text{Line diameter factor} = D^{2.5} = 23.398^{2.5} = 2620 \text{ in.}^{2.5} \quad (\text{C-10})$$

5. Average Pressure

$$\begin{aligned}
 P_{AVG} &= \frac{2}{3} \left(P_1 + P_2 - \frac{P_1 P_2}{P_1 + P_2} \right) \\
 &= \left(1810 + 1560 - \frac{1810 \times 1560}{1810 + 1560} \right) \\
 &= \frac{2}{3}(2532) = 1690 \text{ psia} \quad (\text{C-11})
 \end{aligned}$$

6. Calculation of Compressibility Factor (Z) and FPV

This is calculated by Sarem's method using Lagrange polynomials which fit best the compressibility factor chart. A subroutine ZFAC has been prepared which calculates Z based on TR and PR, the reduced temperature and pressure, respectively.

Pseudo Critical Properties

$$TC = 0.8 \times 364.7 + 0.2 \times 666 = 424^{\circ}\text{R} \quad (422) \quad (\text{C-12})$$

$$PC = 0.8 \times 673.6 + 0.2 \times 617.4 = 662.4 \text{ psia} \quad (662.3) \quad (\text{C-13})$$

$$TR = \frac{520}{424} = 1.23 \quad (1.232) \quad (\text{C-14})$$

$$PR = \frac{1810}{662.4} = 2.73 \quad (\text{C-15})$$

$$Z = 0.58 \quad (\text{C-16})$$

$$FPV = \sqrt{\frac{1}{Z}} = \sqrt{\frac{1}{0.58}} = 1.315 \quad (\text{C-16a})$$

7. Calculation of Viscosity (Bicher and Katz Method)

$$\begin{aligned}
 TR &= 1.23 \\
 PR &= 2.73
 \end{aligned}
 \boxed{\quad \text{Eqs. (C-14), (C-15) above}}$$

$$\text{Molecular weight (MWT)} = 22.73$$

Fig. 11,

$$KI = 1.1 \quad (\text{C-17})$$

and

$$\begin{aligned} \frac{VISKM}{KIX \sqrt{MWT}} &= 5.18 \quad \text{or} \quad VISKM = 51.8 \times 1.1 \times \sqrt{22.73} \\ &= 272 \text{ micropoises} \\ MU &= VISKM = 272 \times 6.72 \times 10^{-8} \\ &= 1.825 \times 10^{-5} \text{ lb/ft sec} \end{aligned} \quad (\text{C-18})$$

8. Calculation of Transmission Factor (FT)

$$FT = \frac{2}{\sqrt{\overline{FM}}} \quad (\text{C-19})$$

where FM = Moody friction factor

$$RE = \frac{D \rho V}{\mu} = 1.3526 \times 10^{-5} \frac{QBxG}{DxMU} \quad (\text{C-20})$$

from Eqs. (C-2), (C-9), and (C-18)

$$\begin{aligned} &= 1.3526 \times 600 \times 10^6 \times \frac{0.784}{23.4 \times 1.825} \\ &= 1.49 \times 10^7 \end{aligned} \quad (\text{C-21})$$

$$\frac{2}{\sqrt{\overline{FM}}} = 4 \log \frac{D}{EE} + 2.28 - 4 \log (1 + 9.34 \frac{D/EE}{RE \sqrt{\overline{FM}}})$$

$$\text{Let } \frac{2}{\sqrt{\overline{FM}}} = 21.45$$

Is

$$\begin{aligned} 21.45 &= 4 \log \frac{23.4}{250 \times 10^{-6}} + 2.28 - 4 \log (1 + 9.34 \frac{250 \times 10^{-6}}{1.49 \times 10^7 \times 2}) \\ &= 4 \times 4.974 + 2.28 - 4 \log (1 + 6.3 \times 10^{-1}) \\ &= 22.17 - 0.82 = 21.35 \\ \therefore \frac{2}{\sqrt{FM}} &= 21.40 \end{aligned} \quad (C-22)$$

9. Drag Factor (FF)

It was agreed to use a bend index of 200°/mile, plastic lined pipe (Fig. 13)

$$FF = 0.936 \quad (C-23)$$

10. Checking for Assumed Compression Ratio

Let

$$K_6 = 77.5 \times FGR \times FTF \times FPV \times FT \times FF \times FFE \times FD$$

From Eqs. (C-3), (C-4), (C-10), (C-16a), (C-22), (C-23)

$$K_6 = 77.5 \times 0.874 \times 1 \times 1.34 \times 21.4 \times 0.936 \times 1 \times 2620$$

$$K_6 = 4.76 \times 10^6$$

$$\begin{aligned} P_1 &= \sqrt{P_2^2 + \frac{QB^2 \times L}{(K_6)^2}} \\ &= \sqrt{1560^2 + (600 \times 10^6)^2} \times \frac{47.6}{(4.76 \times 10^6)^2} \\ &= 1790 \text{ (1790) psia} \end{aligned}$$

$$CR = \frac{P_1}{P_2} = \frac{1790}{1560} = 1.147$$

Second Trial for Compression Ratio

Therefore let us assume CR as 1.147. Repeating steps 2-10 we have

$$P_1 = 1.147 \times 1560 = 1790 \text{ psia}$$

$$THK = \frac{P_1 \times OD}{2 \times F \times E \times S} = \frac{1790 \times 24}{2 \times 0.72 \times 1 \times 100,000} = 0.298 \text{ in. (0.298)}$$

$$D = 24 - 2 \times 0.298 = 23.40 \text{ in. (23.403)}$$

$$FD = 23.403^{2.5} = 26.25 \text{ in.}^{2.5}$$

$$\text{Average pressure} = P_{AVG} = \frac{2}{3} (1790 + 1560 - \frac{1790 \times 1560}{1790 + 1560})$$

$$= 1677 \text{ psia (1678)}$$

$$TR = 1.23 (1.232)$$

$$PR = \frac{1677}{662.4} = 2.53 (2.533)$$

$$Z = 0.565$$

$$FPV = \sqrt{\frac{1}{0.565}} = 1.33$$

$$\frac{VI SKM}{KI \sqrt{MWT}} = 39.3 \text{ or } VI SKM = 39.3 \times 1.1 \times \sqrt{22.73}$$

$$= 206 \text{ micropoises}$$

$$MU = 206 \times 6.72 \times 10^{-8} = 1.39 \times 10^{-5} \text{ lb/ft sec}$$

$$\begin{aligned}
RE &= \frac{D \rho v}{\mu} = 1.3526 \times 10^{-5} \frac{QBxG}{DxMU} \\
&= 1.3526 \times 10^{-5} \times 600 \times 10^6 \times \frac{0.784}{23.4 \times 1.39 \times 10^{-5}} \\
&= 1.96 \times 10^7
\end{aligned}$$

$$\frac{2}{\sqrt{FM}} = 4 \log \frac{D}{EE} + 2.28 - 4 \log (1 + 9.34 \frac{D/EE}{RE \sqrt{FM}})$$

Let $\frac{2}{\sqrt{FM}} = 21.45$

$$\begin{aligned} \frac{2}{\sqrt{FM}} &= 4 \log \frac{23.4}{250 \times 10^{-6}} + 2.28 - 4 \log (1 + 9.34 \underbrace{\frac{250 \times 10^{-6}}{1.96 \times 10^7 \times 2}}_{0.48}) \\ &= 4 \times 4.97 + 2.28 - 4 \times 0.17 \\ &= 21.49 \end{aligned}$$

Verification from Moody Friction Factor Chart

$$EE/D = \frac{250 \times 10^{-6}}{23.4} = 10.7 \times 10^{-6} = 0.0000107$$

$$FM = 0.0086$$

$$\frac{2}{\sqrt{FM}} = \frac{2}{9.22 \times 10^{-1}} = 21.6$$

$$\begin{aligned} K_6 &= 77.5 \times FGR \times FTF \times FPV \times FT \times FF \times FFE \times FD \\ &= 77.5 \times 0.874 \times 1 \times 1.33 \times 21.46 \times 0.936 \times 1 \times 2625 \\ &= 4.77 \times 10^6 \end{aligned}$$

$$\begin{aligned} P_1 &= \sqrt{P_2^2 + \frac{QB^2 L}{K_6^2}} \\ &= \sqrt{1560^2 + (600 \times 10^6)^2 \times \frac{47.6}{(4.77 \times 10^6)^2}} \\ &= 1790 \text{ psia} \end{aligned}$$

$$CR = \frac{P_1}{P_2} = \frac{1790}{1560} = \underline{1.147} \text{ which checks}$$

Calculation of Horsepower of Compressor

Using Equation (22) to obtain the horsepower per station

$$A = \frac{0.0854 \times K \times TF \times Z^{\frac{K-1}{K}} (CR - 1)}{(K-1) \times EFF} \times \frac{14.73}{14.65}$$

*Z is based on inlet conditions.

$$PR = \frac{1560}{662.4} = 2.353$$

$$TR = 1.232$$

$$Z = 0.575 (0.591)$$

$$\text{Molecular weight} = 22.73$$

Using chart in NGSMA, p. 27 (1957) to find K

$$K = \frac{C_p}{C_v} = 1.22 (1.22)$$

$$\text{Horsepower } A = \frac{0.0854 \times 1.22 \times 520 \times 0.575 \times \left(1.147 \frac{1.22-1}{1.22} - 1 \right)}{0.22 \times 0.8} \times \frac{14.73}{14.65}$$

$$= 4.48 (4.588) \text{ hp/MMcf/day}$$

Cost of Pipeline and Compression Stations

From Eqn. (A-3) the cost of the pipeline in dollars per mile (YW) is computed

$$YW = 28.2 \times (D + THK) \times THK \times Y$$

$$= 28.2 \times (23.4 + 0.298) \times 0.298 \times 384$$

$$= 7.65 \times 10^4 (7.656 \times 10^4) \text{ \$/mile}$$

The Initial Investment in Pipeline (IINVL)

$$IINVL = (YW + N \times OD + H) \times \frac{10^7}{QB}$$

$$= (7.65 \times 10^4 + 1200 \times 24 + 3000) \times \frac{10^7}{600 \times 10^6}$$

$$= 1805 \text{ (1827*) cents/100 miles/Mcf/day}$$

Note: *This value takes into account the adjustment in quantity of gas flowing through the pipeline due to fuel consumption at each station. This is the reason for the discrepancy in the two values. This also occurs in the costs stated below.

Initial Investment for Stations (IINVS)

The initial investment for stations is given by Eq. (26)

$$\begin{aligned} IINVS &= \left(X + \frac{X_1 \times 10^6}{A \times Q_B} \right) \times \frac{A}{L_T} \times NOS \times 10 \\ &= \frac{10(X)(A)(NOS)}{L_T} + \frac{(X_1)(10^7)(NOS)}{(Q_B)(L_T)} \\ &= \frac{(10)(165)(4.48)(21)}{1000} + \frac{270,000 \times 10^7 \times 21}{600 \times 10^6 \times 1000} \\ &= 155.2 + 94.4 = 249.6 \text{ (254.3) cents/100 miles/Mcf/day} \end{aligned}$$

Operating Cost (CMMOP)

The operating cost is given by Eq. (28)

$$\begin{aligned} CMMOP &= ((365 \times 24 \times FPHR \times CF + CLMS) \times \frac{A \times NOS \times FOOP}{365 \times L_T} \\ &\quad + LG \times GASCST \times \frac{10^3}{L_T} + \frac{CLML \times 10^6}{Q_B \times 365}) \times 10 + \frac{AD}{3650} \\ &= (\underbrace{(365 \times 24 \times 8.7 \times 10^{-3} \times 0.20 + 19)}_{15.2} \times \frac{4.48 \times 21 \times 1}{365 \times 1000} \\ &\quad + 0.005 \times 0.20 \times \frac{10^3}{1000} + \frac{850 \times 10^6}{600 \times 10^6 \times 365}) \times 10 + \frac{1}{3650} \\ &= (35.2 \times 2.58 \times 10^{-4} + 0.001 + 3.88 \times 10^{-3}) \times 10 + \frac{1}{3650} \end{aligned}$$

$$= (0.00882 + 0.001 + 0.00388) \times 10 + \frac{1}{3650} *$$

$$= 0.1395 (0.139965) \text{ cents/100 miles/Mcf.}$$

Amortization Cost

The amortization cost is based on paying 15% of investment per year, Eq. (27).

$$\begin{aligned} \text{CMMAM} &= (((Y_W + N \times O_D + H) \times \frac{B \times 10^6}{Q_B}) \\ &\quad + \text{ALPHA} \times (X + \frac{X_1 \times 10^6}{A \times Q_B}) \left(\frac{A \times N_O_S}{L_T} \right) \left(\frac{F_OOP}{365} \right) \times 10 \\ &= ((7.65 \times 10^4 + 1200 \times 24 + 3000) \frac{(0.15) \times 10^6}{600 \times 10^6} \\ &\quad + 0.15 (165 + \frac{270,000 \times 10^6}{4.48 \times 600 \times 10^6}) \left(\frac{4.48 \times 21}{1000} \right) \times \frac{1 \times 10}{365} \\ &= (27.1 + 3.75) \times \frac{10}{365} \\ &= 0.845 (0.8555*) \text{ cents/100 miles/Mcf} \end{aligned}$$

*See note at the end of the Initial Investment on Pipeline.

Fuel Consumption at the Compressor Stations (LSG)

This is needed to obtain the pipe line delivery.

$$\begin{aligned} \text{LSG} &= (24 \times F_H P_H R \times A \times Q_B \times 10^{-3}) \times N_O_S + L_G \times Q_B \\ &= 24 \times 8.7 \times 10^{-3} \times 4.48 \times 600 \times 10^{+6} \times 10^{-3} \times 21 + 0.005 \times 600 \times 10^6 \\ &= (11.75 + 3) \times 10^6 = 14.75 \times 10^6 \text{ ft}^3/\text{day} \end{aligned}$$

*As indicated earlier, this should be AD/36.5. For this example the cost of transporting gas, CY, would be raised about 3% by the correction.

Quantity of gas, at the exit

$$= (600 - 14.75) \times 10^6$$

$$= \underline{585.25} \times 10^6 (\underline{585.33} \times 10^6) \text{ ft}^3/\text{day}$$

$$\text{CY} = \text{CMMAM} + \text{CMMOP} = 0.845 + 0.1395 = \underline{0.9845} (0.9955) \text{ cents/100 miles/Mcf}$$

$$\text{IINV} = \text{IINVL} + \text{IINVS} = 1805 + 250 = 2075 \text{ cents/100 miles/Mcf}$$

In reviewing the method of finding amortization and operating cost, the full 600 million cu ft of gas are used even though only 585.25 million are sold. The computer program finds costs by using the actual flow in each segment, and therefore is based on an average pipeline delivery somewhere between 600 and 585 million cu ft.

$$\text{Total Investment (TINV)} = \text{IINV} \times \text{QB} \times 10^{-7} \times \text{LT}$$

$$= 2075 \times 600 \times 10^6 \times 10^{-7} \times 1000$$

$$= 1.245 \times 10^8 (1.250 \times 10^8) \text{ dollars/day}$$

$$\text{Total Operating Cost (TOPCST)} = \text{CY} \times \text{QB} \times 10^{-7} \times \text{LT}$$

$$= 0.9845 \times 600 \times 10^{+6} \times 10^{-7} \times 1000$$

$$= 59,070 (59,730) \text{ dollars/day}$$

APPENDIX D

PHASE DIAGRAMS DETERMINED BY KURATA FOR MIXTURES NOT STUDIED IN THIS REPORT

These diagrams, reproduced from the Ph.D. thesis of Fred Kurata at The University of Michigan in 1941, under the direction of Professor Donald L. Katz, represent the experimentally determined phase diagrams. The compositions of the mixtures are listed on the diagrams.

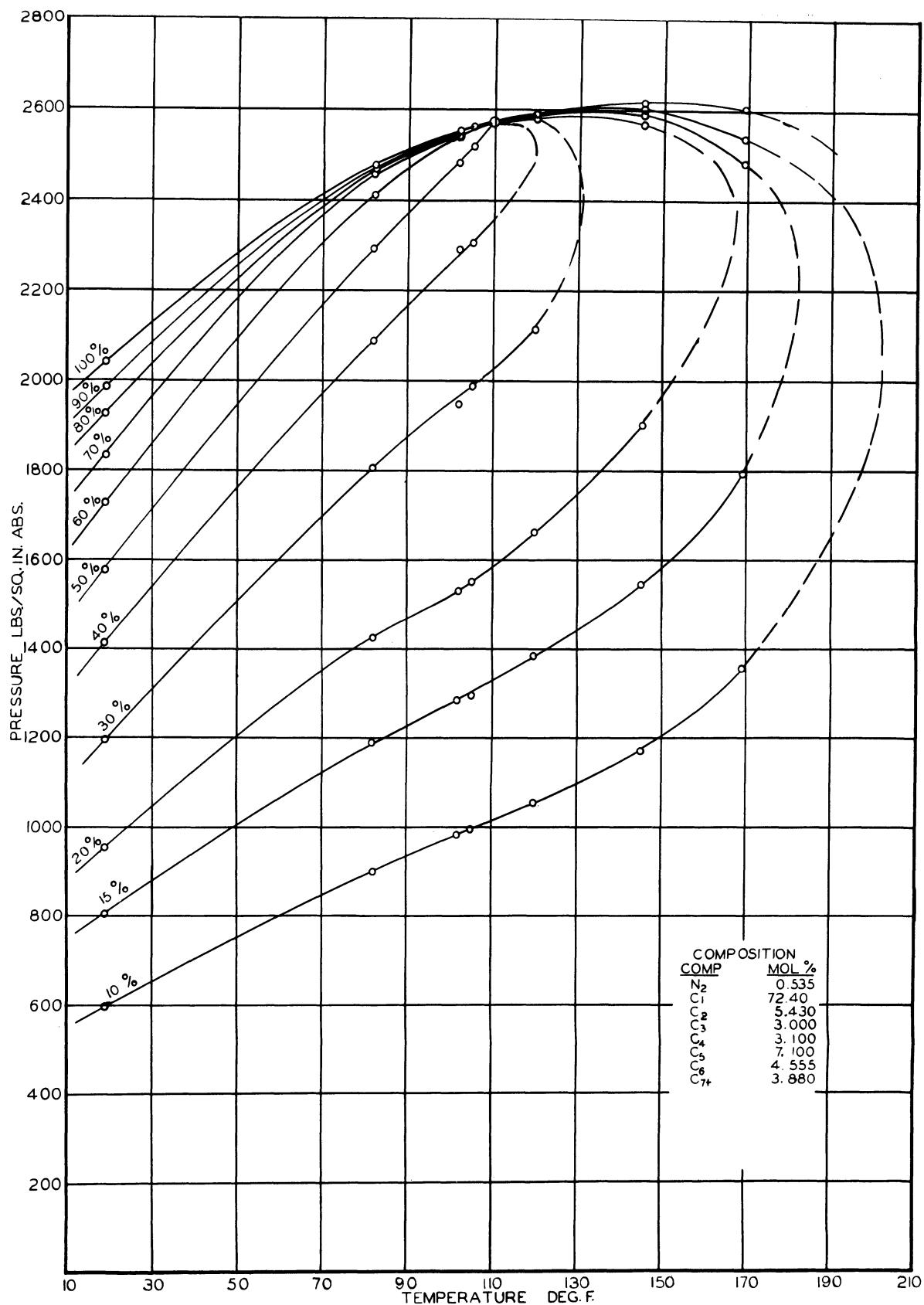


Fig. D-1. Phase diagram for mixture S-3.

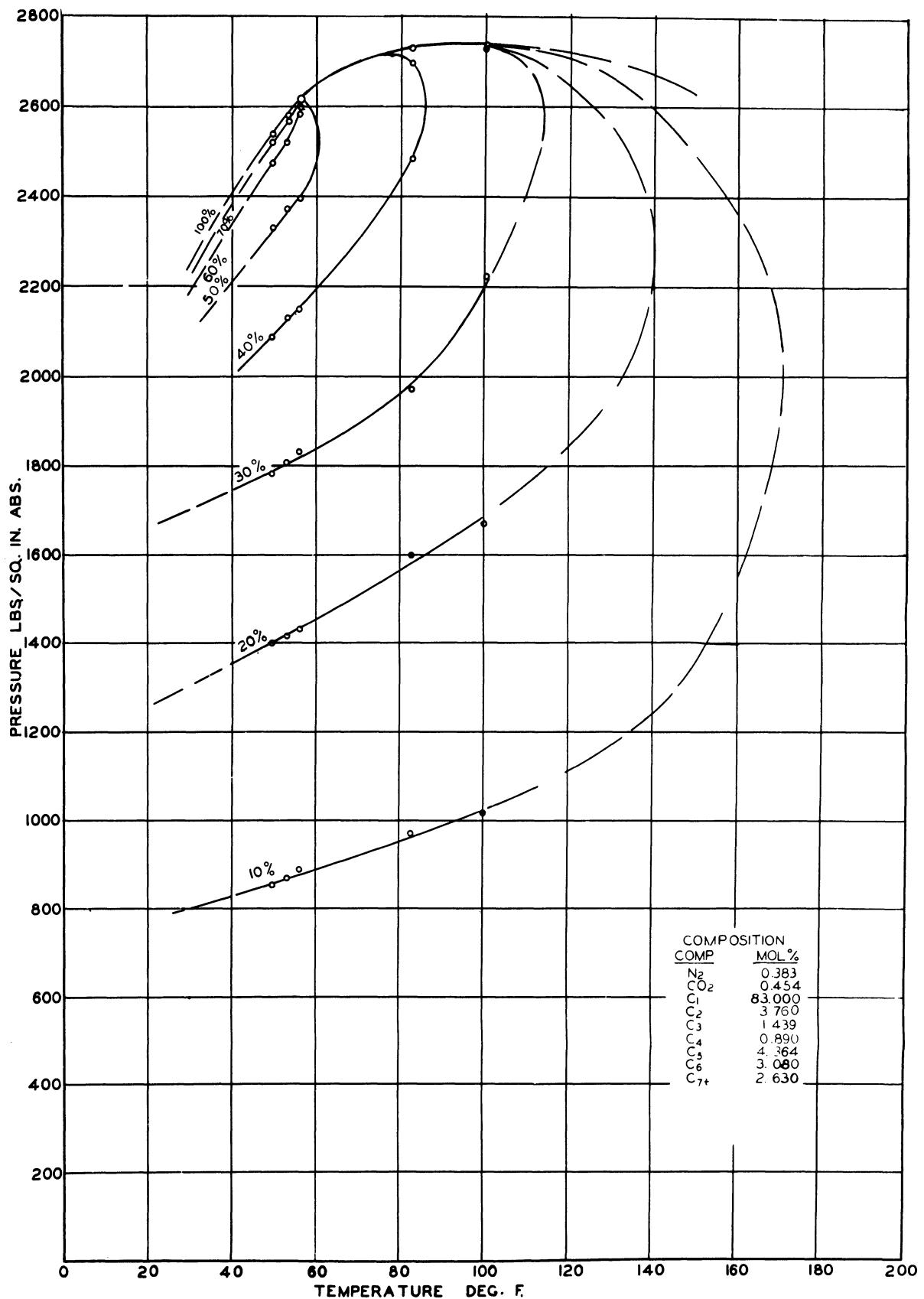


Fig. D-2. Phase diagram for mixture T-1.

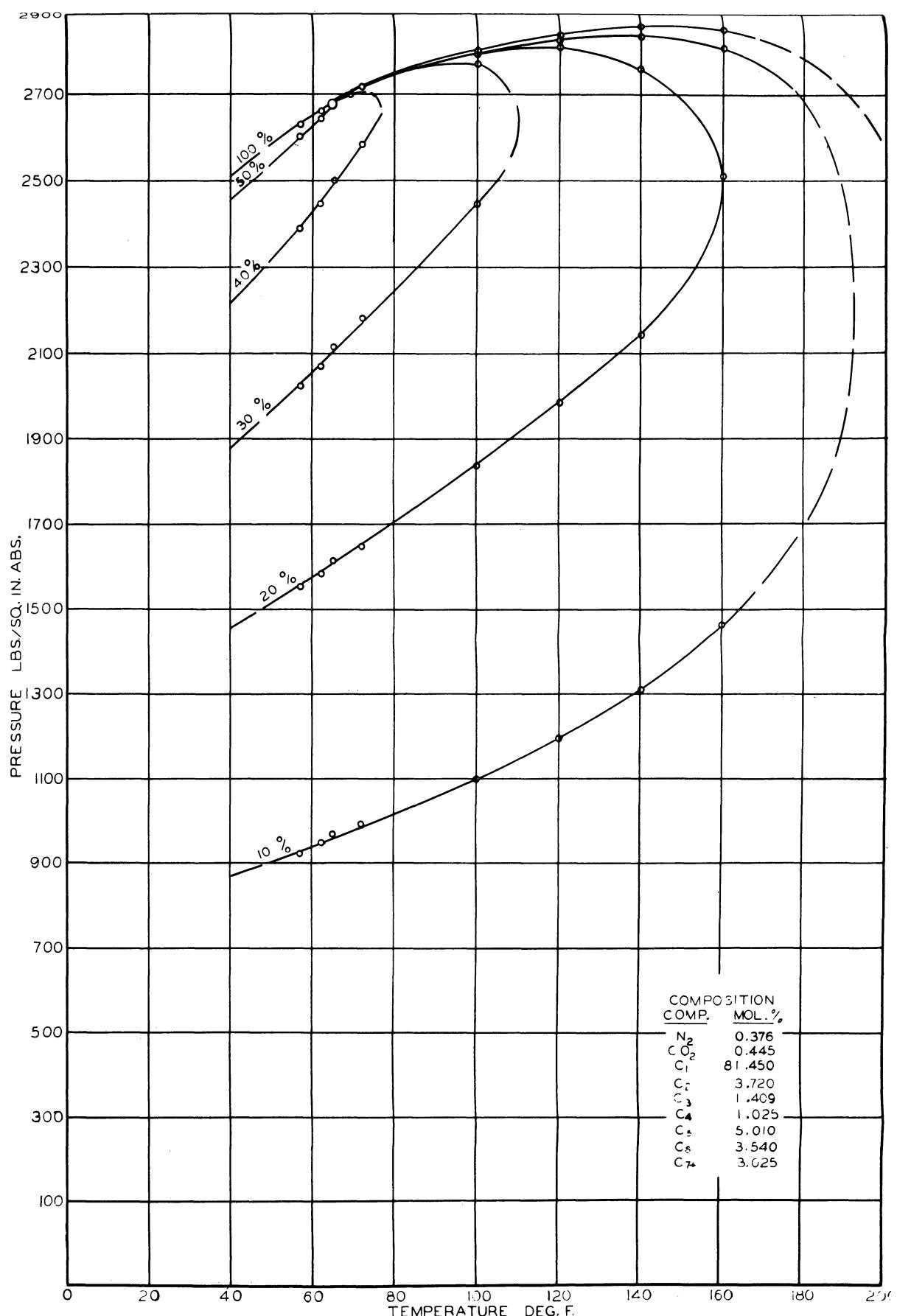


Fig. D-3. Phase diagram for mixture T-3.

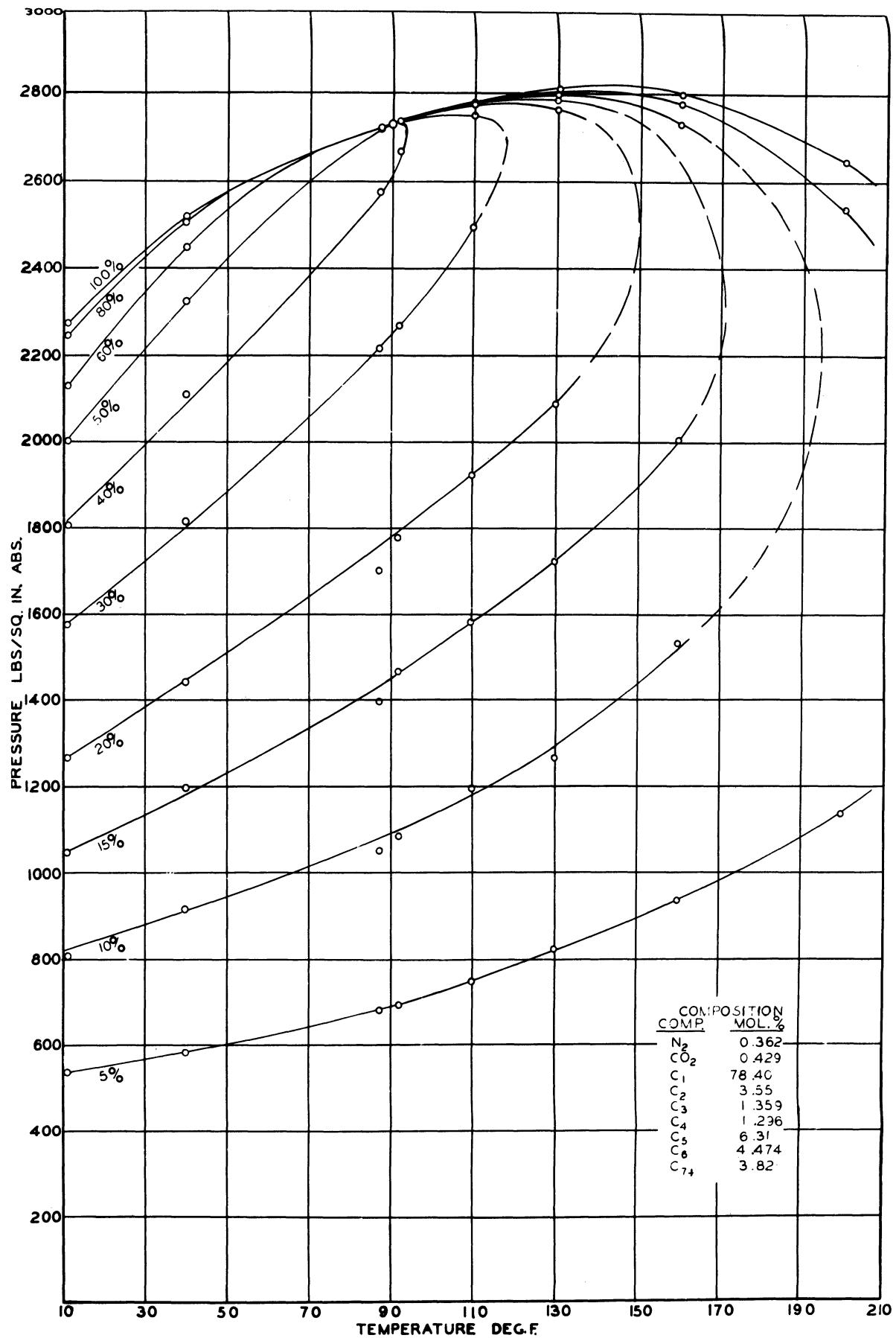


Fig. D-4. Phase diagram for mixture T-4.

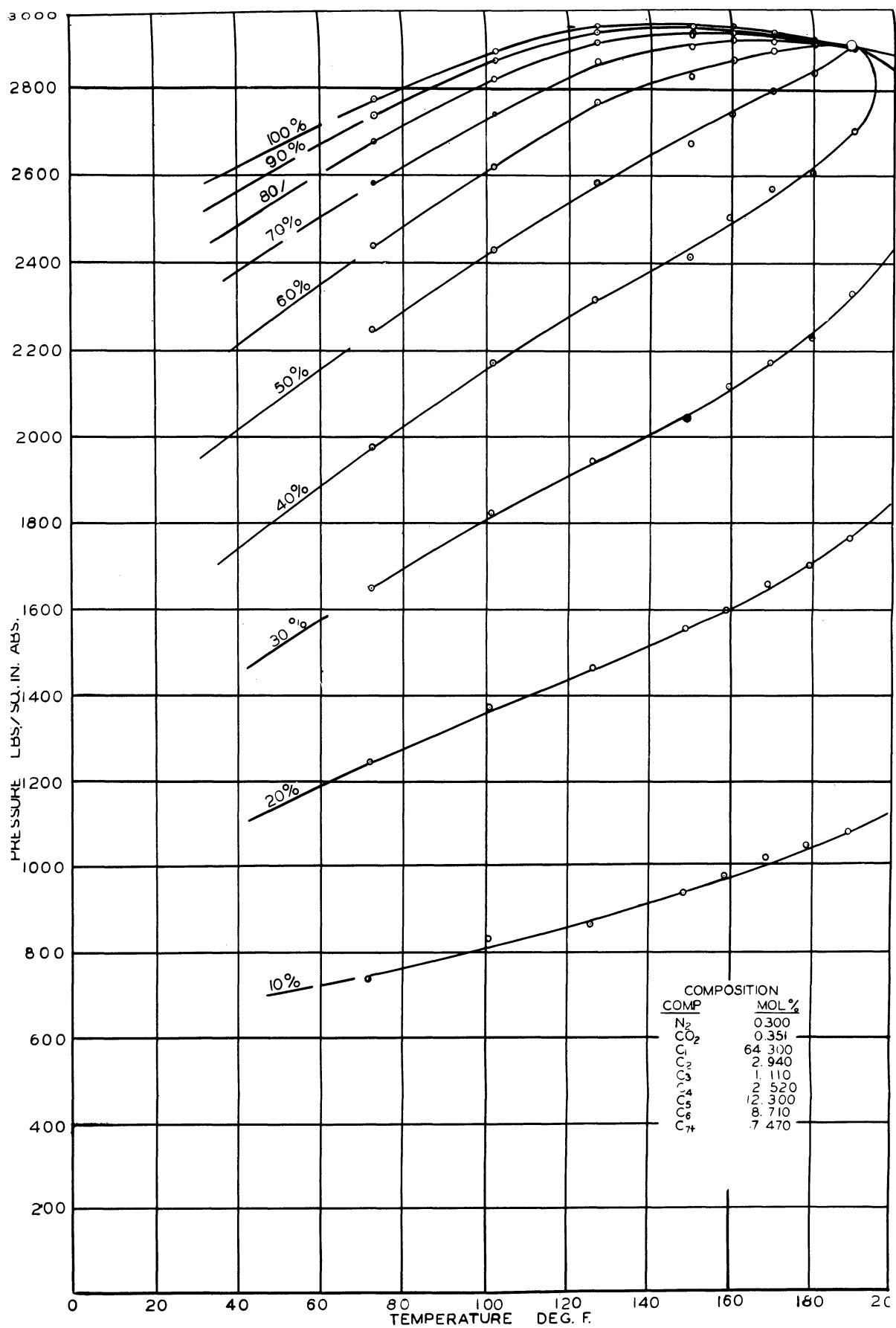


Fig. D-5. Phase diagram for mixture T-5.

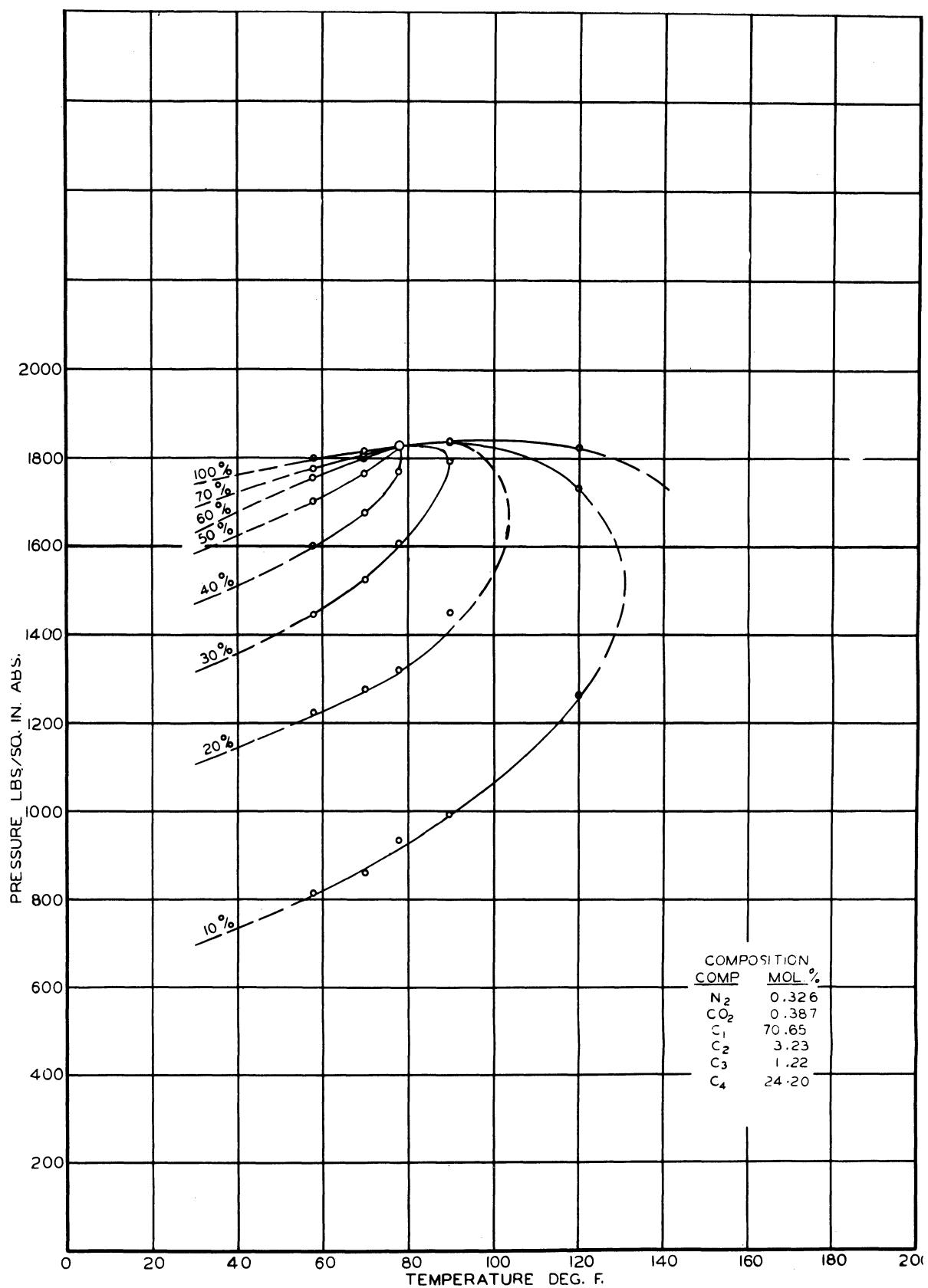


Fig. D-6. Phase diagram for mixture B-1.

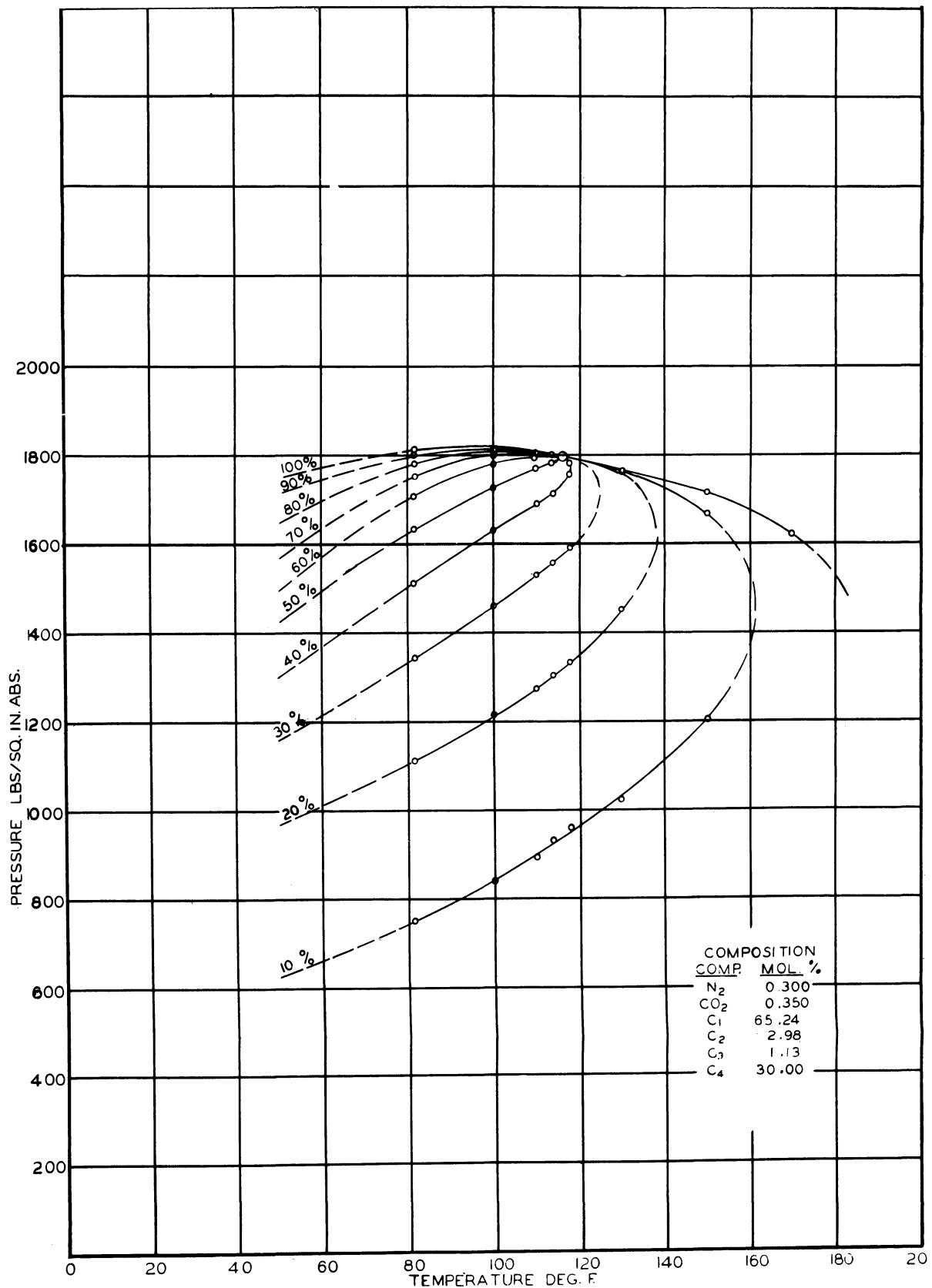


Fig. D-7. Phase diagram for mixture B-2.

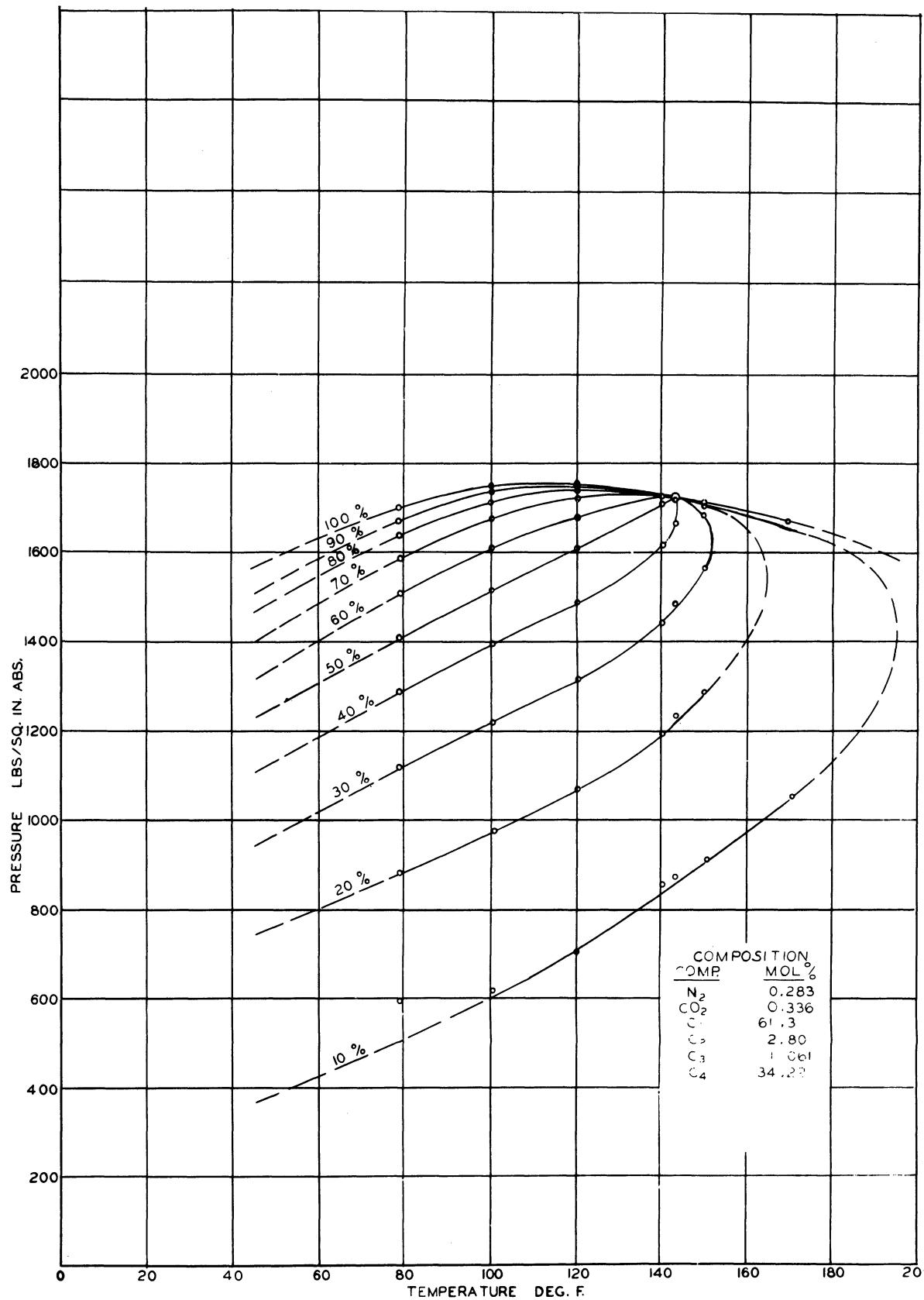


Fig. D-8. Phase diagram for mixture B-3.

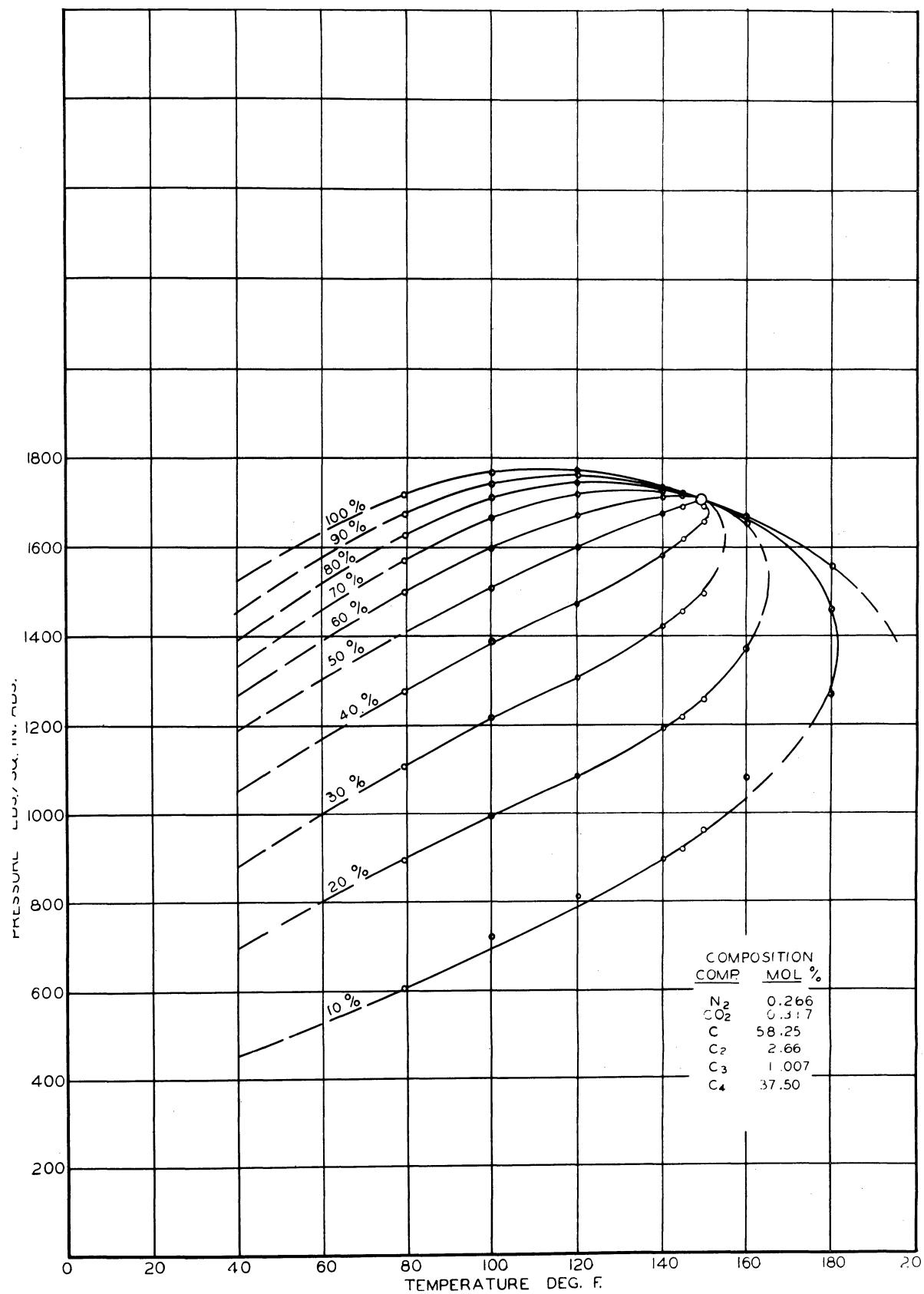


Fig. D-9. Phase diagram for mixture B-4.

APPENDIX E
CHARTS GIVING RESULTS OF ECONOMIC CALCULATIONS

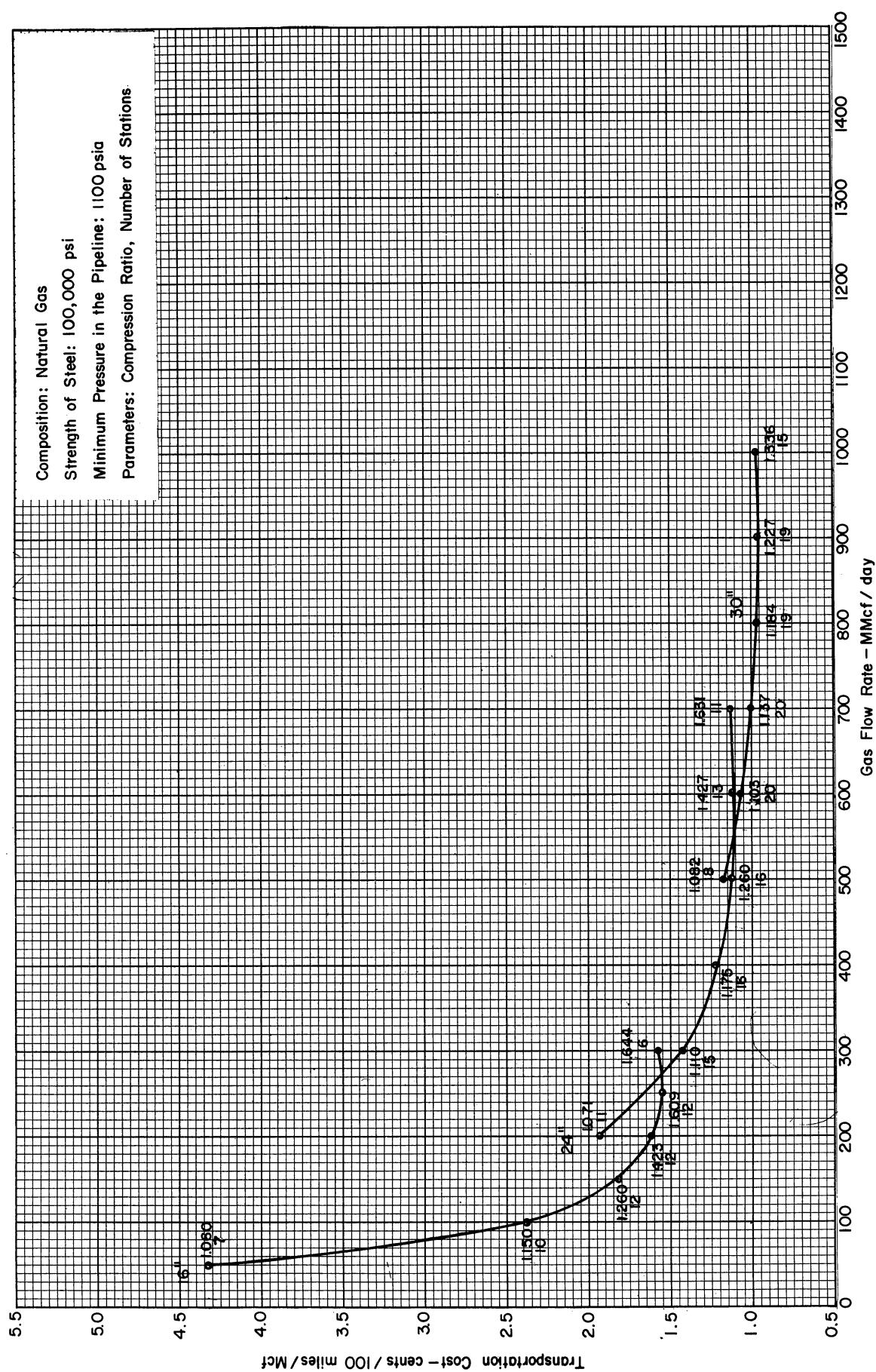


Figure 15E-1. Cost of Transportation versus Flow Rate

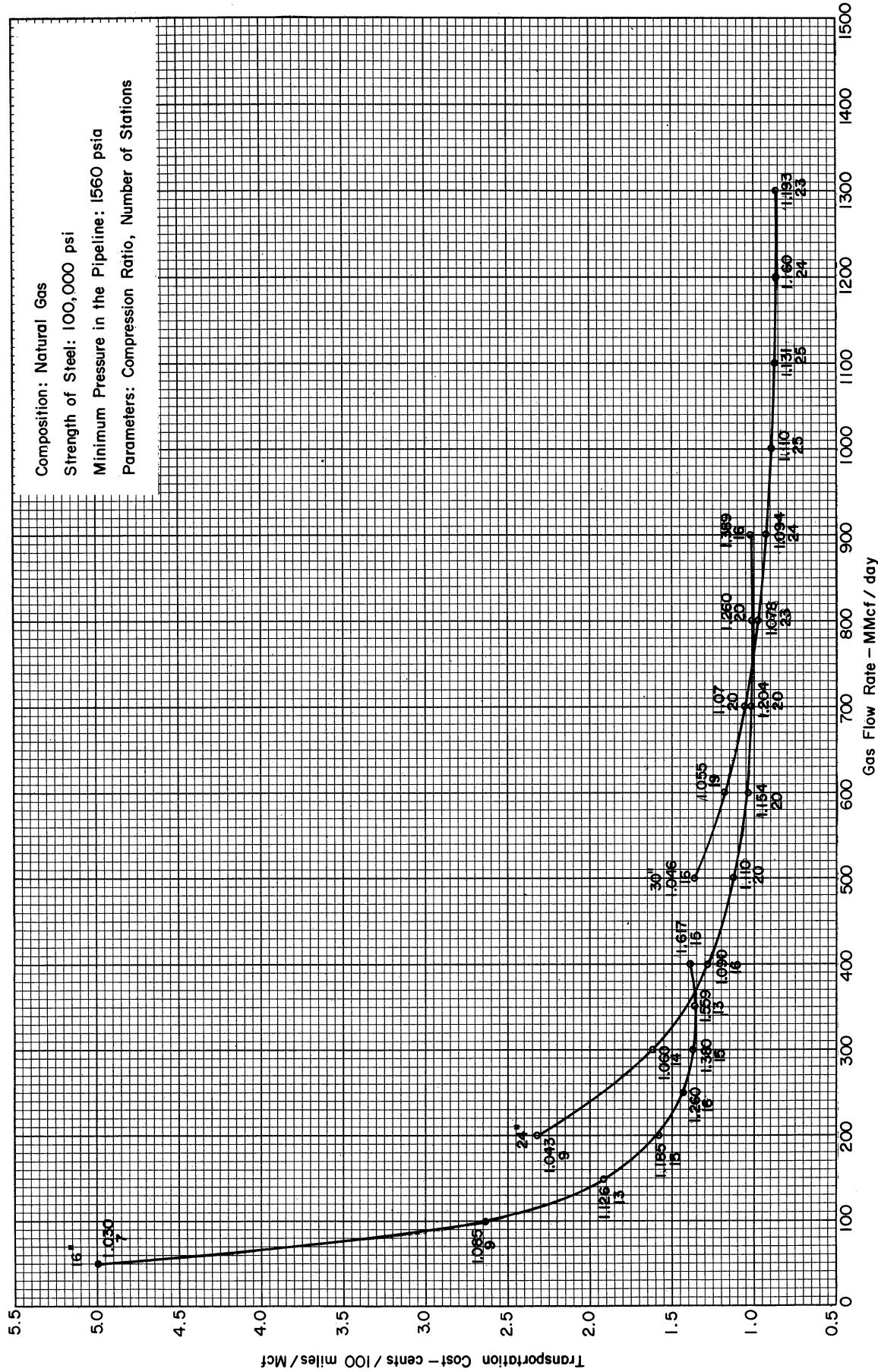


Figure 15E-2. Cost of Transportation versus Flow Rate

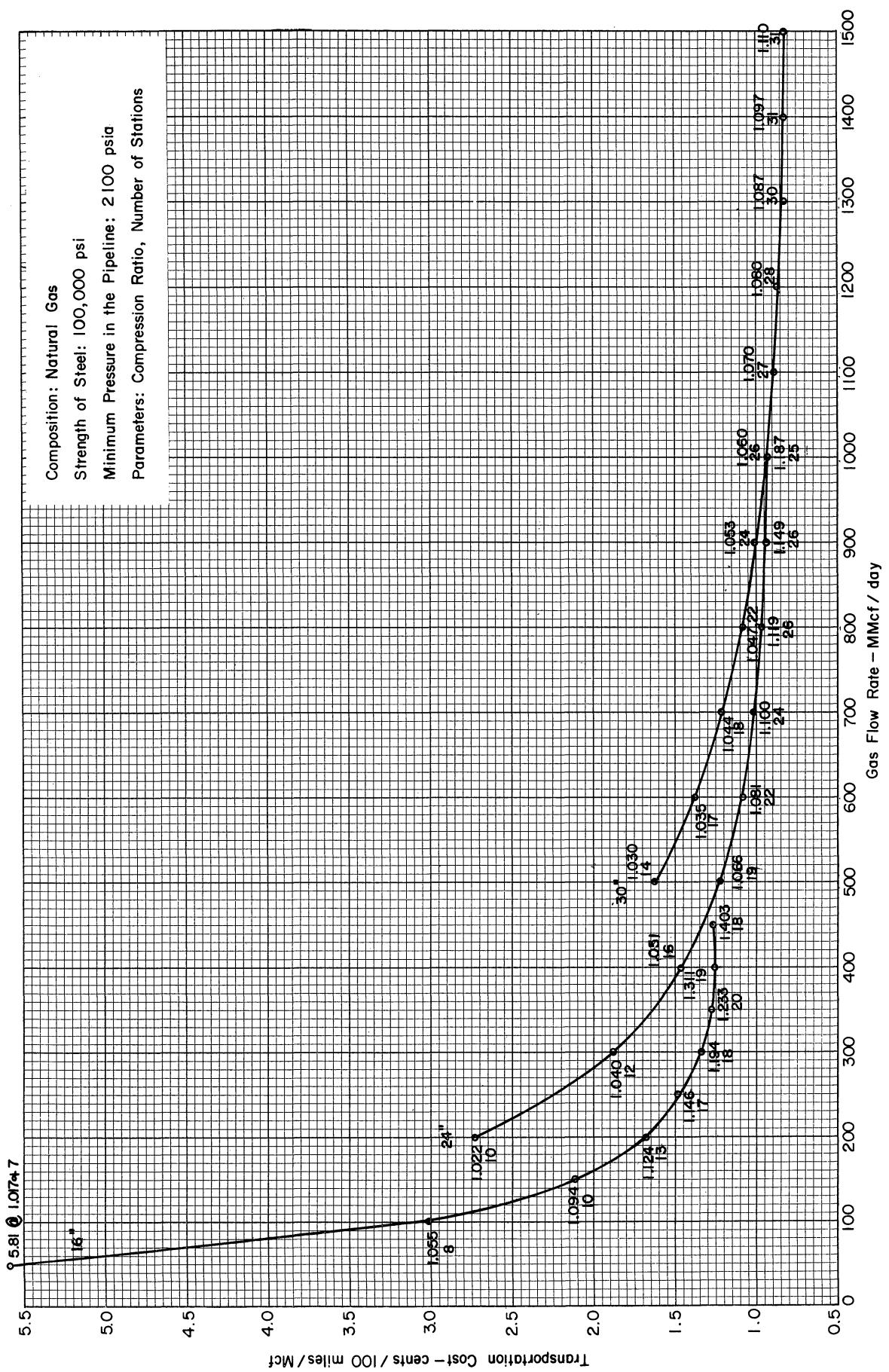


Figure 15E-3. Cost of Transportation versus Flow Rate

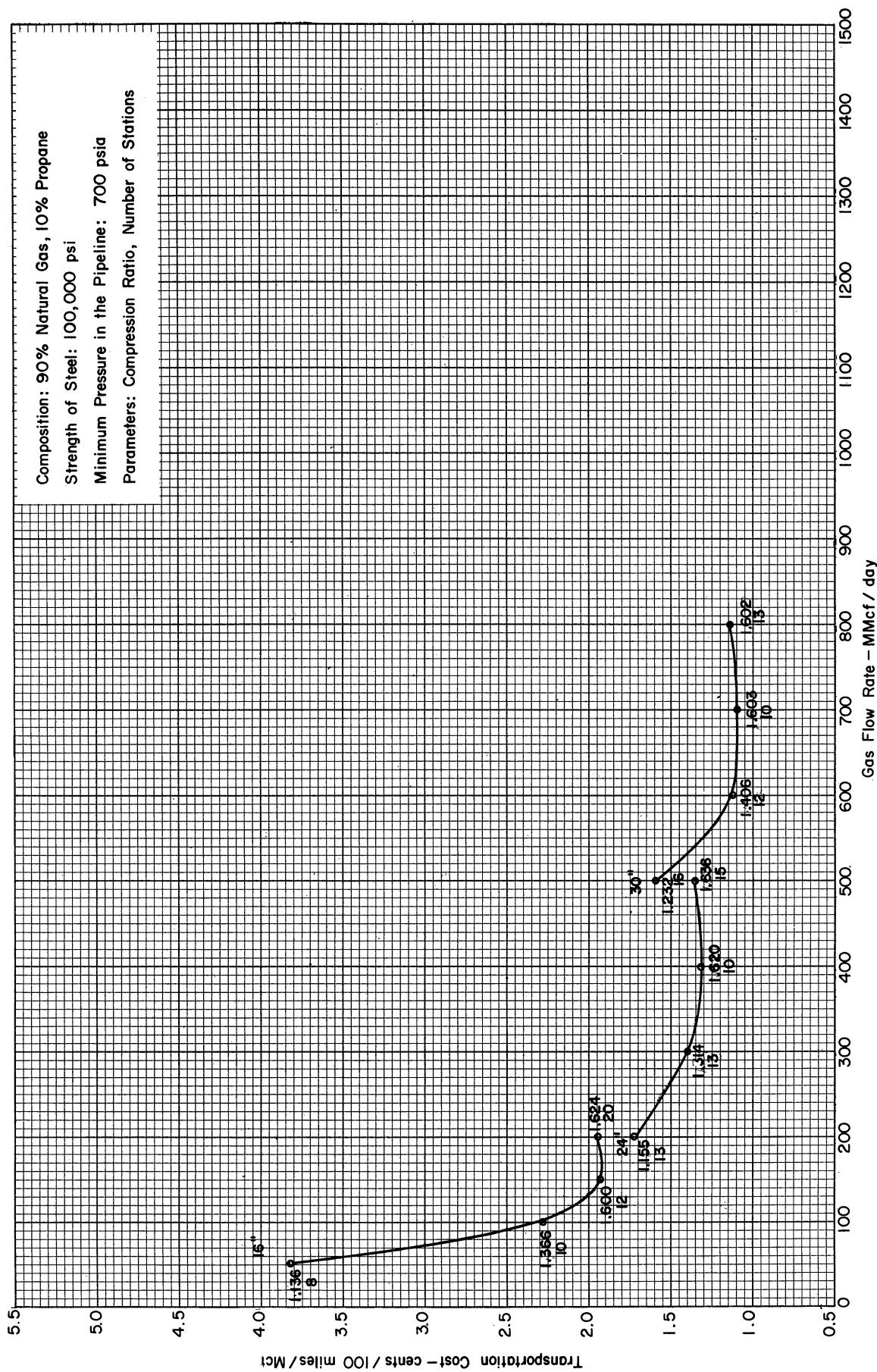


Figure 15E-4. Cost of Transportation versus Flow Rate

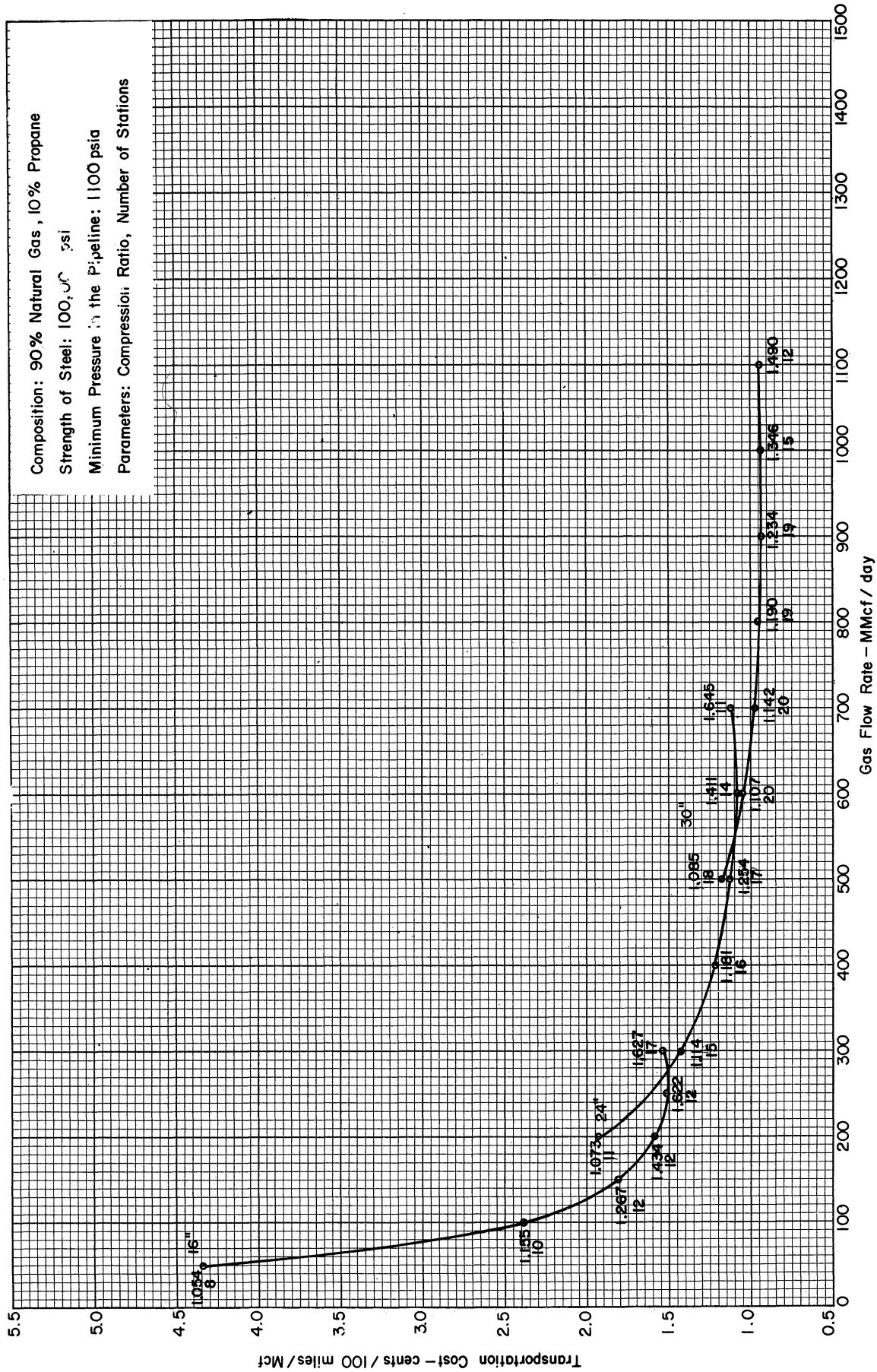


Figure 15E-5. Cost of Transportation versus Flow Rate

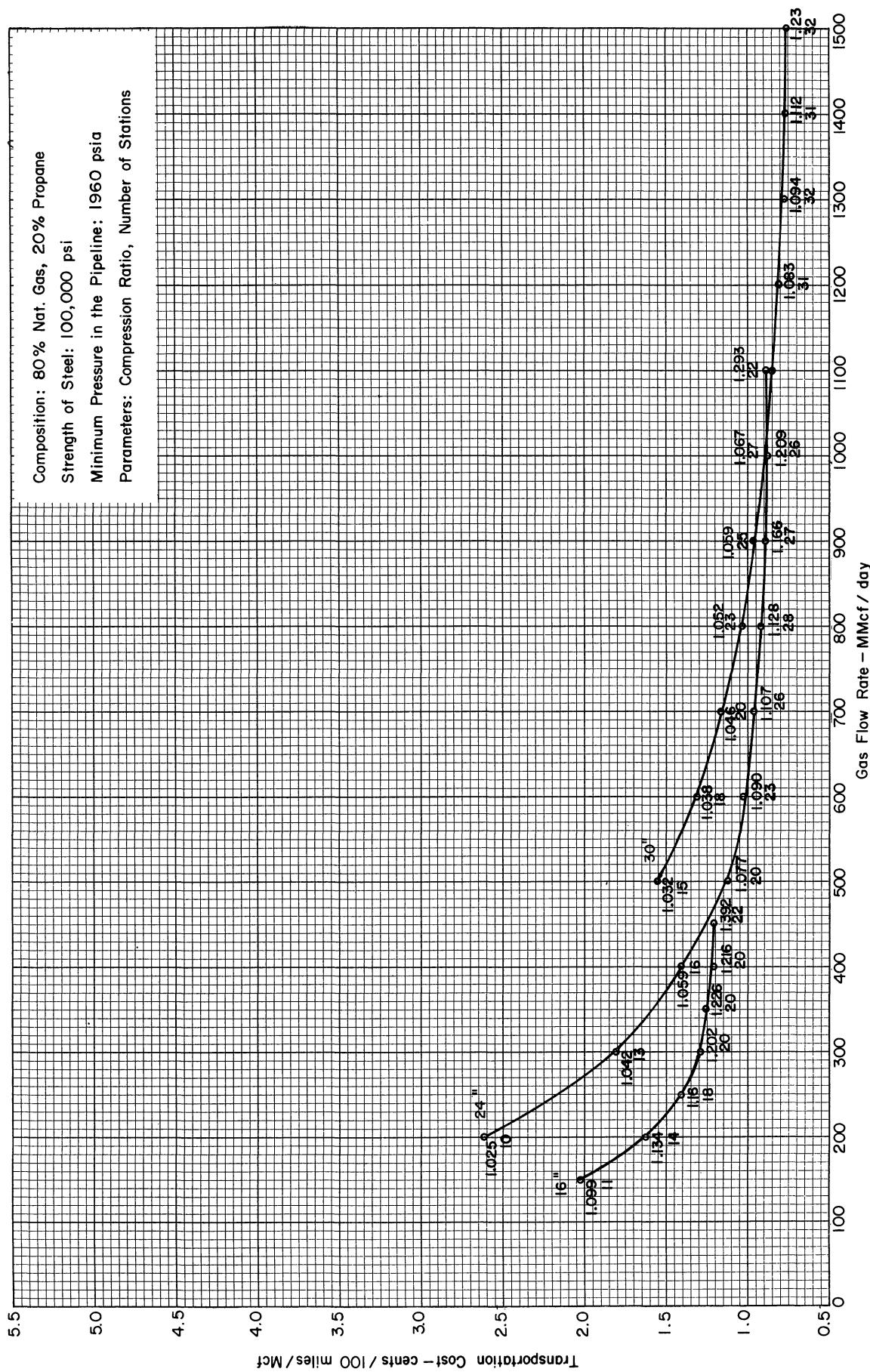


Figure 15E-6. Cost of Transportation versus Flow Rate

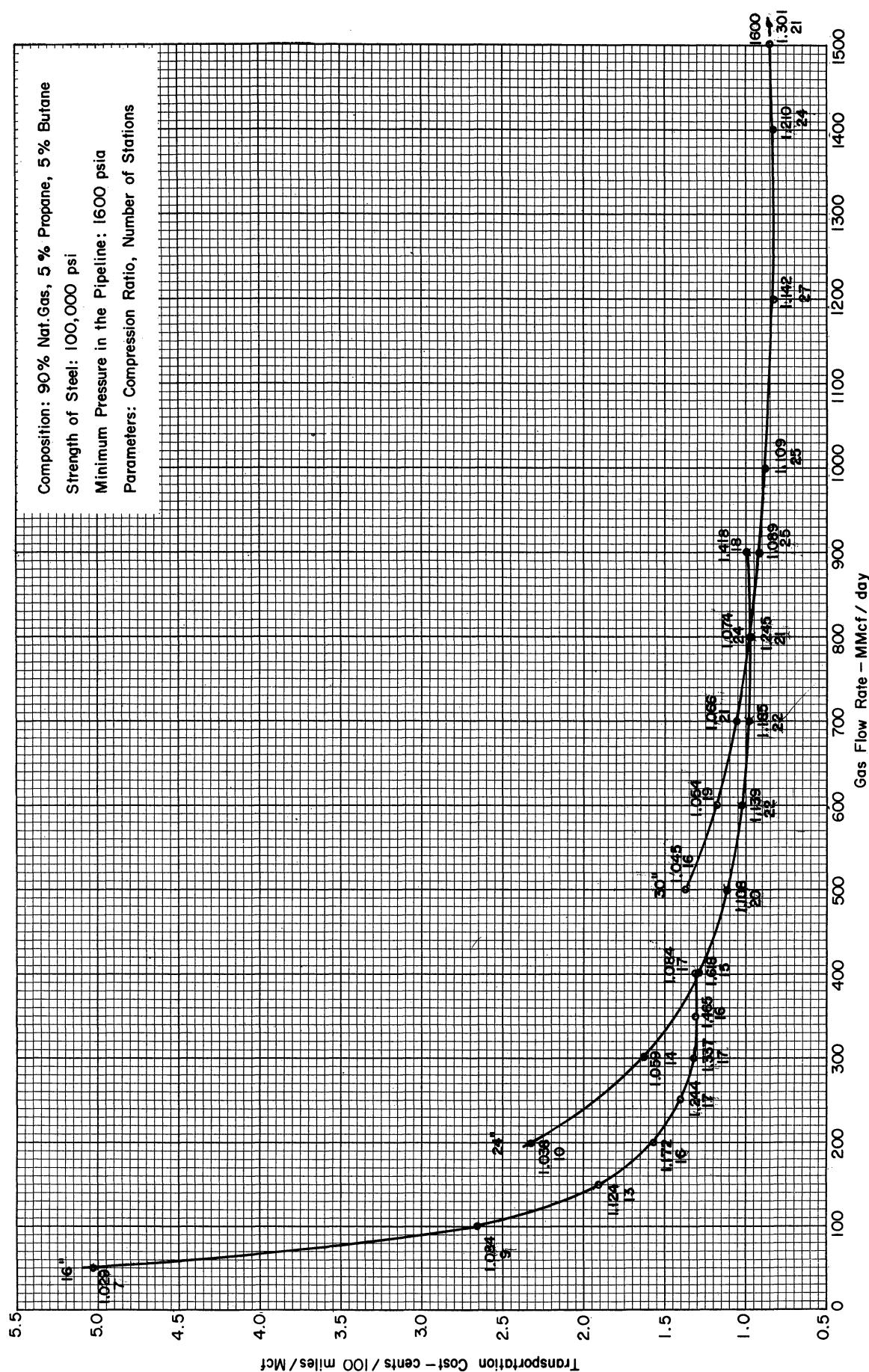


Figure 15E-7. Cost of Transportation versus Flow Rate

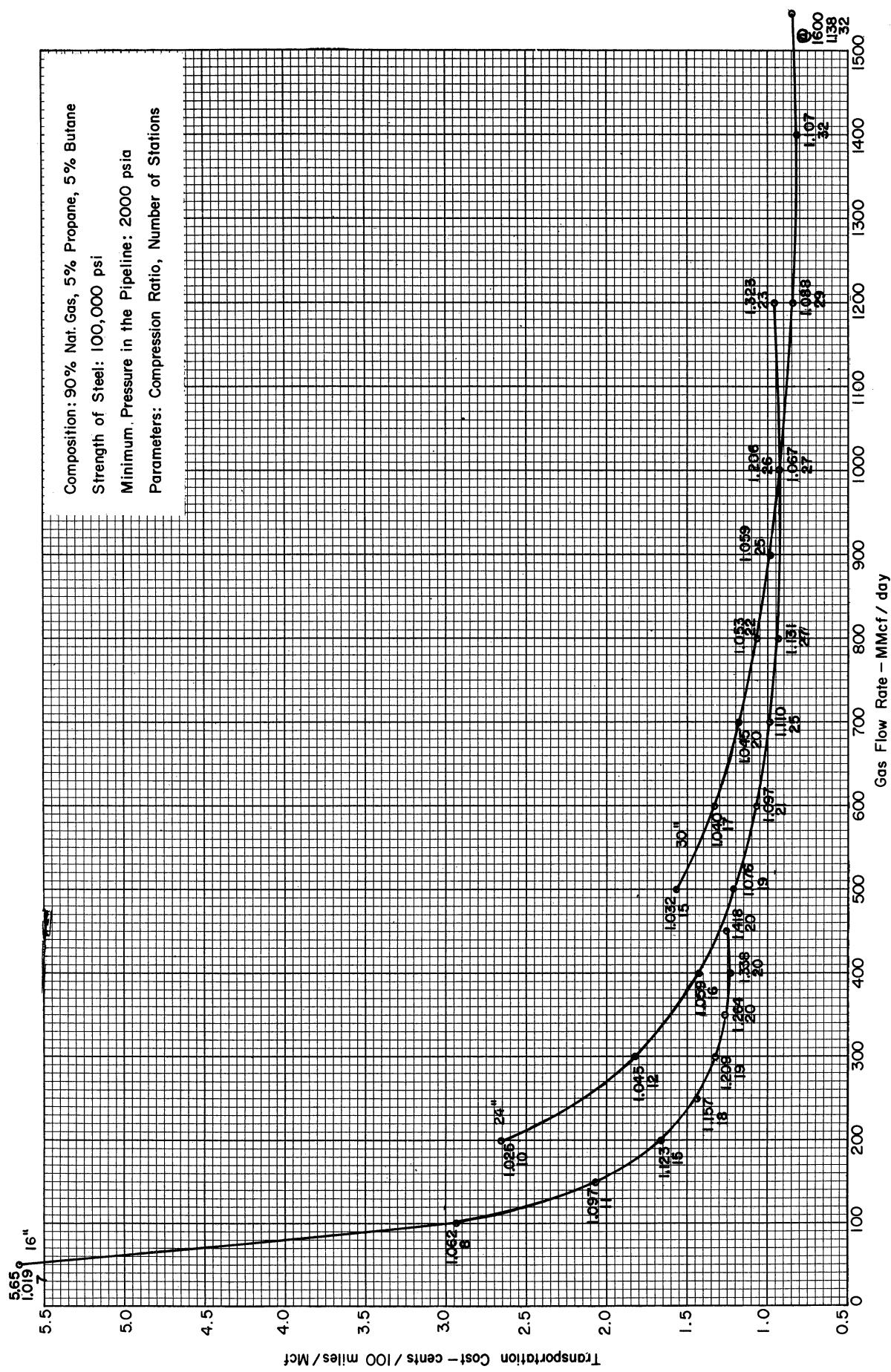


Figure 15E-8. Cost of Transportation versus Flow Rate

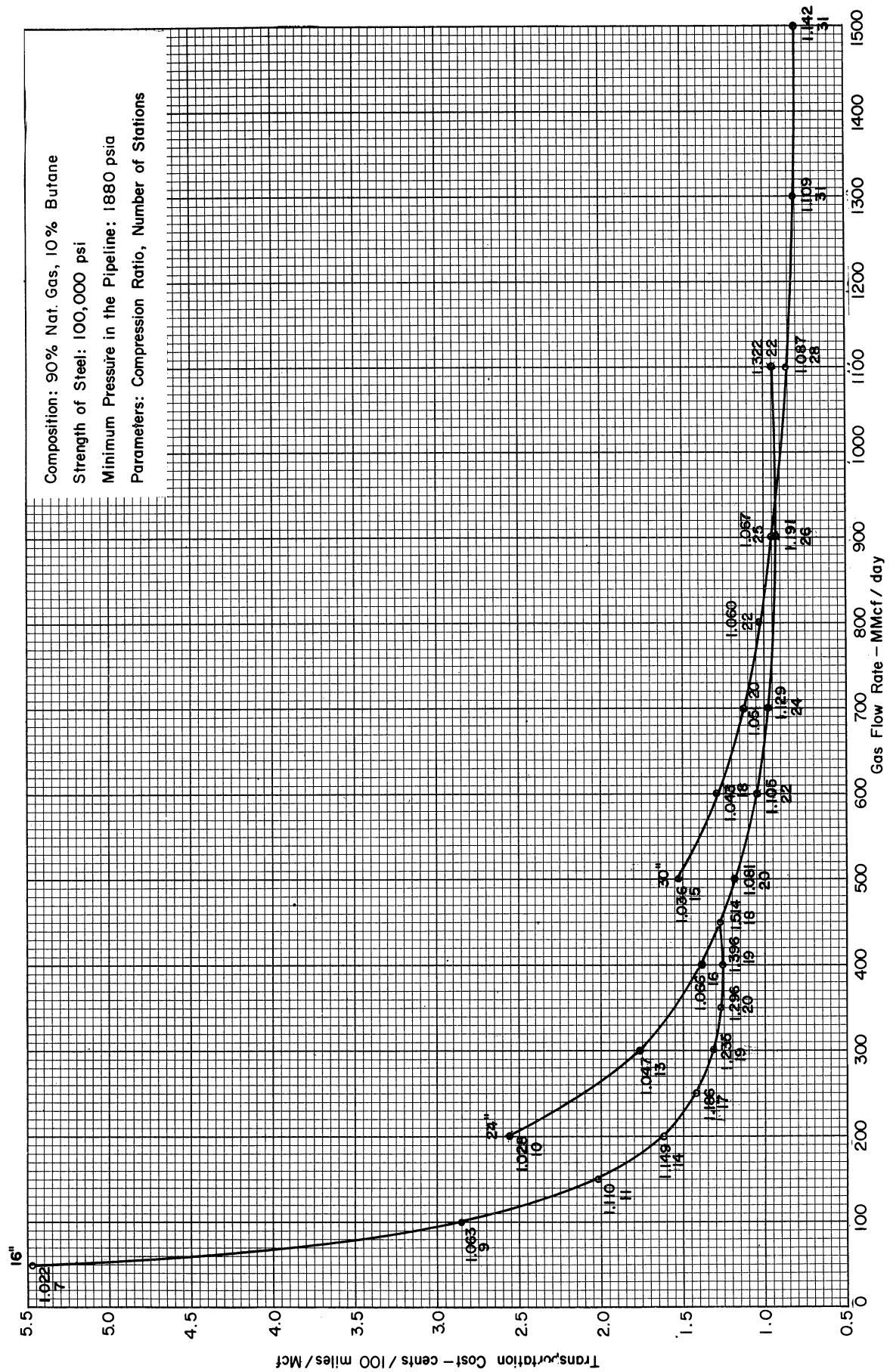


Figure 15E-9. Cost of Transportation versus Flow Rate

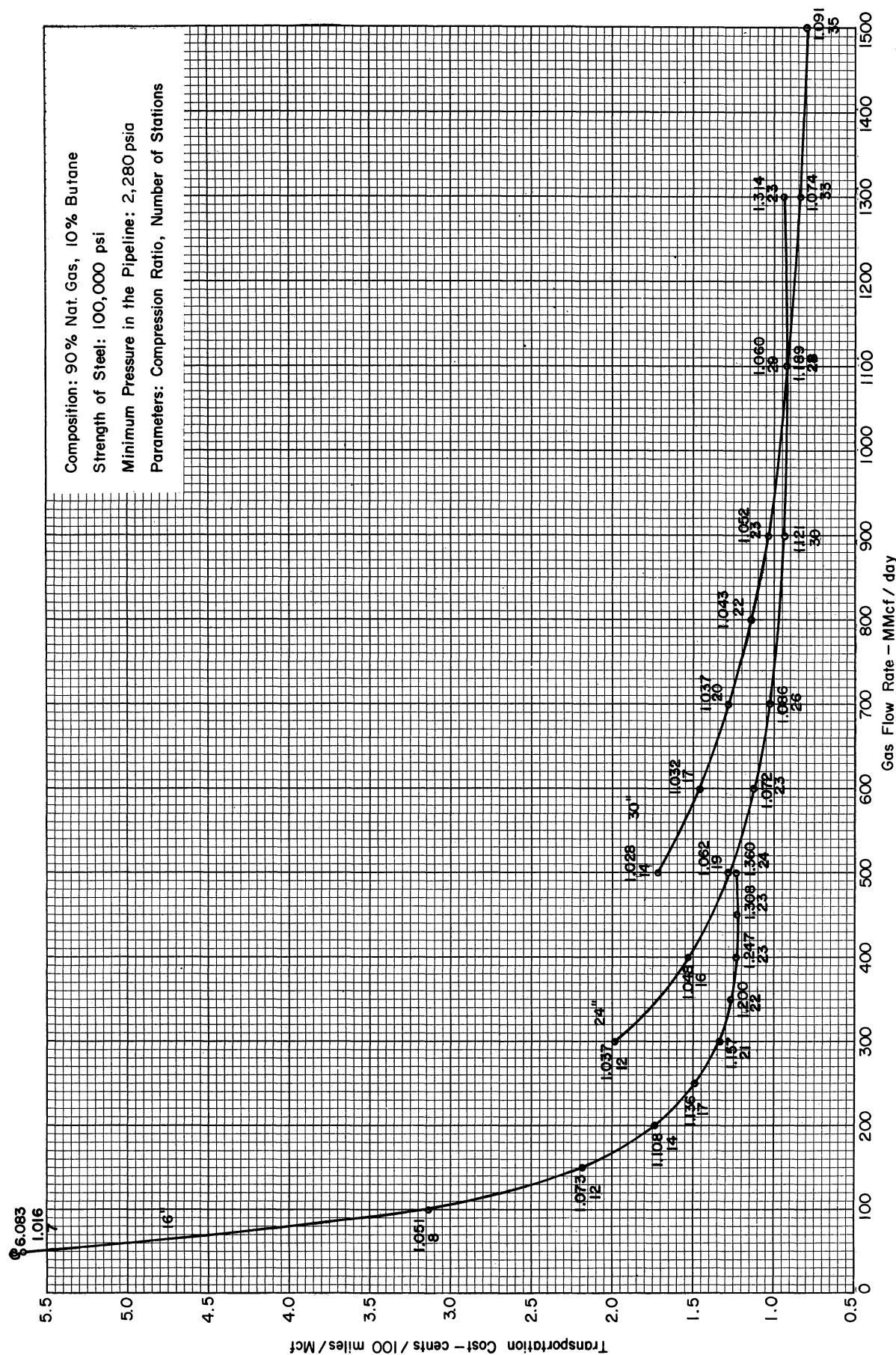


Figure 15E-10. Cost of Transportation versus Flow Rate

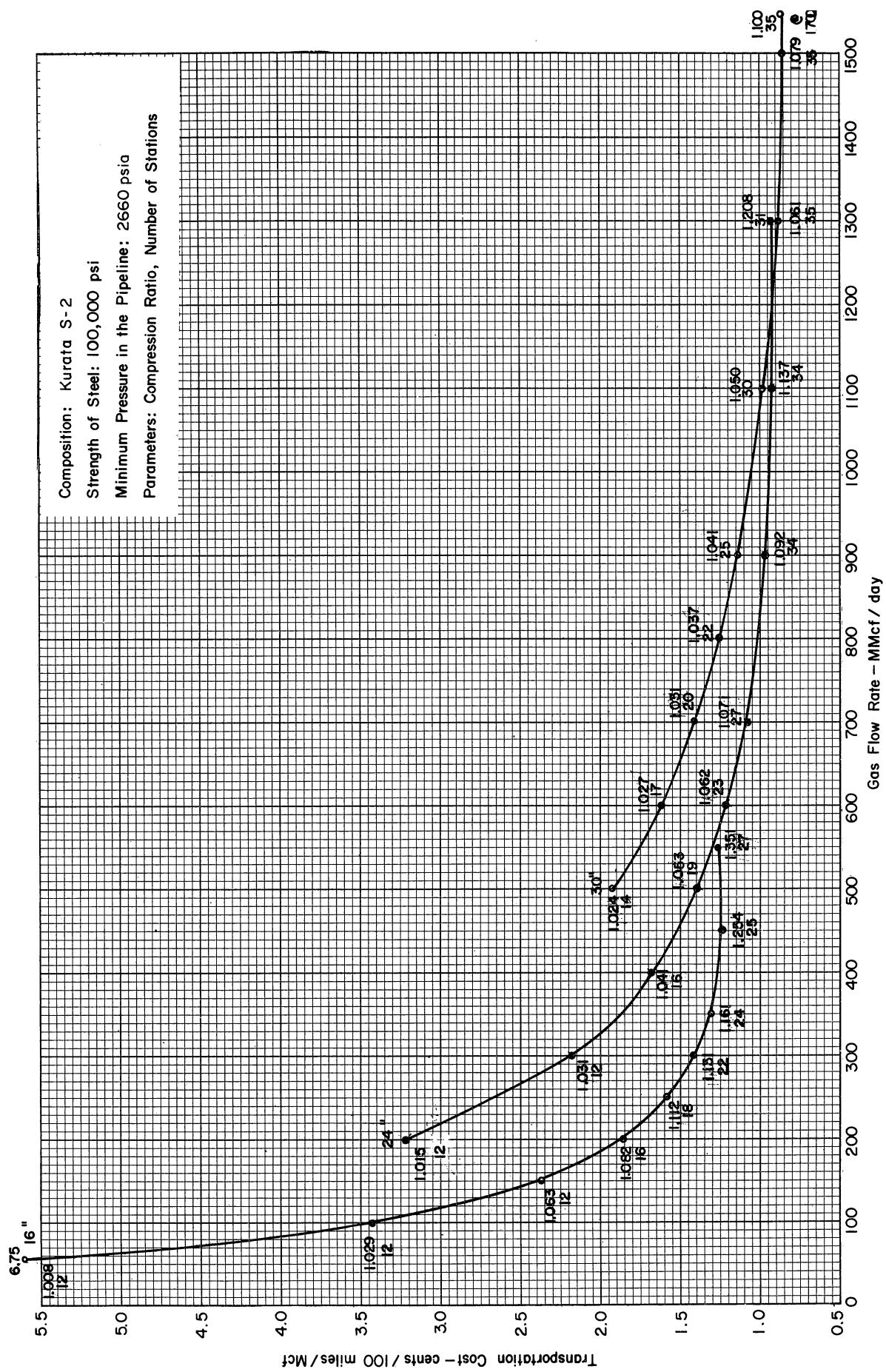


Figure 15E-11. Cost of Transportation versus Flow Rate

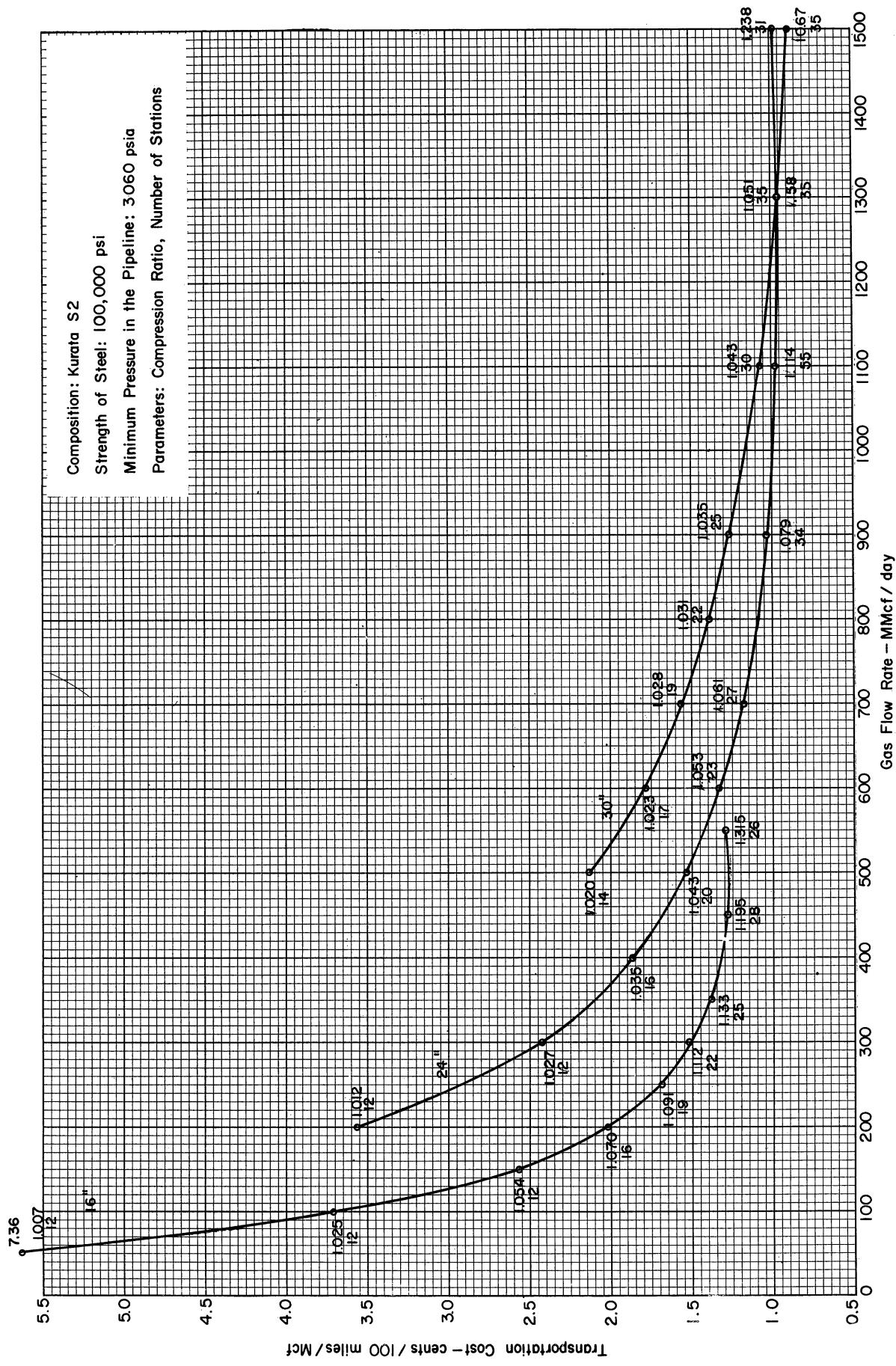


Figure 15E-12. Cost of Transportation versus Flow Rate

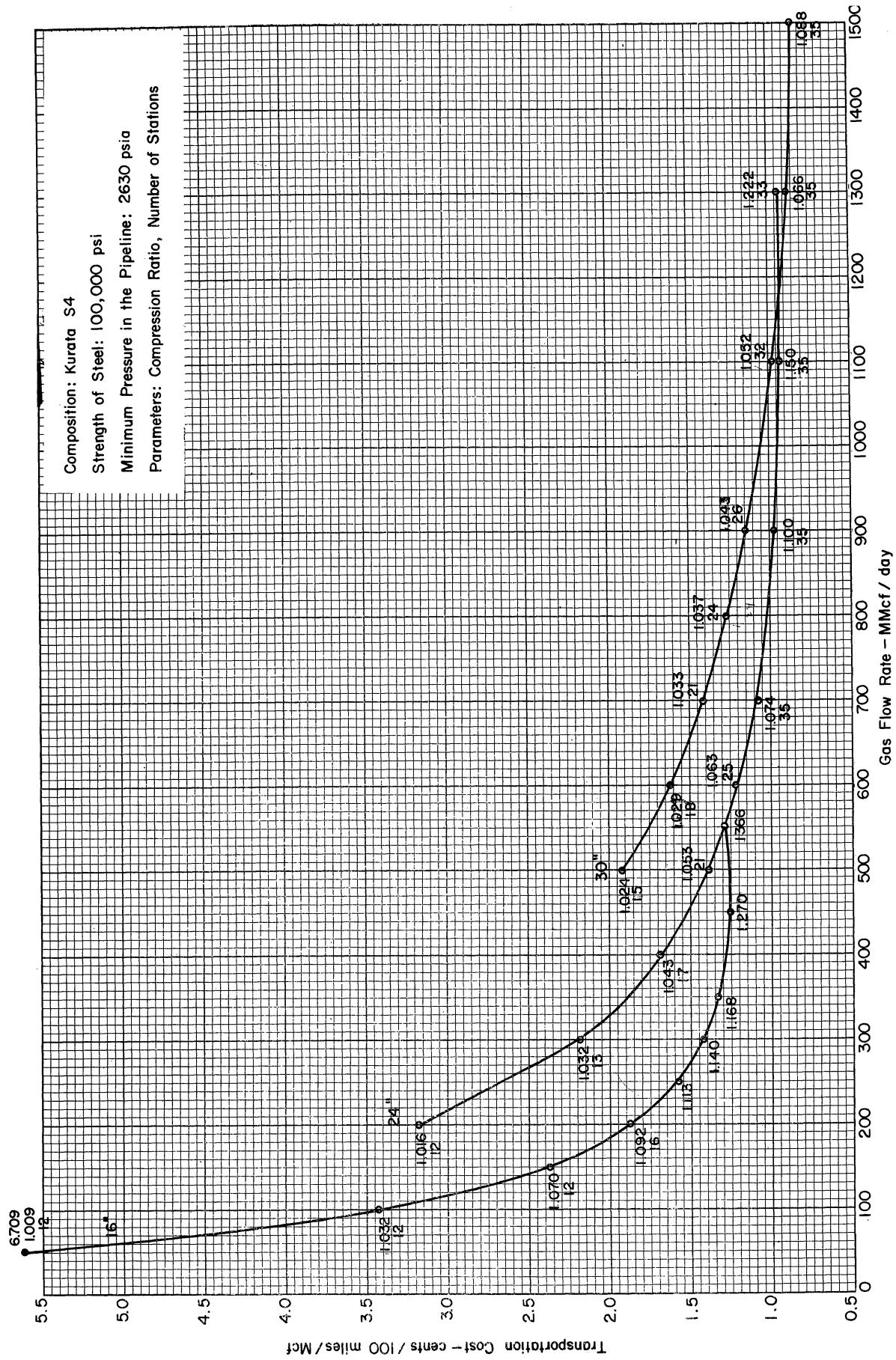


Figure 15E-13. Cost of Transportation versus Flow Rate

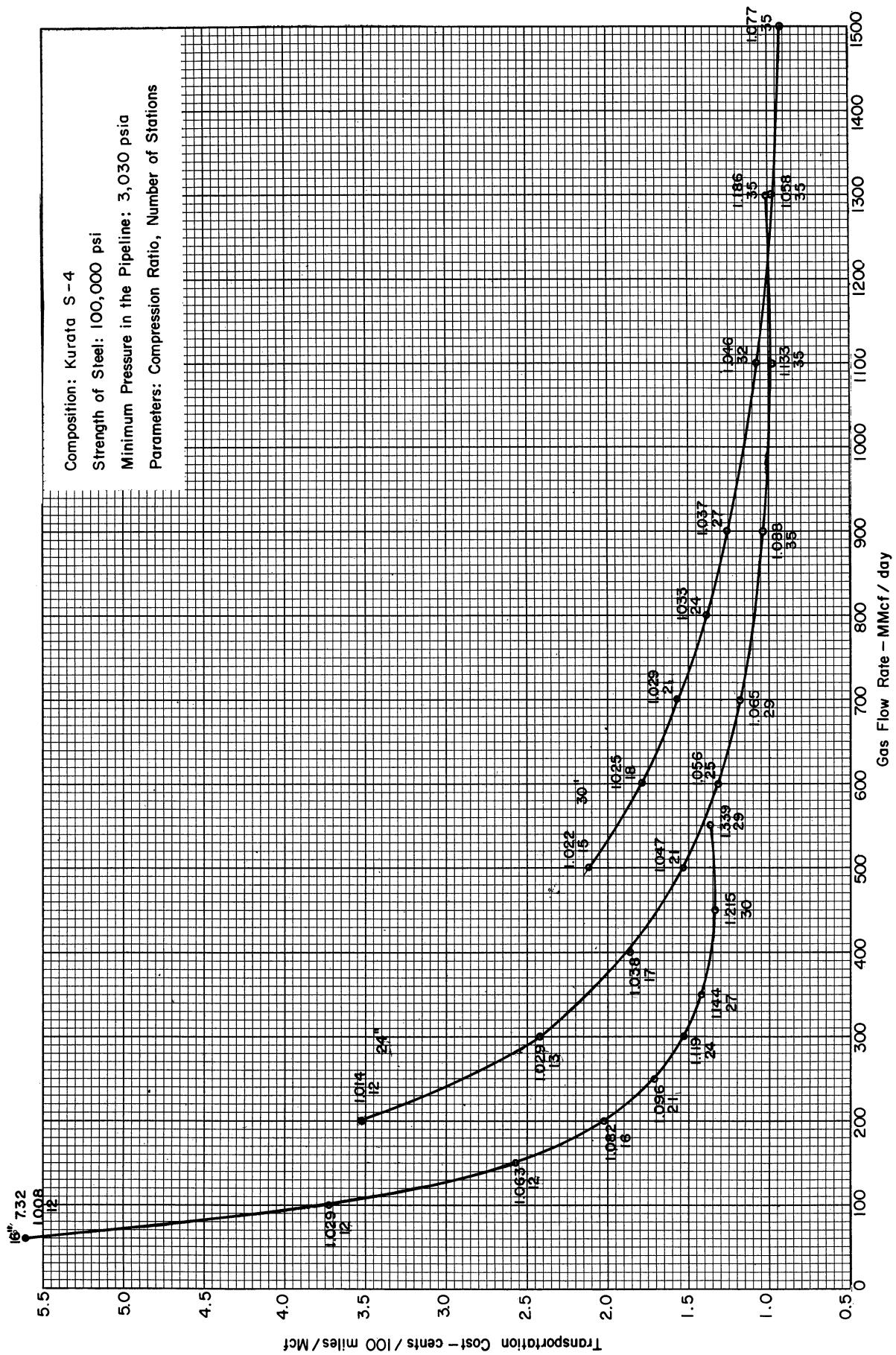


Figure 15E-14. Cost of Transportation versus Flow Rate

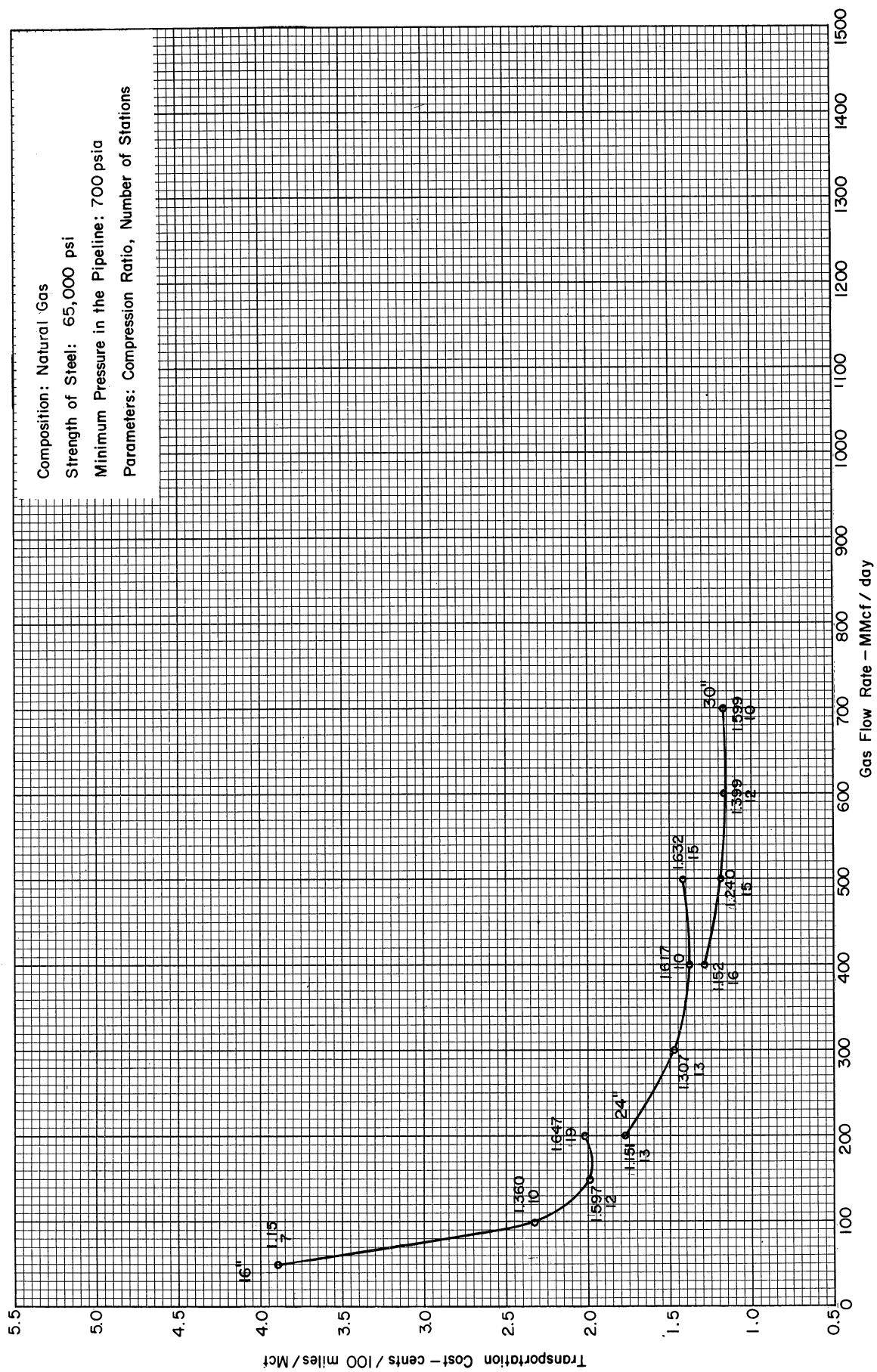


Figure 15E-15. Cost of Transportation versus Flow Rate

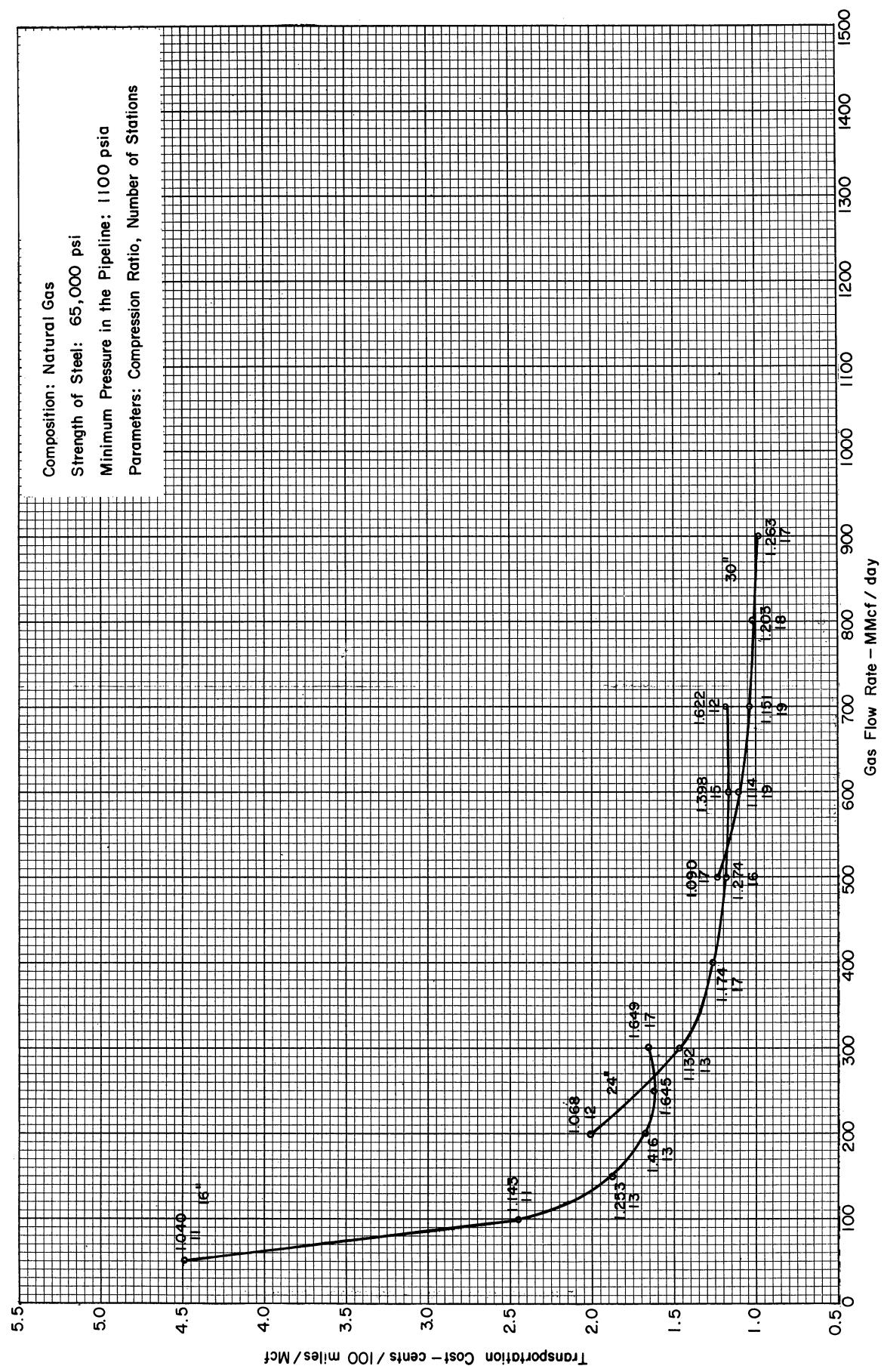


Figure 15E-16. Cost of Transportation versus Flow Rate

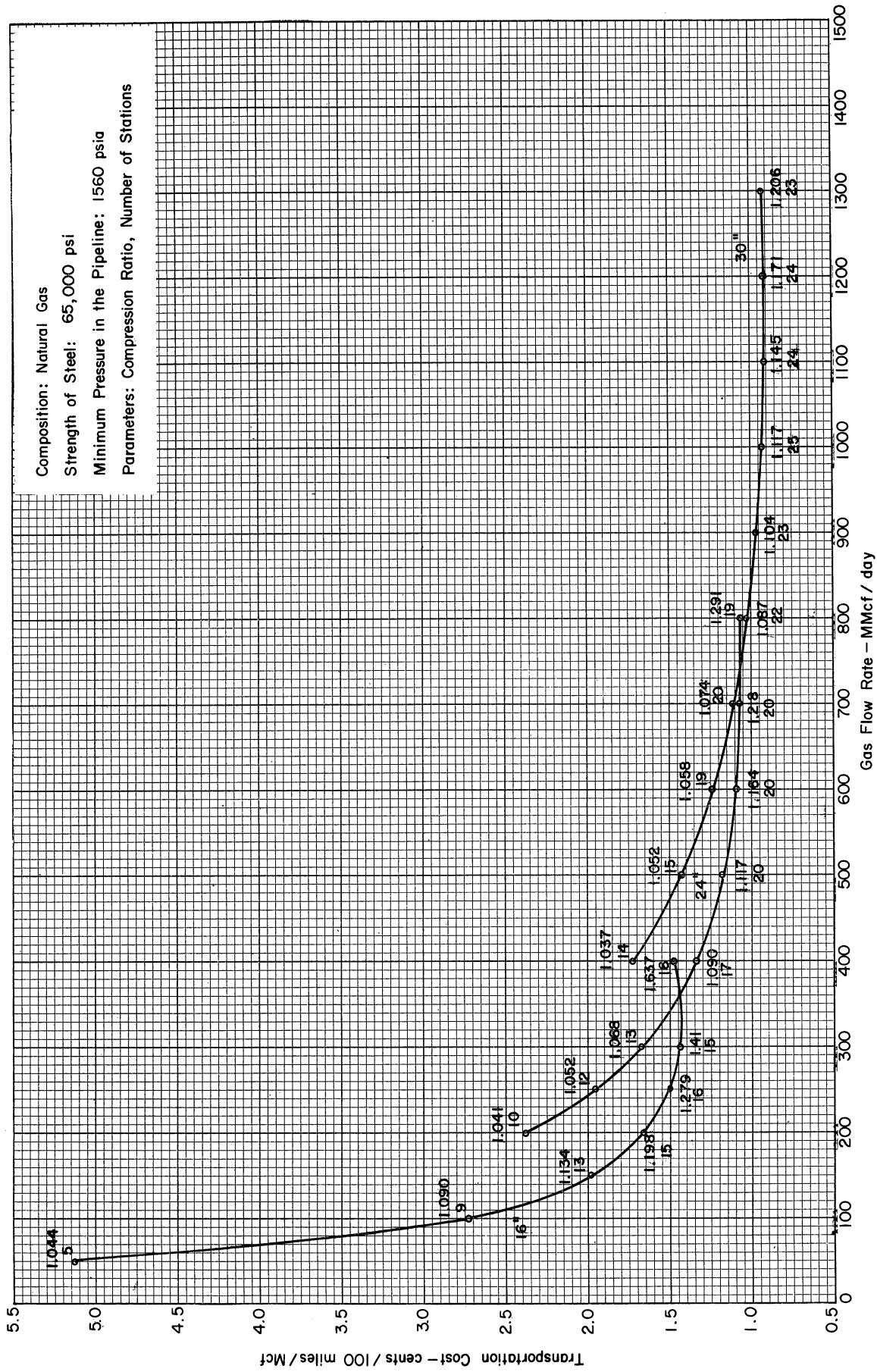


Figure 15E-17. Cost of Transportation versus Flow Rate

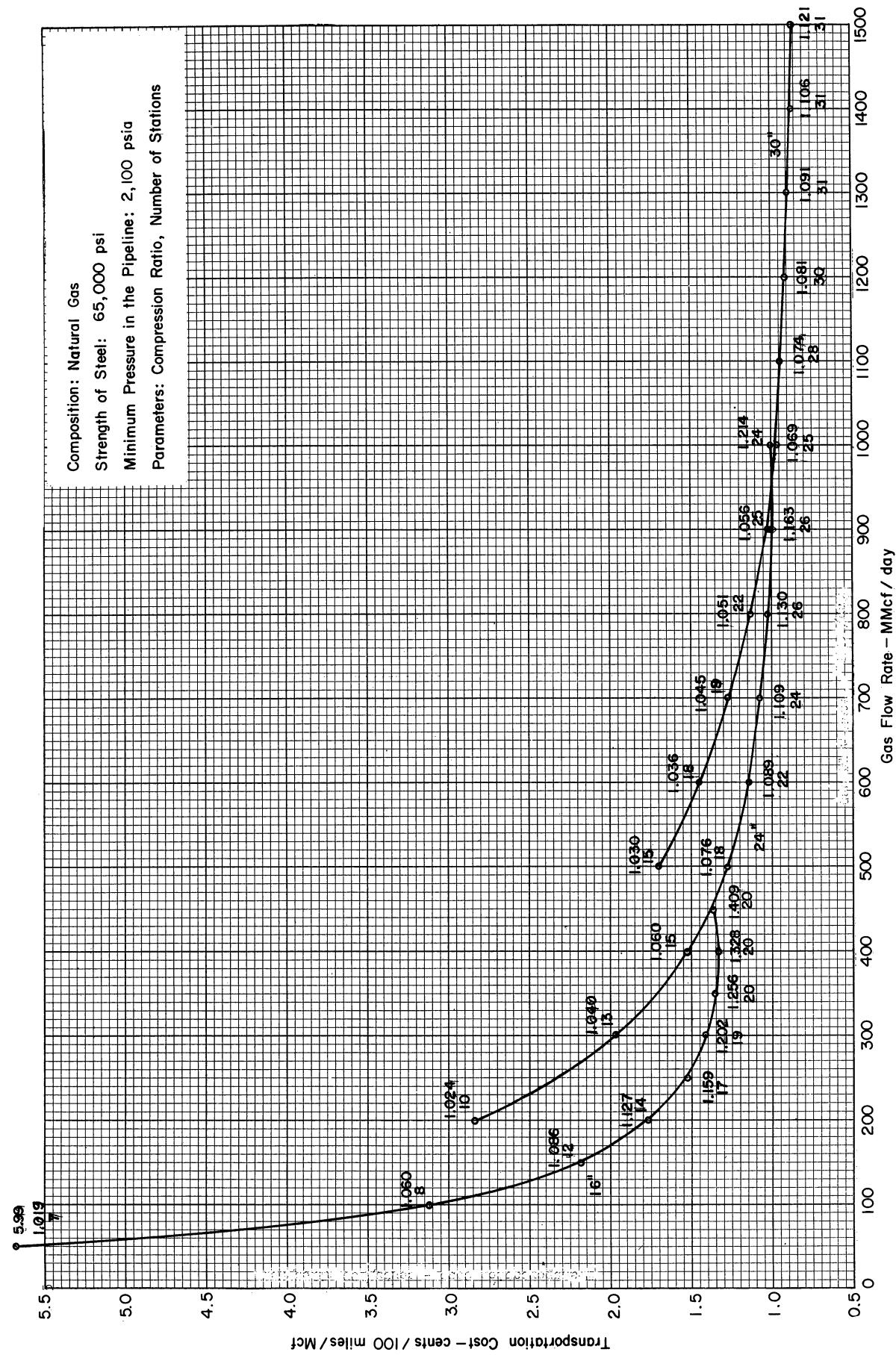


Figure 15E-18. Cost of Transportation versus Flow Rate

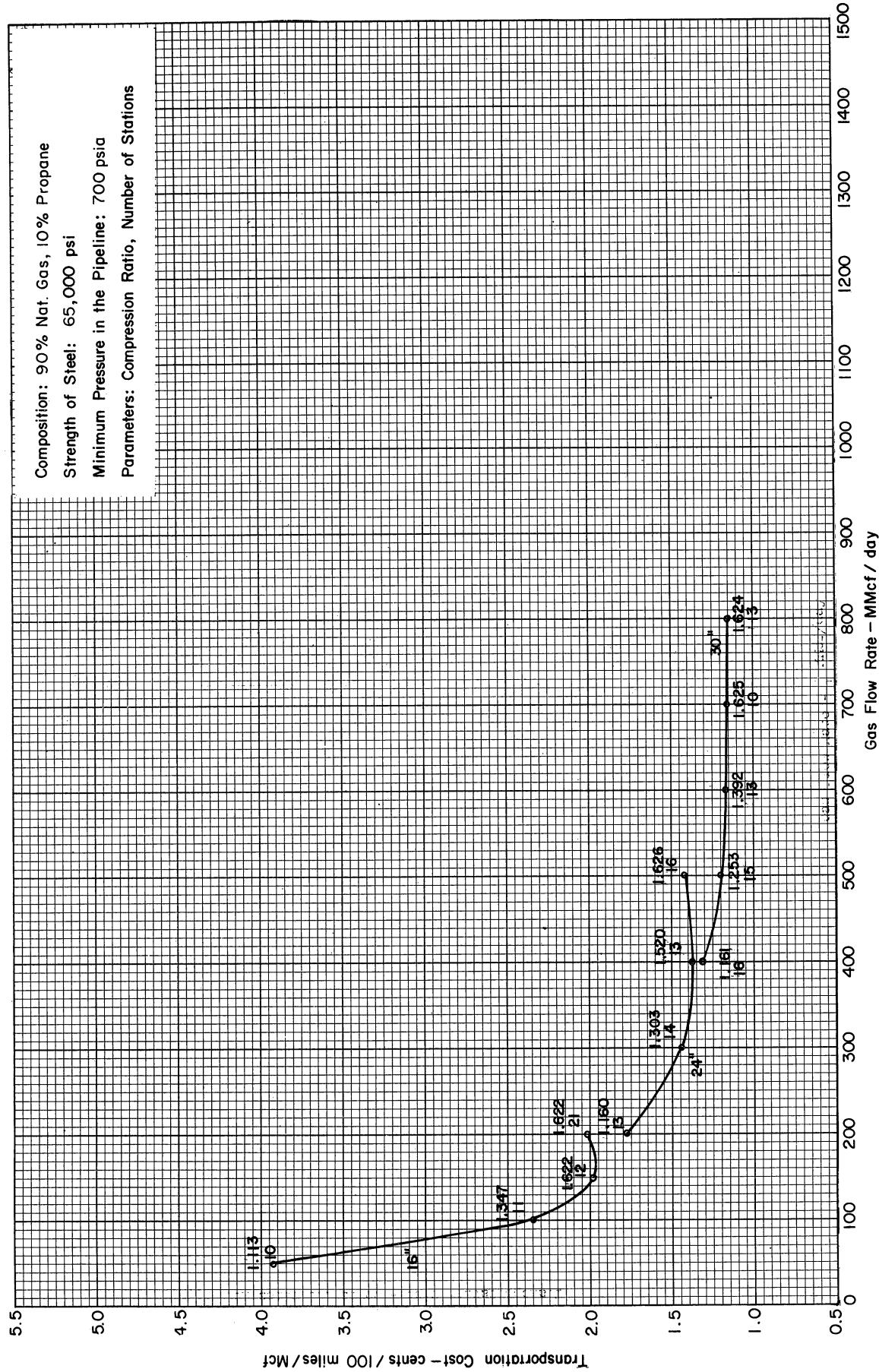


Figure 15E-19. Cost of Transportation versus Flow Rate

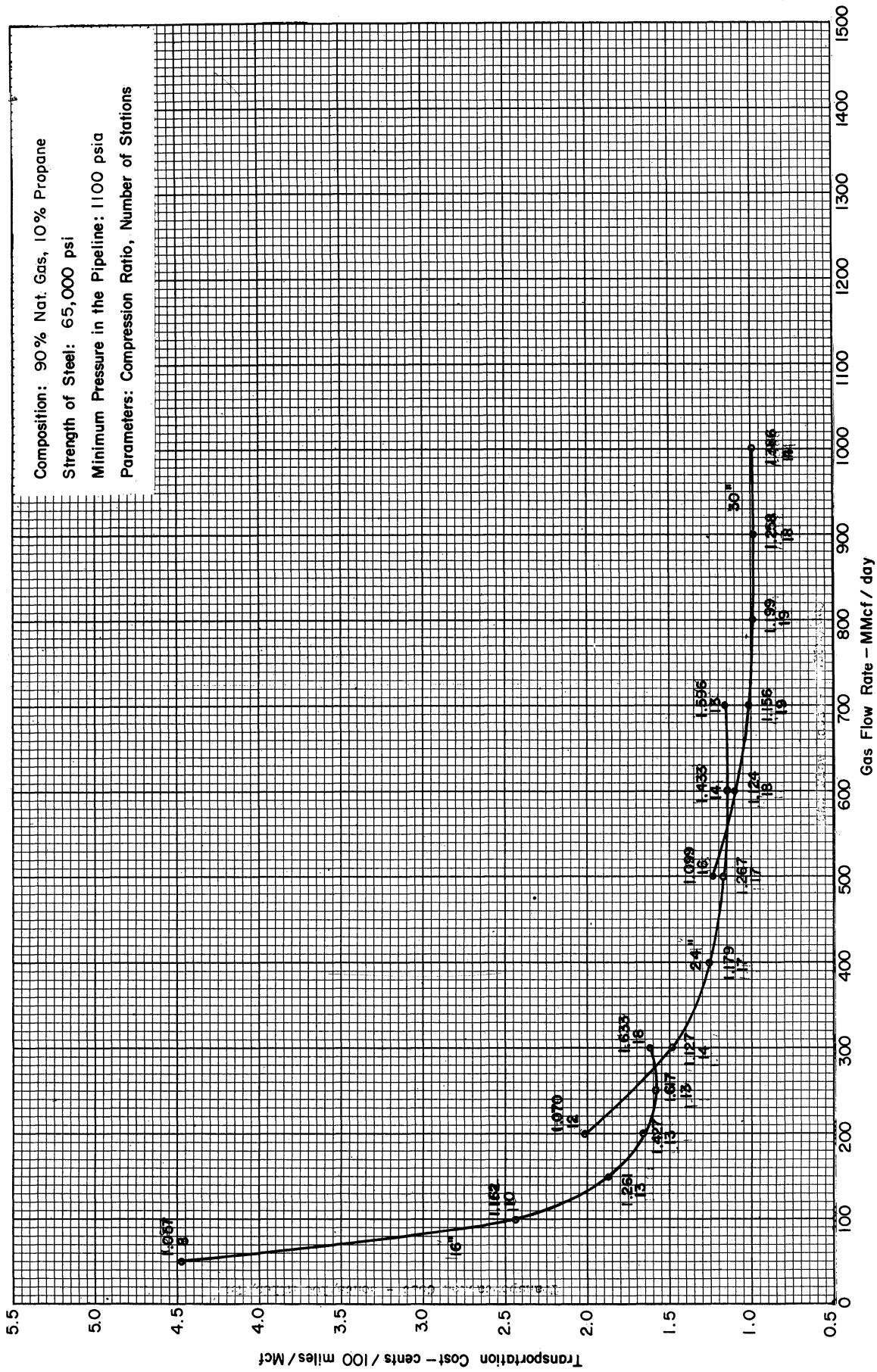


Figure 15E-20. Cost of Transportation versus Flow Rate

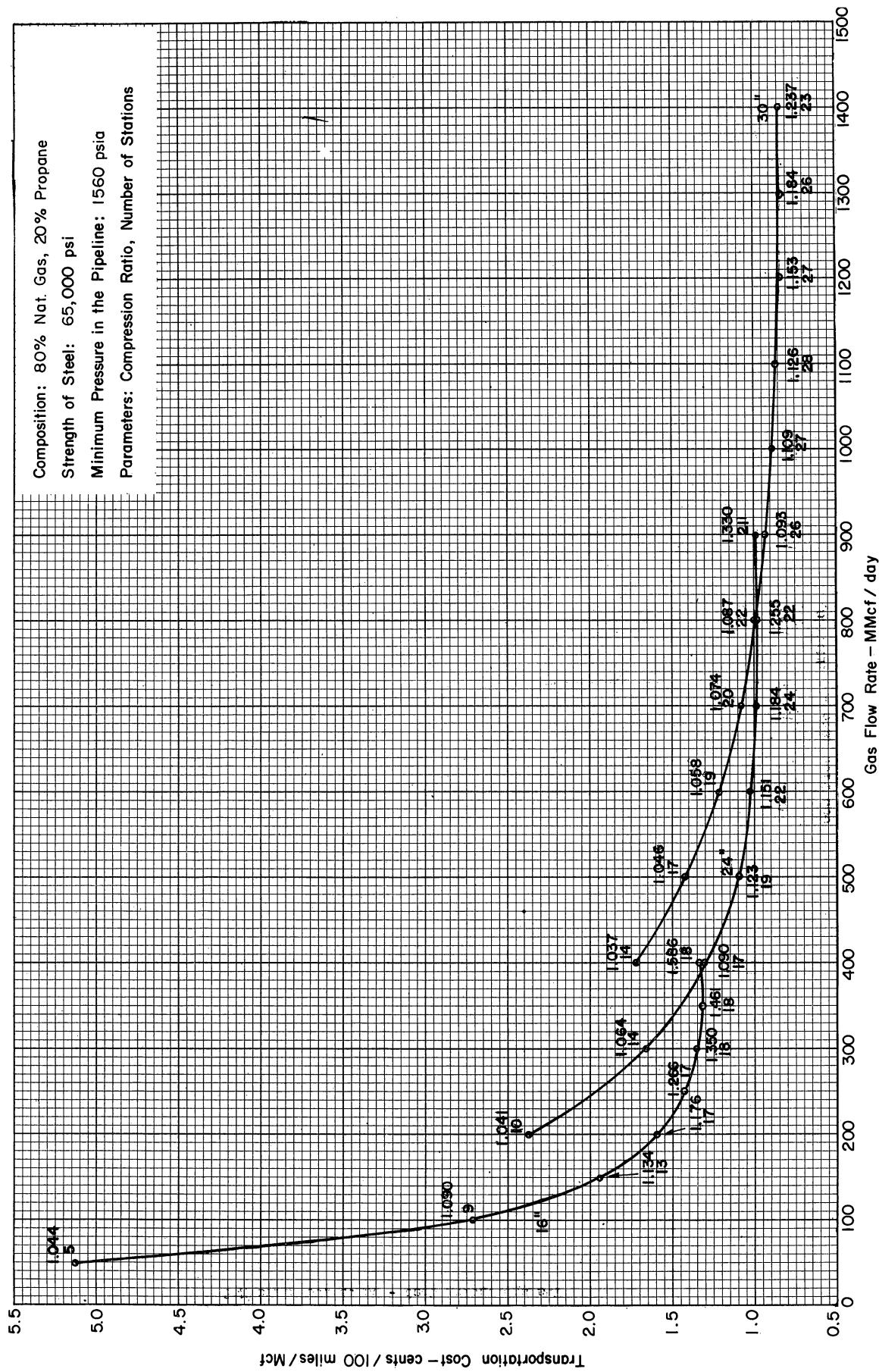


Figure 15E-21. Cost of Transportation versus Flow Rate

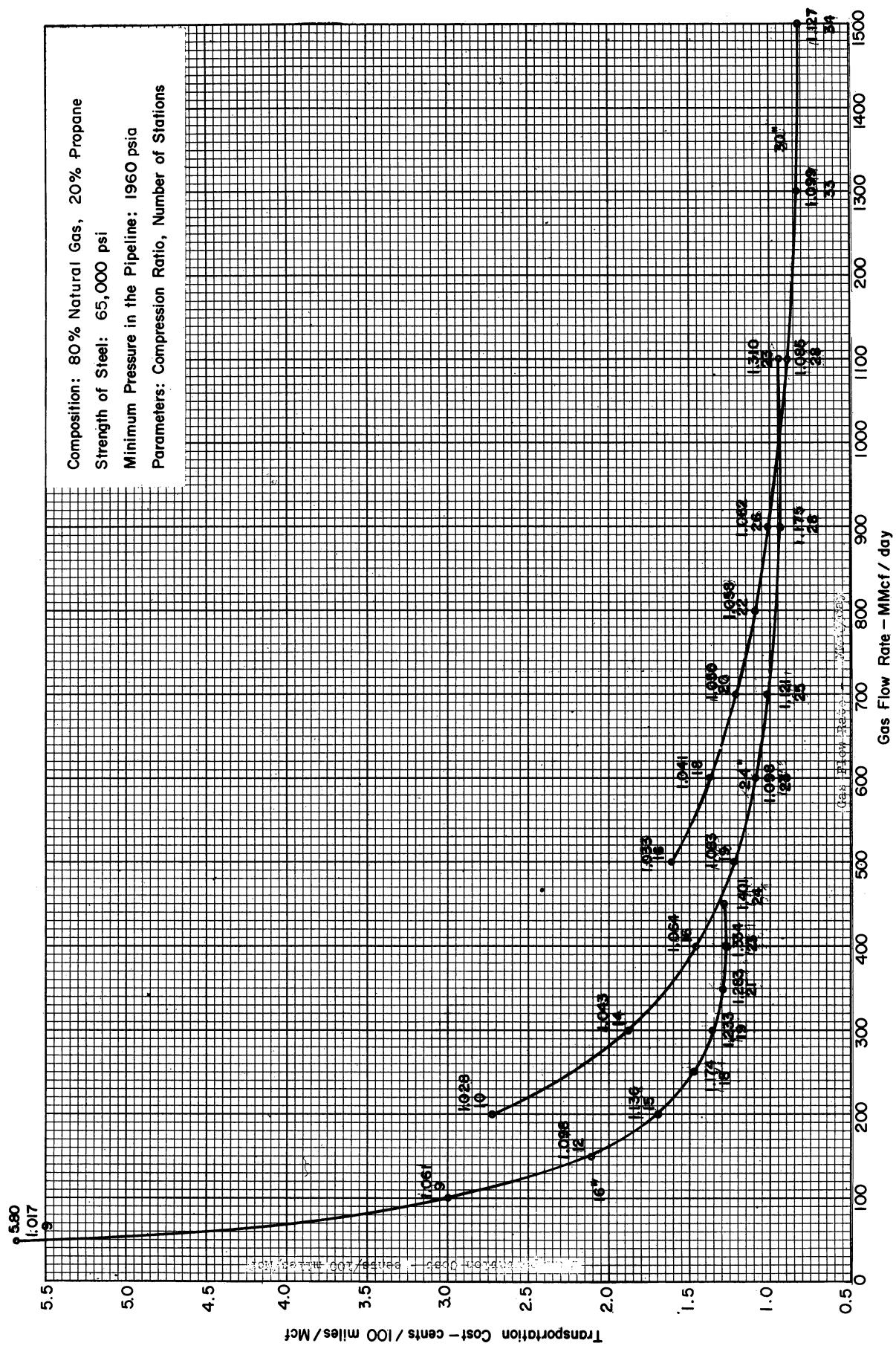


Figure 15E-22. Cost of Transportation versus Flow Rate

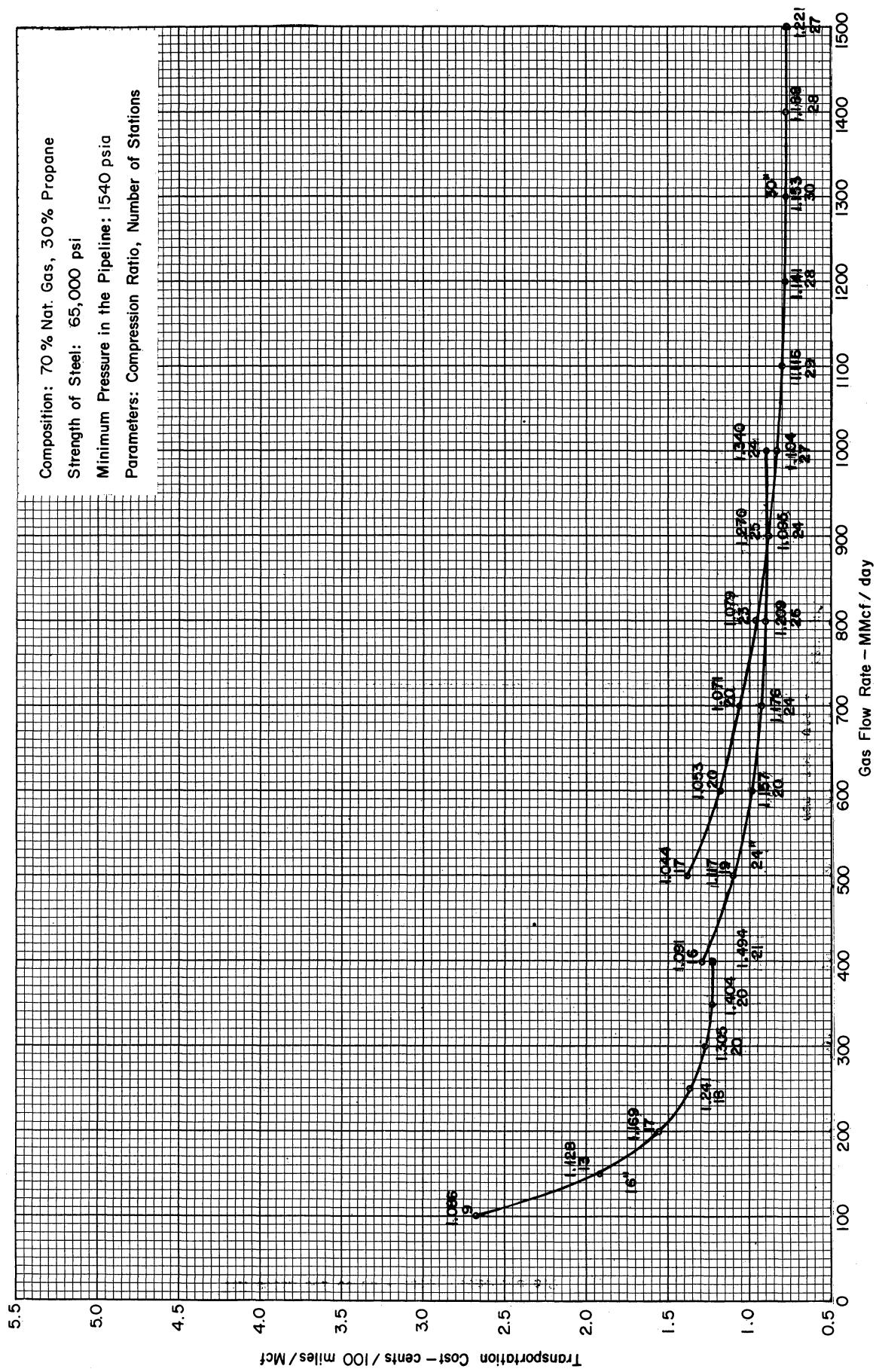


Figure 15E-23. Cost of Transportation versus Flow Rate

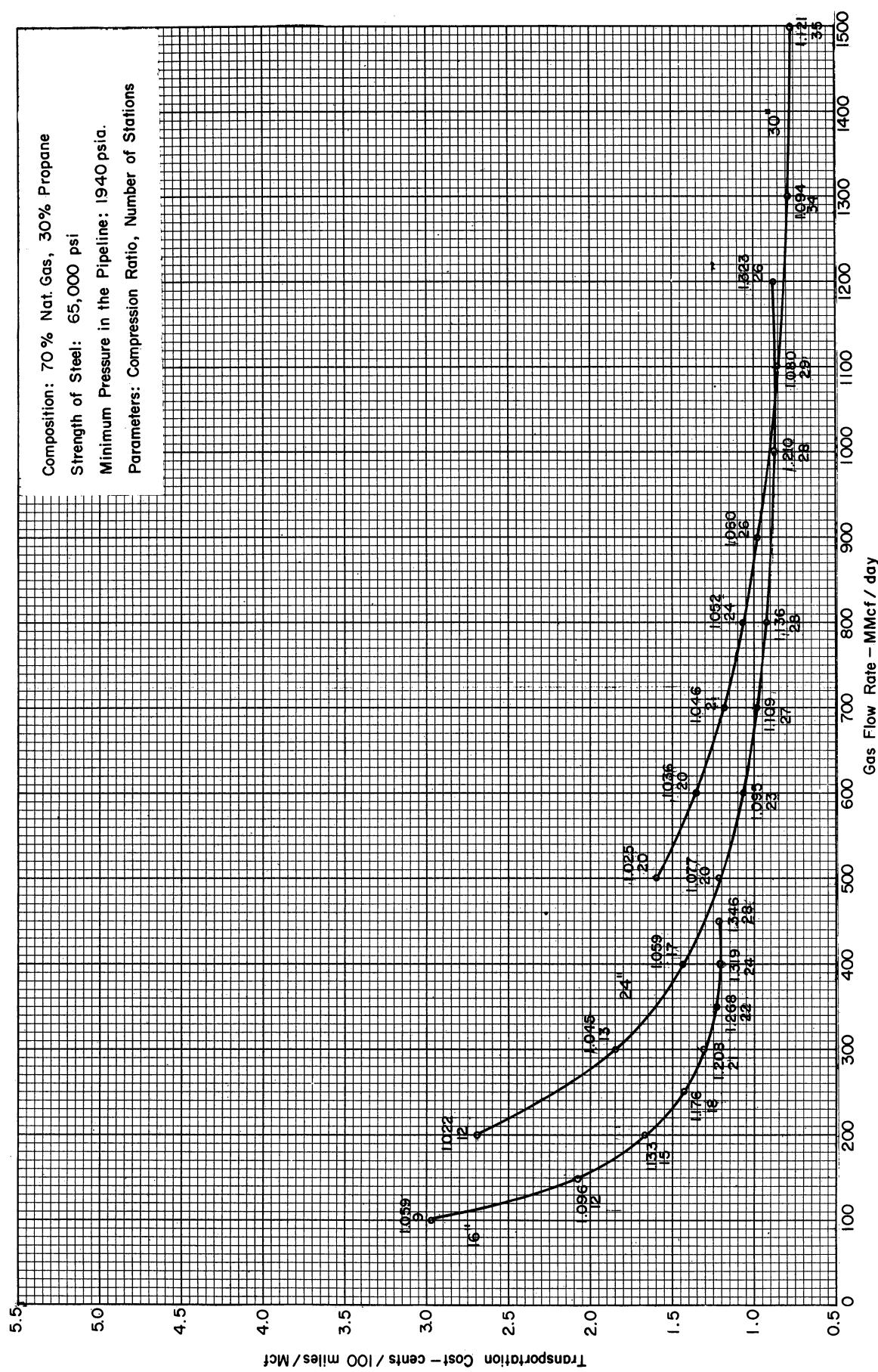


Figure 15E-24. Cost of Transportation versus Flow Rate

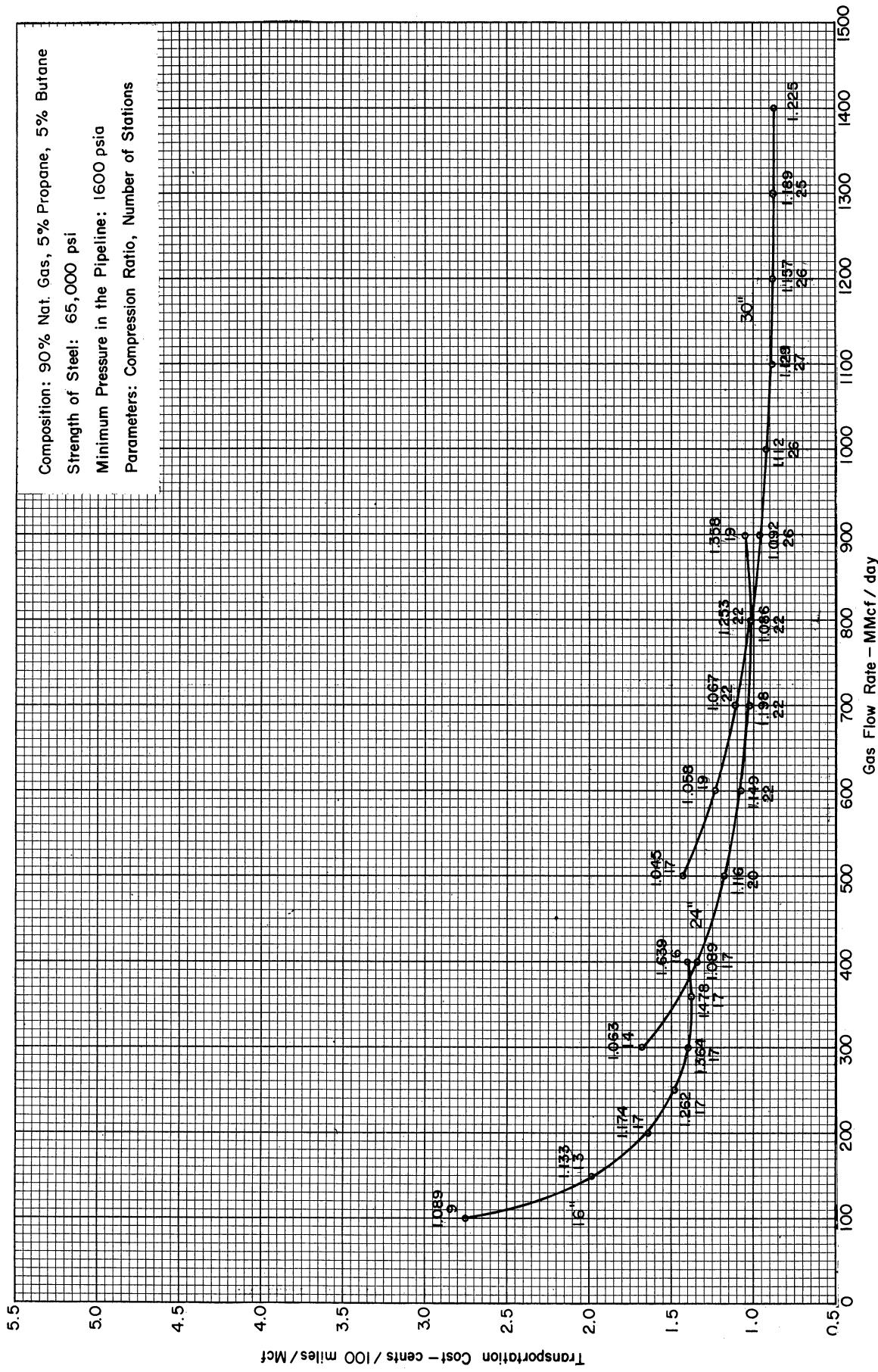


Figure 15E-25. Cost of Transportation versus Flow Rate

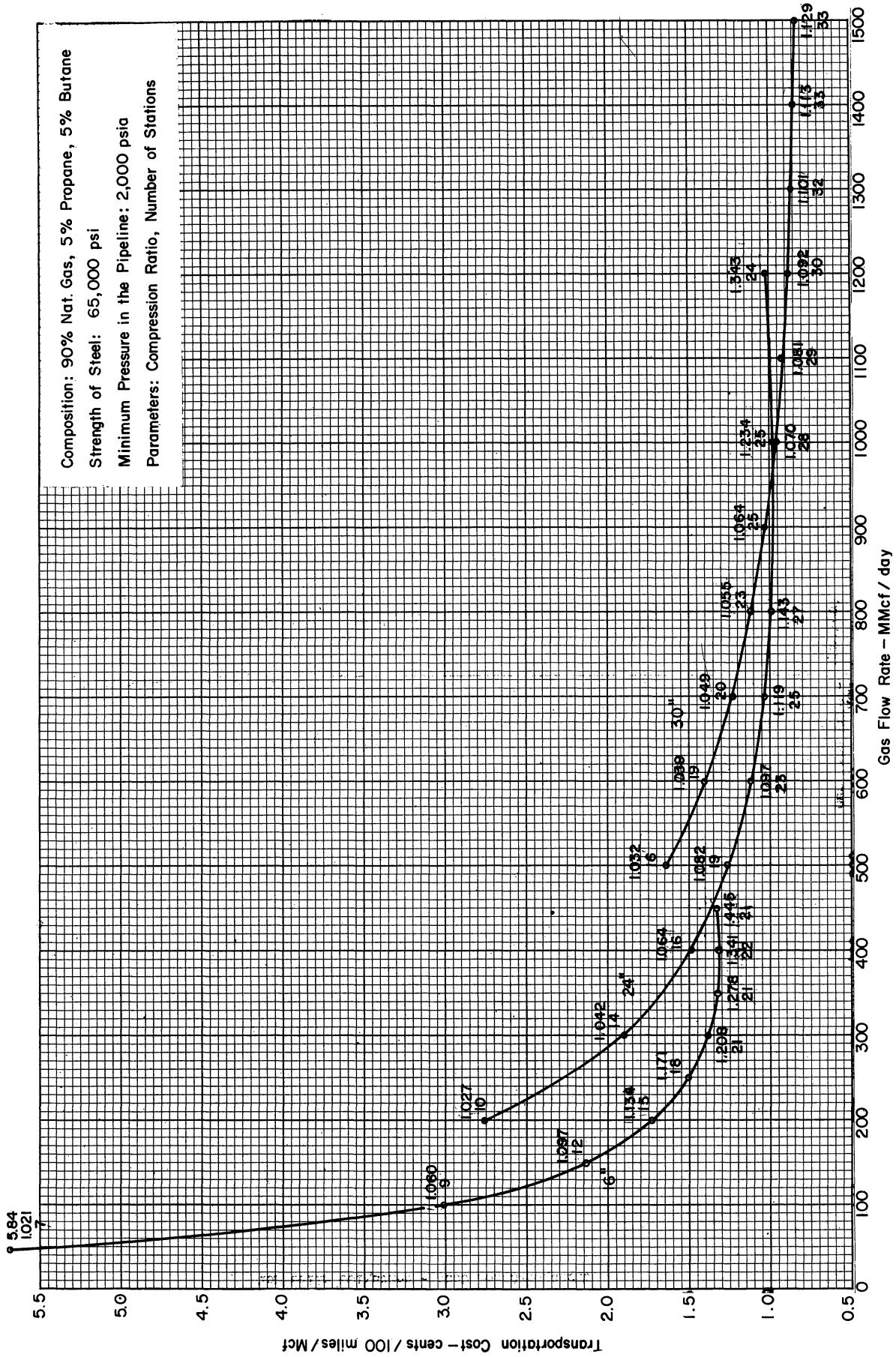


Figure 15E-26. Cost of Transportation versus Flow Rate

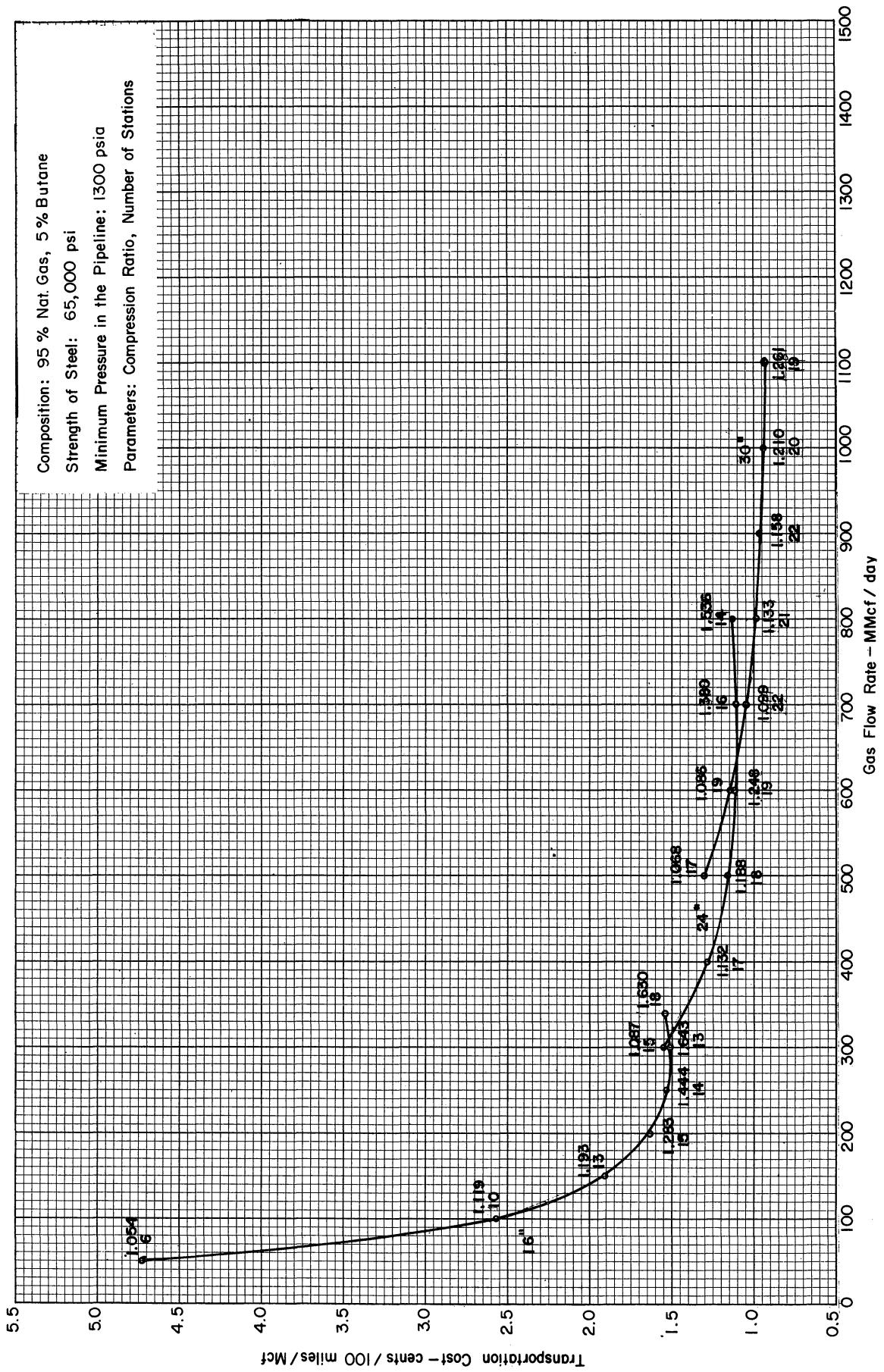


Figure 15E-27. Cost of Transportation versus Flow Rate

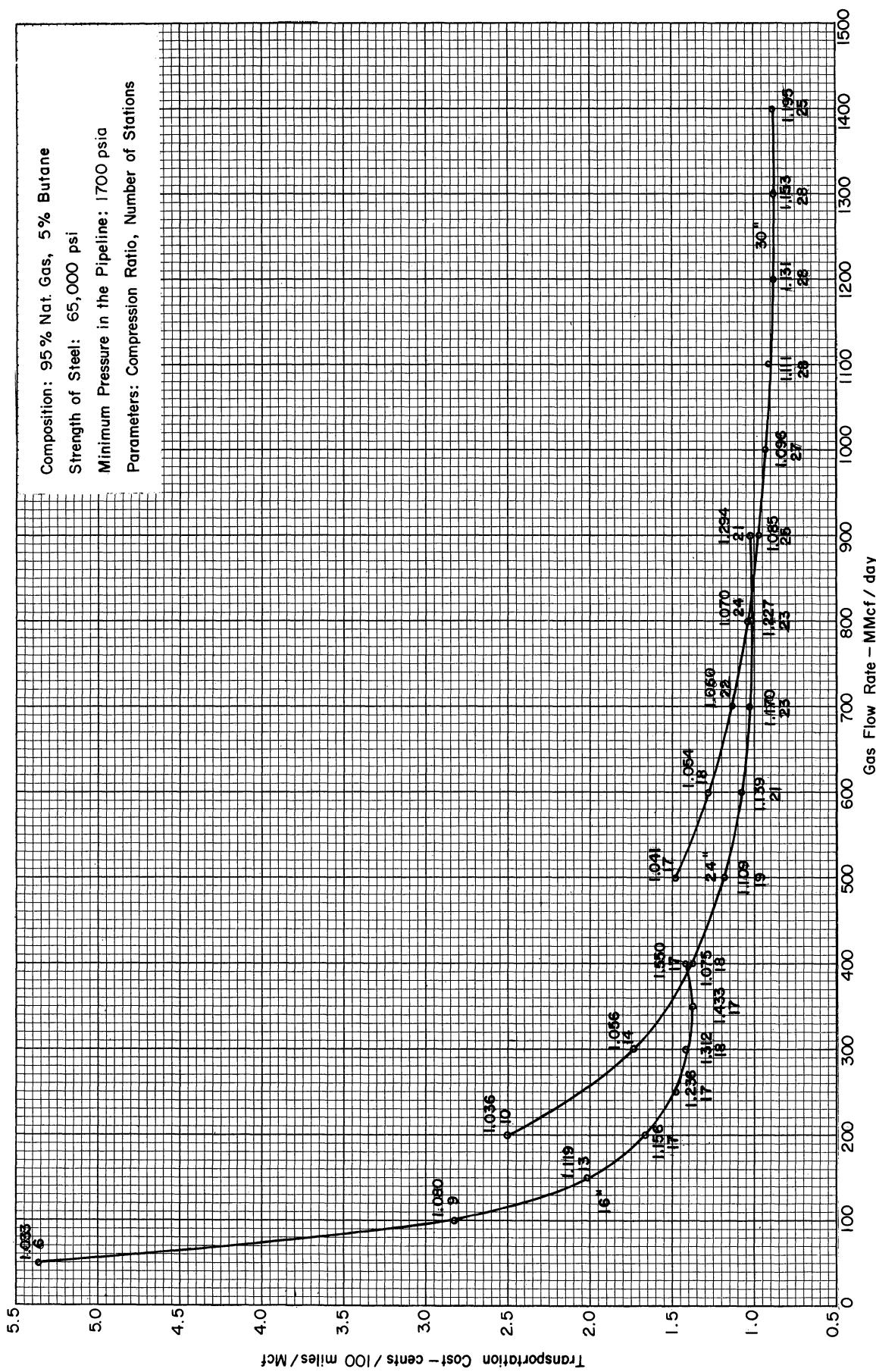


Figure 15E-28. Cost of Transportation versus Flow Rate

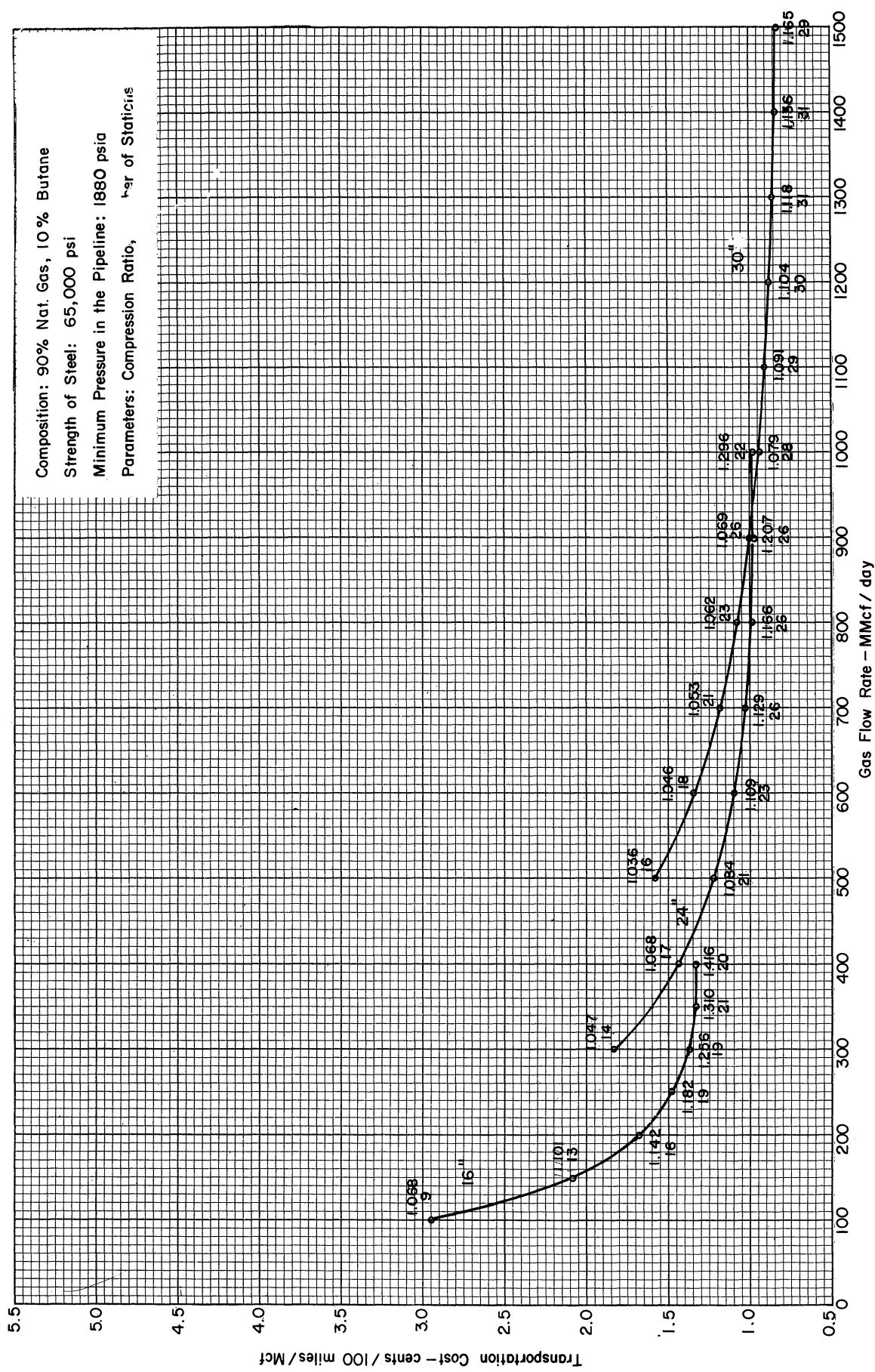


Figure 15E-29. Cost of Transportation versus Flow Rate

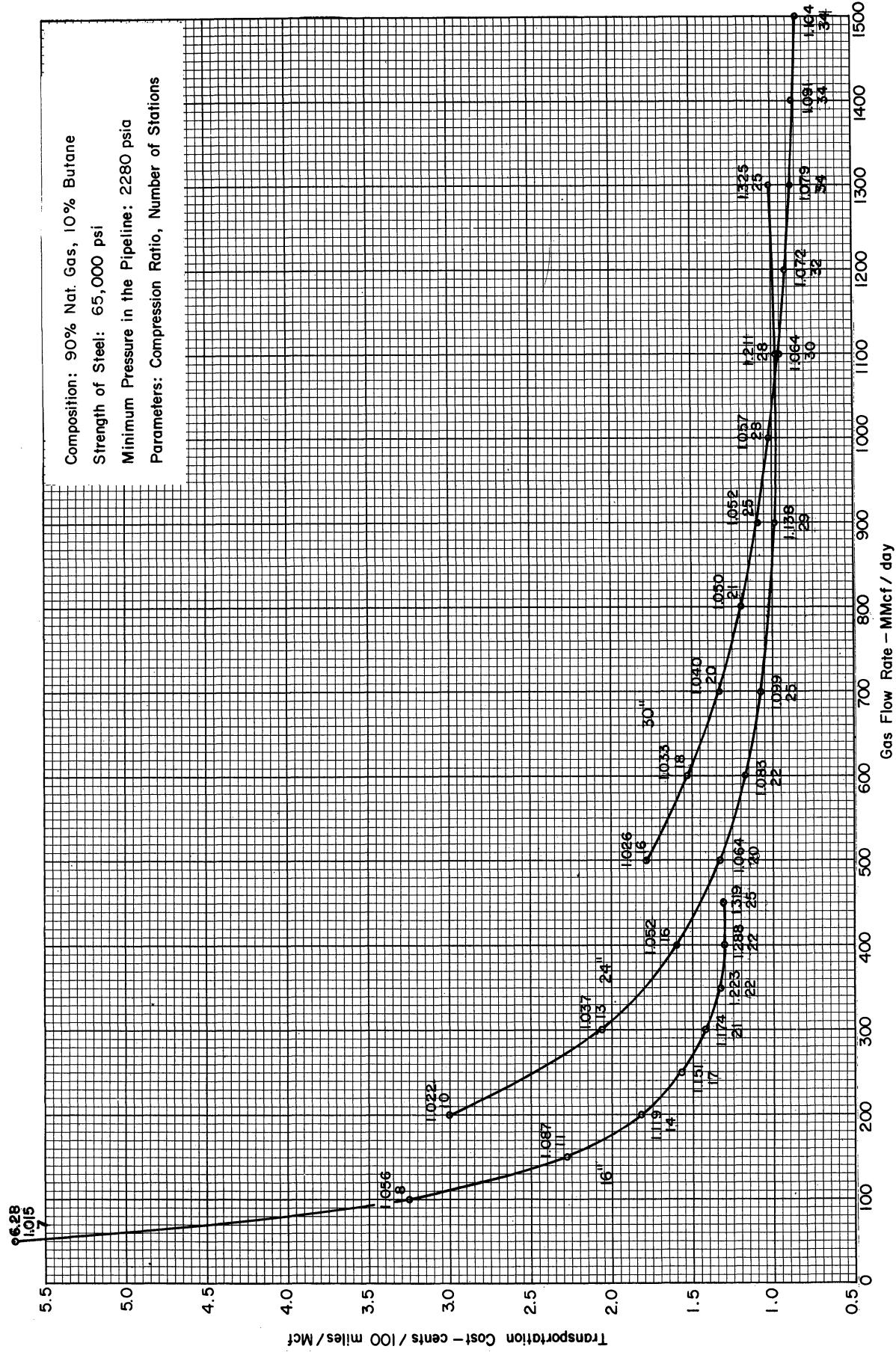


Figure 15E-30. Cost of Transportation versus Flow Rate

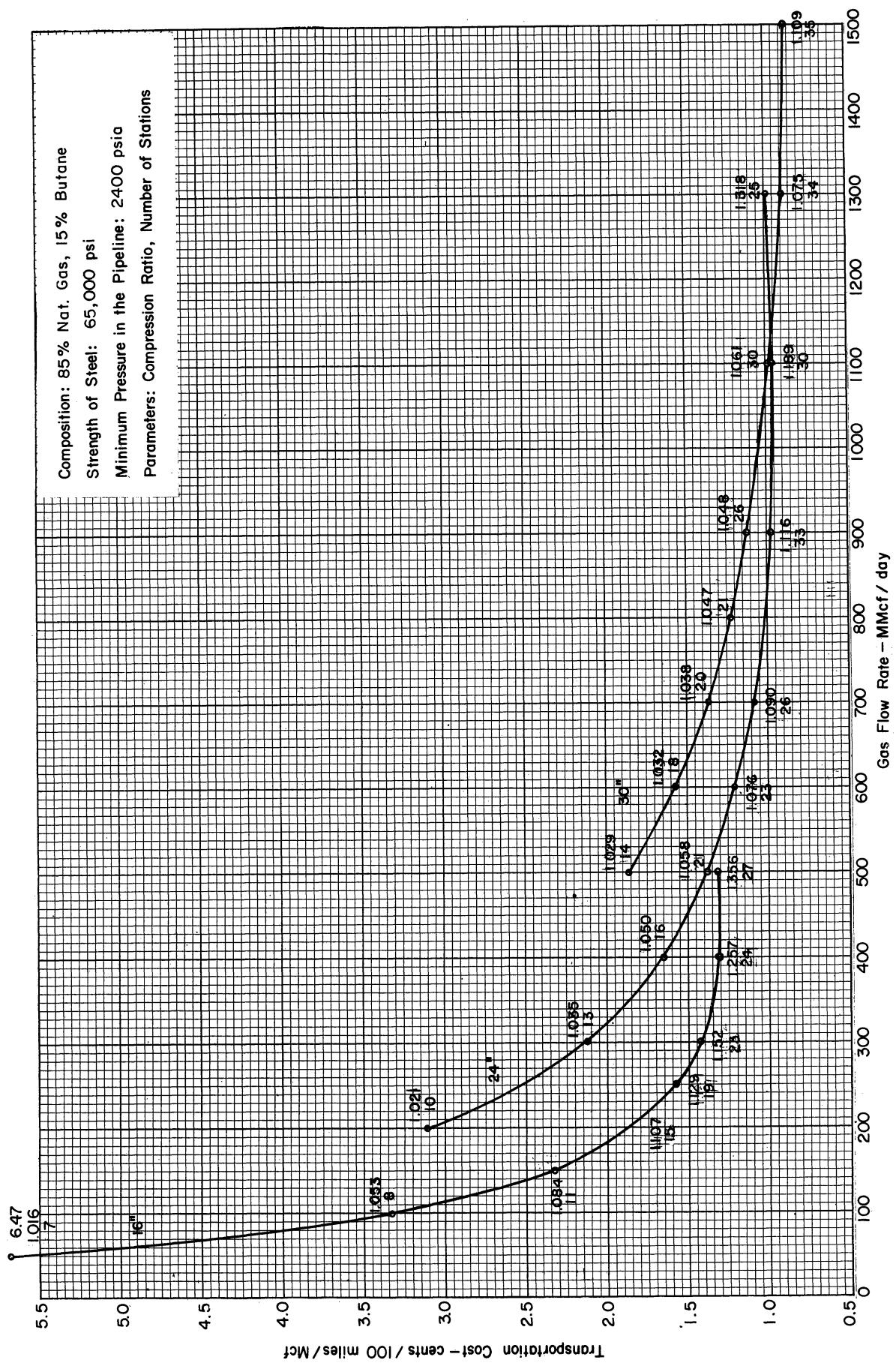


Figure 15E-31. Cost of Transportation versus Flow Rate

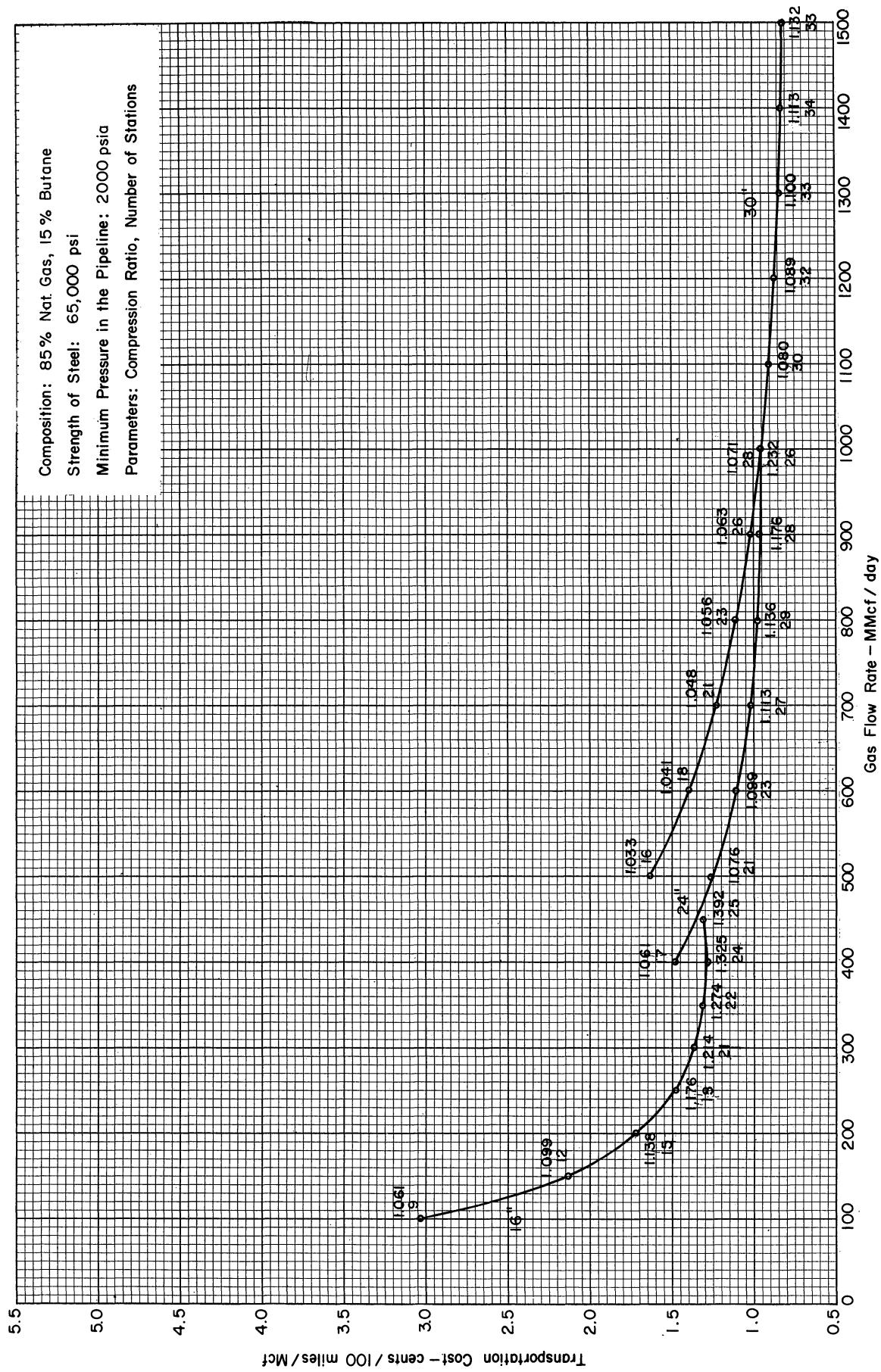


Figure 15E-32. Cost of Transportation versus Flow Rate

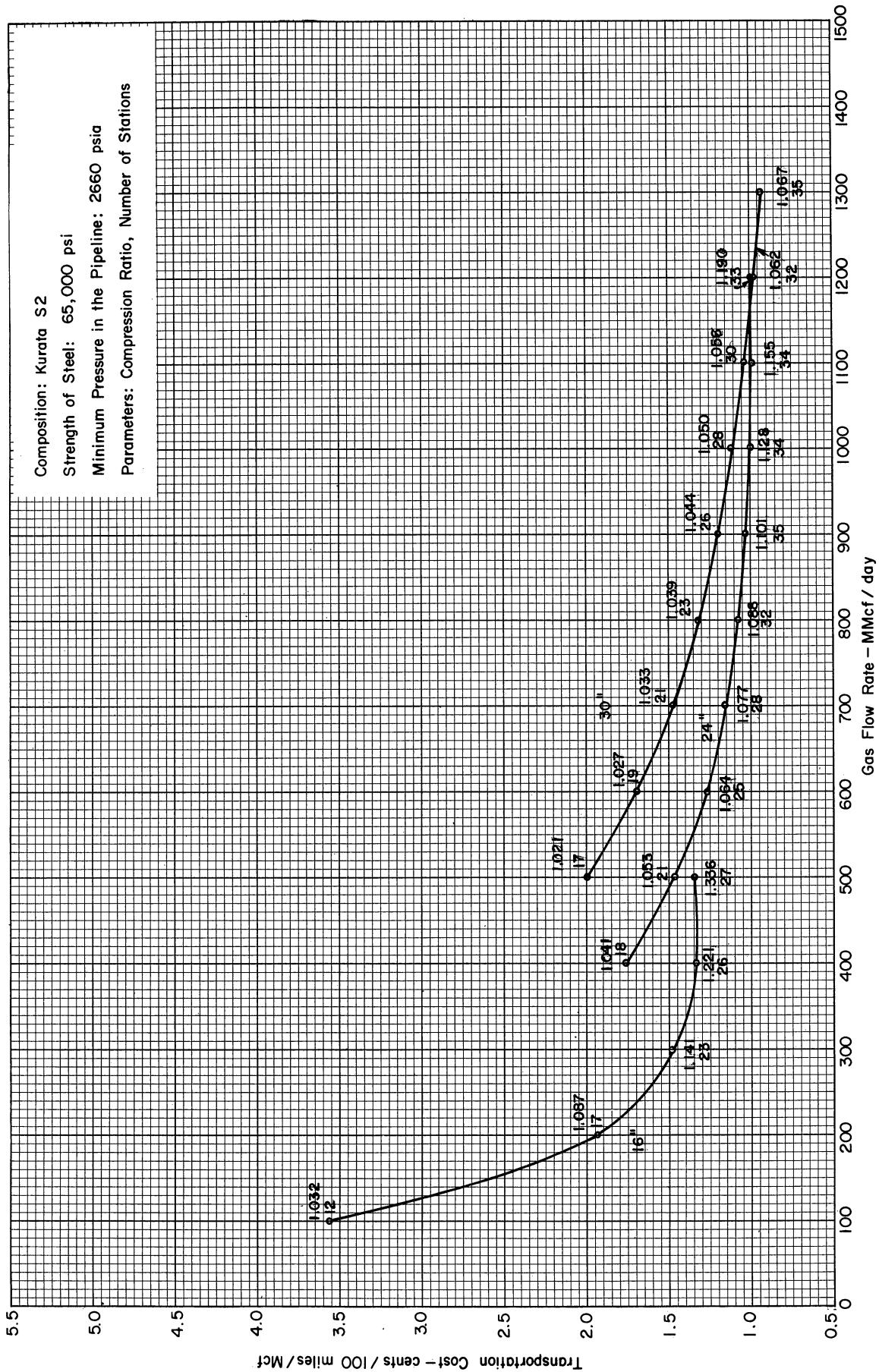


Figure 15E-33. Cost of Transportation versus Flow Rate

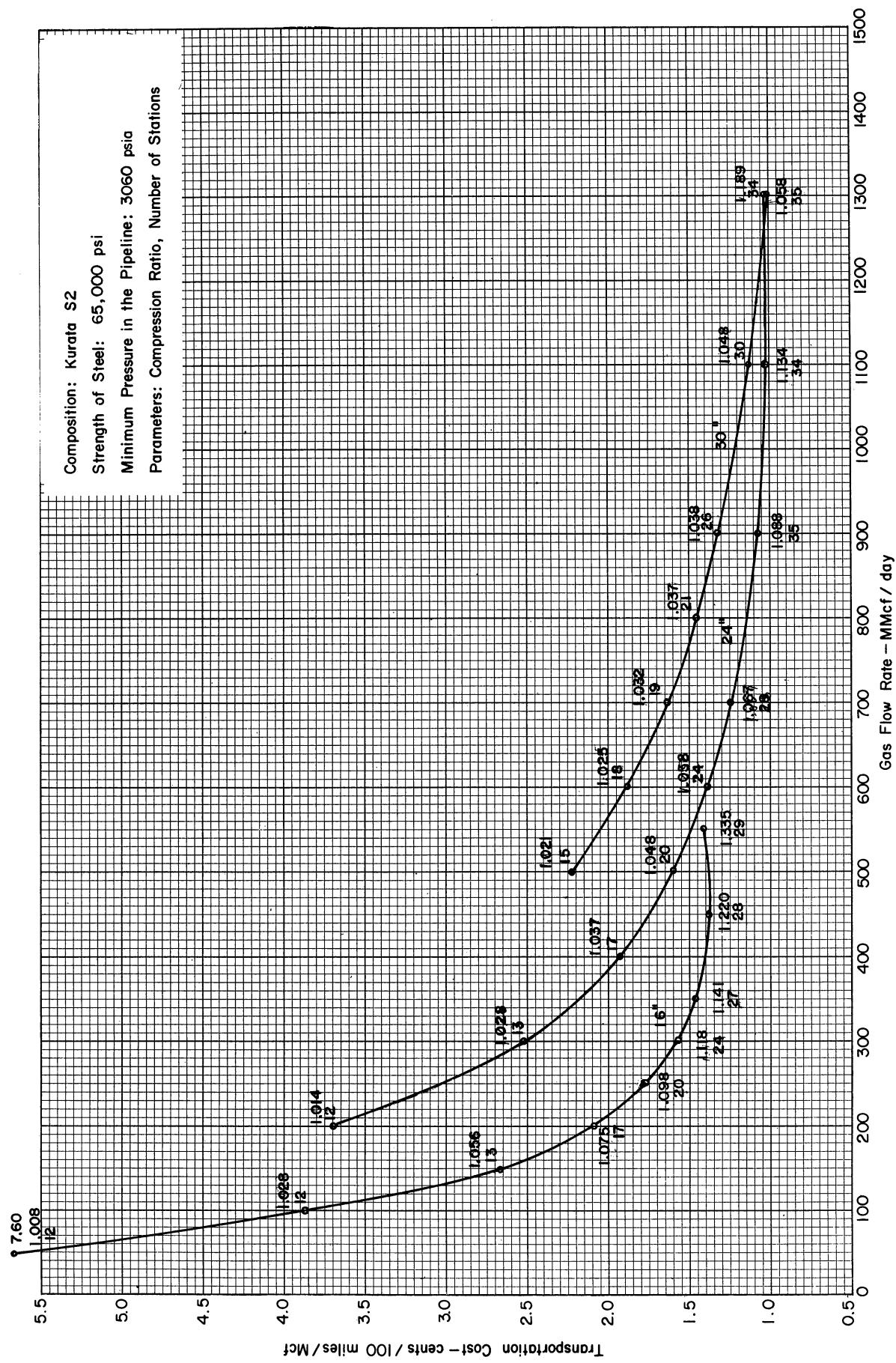


Figure 15E-34. Cost of Transportation versus Flow Rate

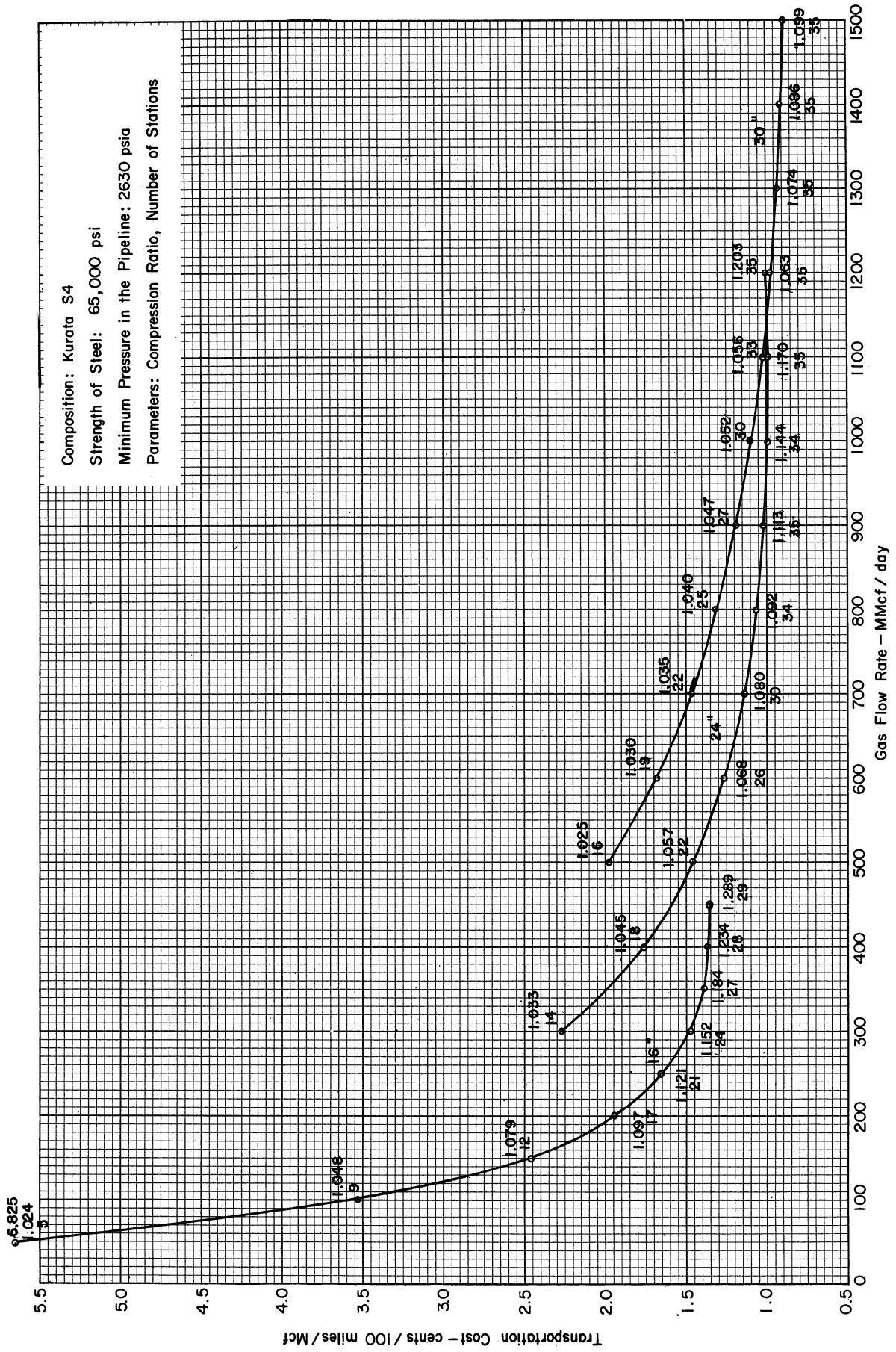


Figure 15E-35. Cost of Transportation versus Flow Rate

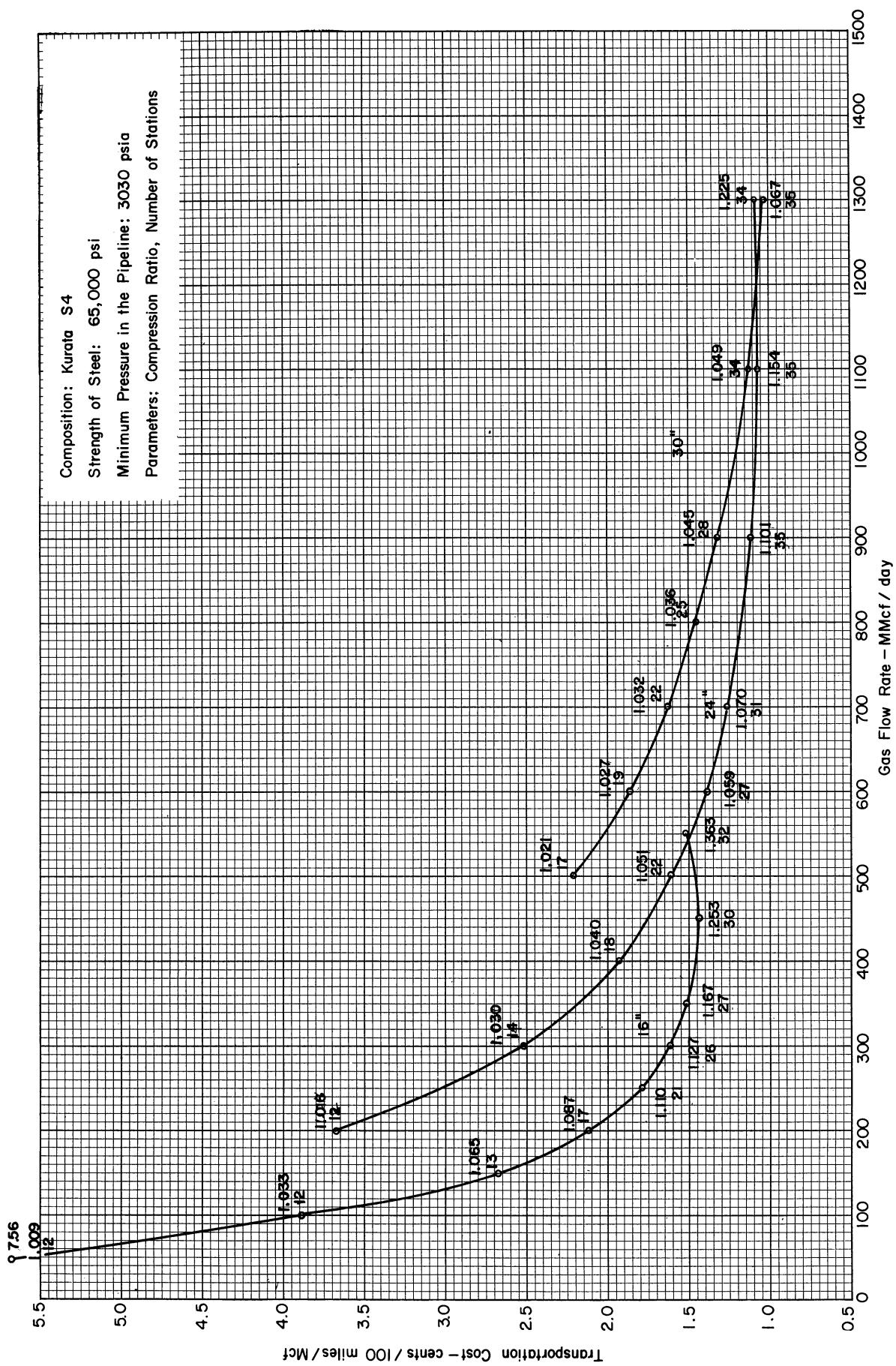


Figure 15E-36. Cost of Transportation versus Flow Rate

APPENDIX F

EXAMPLES OF SAREM'S METHOD FOR COMPUTING COMPRESSIBILITY FACTOR

Example of the Use of Subroutine ZFAC

```

TEST PROGRAM FOR SUBROUTINE ZFAC
THIS PROGRAM CAN ACCEPT MIXTURES CONTAINING NITROGEN,
METHANE TO N-HEXANE AND THREE ADDITIONAL COMPONENTS
WITH KNOWN TC, PC, AND MOL. WT.
ZERO. ( FN2, FC02, FCH4, FC2H6, FC3H8, FIC4H4, *C01
1   FIC5H2, FNC5H2, FNC6H4, FX, FY, FZ, TX, TY, TZ, PX, PY,
2   PZ, MX, MY, MZ )

PRINT FORMAT TITLE
VECTOR VALUES TITLE= $1H1, T1G, H*TEMP.*, T25, H*PRESS.*, T4C, H*TC*, -T
1   52, H*PC*, T64, H*Z, CALCULATED*, T86, H*FROM Z CHART*, T1G
2   6, H*G* /$

READ DATA
PSUM= FX*PX+ FY*PY + FZ*PZ
TSUM= FX*T*X+ FY*T*Y + FZ*T*Z
MSUM= FX*M*X+ FY*M*Y + FZ*M*Z
TC= TSUM+ FN2*TN2+ FC02* TC02+ FCH4* TCH4+ FC2H6* TC2H6+
1   FC3H8* TC3H8+ FIC4H4*TIC4H4+ FNC4H4* TNC4H4+ FIC5H2*TIC5H2
2   + FNC5H2* TNC5H2+ FNC6H4* TN6H4
PC= PSUM+ FN2*PN2+ FC02* PC02+ FCH4* PCH4+ FC2H6* PC2H6+
1   FC3H8* PC3H8+ FIC4H4* PIC4H4+ FNC4H4* PNC4H4+ FIC5H2*PIC5H2
2   + FNC5H2* PNC5H2+ FNC6H4* PNC6H4
MWT=MSUM+ FN2*MN2+ FC02* MC02+ FCH4* MCH4+ FC2H6* MC2H6+
1   FC3H8* MC3H8+ FIC4H4*MIC4H4+ FNC4H4* MNC4H4+ FIC5H2*MIC5H2
2   + FNC5H2* MNC5H2+ FNC6H4* MN6H4
G= MWT/ 29.
ZFAC.(T, P, TC, PC, G, Z)
PRINT FORMAT RESULT, T, P, TC, PC, Z, Z1 , G
VECTOR VALUES RESULT= $1H , T6, F10.2,T21,F1C.2,T36,F1D.2,T48,F10.2,T62,
1   F7.4, T84,WF7.4 , T194, F7.4*$

TRANSFER TO START
END OF PROGRAM

```

CALL -ACTION OF COMPRESSIBILITY FACTOR(Z) BY US' JF

LEGENDRE POLYNOMIALS.

REF. SAREM, OIL AND GAS JOURNAL, SEP. 18, 1961, PAGE 118.

EXTERNAL FUNCTION (T, P, TC, PC, G, Z)

ENTRY TO ZFAC.

DIMENSION PM(10), PN(10)

INTEGER M,N

DIMENSION AI(G .0. 5)*(0 .0. 5)

VECTOR VALUES A(1) = 2.1433504, J.33123524, C.10572871,

1 -0.52184 40.0.019703985, -J.06530959, C.083176184,

2 -C.13403614, -0.050393654, J.044312146, -C.026383354,

3 C.089178330, -0.021467042, J.066882961, C.0650924798

4 , -C.019329465, 5.019262143, -0.010894821, -Q.00087140318

5 , -Q.027174261, C.010551336, C.0058972516, -C.01153539, 0

6 .009559389, C.0042846283, C.0088512291, -Q.0073181933,

8 C.0015366676, C.0042910089, -C.0060114017, -C.0016595343,

9 -0.002152 929, C.0026959963, -0.0028326809, -0.00081302526,

1 0.0031175170

WHENEVER TC .L. 0.01

TC= G*100./J.32 + 171

PC= -G*25./J.545 + 699.7

END OF CONDITIONAL

PR= P/PC

TR= T/TC

X= (2.*PR-15.)/14.8

Y= (2.*TR-4.)/ 1.9

PM(G)= 0.7571068

PN(G)= PM(G)

PM(1)= 1.224745*X

PM(2)= C.7905695 *(3.*X*X-1.)

PN(2)= C.7905695 *(3.*Y*Y-1.)

PM(3)= 0.9354145*(5.*X.*P. 3. -3.*Y*)

PN(3)= 0.9354145*(5.*Y.*P. 3. -3.*Y*)

PM(4)= 0.265165*(35.*X.*P. 4.- 35.*X*X+3.)

PN(4)= 0.265165*(35.*Y.*P. 4.- 35.*Y*Y+3.)

PM(5)= C.293151*(63. *X.*P. 5.- 70.*X.*P. 3.+ 15.*X)

PN(5)= J.293151*(63. *Y.*P. 5.- 70.*Y.*P. 3.+ 15.*Y)

PN(1)= 1.*224745*Y

Z=G.

THROUGH ZLOOP, FOR N=0,1, N.G. 5

THROUGH JLOOP, FOR M=0,1, M.G. 5

Z= Z+ A(H,M)* PM(M)* PN(N)

CONTINUE

FUNCTION RETURN

END OF FUNCTION

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Computer Output

TEMP.	PRES.	TC	PC	Z, CALCULATED	FROM Z CHART	G
480.00	1000.00	378.43	669.68	.7280	.7267	.6534
480.00	1400.00	378.43	669.68	.6350	.6453	.6534
480.00	1800.00	376.43	669.68	.6000	.6181	.6534
480.00	2200.00	378.43	669.68	.6030	.6175	.6534
480.00	2600.00	378.43	669.68	.6369	.6320	.6534
480.00	3000.00	378.43	669.68	.6708	.6720	.6534
480.00	3400.00	378.43	669.68	.7515	.7720	.6534
500.00	1000.00	378.43	669.68	.6948	.6948	.6534
500.00	1400.00	378.43	669.68	.6450	.6450	.6534
500.00	1800.00	378.43	669.68	.6663	.6450	.6534
500.00	2200.00	378.43	669.68	.6605	.6400	.6534
500.00	2600.00	378.43	669.68	.6726	.6584	.6534
500.00	3000.00	378.43	669.68	.6986	.6886	.6534
520.00	1000.00	378.43	669.68	.7886	.8020	.6534
520.00	1400.00	378.43	669.68	.7369	.7390	.6534
520.00	1800.00	378.43	669.68	.7084	.7000	.6534
520.00	2200.00	378.43	669.68	.6993	.6850	.6534
520.00	2600.00	378.43	669.68	.7063	.6950	.6534
520.00	3000.00	378.43	669.68	.7261	.7200	.6534
540.00	1000.00	378.43	669.68	.8193	.8300	.6534
540.00	1400.00	378.43	669.68	.7726	.7780	.6534
540.00	1800.00	378.43	669.68	.7451	.7350	.6534
540.00	2200.00	378.43	669.68	.7343	.7230	.6534
540.00	2600.00	378.43	669.68	.7377	.7270	.6534
540.00	3000.00	378.43	669.68	.7530	.7460	.6534

* ALL INPUT DATA HAVE BEEN PROCESSED.
AT LOC 16127

Z-factor equation developed for use in digital computers

Only 36 coefficients need be stored to permit a digital computer to generate Z factors. Capacity of small computers is not taxed, and mean deviation is as low as 0.004

NATURAL-GAS and reservoir-engineering calculations involving the compressibility factor Z can best be handled in a computer program if the Z factors can be generated internally by a digital computer.

An accurate method of getting Z factors from a computer by use of a 20-by-20 input table was recently developed by Gray and Sims.¹ But for small and medium-sized computers the memory storage needed for this table lookup method seriously limits the scope of related programs. What is needed is a method that will supply the Z factors without using much storage capacity in a small computer.

The solution. To solve this problem, an equation requiring storage of only 36 coefficients was fitted to a chart in which Z was expressed as a function of p_r and T_r . Two sets of Legendre polynomials were used and the coefficients were found by the method of least squares.

Accuracy of such an approximation depends on the degree of the polynomials and the magnitude of the intervals chosen for fitting, among other things. For convenience and desired accuracy, the compressibility chart was tabulated at intervals of 0.1 on p_r and T_r for values of p_r between 0.1 and 14.9, and values of T_r between 1.05 and

2.95. Polynomials up to fifth degree were used.

The general equation is:

$$Z = \sum_{m=0}^5 \sum_{n=0}^5 A_{nm} P_m(x) P_n(y)$$

Where:

$$x = \frac{2p_r - 15}{14.8}$$

$$y = \frac{2T_r - 4}{1.9}$$

P_m , P_n are Legendre polynomials of degree m and n , respectively.

$$A_{nm} = \frac{\int_x \int_y Z(x,y) P_m(x) P_n(y) dy dx}{\int_x \int_y P_m^2(x) P_n^2(y) dy dx}$$

Expressions for the Legendre polynomials of degree 0-5 are given in Table 1. Calculated coefficients, obtained by numerical integration from the Z factor chart, are listed in Table 2.

TABLE 1—LEGENDRE POLYNOMIALS OF DEGREE 0-5

$$P_0(x) = 0.7071068$$

$$P_1(x) = 1.224745x$$

$$P_2(x) = 0.7905695 (3x^2 - 1)$$

$$P_3(x) = 0.9354145 (5x^3 - 3x)$$

$$P_4(x) = 0.265165 (35x^4 - 30x^2 + 3)$$

$$P_5(x) = 0.293151 (63x^5 - 70x^3 + 15x)$$



BY A. M. SAREM

... research engineer in the production research department of Sinclair Research, Inc. He joined Sinclair in 1954 and since that time has done research in the field of fluid flow in porous media and phase behavior of hydrocarbons and associated fluids. He holds BS and MS degrees from University of Tulsa where he has also done post-graduate work in mathematics.

Mean deviation between Z factors calculated by the method shown here and those appearing in the original chart was found to be acceptable. On the boundaries of the chart, where $T_r = 1.05$ and 2.95, and $p_r = 0.1$ and 14.9, 53 points showed a mean deviation of 0.013. For 300 points scattered uniformly throughout the chart, mean deviation was found to be 0.004.

References

1. Gray, E. H., and Simms, H. L., "Z Factor Determination in Digital Computers"; The Oil and Gas Journal, July 20, 1959, p. 80.
2. Poettman, F. H., and Carpenter, P. G., "The Multiphase Flow of Gas, Oil, and Water Through Vertical Flow Strings with Application to the Design of Gas-Lift Installations"; API Drilling & Production Practices, (1952), p. 257.
3. Brown, G. G., Katz, D. L., Oberfell, G. G., Alden, R. G., "Natural Gasoline and the Volatile Hydrocarbons"; Section One Sponsored by NGAA, Tulsa, (1948), p. 38.

TABLE 2—VALUES OF THE COEFFICIENTS, A_{mn}

	$m=0$	$m=1$	$m=2$	$m=3$	$m=4$	$m=5$
$n=0$	2.1433504	0.083176184	-0.021467042	-0.00087140318	0.0042846283	-0.0016595343
$n=1$	0.33123524	-0.13403614	0.066880961	-0.027174261	0.0088512291	-0.0021520929
$n=2$	0.10572871	-0.050393654	0.0050924798	0.010551336	-0.0073181933	0.0026959963
$n=3$	-0.052184040	0.044312146	-0.019329465	0.0058972516	0.0015366676	-0.0028326809
$n=4$	0.019703980	-0.026383354	0.019262143	-0.01153539	0.0042910089	-0.00081302526
$n=5$	-0.0053095900	0.0089178330	-0.010894821	0.009559389	-0.0060114017	0.0031175170

APPENDIX G

VISCOSITY DATA USED TO REPRESENT CORRELATION

The points on the curves of Figs. 11 and 12 are listed in matrix form for interpolation by TAB to obtain viscosity at any reduced temperature, pressure, and molecular weight.

```
V(1)= 36.9, 38., 39.1, 40., 41.1, 42.1, 43.2, 44.1, 45.4, 46.1,  
32.4, 34.1, 36., 38.1, 40., 42.3, 44.8, 46.4, 48.2, 50.,  
29., 31., 33.9, 37., 40.8, 44.2, 48., 51., 54.1, 57.,  
26., 29.2, 34., 38.3, 43.2, 47.8, 51.8, 55.2, 59., 62.3,  
24.2, 29., 34.1, 40.2, 46.2, 52., 57., 62., 66.3, 70.8,  
23., 30., 39.8, 48.8, 57.2, 64.8, 72., 79., 85.9, 92.,  
23., 34., 48.1, 60.6, 71.3, 80., 87.8, 95.1, 102.1, 108.3,  
26., 42.2, 63., 77.8, 89., 98.4, 108., 116., 124.1, 130.3  
52., 78., 95., 101.6, 116., 124., 132., 140., 148., 152.,  
90., 102., 113., 123., 132., 142., 151., 159., 163., 171.,  
115., 126., 139., 149., 156., 163., 172., 180., 186., 192.,  
135., 147., 158., 168., 178., 186., 195., 202., 210., 215.,  
160., 170., 181., 192., 202., 212., 222., 231., 240., 248.,  
186., 200., 210., 219., 228., 238., 247., 256., 265., 273.,  
222., 233., 246., 256., 265., 273., 282., 290., 300., 305.,  
265., 278., 290., 300., 310., 320., 330., 340., 350., 360.  
V1(1)= 3., 2.5, 2., 1.7, 1.5, 1.3, 1.2, 1.1, 1.,  
.95, .90, .85, .75, .7, .65  
MT(1)=15., 18., 21., 24., 27., 30., 33., 35.  
KS(1)= 1.3, 1.205, 1.14, 1.08, 1.04, 1.008, 1.004, 1.  
PRD(1)= 1., 2., 3., 4., 5., 6., 7., 8., 9., 10.
```

NP=10

APPENDIX H

DATA FOR COMPRESSIBILITY FACTOR OF METHANE-PROPANE SYSTEM IN MATRIX FORM

$PP(1) = 200\cdot, 400\cdot, 600\cdot, 800\cdot, 1000\cdot, 1250\cdot, 1500\cdot, 1750\cdot, 2000\cdot,$
 $2250\cdot, 2500\cdot, 2750\cdot, 3000\cdot, 3500\cdot, 4000\cdot, 4500\cdot, 5000\cdot$
 $VFM(1) = 8\cdot37, 2\cdot23, 1\cdot287, 1\cdot262, 1\cdot26, 1\cdot212, 1\cdot241,$
 $1\cdot238, 1\cdot227, 1\cdot217, 1\cdot213, 1\cdot206, 1\cdot198, 1\cdot184, 1\cdot174, 1\cdot164, 1\cdot154,$
 $13\cdot06, 5\cdot26, 1\cdot917, 1\cdot25, 1\cdot251, 1\cdot208, 1\cdot219, 1\cdot213, 1\cdot201, 1\cdot192,$
 $1\cdot182, 1\cdot173, 1\cdot153, 1\cdot142, 1\cdot125, 1\cdot111, 1\cdot096,$
 $17\cdot77, 5\cdot63, 2\cdot963, 1\cdot809, 1\cdot266, 1\cdot236, 1\cdot215, 1\cdot2, 1\cdot185,$
 $1\cdot174, 1\cdot162, 1\cdot153, 1\cdot142, 1\cdot125, 1\cdot111, 1\cdot096, 1\cdot084,$
 $22\cdot42, 7\cdot32, 4\cdot0, 2\cdot577, 1\cdot74, 1\cdot289, 1\cdot234, 1\cdot203, 1\cdot185, 1\cdot167,$
 $1\cdot15, 1\cdot139, 1\cdot123, 1\cdot1, 1\cdot083, 1\cdot068, 1\cdot054,$
 $24\cdot14, 9\cdot02, 5\cdot04, 3\cdot34, 2\cdot341, 1\cdot559, 1\cdot317, 1\cdot258, 1\cdot222, 1\cdot186,$
 $1\cdot162, 1\cdot142, 1\cdot12, 1\cdot091, 1\cdot067, 1\cdot046, 1\cdot029,$
 $24\cdot40, 10\cdot71, 6\cdot23, 4\cdot11, 2\cdot941, 2\cdot02, 1\cdot525, 1\cdot402, 1\cdot311, 1\cdot252,$
 $1\cdot205, 1\cdot167, 1\cdot138, 1\cdot045, 1\cdot06, 1\cdot033, 1\cdot01,$
 $24\cdot75, 11\cdot71, 7\cdot11, 4\cdot87, 3\cdot54, 2\cdot49, 1\cdot868, 1\cdot632, 1\cdot459$
 $1\cdot353, 1\cdot27, 1\cdot214, 1\cdot173, 1\cdot109, 1\cdot062, 1\cdot027, 0\cdot9974$
 $VFM1(1) = 24\cdot86, 7\cdot26, 3\cdot5, 1\cdot465, 1\cdot442, 1\cdot414, 1\cdot389, 1\cdot366,$
 $1\cdot353, 1\cdot338, 1\cdot325, 1\cdot313, 1\cdot303, 1\cdot283, 1\cdot266, 1\cdot25, 1\cdot236,$
 $25\cdot8, 9\cdot2, 4\cdot59, 2\cdot5, 1\cdot482, 1\cdot431, 1\cdot399, 1\cdot364, 1\cdot345, 1\cdot326,$
 $1\cdot31, 1\cdot292, 1\cdot279, 1\cdot255, 1\cdot234, 1\cdot217, 1\cdot202,$
 $26\cdot61, 9\cdot75, 5\cdot6, 3\cdot54, 2\cdot42, 1\cdot533, 1\cdot456, 1\cdot401, 1\cdot363, 1\cdot333, 1\cdot307,$
 $1\cdot286, 1\cdot266, 1\cdot235, 1\cdot213, 1\cdot191, 1\cdot173,$
 $27\cdot29, 12\cdot11, 6\cdot45, 4\cdot32, 3\cdot09, 2\cdot21, 1\cdot615, 1\cdot497, 1\cdot424, 1\cdot373,$
 $1\cdot3333, 1\cdot3, 1\cdot273, 1\cdot23, 1\cdot197, 1\cdot17, 1\cdot147,$
 $27\cdot88, 12\cdot81, 7\cdot71, 5\cdot9, 3\cdot67, 2\cdot63, 1\cdot946, 1\cdot704, 1\cdot55, 1\cdot453, 1\cdot388,$
 $1\cdot34, 1\cdot301, 1\cdot241, 1\cdot198, 1\cdot162, 1\cdot134,$
 $28\cdot38, 13\cdot33, 8\cdot29, 5\cdot78, 4\cdot28, 3\cdot06, 2\cdot445, 2\cdot033, 1\cdot767, 1\cdot594,$
 $1\cdot494, 1\cdot415, 1\cdot356, 1\cdot268, 1\cdot208, 1\cdot161, 1\cdot125,$
 $28\cdot8, 13\cdot75, 8\cdot73, 6\cdot24, 4\cdot75, 3\cdot6, 2\cdot861, 2\cdot366, 2\cdot039, 1\cdot811,$
 $1\cdot652, 1\cdot531, 1\cdot441, 1\cdot317, 1\cdot234, 1\cdot172, 1\cdot127$
 $COM1(1) = 0\cdot2, 0\cdot3, 0\cdot4, 0\cdot5, 0\cdot6, 0\cdot7, 0\cdot8, 0\cdot9$

APPENDIX I

SUBROUTINE FNGFRI FOR CALCULATION OF MOODY FRICTION FACTOR
BY COLEBROOK RELATIONSHIP

EXPLANATION OF THE EXTERNAL FUNCTION FNGFRI

This external function solves by trial and error for the Moody friction factor using Colebrook relationship. It is good for both fully turbulent flow and for the flow in the transition zone. Explanation of its usage is as follows:

FNGFRI (FTB, D, EE, QB, G, MV, FT)

FTB = Set FTB = 1 for fully turbulent flow
= 0 for transition zone

D = Inside diameter of the pipe, in.

EE = Pipe roughness, in.

QB = Gas flow rate, ft³/day

G = Gas gravity (mol wt/29.0)

MV = Viscosity of the mixture, lb/ft/sec

FT = $2/\sqrt{FM}$

FM = Moody friction factor

```
$COMPILE MAD, EXECUTE, DUMP ,PRINT OBJECT          FNGFR
RSUBROUTINE FOR FANNING FRICTION FACTOR(1/SQRT(F))
RFOR FULLY TURBULANT SET FTB = 1.0, OTHERWISE SET IT ANYTHING
RFT= 1/ SQRT(F) = 2/SQRT(FM)
RREF. KATZ HANDBOOK EQUATION 7-23, PAGE 302
R
      EXTERNAL FUNCTION (FTB, D,           EE, QB, G, MU, FT)
E'O  FNGFRI.
INTEGER FTB
FT= 4.* LOG10.(D/EE)+2.28
FT2 = FT
W'R  FTB .E. 1 , T'O  FB
RE= 1.3526E-5* QB*G/(MU*D)
LOOP   FT1= FT2-4. *LOG10.(1.+ 4.67*D* FT/(EE*RE) )
W'R  .ABS. (FT1-FT)/FT .L. .001
T'O  FB
O'E
FT= FT1
T'O  LOOP
E'L
FB   F'N
E'N
```


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