

Robust Control Design for a Benchmark Problem

IT is quite possible that twenty years from now controls researchers will look back at our era as a "golden age" of control theory development. The scope and pace of current developments have reached a level beyond even the tremendous strides of the 1960s and 1970s. This special section of the JGCD, which is devoted to the treatment of a benchmark problem, clearly demonstrates the extent of ongoing activity.

As we passed, in approximately 1960, from the "classical" era to the "modern" era in feedback control, the research pendulum seemed to swing irretrievably to the state-space side of the subject. Yet the replacement of Nyquist plots with Riccati equations was to be only the first act in a drama with many subplots and unexpected twists: LQR, LQG, QFT, LQG/LTR, guaranteed cost control, H_∞ , μ , and, now, state-space H_∞ . The pace of such developments continues unabated.

The intent of this issue is to provide a glimpse of a broad spectrum of this activity. Specifically, each of the papers focuses on the same problem, namely, a two-mass system with uncertain spring constant and noncollocated sensor and actuator. In spite of its simplicity, the problem is nontrivial in that it captures both rigid body mode and flexible body mode with uncertainty. This benchmark problem has been examined in more than 45 journal and conference papers. Although it would have been easy to pose more involved extensions of this problem for this section, transparency of the problem and comparison of the various techniques would have been lost.

Besides addressing the benchmark problem, each paper provides a brief description of the technical approach. It is hoped that this combination of theoretical overview and illustrative example will be useful both to students and researchers. The section begins with an introductory paper by Wie and Bernstein, followed by a paper by Stengel and Marrison that provides a detailed comparison of a collection of earlier designs for the benchmark problem. A variety of techniques are represented by six papers, namely, minimax method (Mills and Bryson), game theoretic control (Rhee and Speyer), pole placement (Lilja and Astrom), quantitative feedback theory (Jayasuriya et al.), maximum entropy (Collins et al.), and μ synthesis (Braatz and Morari). These papers are then followed by five papers on a variety of techniques based on state-space H_∞ theory (Chiang and Safonov; Byrns and Calise; Wang et al.; Adams and Banda; and Wie et al.).

Besides the authors themselves, the success of this section is due to Bong Wie of Arizona State University and Dennis S. Bernstein of the University of Michigan. I'd like to express my appreciation for their efforts in making this unusual and valuable issue a reality. Their accomplishment continues to teach us that in spite of the advances of the past, our knowledge is but a starting point for our future endeavors.

K. Terry Alfriend
Editor-in-Chief

Benchmark Problems for Robust Control Design

Bong Wie*

Arizona State University, Tempe, Arizona 85287

and

Dennis S. Bernstein†

University of Michigan, Ann Arbor, Michigan 48109

I. Introduction

SIMPLE, yet meaningful, control problems to highlight issues in robust control design and to provide a forum for the application of a variety of robust control design methodologies are formulated in this paper. Such problems have been studied by several researchers under a variety of assumptions.¹⁻⁹ More recently, these problems were refined and addressed as benchmark problems for robust control design at the 1990 American Control Conference,¹⁰⁻¹⁶ the 1991 American Control Conference,¹⁷⁻²⁸ and the 1992 American Control Conference.²⁹⁻⁴⁴

The papers in this special section of the *Journal of Guidance, Control, and Dynamics* are concerned with the original three problems of Ref. 10, which address primarily a disturbance rejection control problem in the presence of parametric uncertainty. These problems are augmented with a new problem, called problem 4, which is concerned with a command tracking control problem in the presence of plant modeling uncertainty.²⁹ For problem 4, control input saturation limit is

specified explicitly. The previous problems 1, 2, and 3 are refined in Ref. 29; however, they are essentially the same as those presented in Ref. 10.

It is emphasized that in our problem statement certain aspects, such as parameter uncertainty with given nominal parameter values and nominal desired performance, are specified concretely, while other aspects, such as the sensor noise model, definition of settling time, measure of control effort, controller complexity, bandwidth, etc., are deliberately left vague. Each designer is thus given the opportunity to emphasize additional design tradeoffs for a realistic control design as desired.

II. Benchmark Problems

Consider the two-mass-spring system shown in Fig. 1, which is a generic model of an uncertain dynamical system with a rigid-body mode and one vibration mode.

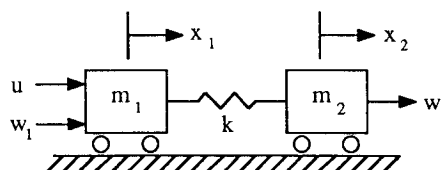


Fig. 1 Two-mass-spring system with uncertain parameters.

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*Professor, Department of Mechanical and Aerospace Engineering, Associate Fellow AIAA.

†Associate Professor, Department of Aerospace Engineering.

It is assumed that for the nominal system $m_1 = m_2 = 1$ and $k = 1$ with appropriate units and time is in units of seconds. A control force acts on body 1, and the position of body 2 is measured, resulting in a noncollocated actuator/sensor control problem.

This system can be represented in state-space form as

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -k/m_1 & k/m_1 & 0 & 0 \\ k/m_2 & -k/m_2 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1/m_1 \\ 0 \end{bmatrix} (u + w_1) + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 1/m_2 \end{bmatrix} w_2 \quad (1a)$$

$$y = x_2 + v \quad (1b)$$

$$z = x_2 \quad (1c)$$

where x_1 and x_2 are the positions of body 1 and body 2, respectively; x_3 and x_4 the velocities of body 1 and body 2, respectively; u the control input acting on body 1; y the sensor output; w_1 and w_2 the plant disturbances acting on body 1 and body 2, respectively; v the sensor noise; and z the output to be controlled (i.e., the performance variable).

Problem 1

Design a constant-gain linear feedback controller of the form

$$\dot{x}_c = A_c x_c + B_c y$$

$$u = C_c x_c + D_c y$$

with the following properties:

- 1) For a unit impulse disturbance exerted on body 1 and/or body 2, the controlled output ($z = x_2$) has a settling time of about 15 s for the nominal system with $m_1 = m_2 = k = 1$.
- 2) The closed-loop system is stable for $0.5 \leq k \leq 2.0$ and $m_1 = m_2 = 1$.
- 3) The closed-loop system is insensitive to high-frequency sensor noise.
- 4) Reasonable performance/stability robustness and reasonable gain/phase margins are achieved with reasonable bandwidth.
- 5) Reasonable control effort (e.g., peak control input) is used.
- 6) Reasonable controller complexity (e.g., controller order) is needed.

Problem 2

The same as problem 1 except in place of property 2 insert:

- 2) A stability robustness measure with respect to the three uncertain parameters m_1 , m_2 , and k (with nominal values of $m_1 = m_2 = k = 1$) is maximized.

Problem 3

The same as problem 1 except in place of property 1 insert:

- 1) For a sinusoidal disturbance with known frequency of 0.5 rad/s acting on body 1 and/or body 2, but with unknown constant amplitude and phase, the closed-loop system achieves asymptotic disturbance rejection of the controlled output z with approximately a 20-s settling time for $m_1 = m_2 = 1$ and $0.5 \leq k \leq 2.0$.

Problem 4

Design a feedback/feedforward controller for a unit-step output command tracking problem for the controlled output z with the following properties:

- 1) The control input is limited as $|u| \leq 1$.
- 2) Performance requirement: settling time and overshoot are both to be minimized.
- 3) Robustness requirement: performance robustness and stability robustness with respect to the three uncertain parameters m_1 , m_2 , and k (with the nominal values of $m_1 = m_2 = k = 1$) are both to be maximized.
- 4) If there are conflicts between properties 2 and 3, then performance vs robustness tradeoffs must be considered.

For each design (where applicable), please provide the following:

- a) Controller matrices (A_c , B_c , C_c , and D_c) and/or poles and zeros of compensators.
- b) Gain and phase margins.
- c) Time responses of x_1 , x_2 , and u for the nominal system.
- d) Evidence of ability to withstand sensor noise.
- e) Evidence of performance/stability robustness (e.g., real-parameter margin or μ).

Remark

It is again emphasized that each designer is given the opportunity to consider additional design tradeoffs for a realistic control design by including the effects of unmodeled high-frequency dynamics, actuator/sensor dynamics, bandwidth limit, time delay, etc.

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